

# HOLT CONSULTING ENGINEERS (PTY) LTD

## TUMELA 18 TON SKIP DESIGN

### H-MAC603

### H-MAX603-CAL-MM-18SKIP-001-SHT-001

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**REVISION HISTORY**

REV	DESCRIPTION	DATE	ISSUED BY	REVIEWED BY	APPROVED
A	ISSUED FOR REVIEW	2022-10-15	G.G. HOLT	H.F. HOLT	

## 1 CUSTOMER DETAILS

<b>Customer:</b>	Max Power Services (Pty) Ltd
<b>Customer Name:</b>	Herman de Koker
<b>Customer Email:</b>	harry@maxpower.co.za

## 2 CALCULATION INPUT DATA

### 2.1 Applicable Design Codes

SANS 10208: 3 - 2017: Design of structures for the mining industry Part 3: Conveyances

SANS 10610: Building loading code

SANS 10162: Steel design

### 2.2 General Data

<b>Design Method</b>	Limit States (Rope Break Conditions)	
<b>Material of Construction</b>	Main Body: EN10025 S355JR	
<b>Material of Construction</b>	Liners: VRN 500	
<b>Yield Stress</b>	355	<i>MPa</i>
<b>Skip Weight</b>	9878	<i>kg</i>
<b>Payload</b>	18000	<i>kg</i>
<b>Winding Speed</b>	15	<i>m/s</i>
<b>Winding Rope Diameter</b>	54	<i>mm</i>
<b>Winding Rope Unit Mass</b>	12.45	<i>kg/m</i>
<b>Rope Break Force</b>	2319	<i>kN</i>
<b>Ultimate Tensile Strength</b>	1900	<i>MPa</i>
<b>Winder Acceleration</b>	0.8	<i>m/s<sup>2</sup></i>
<b>Winder Trip Acceleration</b>	5	<i>m/s<sup>2</sup></i>
<b>Winder Travel Distance</b>	1023	<i>m</i>
<b>Number of Cycles per Month</b>	3000	
<b>Skip Internal Height</b>	5600	<i>mm</i>
<b>Skip Internal Width</b>	1557	<i>mm</i>
<b>Skip Internal Depth</b>	1400	<i>mm</i>
<b>Skip Overall Height</b>	10713	<i>mm</i>
<b>Skip Overall Width</b>	1856	<i>mm</i>
<b>Skip Overall Depth</b>	1743	<i>mm</i>
<b>Ore Bulk Density</b>	1950	<i>kg/m<sup>3</sup></i>

### 2.3 Assumption Data

Spacing between rails	1800	<i>mm</i>
Top Hat Guide Specification	340 x 175mm	
Top Hat Guide Material Specification	EN10025 S355JR	
Top Hat Guide Unit Mass	85.95	<i>kg/m</i>
Top Hat Guide Width	175	<i>mm</i>
Bunton Stiffness	1608000	<i>N/m</i>
Guide Stiffnes	1600000	<i>N/m</i>

