

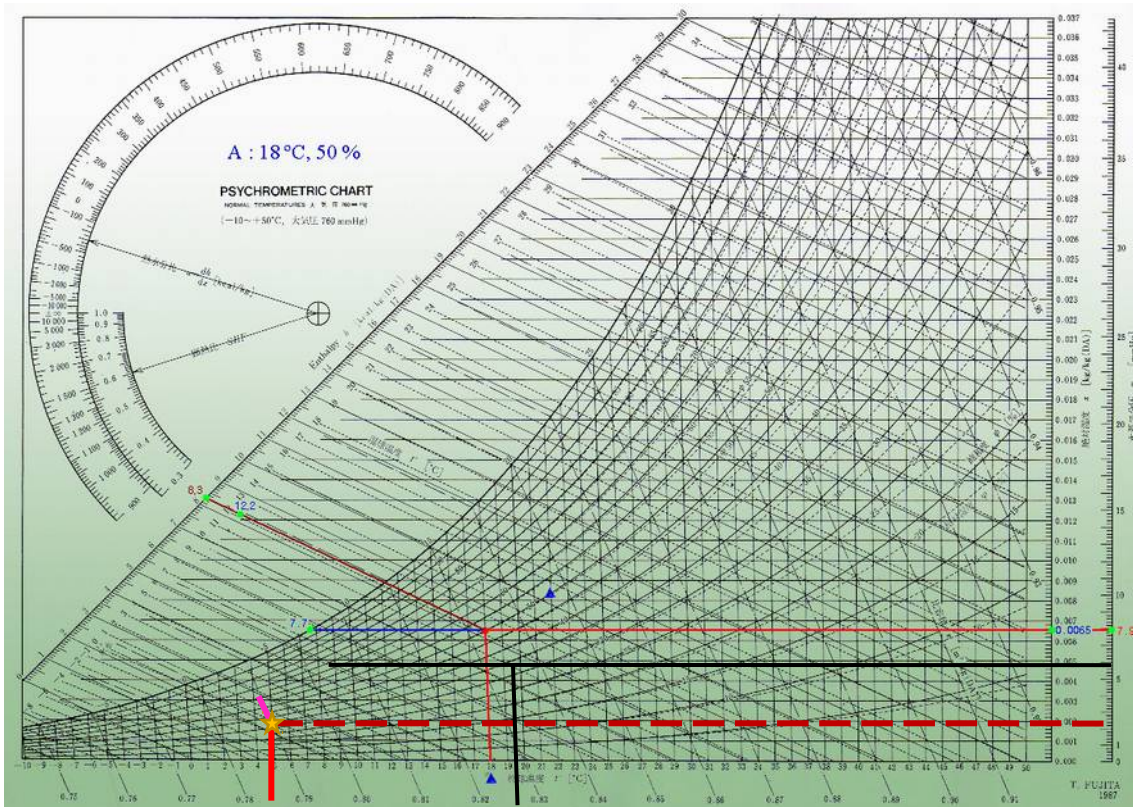
WEEK 9 ASSIGNMENT

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QUESTION 1

Use a weather forecast website, and utilize the psychrometric chart and the formula we went through in the class to determine the absolute humidity, the wet-bulb temperature and the mass of water vapour in the air in Classroom A (Aula A) of Piacenza campus in the moment that you are solving this exercise.

ANSWER:



	1:00 pm	14:00	4:00 pm	18:00	8:00 pm	21:00	22:00
Effective temperature	9 °C	10 °C	8 °C	6 °C	4 °C	2 °C	2 °C
Perceived temperature	7 °C	10 °C	6 °C	4 °C	2 °C	0 °C	0 °C
Rainfall	0 mm	0 mm	0 mm	0 mm	0 mm	0 mm	0 mm
Humidity	67 %	65 %	69 %	70 %	75 %	83 %	87 %
Atmospheric pressure	1025 hPa	1025 hPa	1025 hPa	1026 hPa	1027 hPa	1027 hPa	1028 hPa
Wind intensity	15 km / h	14 km / h	9 km / h	9 km / h	7 km / h	8 km / h	8 km / h
Wind direction	←	←	←	←	↖	↖	↖
	IS	IS	IS	IS	SELF	SELF	SELF
Probability of fog	0 %	0 %	0 %	0 %	0 %	0 %	0 %
Dew point	3 °C	3 °C	3 °C	1 °C	-1 °C	0 °C	-1 °C
Clouds	21 %	13 %	42 %	15 %	2 %	3 %	3 %
Low clouds	11 %	7 %	42 %	15 %	2 %	3 %	3 %
Medium clouds	18 %	12 %	2 %	0 %	1 %	0 %	0 %
High clouds	0 %	0 %	0 %	0 %	0 %	0 %	0 %

a) Absolute Humidity

From the chart, taking dry bulb temperature as 4°C and relative Humidity as 75%, the specific humidity is 0.0037.

