

Principal Strains:

ton 
$$20\rho = \frac{(\Upsilon_{xy}/2)}{(\varepsilon_{x} - \varepsilon_{y})/2} = \frac{\Upsilon_{xy}}{\varepsilon_{x} - \varepsilon_{y}}$$

$$\varepsilon_{1,2} = \varepsilon_{\text{avg}} + \Omega$$

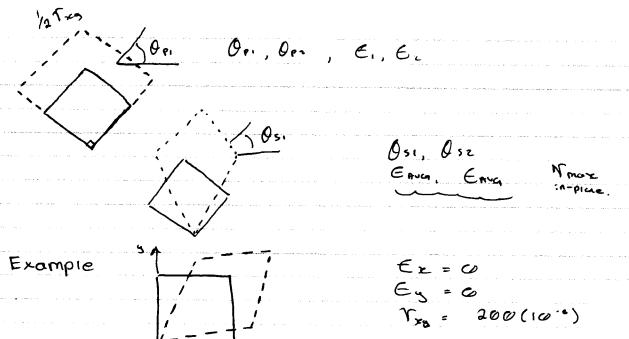
$$\varepsilon_{\text{avg}} = \frac{\varepsilon_{x} - \varepsilon_{y}}{2} + \frac{(\Upsilon_{xy})^{2}}{2} + \frac{(\Upsilon_{xy})^{2}}{2}$$

Max in-plane shear strain:

$$\frac{1}{2} \Upsilon \max_{\text{in-plane}} = R = \sqrt{\left(\frac{E_x - E_y}{2}\right)^2 + \left(\frac{\Upsilon - y}{2}\right)^2}$$

$$\tan 2\theta_s = -\frac{E_x - E_y}{\Upsilon_{xy}}$$





Find the State of strain on an element oriented at the point c.c.w. 45°

Solution: 0 = 45.

$$E_{x'} = E_{AUG} + \underbrace{E_{x}-E_{y}}_{\partial} \cos 2\theta + \underbrace{Y_{xy}}_{2} \sin 2\theta$$

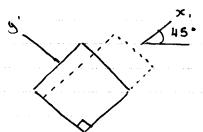
$$= 0 + 0 + \underbrace{200(_{210}^{-6})}_{2} \sin 2\theta$$

$$= (0) + (0) + \underbrace{100(_{10}^{-6})}_{2} (7)$$

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$$\frac{1}{2} \int x'y' = \frac{-Ex - Ey}{2} \sin 2\theta + \int xy \cos 2\theta$$
  
= 0 + 0 => 0



Example 
$$E_{x} = -3500 (10^{-6})$$
  
 $E_{y} = 2000 (10^{-6})$   
 $T_{xy} = 800 (10^{-6})$ 

## Determine:

a) The principle strains and associated orientations

b) The max in-plane shear strain and orientation

Earch = -75 (10.6)

$$A = 277.9 (10.6)$$
 $E = 202.9 (10.6)$ 
 $E_1 = E_{AVG} + R = (-75 + 277.9)(10.6)$ 
 $= 202.9 (10.6)$ 
 $E_2 = E_{AVG} - R$ 
 $= -352.9 (10.6)$ 
 $E_3 = E_{AVG} - R$ 
 $= -352.9 (10.6)$ 
 $E_4 - E_5$ 
 $= -0.1445$ 
 $20P = -8.28^{\circ} \text{ and } -8.28^{\circ} + 180^{\circ}$ 
 $O_7 = -4.14^{\circ} \text{ and } 85.86^{\circ}$ 

