

Rotational Dynamics

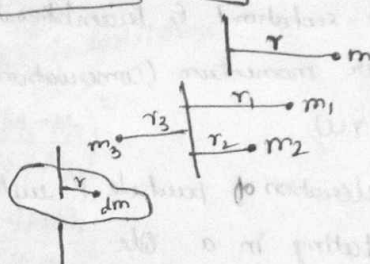
⇒ **Moment of inertia** - It is measurement of the resistance of a body to a change in its rotational motion.

Translatory motion	Rotatory motion
mass (m)	(I) moment of inertia
displacement (w)	angular displacement (θ)
velocity (v)	angular velocity (ω)
acceleration (a)	angular acceleration (α)
momentum (p)	angular momentum (L)
Force (F)	$L = I\omega$
work (W) = F.S	Torque (T)
	work (W) = T θ

→ for single particle $I = mr^2$

system of particles $I = \sum m_i r_i^2$

for rigid bodies $I = \int r^2 dm$



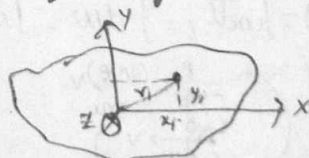
③ **Parallel axis theorem** - Applicable for 2D & 3D bodies

$$I = I_{\text{COM}} + Mr^2$$

r = distance b/w axis to point.

Perpendicular axis theorem - applicable for only 2D bodies.

$$I_z = I_x + I_y$$



Radius of gyration (K) - a distance from the axis where the whole mass can be assumed to be concentrated

$$K = \sqrt{\frac{I}{M}}$$

④ **M.O.I of standard bodies**

i) Ring

$I_C = MR^2$, $I = 2MR^2$
axis in plane axis out of plane

radius



$I = MR^2/2$
axis in plane

ii) Disc
 $I_C = \frac{1}{2}MR^2$
 $I = \frac{1}{2}MR^2$


ii) Disc

$I_C = MR^2/2$


① $I_C = \frac{1}{2}MR^2$
 $I = \frac{3}{2}MR^2$

② $I = \frac{MR^2}{4}$

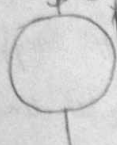
$I = \frac{ml^2}{3}$

(iv) Solid


$$I_C = \frac{MR^2}{2} \quad I = \frac{1}{2}MR^2$$


(v) hollow


$$I_C = MR^2$$

(vi) Hollow sphere


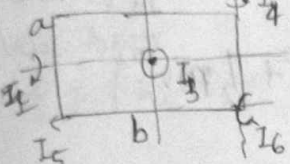
$$I_C = \frac{2}{3}MR^2$$

$$I = \frac{5}{3}MR^2$$

(vii) Solid


$$I_C = \frac{2}{5}MR^2$$

$$I = \frac{7}{5}MR^2$$

(viii)



$$I_1 = I_2 = \frac{Ma^2}{12}$$

$$I_3 = \frac{Mb^2}{12}$$

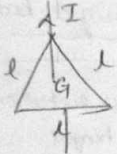
$$I_4 = I_5 = \frac{Ma^2}{12}$$

$$I_6 = \frac{Mb^2}{12}$$


$$I_1 = I_2 + I_3 = I_4 + I_5$$

(ix) Hollow cone



$$I_C = \frac{MR^2}{2}$$

(x)


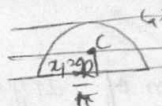
$$I = \frac{mb^2}{12}$$

(xi) Solid cone


$$I = \frac{3MR^2}{10}$$

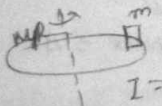
(xii) Semicircular arc


$$I = MR^2$$

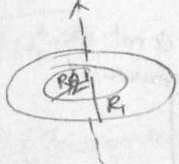
(c)


$$I = I + M(x_1^2 - x_2^2)$$

5 Addition (or) subtraction of MOI

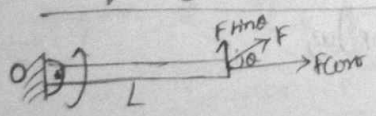
(a)


