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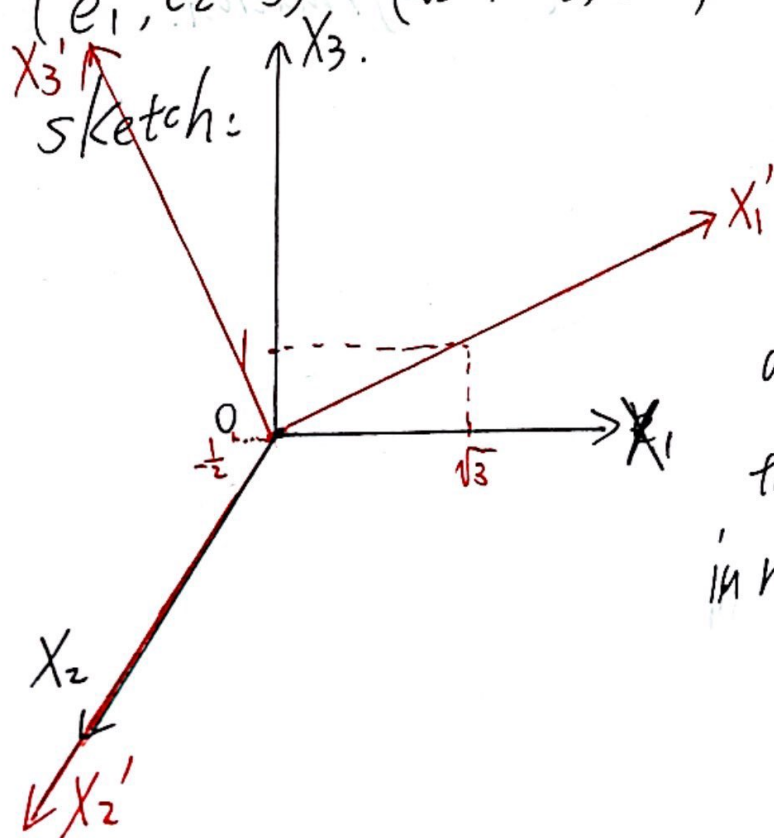
$$(a) \hat{e}_1' = T \cdot \hat{e}_1 = \begin{bmatrix} \sqrt{3} & 0 & -\frac{1}{2} \\ 0 & 1 & 0 \\ 1 & 0 & \frac{1}{2}\sqrt{3} \end{bmatrix} \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix} = \begin{bmatrix} \sqrt{3} \\ 0 \\ 1 \end{bmatrix}$$

$$\hat{e}_2' = T \cdot \hat{e}_2 = \begin{bmatrix} 0 \\ 1 \\ 0 \end{bmatrix} = \hat{e}_2$$

$$\hat{e}_3' = T \cdot \hat{e}_3 = \frac{1}{2} \begin{bmatrix} -1 \\ 0 \\ \sqrt{3} \end{bmatrix}$$

so we have the transformed coordinate system.

$$(\hat{e}_1', \hat{e}_2', \hat{e}_3') = (\sqrt{3}\hat{e}_1 + \hat{e}_3, \hat{e}_2, -\frac{1}{2}\hat{e}_1 + \frac{\sqrt{3}}{2}\hat{e}_3)$$



as the graph on the left,
the coordinate (X_1', X_2', X_3') plotted
in red is the modified coordinate.

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16) No, it's not.

if T is an orthonormal transformation,
its column vectors should all be normal.

but ~~apparent~~ obviously, let $T = [V_1, V_2, V_3]$.

$$V_1 = \begin{bmatrix} \sqrt{3} \\ 0 \\ 1 \end{bmatrix} \Rightarrow \|V_1\| = 2, \text{ not normal, that is to}$$

$$\text{say, for } \hat{e}_1' = T \cdot \hat{e}_1 = \begin{bmatrix} \sqrt{3} \\ 0 \\ 1 \end{bmatrix}, \|\hat{e}_1'\| = 2 \neq \|\hat{e}_1\|.$$

so some vector changes norm after transformation,

so T is not an orthonormal transformation.

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(2)

(a) let ξ_a be some point on the material filament at $t=0$.

so $\xi_a = (a, 0, 0)$, $a \in [0, 1]$.