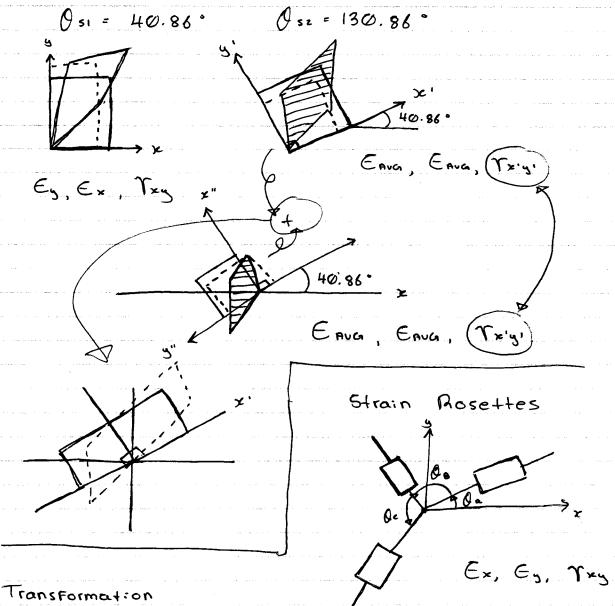
When  $O_7 = -4.14^{\circ}$ 
 $O_7 = -350.9 (10.6) = E_2$ 
 $O_7 = -352.9 (10.6) = E_2$ 

b) Max in-plane shear

1/2 T max =  $R = 277.9 (10^{-6})$ T max =  $2R = 555.8 (10^{-6})$ in-plane  $\tan 20s = -\frac{Ex - Ey}{Vxy}$ =  $-\frac{(-350 - 200)}{80}$ 

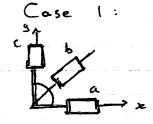
205 = 81.72° and 81.72° + 180° 05 = 40.86° and 130,86° When Q = 40.86.

## (2?) Yx'y = 277.9 (10-6)



Fansformation

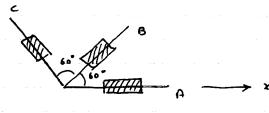
\[
\int \text{Ea} = \int \text{Cos}^2 \text{Da} + \int \text{Ey} \text{Sin}^2 \text{Da} + \text{Txy} \text{Sin}^2 \text{Da} + \text{Txy} \text{Sin} \text{Da} \text{Cos} \text{Da} \\
\int \text{E} = \int \text{Cos}^2 \text{Dc} + \int \text{Ey} \text{Sin}^2 \text{Dc} + \text{Txy} \text{Sin} \text{Dc} \text{Cos} \text{Dc} \\
\int \text{E} = \int \text{Cos}^2 \text{Dc} + \int \text{Ey} \text{Sin}^2 \text{Dc} + \text{Txy} \text{Sin} \text{Dc} \text{Cos} \text{Dc} \\
\int \text{E} = \int \text{Cos}^2 \text{Dc} + \int \text{Ey} \text{Sin}^2 \text{Dc} + \text{Txy} \text{Sin} \text{Dc} \text{Cos} \text{Dc} \\
\int \text{Txy} \text{Sin} \text{Dc} \text{Dc} \\
\int \text{Txy} \text{Sin} \text{Dc} \text{Dc} \\
\int \text{Dc} \text{Dc} \text{Dc} \\
\int \text{Dc} \text{Dc} \text{Dc} \\
\int \text{Dc} \text{Dc} \\
\int \text{Dc} \text{Dc} \\
\int \text{Dc} \text{Dc} \\
\int \text{Dc} \\\
\



$$\begin{array}{cccc}
Q_{\mu} &= & \bigcirc^{\circ} \\
Q_{\nu} &= & 45^{\circ} \\
Q_{c} &= & Q\bigcirc^{\circ}
\end{array}$$

and

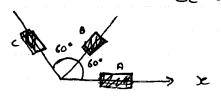
Case 2



.. Ex = Ea

Example :

Determine principle Strains and orientations.



From previous lecture -

Determine the principal strains and orientations.

