Third year

Module 3F2: Systems and Control

LECTURE NOTES 2: 'CLASSICAL' METHODS

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1 Revision of Feedback Control

1.1 Why use Feedback?

• To reduce effects of uncertainty:

- Disturbances

Wind/waves, Friction, Impurities, . . .

- Model errors

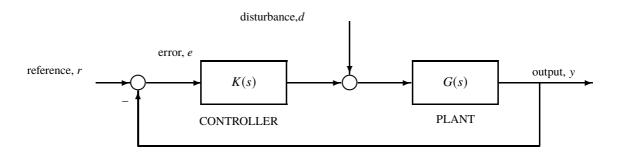
Approximations, Tolerances, Ageing, . . .

- To stabilise unstable system:
 - Inverted pendulum,
 - High-performance fighter aircraft (Fly-by-wire)
 - Helicopter, Submarine (depth)
 - Exothermic chemical reactor, Nuclear reactor

Problems with feedback:

- May destabilise system
- Sensors introduce noise

1.2 The Standard Feedback Loop



1.3 Sensitivity and Complementary Sensitivity

Let L(s) be the (open) loop transfer function: L(s) = G(s)K(s)

"Return-ratio"

Complementary Sensitivity: $T(s) = \frac{L(s)}{1+L(s)}$

(Multivariable: $T(s) = L(s)[I + L(s)]^{-1}$

$$\bar{y} = T\bar{r} = -T\bar{n}$$
 (\bar{n} = sensor noise)

 $T \approx 1 \Rightarrow \text{good "tracking" but no noise filtering.}$

Sensitivity: $S(s) = \frac{1}{1 + L(s)}$

(Multivariable: $S(s) = [I + L(s)]^{-1}$)

$$\bar{e} = S\bar{r}, \quad S = \frac{dT/T}{dG/G}, \qquad \qquad \bar{y} = GS\bar{d}$$

Small $|S| \Rightarrow$ feedback is beneficial.

Fact: S(s) + T(s) = 1

(Multivariable: S(s) + T(s) = I)

Hence trade-off:

- Small $|S(j\omega)|$ at low frequencies
- Small $|T(j\omega)|$ at high frequencies.

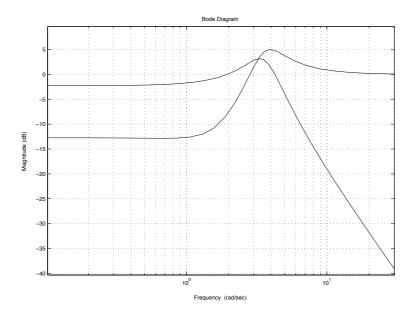


Figure 1.1: Sensitivity (S) and Complementary Sensitivity (T)

1.4 Steady-state Error

Constant reference: $r(t) = \alpha$, $\bar{r}(s) = \frac{\alpha}{s}$. Assume closed-loop is *asymptotically stable*.

$$\lim_{t \to \infty} e(t) = S(0)\alpha = \frac{1}{1 + L(0)}\alpha$$

(Final Value Theorem or
$$S(j\omega)$$
 with $\omega = 0$)

$$\lim_{t\to\infty} e(t) = 0$$
 if $|G(0)K(0)| = \infty \iff G(s)K(s)$ has pole at $s = 0$ — integral action

Constant disturbance: $d(t) = \beta$, $\bar{d}(s) = \frac{\beta}{s}$.

 $\bar{e} = -GS\bar{d}$. Zero steady-state error $\Leftarrow K(s)$ has pole at s = 0.

Ramp reference: $r(t) = \alpha t$, $\bar{r}(s) = \frac{\alpha}{s^2}$.

$$\lim_{t\to\infty}e(t)=\lim_{s\to 0}s\bar{e}(s)=\lim_{s\to 0}\tfrac{\alpha}{s}S(s).$$

(Final Value Theorem)

Hence:

- Finite steady-state error $\Leftarrow G(s)K(s)$ has a pole at s=0.
- Zero steady-state error $\Leftarrow G(s)K(s)$ has two poles at s=0.

Small steady-state error requires high gain at "DC".

1.5 The Nyquist Stability Theorem

Motivation:

- The frequency response can be determined experimentally.
- Or from transfer function or state-space model.
- Want a test for closed-loop stability that uses open-loop information.

Theorem:

- Plot $L(j\omega) = G(j\omega)K(j\omega)$ on the Argand diagram, for $-\infty < \omega < +\infty$ the *Nyquist plot*.
- The closed loop is stable if and only if the Nyquist plot encircles the point -1 + j0 p_u times counterclockwise, where p_u is the number of *unstable poles* of G(s) and K(s).

2 The Root-Locus Method

2.1 An Example

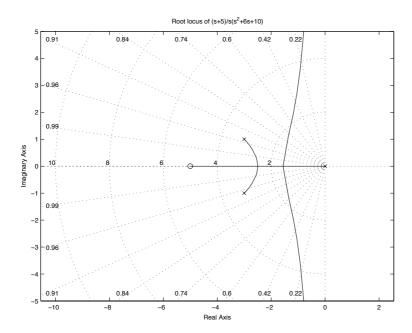


Figure 2.1:

Root-locus diagram for

$$L(s) = \frac{s+5}{s(s^2+6s+10)}$$

This shows the locations of the roots of 1 + kL(s) = 0 for k > 0.

