













Week 9 Assignment

Wednesday, December 4, 2019 12:19 PM

Task one:

Use a weather forecast website, and utilize the psychrometric chart and the formula we went through in the class to determine the absolute humidity, the wet-bulb temperature and the mass of water vapor in the air in Class Room A (Aula A) of Piacenza campus in the moment that you are solving this exercise (provide the inputs that you utilized)

Il tempo oggi in Piacenza Lunedì, 16 Dicembre 2019							
	17:00	18:00	19:00	20:00	21:00	22:00	23:00
	 Cloud	 Cloud	 Cloud	 Cloud	 Drizzle	 LightRain	 LightRain
Temperatura effettiva	7°C	7°C	6°C	6°C	6°C	6°C	7°C
Temperatura percepita	7°C	7°C	6°C	5°C	6°C	6°C	7°C
Precipitazioni	0 mm	0 mm	0 mm	0 mm	0 mm	0 mm	0 mm
Umidità	95 %	95 %	97 %	98 %	96 %	96 %	95 %
Pressione atmosferica	1021 hPa	1021 hPa	1021 hPa	1021 hPa	1021 hPa	1021 hPa	1020 hPa
Intensità del vento	3 km/h	3 km/h	4 km/h	5 km/h	4 km/h	3 km/h	2 km/h
Direzione del vento							
	NO	O	O	O	O	O	O
Probabilità di nebbia	0 %	0 %	0 %	0 %	0 %	0 %	0 %
Punto di rugiada	6°C	6°C	6°C	5°C	6°C	6°C	6°C
Nuvole	100 %	95 %	98 %	99 %	100 %	100 %	100 %
Nuvole basse	99 %	67 %	73 %	89 %	100 %	100 %	100 %
Nuvole medie	75 %	53 %	19 %	19 %	99 %	99 %	92 %
Nuvole alte	91 %	88 %	89 %	95 %	95 %	95 %	100 %

Umidità: Relative humidity, Atmospheric Pressure : Air total pressure (1 hPa: 0.1 kPa} Effective

Temperature: temperature to be utilized.

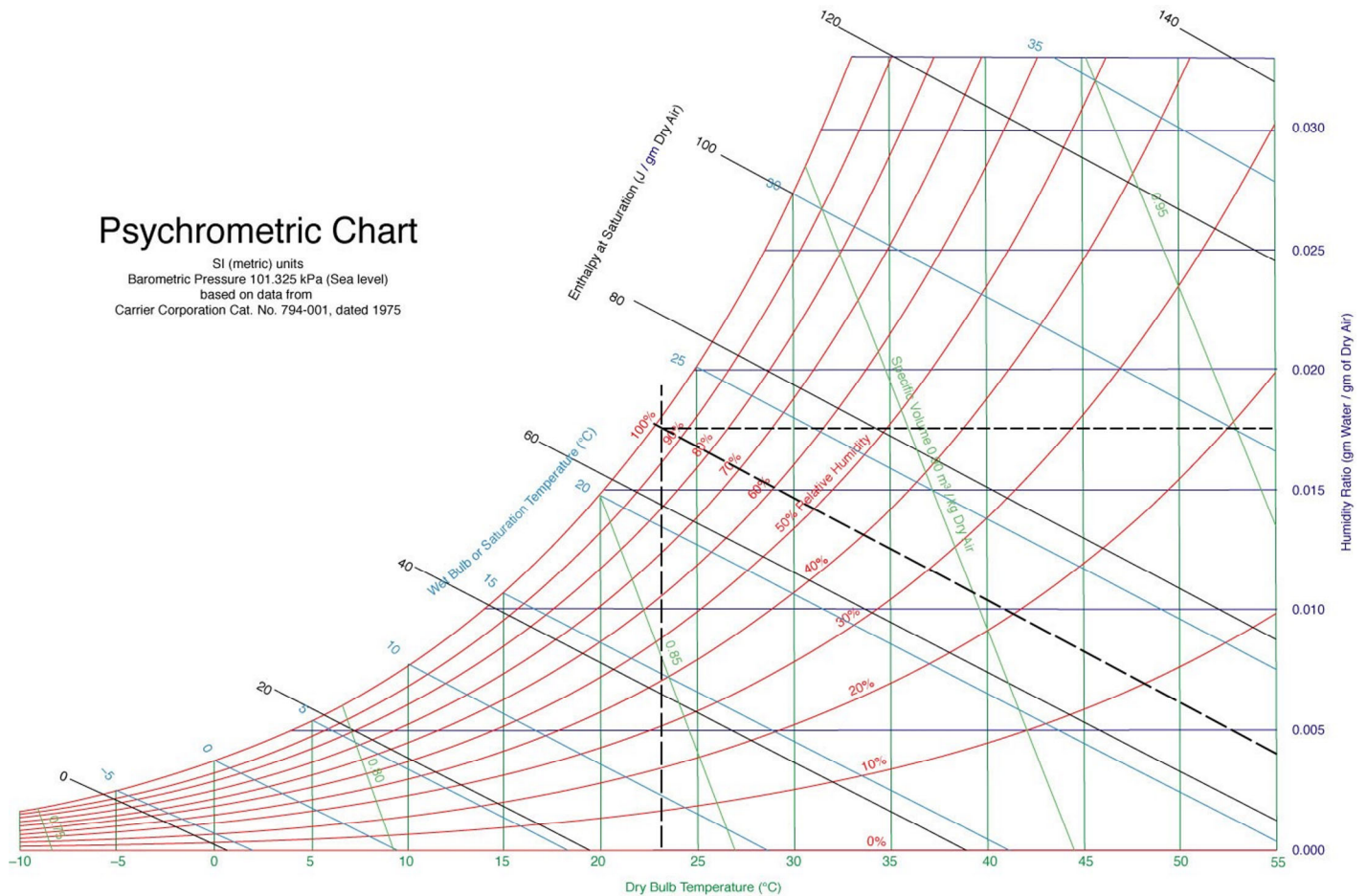
Chosen time : 19:00 Relative humidity $\Phi = 97\%$

