ChE-381: Process Dynamics and Control Simulation Session 6

Problem statement

Consider the three noninteracting tanks in series shown in Figure 1 with feedback control.

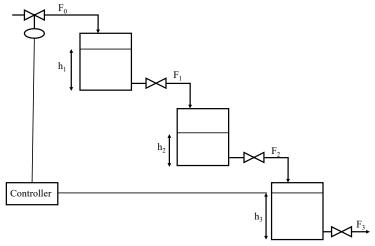


Figure 1. Three non-interacting tanks in series with feedback controller.

A linear model for the liquid level h_i of each tank will take the form:

$$A_{1} \frac{dh_{1}}{dt} = F_{0} - c_{1}h_{1}$$

$$A_{2} \frac{dh_{2}}{dt} = c_{1}h_{1} - c_{2}h_{2}$$

$$A_{3} \frac{dh_{3}}{dt} = c_{2}h_{2} - c_{3}h_{3}$$

where A_i and c_i represent the cross-section area and the outlet valve discharge constant for each tank. Defining deviation variables and taking Laplace transform, we get:

$$H_3'(s) = \frac{K}{(\tau_1 s + 1)(\tau_2 s + 1)(\tau_3 s + 1)} F_0'(s)$$

We observe that h_3 is measured and F_0 is adjusted by the controller. Assume that K = 6, $\tau_1 = 2$, $\tau_2 = 4$, and $\tau_3 = 6$.

- a) Tune PI controller using the following methods: robust response time, Skogestad IMC, and Zeigler-Nichols step response. Compute your performance metric used in lab 5 for the tuned PI controller. Which method performs the best in terms of performance metrics?
- b) Tune PI controller obtained using Skogestad IMC and Zeigler-Nichols further to reduce your performance metric while ensuring the poles of closed loop transfer function remains sufficiently in the left half plane.
- c) Show the response plot with the PI controller obtained in (b) and PID parameters obtained in the previous lab in the **same figure**. Ensure that the figure is well labeled, and legend is clear.

Submit a zipped folder containing a .txt file containing the solution of parts (a) and (b), and the simulink file for part (c). Name the folder as your roll number.