

## Technical environmental system – Weekly submission IX

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### Task I

Considering:

Classroom A (Aula A) =  $10m \times 5m \times 4m$

Temperature =  $7^{\circ}C$

Saturation pressure of water =  $1.0021 \text{ kPa}$

Atmospheric pressure =  $102 \text{ kPa}$

$R_v = 0.4615$

$$\phi = \frac{m_v}{m_g} = \frac{P_v}{P_g}$$

$$P_v = \phi \times P_g = 0.84 \times 1.0021 = 0.84 \text{ kPa}$$

$$P_a = P - P_v = 102 \text{ kPa} - 0.84 \text{ kPa} = 101.16 \text{ kPa}$$

### Absolute humidity

$$\omega = 0.622 \frac{P_v}{P_a} = 0.622 \frac{0.84}{101.16} = 0.0052 \frac{\text{kg vapour}}{\text{kg dry Air}}$$

### Mass of water vapor

$$m = \frac{PV}{R_{sp}T}; m_v = \frac{P_v V_v}{R_v T}$$

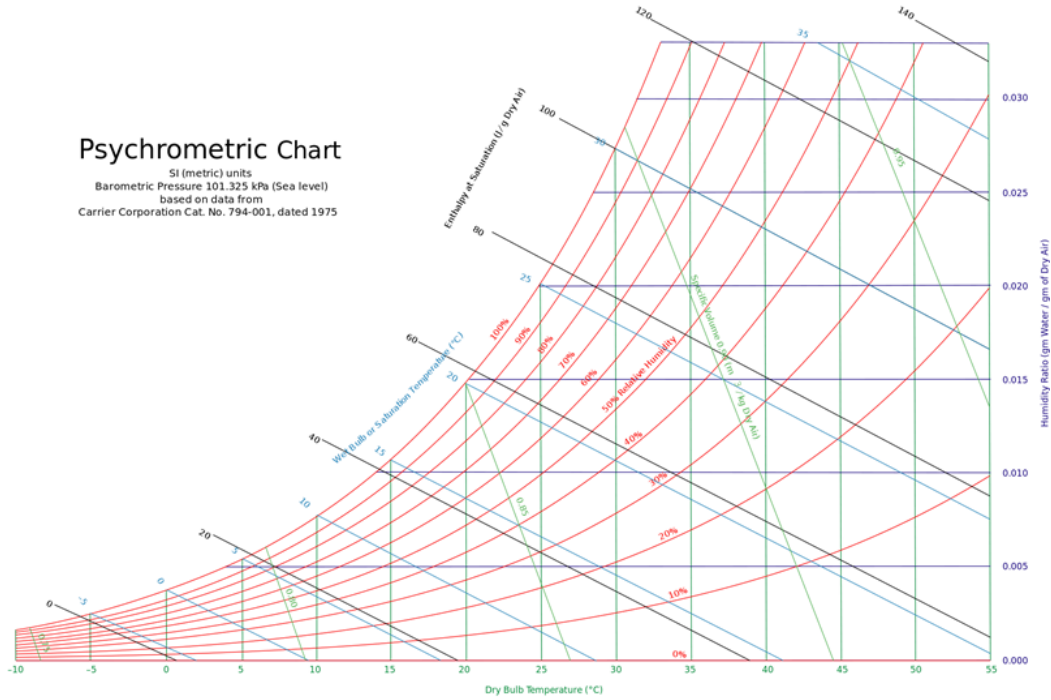
$$m_v = \frac{0.84 \times (10 \times 5 \times 4)}{0.4615 \times (273 + 7)} = 1.3 \text{ kg water vapor}$$

### Enthalpy

$$h = h_a + \omega h_v = (1.005 \times 7) + 0.0052 (2501 + (1.82 \times 7)) = 20.11 \frac{\text{kJ}}{\text{kg dry Air}}$$

## Psychrometric Chart

SI (metric) units  
Barometric Pressure 101.325 kPa (Sea level)  
based on data from  
Carrier Corporation Cat. No. 794-001, dated 1975



Wet-bulb temperature  
≈ 5.5°C

## Task 2

### BRINDISI, Italy

WMO#: 163200

Lat: 40.65N Long: 17.95E Elev: 10 StdP: 101.2 Time Zone: 1.00 (EUW) Period: 86-10 WBAN: 99999

#### Annual Heating and Humidification Design Conditions

Coldest Month	Heating DB			Humidification DP/MCDB and HR						Coldest month WS/MCDB				MCWS/PCWD	
	99.6%	99%		DP	HR	MCDB	DP	HR	MCDB	WS	MCDB	WS	MCDB	MCWS	PCWD
(a)	(b)	(c)		(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)
(1) 2	2.9	4.1		-5.1	2.5	7.2	-3.0	3.0	7.4	13.4	10.2	12.4	10.6	3.4	250

