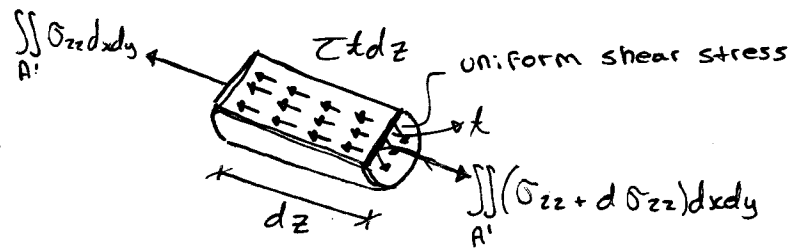
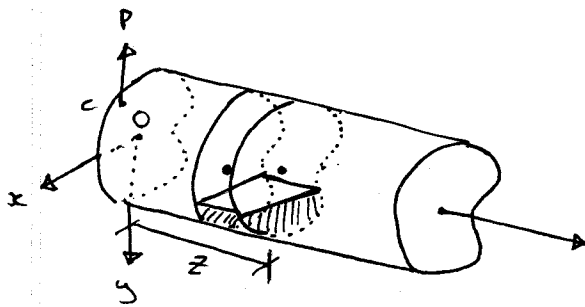


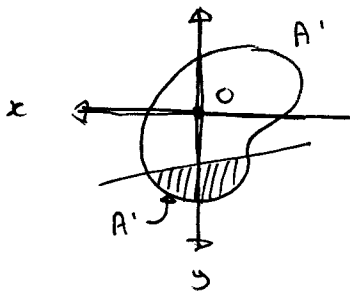
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$$\sum F_z = 0 :$$

$$\iint_{A'} (\sigma_{zz} + d\sigma_{zz}) dx dy - \iint_{A'} \sigma_{zz} dx dy - \tau dz = 0$$

$$= q = \tau dz = \iint_{A'} \frac{d\sigma_{zz}}{dz} dx dy$$



$$\text{Since } \sigma_{zz} = \frac{M_x I_y + M_y I_{xy}}{\Delta} y - \frac{M_y I_x + M_x I_{xy}}{\Delta} x$$

$$\Rightarrow \frac{d\sigma_{zz}}{dz} = \frac{\frac{dM_x}{dz} I_y + \frac{dM_y}{dz} I_{xy}}{\Delta} y - \frac{\frac{dM_y}{dz} I_x + \frac{dM_x}{dz} I_{xy}}{\Delta} x$$

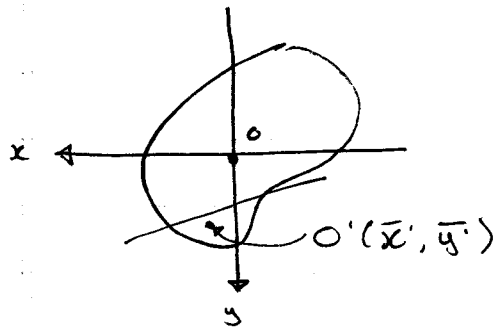
$$\text{because of: } \frac{dM_x}{dz} = V_y \quad \frac{dM_y}{dz} = -V_x$$

$$\Rightarrow \frac{d\sigma_{zz}}{dz} = \frac{V_y I_y - V_x I_{xy}}{\Delta} y - \frac{-V_x I_x + V_y I_{xy}}{\Delta} x$$

$$= \frac{V_y I_y - V_x I_{xy}}{\Delta} y + \frac{V_x I_x + V_y I_{xy}}{\Delta} x$$

$$\Rightarrow q = \tau dz = \iint_{A'} \frac{d\sigma_{zz}}{dz} dx dy$$

$$= \frac{V_y I_y - V_x I_{xy}}{\Delta} \iint_{A'} y dx dy + \frac{V_x I_x + V_y I_{xy}}{\Delta} \iint_{A'} x dx dy$$



