

week9_MelissaMartinoli

martedì 3 dicembre 2019 23:12

01. Use a weather forecast website, and utilize the psychrometric chart and the formula we went through in the class to determine the absolute humidity, the wet-bulb temperature and the mass of water vapour in the air in Classroom A (Aula A) of Piacenza campus in the moment that you are solving this exercise (provide the inputs that you utilized)

Il tempo oggi in Piacenza Lunedì, 02 Dicembre 2019							
	13:00	14:00	16:00	18:00	20:00	21:00	22:00
	 PartlyCloud	 PartlyCloud	 LightCloud	 LightCloud	 PartlyCloud	 Cloud	 PartlyCloud
Temperatura effettiva	10°C	10°C	9°C	6°C	7°C	7°C	8°C
Temperatura percepita	10°C	10°C	8°C	5°C	7°C	6°C	7°C
Precipitazioni	0 mm	0 mm	0 mm	0 mm	0 mm	0 mm	0 mm
Umidità	79 %	77 %	89 %	90 %	90 %	92 %	91 %
Pressione atmosferica	1016 hPa	1015 hPa	1016 hPa	1017 hPa	1019 hPa	1019 hPa	1020 hPa

Humidity: 90%

Relative humidity (ϕ): 90%

Total air pressure: 101,9 KPa

Atmospheric Pressure: 1019 hPa

Temperature: 7°C = 230 K

Starting from a psychrometric chart we can see 4 different values:

Humidity ratio

I.E.

absolute humidity (ω) = 0,0055

wet-bulb temperature T_{wb} = 6 °C

$$\omega = 0.622 \frac{P_v}{P_a}$$

$$P_v = 0.893 \text{ kPa}$$

As we have learn in class we already know that $R_{sp} = 0.4615$ so we can use this data (and the previous ones) in the following formula:

$$M = \frac{P_v}{R_{sp} T}$$

$$M = \frac{0.893 \text{ V}}{0.4615 \cdot 230} = 8,41 \times 10^{-3} \text{ V}$$

Remember that $\Phi = \frac{M_v}{M_g} = 90\%$

So we can find the inverse formula $M_g = \frac{M_v}{90\%} = 9,34 \times 10^{-3} \text{ v}$

02. Utilize the same methodology we went through in the class and determine the sensible and latent load corresponding to internal gains, the ventilation, and the infiltration in a house with a good construction quality and with the same geometry as that of the example which is located in Brindisi, Italy

BRINDISI, Italy

WMO#: 163200

Lat: 40.65N Long: 17.95E Elev: 10 StdP: 101.2 Time Zone: 1.00 (EUW) Period: 86-10 WBAN: 99999

Annual Heating and Humidification Design Conditions

Coldest Month	Heating DB		Humidification DP/MCDB and HR								Coldest month WS/MCDB				MCWS/PCWD to 99.6% DB	
			99.6%				99%				0.4%		1%		MCWS	PCWD
	99.6%	99%	DP	HR	MCDB	DP	HR	MCDB	WS	MCDB	WS	MCDB				
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)		
2	2.9	4.1	-5.1	2.5	7.2	-3.0	3.0	7.4	13.4	10.2	12.4	10.6	3.4	250		

Annual Cooling, Dehumidification, and Enthalpy Design Conditions

Hottest Month	Hottest Month DB Range	Cooling DB/MCWB						Evaporation WB/MCDB						MCWS/PCWD to 0.4% DB	
		0.4%		1%		2%		0.4%		1%		2%		MCWS	PCWD
		DB	MCWB	DB	MCWB	DB	MCWB	WB	MCDB	WB	MCDB	WB	MCDB		
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)	(p)
8	7.1	32.8	23.6	31.1	24.3	29.9	24.3	27.2	29.7	26.3	29.0	25.6	28.3	4.2	180

