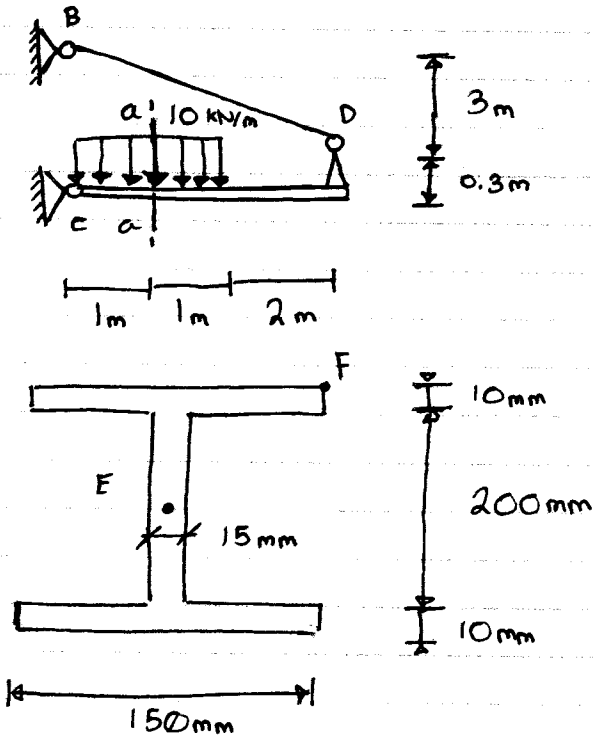


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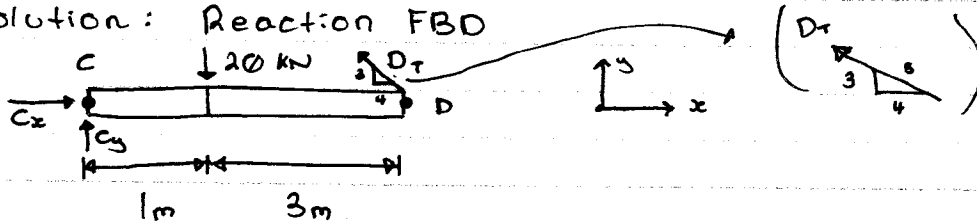
Assignment #1 due Jan. 23RD in class.
Chapter 8: 6, 9, 16, 20, 38, 49

EXAMPLE:



Determine the state of stress at points E and F at section a-a.

Solution: Reaction FBD



$$\sum M_C = 0$$

$$-20 \text{ kN}(1\text{m}) + (3/5) D_T (4\text{m}) + (4/5) D_T (0.3\text{m}) = 0$$

$$D_T [(3/5)(0.3) + (4/5)(4)] = 20$$

$$D_T = 7.576 \text{ kN}$$

$$\sum F_x = 0$$

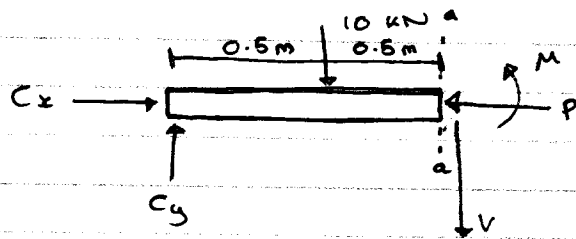
$$-(4/5) D_T + C_x$$

$$C_x = 6.0608 \text{ kN}$$

$$\sum F_y = 0$$

$$+ (3/5) D_T + C_y$$

$$C_y = 15.4545 \text{ kN}$$



$$P = 6.0606 \text{ kN}$$

$$V_0 = 5.4545 \text{ kN}$$

$$M = 10.4545 \text{ kN}\cdot\text{m}$$

Geometric properties of section a-a:

E is the centroid of the section.

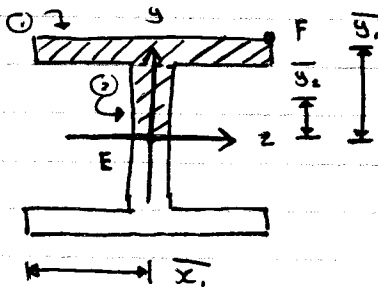
$$A = (150 \times 10 \times 2) + (200 \times 15) = 16000 \text{ mm}^2$$

$$= 6000 \times 10^{-6} \text{ m}^2 = 6 \times 10^{-3} \text{ m}^2$$

$$I = \left(\frac{1}{12}\right)(15)(200)^3 + 2 \left[\left(\frac{1}{12}\right)(150)(10)^3 + (150)(10)(105)^2 \right]$$

$$= 43.1 \times 10^6 \text{ mm}^4$$

$$= 43.1 \times 10^{-6} \text{ m}^4$$



$$Q_E = A_1 \bar{y}_1 + A_2 \bar{y}_2$$

$$= (150)(10)(105) + (100)(15)(60)$$

$$= 0.2325 (10^6) \text{ mm}^3$$

$$= 0.2325 \times 10^{-3} \text{ m}^3$$

$$Q_F = 0$$

At E:

