

Page-1

Q1.

(i) There are 7 variables ( $T_0, \mu_0, \rho, g, V, M, r$  and  $g$ ) and 3 dimensions (M, L, T) and so 4 dimensionless groups are required.

(ii). According to Buckingham Pi Theory, write as

$$\begin{matrix} T_0 & \mu_0 & \rho & g & V & M & r \\ M & \begin{pmatrix} 1 & 1 & 1 & 0 & 0 & 1 & 0 \\ -1 & -1 & -3 & 1 & -1 & 0 & 1 \\ -2 & -1 & 0 & 2 & -1 & 0 & 0 \end{pmatrix} \\ L & \\ T & \end{matrix}$$

so

$$A = \begin{pmatrix} 1 & 1 & 1 & 0 & 0 & 1 & 0 \\ -1 & -1 & -3 & 1 & -1 & 0 & 1 \\ -2 & -1 & 0 & 2 & -1 & 0 & 0 \end{pmatrix} \text{ let } \bar{X} = (X_{T_0} X_{\mu_0} X_{\rho} X_g X_V X_M X_r)$$

$$A_1 = \begin{pmatrix} 1 & 1 & 1 \\ -1 & -1 & -3 \\ -2 & -1 & 0 \end{pmatrix} \quad A_2 = \begin{pmatrix} 0 & 0 & 1 \\ 1 & 1 & 0 \\ -2 & -1 & 0 \end{pmatrix} \quad \begin{matrix} \bar{X}_1 = (X_{T_0} X_{\mu_0} X_{\rho}) \\ \bar{X}_2 = (X_g X_V X_M X_r) \end{matrix}$$

then the linear equation  $A\bar{X} = 0$  is equivalent to.

$$A_1 \bar{X}_1 + A_2 \bar{X}_2 = 0 \Rightarrow A_1 \bar{X}_1 = -A_2 \bar{X}_2 \Rightarrow \bar{X}_1 = -A_1^{-1} A_2 \bar{X}_2$$

compute  $A_1^{-1}$  by python code, we have  $A_1^{-1} = \begin{pmatrix} -\frac{3}{4} & \frac{1}{4} & -1 \\ \frac{3}{2} & -\frac{1}{2} & 1 \\ \frac{1}{4} & \frac{1}{4} & 0 \end{pmatrix}$

so take  $\bar{X}_2 = \begin{pmatrix} 1 \\ 0 \\ 0 \end{pmatrix}$  or  $\begin{pmatrix} 0 \\ 1 \\ 0 \end{pmatrix}$  or  $\begin{pmatrix} 0 \\ 0 \\ 1 \end{pmatrix}$  or  $\begin{pmatrix} 0 \\ 0 \\ 0 \end{pmatrix}$ .

Page. 2

we solve  $\bar{X}$  to be

$$\bar{X} = \begin{pmatrix} -\frac{5}{2} \\ -\frac{1}{2} \\ -\frac{1}{2} \\ 0 \\ 0 \\ 0 \\ 0 \end{pmatrix} \text{ or } \begin{pmatrix} -\frac{5}{2} \\ -\frac{1}{2} \\ -\frac{1}{2} \\ 0 \\ 0 \\ 0 \\ 0 \end{pmatrix} \text{ or } \begin{pmatrix} -\frac{5}{2} \\ -\frac{1}{2} \\ -\frac{1}{2} \\ 0 \\ 0 \\ 0 \\ 0 \end{pmatrix} \text{ or } \begin{pmatrix} -\frac{1}{2} \\ -\frac{1}{2} \\ -\frac{1}{2} \\ 0 \\ 0 \\ 0 \\ 1 \end{pmatrix}$$

thus we have the following dimensionless groups:

$$N_1 = \frac{\mu \omega^{\frac{5}{2}} g}{\tau_0^{\frac{5}{2}} p^{\frac{1}{2}}} \quad N_2 = \frac{\mu \omega^{\frac{3}{2}} V}{\tau_0^{\frac{5}{2}} p^{\frac{1}{2}}} \quad N_3 = \frac{\tau_0^{\frac{3}{2}} M}{\mu \omega^{\frac{3}{2}} p^{\frac{1}{2}}}$$

$$N_4 = \frac{\mu \omega^{\frac{1}{2}} r}{\tau_0^{\frac{1}{2}} p^{\frac{1}{2}}}$$

(iii) take new  $N_2 = \frac{N_2}{\sqrt{N_1 N_4}} = \frac{\frac{\mu \omega^{\frac{3}{2}} V}{\tau_0^{\frac{5}{2}} p^{\frac{1}{2}}}}{\sqrt{\frac{\mu \omega^{\frac{5}{2}} g}{\tau_0^{\frac{5}{2}} p^{\frac{1}{2}}} \cdot \frac{\mu \omega^{\frac{1}{2}} r}{\tau_0^{\frac{1}{2}} p^{\frac{1}{2}}}}}$

