

①

Feb. 4/19

(2-27) • using eq'n (2-24) $\rightarrow T(x=0) = T_{s,1} = C_2$

Now, sub. eq'n (2-27) in (2-26), gives

(2-28) $KC_1 = h(C_2 - T_\infty)$

Solving For C_1 & C_2 (finally), gives

(2-29)a $T(x) = \left(\frac{L-x}{L + (k/h)} \right) (T_\infty - T_{s,2}) + T_{s,2}$ verify!

Remarks:

• $T_{s,1}$ can be determined using eq (2-27)

(2-29)b • $T_{s,1} = \left(\frac{L}{L + k/h} \right) (T_\infty - T_{s,2}) + T_{s,2}$

Table (2-1): Standard BC's For 1-D, Steady-state, heat cond.

① convection BC

(each equation is obtained by performing energy balance over the surface)

(2-37) @ $x=0$

$-K dT/dx|_{x=0} = h(T_\infty - T_{s,1})$

(2-38) @ $x=L$

$-K dT/dx|_{x=L} = h(T_{s,2} - T_\infty)$

② Specified heat flux

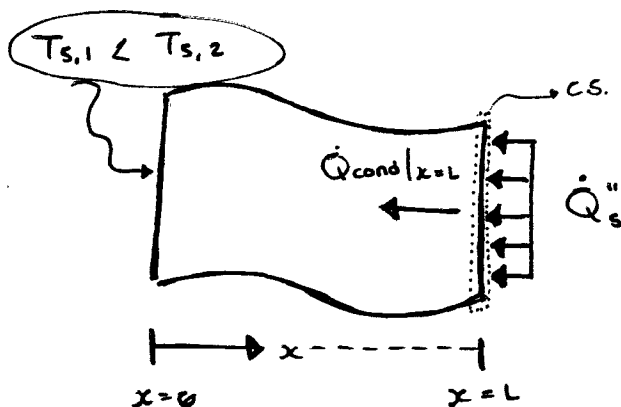
$(\dot{Q}/A = \dot{Q}''_s \text{ } \overset{\text{double prime}}{\text{true in}} \text{ } x\text{-dir})$

(2-39a) @ $x=0$

$\dot{Q}''_s = -K dT/dx|_{x=0}$

(2-40) @ $x=L$

$-K dT/dx|_{x=L} = \dot{Q}''_s$



$\dot{E}_{in} - \dot{E}_{out} + \dot{E}_{gen} = \dot{E}_{st}$
 $\dot{E}_{in} = \dot{E}_{out}$

$\dot{Q}_s = \dot{Q}_{cond}|_{x=L}$
 $= -KA \frac{dT}{dx}|_{x=L}$

$\dot{Q}''_s A = -KA \frac{dT}{dx}|_{x=L}$

$\rightarrow \dot{Q}''_s = -K \frac{dT}{dx}|_{x=L}$

③ Insulated Surface (ideal)

(2-41) @ $x = 0$

$$0 = -k \left(\frac{dT}{dx} \right) \Big|_{x=0}$$

$\leftarrow \dot{Q}_{in} = \dot{Q}_{out} \rightarrow$ (surface energy balance at $x = 0$)

(2-42) @ $x = L$

$$-k \left(\frac{dT}{dx} \right) \Big|_{x=L} = 0$$

④ Black-body radiative HT

(2-43) @ $x = 0$

$$\epsilon E_s (T_{surr}^4 - T_{s,1}^4) = -k \frac{dT}{dx} \Big|_{x=0}$$

\leftarrow emissivity of the surface

(2-44) @ $x = L$

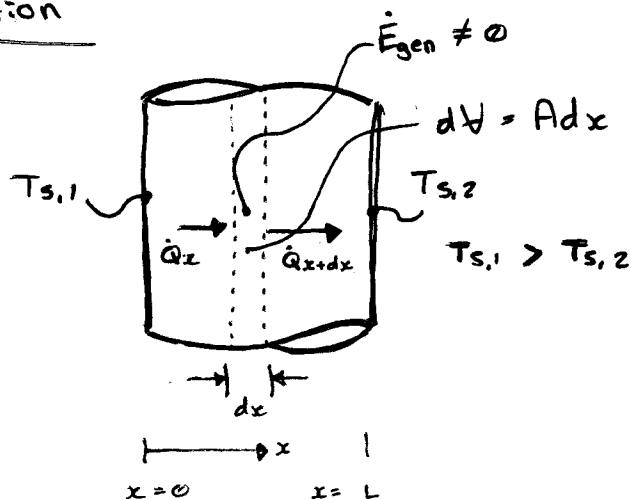
$$-k \frac{dT}{dx} \Big|_{x=L} = \epsilon E_s (T_{s,2}^4 - T_{surr}^4)$$