$$\iint_{A'} y dx dy = A' \bar{y}$$

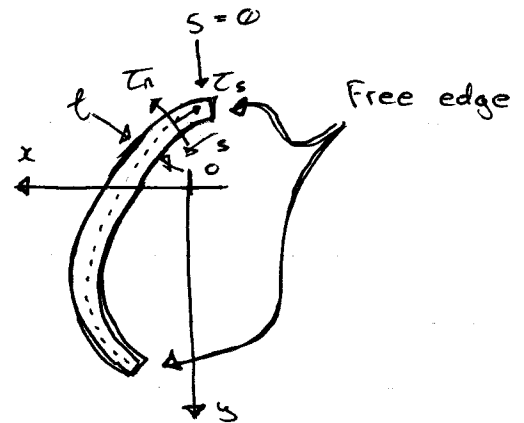
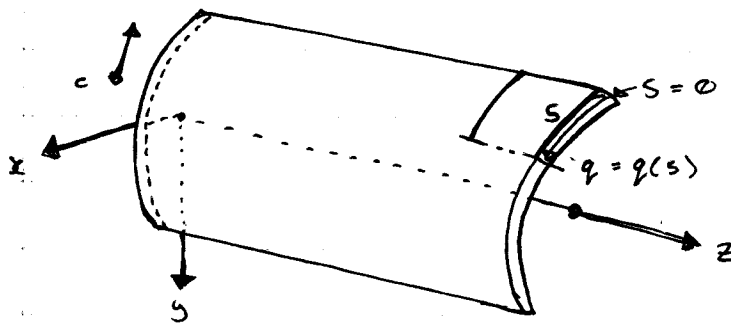
$$\iint_{A'} x dx dy = A' \bar{x}$$

$$q = \tau t = \frac{V_y I_y + V_x I_{xy}}{\Delta} A' \bar{y} + \frac{V_x I_x + V_y I_{xy}}{\Delta} A' \bar{x}$$

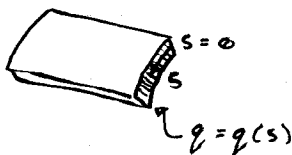
If  $I_{xy} = 0$ ,  $q = (V_y / I_x) A' \bar{y} + (V_x / I_y) A' \bar{x}$

If  $V_x = 0$ ,  $q = (V_y / I_x) A' \bar{y}$

## 2. Thin-wall open section

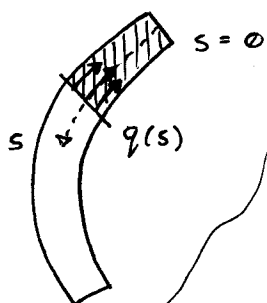


- 1°  $\tau_n = 0$  (thickness very small)
- 2°  $\tau_s$  is uniform through thickness of the wall
- 3°  $q = \tau t$  : shear flow
- 4° Shear flow is zero at the free edge

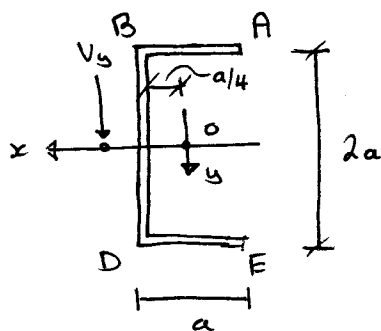
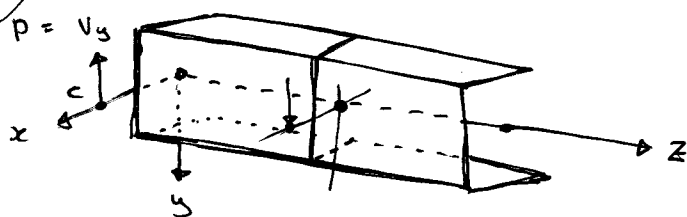


Positive Shear Flow:

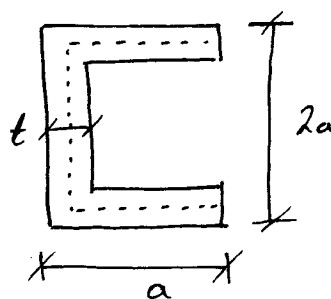
the shear flow points into the area



Example: Determine the shear flow in a C channel section due to a shear force  $V_y$  through its shear center.