2.4 Sketches and Drawings

2.4.1 General Arrangement

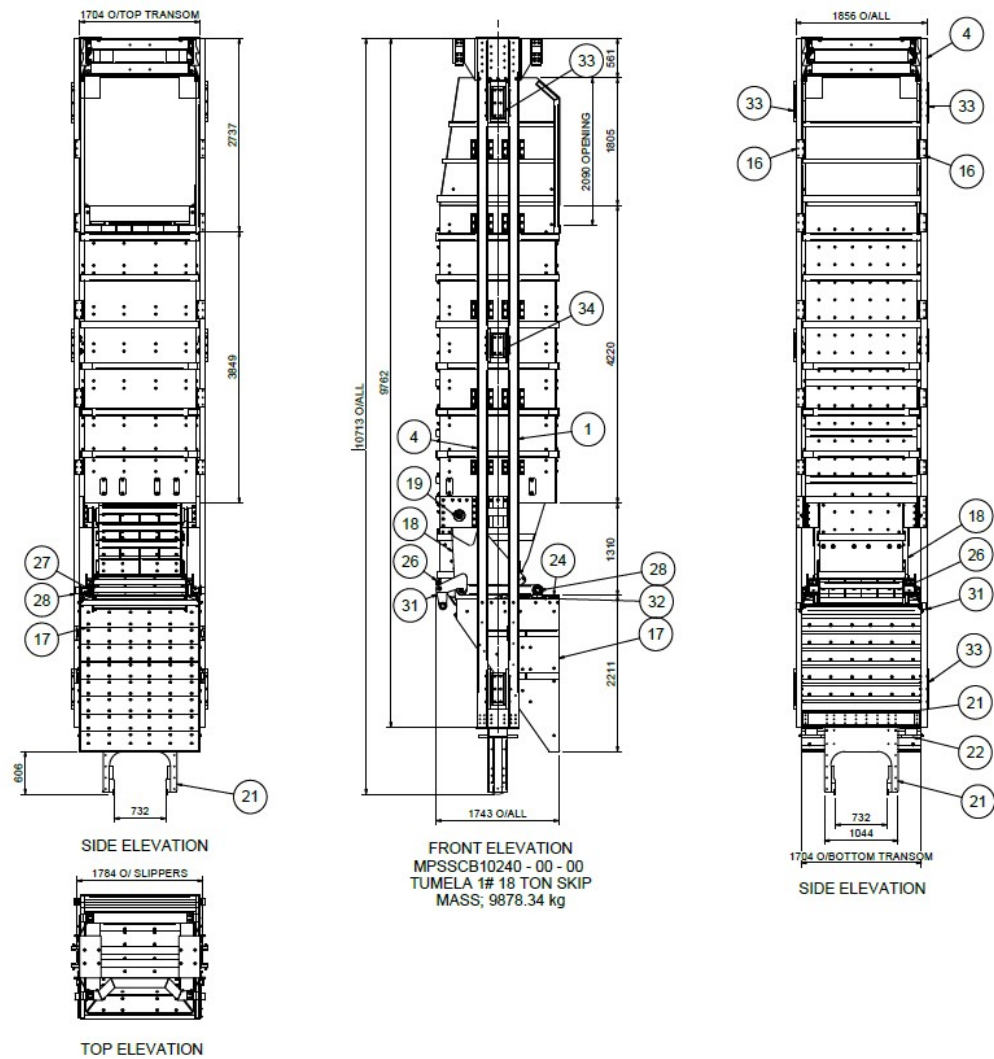
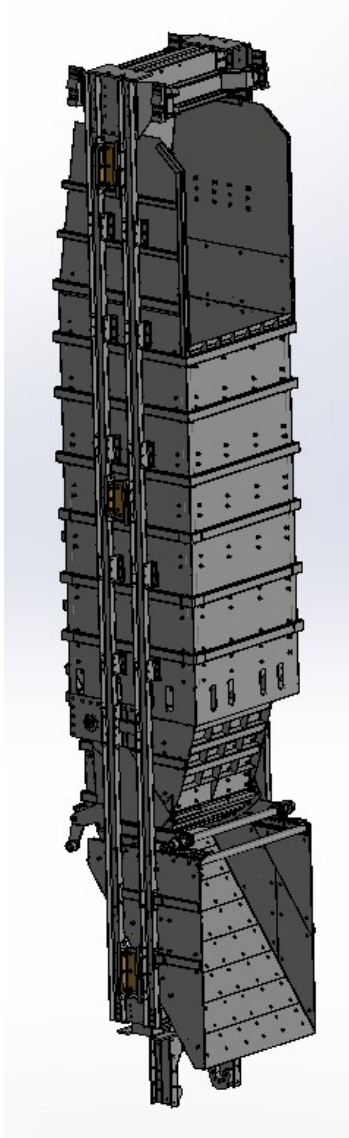
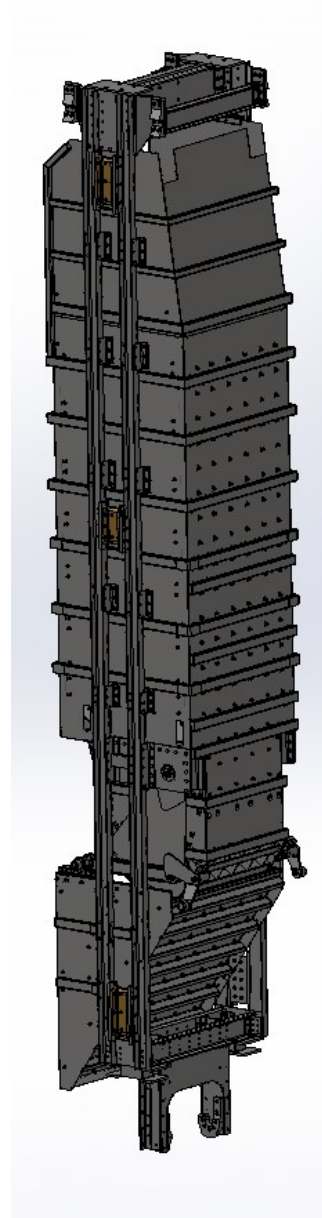