$$I = \frac{MR^2}{2} + MR^2$$

(b)


$$I = \frac{M(R_1^2 + R_2^2)}{2}$$

6 Torque ($\vec{\tau} = \vec{r} \times \vec{F}$) = N.m



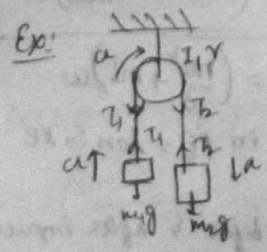
- i) $\tau = LF \sin \theta$, ACW
- ii) $F \cos \theta$ passes through point O, can't have rotational effect

$$\sum \vec{\tau} = I \vec{\alpha}$$
 angular accel.

static equilibrium problems

$$\sum \vec{\tau} = 0 \quad \& \quad \sum \vec{F} = 0$$
 both

$$W \& V_{cm} = 0$$



Ex:

$$m_2 g - T_2 = m_2 a \rightarrow (1)$$

$$T_1 - m_1 g = m_1 a \rightarrow (2)$$

$$T_2 - T_1 = I \alpha$$

$$(T_2 - T_1) r = I \frac{a}{r}$$

$$T_2 - T_1 = \frac{I a}{r^2} \rightarrow (3)$$

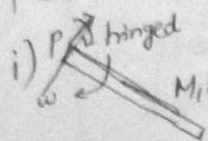
from (1) + (2) + (3) \Rightarrow

$$a = \frac{(m_2 - m_1) g}{(m_2 + m_1 + \frac{I}{r^2})}$$

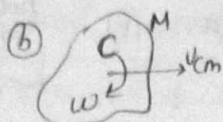
$$a = \frac{g \sin \theta}{1 + k^2/r^2}$$

$$t = \sqrt{\frac{2 g \sin \theta (1 + k^2/r^2)}{g \sin \theta}}$$

7) a) $KE \Rightarrow \int d(KE) = \frac{1}{2} \int dm v^2$
 $KE = \frac{1}{2} I \omega^2$



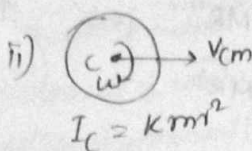
$KE = \frac{1}{2} I_p \omega^2$
 $= \frac{1}{2} \frac{ML^2}{3} \omega^2$



here axis of rotation is @ M

$KE = KE_R + KE_T$

$KE = \frac{1}{2} I \omega^2 + \frac{1}{2} m v_{cm}^2$



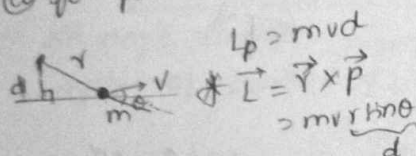
$I_c = k m r^2$

$k = \frac{1}{2}$ for ring / thin cylinder
 $\frac{1}{2}$ for disk / solid cylinder
 $\frac{2}{3}$ for hollow sphere
 $\frac{2}{5}$ for solid sphere

$KE = \frac{1}{2} I_c \omega^2 + \frac{1}{2} m v_{cm}^2$

8) Angular momentum (L)

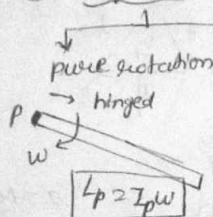
i) a) for point mass



$L_p = m v d$

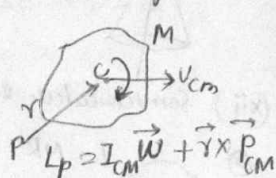
$\vec{L} = \vec{r} \times \vec{p}$
 $= m v r \sin \theta$

b) rigid body



$L_p = I_p \omega$

rolling (rot + trans)



