

2nd Unit Problems

Q) For the servomechanism with open loop transfer function given below, what type of input signal gives rise to a constant steady state error and calculate their values

$$G(s) = \frac{10}{s^2(s+2)(s+3)}$$

$$\text{Given : } G(s) = \frac{10}{s^2(s+2)(s+3)}$$

Let us assume unity feedback system

$$H(s) = 1$$

The open loop system has two poles at origin. Hence it is a type-2 system.

$$s^2 = 0, \quad s+2=0, \quad s+3=0$$

$$s=0, s=0, \quad s=-2, \quad s=-3$$

In systems with type number-2 the acceleration (parabolic) input will give a constant steady state error.

The steady state error with unit acceleration input,

$$e_{ss} = \frac{1}{K_a}$$

Acceleration error constant,

$$K_a = \lim_{s \rightarrow 0} s^2 G(s) H(s)$$

$$= \lim_{s \rightarrow 0} s^2 \frac{10}{s^2(s+2)(s+3)} \quad (1)$$

$$= \frac{10}{(0+2)(0+3)}$$

$$= \frac{10}{(2)(3)} = \frac{10}{6} = 1.67$$

$$K_a = 1.67$$

Steady state error,

$$e_{ss} = \frac{1}{k_a} = \frac{1}{1.67} = 0.6$$

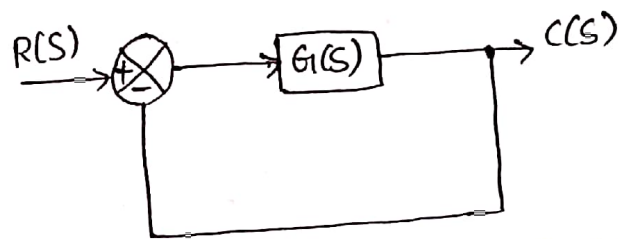
In the given system, with unit acceleration input,
steady state error = 0.6

Q) A unity feedback control system has an open loop transfer function, $G(s) = \frac{10}{s(s+2)}$. Find the time domain specification for a step input of 12 units.

The unity feedback system is shown in fig.

The closed loop transfer

$$\text{function, } \frac{C(s)}{R(s)} = \frac{G(s)}{1+G(s)}$$



The closed loop transfer function, Fig: Unity Feedback System

$$\text{G.T, } G(s) = \frac{10}{s(s+2)}$$

$$\therefore \frac{C(s)}{R(s)} = \frac{\frac{10}{s(s+2)}}{1 + \frac{10}{s(s+2)}} = \frac{\frac{10}{s(s+2)}}{\frac{s(s+2) + 10}{s(s+2)}}$$

$$\frac{C(s)}{R(s)} = \frac{10}{s^2 + 2s + 10} \rightarrow \textcircled{1}$$

The values of damping ratio and natural frequency of oscillation ω_n are obtained by comparing the system transfer function with standard form of second order transfer function,

Standard form of second order transfer function is,

$$\frac{C(s)}{R(s)} = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} \rightarrow \textcircled{2}$$

On comparing eq ① & ② we get,

$$\omega_n^2 = 10$$

$$\omega_n = \sqrt{10}$$

$$\omega_n = 3.162 \text{ rad/sec}$$

$$2\zeta\omega_n = 2$$

$$\zeta = \frac{2}{2\omega_n}$$

$$= \frac{1}{3.162}$$

$$\boxed{\zeta = 0.316}$$

$$\theta = \tan^{-1} \frac{\sqrt{1-\zeta^2}}{\zeta} = \tan^{-1} \frac{\sqrt{1-0.316^2}}{0.316} = 1.249 \text{ rad}$$

$$\omega_d = \omega_n \sqrt{1-\zeta^2} = 3.162 \sqrt{1-0.316^2} = 3 \text{ rad/sec}$$

