

ENERGY ENGINEERING (MEEN 10050)
Numerical **Answers** to **Section B** Examination Questions

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Summer 2013 (2012-13 Academic Session, End of Semester Paper)

Question B1:

- (a) What are the U-values of the two walls? $U_A = 2.855 \text{ W/m}^2 \cdot \text{K}$ $U_B = 1.4235 \text{ W/m}^2 \cdot \text{K}$
- (b) What is the temperature T_2 of the air in the unheated room? $T_2 = 14.0^\circ \text{C}$
- (c) What is the temperature T_1 of the air in the heated room? $T_2 = 21.0^\circ \text{C}$

Question B2:

For H_2O at **State 1**: 200 kPa, 120.23°C , **State 2**: 10 MPa, 500°C

- (a) the specific volume from the data tables,
 $v = 0.8857 \text{ m}^3 \cdot \text{kg}$ $v = 0.03279 \text{ m}^3 \cdot \text{kg}$
- (b) the specific volume calculated using the ideal gas equation and,
 $v = 0.9085 \text{ m}^3 \cdot \text{kg}$ $v = 0.03571 \text{ m}^3 \cdot \text{kg}$
- (c) the percentage error involved in assuming ideal gas behaviour.
 $\% \text{ error} = -2.57\%$ $\% \text{ error} = +8.91\%$

Question B3

- (i) The outlet temperature of the steam ($^\circ \text{C}$), $T_{\text{exit}} = \mathbf{99.63^\circ \text{C}}$
- (ii) The change in specific enthalpy of the steam (kJ/kg), $\mathbf{878.9 \text{ kJ/kg}}$
- (iii) The mass flow rate of steam required to yield a turbine mechanical power output of 50.0 MW (Neglecting potential energy changes and any mechanical friction losses in the turbine). $\dot{m} = 5.69 \text{ kg/s}$

Summer 2012 (2011-12 Academic Session, End of Semester Paper)

Question B1:

- (a) What is the U-value of the wall? $U = 0.349 \text{ W/m}^2 \text{K}$
- (b) What is the minimum capacity of a heater for that room? $\dot{Q} = 157 \text{ W}$
- (c) What is the temperature of the external surface of the wall when the minimum temperature is observed? $T_1 = -4.62^\circ \text{C}$

Question B2:

- (1) the final volume, and $V_2 = 0.02143 \text{ m}^3$
- (2) the final temperature ($^\circ \text{C}$) $T_2 = 497.86 \text{ K} = 224.7^\circ \text{C}$

Question B3

- (A) What inlet water flow rate (in litres per minute) can be sustained if the outlet water temperature is to be 50°C at the exit from the heater? $\dot{Q} \text{ (litres/min)} = 2.3 \text{ litres/min}$
- (B) What exit temperature ($^\circ \text{C}$) would be achieved if this water flow rate were increased by 60%?
 $T_e = 33.12^\circ \text{C}$
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Summer 2011 (2010-11 Academic Session, End of Semester Paper)

Question B1:

- (a) the steady state rate of heat transfer through this glass window $\dot{Q} = 502.9 \text{ W}$
(b) the overall heat transfer coefficient $U = 5.466 \text{ W.m}^{-2}.\text{K}^{-1}$
(c) the temperature of its inner surface for a time during the day $T_2 = 2.3^\circ \text{C}$

Question B2:

- (a) The work done during this process ${}_1W_2 = -9.46 \text{ kJ}$
(b) The heat transferred. ${}_1Q_2 = -9.46 \text{ kJ}$

Question B3

- (iv) The outlet temperature of the steam ($^\circ\text{C}$), **60.06 $^\circ\text{C}$**
(v) The change in specific enthalpy of the steam (kJ.kg^{-1}) **1104.6 kJ/kg**
(vi) The change in steam kinetic energy per unit mass (kJ.kg^{-1}) -1.2 kJ/kg
(vii) The mass flow rate of steam required to yield a turbine mechanical power output of 50.0 MW (Neglecting any mechanical friction losses in the turbine) $\dot{m} = 45.31 \text{ kg/s}$
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Christmas 2011 (2010-11 Academic Session, Resit Exam Paper)

Question B1

- (a) the steady state rate of heat transfer through this glass window, $\dot{Q} = 144.3 \text{ W}$
(b) the overall heat transfer coefficient, and $U = 4.357 \text{ W.m}^{-2}.\text{K}^{-1}$
(c) the temperature of its outer surface for a time during the day $T_2 = 2.6^\circ \text{C}$

Question B2

- (i) the pressure rise (in kPa) if the air temperature in the tyre rises to 80°C under racing conditions, neglecting any changes in the internal volume, $\Delta P = P_2 - P_1 = 61.4 \text{ (kPa)}$
(ii) the mass of air in the tyre, $m_1 = 0.03564 \text{ (kg)}$
(iii) the mass of air which must be bled off to restore the pressure to 300 kPa, at 80°C .
 $\Delta m = m_1 - m_3 = 0.00606 \text{ (kg)}$