$= \frac{V}{\sqrt{g r}} \sim$  the Froude Number, represents the ratio of inertia to gravity, and  $N_3 = \frac{N_3}{N_4} = \frac{\tau_0 M}{\mu^2}$

$$\text{new } N_1 = N_1 N_3^3 = \frac{g M^3}{\mu \omega^2 p}, \quad \text{new } N_4 = N_3^3 N_4^3 = \frac{M \cdot r^3}{p}$$



Page. 3

Thus our new dimensionless group

should be:

$$N_1 = \frac{gM^3}{\mu \omega^2 \rho}$$

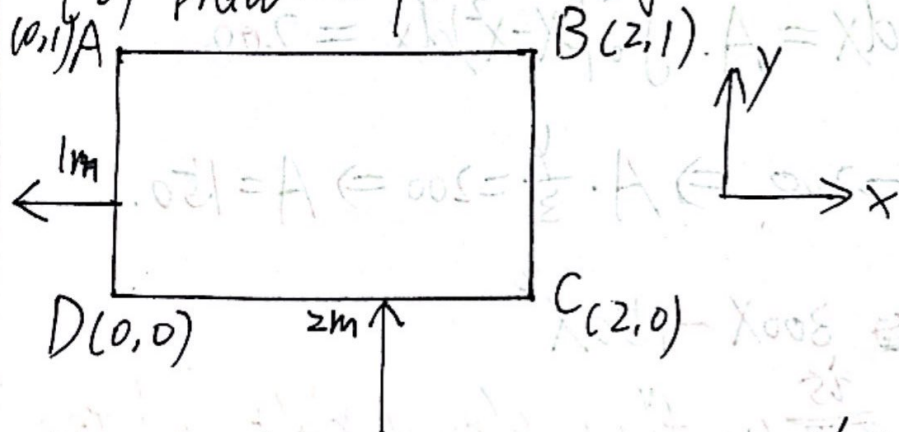
$$N_2 = \frac{v}{\sqrt{gr}} \sim \text{the Froude Number.}$$

$$N_3 = \frac{T_0 M}{\mu \omega^2}$$

$$N_4 = \frac{M \cdot r^3}{\rho}$$

Question 2

(i) Draw the picture of this rock:



From the question, we notice that there is only inlet heat flow at the bottom, and the only outlet is the left edge. No heat flows through the right and top sides.

Page 4.

And also compute the total heat flow in let

$$Q_{in} = 100 \text{ W/m} \cdot 2\text{m} = 200 \text{ W} = Q_{out}.$$

the edges are being evenly cooled, thus at the left edge we have:  $F_x = -\frac{200}{1} = -200$  (Note the out let flow direction is negative  $-x$  direction.).

$$\text{So } F_{AD} = -200.$$

At the bottom, assuming  $F_{CD}(x) = A \cdot (2-x) \cdot x$ ,  $A$  some constant.

we have  $F_{CD}(2) = F_{CD}(0) = 0$  and  $F_{CD}(1) = \max_{x \in [0,2]} F_{CD}(x)$ .

$$\text{Note that } Q_{in} = \int_0^2 F_{CD}(x) dx = A \cdot \int_0^2 (2x - x^2) dx = 200.$$

$$\text{we have } A \cdot \left[ x^2 - \frac{x^3}{3} \right]_{x=0}^{x=2} = 200 \Rightarrow A \cdot \frac{4}{3} = 200 \Rightarrow A = 150.$$

$$\text{thus we have } F_{CD}(x) = \cancel{150} 300x - 150x^2$$

As mentioned before, there ~~are~~<sup>is</sup> no flow through right and top sides, thus naturally,  $F_{AB} = F_{BC} = 0$ .

To conclude,

we have

$$\left\{ \begin{array}{l} F_{left} = -200 \\ F_{top} = 0 \\ F_{right} = 0 \\ F_{bottom} = 300x - 150x^2 \end{array} \right.$$