(i) Rise time, $t_r = \frac{\pi - \theta}{\omega_d}$

$$t_r = \frac{\pi - 1.249}{3} = 0.63 \text{ sec}$$

$$\boxed{t_r = 0.63 \text{ sec}}$$

(ii) Percentage overshoot, %Mp

$$\%Mp = e^{\frac{-\zeta\pi}{\sqrt{1-\zeta^2}}} \times 100$$

$$= e^{\frac{-0.316\pi}{\sqrt{1-0.316^2}}} \times 100$$

$$= 0.3512 \times 100 = 35.12\%$$

(iii) Peak overshoot = $\frac{35.12}{100} \times 12 \text{ units}$

$$= 4.2144 \text{ units}$$

(iv) Peak time,

$$t_p = \frac{\pi}{\omega_d} = \frac{\pi}{3} = 1.047 \text{ sec}$$

$$\boxed{t_p = 1.047 \text{ sec}}$$

(V) Time constant

$$T = \frac{1}{\zeta \omega_n}$$
$$= \frac{1}{0.316 \times 3.162} = 1 \text{ sec}$$

$$\boxed{T = 1 \text{ sec}}$$

\therefore For 5% error, settling time, $t_s = 3T = 3 \text{ sec}$

\therefore For 2% error, settling time, $t_s = 4T = 4 \text{ sec}$.

vi) Settling time $t_s = 3 \text{ sec}$ for 5% error
 $= 4 \text{ sec}$ for 2% error.

Q

A unity feedback system has the forward transfer function $G(s) = \frac{k_1(2s+1)}{s(5s+1)(1+s)^2}$. The input $r(t) = 1+6t$

is applied to the system. Determine the value of k_1 if the steady error is to be less than 0.1.

Given that, input $r(t) = 1+6t$

on taking laplace transform of $r(t)$ we get $R(s)$.

$$\therefore R(s) = \mathcal{L}\{r(t)\}$$
$$= \mathcal{L}\{1+6t\}$$

$$R(s) = \frac{1}{s} + \frac{6}{s^2}$$

The error signal in s-domain $E(s)$ is given by

$$\therefore E(s) = \frac{R(s)}{1+G(s)H(s)} = \frac{\frac{1}{s} + \frac{6}{s^2}}{1 + \frac{k_1(2s+1)}{s(5s+1)(1+s)^2}} \quad [\because H(s)=1]$$
$$= \frac{\frac{1}{s} + \frac{6}{s^2}}{\frac{s(5s+1)(1+s)^2 + k_1(2s+1)}{s(5s+1)(1+s)^2}}$$

$$E(s) = \frac{1/s}{\frac{s(s+1)(1+s)^2 + k_1(2s+1)}{s(s+1)(1+s)^2}} + \frac{6}{s^2} \cdot \frac{s(s+1)(1+s)^2}{s(s+1)(1+s)^2 + k_1(2s+1)}$$

$$= \frac{1}{s} \left[\frac{s(s+1)(1+s)^2}{s(s+1)(1+s)^2 + k_1(2s+1)} \right] + \frac{6}{s^2} \left[\frac{s(s+1)(1+s)^2}{s(s+1)(1+s)^2 + k_1(2s+1)} \right]$$

The steady state error e_{ss} can be obtained from final value theorem.

$$e_{ss} = \lim_{t \rightarrow \infty} e(t) = \lim_{s \rightarrow 0} s E(s)$$

$$= \lim_{s \rightarrow 0} s \left\{ \frac{1}{s} \left[\frac{s(s+1)(1+s)^2}{s(s+1)(1+s)^2 + k_1(2s+1)} \right] + \frac{6}{s^2} \left[\frac{s(s+1)(1+s)^2}{s(s+1)(1+s)^2 + k_1(2s+1)} \right] \right\}$$

$$e_{ss} = \lim_{s \rightarrow 0} \left\{ \frac{s(s+1)(1+s)^2}{s(s+1)(1+s)^2 + k_1(2s+1)} + \frac{6(s+1)(1+s)^2}{s^2(s+1)(1+s)^2 + k_1(2s+1)} \right\}$$

$$e_{ss} = 0 + \frac{6}{k_1} = \frac{6}{k_1}$$

Given that, $e_{ss} < 0.1$,

$$\therefore 0.1 = \frac{6}{k_1} \quad \text{or} \quad k_1 = \frac{6}{0.1} = 60$$

For steady state error, $e_{ss} < 0.1$, the value of k_1 should be greater than 60.