Normal Stress

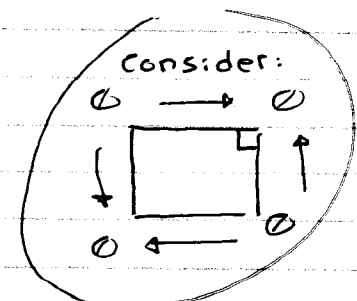
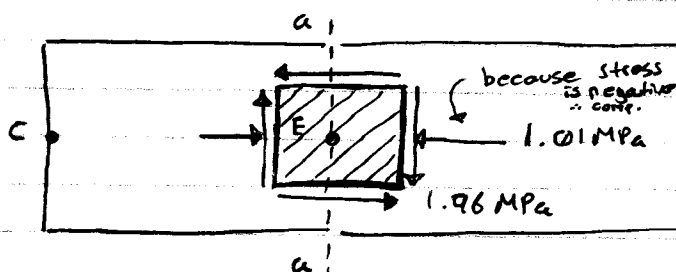
$$\sigma_E = \frac{-6.0606 \text{ kN}}{6 \times 10^{-3} \text{ m}^2} + \frac{10.4545 \text{ kN}}{43.1 \times 10^{-6}} (0)$$

$$\sigma_E = -1.0101 \text{ MPa}$$

Shear Stress

$$\tau_E = \frac{V Q_E}{I t} = \frac{(5.4545 \text{ kN})(0.2325 \times 10^{-3} \text{ m}^3)}{(43.1 \times 10^{-6} \text{ m}^4)(15 \times 10^{-3} \text{ m})}$$

$$\tau_E = 1.96 \text{ MPa}$$



A + F:

Normal Stress

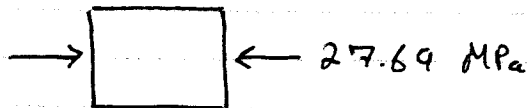
$$\sigma_F = \frac{-6.0606 \text{ kN}}{6 \times 10^{-3} \text{ m}^2} - \frac{10.4545 \text{ kN}}{43.1 \times 10^{-6} \text{ m}^4} (\phi.11 \text{ m})$$

$$\sigma_F = -27.69 \text{ MPa}$$

Shear Stress

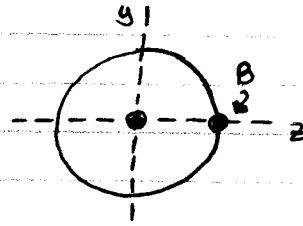
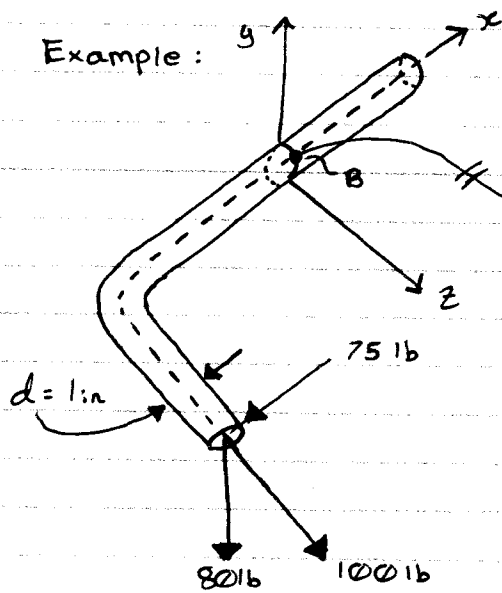
$$\tau_F = \frac{VQ_F}{It} = 0$$

State of stress



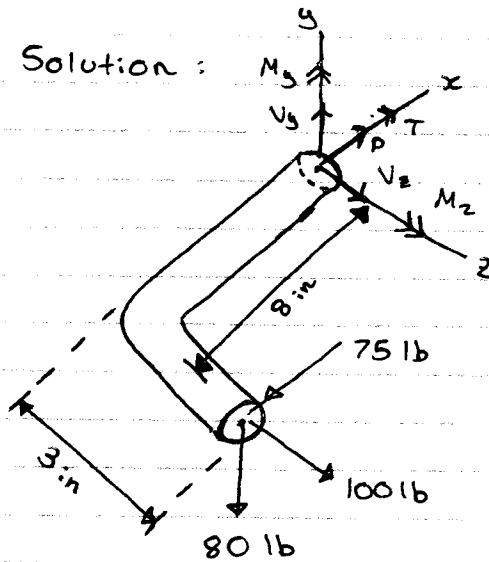
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Example:



Determine the state of stress at B.