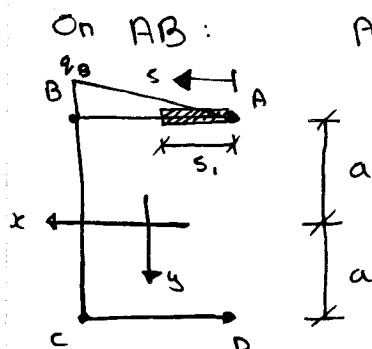
$$(t \ll a)$$



$$q = \tau t = \frac{V_y}{I_x} A' \bar{y}'$$

$$I_x = \left(\frac{1}{12}\right) t (2a)^3 + \left[\left(\frac{1}{12}\right) (a) (t)^3 + a t \cdot a^2\right] \times 2$$

$$= \left(\frac{8}{3}\right) a^3 t \quad (at^3 \ll a^3 t)$$



On AB:

$$A' = s \cdot t$$

$$\bar{y}' = -a$$

$$q(s) = \frac{V_y}{I_x} A' \bar{y}'$$

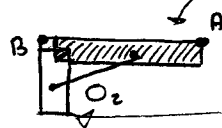
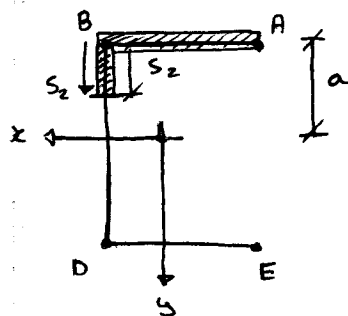
$$= \frac{V_y}{\frac{8}{3} a^3 t} (s \cdot t) (-a) = -\frac{3V_y (s)}{8a^2}$$

(when  $0 \leq s \leq a$ )

at B,  $s = a$

$$q_B = -\frac{3V_y}{8a^2} (a) = -\frac{3V_y}{8a}$$

On BD:



$$A' \bar{y}' = A \bar{y}_1' + A_2 \bar{y}_2'$$

$$\begin{aligned} q(s_2) &= q_B + \frac{V_y}{I_x} A_2 \bar{y}_2' \\ \Rightarrow \frac{-3V_y}{8a} - \frac{V_y}{8/3 a^2} \cdot (s_2 t)(a - s_2/2) \\ &= \frac{3V_y}{8a} \left( \frac{1}{2} \left( \frac{s_2}{a} \right)^2 - \frac{s_2}{a} - 1 \right) \end{aligned}$$

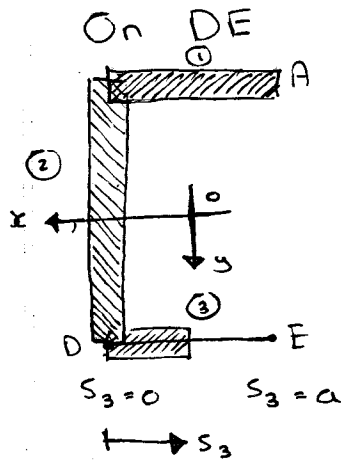
$$A_2 = S_2 t$$

$$\bar{y}_2' = -(a - s_2/2)$$

At D,  $S_2 = 2a$

$$q_D = q(2a) = -\frac{3V_y}{8a} = q_B$$

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$$q(s_3) = (V_y / I_x) A' \bar{y}'$$

$$A' \bar{y}' = A_1' \bar{y}_1' + A_2' \bar{y}_2' + A_3' \bar{y}_3'$$

$$q(s_3) = \frac{V_y}{I_x} [A_1' \bar{y}_1' + A_2' \bar{y}_2' + A_3' \bar{y}_3']$$

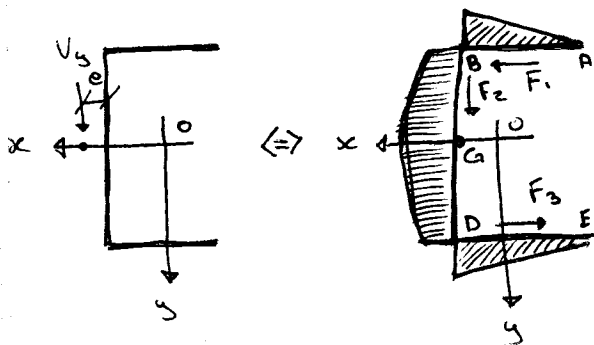
$$= q_D + \frac{V_y}{I_x} A_3' \bar{y}_3'$$

