```
clc;
close all;
clear;
%% 1. Input Data
% --- Physical Data ---
FLUID = 'T';
                        % Insert W, T or A for water, Therminol 66 or Air
Di = 0.020;
                         % Internal diameter (m)
Do = 0.024;
                        % External diameter (m)
                        % Tube length (m)
L = 20;
pin = 2e5;
                        % Inlet pressure (Pa)
Tin = 95 + 273.15; % Inlet temperature (K)
Text = 20 + 273.15; % Ambient temperature (K)
pext = 101325;
                        % Ambient pressure (Pa)
lambda t = 36;
                        % Thermal conductivity of tube (W/m·K)
g = 9.81;
beta = 1 / Text;
% --- Numerical Data ---
                        % Number of control volumes
N = 100;
tolerance = 1e-5;
                        % Convergence criterion
del x = L / N;
                        % Control volume length
%% Assign thermophysical property functions based on the fluid type
if strcmp(FLUID, 'W')
   vin = 1;
                             % Inlet velocity (m/s)
    density = @density water;
    viscosity = @viscosity_water;
    conductivity = @conductivity water;
    SP = @SP water;
elseif strcmp(FLUID, 'T')
    vin = 1;
                             % Inlet velocity (m/s)
    density = @density therminol;
    viscosity = @viscosity therminol;
    conductivity = @conductivity therminol;
    SP = @SP therminol;
elseif strcmp(FLUID, 'A')
    vin = 30;
                              % Inlet velocity (m/s)
    density = @density air;
    viscosity = @dynamic visc air;
    conductivity = @conduct air;
    SP = @SP_air;
end
%% 2. Geometry and Inlet Properties
% --- Geometry Calculations ---
S_f = pi * (Di / 2)^2;
                             % Fluid cross-sectional area (m²)
                            % Fluid cross-sectional area (m²)
S t = pi * (Do^2-Di^2)/4;
Pi = pi * Di;
                             % Internal perimeter (m)
Po = pi * Do;
                              % Internal perimeter (m)
% --- Inlet Properties ---
```

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%% 2.1 Mesh Definition
                               % Control volume boundaries
Xp f = linspace(0, L, N + 1);
Xp_t = [0, (Xp_f(1:end-1) + Xp_f(2:end)) / 2, L]; % Node positions for the tube
%% 3. Initial Conditions
T t = linspace(Tin, Text, N + 2); % Initial temperature for tube nodes
v_f = vin * ones(1, N + 1);
                                           % Initial velocity (m/s)
P_f = pin * ones(1, N + 1);
                                           % Initial pressure (Pa)
T f = linspace(Tin, Text, N + 1);
                                                % Initial temperature (K)
ro f = roin * ones(1, N + 1);
                                          % Initial density
%% Start global iteration
convergenza_glo = false;
while ~convergenza glo
    %% 4. Internal Fluid Evaluation
    convergenza int = false; % Convergence flag for internal fluid
    alpha i = zeros(1, N); % Initialize the vector for alpha <math>i
    while ~convergenza int
        T old = T f; % Store previous iteration values
       P old = P_f;
       ro old = ro f;
       v old = v f;
        for i = 1 : N
            % Average properties in control volume