#### Annual Cooling, Dehumidification, and Enthalpy Design Conditions

Hottest Month	Hottest Month DB Range	Cooling DB/MCWB						Evaporation WB/MCDB						MCWS/PCWD to 0.4% DB	
		0.4%	1%	2%	0.4%	1%	2%	0.4%	1%	2%	0.4%	1%	2%	MCWS	PCWD
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)	(p)
(2) 8	7.1	32.8	23.6	31.1	24.3	29.9	24.3	27.2	29.7	26.3	29.0	25.6	28.3	4.2	180

	Dehumidification DP/MCDB and HR									Enthalpy/MCDB						Hours 8 to 4 & 12.8/20.6
	DP	HR	MCDB	DP	HR	MCDB	DP	HR	MCDB	Enth	MCDB	Enth	MCDB	Enth	MCDB	
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)	(p)	
(3) 26.3	21.8	29.2	25.4	20.7	28.5	24.7	19.7	27.9	86.0	30.1	82.2	29.1	78.5	28.3	1236	(4)

#### Extreme Annual Design Conditions

Extreme Annual WS			Extreme Max WB	Extreme Annual DB				n-Year Return Period Values of Extreme DB											
1%	2.5%	5%		Mean	Standard deviation	Min	Max	Min	Max	Min	Max	Min	Max	Min	Max	Min	Max	Min	Max
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)	(p)				
(4) 11.3	9.9	8.7	31.4	0.4	37.3	1.4	3.0	-0.6	39.4	-1.4	41.1	-2.2	42.8	-3.2	44.9				

Building height = 2.5m  
 Floor area =  $200 \text{ m}^2$   
 Number of occupants = 2  
 Number of bedrooms = 1  
 Wall area =  $144 \text{ m}^2$

### Temperature for cooling and heating

$$T_{cooling} = 31.1 \text{ }^{\circ}\text{C}$$

$$T_{heating} = 4.1 \text{ }^{\circ}\text{C}$$

### Temperature difference

$$\Delta T_{cooling} = 31.1 - 24 = 7.1 \text{ }^{\circ}\text{C}$$

$$\Delta T_{heating} = 20 - 4.1 = 15.9 \text{ }^{\circ}\text{C}$$

### Internal gains

$$\dot{Q}_{ig_{sensible}} = 136 + 2.2A_{cf} + 22N_{oc} = 136 + 2.2 \times 200 + 22 \times 2 = 620 \text{ W}$$

$$\dot{Q}_{ig_{latent}} = 20 + 0.22A_{cf} + 12N_{oc} = 20 + 0.22 \times 200 + 12 \times 2 = 88 \text{ W}$$

### Infiltration

$$A_{ul} = 1.4 \frac{\text{cm}^2}{\text{m}^2}$$

$$A_{es} = 200 + 144 = 344 \text{ m}^2$$

$$A_L = A_{es} \times A_{ul} = 344 \times 1.4 = 481.6 \text{ cm}^2$$

$$IDF_{heating} = 0.065 \frac{L}{s \cdot \text{cm}^2}$$

$$IDF_{cooling} = 0.032 \frac{L}{s \cdot \text{cm}^2}$$

$$\dot{Q}_{i_{heating}} = A_L \times IDF = 481.6 \times 0.065 = 31.30 \frac{L}{s}$$

$$\dot{Q}_{i_{cooling}} = A_L \times IDF = 481.6 \times 0.032 = 15.41 \frac{L}{s}$$

### Ventilation

$$\dot{Q}_v = 0.05A_{cf} + 3.5(N_{br} + 1) = 0.05 \times 200 + 3.5 \times 2 = 17 \frac{L}{s}$$

$$\dot{Q}_{inf-ventilation_{heating}} = 31.30 + 17 = 48.30 \frac{L}{s}$$

$$\dot{Q}_{inf-ventilation_{cooling}} = 15.41 + 17 = 32.41 \frac{L}{s}$$

$$\dot{Q}_{inf-ventilation_{cooling_{sensible}}} = C_{sensible} \dot{V} \Delta T_{cooling} = 1.23 \times 32.41 \times 7.1 = 283.04 \text{ W}$$

$$\dot{Q}_{inf-ventilation_{cooling_{latent}}} = C_{latent} \dot{V} \Delta \omega_{cooling} = 3010 \times 32.41 \times 0.0045 = 438.99 \text{ W}$$

$$\dot{Q}_{inf-ventilation_{heating_{sensible}}} = C_{sensible} \dot{V} \Delta T_{heating} = 1.23 \times 48.30 \times 15.9 = 944.60 \text{ W}$$

$$\dot{Q}_{inf-ventilation_{heating_{latent}}} = C_{latent} \dot{V} \Delta \omega_{heating} = 3010 \times 48.30 \times 0.0046 = 668.76 \text{ W}$$