$L_p = I_{cm} \omega + \vec{r} \times \vec{p}_{cm}$

ii) Conservation of angular momentum

a) $T = \frac{dL}{dt}$

if $T_{ext} = 0 \Rightarrow L$ is const. @ conserved

only to $F = \frac{dp}{dt}$

b) If $T_{ext} \neq 0$

$T dt = dL \Rightarrow \int T dt = \Delta L$

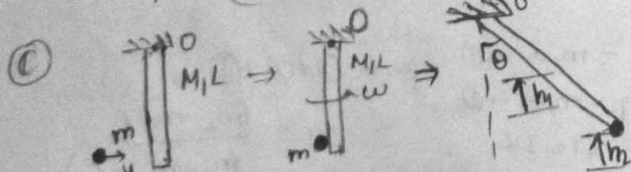
angular impulse = ΔL

iii) Examples

a) $M/R \omega_1 \Rightarrow M/R \omega_2 \Rightarrow \frac{Li = L_f}{\omega_1} = \left(\frac{MR^2}{2} + MR^2 \right) \omega_2$

b) $I_1 \omega_1 \Rightarrow I_2 \omega_2$ (ii) Law of $KE = \frac{1}{2} I_1 \omega_1^2 = \frac{1}{2} (I_1 + I_2) \omega_2^2$

i) $I_1 \omega = (I_1 + I_2) \omega$



$m u L = (I_{rod} + I_{man}) \omega$

$m u L = \left(\frac{m L^2}{3} + m L^2 \right) \omega$

gain in PE = loss in KE

T_{net} about O is zero during impact, L is same just before & after impact

① fixed & free collision (e = coeff. of restitution)

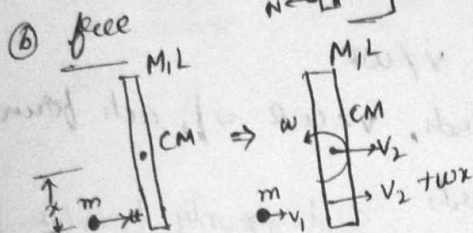


$$L_i = L_f$$

$$m u x = m v_1 x + \frac{m L^2}{3} \omega \rightarrow (i)$$

$$e = \frac{\omega x - v_1}{u} \rightarrow (ii)$$

hence solve 1 & 2 we get v, ω



(i) L_i is cons about any point about CM $\Rightarrow L_i = L_f$

$$m u x = m v_1 x + \frac{m L^2}{12} \omega \rightarrow (1)$$

(ii) P is cons (As $F_{net} = 0$)

$$P_i = P_f \Rightarrow m u = m v_1 + m v_2 \rightarrow (2)$$

$$(iii) e = \frac{v_2 + \omega x - v_1}{u} \rightarrow (3)$$

by solving ① & ②, ③ we get v_1, v_2 & ω

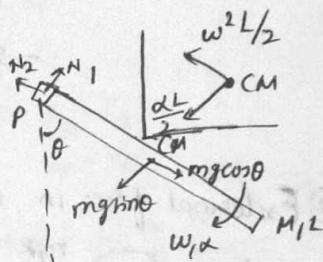
④ hinge reaction

① Always take 2 later components of this model

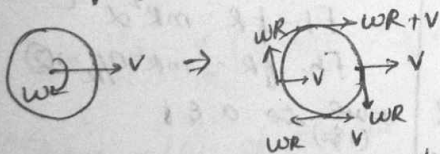
$$mg \sin \theta = N_1 = \frac{m \omega^2 L}{2}$$

$$N_2 - mg \cos \theta = m \omega^2 \frac{L}{2}$$

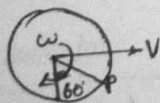
$$\therefore R \times N = \sqrt{N_1^2 + N_2^2}$$



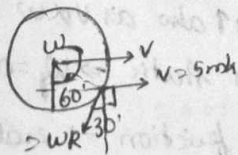
⑨ Velocity of point on circumference (Translational + Rotational) = rolling



Ex-1 - $V = 5 \text{ m/s}$, $\omega = 2.5 \text{ rad/s}$, $R = 2 \text{ m}$, find V_p ?