            T i = (T f(i+1) + T f(i)) / 2;
            P i = (P f(i+1) + P f(i)) / 2;
            v_i = (v_f(i+1) + v_f(i)) / 2;
            ro i = (ro f(i+1) + ro f(i)) / 2;
            % Thermodynamic and flow properties
            mu i = viscosity(T i, ro i);
            muw i = viscosity(T t(i), ro i);
            cp i = SP(T i);
            lambda i = conductivity(T i);
            Re i = (ro_i * v_i * Di) / mu_i;
            Pr_i = (mu_i * cp_i) / lambda_i;
            Gz i = Re i * Pr i * Di / del x;
            alpha_i (i) = alpha(Re_i, Pr_i, Gz_i, mu_i, muw_i, lambda_i, Di, 🗸
del x);
           f i = f(Re i);
            % Update equations
            P_f(i+1) = P_f(i) - (m_in * (v_f(i+1)-v_f(i)) + f_i * Pi * del_x * \checkmark
((ro i * v i^2) / 2)) / S f ;
            T_f(i+1) = T_f(i) + (alpha_i(i) * Pi * del_x * (T t(i+1) - T i) - m in 
* (v f(i+1)^2-v f(i+1)^2)/2)/(m in*cp i);
            ro_f(i+1) = density(T_f(i+1), P_f(i+1));
            v_f(i+1) = m_in / (ro_f(i+1) * S_f);
        end
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```
% Compute residuals for convergence check
     residual T = max(abs(T f - T old));
     residual_P = max(abs(P_f - P_old));
     residual_ro = max(abs(ro_f - ro_old));
     residual v = max(abs(v f - v old));
     % Check convergence
     if max([residual_T, residual_P, residual_ro, residual_v]) < tolerance</pre>
        convergenza int = true;
     end
end
alpha ext = zeros(1, N); % Save vector
 for i = 1:N
     % All variables evaluated at film temperature Tm = 0.5 * (Text + T t(i))
     Tm = 0.5 * (Text + T_t(i+1));
    mu = dynamic_visc_air(Tm);
    ro= density air(Tm, pext);
     lambda= conduct air(Tm);
    beta= 1/Tm;
     cp= SP air(Tm);
     Gr = g * beta * ro^2 * abs(T t(i+1) - Text) * Do^3 / mu^2;
     Pr = mu * cp / lambda;
     Ra = Gr * Pr;
     if Ra < 10^9
        C = 0.47;
        n = 1/4;
        k = 1;
     else
        C = 0.1;
        n = 1/3;
        k = 1;
     end
     Nu ext = C * Ra^n * k;
     alpha ext(i) = (lambda * Nu ext) / Do;
 end
 %% 5. Solve the Tube Temperature
 % Update tube node temperatures iteratively
 % Get the parameter P(i)
 % (i == 1) Internal Boundary condition
aE = 1;
 aP = 1;
 P(1) = aE / aP;
 for i = 2:(N+1)
     % Calculation of coefficients
     aW = lambda t * S t / (Xp t(i) - Xp t(i-1));
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aE = lambda_t * S_t / (Xp_t(i+1) - Xp_t(i));
        aP = aE + aW + alpha i(i-1)*Pi*(Xp t(i) - Xp t(i-1))...
                     + alpha ext(i-1) *Po*(Xp t(i) - Xp t(i-1));
        P(i) = aE / (aP - aW * P(i-1));
    end
    % (i == N+2) External Boundary condition
    % aE = 0;
    % aW = lambda / (r(N+2) - r(N+1));
    % aP = aW + alpha b;
    % P(N+2) = aE / (aP - aW * P(N+1)); % This part of the code is useless
    P(N+2) = 0;
    % Get the parameter R(i)
    % (i == 1) Internal Boundary condition
    aE = 1;
    aP = aE;
   bP = 0;
    R(1) = bP / aP;
    for i = 2 : (N+1) % Internal nodes
        aW = lambda t * S t / (Xp t(i) - Xp t(i-1));
        aE = lambda_t * S_t / (Xp_t(i+1) - Xp_t(i));
        aP = aE + aW + alpha_i(i-1)*Pi*(Xp_t(i) - Xp_t(i-1))...