Figure 1: 18 ton Skip Drawing

## 2.4.2 Isometric Views



(a) 18 ton Skip Isometric View 1



(b) 18 ton Skip Isometric View 3

### 3 CALCULATIONS

#### 3.1 Skip Loads

##### 3.1.1 Basic Skip Parameters

Payload	$R$	18000	$kg$
Payload	$R_l$	176580.0	$N$
Winding Speed	$V_w$	15	$m/s$
Winder Acceleration	$a_w$	0.8	$m/s^2$
Winder Trip Acceleration	$a_t$	5	$m/s^2$
Winding Rope Diameter	$Rope_d$	54	$m/s^2$
Rope Break Force	$RBF$	2319	$kN$
Ultimate Tensile Strength	$UTS$	1900	$MPa$
['Ore Bulk Density', 1950, 'kg/m <sup>3</sup> ']	$\rho_b$	1950	$kg/m^3$
Skip Internal Width	$b$	1557	$mm$
Skip Internal Depth	$w$	1557	$mm$
Skip Volume Required	$Vol = R/\rho_b$	9.2	$m^3$
Ore Height in Skip	$h = Vol/(bw)$	4.2	$m$
Rope Stretch Under Payload	$\Delta L$	765.8	$mm$

##### 3.1.2 Permanent Loads

Skip Bridle Sides	$m_1$	1167	$kg$
Skip Bridle Top Transom	$m_2$	1522	$kg$
Skip Bridle Bottom Transom	$m_3$	850	$kg$
Skip Unit	$m_4$	6336	$kg$
Permanent Load	$G_c = (m_1 + m_3 + m_4)g$	81943	$N$



