

WEEK 9 SUBMISSION

TASK 1

Aula A = $10m \times 5m \times 4m$

Temperature = $7^{\circ}C$

Saturation pressure of water = 1.0021 kPa

Atmospheric pressure = 102 kPa

Relative humidity = 84%

$R_v = 0.4615$

$$\phi = \frac{m_v}{m_g} = \frac{P_v}{P_g}$$

$$P_v = \phi \times P_g = 0.84 \times 1.0021 = 0.84 \text{ kPa}$$

$$P_a = P - P_v = 102 \text{ kPa} - 0.84 \text{ kPa} = 101.16 \text{ kPa}$$

Absolute humidity:

$$\omega = 0.622 \frac{P_v}{P_a} = 0.622 \frac{0.84}{101.16} = 0.0052 \frac{\text{kg}_{\text{vapour}}}{\text{kg}_{\text{dryAir}}}$$

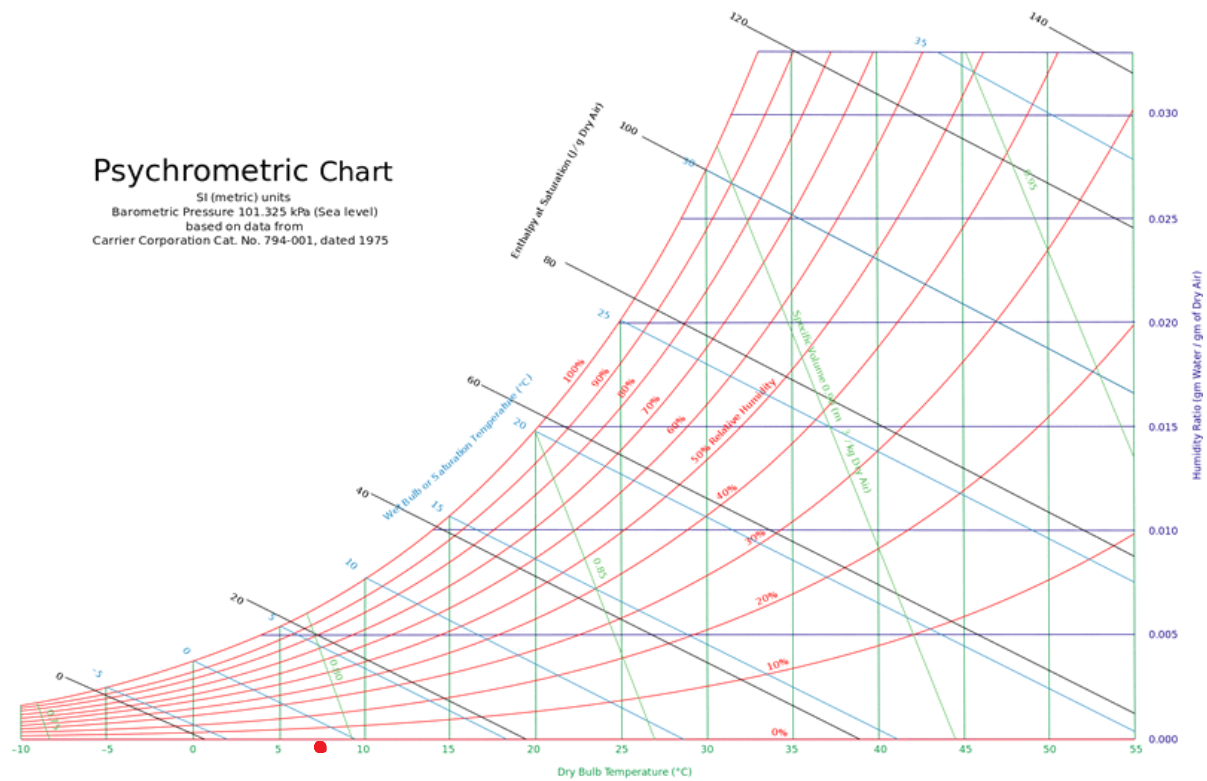
Mass of water vapor:

$$m = \frac{PV}{R_{sp}T} ; m_v = \frac{P_v V_v}{R_v T}$$

$$m_v = \frac{0.84 \times (10 \times 5 \times 4)}{0.4615 \times (273 + 7)} = 1.3 \text{ kg water vapor}$$

Enthalpy:

$$h = h_a + \omega h_v = (1.005 \times 7) + 0.0052 (2501 + (1.82 \times 7)) = 20.11 \frac{\text{kJ}}{\text{kg}_{\text{dryAir}}}$$



Wet-bulb temperature
 $\approx 5.5^{\circ}\text{C}$

TASK 2

BRINDISI, Italy

WMO#: 163200

Lat: 40.65N Long: 17.95E Elev: 10 StdP: 101.2 Time Zone: 1.00 (EUW) Period: 86-10 WBAN: 99999

Annual Heating and Humidification Design Conditions

Coldest Month	Heating DB		Humidification DP/MCDB and HR						Coldest month WS/MCDB				MCWS/PCWD to 99.6% DB	
			99.6%			99%			0.4%		1%			
	99.6%	99%	DP	HR	MCDB	DP	HR	MCDB	WS	MCDB	WS	MCDB	MCWS	PCWD
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)
2	2.9	4.1	-5.1	2.5	7.2	-3.0	3.0	7.4	13.4	10.2	12.4	10.6	3.4	250

