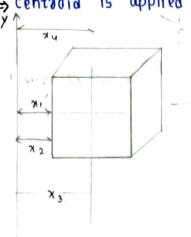
\* Centroid:

centroid of an area lies on the axis of symmetry if it exists

=> centroid is applied to plane areas or lines (geometry).



$$Y \sqrt{x} = \sqrt{1} / x_1 + \sqrt{2} / x_2 + \sqrt{3} / x_3 + \sqrt{4} / \sqrt{x_4}$$

$$\sqrt{2} = \sqrt{2} / x_1 + \sqrt{2} / x_2 + \sqrt{3} / x_3 + \sqrt{4} / \sqrt{x_4}$$

$$\sum \hat{V}_i \bar{x} = \sum \hat{V}_i x_i$$

$$\bar{x} = \frac{\sum \hat{y} \cdot \hat{x}^{i}}{\sum \hat{y}^{i}} = \frac{\sum \hat{A} \cdot \hat{x}^{i}}{\sum \hat{A}^{i}} = \frac{\sum \hat{L}^{i} \hat{x}^{i}}{\sum \hat{L}^{i}}$$

Areas :

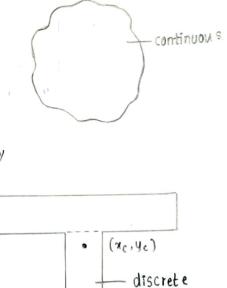
for continuous body,

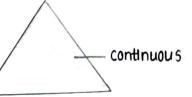
$$\frac{x_{c} = \int_{A} x dA}{\int_{A} dA} = \int_{0}^{1} \int_{0}^{1} x dx dy$$

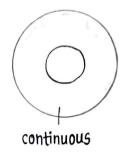
$$\frac{y_c = \int_A y dA}{\int_A dA}$$

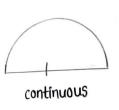
For discrete body,

$$\chi_c = \frac{\sum A_i \chi_i}{\sum A_i}$$









Ye = 
$$\int_{A} y dA$$
 $\Delta ABC \sim \Delta ABF$ 
 $\frac{b_1}{b} = \frac{h-y}{h}$ 
 $b_1 = b \left(1 - \frac{y}{h}\right)$ 

Area of element  $EF = b_1 \times dy = dA$ 

b 
$$\left(1 - \frac{3}{h}\right)$$
  
element  $EF = b_1 \times c$ 

$$dA = b\left(1 - \frac{y}{h}\right) dy$$

$$N = \int y b\left(1 - \frac{y}{h}\right) dy = b \int_{0}^{h} \left(y - \frac{y^{2}}{h}\right) dy$$

$$-\frac{y}{h} dy = b \int_{0}^{h} (y - \frac{y^{2}}{h}) dy$$
$$= b \int_{0}^{h} (y - \frac{y^{2}}{h}) dy$$

$$= b \left[ \frac{y^2}{4} - \frac{y^3}{3h} \right]_0^h$$

$$b = b \left( \frac{y^2}{2} - \frac{y^3}{3h} \right)^h$$

$$= b \left( \frac{y^2}{2} - \frac{y^3}{3h} \right)_0^h$$

$$= b \left( \frac{h^2}{2} - \frac{h^3}{3h} \right) = h^4$$

$$\frac{1}{2} \int_{\mathbb{R}^{3}} \frac{dh}{dh} dh = \frac{1}{2} \int_{\mathbb{R}^{3}} \frac{dh}{dh} \int_{\mathbb{R}^{3}} \frac{dh}{dh}$$

$$= b \left( \frac{h^2}{2} - \frac{h^2}{3} \right)$$

$$= \frac{bh^2}{6}$$
 year snownitus sot

$$= b \int_{0}^{h} (1 - \frac{y}{h}) dy = b \left[ y - \frac{y^{2}}{ah} \right]_{0}^{h}$$

$$= b \left[ h \cdot h^{2} \right]$$

$$= b \left[ h - \frac{h^2}{2h} \right]$$

$$= bh$$

$$\frac{a}{bh/2}$$

$$\frac{bh^2/6}{bh/2}$$

$$\frac{bh^2/6}{bh/2}$$

$$\frac{a}{bh/2}$$

$$\frac{a}{bh/2}$$

$$y_{c} = \frac{bh^{2}/6}{bh/2}$$

$$y_{c} = \frac{bh^{2}/6}{bh/3}$$

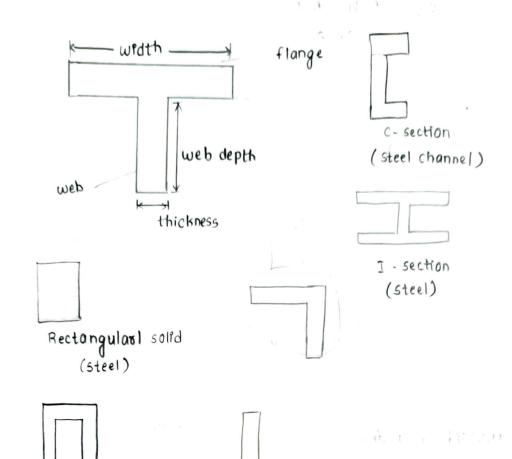
$$y_{c} = \frac{bh^{2}/6}{bh/3}$$

$$y_{c} = \frac{bh^{2}/6}{bh/3}$$

 $D = \int_{a}^{h} dA$ 

ailiolania

\*



Symmetry: For centroid => area is in same ratio.