Total air pressure = 1021hPa = 102.1kPa Temperature = 6°C

Aula A= 10m x 5m x 4m = 200 m³

Psychrometric Chart

SI (metric) units
Barometric Pressure 101.325 kPa (Sea level)
based on data from
Carrier Corporation Cat. No. 794-001, dated 1975



From the chart with the weather data we can get:

The absolute humidity $\omega = 0.0175$

Wet bulb temp T_{wb} : 22.5

$$\text{Therefore } \omega = \frac{0.622pv}{pa} = \frac{0.622pv}{p-pv} = 0.0175$$

$$P = 102.1 \text{ Kpa then, } 0.0175 = \frac{0.622pv}{102.2-pv}$$

$$\text{Then } Pv = 2.8 \text{ Kpa}$$

$$\text{For any ideal gas, } m = \frac{p \times v}{RSpT}$$

And $Rsp = 0.4615$, Volume of aula $A = 200 \text{ m}^3$

$$mv = \frac{2.8 \times 200}{0.4615 \times (273 + 6)} = 4.35$$

$$\text{Autem, } \Phi = \frac{mv}{mg} = \frac{pv}{pg} = 97\%$$

$$0.97 \times mg = mv$$

$$Mg = \frac{4.35}{0.97} = 4.48 \text{ kg}$$

Task 2:

Utilize the same methodology we went through in the class and determine the sensible and latent load corresponding to internal gains, the ventilation, and the infiltration in a house with a *good* construction quality and with the same geometry as that of the example which is located in Brindisi, Italy

BRINDISI, Italy

WMO#: 163200

Lat: 40.65N Long: 17.95E Elev: 10 StdP: 101.2 Time Zone: 1.00 (EUW) Period: 86-10 WBAN: 99999

Annual Heating and Humidification Design Conditions

Coldest Month	Heating DB		Humidification DP/MCDB and HR						Coldest month WS/MCDB				MCWS/PCWD to 99.6% DB	
			99.6%			99%			0.4%		1%			
	99.6%	99%	DP	HR	MCDB	DP	HR	MCDB	WS	MCDB	WS	MCDB	MCWS	PCWD
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)
2	2.9	4.1	-5.1	2.5	7.2	-3.0	3.0	7.4	13.4	10.2	12.4	10.6	3.4	250

Annual Cooling, Dehumidification, and Enthalpy Design Conditions

Hottest Month	Hottest Month DB Range	Cooling DB/MCWB						Evaporation WB/MCDB						MCWS/PCWD to 0.4% DB	
		0.4%		1%		2%		0.4%		1%		2%			
		DB	MCWB	DB	MCWB	DB	MCWB	WB	MCDB	WB	MCDB	WB	MCDB	MCWS	PCWD
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)	(p)
8	7.1	32.8	23.6	31.1	24.3	29.9	24.3	27.2	29.7	26.3	29.0	25.6	28.3	4.2	180

	Dehumidification DP/MCDB and HR						Enthalpy/MCDB						Hours 8 to 4 & 12.8/20.6		
	0.4%		1%		2%		0.4%		1%		2%				
	DP	HR	MCDB	DP	HR	MCDB	DP	HR	MCDB	Enth	MCDB	Enth	MCDB	Enth	MCDB
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)	(p)
26.3	21.8	29.2	25.4	20.7	28.5	24.7	19.7	27.9	86.0	30.1	82.2	29.1	78.5	28.3	1236