Annual Cooling, Dehumidification, and Enthalpy Design Conditions

Hottest Month	Hottest Month DB Range	Cooling DB/MCWB						Evaporation WB/MCDB						MCWS/PCWD to 0.4% DB	
		0.4%		1%		2%		0.4%		1%		2%			
		DB	MCWB	DB	MCWB	DB	MCWB	WB	MCDB	WB	MCDB	WB	MCDB	MCWS	PCWD
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)	(p)
8	7.1	32.8	23.6	31.1	24.3	29.9	24.3	27.2	29.7	26.3	29.0	25.6	28.3	4.2	180

Dehumidification DP/MCDB and HR									Enthalpy/MCDB						Hours 8 to 4 & 12.8/20.6	
0.4%		1%		2%		0.4%		1%		2%						
DP	HR	MCDB	DP	HR	MCDB	DP	HR	MCDB	Enth	MCDB	Enth	MCDB	Enth	MCDB		
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)	(p)	
26.3	21.8	29.2	25.4	20.7	28.5	24.7	19.7	27.9	86.0	30.1	82.2	29.1	78.5	28.3	1236	

Extreme Annual Design Conditions

Extreme Annual WS			Extreme Max WB	Extreme Annual DB				n-Year Return Period Values of Extreme DB							
				Mean		Standard deviation		n=5 years		n=10 years		n=20 years		n=50 years	
1%	2.5%	5%		Min	Max	Min	Max	Min	Max	Min	Max	Min	Max	Min	Max
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)	(p)
11.3	9.9	8.7	31.4	0.4	37.3	1.4	3.0	-0.6	39.4	-1.4	41.1	-2.2	42.8	-3.2	44.9

Building height = 2.5m

Floor area = 200 m^2

Number of occupants = 2

Number of bedrooms = 1

Wall area = 144 m^2

Temperature for cooling and heating

$$T_{cooling} = 31.1 \text{ }^{\circ}\text{C}$$

$$T_{heating} = 4.1 \text{ }^{\circ}\text{C}$$

Temperature difference

$$\Delta T_{cooling} = 31.1 - 24 = 7.1 \text{ }^{\circ}\text{C}$$

$$\Delta T_{heating} = 20 - 4.1 = 15.9 \text{ }^{\circ}\text{C}$$

Internal gains

$$\dot{Q}_{ig_{sensible}} = 136 + 2.2A_{cf} + 22N_{oc} = 136 + 2.2 \times 200 + 22 \times 2 = 620 \text{ W}$$

$$\dot{Q}_{ig_{latent}} = 20 + 0.22A_{cf} + 12N_{oc} = 20 + 0.22 \times 200 + 12 \times 2 = 88 \text{ W}$$

Infiltration

$$A_{ul} = 1.4 \frac{\text{cm}^2}{\text{m}^2}$$

$$A_{es} = 200 + 144 = 344 \text{ m}^2$$

$$A_L = A_{es} \times A_{ul} = 344 \times 1.4 = 481.6 \text{ cm}^2$$

$$IDF_{heating} = 0.065 \frac{\text{L}}{\text{s} \cdot \text{cm}^2}$$

$$IDF_{cooling} = 0.032 \frac{\text{L}}{\text{s} \cdot \text{cm}^2}$$

$$\dot{Q}_{i_{heating}} = A_L \times IDF = 481.6 \times 0.065 = 31.30 \frac{\text{L}}{\text{s}}$$

$$\dot{Q}_{i_{cooling}} = A_L \times IDF = 481.6 \times 0.032 = 15.41 \frac{\text{L}}{\text{s}}$$

Ventilation

$$\dot{Q}_v = 0.05A_{cf} + 3.5(N_{br} + 1) = 0.05 \times 200 + 3.5 \times 2 = 17 \frac{\text{L}}{\text{s}}$$

$$\dot{Q}_{inf-ventilation_{heating}} = 31.30 + 17 = 48.30 \frac{\text{L}}{\text{s}}$$

$$\dot{Q}_{inf-ventilation_{cooling}} = 15.41 + 17 = 32.41 \frac{\text{L}}{\text{s}}$$

$$\dot{Q}_{inf-ventilation_{cooling_{sensible}}} = C_{sensible} \dot{V} \Delta T_{cooling} = 1.23 \times 32.41 \times 7.1 = 283.04 \text{ W}$$

$$\dot{Q}_{inf-ventilation_{cooling_{latent}}} = C_{latent} \dot{V} \Delta \omega_{cooling} = 3010 \times 32.41 \times 0.0045 = 438.99 \text{ W}$$

$$\dot{Q}_{inf-ventilation_{heating_{sensible}}} = C_{sensible} \dot{V} \Delta T_{heating} = 1.23 \times 48.30 \times 15.9 = 944.60 \text{ W}$$

$$\dot{Q}_{inf-ventilation_{heating_{latent}}} = C_{latent} \dot{V} \Delta \omega_{heating} = 3010 \times 48.30 \times 0.0046 = 668.76 \text{ W}$$