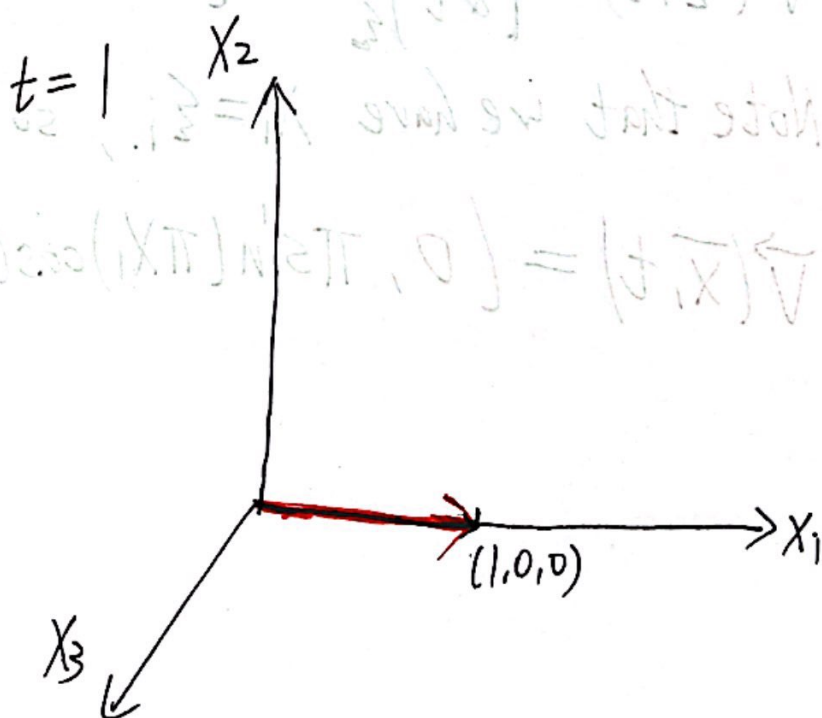
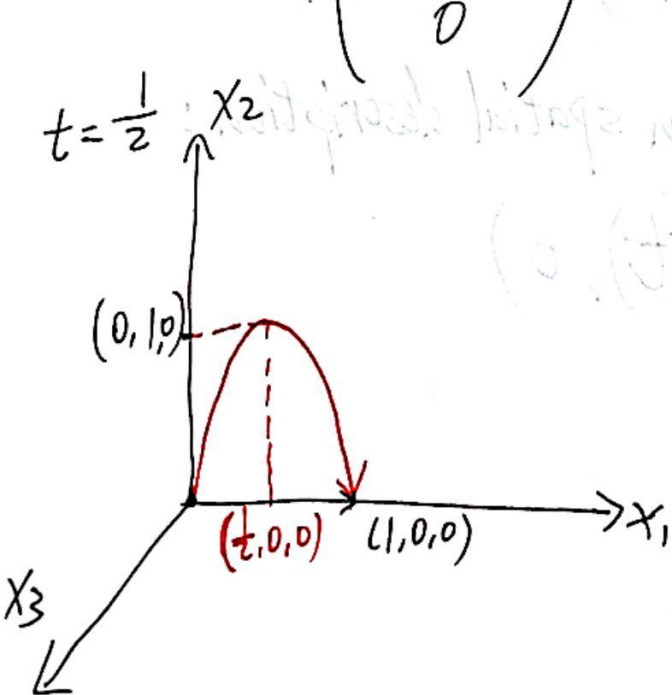
given.
$$X_a = \begin{pmatrix} \xi_{a1} \\ \xi_{a2} + \sin(\pi t)(\sin \pi \xi_{a1}) \\ \xi_{a3} \end{pmatrix}$$