Solution: 
$$(E_x = E_a)$$

$$(E_y = \frac{1}{3}(2E_b + 2E_c - E_a))$$

$$(F_{xy} = \frac{2}{\sqrt{3}}(E_b - E_c))$$

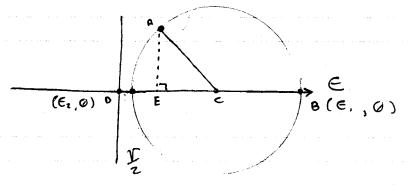
=> 
$$E_x = 60(10^{-6})$$
  
 $E_y = 246(10^{-6})$   
 $Y_{xy} = -149(10^{-6})$ 

Mohrs circle:  

$$E_{\text{aug}} = \underbrace{E_{\text{x}} + E_{\text{y}}}_{2} = \underbrace{60 + 246}_{2} (10^{-6})$$

$$= (53(10^{-6}))$$

:. Centre 
$$C(Eaug, 0) = C(153(10^{-6}), 0)$$
  
Reference Point  $A(Ex, 12 Txy)$   
=  $(60 \times 10^{-6}, -74.5 \times 10^{-6})$ 



R = 
$$\sqrt{\frac{(x-69)^2}{2}}$$
 ... etc. use Formula.

R =  $119.2(10^{-6})$ 

$$E_{1} = E_{AVG} + R = 153(10^{-6}) + 119.20(10^{-6})$$

$$= 272.2(10^{-6})$$

$$E_{2} = E_{AVG} - R = 153(10^{-6}) - (19.20(10^{-6}))$$

$$= 33.8(10^{-6})$$

$$\triangle$$
ACE: AE = 74.5 (10<sup>-6</sup>)

CE = 153 (10<sup>-6</sup>) - 60 (10<sup>-6</sup>)

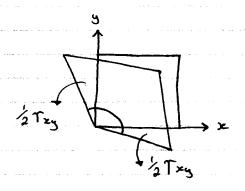
= 93 (10<sup>-6</sup>)

 $\therefore$  ton  $\angle$ ACE = AE = 74.5 (10<sup>-6</sup>)

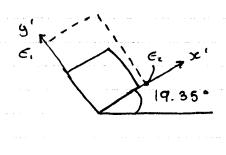
CE 93 (10<sup>-6</sup>)

 $\angle$ ACE = 38.7°

$$\theta_{P1} = 19.35^{\circ}$$
 and  $\theta_{P1} = 90^{\circ} + 19.35^{\circ} = 109.35^{\circ}$ 



Ez = 02



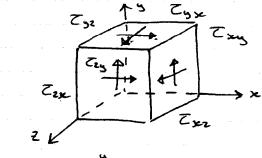
## 10.6 Material Property Relationships

General Hookes Law
4 A 3D State of Stress

Normal stress and normal strains 0x, 0y, 0z 0x 0x

$$\begin{cases} Ex = \frac{Gx}{E} - \frac{VGy}{E} - \frac{VGz}{E} = \frac{1}{E} \left[Gx - V(Gy + Gz)\right] \\ Ey = -\frac{VGx}{E} + \frac{Gy}{E} - \frac{2Gz}{E} = \frac{1}{E} \left[Gy - V(Gz + Gz)\right] \\ Ez = -\frac{VGx}{E} - \frac{VGy}{E} + \frac{Gz}{E} = \frac{1}{E} \left[Gz - V(Gz + Gy)\right] \end{cases}$$

Shear Stress and Shear strain:



Principal Stress and principal Strain:

Xyz +> Principal Stress directions

Txy = Zyz = Tzx = 0

Hooke's Law

Txy = Tyz = Yzx = 0

Principal Stress directions

Furthermore,  $\sigma_{1}, \sigma_{2}, \sigma_{3}$ ;  $\varepsilon_{1}, \varepsilon_{2}, \varepsilon_{3}$   $\varepsilon_{2} = \frac{1}{E} \left[\sigma_{1} - V(\sigma_{2} + \sigma_{3})\right]$   $\varepsilon_{3} = \frac{1}{E} \left[\sigma_{2} - V(\sigma_{3} + \sigma_{3})\right]$  $\varepsilon_{3} = \frac{1}{E} \left[\sigma_{3} - V(\sigma_{2} + \sigma_{3})\right]$  Plane Stress and plane strain

Hoose's Law:  

$$E_{z} = \frac{1}{E} \left[ G_{z} - V \left( G_{x} + G_{y} \right) \right] = \frac{V}{E} \left( G_{x} + G_{y} \right)$$

$$\neq \emptyset$$

Not a plane strain.