In this case $0 \le k \le 2 \times 10^4$.

Useful when the loop dynamics are fixed, and only the gain varies.

2.2 The Angle Condition

$$L(s) = \frac{n(s)}{d(s)} = \frac{c(s - z_1)(s - z_2) \dots (s - z_m)}{(s - p_1)(s - p_2) \dots (s - p_n)}$$

where any complex zeros or poles occur in conjugate pairs and $m \le n$.

We assume for the moment that c > 0.

Suppose that s_0 is on the root-locus:

$$1 + kL(s_0) = 0 \Rightarrow L(s_0) = -\frac{1}{k}$$
 real and negative (2.1)

Hence *angle condition* for s_0 to be on the root-locus:

$$\angle L(s_0) = (2\ell + 1)\pi \tag{2.2}$$

$$\sum_{i=1}^{m} \angle(s_0 - z_i) - \sum_{i=1}^{n} \angle(s_0 - p_i) = (2\ell + 1)\pi$$
 (2.3)

2.3 Finding Gain from the Root-Locus Plot

Once a root-locus plot has been obtained, it can be calibrated with k values. From (2.1) we have, at a point s_0 on the root-locus:

$$k = \frac{1}{|L(s_0)|}$$

$$= \frac{1}{c} \times \frac{|s_0 - p_1| \times |s_0 - p_2| \times \dots \times |s_0 - p_n|}{|s_0 - z_1| \times |s_0 - z_2| \times \dots \times |s_0 - z_m|}$$

2.4 Constructing the Root-Locus Plot

Nowadays we can use software to draw root-locus diagrams (eg control.rlocus in Python).

But it is useful to have some understanding of how the form of the locus is determined. A set of about 15 'construction rules' has been developed. The 5 most important ones are given here. They are all consequences of (2.3) and properties of polynomials.

- Rule 1. The root-locus diagram is symmetric with respect to the real axis and consists of n branches.
- Rule 2. For k = 0 the *n* branches start at the open loop poles p_i . As $k \to \infty$, *m* branches tend to the zeros z_i and n m branches tend to infinity.
- Rule 3. Points on the real axis which lie to the left of an odd number of poles and zeros are on the root-locus.
- Rule 4. The breakaway points are those points on the root-locus for which $\frac{d}{ds}L(s) = 0$ (same as dk/ds = 0).
- Rule 5. As $k \to \infty$, the n-m branches which tend to infinity do so along straight line asymptotes at angles $(2\ell+1)\pi/(n-m)$ to the +ve real axis $(\ell=0,\ldots,n-m-1)$, and emanate from the point ('centre of gravity' pole +ve mass, zero -ve mass):

$$\frac{\sum_{i=1}^n p_i - \sum_{i=1}^m z_i}{n-m}.$$

Proof of Rule 3:

Consider a point s_0 , a pole p_i , and a zero z_i , all on the real axis.

$$\angle(s_0 - p_i) = \begin{cases} 0 \text{ if } s_0 > p_i \\ \pi \text{ if } s_0 < p_i \end{cases}$$

The same holds for $\angle(s_0 - z_i)$. Rule 3 follows from (2.3).

Example of use of Rule 3:

Suppose that G(s) has one pole and one zero in the right half-plane, eg $p_1 = +5$, $z_1 = +2$. Rule 3 shows that K(s) must have at least one pole to the right of +2.

— the controller must be *unstable*!

(Figuring out the details is often easier from Bode plots etc.)

Proof of Rule 2:

$$d(s_0) + kn(s_0) = 0$$
 from (2.1)

So if k = 0 then $d(s_0) = 0 \Rightarrow$ Branches start at poles.

Also, for any fixed k,

(2.1)
$$\iff \frac{1}{k}d(s_0) + n(s_0) = 0$$

So, as $k \to \infty$ the finite roots tend to zeros (i.e. finite branches end at zeros). There can be at most m of them.

(to see this, put $n(s) = (s - s_0)n'(s)$, then for s close to s_0 we have $s = s_0 - \frac{1}{k} \frac{d(s_0)}{n'(s_0)} \rightarrow s_0$.)

The remaining n - m branches go to ∞ , but how?

Proof of Rule 5:

Application to previous example:

n=3, m=1, so n-m=2. Two asymptotes at angles $\pi/2$ and $3\pi/2$.