$$\begin{array}{c|c}
 & 100 \text{ mm} \\
\hline
 & A_2 \\
\hline
 & A_1 \\
\hline
 & 80 \text{ mm} \\
\hline
 & 0 - 35 - 15 \\
\hline
 & 30 \text{ mm}
\end{array}$$

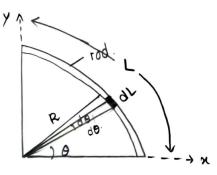
$$\alpha_c = \frac{A_1 x_1 + A_2 x_2}{A_1 + A_2}$$

Rectangular hollow

$$y_c = \frac{A_1 y_1 + A_2 y_2}{A_1 + A_2}$$

centroid of a line (rod)

\*



$$x_c = \int_{\mathcal{L}} x dL$$
 L = R0 dl = Rd0

$$\int_{\mathcal{L}} dL \qquad \qquad \chi = R d\theta$$

$$\int x \, dL = \int_{0}^{\pi/3} R \cos \theta \cdot R \, d\theta$$

$$= R^{2} \int_{0}^{\pi/2} \cos\theta \, d\theta$$

$$= R^{2} [\sin \theta]_{0}^{\pi/2} = R^{2}(1) = R^{2}$$

$$\int dL = \int_0^{\pi/2} R d\theta$$
$$= R \int_0^{\pi/2} d\theta$$
$$= R \left[\theta\right]_0^{\pi/2}$$

$$= R \frac{\Pi}{2}$$

$$x_{c} = \frac{R^{2}}{R \pi I_{d}}$$

$$= \frac{R}{\pi}$$

$$y_{c} = \int y dL$$

$$\int dL = \int_{0}^{\pi/2} R \sin \theta \cdot R d\theta$$

$$= R^{2} \int_{0}^{\pi/2} \sin \theta \cdot d\theta$$

$$= -R^{2} \left[\cos \theta\right]_{0}^{\pi/2} \cos \theta$$

$$= \frac{R\pi}{2}$$

$$\therefore (x_{c}, y_{c}) = \left(\frac{2R}{\pi}, \frac{2R}{\pi}\right)$$

$$\therefore (x_{c}, y_{c}) = \left(\frac{2R}{\pi}, \frac{2R}{\pi}\right)$$

$$\Rightarrow 0$$

4)

xc=0 yc = 29

1)

0)

Composite area or hybrid area centrold.

\*

1)

$$A_1 = \frac{1}{2} \times 300 \times 300 = 45000 \text{ mm}^2$$

$$A_2 = 800 \times 300 = 60000 \, \text{mm}^2$$

$$\chi_1 = \frac{h}{3} = \frac{300}{3} = 100 \text{ mm}$$
 (b)

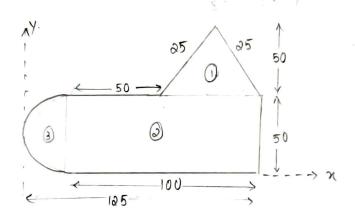
$$x_2 = \frac{b}{a} = \frac{800}{a} = -100 \text{ mm}$$

$$\alpha_{c} = \frac{A_{1}x_{1} + A_{2}x_{2} + A_{3}x_{3}}{A_{1} + A_{2} + A_{3}} \left( \frac{86}{10}, \frac{16}{11} \right) = \left( \frac{8}{10}, \frac{16}{11} \right)$$

$$y_1 = \frac{b}{9} = \frac{300}{9} = 150 \text{ mm}$$

$$y_2 = \frac{300}{2} = 150 \text{ mm}$$

$$y_3 = \frac{100}{3} = 50 \text{ mm}$$



$$A_1 = \frac{1}{2} \times 50 \times 50 = 1250 \text{ mm}^2$$

$$A_3 = \frac{1}{2} \pi \delta^2$$

$$=\frac{1}{2} \times 3.14 \times 25^2 = 961.25 \text{ mm}^2$$

$$x_1 = \frac{50/2}{25 + 50 + 25} = 100 \text{ mm}$$

$$x_2 = \frac{100}{2} + 25 = 75 \text{ mm}$$

$$x_3 = R - \frac{qR}{3\Pi} = R\left(1 - \frac{q}{3\Pi}\right)$$

$$A_1 x_1 + A_2 x_2 + A_2 x_3 = 14.4 \text{mm}$$

$$x_c = R \quad y_c = \frac{uR}{3R}$$

$$x_{c} = \frac{A_{1}x_{1} + A_{2}x_{2} + A_{3}x_{3}}{A_{1} + A_{2}x_{2} + A_{3}x_{3}} = 14.4mm$$