                     + alpha ext(i-1) *Po*(Xp t(i) - Xp t(i-1));
        bP = alpha_i(i-1)*0.5*(T_f(i) + T_f(i-1))*Pi*(Xp_t(i) - Xp_t(i-1)) \dots
           + alpha_ext(i-1) *Text*Po*(Xp_t(i) - Xp_t(i-1));
        R(i) = (bP + aW * R(i-1)) / (aP - aW * P(i-1));
    end
    % (i == N+2) External Boundary condition
    aW = 1;
    aP = aW;
   bP = 0;
   R(N+2) = (bP + aW * R(N+1)) / (aP - aW * P(N+1));
   T_{new(N+2)} = R(N+2); % Initialize the vector
    for i = (N+1):-1:1
        T \text{ new}(i) = P(i) * T \text{ new}(i+1) + R(i);
    end
    %% 6. Global Convergence Check
    if max(abs(T new - T t)) < tolerance</pre>
        convergenza glo = true;
    end
    T_t = T_new;
end
%% 7a. Final calculations
alpha i = alpha i(end);
alpha o=alpha ext(end);
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vout=v_f (end);
                                                                              %final∠
outlet velocity
                                                                              %final∠
Pout=P f(end);
outlet pressure
Tout=T_f(end) - 273.15;
                                                                              %final∠
outlet temperature
%% Table of Variables and Final Results
% Calculation of key variables and creation of the table
% Final variables
alpha i mean = mean (alpha i);
alpha_o_mean = mean (alpha_o);
T t ave = mean(T t);
                                          % Average temperature of the tube (K)
T f ave = mean(T f);
                                          % Average temperature of the fluid (K)
Qw final = m in * cp i * (Tin - T f(end)); % Total heat exchanged (W)
                                          % Final Reynolds number inside the tube
Re i final = Re i;
Pr i_final = Pr_i;
                                          % Final Prandtl number inside the tube
K i final = (mu i / muw i)^(0.14);
                                         % Nusselt correction factor
f i final = f i;
                                          % Final friction factor
alpha_i_final = alpha_i(end);
                                          % Final internal convection coefficient
Gro final = Gr;
                                          % Grashof number
Pr o final = Pr;
                                          % Final Prandtl number outside the tube
alpha_o_final = alpha_ext(end);
                                          % Final external convection coefficient
Rcond = 1 / (2 * pi * lambda t) * log(Do/Di);
Utot = ( 1 /(alpha_i_mean * Pi * L) + Rcond + (1/(alpha_o_mean * Po * L)));
Uo_final = 1 / (Utot * Po * L); % External heat transfer coefficient
Ui final = 1 / (Utot * Pi * L); % Internal heat transfer coefficient
                                          % Final outlet velocity (m/s)
vout final = vout;
                                          % Final outlet pressure (Pa)
Pout final = Pout;
                                          % Final outlet temperature (K)
Tout final = Tout;
% Create the table with the results
T = table(Re i final, Pr i final, K i final, f i final, alpha i final, ...
    Gro final, Pr o final, alpha o final, Uo final, Ui final, ...
    vout final, Pout final, Tout final, Qw final, ...
    'VariableNames', {'Re_i', 'Pr_i', 'K_i', 'f_i', 'alpha_i', 'Gro', 'Pr_o', ...
    'alpha o', 'U o', 'U i', 'v out', 'P out', 'T out', 'Q w'});
% Display the table
disp('Table of Final Results:');
disp(T);
%% Figura (1): Temperature T f e T t
figure(1);
x = Xp_f;
T_t = T_t(2:end);
% Grafico con un unico asse
plot(x, T f - 273.15, 'b-', 'LineWidth', 1.5); % T f in blu
hold on;
plot(x, T t - 273.15, 'r--', 'LineWidth', 1.5); % T t in rosso
hold off;
```

```
% Etichette e dettagli
ylabel('Temperature (°C)');
xlabel('Position (m)');
grid on;
title('Figure (1): Temperature of Fluid (T f) and Tube (T t)');
legend({'T f (Fluid Temperature)', 'T t (Tube Temperature)'}, 'Location', ✓
'northeast');
%% Figura (2): Pressione P f e Velocità v f
figure(2);
% Asse per P_f (sinistra)
yyaxis left;
plot(x, P f / 10^5, 'b-.', 'LineWidth', 1.5); % P f in verde
ylabel('P f (Pressure, bar)', 'Color', 'b'); % Colore asse coerente con P f
ylim([0, max(P f / 10^5) + 1]);
xlabel('Position (m)');
grid on;
% Asse per v f (destra)
yyaxis right;
plot(x, v f, 'r-', 'LineWidth', 1.5); % v f in magenta
ylabel('v_f (Velocity, m/s)', 'Color', 'r'); % Colore asse coerente con v_f
ylim([0, max(v f) + 5]);
% Titolo e legenda
title('Figure (2): Pressure (P f) and Velocity (v f)');
legend({'P f', 'v f'}, 'Location', 'northeast');
%% Functions liquid water
% Density calculation liquid water
function sum = density water(T,~)
    sum = 847.2 + 1.298 * T - (2.657e-3) * T^2;
end
% Viscosity calculation liquid water
function sum = viscosity_water(T,~)
    if T < 353
        sum = 0.9149 - (1.2563e-2) * T + (6.9182e-5) * T^2 - ...