Since 
$$E_2 = \frac{1}{E} \left[ \delta_2 - \mathcal{V}(\delta_{x} * \delta_{y}) \right] = \emptyset$$

(modulus of elasticity)

$$= \Rightarrow \qquad G = \frac{E}{2(1+V)}$$

5:nce 
$$G_{x'} = Z$$
,  $G_{y'} = -Z$ 

$$= \frac{1}{2} \int_{\mathbb{R}^{n}} \int_{\mathbb{R}^{n}} \left( Z + \mathcal{V} Z \right) = \underbrace{1 + \mathcal{V}}_{\mathbb{E}} Z$$

$$= \frac{1}{2} \int_{\mathbb{G}} \int_{\mathbb{R}^{n}} \frac{1 + \mathcal{V}}{\mathbb{E}} dx$$

$$= \frac{1}{2} \int_{\mathbb{R}^{n}} \frac{1 + \mathcal{V}}{\mathbb{E}} dx$$

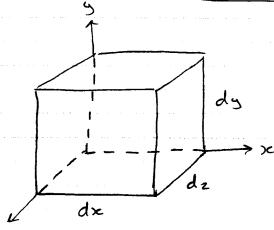
$$= \frac{1}{2} \int_{\mathbb{R}^{n}} \frac{1 + \mathcal{V}}{\mathbb{E}} dx$$

$$= \frac{1}{2} \int_{\mathbb{R}^{n}} \frac{1 + \mathcal{V}}{\mathbb{E}} dx$$

## Dilation and Burk Modules

$$\begin{aligned}
& \in_{\mathbf{x}} : & \in_{\mathbf{x}} &= \frac{d\mathbf{u}}{d\mathbf{x}} \\
& d\mathbf{u} &= \in_{\mathbf{x}} d\mathbf{x}
\end{aligned}$$

$$dx' = du + dx = (1 + \epsilon_x) dx$$



$$E_x$$
,  $E_y$ ,  $E_z$ 

$$dx \rightarrow dx' = (1+\epsilon_x)dx$$

$$dy \rightarrow dy' = (1+\epsilon_y)dy$$

$$dz \rightarrow dz' = (1+\epsilon_z)dz$$

Volume oid du = dxdydz
new du' = dx'dy'dz'

Volume change  

$$\delta V = dV' - dV$$

$$= (1 + E_x)(1 + E_y)(1 + E_z) dxdydz - dV$$

$$= (1 + E_x + E_y + E_z + E_x E_y + E_y E_z + E_z E_x$$

$$+ E_x E_y E_z) dV - dV$$

$$= (E_x + E_y + E_z) dV$$
Volumetric Strain  $\longleftrightarrow$  dilation

Hooke's Law
$$E_{x} = \frac{1}{E} (G_{x} - V(G_{y} + G_{z}))$$

$$E_{y} = \frac{1}{E} (G_{y} - V(G_{z} + G_{z}))$$

$$E_{z} = \frac{1}{E} (G_{z} - V(G_{z} + G_{y}))$$

$$E_{x} + E_{y} + G_{z} = \frac{1 - 2\nu}{E} \left( \sigma_{x} + \sigma_{y} + \sigma_{z} \right)$$

$$\frac{P}{e} = -\frac{3(1-2v)}{E} P$$

DeF:ne

$$R = \frac{E}{3(1-2\nu)}$$
but Modulus
$$= P = -Ke$$

No volume change

$$1 - 2v = \emptyset$$

$$V = \frac{1}{2}$$