2.3 Conduction Heat Transfer with Internal Energy Generation

2.3.1:

Conditions (assumptions):

- 1-D (in x -dir)
- Steady state HT
- $k = \text{const.}$



Formulations:

Application of energy balance for the shown diff. volume (dV) gives:

$$\dot{E}_{in} - \dot{E}_{out} + \dot{E}_{gen} = \dot{E}_{st}$$

$\rightarrow = 0$ (steady-state)

(2-110)a $\dot{Q}_x - \dot{Q}_{x+dx} + \dot{E}_{gen} = 0$

Recall, the def'n of a derivative

(2-110)b $\dot{Q}_{x+dx} = \dot{Q}_x + d\dot{Q}_x/dx$

sub eq'n (2-110)b in (2-110)a, gives

(2-111) $-d\dot{Q}_x/dx \, dx + \dot{E}_{gen} = 0$

{ Fig (2-17)a : Conduction w/ $\dot{E}_{gen} \neq 0$

Recall,

$$(2-112) \quad \dot{e}_{\text{gen}} = \frac{\dot{E}_{\text{gen}}}{dV} = \frac{\dot{E}_{\text{gen}}}{A dx}$$

Fourier's Law of Conduction

$$(2-113) \quad \dot{Q}_x = -KA(dT/dx)$$

Sub eqs (2-112) & (2-113) in (2-111), gives

$$-d/dx (-KA dT/dx) dx + \dot{e}_{\text{gen}} dV = 0$$

$$KA(d^2T/dx^2) dx + \dot{e}_{\text{gen}} (A dx) = 0$$

Rearranging gives

$$(2-114) \quad \boxed{\frac{d^2T}{dx^2} + \frac{\dot{e}_{\text{gen}}}{K} = 0}$$

$$(2-115)a \quad \underline{2 \text{ BC's:}} \quad * T(x=0) = T_{s,1}$$

$$(2-115)b \quad * T(x=L) = T_{s,2}$$

$$\frac{d}{dx} \left(\frac{dT}{dx} \right) = - \frac{\dot{e}_{\text{gen}}}{K}$$

$$\text{Integrating once:} \quad dT/dx = -(\dot{e}_{\text{gen}}/K)x + C_1$$

$$(2-116) \quad \text{Integrating twice:} \quad T(x) = -(\dot{e}_{\text{gen}}/2K)x^2 + C_1x + C_2$$

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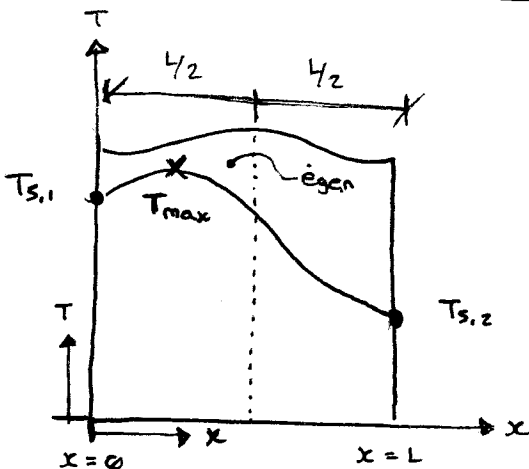
C_1 & C_2 can be determined by applying 2 BC's as follows:

(2-117)a 1st BC: $C_2 = T_{s,1}$

(2-117)b 2nd BC: $C_1 = \frac{\dot{e}_{gen} L}{2k} + \frac{(T_{s,2} - T_{s,1})}{L}$ Verify!

Sub C_1 & C_2 using eq's (2-117)a,b in eq (2-116), gives

(2-118)
$$T(x) = \frac{-\dot{e}_{gen} x^2}{2k} + \frac{\dot{e}_{gen} L x}{2k} + \frac{T_{s,2} - T_{s,1}}{L} x + T_{s,1}$$



Verify w/ $T(0)$, $T(L)$
 $T(0) = T_{s,1}$, $T(L) = T_{s,2}$

$$T_{s,1} > T_{s,2}$$

- non-linear profile

$\dot{e}_{gen} = \text{const.} = \text{uniform}$

* Temp. profile for the case
 $(T_{s,1} > T_{s,2})$ given

* Eq. (2-118) can also be simplified for the case

$T_{s,1} = T_{s,2}$ to

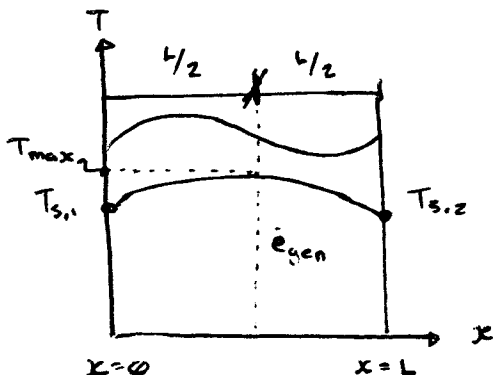
(2-119)

$$T(x) = \frac{\dot{e}_{gen} L^2}{2k} \left[\left(\frac{x}{L} \right) - \left(\frac{x}{L} \right)^2 \right] + T_{s,1}$$