$$q(s_3) = -\frac{3V_y}{8a} + \frac{V_y}{8a^3} s_3 t$$

$$= -\frac{3V_y}{8a} + \frac{3V_y}{8a} \cdot \frac{s_3}{a}$$

@ E,  $s_3 = a$

$$\therefore q_E = -\frac{3V_y}{8a} + \frac{3V_y}{8a} = 0$$



Since  $q_B = \frac{3V_y}{8a}$

$$\therefore F_i = \frac{1}{2} q_B \cdot AB$$

$$= \frac{1}{2} \left( \frac{3V_y}{8a} \right) a = \frac{3}{16} V_y$$

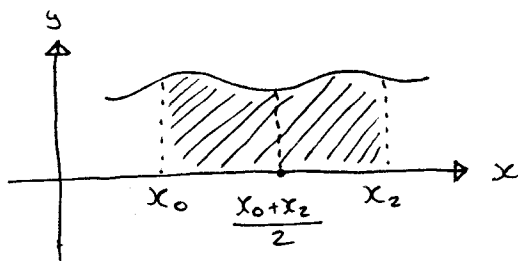
D is the moment center

$$F_i \cdot 2a = V_y e$$

$$\Rightarrow e = \frac{F_i \cdot 2a}{V_y} = \frac{\frac{3}{16} V_y \cdot 2a}{V_y} = \frac{3a}{8}$$

negative?

$$F_2 = -\int_0^{2a} q(s_2) ds_2$$



$$\int_{x_0}^{x_2} f(x) dx = \frac{x_2 - x_0}{6} (f_0 + 4f_1 + f_2)$$

$$f_0 = f(x_0), \quad f_1 = f\left(\frac{x_0 + x_2}{2}\right)$$

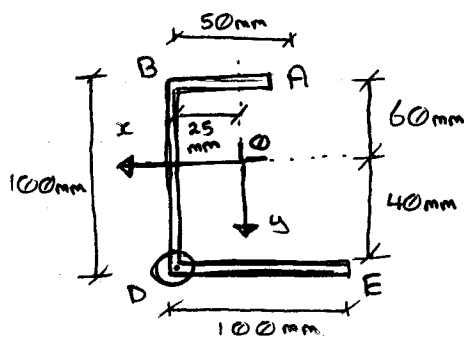
$$f_2 = f(x_2)$$

$$F_2 = \frac{2a}{6} (q_B + 4q_G + q_D)$$

$$= \frac{a}{3} \left( \frac{-8V_y}{8a} - 4 \left( \frac{9V_y}{16a} \right) - \frac{3V_y}{8a} \right)$$

$$F_2 = -V_y$$

Example: Find the Shear Center of a C-section.



$$t = 4 \text{ mm}$$

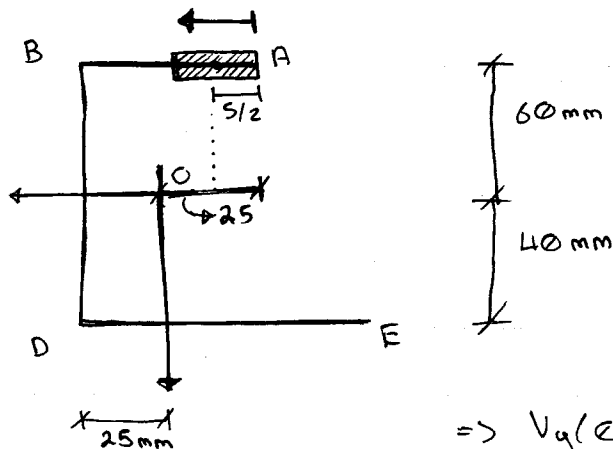
Solution: 1° Find the centroid and moment of area about centroid axes

$$I_x = 1.733 \times 10^6 \text{ mm}^4$$

$$I_y = 0.876 \times 10^6 \text{ mm}^4$$

$$I_{xy} = -0.500 \times 10^6 \text{ mm}^4$$

2° Find internal Shear Flow due to the internal Shear Forces  $V_x$  and  $V_y$



$$\text{At } 0 \leq s \leq 50$$

$$A' = st$$

$$\text{Centroid } O' (s/2 - 25, -60)$$

$$\therefore q(s) = \frac{V_y I_y - V_x I_{xy}}{I} A' \bar{y}' + \dots$$

$$\dots + \frac{V_x I_x - V_y I_{xy}}{I} A' \bar{x}'$$

$$\Rightarrow \frac{V_y (0.876)(10^6) - V_x (-0.500)(10^6) \cdot S (4)(-60)}{(1.268 \times 10^{12})} + \dots$$