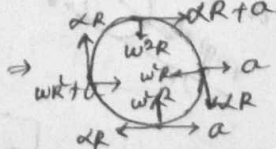
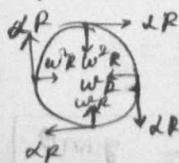
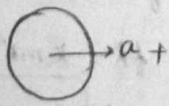
10/1:



$$V_p = \sqrt{5^2 + 5^2 + 2 \cdot 5 \cdot 5 \cos 120^\circ}$$

$$= 5 \text{ m/s}$$

⑩ Acceleration of point on circumference



13 Rolling motion

① Pure rolling: No relative motion of pt. of contact & pt. of contact have same velocity

$$V_{\text{ground}} = 0 \Rightarrow V - WR = 0 \Rightarrow \boxed{V = WR}$$



(i) $V = WR$ ($\because a = R\alpha$), no friction acting

② Rolling without slipping: $V \neq WR$

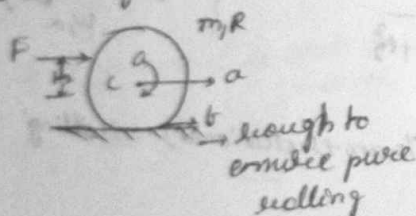
\rightarrow If $V > WR \Rightarrow f_k$ acts backwards, $V < WR \Rightarrow f_k$ acts forwards
 f_k until pure rolling starts
 Sliding only: translational

14 KE in pure rolling

$\beta = \frac{K^2}{R^2} \rightarrow \beta = 1$ if ring / H. cyl
 $= \frac{1}{2}$ if disc / sol. cyl
 $= \frac{2}{3}$ for H. sphere
 $= \frac{2}{5}$ for sol. sphere

$$\boxed{KE_T = \frac{1}{2} m v^2 (1 + \beta)}$$

12 External force in rolling



Note: (i) pure rolling $\rightarrow f$ is static

(ii) pt of contact is at instantaneous rest $\Rightarrow W_f = 0$

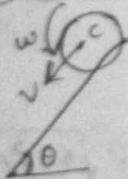
$$F + f = ma \rightarrow (1) \quad \left[\alpha = \frac{a}{R} \right]$$

$$Fh - fR = mR^2 \alpha$$

$$Fh - fR = mR^2 \frac{a}{R} \rightarrow (2)$$

solve to a & f
 (1 & 2)

13 Energy conservation (pure rolling)



$$V = WR \text{ \& } a = R\alpha$$

i) $V \uparrow$ due to gravity

ii) $W \uparrow$ also as $V \uparrow$ & $a \uparrow$

f acts static $\Rightarrow W_f = 0$

14 acceleration & friction \rightarrow rolling on inclined



$$mg \sin \theta - f = ma \rightarrow (1)$$

$$fR = K m R^2 \alpha \quad \left\{ \alpha = \frac{a}{R} \right\}$$

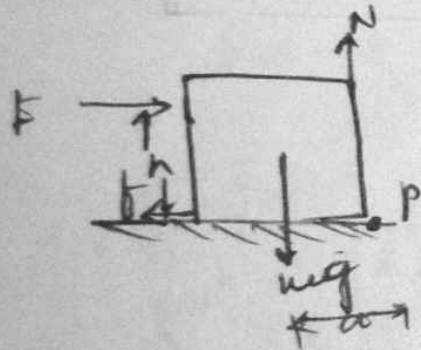
$$fR = K m R^2 \frac{a}{R} \Rightarrow f = K m a \rightarrow (2)$$

$$\boxed{a = \frac{g \sin \theta}{1 + \beta}}$$

$$\boxed{f = \frac{\beta (m g \sin \theta)}{1 + \beta}}$$

$$a_{\text{solid-sph}} > a_{\text{disc}} > a_{\text{H.s}} > a_{\text{ring}}$$

(15) Toppling



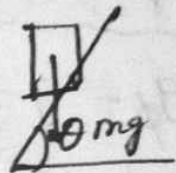
N should pass through P
for toppling about P

$$\tau_f > \tau_{mg}$$

$$Fh > mga$$

$$F > mga/h$$

$$F_{\min} = \frac{mga}{h}$$



$\theta \uparrow$ in inclined
plane

toppling also \uparrow