Solution:



$$\sum F_x = 0$$

$$P - 75 = 0 \quad P = 75 \text{ lb}$$

$$\sum F_y = 0$$

$$V_y - 80 = 0 \quad V_y = 80 \text{ lb}$$

$$\sum F_z = 0$$

$$V_z + 100 = 0 \quad V_z = -100 \text{ lb}$$

$$\sum M_x = 0$$

$$0 = T + (80 \text{ lb})(3 \text{ in})$$

$$T = -240 \text{ lb} \cdot \text{in}$$

$$\sum M_y = 0$$

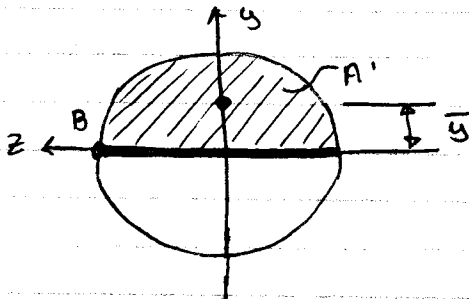
$$0 = M_y - (75 \text{ lb})(3 \text{ in}) + (100 \text{ lb})(8 \text{ in})$$

$$M_y = -575 \text{ lb} \cdot \text{in}$$

$$\sum M_z = 0$$

$$0 = M_z + (80 \text{ lb})(8 \text{ in})$$

$$M_z = -640 \text{ lb} \cdot \text{in}$$



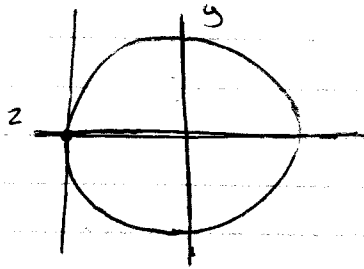
$$\begin{aligned} Q_{Bz} &= A' \bar{y} \\ &= \left(\frac{1}{2}\right) \pi r^2 \times \frac{4r}{3\pi} \\ &= 0.083333 \end{aligned}$$

$$\begin{aligned} I_y &= I_z = \left(\frac{1}{4}\right) \pi r^4 \\ &= \left(\frac{1}{4}\right) \pi (0.5)^4 \\ &= 0.049087 \end{aligned}$$

$$\begin{aligned} J_c &= I_y + I_z = \frac{1}{2} \pi (r)^4 \\ &= 0.098174 \end{aligned}$$

$$\begin{aligned} A &= \pi r^2 = \pi (0.5)^2 \\ &= 0.78540 \end{aligned}$$

(2)



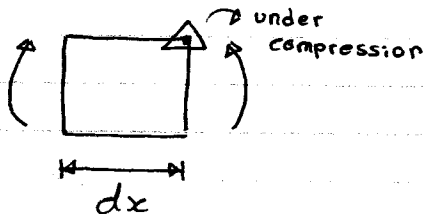
$$Q_{By} = A' \bar{z} = 0$$

Normal:

$$\text{Axial Force } \sigma_B = \frac{P}{A} = \frac{75}{0.78540} = 95.493$$

Bending moment M_y

$$\sigma_{B2} = - \left| \frac{M_y}{I_y} z \right| \Rightarrow - \left| \frac{-575}{0.04908} \cdot (0.5) \right| = -5846.9$$



Bending moment M_z

$$\sigma_{B3} = \left| \frac{M_z}{I_z} y \right| = 0 \quad \text{distance from neutral axis}$$

$$\therefore \sigma_B = \sigma_{B1} + \sigma_{B2} + \sigma_{B3} = 95.493 - 5846.8 = -5751.3 \text{ psi}$$

Shear:

$$\text{Torque } T = -240 \text{ lb} \cdot \text{in}$$

$$\tau_{B1} = \frac{T}{J} \rho = \frac{240}{0.098174} \cdot (0.5) = 1223.3 \uparrow$$

