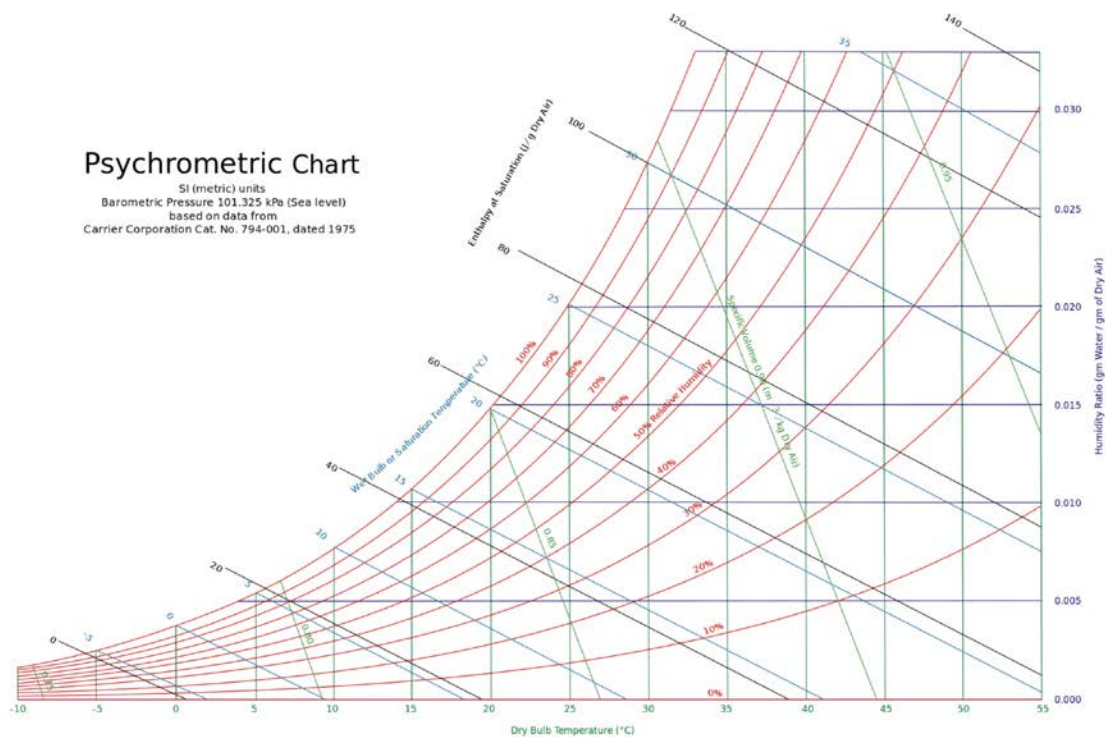


Task 1 Use a weather forecast website, and utilize the psychrometric chart and the formula we went through in the class to determine the absolute humidity, the wet-bulb temperature and the mass of water vapour in the air in Classroom A (Aula A) of Piacenza campus in the moment that you are solving this exercise (provide the inputs that you utilized)

Umidità: Relative humidity, Pressione atmosferica: Air total pressure (1 hPa: 0.1 kPa), Temperatura effettiva: temperature to be utilized.



Relative humidity $\phi=90\%$;

Air pressure $P=101.7\text{kPa}$;

Temperature 6°C

Absolute humidity: $\omega = 0.0052 \frac{\text{kg}_{\text{water}}}{\text{kg}_{\text{dryair}}}$

Wet-bulb temperature: $T_{wb} = 5.2^\circ\text{C}$

Mass of water vapor

$$V_{\text{roomA}} = 20 * 20 * 6 = 720 \text{ m}^3$$

$$P_v = \frac{p\omega}{0.622 + \omega} = \frac{101.7 * 0.0052}{0.622 + 0.0052} = 0.84 \text{ kg}$$

$$m_v = \frac{P_v * V}{R_v * T} = \frac{0.84 * 720}{0.415 * (273 + 6)} = 4.7 \text{ kg}$$