Just sub. $T_{s,1} = T_{s,2}$ - rewritten (2-118)

* NOTE when $\dot{e}_{gen} = 0$, eq (2-118) reduces to the case previously dealt with ($\dot{e}_{gen} = 0$)

* The temp. profile for the case $T_{s,1} = T_{s,2}$ is parabolic and symmetric about the centerline



$\dot{e}_{gen} = \text{constant or uniform throughout}$

(*NOTE that (For the above special case) at the centerline ($L/2 = x$), $T = T_{max}$ and $dT/dx = 0$ as if the plane wall looks like:

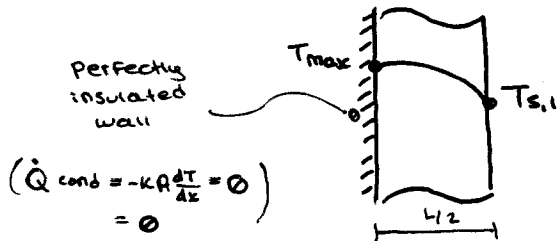


Fig (2-17)d

T_{max} is given by ($x = L/2, T_{s,1} = T_{s,2}$)

$$(2-120) \quad T_{max} = \frac{\dot{E}_{gen} L^2}{8k} + T_{s,1}$$

T_{max} with symmetric B.C. ($T_{s,1} = T_{s,2}$)

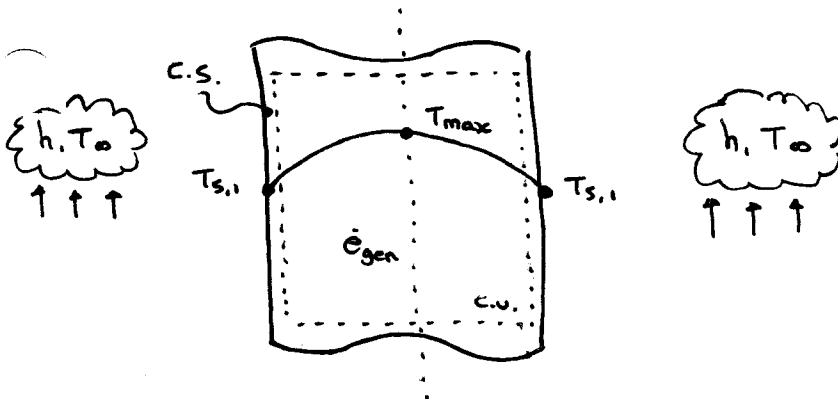


Fig (2-17)e

Application of Energy Balance for the shown C.V., for this case

$$(2-122) \quad \dot{E}_{gen} = \dot{E}_{out}$$

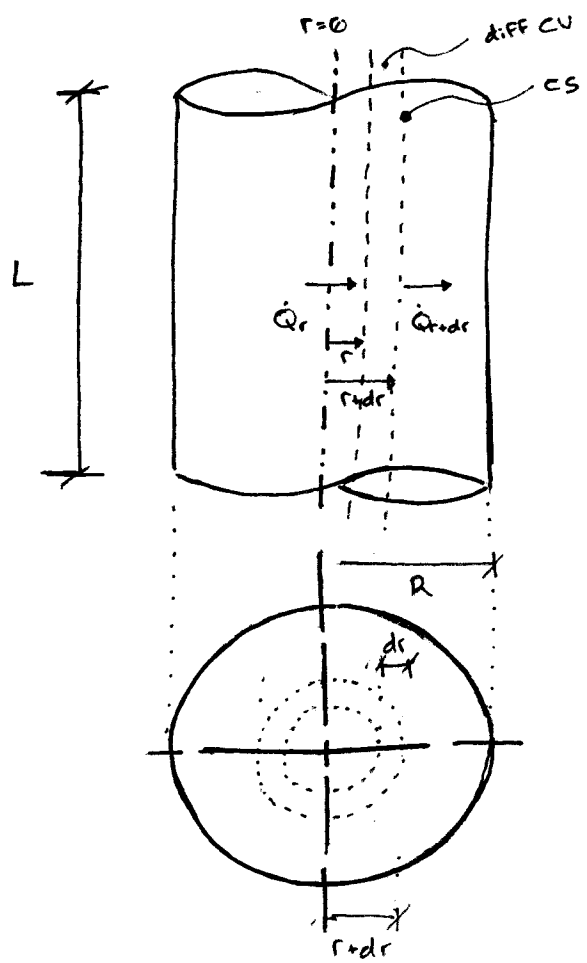
$$\dot{E}_{gen} (AL) = hA(T_{s,1} - T_{\infty})$$