Question B3

- (i) The air temperature at exit from the duct ($^\circ\text{C}$). $T_{\text{exit}} = 31.3^\circ \text{C}$
(ii) The air velocity at exit from the duct (m/s). $\bar{V}_2 = 4.17 \text{ m/s}$
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**ENGINEERING THERMODYNAMICS & FLUID MECHANICS
(MEEN 10010)**

Numerical **Answers** to **Section B** of Examination Questions (2005-2010)

Summer 2010 (2009-10 Academic Session, End of Semester Paper)

Question B1: Overall Height of Left leg = 14.33 cm, Right leg = 15.86 cm

Question B2: (i) $V_{total} = 0.006848 \text{ m}^3$ (ii) $m_{vap} = 0.003721 \text{ kg}$

(iii) $U_1 = 259.1 \text{ kJ}$ (iv) $T_{final} = 151.86 \text{ }^\circ\text{C}$

Question B3 (1) $V_2 = 0.0535 \text{ m}^3$ (2) ${}_1W_2 = -33.61 \text{ kJ}$ (3)

$T_2 = 491.61 \text{ K} = 218.46 \text{ }^\circ\text{C}$

Summer 2009 (2008-09 Academic Session, End of Semester Paper)

Question B1: (a): $F = F_1 - F_2 = 2.04 \times 10^8 \text{ N}$ (b) $X = 27.9 \text{ m}$

Question B2: (1) $m_1 = 0.0011 \text{ kg}$ (2) $P_2 = 12.981 \text{ MPa}$

(3) ${}_1W_2 = 612.93 \text{ J}$ (4) ${}_1Q_2 = -76.48 \text{ J}$

Question B3 (a) $T_1 = 151.86^\circ\text{C}$ (b) $m_{total} = 915.58 \text{ kg}$

(c) $x_1 = 0.0007283$ (d) $U_2 = 699.4 \text{ MJ}$

Summer 2008 (2007-08 Academic Session, End of Semester Paper)

Question B1: (1): $P_A - P_B = 34688 \text{ N/m}^2$

Question B2: (1) $V_2 = 0.0179 \text{ m}^3$ (2) $T_2 = 142.65 \text{ }^\circ\text{C}$

Question B3 (i) $T_{exit} = 99.63 \text{ }^\circ\text{C}$ (ii) 880.6 kJ/kg (iii) $\dot{m} = 5.68 \text{ kg/s}$

Summer 2007 (2006-07 Academic Session, **Re-Sit**)

Question B1: (1): $P_A - P_B = 34688 \text{ N/m}^2$

Question B2: Head Loss : $h_{loss} = 9.4 \text{ m}$

Question B3 (i) $\Delta P = P_2 - P_1 = 61.4 \text{ (kPa)}$ (ii) $m_1 = 0.03564 \text{ (kg)}$

(iii) $\Delta m = 0.00606 \text{ (kg)}$

Question B4 (i) $T_{exit} = 25^\circ\text{C}$ (ii) $\bar{V}_2 = 5.16 \text{ m/s}$

Christmas 2006 (2006-07 Academic Session, End of Semester Paper)

Question B1: (1): $Q = \frac{2 \cdot g \cdot h \cdot \left(\frac{\rho_m}{\rho_w} - 1 \right)}{1 - \left(\frac{A_2}{A_1} \right)^2} \cdot A_2$ (2) $Q = 0.029 \text{ m}^3/\text{s}$

Question B2: Head Loss : $h_{loss} = 9.4 \text{ m}$

Question B3 (a) ${}_1W_2 = -696 \text{ kJ}$ (b) ${}_1Q_2 = -696 \text{ kJ}$

Question B4 (i) $T_{exit} = 25^\circ\text{C}$ (ii) $\bar{V}_2 = 5.16 \text{ m/s}$

Summer 2006 (2005-06 Academic Session, **Re-Sit**)

Question B1: (i): $v_c = 7.004 \text{ m/s}$ (ii) $Q = 0.220 \text{ m}^3/\text{s}$ (iii) $\frac{P_B}{\rho g} = -4.5 \text{ m}$

Question B2: (i) $\Delta P = 61.4 \text{ (kPa)}$ (ii) $m_1 = 0.03564 \text{ (kg)}$ (iii) $\Delta m = 0.00606 \text{ (kg)}$

Question B3: (1) $V_2 = 0.0535 \text{ m}^3$ (2) ${}_1W_2 = -33.61 \text{ kJ}$ (3) $T_2 = 491.61 \text{ K} = 218.46 \text{ }^\circ\text{C}$

Christmas 2005 (2005-06 Academic Session, End of Semester Paper)

Question B1: (i) $Q = 1.45 \text{ m}^3/\text{s}$ (ii): $\frac{P_B}{\rho g} = -5.0 \text{ m}$

Question B2: IDEAL GAS : $v = 0.035711 \text{ m}^3/\text{kg}$

TABLES $v = 0.03279 \text{ m}^3/\text{kg}$ (MORE ACCURATE)

Question B3 $T_e = 28 \text{ }^\circ\text{C}$, slightly "cold"

D. Timoney, November 2012