Q A unity feed back control system is characterized by the following open loop transfer function

$G(s) = \frac{0.4s+1}{s(s+0.6)}$. Determine its transient response for unit step input. Evaluate the maximum overshoot and corresponding peak time.

The closed loop transfer function,

$$\frac{C(s)}{R(s)} = \frac{G(s)}{1 + G(s)H(s)}$$

Given that, $G(s) = \frac{(0.4s+1)}{s(s+0.6)}$

For unity feedback system, $H(s) = 1$

$$\therefore \frac{C(s)}{R(s)} = \frac{G(s)}{1 + G(s)} = \frac{\frac{0.4s+1}{s(s+0.6)}}{1 + \frac{0.4s+1}{s(s+0.6)}} = \frac{0.4s+1}{s(s+0.6) + 0.4s+1}$$

$$= \frac{0.4s+1}{s^2 + 0.6s + 0.4s + 1}$$

$$\frac{C(s)}{R(s)} = \frac{0.4s+1}{s^2 + s + 1}$$

The s-domain response, $C(s) = R(s) \times \frac{0.4s+1}{s^2 + s + 1}$

For step input, $R(s) = 1/s$.

$$\therefore C(s) = \frac{1}{s} \frac{0.4s+1}{s^2 + s + 1} = \frac{0.4s+1}{s(s^2 + s + 1)}$$

By partial fraction expansion $C(s)$ can be expressed as

$$C(s) = \frac{0.4s+1}{s(s^2 + s + 1)} = \frac{A}{s} + \frac{Bs+C}{s^2 + s + 1}$$

The residue A is solved by multiplying $C(s)$ by s and letting $s=0$

$$\therefore A = C(s) \times s|_{s=0}$$

$$A = \frac{0.4s+1}{s'(s^2 + s + 1)} \Big|_{s=0}$$

$$A = \frac{0+1}{0+0+1}$$

$$\boxed{A=1}$$

B and c are solved by cross multiplication and equating coefficients

$$\frac{0.4s+1}{s(s^2+s+1)} = \frac{A}{s} + \frac{Bs+C}{s^2+s+1}$$

$$0.4s+1 = A(s^2+s+1) + (Bs+C)(s)$$

$$\text{Equating } s^2 \text{ coefficients} \quad 0.4s+1 = A(s^2+s+1) + Bs^2+Cs$$

$$A+B=0$$

$$1+C=0$$

$$\boxed{B=-1}$$

s- coefficients

$$A+C=0.4$$

$$C=0.4-A$$

$$C=0.4-1$$

$$\boxed{C=-0.6}$$

$$\therefore C(s) = \frac{1}{s} + \frac{-s-0.6}{s^2+s+1} = \frac{1}{s} - \frac{s+0.6}{s^2+s+1}$$

$$C(s) = \frac{1}{s} - \frac{s+0.5+0.1}{s^2+2 \times 0.5s+0.5^2+1-0.25}$$

$$C(s) = \frac{1}{s} - \frac{s+0.5+0.1}{(s+0.5)^2+0.75} = \frac{1}{s} - \frac{s+0.5}{(s+0.5)^2+0.75} - \frac{0.1}{\sqrt{0.75}} \frac{\sqrt{0.75}}{(s+0.5)^2+0.75}$$

The time domain response is obtained by taking inverse Laplace transform.

$$C(t) = \mathcal{L}^{-1}\{C(s)\} = \mathcal{L}^{-1}\left\{\frac{1}{s} - \frac{s+0.5}{(s+0.5)^2+0.75} - \frac{0.1}{\sqrt{0.75}} \frac{\sqrt{0.75}}{(s+0.5)^2+0.75}\right\}$$

$$C(t) = 1 - e^{-0.5t} \cos \sqrt{0.75}t - \frac{0.1}{\sqrt{0.75}} e^{-0.5t} \sin \sqrt{0.75}t$$

$$C(t) = 1 - e^{-0.5t} [0.1155 \sin(\sqrt{0.75}t) + \cos(\sqrt{0.75}t)]$$

The transient response is the part of the output which vanishes as t tends to infinity. Here as t tends to infinity the exponential component $e^{-0.5t}$ tends to zero. Hence the transient response is

given by damped sinusoidal component.