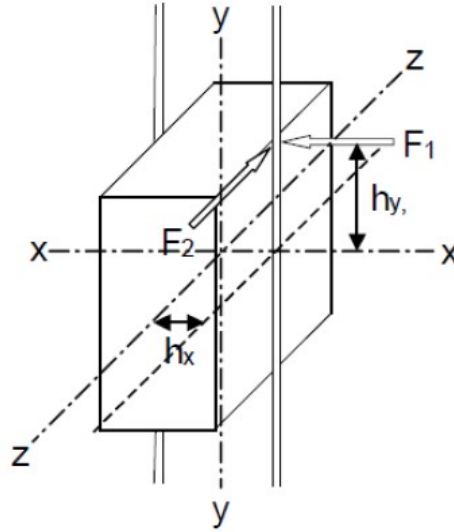


Figure 3: Properties Diagram

### 3.1.3 Lateral Imposed Loads (H) - Fixed Guide Systems in Vertical Shafts

Clearance between Roller and Slipper	$\Delta_c$	10	mm
Slipper Plate Impact Factor	$\alpha_n$	2	
Guide Roller Assembly Stiffness	$k_r$	500000	N/m
Bunton Stiffness	$k_b$	1608000	N/m
Guide Stiffness	$k_g$	1600000	N/m
Moment of Inertia about X-axis	$I_x$	80510	kg.m <sup>2</sup>
Moment of Inertia about Y-axis	$I_y$	6838	kg.m <sup>2</sup>
Moment of Inertia about Z-axis	$I_z$	82050	kg.m <sup>2</sup>
Distance from slipper to center of gravity	$h_x$	892	mm
Distance from slipper to center of gravity	$h_y$	4847	mm
Distance from slipper to center of gravity	$h_z$	28	mm
Guide Roller Lateral Load	$H_f$	5000000	N
Steelwork Stiffness Ratio	$r_k$	1.005	
Weight of Skip System	$m_c$	8353	kg
Effective Mass About y - x Plane	$m_x = (m_c I_z) / (I_z + m_c (h_y)^2)$	2463.0	kg
Effective Mass About y - z Plane	$m_z = (m_c I_x I_y) / (I_x I_y + (m_c I_x (h_y)^2) + (m_c I_y (h_x)^2)$	280.0	kg
Non-Dimensional Lateral Stiffness	$K_x = (k_b L_b^2) / m_x V^2$	9	
Non-Dimensional Lateral Stiffness	$K_z = (k_b L_b^2) / m_z V^2$	83	
Plate Coefficient from graph	$P_b$	0.05	
Maximum Moving Misalignment	$e$	0.01	m
Lateral Slipper Pad Load	$H_s$	7791	N