              (1.9067e-7) * T^3 + (2.6275e-10) * T^4 - ...
              (1.4474e-13) * T^5;
    else
        sum = (3.7471e-2) - (3.5636e-4) * T + (1.3725e-6) * T^2 - ...
              (2.6566e-9) * T^3 + (2.5766e-12) * T^4 - \dots
              (1e-15) * T^5;
    end
end
```

```
% Thermal conductivity calculation liquid water
function sum = conductivity water(T)
    sum = -0.722 + 7.168e - 3 * T - 9.137e - 6 * T^2;
end
% Specific heat calculation
function sum = SP water(T)
    sum = 5648.8 - 9.140 * T + 14.21e-3 * T^2;
end
%% Therminol 66
% Funzioni per Therminol 66
function sum = density_therminol(T,~)
    sum = 1164.45 - 0.4398 * T - 3.21e-4 * T^2;
end
function sum = viscosity therminol(T,ro)
    sum = ro * exp(-16.096 + (586.38 / (T - 210.65)));
end
function sum = conductivity therminol(T)
    sum = 0.116 + 4.9e-5 * T - 1.5e-7 * T^2;
end
function sum = SP therminol(T)
    sum = 658 + 2.82 * T + 8.97e-4 * T^2;
end
%% Internal heat transfer and friction factor functions
% Heat transfer coefficient calculation
function sum = alpha(Re, Pr, Gz, mui, muwi, lambda, D, del x)
    if Re < 2000 && Gz > 10
        K = ((D / del x)^(1/3)) * ((mui / muwi)^(0.14));
        C = 1.86; m = 1/3; n = 1/3;
    elseif Re < 2000 && Gz < 10
        K = 1;
        C = 3.66; m = 0; n = 0;
    elseif Re > 2000 && (0.6 < Pr && Pr < 100)
        K = (mui / muwi)^{(0.14)};
        C = 0.027; m = 0.8; n = 0.33;
    else
       K = 1;
        C = 0.023; m = 0.8; n = 0.04;
    Nu = C * (Re^m) * (Pr^n) * K;
    sum = (Nu * lambda) / D;
end
% Darcy friction factor calculation
function sum = f(Re)
    if Re <= 2000
        sum = 16 / Re;
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elseif Re > 5*1e3 && Re < 3*1e4
        sum = 0.079*Re^{(-0.25)};
    else
       sum = 0.046*Re^{(-0.20)};
    end
end
%% Functions for external fluid (dry air)
% Density dry air
function sum = density_air(T,p)
    sum = p / (287 * T);
end
% Dynamic viscosity
function sum = dynamic_visc_air(T,~)
    sum = (2.5393*1e-5 * (T/273.15)^0.5) / (1 + 122/T);
end
% Heat coefficient
function sum = SP air(T)
    sum = 1031.5 - 0.210 * T + 4.143*1e-4 * T^2;
end
% Conductivity
function sum = conduct air(T)
    sum = (2.728*1e-3 + 7.776*1e-5 * T);
end
```