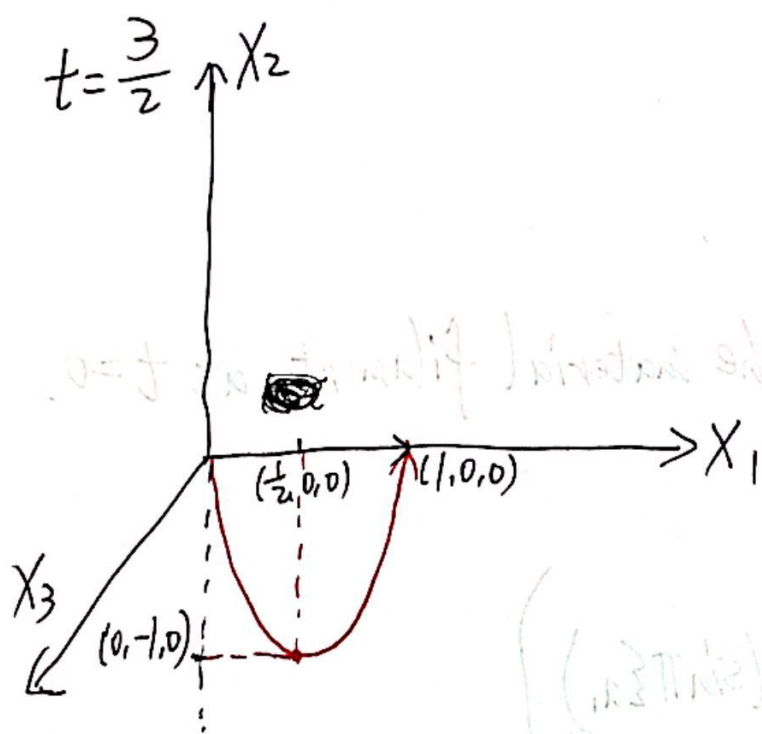
we have
$$X_a|_{t=\frac{1}{2}} = \begin{pmatrix} a \\ \sin(a\pi) \\ 0 \end{pmatrix}$$

$$X_a|_{t=1} = \begin{pmatrix} a \\ 0 \\ 0 \end{pmatrix}$$

$$X_a|_{t=\frac{3}{2}} = \begin{pmatrix} a \\ -\sin(a\pi) \\ 0 \end{pmatrix}$$

plots are following:
(red line is the deformed shape.)





To conclude, at $t = \frac{1}{2}$ and $t = \frac{3}{2}$, the filament deforms to be a trigonometric-like shape. And at $t = 1$, the deformed shape ~~can~~ coincides with the initial shape.

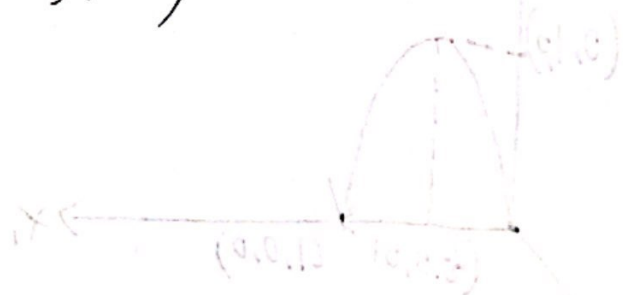
(b) For velocity \vec{V} ,

In material description, we have

$$\vec{V}(\bar{\xi}, t) = \left[\frac{\partial \bar{X}}{\partial t} \right]_{\bar{\xi}} = (0, \pi \sin(\pi \bar{\xi}_1) \cos(\pi t), 0).$$

Note that we have $X_1 = \bar{\xi}_1$, so in spatial description:

$$\vec{V}(\bar{X}, t) = (0, \pi \sin(\pi X_1) \cos(\pi t), 0)$$



For acceleration \vec{a} in material description:

$$\vec{a}(\vec{\xi}, t) = \left(\frac{\partial^2 \vec{X}}{\partial t^2} \right)_{\vec{\xi}} = (0, -\pi^2 \sin(\pi \xi_1) \sin(\pi t), 0)$$

and still we have $X_1 = \xi_1$, so in spatial description we have:

$$\vec{a}(\vec{X}, t) = (0, -\pi^2 \sin(\pi X_1) \sin(\pi t), 0)$$

(3)

(a) Displacement field.

$$u_1 = k(X_1 + X_3^2), u_2 = kX_1X_3, u_3 = k(X_1^2 - X_3)$$

$$\text{compute } \nabla u = \begin{bmatrix} \frac{\partial u_1}{\partial X_1} & \frac{\partial u_1}{\partial X_2} & \frac{\partial u_1}{\partial X_3} \\ \frac{\partial u_2}{\partial X_1} & \frac{\partial u_2}{\partial X_2} & \frac{\partial u_2}{\partial X_3} \\ \frac{\partial u_3}{\partial X_1} & \frac{\partial u_3}{\partial X_2} & \frac{\partial u_3}{\partial X_3} \end{bmatrix} = \begin{bmatrix} k & 0 & 2kX_3 \\ kX_3 & 0 & kX_1 \\ 2kX_1 & 0 & -k \end{bmatrix}$$

and the stress is given by

$$E = \frac{1}{2}(\nabla u + \nabla u^T) = \begin{bmatrix} k & \frac{k}{2}X_3 & k(X_1 + X_3) \\ \frac{k}{2}X_3 & 0 & \frac{k}{2}X_1 \\ k(X_1 + X_3) & \frac{k}{2}X_1 & -k \end{bmatrix}$$

The components of E are the stress components.