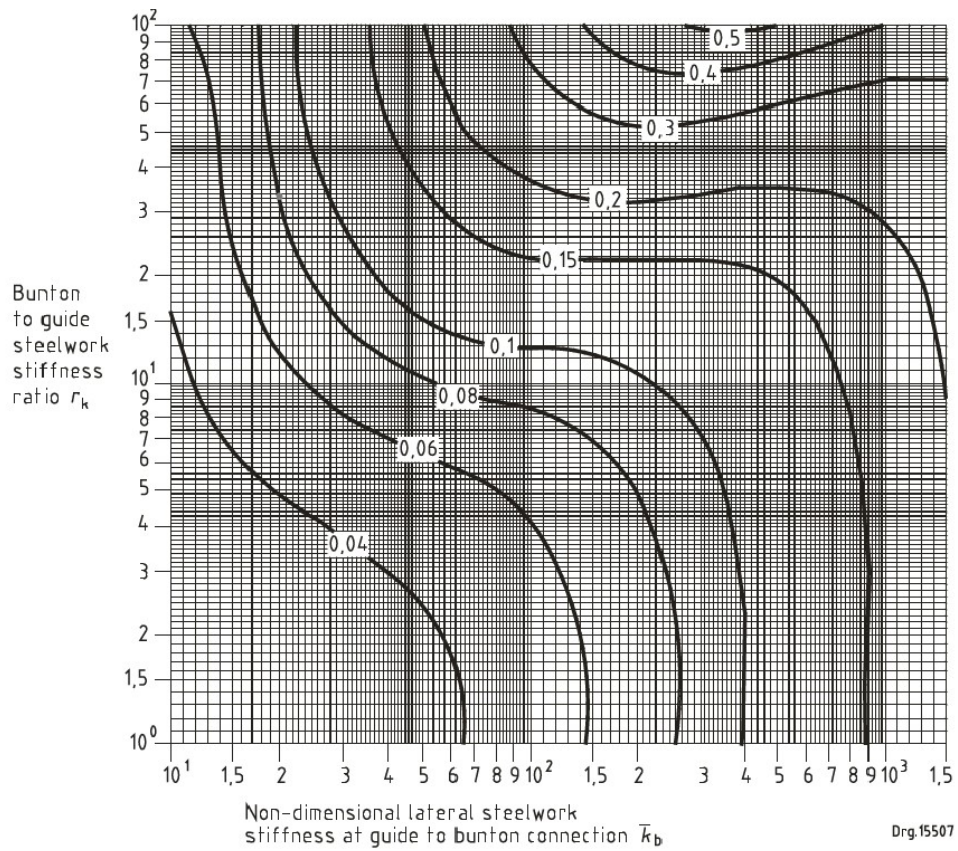


Figure 1 — Contour plot of slipper plate load coefficient  $\overline{P_b}$

Figure 4: Slipper Plate Load Coefficient  $P_b$

### 3.1.4 Winder System Loads

Dynamic Impact Factor	$\alpha_d$	2	
Winder Acceleration and Deceleration	$a_o$	0.8	$m/s^2$
Winder Trip Acceleration	$a_t$	0.8	$m/s^2$
Skip Self Weight	$G_c$	81943	$N$
Content Load	$C_y$	176580.0	$N$
Tail Rope Load	$T$	0	$N$
Acceleration Load	$A_o = (\alpha_d)a_o(G_c + C_y + T)/g$	42165	$N$
Acceleration Trip Out Load	$A_t = (\alpha_d)a_t(G_c + C_y + T)/g$	42165	$N$

### 3.1.5 Emergency Loads

Emergency Load  $E_r$  2319000  $N$

### 3.1.6 Vertical Friction Loads

Lateral Slipper Pad Load	$H_s$	7791	$N$
Vertical Friction Load	$F_v = 0.5H_s$	3895.5	$N$

### 3.1.7 Rock Loads

Filling Impact Factor in Stationary Position	$\alpha_v$	1.5	
Load on Tipping Rollers Impact Factor	$\alpha_t$	2	
Static Load	$R$	176580.0	$N$
Bridle Transom Load While Filling	$R_d = (\alpha_v)(R)$	27000.0	$N$
Rock Pressure	$p_o = \rho_b g h$	80.3439	$N/m^2$
Pressure on the Door	$p_1 = 1p_o$	80.3	$N/m^2$
Pressure on the Back of Skip	$p_2 = 0.5p_o$	40.2	$N/m^2$
Pressure on the Lower Portion Skip Back	$p_3 = 0.5p_o$	120.5	$N/m^2$
Pressure on the Front and Sides of Skip	$p_4 = 0.2p_o$	16.1	$N/m^2$

## 3.2 Skip Element Design

### 3.2.1 Top Transom

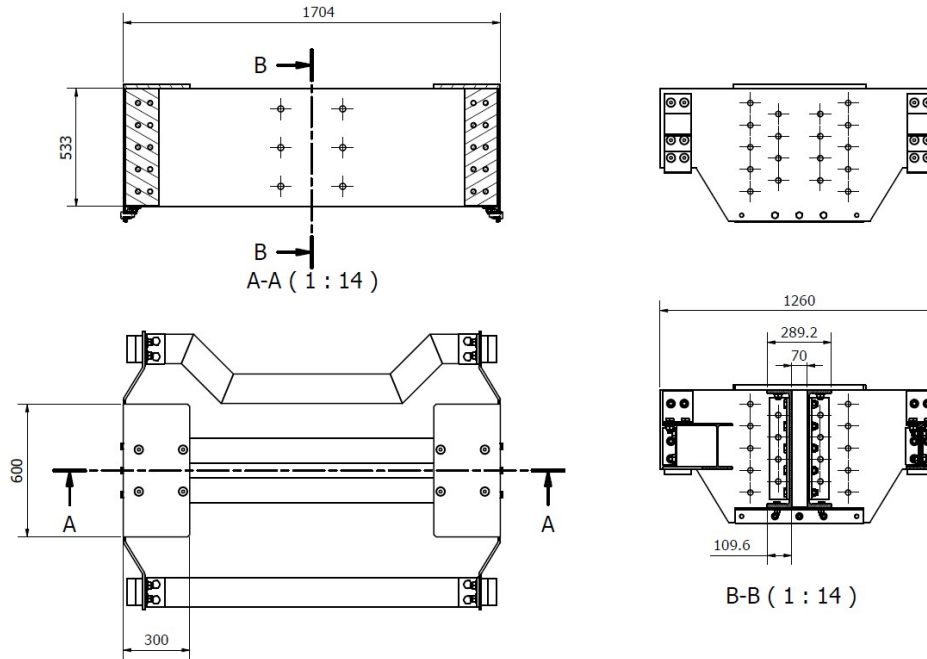


Figure 5: Top Transom Configuration

**Material Data**

2 / 533 x 110 CHANNEL (MODIFIED 533 X 210 X 93 UB) WITH TOP AND BOTTOM 20 X 100 PLATE

4 / 20 X 100 PLATES (TOP AND BOTTOM)