Extreme Annual Design Conditions

Extreme Annual WS			Extreme Max WB	Extreme Annual DB				n-Year Return Period Values of Extreme DB							
				Mean		Standard deviation		n=5 years		n=10 years		n=20 years		n=50 years	
1%	2.5%	5%		Min	Max	Min	Max	Min	Max	Min	Max	Min	Max	Min	Max
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)	(p)
11.3	9.9	8.7	31.4	0.4	37.3	1.4	3.0	-0.6	39.4	-1.4	41.1	-2.2	42.8	-3.2	44.9

Height= 2.5 m; Floor area= 200 m²; Wall area= 144 m²

Internal Gains:

Sensible cooling load from internal gains,

$$Q_{ig,sensible} = 136 + 2.2A_{cf} + 22N_{oc} = 136 + 2.2 \cdot 200 + 22 \cdot 2 = 620W$$

Latent cooling load from internal gains,

$$Q_{ig,latent} = 20 + 0.22A_{cf} + 12N_{oc} = 20 + 0.22 \cdot 200 + 12 \cdot 2 = 88W$$

Infiltration:

Unit leakage area $A_{ul} = 1.4 \text{ cm}^2/\text{m}^2$

Exposed surface $A_L = A(\text{wall}) + A(\text{roof}) = 200 + 144 = 344\text{m}^2$

$$\text{So, } A_L = A_{es} \cdot A_{ul} = 344 \cdot 1.4 = 481.6 \text{ cm}^2$$

Define the cooling temperature $T_{cooling} = 24^\circ\text{C}$, and heating temperature $T_{heating} = 20^\circ\text{C}$ in Brindisi,

$$\Delta T_{cooling} = 31.1 - 24 = 7.1^\circ\text{C} = 7.1 \text{ k}$$

$$\Delta T_{heating} = 20^\circ\text{C} - (-4.1^\circ\text{C}) = 24.1^\circ\text{C} = 24.1 \text{ k}$$

$$DR = 7.1^\circ\text{C} = 7.1 \text{ k}$$

$$\text{Given: } IDF(\text{heating}) = 0.073 \frac{L}{s \cdot \text{cm}^2}$$

$$IDF(\text{cooling}) = 0.033 \frac{L}{s \cdot \text{cm}^2}$$

Calculate infiltration airflow rate.

$$\Delta T_{\text{heating}} = 20^{\circ}\text{C} - (-4.1^{\circ}\text{C}) = 24.1^{\circ}\text{C} = 24.1 \text{ K}$$

$$\Delta R = 7.1^{\circ}\text{C} = 7.1 \text{ K}$$

$$\text{Given: IDF (heating)} = 0.073 \frac{\text{L}}{\text{s} \cdot \text{cm}^2}$$

$$\text{IDF (cooling)} = 0.033 \frac{\text{L}}{\text{s} \cdot \text{cm}^2}$$

Calculate infiltration airflow rate,

$$Q_{i, \text{heating}} = A_L * \text{IDF}_{\text{heating}} = 481.6 * 0.073 = 35.157 \frac{\text{L}}{\text{s}}$$

$$Q_{i, \text{cooling}} = A_L * \text{IDF}_{\text{cooling}} = 481.6 * 0.033 = 15.893 \frac{\text{L}}{\text{s}}$$

The required minimum whole building ventilation rate is

$$Q_V = 0.05 A_{cf} + 3.5 (N_{br} + 1) = 0.05 * 200 + 3.5 * (1+1) = 17 \frac{\text{L}}{\text{s}}$$

Thus,

$$Q_{i-v, \text{heating}} = Q_{i, \text{heating}} + Q_V = 35.157 + 17 = 52.157 \frac{\text{L}}{\text{s}}$$

$$Q_{i-v, \text{cooling}} = Q_{i, \text{cooling}} + Q_V = 15.893 + 17 = 32.893 \frac{\text{L}}{\text{s}}$$

Given that $C_{\text{sensible}} = 1.23$, $C_{\text{latent}} = 3010$, $\Delta \omega_{\text{cooling}} = 0.0039$

$$\dot{q}_{\text{inf-ventilation (cooling sensible)}} = C_{\text{sensible}} Q_{i-v, \text{cooling}} \Delta T_{\text{cooling}} = 1.23 * 32.893 * 7.1 = 287.25 \text{ W}$$

$$\dot{q}_{\text{inf-ventilation (cooling latent)}} = C_{\text{latent}} Q_{i-v, \text{cooling}} \Delta \omega_{\text{cooling}} = 3010 * 32.893 * 0.0039 = 386.13 \text{ W}$$

$$\dot{q}_{\text{inf-ventilation (heating sensible)}} = C_{\text{sensible}} Q_{i-v, \text{cooling}} \Delta T_{\text{heating}} = 1.23 * 52.157 * 24.1 = 1546.09 \text{ W}$$