By formula Method

$$\Phi = m_v/m_g$$

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$$= P_v/P_g \quad (P_v = P_{\text{sat}} @ 4^\circ\text{C} = 0.8132 \text{ Kpa})$$

Partial pressure of water vapour

$$\Phi = P_v/P_g$$

$$P_v = \Phi \times P_g$$

$$P_v = 0.75 \times 0.8132 = \mathbf{0.6099 \text{ Kpa}}$$

$$P_a = P - P_v$$

$$P_a = 102.7\text{kPa} - 0.6099 = \mathbf{102.0901 \text{ kPa}}$$

$$\omega = 0.622(P_v/P_a)$$

$$\omega = 0.622 \times 0.6099/102.0901$$

$$\omega = \mathbf{0.00368 \text{ Kg vapour/Kg dry air}}$$

b) Wet bulb temperature

From the chart, taking dry bulb temperature as 4°C and relative Humidity as 75%, the specific humidity is **2°C**.

c) Mass of water vapour in the air (m_v) (taking classroom dimensions as 20mx5mx5m

$$M_v = P_v \times V_v / R_v \times T$$

$$m_v = 0.6099 \times (20 \times 5 \times 5) / 0.4615 \times (4 + 273)$$

$$m_v = \mathbf{2.38 \text{ kg}}$$

Utilize the same methodology we went through in the class and determine the sensible and latent load corresponding to internal gains, the ventilation, and the infiltration in a house with a *good* construction quality and with the same geometry as that of the example which is located in Brindisi, Italy.

ANSWER:

a) Internal Gains

$$\begin{aligned} Q_{ig.sensible} &= 136 + 2.2A_{cf} + 22N_{oc} \\ &= 136 + 2.2 \cdot 200 + 22 \cdot 2 \\ &= \mathbf{620W} \end{aligned}$$

$$\begin{aligned} Q_{ig.latent} &= 20 + 0.22A_{cf} + 12N_{oc} \\ &= 20 + 0.22 \cdot 200 + 12 \cdot 2 \\ &= \mathbf{88W} \end{aligned}$$

b) Infiltration

$$Q_i = A_L \cdot IDF$$

$$A_{UL} = 1.4 \text{ cm}^2/\text{m}^2 \text{ (From the table)}$$

Exposed surface = Wall area + roof area

$$A_{es} = 200 + 144$$

$$A_{es} = 344 \text{ m}^2$$

$$\begin{aligned} A_L &= A_{es} \times A_{UL} = 344 \times 1.4 \\ &= \mathbf{481.6 \text{ cm}^2} \end{aligned}$$

Infiltration rate

$$Q_i = A_i \times IDF$$

$$IDF_{heating} = \mathbf{0.065 \text{ L/s}}$$

$$IDF_{cooling} = \mathbf{0.032 \text{ L/s}}$$

Infiltration Rate- Winter

$$\begin{aligned} V_{i.heating} &= (481.6 \text{ cm}^2) (0.065 \text{ L/s}) \\ &= \mathbf{35.156 \text{ L/s}} \end{aligned}$$

$$\begin{aligned} V_{i.cooling} &= (481.6 \text{ cm}^2) (0.0375 \text{ L/s}) \\ &= \mathbf{18.06 \text{ L/s}} \end{aligned}$$

c) Ventilation

$$Q_v = 0.05 A_{cf} + 3.5 (N_{br} + 1)$$

$$V(\dot)_{\text{Ventilation}} = (0.05 \times 200) + (3.5 \times (1 + 1)) = \mathbf{17.0 \text{ L/s}}$$

$$\begin{aligned} V(\dot)_{\text{infiltration-ventilation heating}} &= 31.30 \text{ L/s} + 17 \text{ L/s} \\ &= \mathbf{48.3 \text{ L/s}} \end{aligned}$$

$$\begin{aligned} V(\dot)_{\text{infiltration-ventilation cooling}} &= 15.41 \text{ L/s} + 17 \text{ L/s} \\ &= \mathbf{32.41 \text{ L/s}} \end{aligned}$$

Sensible and Latent load

$$C_{sensible} = 1.23$$

$$C_{latent} = 3010$$

$$\Delta T_{heating} = 20^\circ\text{C} - 4.1^\circ\text{C} = \mathbf{15.9^\circ\text{C}}$$

$$\Delta T_{cooling} = 31.1^\circ\text{C} - 24^\circ\text{C} = \mathbf{7.1^\circ\text{C}}$$

$$\begin{aligned} Q(\dot)_{i-v \text{ heating sensible}} &= C_{sensible} V(\dot) \Delta T_{heating} \\ &= 1.23 \times 48.3 \times 15.9 \\ &= \mathbf{944.60W} \end{aligned}$$

$$\begin{aligned} Q(\dot)_{i-v \text{ cooling sensible}} &= C_{sensible} V(\dot) \Delta T_{cooling} \\ &= 1.23 \times 32.41 \times 7.1 \\ &= \mathbf{283.04W} \end{aligned}$$

$$\begin{aligned} Q(\dot)_{i-v \text{ cooling latent}} &= C_{latent} V(\dot) \Delta \omega_{cooling} \\ &= 3010 \times 32.41 \times (0.014 - 0.0095) \end{aligned}$$

$$= 438.99W$$