2 / 150 x 150 ANGLE END CONNECTIONS

Length of Transom	$L$	1700	mm
Channel Width	$b$	109	mm
Channel Height	$h$	533	mm
Channel Web Thickness	$t_w$	10.2	mm
Channel Flange Thickness	$t_f$	15.6	mm
Top and Bottom Plate Width	$w$	110	mm
Top and Bottom Plate Thickness	$t_p$	20	mm
Yield Stress	$f_y$	355	MPa
Channel Cross Sectional Area	$A = 2bt_f + h_w * t_w$	8519.0	mm <sup>2</sup>
Channel Centroid Distance	$x_c = (0.5h_w t_w^2 + t_f b^2) / A$	25.0	mm
Channel Second Moment of Area about x-x	$I_x = bh^3 / 12 - b_f h_w^3 / 12$	335071491.0	mm <sup>4</sup>
Channel Second Moment of Area about y-y	$I_y = h_w t_w^3 / 3 + 2t_f b^3 / 3 - Ax_c^2$	8397644.0	mm <sup>4</sup>
Double Channel Polar Moment	$J = I_x + I_y$	340973234.0	mm <sup>4</sup>
Channel Warping Constant	$C_w = (1/144)(t_f^3 b^3) + (1/36)(h - t_f/2)^3 t_w^3$	4304577645.0	mm <sup>6</sup>
Channel Plastic Section Modulus	$Z_p = bh^2 / 4 - b_f h_w^2 / 4$	1521885.0	mm <sup>3</sup>
Double Plate Cross Sectional Area	$A = 2(wt_p)$	4400	mm <sup>2</sup>
Double Plate Second Moment of Area about x-x	$I_x = w/12(H^3 - h^3)$	336536567.0	mm <sup>4</sup>
Double Channel Second Moment of Area about y-y	$I_y = t_p w^3 / 6$	4436667.0	mm <sup>4</sup>
Double Plate Plastic Section Modulus	$Z_p = 2(wt_p(h - t_p/2))$	2301200.0	mm <sup>3</sup>
Plastic Moment	$M_p = Z_p f_y$	1357.2	kNm
Adjusted Plastic Moment	$0.67M_p$	909.3	kNm
Critical Elastic Moment	$M_c = ((\pi^4 E^2 C_w I_y) / L^4 + (\pi^2 E I_y G J) / L^2)^{0.5}$	20494.3	kNm
Factored Moment Resistance	$M_r = 1.15\phi M_p (1 - (0.28M_p / M_c))$	1378.7	kNm
Factored Shear Resistance	$V_r = 0.55\phi A f_y$	2270.0	kNm

**Design for Emergency Loads**

Rope Break Force	$RBF$	2319	kN
Number of Beams	$No.$	2	
Combined Shear Resistance	$M_r$	4540.0	kN
Ultimate Bending Moment	$M_u = RBF L / 4$	986.0	kN
Ultimate Shear Force	$M_u = RBF / 2$	1160.0	kN
Interaction Check	$M_u / M_r + V_u / V_r < 1$	0.61	Pass

**Design for Operation Loads**

Operation Force	$LC = 1.2G_c + 1.6R$	127.0	$kN$
Number of Beams	$No.$	2	
Combined Shear Resistance	$M_r$	4540.0	$kN$
Ultimate Bending Moment	$M_u = RBF L/4$	54.0	$kN$
Ultimate Shear Force	$M_u = RBF/2$	64.0	$kN$
Interaction Check	$M_u/M_r + V_u/V_r < 1$	0.03	$Pass$

**4 SUMMARY**

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