

Homework 9

Tuesday, December 3, 2019

11:39 PM

Task 1

Use a weather forecast website, and utilize the psychrometric chart and the formula we went through in the class to determine the absolute humidity, the wet-bulb temperature and the mass of water vapor in the air in Classroom A (Aula A) of Piacenza campus in the moment that you are solving this exercise (provide the inputs that you utilized)

Aula A = $10m \times 5m \times 4m$

Temperature = $7^{\circ}C$

Saturation pressure of water = 1.0021 kPa

Atmospheric pressure = 102 kPa

Relative humidity = 84%

$R_v = 0.4615$

$$\phi = \frac{m_v}{m_g} = \frac{P_v}{P_g}$$

$$P_v = \phi \times P_g = 0.84 \times 1.0021 = 0.84 \text{ kPa}$$

$$P_a = P - P_v = 102 \text{ kPa} - 0.84 \text{ kPa} = 101.16 \text{ kPa}$$

Absolute humidity

$$\omega = 0.622 \frac{P_v}{P_a} = 0.622 \frac{0.84}{101.16} = 0.0052 \frac{\text{kg}_{\text{vapour}}}{\text{kg}_{\text{dryAir}}}$$

Mass of water vapor

$$m = \frac{PV}{R_{sp}T}; m_v = \frac{P_v V_v}{R_v T}$$

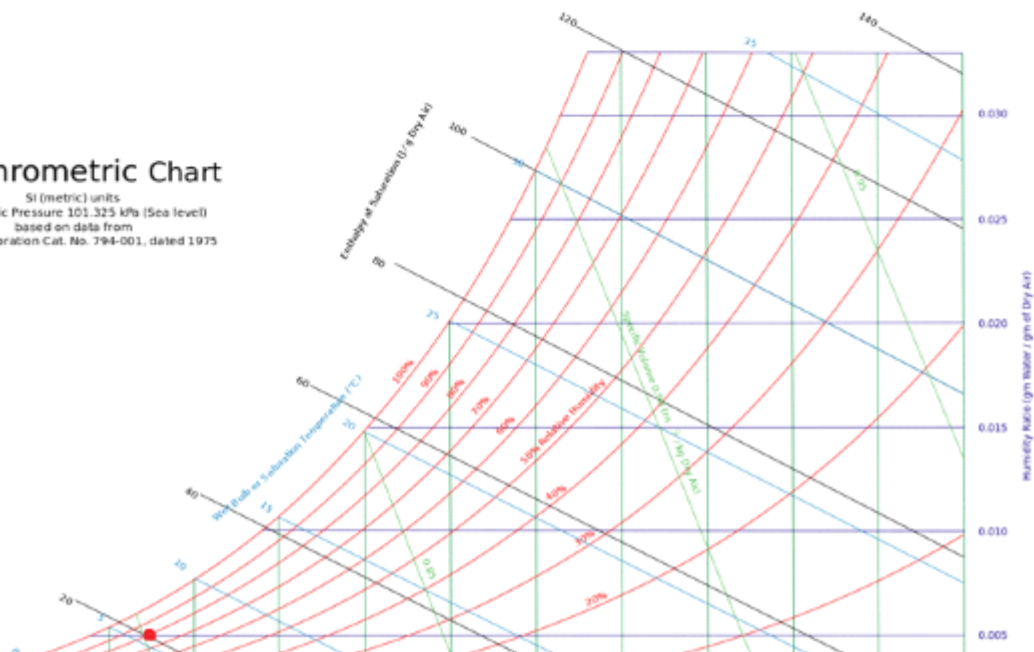
$$m_v = \frac{0.84 \times (10 \times 5 \times 4)}{0.4615 \times (273 + 7)} = 1.3 \text{ kg water vapor}$$

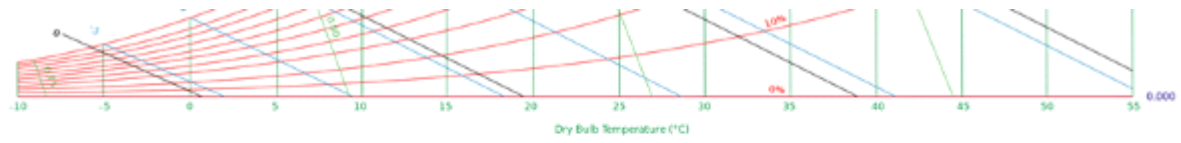
Enthalpy

$$h = h_a + wh_v = (1.005 \times 7) + 0.0052 (2501 + (1.82 \times 7)) = 20.11 \frac{\text{kJ}}{\text{kg}_{\text{dryAir}}}$$

Psychrometric Chart

SI (metric) units
Barometric Pressure 101.325 kPa (Sea level)
based on data from
Carrier Corporation Cat. No. 794-001, dated 1975





Wet-bulb temperature

$\simeq 5.5^{\circ}\text{C}$

Task 2

Utilize the same methodology we went through in the class and determine the sensible and latent load corresponding to internal gains, the ventilation, and the infiltration in a house with a good construction quality and with the same geometry as that of the example which is located in Brindisi, Italy

BRINDISI, Italy

WMO#: 163200

Lat: 40.65N Long: 17.95E Elev: 10 StdP: 101.2 Time Zone: 1.00 (EUW) Period: 86-10 WBAN: 99999