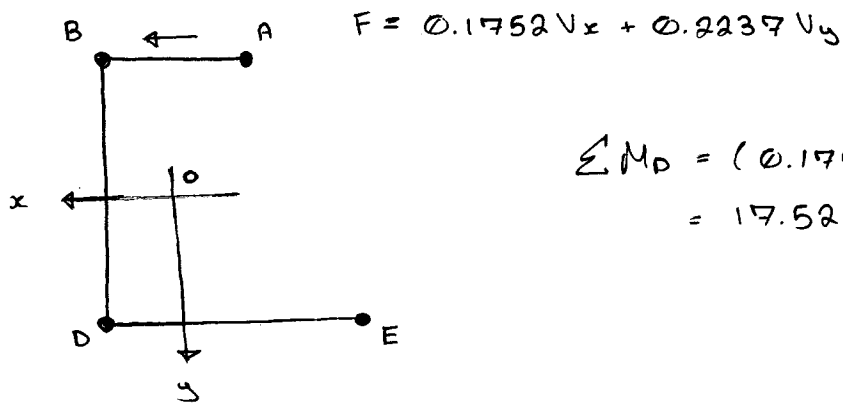
$$\dots \frac{V_x (1.733)(10^6) - V_y (-0.500)(10^6) S (4)(s/2 - 25)}{(1.268 \times 10^{12})}$$

$$\Rightarrow V_x [2.733(s - 50) - 94.635](10^6) + V_y [0.788(s - 50) - 165.795](10^6)$$

Resultant on AB :

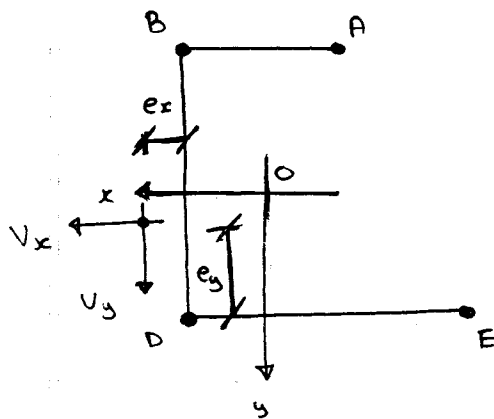
$$F = \int_0^{50} q(s) ds$$

$$= -(0.1752 V_x + 0.2237 V_y)$$



$$\sum M_D = (0.1752 V_x + 0.2237 V_y)(100)$$

$$= 17.52 V_x + 22.37 V_y$$



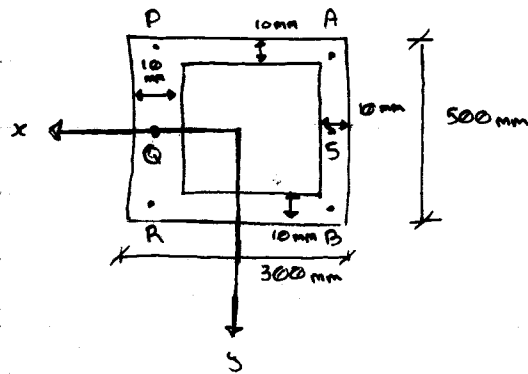
Shear Force  $V_x$  and  $V_y$

$$\sum M_D = V_x e_y + V_y e_x$$

$$\Rightarrow \begin{cases} e_y = 17.52 \text{ mm} \\ e_x = 22.37 \text{ mm} \end{cases}$$

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**Example** Box Beam



1° thin wall

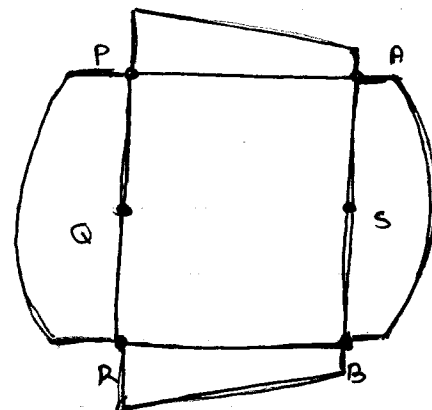
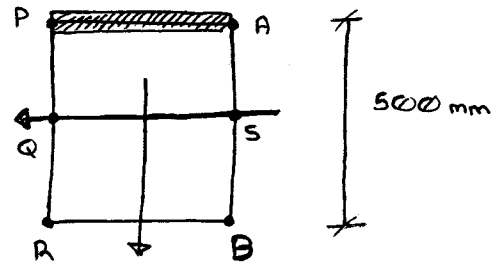
2° X-axis is Symmetric axis