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comments on the style of deformation:

write $\nabla u = \frac{1}{2}(\nabla u + \nabla u^T) + \frac{1}{2}(\nabla u - \nabla u^T)$

$$E = \frac{1}{2}(\nabla u + \nabla u^T), W = \frac{1}{2}(\nabla u - \nabla u^T).$$

E represents the internal deformation, also known as strain.

W represents the rigid body rotation.

to further comment on E , suppose we have our original vector $d\bar{\xi}_1 = (d\xi_1, 0, 0)$, $d\bar{\xi}_2 = (0, d\xi_2, 0)$ at $\bar{X} = (X_1, X_2, X_3)$.
the change of length of $d\bar{\xi}_1$ is $d\bar{\xi}_1' - d\xi_1 \approx (1 + E_{11} - 1) d\xi_1$.

so $\frac{D(d\bar{\xi}_1)}{Dt} = E_{11} = K$, similarly, change in length of $d\bar{\xi}_2$ is given by $E_{22} = 0$

the change in angle between $d\bar{\xi}_1', d\bar{\xi}_2'$ is given by

$$2E_{12} = KX_3$$

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(b) Recall the equation of motion:

$$\vec{F}_{\text{body}} + \vec{F}_{\text{stress}} = m\vec{a}.$$

In this question, $\left\{ \begin{array}{l} \text{equilibrium} \Rightarrow \vec{a} = 0 \\ \text{in the absence of body forces} \Rightarrow \vec{F}_{\text{body}} = 0. \end{array} \right.$

Thus we have:

$$\vec{F}_{\text{stress}} = \nabla \cdot \vec{\sigma} = 0.$$

$$\text{so } \frac{\partial \sigma_{ji}}{\partial x_j} = 0.$$

In x_1 direction:

$$\frac{\partial \sigma_{11}}{\partial x_1} + \frac{\partial \sigma_{21}}{\partial x_2} + \frac{\partial \sigma_{31}}{\partial x_3} = 0 + 0 + 0 = 0 \text{ (Already satisfied.)}$$

~~26~~ In x_2 direction:

$$\frac{\partial \sigma_{12}}{\partial x_1} + \frac{\partial \sigma_{22}}{\partial x_2} + \frac{\partial \sigma_{32}}{\partial x_3} = 0 + 0 + 0 = 0 \text{ (Already satisfied.)}$$

In x_3 direction:

$$\frac{\partial \sigma_{13}}{\partial x_1} + \frac{\partial \sigma_{23}}{\partial x_2} + \frac{\partial \sigma_{33}}{\partial x_3} = 1 + C - B = 0$$

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write σ as

$$\sigma = \begin{pmatrix} AX_3 & CX_3 & X_1+X_2 \\ CX_3 & X_1+X_3 & CX_2 \\ X_1+X_2 & CX_2 & CX_1-BX_3 \end{pmatrix}$$

the plane $X_1+X_3=0$ is stress free,

let $S = (a, b, c)$ on plane $X_1+X_3=0$, also $\vec{0} = (0, 0, 0)$ is on the plane,

we have $a+c=0 \Rightarrow c=-a$.

let \hat{n} be unit normal of $X_1+X_3=0$, suppose $\hat{n} = (n_1, n_2, n_3)$

we have $\|\hat{n}\|=1$

$$\hat{n} \cdot S = n_1 a + n_2 b + n_3 (-a) = a(n_1 - n_3) + b n_2 = 0, \quad \forall a, b \in \mathbb{R}.$$

so we have $\hat{n} = \frac{\sqrt{2}}{2} (1, 0, 1)$

stress on such plane $\vec{T} = \sigma \cdot \hat{n}$,

$$\text{so } \vec{T} = \frac{\sqrt{2}}{2} (AX_3 + X_1 + X_2, C(X_2 + X_3), CX_1 - BX_3 + X_1 + X_2) = \vec{0}.$$

also, we have $X_1+X_3=0$ on the plane.

$$\text{so we have } \begin{cases} A-1=0 \\ C=0 \\ C+1-B=0 \end{cases} \Rightarrow \begin{cases} A=1 \\ B=1 \\ C=0 \end{cases}$$