BRINDISI, Italy														WMO#: 163200	
Lat: 40.65N		Long: 17.95E		Elev: 10		StdP: 101.2		Time Zone: 1.00 (EUW)		Period: 86-10		WBAN: 99999			
Annual Heating and Humidification Design Conditions															
Coldest Month	Heating DB		Humidification DP/MCDB and HR						Coldest month WS/MCDB				MCWS/PCWD to 99.6% DB		
			99.6%			99%			0.4%		1%				
	99.6%	99%	DP	HR	MCDB	DP	HR	MCDB	WS	MCDB	WS	MCDB	MCWS	PCWD	
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)	
2	2.9	4.1	-5.1	2.5	7.2	-3.0	3.0	7.4	13.4	10.2	12.4	10.6	3.4	250	
(1)															
Annual Cooling, Dehumidification, and Enthalpy Design Conditions															
Hottest Month	Hottest Month DB Range	Cooling DB/MCWB						Evaporation WB/MCDB						MCWS/PCWD to 0.4% DB	
		0.4%		1%		2%		0.4%		1%		2%			
		DB	MCWB	DB	MCWB	DB	MCWB	WB	MCDB	WB	MCDB	WB	MCDB	MCWS	PCWD
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)	(p)
8	7.1	32.8	23.6	31.1	24.3	29.9	24.3	27.2	29.7	26.3	29.0	25.6	28.3	4.2	180
(2)															
	Dehumidification DP/MCDB and HR						Enthalpy/MCDB						Hours 8 to 4 & 12.8/20.6		
	0.4%		1%		2%		0.4%		1%		2%				
	DP	HR	MCDB	DP	HR	MCDB	DP	HR	MCDB	Enth	MCDB	Enth	MCDB	Enth	MCDB
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)	(p)
26.3	21.8	29.2	25.4	20.7	28.5	24.7	19.7	27.9	86.0	30.1	82.2	29.1	78.5	28.3	1236
(3)															
Extreme Annual Design Conditions															
Extreme Annual WS			Extreme Max WB	Extreme Annual DB				n-Year Return Period Values of Extreme DB							
				Mean		Standard deviation		n=5 years		n=10 years		n=20 years		n=50 years	
1%	2.5%	5%		Min	Max	Min	Max	Min	Max	Min	Max	Min	Max	Min	Max
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)	(p)
11.3	9.9	8.7	31.4	0.4	37.3	1.4	3.0	-0.6	39.4	-1.4	41.1	-2.2	42.8	-3.2	44.9
(4)															

Task 2 Utilize the same methodology we went through in the class and determine the sensible and latent load corresponding to internal gains, the ventilation, and the infiltration in a house with a *good* construction quality and with the same geometry as that of the example which is located in Brindisi, Italy

$h=2.5\text{m}$; $\text{area}=200 \text{ m}^2$; $\text{wall area}=144 \text{ m}^2$; $A_{ul}=1.4\text{cm}^2/\text{m}^2$

$$Q_{igsensible} = 20 + 2.2 * A_{cf} + 22N_{oc} = 136 + 2.2 * 200 + 22 * 2 = 620\text{w}$$

$$Q_{igsensible2} = 20 + 0.22 * A_{cf} + 12N_{oc} = 20 + 0.22 * 200 + 12 * 2 = 88\text{w}$$

$$IDF_{heating} = 0.06369 \frac{L}{S * cm^2}$$

$$V_{infiltrationheating} = A_l * IDF = 481.6 * 0.06369 = 30.67 \frac{L}{S}$$

$$IDF_{cooling} = 0.03188 \frac{L}{S * cm^2}$$

$$V_{infiltrationcooling} = A_l * IDF = 481.6 * 0.03188 = 15.35 \text{ L/S}$$

$$V_{ventilation} = 0.05A_{cf} + 3.5(N_{br} + 1) = 0.05 * 200 + 3.5 * 2 = 17 \frac{L}{S}$$

$$V_{infventiheat} = 30.67 + 17 = 47.67 \frac{L}{s}$$

$$V_{infventicool} = 15.35 + 17 = 32.35 \frac{L}{s}$$

$$Q_{infventicoolsens} = C_{sensible} V \Delta t_{cool} = 1.23 * 32.35 * 7.1 = 282.51w$$

$$Q_{infventicoollatent} = C_{latent} V \Delta \omega_{cool} = 3010 * 32.35 * 0.0039 = 379.75w$$

$$Q_{infventiheatsens} = C_{sensible} V \Delta t_{heat} = 1.23 * 47.67 * 15.9 = 932.28w$$

$$Q_{infventiheatlatent} = C_{latent} V \Delta \omega_{heat} = 3010 * 47.67 * 0.0065 = 932.66w$$