Page. 5

(ii) For the stream function  $\psi$ , we always have:

$$F_x = \frac{\partial \psi}{\partial y}, \quad F_y = -\frac{\partial \psi}{\partial x}, \quad \text{suppose } \psi(c) = 0.$$

at the bottom we have:

$$F_{DC}(x) = 300x - 150x^2 = -\frac{\partial \psi_{CD}}{\partial x} = -\frac{d\psi_{CD}}{dx}$$

$$\text{so } \psi_{CD}(x) = \int 150x^2 - 300x \, dx = 150\left(\frac{x^3}{3} - x^2\right) + C.$$

$$\psi(c) = 0 \Rightarrow \psi_{CD}(2) = 0 \Rightarrow C = 200.$$

thus we have  $\psi(D) = \psi_{CD}(0) = C = 200.$

at the left side we have:

$$F_{AD} = -200 = \frac{\partial \psi_{AD}}{\partial y} = \frac{d\psi_{AD}}{dy}, \quad \text{so } \psi_{AD}(y) = -200y + D.$$

$$\psi(D) = 200 \Rightarrow \psi_{AD}(0) = 200 \Rightarrow D = 200.$$

$$\text{also indicates } \psi(A) = \psi_{AD}(1) = -200 \cdot 1 + 200 = 0.$$

Note no flow through AB, BC, Thus  $F_{AB} = F_{BC} = 0.$

$$\psi_{AB} = a_1, \quad \psi_{BC} = a_2, \quad (a_1, a_2 \text{ constants}). \quad \text{and}$$

$$\psi(A) = 0 \Rightarrow a_1 = 0$$

$$\psi(C) = 0 \Rightarrow a_2 = 0.$$

Page. 6

To conclude, we have the stream functions around the boundaries:

$$\begin{cases} \psi_{\text{left}} = -200y + 200 \\ \psi_{\text{top}} = 0 \\ \psi_{\text{right}} = 0 \\ \psi_{\text{bottom}} = 50x^3 - 150x^2 + 200 \end{cases}$$

Q3.

(i) the flow is steady, so  $\frac{D\bar{U}}{Dt} = 0$ .

body forces be ignored, so the term  $\rho \bar{g}$  can be thrown away.

Let's write the Navier-Stokes equation in the x-direction.

We have

$$0 = \cancel{\frac{D\bar{U}}{Dt}} = \frac{dP}{dx}$$

$$0 = -\frac{dP}{dx} + \cancel{\frac{D\bar{U}}{Dt}} + \frac{d(T_{yx})}{dx}$$

No pressure gradient in the direction of flow implies  $\frac{dP}{dx} = 0$ .

thus we have  ~~$T_{yx} = 0$~~ .  $T_{yx} = \cancel{0} = A$ , (A some constant.)

~~Also, note at the top we measured  $T_{\text{top}}$ .~~

~~thus we have  $T_{yx} = T_{\text{top}}$ .~~

Page. 7

(ii) Note at the top we have measured  $T_{top}$ .

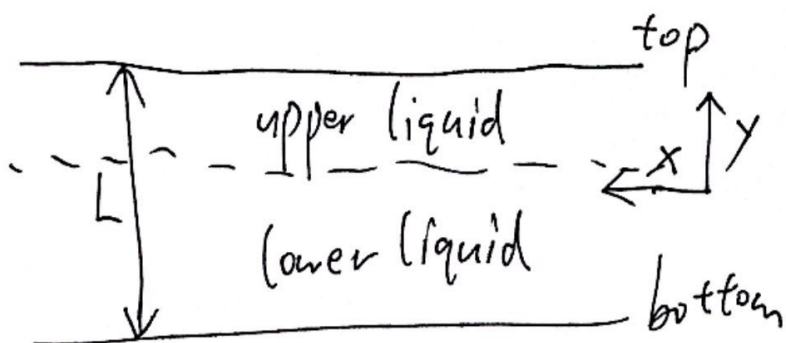
So we have  $T_{yx} = T_{top}$ .



Page. 8

(iii)

Draw the picture:



let  $y$  be the distance from the bottom, we have:

for the upper liquid:

$$\tau_{yx} = \mu_{\text{upper}} \frac{dV_x}{dy} = \tau_{\text{top}}$$

$$V_x = \frac{\tau_{\text{top}}}{\mu_{\text{upper}}} y + B$$

Given  $y=L$ ,  $V_x = V_{\text{top}}$ .

we can derive  $B = -\frac{\tau_{\text{top}}}{\mu_{\text{upper}}} L + V_{\text{top}}$

thus  $V_x = \frac{\tau_{\text{top}}}{\mu_{\text{upper}}} (y-L) + V_{\text{top}}$ ,  $y > 0$ .

for the lower liquid:

$$\tau_{yx} = \mu_{\text{lower}} \frac{dV_x}{dy} = \tau_{\text{top}}$$

thus  $V_x = \frac{\tau_{\text{top}}}{\mu_{\text{lower}}} y + C$  with  $V_x = 0$  at  $y=0$ .



Page. 9

Thus,  $C = 0$ .

So we have  $V_x = \frac{T_{top}}{\mu_{lower}} y$  for  $y < \phi L$