Solving for $T_{s,1}$ gives

$$(2-123) \quad T_{s,1} = \frac{\dot{E}_{gen} L}{h} + T_{\infty}$$

(2-2)B Conduction Heat Transfer in Radial Walls (cylinder)

Consider steady state 1-D (in r-dir) heat conduction in a radial wall with no heat generation ($\dot{E}_{gen} = 0$), as shown:



$$A_r = 2\pi r L$$

$$dV = A_r dr = 2\pi r dr L$$

Fig. (2-11)

Formulations

- $T(r) = ?$
- $\dot{Q}_r = ?$

Applying energy balance For the shown diff. CV (previously), gives

$$\dot{E}_{in} - \dot{E}_{out} + \dot{E}_{gen} = \dot{E}_{st}$$

$\swarrow \begin{matrix} = 0 \\ \text{given} \end{matrix}$
 $\searrow \begin{matrix} = 0 \\ \text{steady state} \end{matrix}$

$$\dot{E}_{in} - \dot{E}_{out} = 0$$

$$\dot{E}_{in} = \dot{E}_{out}$$

(2-77)

$$\dot{Q}_r = \dot{Q}_{r+dr}$$

Using the definition of a derivative

$$(2-78) \quad \dot{Q}_{r+dr} \equiv \dot{Q}_r + d\dot{Q}_r/dr \, dr$$

Sub eq(2-78) in (2-77) gives

$$(2-79) \quad \boxed{\frac{d\dot{Q}_r}{dr} = 0}$$

• This indicates that $\dot{Q}_r \neq f(r) = \text{const.}$

using Fourier's Law of Conduction, we get
 (eq 2-7a)

$$(2-80) \quad \dot{Q}_r = -k A_r \frac{dT}{dr} \Big|_{r=r}$$

$$\frac{d}{dr} \underbrace{\left(-k A_r \frac{dT}{dr} \right)}_{\dot{Q}_r} = 0$$

• consider that $k = \text{const.}$

$$(2-81) \quad \frac{d}{dr} \left(A_r \frac{dT}{dr} \right) = 0$$

$$(2-82) \quad \frac{d}{dr} \left(2\pi r L \frac{dT}{dr} \right) = 0$$

NOTE:
 $(A_r = 2\pi r L)$

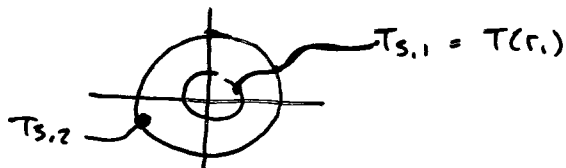
$$(2-83) \quad \boxed{\frac{d}{dr} \left(r \frac{dT}{dr} \right) = 0}$$

The result is valid only for steady-state, $\dot{E}_{\text{gen}} = 0$, 1-D

Next step is to obtain $T(r)$ by solving the diff. eqn given by eq(2-83)

Eq(2-83) is 2nd-order ODE, we need 2 B.C.s

Consider a hollow thick cylinder as shown:



$$(2-84)a \quad 1^{st} \text{ BC} \Rightarrow T(r=r_1) = T_{s,1}$$

$$(2-84)b \quad 2^{nd} \text{ BC} \Rightarrow T(r=r_2) = T_{s,2}$$

Integrating eq. (2-83) once, gives

$$(2-85)a \quad r \frac{dT}{dr} = C_1$$

$$(2-85)b \quad dT = \frac{C_1}{r} dr$$

Integrating again, gives

$$(2-86) \quad T(r) = C_1 \ln r + C_2$$

Applying the 2 B.C.'s in this eq'n (2-86 gives):

$$(2-87)a \quad T(r=r_1) = T_{s,1} = C_1 \ln r_1 + C_2 \quad \text{and}$$

$$(2-87)b \quad T(r=r_2) = T_{s,2} = C_1 \ln r_2 + C_2$$

Solving for C_1 & C_2 and then sub the results in eq (2-86), gives:

(2-88)

$$T(r) = \frac{T_{s,2} - T_{s,1}}{\ln(r_2/r_1)} \ln(r/r_2) + T_{s,2}$$

Verify!