Asymptotes emanate from (Rule 5):

$$\frac{(0-3-3)-(-5)}{3-1}=-\frac{1}{2}$$

Proof of Rule 4:

Application of Rule 4 to example of Fig.2.1:

$$\frac{d}{ds} \left(\frac{s+5}{s^3 + 6s^2 + 10s} \right) = 0$$

$$\Rightarrow \frac{1(s^3 + 6s^2 + 10s) - (s+5)(3s^2 + 12s + 10)}{(s^3 + 6s^2 + 10s)^2} = 0$$

$$\Rightarrow \frac{-2s^3 - 21s^2 - 60s - 50}{(s^3 + 6s^2 + 10s)^2} = 0$$

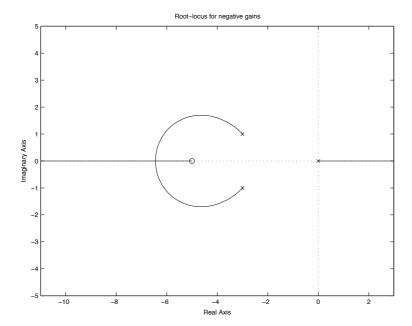
$$\Rightarrow \frac{-2(s+1.5505)(s+2.5)(s+6.4495)}{(s^3 + 6s^2 + 10s)^2} = 0$$

From Fig.2.1 it is seen that the root at -6.4495 is not on the root-locus. The other two roots give the *breakaway points* (ie repeated roots).

2.5 Root-locus for negative k (or negative c)

$$1 + kG(s_0) = 0 \quad \Rightarrow \quad G(s_0) = -\frac{1}{k} > 0$$
$$\Rightarrow \quad \angle G(s_0) = 2\ell\pi$$

- Rules 1,2,4 remain unchanged.
- Rule 3: Replace 'odd' by 'even'.
- Angles of asymptotes become $2\ell\pi/(n-m)$. (Points from which asymptotes emanate remain unchanged.)



Root-locus diagram for negative gain when

$$L(s) = \frac{s+5}{s(s^2+6s+10)}$$

2.6 Studying Parameter Variations

Root-locus diagrams can be used to study the variation of closed-loop poles as other parameters vary — not just the loop-gain k.

All that is needed is to put the closed-loop characteristic equation into the form

$$1 + \lambda H(s) = 0 \tag{2.4}$$

where λ is the parameter that is varying, and H(s) is a transfer function.

Example: Robot placing objects of varying mass

The 1-D equation of motion of a robot moving a mass m with viscous friction c and elastic tether is $m\ddot{x} = u - c\dot{x} - \alpha x$ where x is the mass position and u is the applied force. The use of a PI controller is proposed, with a transfer function k(s+z)/s.

With m = 0.1 kg, c = 0.6 N/(m/sec), $\alpha = 1$ N/m and z = 5 we have

$$G(s) = \frac{1}{0.1s^2 + 0.6s + 1} = \frac{10}{s^2 + 6s + 10}$$
 and $K(s) = k\frac{s + 5}{s}$

Letting L(s) = G(s)K(s)/k the closed-loop characteristic equation is 1 + kL(s) = 0.

Using Fig.2.1, 10k = 1.395 places two closed-loop poles at -1.55 (one of the breakaway points) and the third pole at -2.9.

What if the mass varies?

The closed-loop characteristic equation is

$$1 + k \frac{(s+5)}{s(ms^2 + 0.6s + 1)} = 0$$

which has the same roots as

$$(ms^3 + 0.6s^2 + s) + k(s+5) = 0$$

or

$$1 + \frac{1}{m} \frac{0.6s^2 + [1+k]s + 5k}{s^3} = 0$$

which is in the form of (2.4) with $\lambda = 1/m$. The root-locus plot for this, with k = 1.395/10, is shown in Fig.2.2. The roots with m = 0.1 are marked.

Variations of closed-loop poles as 1/m varies can be clearly seen.

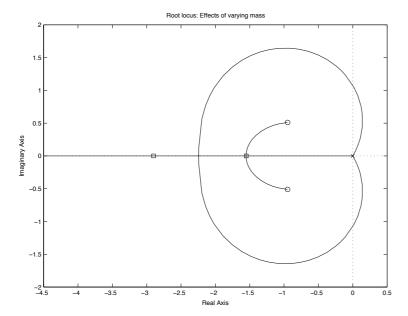


Figure 2.2: Root-locus diagram for variation of mass in robot problem.

3 The Routh-Hurwitz Criterion

The closed-loop characteristic equation

$$1 + G(s)K(s) = 0$$

has the same roots as

$$d_G(s)d_K(s) + n_G(s)n_K(s) = 0$$
 polynomial

The Routh-Hurwitz criterion tests whether a polynomial has any roots with nonnegative real parts. So it tests for asymptotic stability.

Sometimes useful for finding value of k at which root-locus crosses imaginary axis.

Consider the polynomial (assume $a_0 > 0$):

$$a_0s^n + a_1s^{n-1} + a_2s^{n-2} + \dots + a_{n-1}s + a_n$$
 (3.5)

Easy to check that all roots have negative real parts only if $a_i > 0$ for each i.

A Routh array can be constructed for arbitrary n

— see Franklin, Powell and Emami-Naeini, 3rd edition, sec.4.4.3 (for example) for details.

For n = 2, 3, 4 simplifies as follows:

These are in Electrical and Information Data Book

All the roots of (3.5) have negative real parts if and only if:

n = 2: $a_i > 0$, No other conditions

n = 3 : $a_i > 0$, $a_1 a_2 > a_0 a_3$

n = 4 : $a_i > 0$, $a_1 a_2 a_3 > a_0 a_3^2 + a_4 a_1^2$