Dehumidification DP/MCDB and HR										Enthalpy/MCDB				Hours 8 to 4 & 12 & 20	
0.4%			1%			2%			0.4%		1%		2%		
DP	HR	MCDB	DP	HR	MCDB	DP	HR	MCDB	Enth	MCDB	Enth	MCDB	Enth		MCDB
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)	(p)
26.3	21.8	29.2	25.4	20.7	28.5	24.7	19.7	27.9	86.0	30.1	82.2	29.1	78.5	28.3	1236

Extreme Annual Design Conditions

Extreme Annual WS			Extreme Max WB	Extreme Annual DB				n-Year Return Period Values of Extreme DB											
1%	2.5%	5%		Mean	Standard deviation			n=5 years		n=10 years		n=20 years		n=50 years					
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)	(p)				
11.3	9.9	8.7	31.4	0.4	37.3	1.4	3.0	-0.6	39.4	-1.4	41.1	-2.2	42.8	-3.2	44.9				

Internal gains

$$q_{ig,s} = 136 + 2,2 A_{cf} + 22 N_{oc} = 136 + 2,2 \times 200 + 12 \times 2 = 620 \text{ W}$$

$$q_{ig,l} = 20 + 0,22 A_{cf} + 12 N_{oc} = 20 + 0,22 \times 200 + 12 \times 2 = 88 \text{ W}$$

House with high construction quality unit leakage area

$$A_{ul} = 1,4 \text{ cm}^2 / \text{m}^2$$

$$A_{es} = A_{wall} + A_{roof} = 200 + 144 = 344 \text{ m}^2$$

$$A_L = A_{es} \times A_{ul} = 344 \times 1,4 = 481,6 \text{ cm}^2$$

BRINDISI DATAS:

$$T_{cooling} = 24 \text{ }^{\circ}\text{C}$$

$$T_{heating} = 20 \text{ }^{\circ}\text{C}$$

$$\Delta T_{cooling} = 31,1 \text{ }^{\circ}\text{C} - 24 \text{ }^{\circ}\text{C} = 7,1 \text{ }^{\circ}\text{C} = 7,1 \text{ K}$$

$$\Delta T_{heating} = 20 \text{ }^{\circ}\text{C} + 4,1 \text{ }^{\circ}\text{C} = 24,1 \text{ }^{\circ}\text{C} = 24,1 \text{ K}$$

$$IDF_{heating} = 0,073 \text{ L} / \text{s} \times \text{cm}^2$$

$$IDF_{cooling} = 0,033 \text{ L} / \text{s} \times \text{cm}^2$$

$$Q_{i,heating} = A_L \times IDF_{heating} = 481,6 \times 0,073 = 35,1568 \text{ L} / \text{s}$$

$$Q_{i,cooling} = A_L \times IDF_{cooling} = 481,6 \times 0,033 = 15,8928 \text{ L} / \text{s}$$

$$Q_v = 0,05 A_{cf} + 3,5 (N_{br} + 1) = 0,05 \times 200 + 3,5 \times (1 + 1) = 17 \text{ L} / \text{s}$$

$$Q_{i-v,heating} = Q_{i,heating} + Q_v = 35,1568 + 17 = 52,1568 \text{ L} / \text{s}$$

$$Q_{i-v,cooling} = Q_{i,cooling} + Q_v = 15,8928 + 17 = 32,8928 \text{ L} / \text{s}$$

$$C_{sensible} = 0,0039$$

$$Q_{inf-ventilation_coolingsensible} = C_{sensible} Q_{i-v,cooling} \Delta T_{cooling} = 1,23 \times 32,8928 \times 7,1$$

$$Q_{inf-ventilation_coolinglatent} = C_{sensible} Q_{i-v,cooling} \Delta \omega_{cooling} = 3010 \times 32,8928 \times 0,0039$$

$$Q_{inf-ventilation_heatingsensible} = C_{sensible} Q_{i-v,heating} \Delta T_{heating} = 1,23 \times 52,1568 \times 24,1$$