Annual Heating and Humidification Design Conditions

Coldest Month	Heating DB		Humidification DP/MCDB and HR							Coldest month WS/MCDB				MCWS/PCWD to 99.6% DB	
			99.6%			99%				0.4%		1%			
	99.6%	99%	DP	HR	MCDB	DP	HR	MCDB	WS	MCDB	WS	MCDB	MCWS	PCWD	
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)	
2	2.9	4.1	-5.1	2.5	7.2	-3.0	3.0	7.4	13.4	10.2	12.4	10.6	3.4	250	

Annual Cooling, Dehumidification, and Enthalpy Design Conditions

Hottest Month	Hottest Month DB Range		Cooling DB/MCWB						Evaporation WB/MCDB				MCWS/PCWD to 0.4% DB		
			0.4%		1%		2%		0.4%		1%				2%
	DB	MCWB	DB	MCWB	DB	MCWB	WB	MCDB	WB	MCDB	WB	MCDB	MCWS	PCWD	
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)	(p)
8	7.1	32.8	23.6	31.1	24.3	29.9	24.3	27.2	29.7	26.3	29.0	25.6	28.3	4.2	180

	Dehumidification DP/MCDB and HR									Enthalpy/MCDB					Hours 8 to 4 & 12.8/20.6	
	0.4%			1%			2%			0.4%		1%		2%		
	DP	HR	MCDB	DP	HR	MCDB	DP	HR	MCDB	Enth	MCDB	Enth	MCDB	Enth		MCDB
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)	(p)	
26.3	21.8	29.2	25.4	20.7	28.5	24.7	19.7	27.9	27.9	86.0	30.1	82.2	29.1	78.5	28.3	1236

Extreme Annual Design Conditions

Extreme Annual WS			Extreme Max WB	Extreme Annual DB				n-Year Return Period Values of Extreme DB							
				Mean		Standard deviation		n=5 years		n=10 years		n=20 years		n=50 years	
1%	2.5%	5%		Min	Max	Min	Max	Min	Max	Min	Max	Min	Max	Min	Max
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)	(p)
11.3	9.9	8.7	31.4	0.4	37.3	1.4	3.0	-0.6	39.4	-1.4	41.1	-2.2	42.8	-3.2	44.9

Building height = 2.5m

Floor area = 200 m²

Number of occupants = 2

Number of bedrooms = 1

Wall area = 144 m²

Temperature for cooling and heating

$$T_{cooling} = 31.1 \text{ }^{\circ}\text{C}$$

$$T_{heating} = 4.1 \text{ }^{\circ}\text{C}$$

Temperature difference

$$\Delta T_{cooling} = 31.1 - 24 = 7.1 \text{ }^{\circ}\text{C}$$

$$\Delta T_{heating} = 20 - 4.1 = 15.9 \text{ }^{\circ}\text{C}$$

Internal gains

$$\dot{Q}_{ig,sensible} = 136 + 2.2A_{cf} + 22N_{oc} = 136 + 2.2 \times 200 + 22 \times 2 = 620 \text{ W}$$

$$\dot{Q}_{ig,latent} = 20 + 0.22A_{cf} + 12N_{oc} = 20 + 0.22 \times 200 + 12 \times 2 = 88 \text{ W}$$

Infiltration

$$A_{ul} = 1.4 \frac{\text{cm}^2}{\text{m}^2}$$

$$A_{es} = 200 + 144 = 344 \text{ m}^2$$

$$A_L = A_{es} \times A_{ul} = 344 \times 1.4 = 481.6 \text{ cm}^2$$

$$IDF_{heating} = 0.065 \frac{L}{s \cdot \text{cm}^2}$$

$$IDF_{cooling} = 0.032 \frac{L}{s \cdot \text{cm}^2}$$

$$\dot{Q}_i = A_L \times IDF = 481.6 \times 0.065 = 31.30 \frac{L}{s}$$

$$\dot{Q}_{i_{cooling}} = A_L \times IDF = 481.6 \times 0.032 = 15.41 \frac{L}{s}$$

Ventilation

$$\dot{Q}_v = 0.05A_{cf} + 3.5(N_{br} + 1) = 0.05 \times 200 + 3.5 \times 2 = 17 \frac{L}{s}$$

$$\dot{Q}_{inf-ventilation_{heating}} = 31.30 + 17 = 48.30 \frac{L}{s}$$

$$\dot{Q}_{inf-ventilation_{cooling}} = 15.41 + 17 = 32.41 \frac{L}{s}$$

$$\dot{Q}_{inf-ventilation_{cooling_{sensible}}} = C_{sensible} \dot{V} \Delta T_{cooling} = 1.23 \times 32.41 \times 7.1 = 283.04 \text{ W}$$

$$\dot{Q}_{inf-ventilation_{cooling_{latent}}} = C_{latent} \dot{V} \Delta \omega_{cooling} = 3010 \times 32.41 \times 0.0045 = 438.99 \text{ W}$$

$$\dot{Q}_{inf-ventilation_{heating_{sensible}}} = C_{sensible} \dot{V} \Delta T_{heating} = 1.23 \times 48.30 \times 15.9 = 944.60 \text{ W}$$

$$\dot{Q}_{inf-ventilation_{heating_{latent}}} = C_{latent} \dot{V} \Delta \omega_{heating} = 3010 \times 48.30 \times 0.0046 = 668.76 \text{ W}$$