3° only internal shear force

$V_y$  is needed

Take  $V_y = I_x$

4° cut at point A



2 & 4° cond'd

$$\begin{aligned} q_P &= q_A + \frac{V_y}{I_x} A' \bar{y}' \\ &= q_A + (300)(10)(-250) \\ &= q_A - 7500000 \end{aligned}$$

$$\begin{aligned} q_Q &= q_P + \frac{V_y}{I_x} A' \bar{y}' \\ &= (q_A - 7500000) + (250)(20)(-125) \\ &= q_A - 13750000 \end{aligned}$$

$$q_R = q_P ; q_B = q_A$$

$$\begin{aligned} \rightarrow q_S &= q_B + \frac{V_y}{I_x} A' \bar{y}' \\ &= q_A + (250)(10)(125) \\ &= q_A + 3125000 \end{aligned}$$

Angle of twist (per unit length) :

$$\theta = \frac{1}{2GA} \oint q/t \, dl = 0$$

$$\Rightarrow \int_{AP} q/t \, dl + \int_{PR} q/t \, dl + \int_{RB} q/t \, dl + \int_{BA} q/t \, dl$$

$$\Rightarrow \frac{1}{10} \int_{AP} q \, dl + \frac{1}{20} \int_{PR} q \, dl + \frac{1}{10} \int_{RB} q \, dl + \frac{1}{10} \int_{BA} q \, dl$$

$$\begin{aligned} \int_{AP} q \, dl &= \frac{1}{2} (q_R + q_P) (AP) = \frac{1}{2} (q_A + q_A - 7500000) (300) \\ &= 300 q_A - 1125000000 \end{aligned}$$



$$\begin{aligned}
 \int_{PA} q dl &= \frac{PR}{6} (q_P + 4q_Q + q_R) \\
 &= \frac{500}{6} (q_A - 750000 + 4(q_A - 1375000) + q_A - 750000) \\
 &= 500 q_A - 20(2916667)
 \end{aligned}$$

$$\begin{aligned}
 \int_{AB} q dl &= \frac{1}{2} (q_A + q_B) (RB) \\
 &= 300 (q_A) - 112500000
 \end{aligned}$$

$$\begin{aligned}
 \int_{BA} q dl &= \frac{AB}{6} (q_B + 4q_S + q_A) \\
 &= 500 (q_A) + 10416667 \times 10
 \end{aligned}$$

$$\begin{aligned}
 \Rightarrow & \left(\frac{1}{10}\right) (300 q_A - 112500000) \\
 & + \left(\frac{1}{20}\right) (500 q_A - 20(2916667)) \\
 & + \left(\frac{1}{10}\right) (300 q_A - 112500000) \\
 & + \left(\frac{1}{10}\right) (500 q_A + 104166670) = 0
 \end{aligned}$$

$$\Rightarrow q_A = 305556 \text{ N/mm}$$

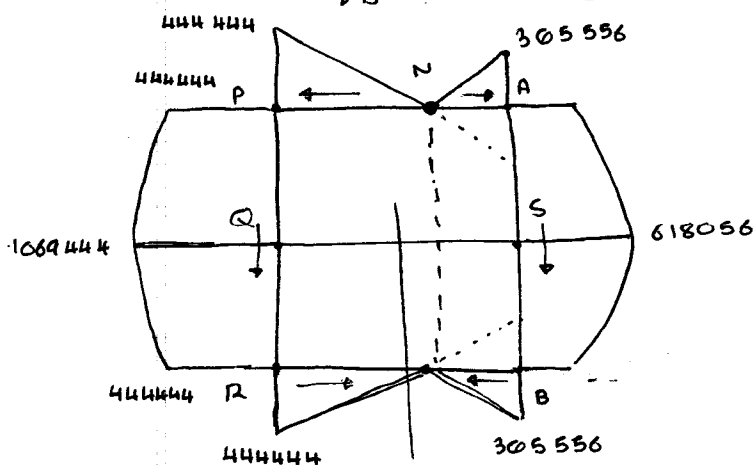
$$\rightarrow q_P = -444444 \text{ N/mm}$$

$$q_R = -444444 \text{ N/mm}$$

$$q_B = 305556 \text{ N/mm}$$

$$q_Q = -1069444 \text{ N/mm}$$

$$q_S = 618056 \text{ N/mm}$$



1° Symmetrical?

2° Edge (Parallel) Shear Force  
 $\Rightarrow$  quadratic

3° Edge (perpendicular) Shear Force  
 $\Rightarrow$  linear

Finding shear center (using Point A)

A: moment center

$$e = 203.0 \text{ mm}$$

Here  $I_x = 687500000 \text{ mm}^4$