Shear Force V_y

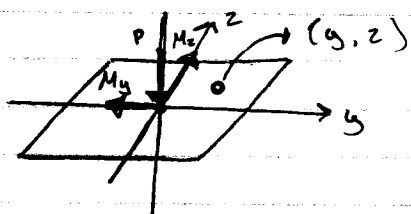
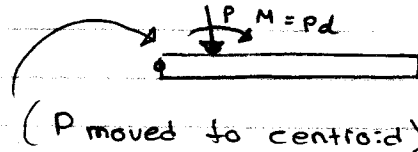
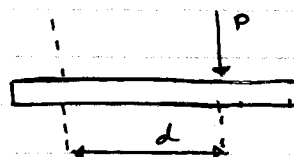
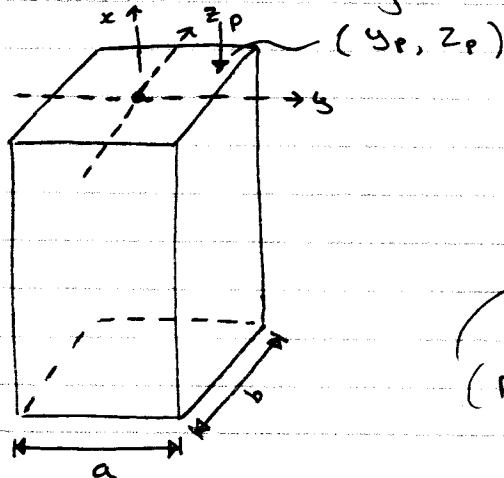
$$\begin{aligned}\tau_{B2} &= \frac{V_y Q_{B2}}{I_z t} \\ &= \frac{(80) \cdot (0.683333)}{(0.049087) \cdot (1)} \\ &= 135.8\end{aligned}$$

Shear force V_z

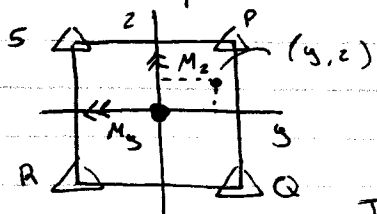
$$\tau_{B3} = \frac{V_z Q_{B3}}{I_y t} = 0$$

$$\begin{aligned}\therefore \tau_B &= \tau_{B1} + \tau_{B2} + \tau_{B3} \\ &= 1359.1 \text{ psi}\end{aligned}$$

Core of a rectangular cross section



$$\begin{aligned}M_y &= P z_p \\ M_z &= P \cdot y_p\end{aligned}$$



Normal Stress:

$$\sigma = -\frac{P}{A} - \frac{M_y}{I_y} z - \frac{M_z}{I_z} y$$

$$I_y = \left(\frac{1}{12}\right) a b^3 = \left(\frac{1}{12}\right) b^2 \times a b = \left(\frac{1}{12}\right) A b^2$$

$$I_z = \left(\frac{1}{12}\right) a^3 b = \left(\frac{1}{12}\right) A a^2$$

$$\therefore \sigma = -P/A - \frac{P y_p}{\frac{1}{12} A b^2} z - \frac{P z_p}{\frac{1}{12} A a^2} y$$

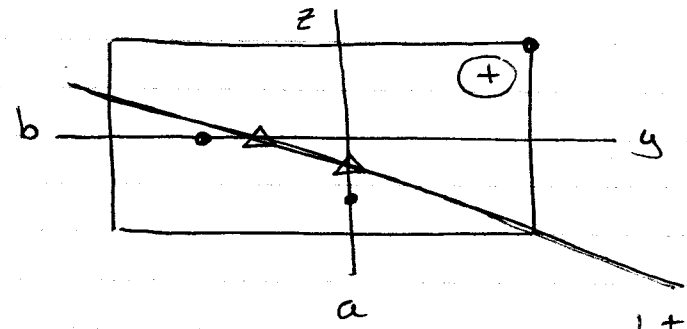
$$= -P/A \left(1 + \frac{12 y_p}{b^2} z + \frac{12 z_p}{a^2} y \right) \leq 0 \quad \curvearrowright$$

$$\Rightarrow 1 + \frac{12y_p}{b^2} z + \frac{12z_p}{a^2} y \geq 0$$

At P: $y = \frac{1}{2}a, z = \frac{1}{2}b$

$$\Rightarrow 1 + \frac{12y_p}{b^2} \times \frac{1}{2}b + \frac{12z_p}{a^2} \times \frac{1}{2}a \geq 0$$

$$\Rightarrow 1 + \frac{6}{b} y_p + \frac{6}{a} z_p \geq 0$$



$$1 + \frac{6}{b} y_p + \frac{6}{a} z_p = 0$$