The transient response of $c(t) = e^{-0.5t} [0.1155 \sin \sqrt{0.75}t + \cos \sqrt{0.75}t]$

Maximum overshoot M_p :

Standard form of second order characteristic equation

$$s^2 + 2\zeta\omega_n s + \omega_n^2 = s^2 + s + 1$$

on comparing we get

$$\begin{aligned} \omega_n^2 &= 1 & 2\zeta\omega_n &= 1 \\ \omega_n &= 1 \text{ rad/sec} & 2\zeta(1) &= 1 \\ & & \zeta &= 1/2 = 0.5 \end{aligned}$$

$$M_p = e^{\frac{-\zeta\pi}{\sqrt{1-\zeta^2}}} = e^{\frac{-0.5\pi}{\sqrt{1-0.5^2}}} = 0.163$$

\therefore Maximum overshoot $\therefore M_p = M_p \times 100$

$$= 0.163 \times 100$$

$$\therefore M_p = 16.3 \%$$

$$\text{Peak time, } t_p = \frac{\pi}{\omega_d} = \frac{\pi}{\omega_n \sqrt{1-\zeta^2}} = \frac{\pi}{1 \times \sqrt{1-0.5^2}}$$

$$\boxed{t_p = 3.628 \text{ sec}}$$

The response of system is underdamped.

Q. For a unity feedback control system the open loop transfer function $G(s) = \frac{10(s+2)}{s^2(s+1)}$.

Find: a) position, velocity and acceleration error constants
b) steady state error when input $R(s) = \frac{3}{s} - \frac{2}{s^2} + \frac{1}{3s^3}$

$$\text{Given, } G(s) = \frac{10(s+2)}{s^2(s+1)}, \quad R(s) = \frac{3}{s} - \frac{2}{s^2} + \frac{1}{3s^3}$$

For, unity feedback system, $H(s) = 1$

Position error constant,

$$K_p = \lim_{s \rightarrow 0} G(s) H(s) \quad [H(s) = 1]$$

$$K_p = \lim_{s \rightarrow 0} \frac{10(s+2)}{s^2(s+1)} = \infty$$

Velocity Error Constant, $K_v = \infty$

$$K_v = \lim_{s \rightarrow 0} s \cdot G(s) H(s)$$

$$= \lim_{s \rightarrow 0} s \cdot \frac{10(s+2)}{s^2(s+1)} \quad (1)$$

$$K_v = \infty$$

Acceleration Error constant:

$$K_a = \lim_{s \rightarrow 0} s^2 G(s) H(s)$$

$$= \lim_{s \rightarrow 0} s^2 \cdot \frac{10(s+2)}{s^2(s+1)} \quad (1)$$

$$= \frac{10(0+2)}{0(1)}$$

$$K_a = 20$$

b) To find steady state error:-

The error signal in s-domain, $E(s) = \frac{R(s)}{1+G(s)H(s)}$

$$\text{GT, } R(s) = \frac{3}{s} - \frac{2}{s^2} + \frac{1}{3s^3}, \quad G(s) = \frac{10(s+2)}{s^2(s+1)}, \quad H(s) = 1$$

$$\therefore E(s) = \frac{\frac{3}{s} - \frac{2}{s^2} + \frac{1}{3s^3}}{1 + \frac{10(s+2)}{s^2(s+1)}} = \frac{\frac{3}{s} + \frac{2}{s^2} + \frac{1}{3s^3}}{\frac{s^2(s+1) + 10(s+2)}{s^2(s+1)}}$$

$$\therefore E(s) = \frac{3}{s} \left[\frac{s^2(s+1)}{s^2(s+1) + 10(s+2)} \right] - \frac{2}{s^2} \left[\frac{s^2(s+1)}{s^2(s+1) + 10(s+2)} \right] + \frac{1}{3s^3} \left[\frac{s^2(s+1)}{s^2(s+1) + 10(s+2)} \right]$$

The steady state error e_{ss} can be obtained from final value theorem.

Steady state error,

$$e_{ss} = \lim_{t \rightarrow \infty} e(t) = \lim_{s \rightarrow 0} s E(s)$$

$$\therefore e_{ss} = \lim_{s \rightarrow 0} s \left\{ \frac{3}{s} \left[\frac{s^2(s+1)}{s^2(s+1)+10(s+2)} \right] - \frac{2}{s^2} \left[\frac{s^2(s+1)}{s^2(s+1)+10(s+2)} \right] + \frac{1}{3s^3} \left[\frac{s^2(s+1)}{s^2(s+1)+10(s+2)} \right] \right\}$$

$$e_{ss} = \lim_{s \rightarrow 0} \left\{ \frac{3s^2(s+1)}{s^2(s+1)+10(s+2)} - \frac{2s(s+1)}{s^2(s+1)+10(s+2)} + \frac{(s+1)}{3s^2(s+1)+30(s+2)} \right\}$$

$$= 0 - 0 + \frac{1}{60}$$

Steady state error, $\boxed{e_{ss} = \frac{1}{60}}$

Q. For the servomechanisms with open loop transfer function given below explain what type of input signal give rise to a constant steady state error and calculate their values.

i) $G(s) = \frac{20(s+2)}{s(s+1)(s+3)}$ (ii) $G(s) = \frac{10}{(s+2)(s+3)}$

(iii) $G(s) = \frac{10(s+2)}{s^2(s+1)(s+2)}$

(i) $G(s) = \frac{20(s+2)}{s(s+1)(s+3)}$

Let us assume unity feedback system $\therefore H(s) = 1$

The *loop open loop system has a pole at origin. Hence it is a type-1 system. In systems with type number -1, the velocity (ramp) input will give a constant steady state error.

The steady state error with unit velocity input,

$$e_{ss} = \frac{1}{k_v}$$

velocity error constant, $k_v = \lim_{s \rightarrow 0} s G(s) H(s) = \lim_{s \rightarrow 0} s G(s)$

$$k_v = \lim_{s \rightarrow 0} s \frac{20(s+2)}{s(s+1)(s+3)}$$
$$= \frac{20(0+2)}{(0+1)(0+3)} = \frac{40}{3}$$

$$k_v = \frac{40}{3}$$

steady state error, $e_{ss} = \frac{1}{k_v} = \frac{3}{40} = 0.075$.

$$(ii) \quad G(s) = \frac{10}{(s+2)(s+3)}$$

$$H(s) = 1$$

The open loop system has no pole at origin. Hence it is a type-0 system. In systems with type number-0, the step input will give a constant steady state error.

The steady state error with unit step input

$$e_{ss} = \frac{1}{1+k_p}$$

position error constant, $k_p = \lim_{s \rightarrow 0} G(s) H(s)$

$$= \lim_{s \rightarrow 0} G(s)$$

$$k_p = \lim_{s \rightarrow 0} \frac{10}{(s+2)(s+3)} = \frac{10}{2 \times 3} = \frac{5}{3}$$

$$\text{steady state error, } e_{ss} = \frac{1}{1+k_p} = \frac{1}{1+\frac{5}{3}} = \frac{3}{3+5} = \frac{3}{8}$$

$$e_{ss} = 0.375$$

$$(iii) \quad G(s) = \frac{10(s+2)}{s^2(s+1)(s+2)}$$

$$H(s) = 1$$

The open loop system has two poles at origin. Hence it is a type-2 system. In systems with type no. 2, the acceleration (parabolic) input will give a constant steady state

error.

The steady state error with unit acceleration input, $e_{ss} = \frac{1}{k_a}$

$$k_a = \lim_{s \rightarrow 0} s^2 G(s) H(s) = \lim_{s \rightarrow 0} s^2 \frac{10(s+2)}{s^2(s+1)(s+2)}$$
$$= \frac{10(2)}{(1)(2)} = 10$$

steady state error, $e_{ss} = \frac{1}{k_a} = \frac{1}{10} = 0.1$