Th.091 mm . 17-8- 5 - 12-12 12 12

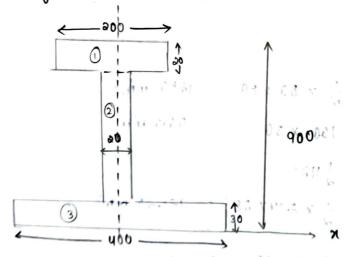
$$y_1 = \frac{60}{3} + 60 = 66.66 \text{ mm}$$

$$y_2 = \frac{50}{2} = 26 \text{ mm}$$

$$y_{c} = \frac{A_{1}y_{1} + A_{2}y_{2} + A_{3}y_{3}}{A_{1} + A_{2} + A_{3}}$$

$$(58.58)$$

$$= \frac{63325 + 125000 + 24531.25}{1231.25} = \frac{232856.25}{1231.25} = 32.201 \text{ mm}$$



$$x_1 = 0$$

3)

$$x_2 = 0$$

$$x_{c} = \frac{A_{1}x_{1} + A_{2}x_{2} + A_{3}x_{3}}{A_{1} + A_{2} + A_{3}}$$

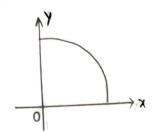
$$y_1 = \frac{30}{2} + 950 + 30 = 10 + 880 \pm 890 \text{ mm}$$

$$y_2 = \frac{850}{2} + 30 = 405 + 30 = 455 \text{ mm}$$

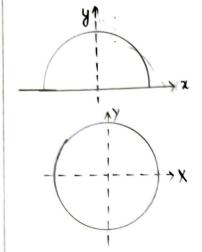
$$y_3 = \frac{30}{2} = 115 \text{ mm}$$

$$y_c = \frac{A_1y_1 + A_2y_2 + A_3y_3}{A_1 + A_2 + A_3}$$

$$= \frac{3560000 + 7735000 + 180000}{33000} = \frac{11475000}{33000} = 347.72 \text{ mm}$$



$$\chi_c = \frac{a_H}{\pi}$$
,  $y_c = \frac{a_H}{\pi}$ 



THE STREET

## \* Centroids of common shapes of areas

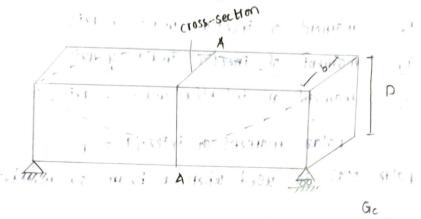
Cerebration of			
Diagram	ā	ý	Area
$\frac{1}{\sqrt{2}}$		Ы°	bh a
C C	<u>47</u> 311	<u>48</u> 311	<u>π</u> γ²
19 0	0	<u>48</u> 311	<u>₩</u> 82
	4 <u>a</u> 311	<u>Чь</u> 3П	11 ab
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	0	<u>Чь</u> 3т	11 ab
	<u>3a</u>	3h 5	aoh 3
$\frac{Q_{1}\bar{\chi}}{\varphi} \leftarrow 0 \leftarrow q \rightarrow 0$	0	3h 5	Yah 3
$ \begin{array}{c c}  & \alpha \\  & y = \kappa n^{2} \\ \hline  & C \\ \hline  & \tilde{\chi} \\ \end{array} $	<u>30</u> 4	3h 10	ah 3
ch	₹ <u>1</u>	<u> 37</u>	<u>1178</u>
3 0 3	0	A8	<u>πσ</u>
ν γ α c	osina a	0	aa7
	Diagram $ \begin{array}{c}                                     $	Diagram $ \frac{1}{3} $ $ \frac{1}{3$	Diagram $\frac{1}{3}$

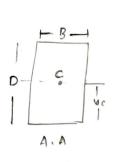
Moment of Inestia

It is a geometrical property

It is used for finding the bending capacity of a structure

It is also used for finding the deflection of a structure.





Moment (M)

Stress

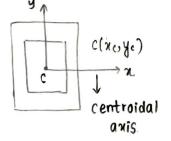
depth is more

emoment of Inertia(I)

Centroid of ye

stress

emoment of inertia is more



The moment of interia can be calculated by both best axis and controidal axis as a reference.

Moment of interia is also known as second moment of Area and moment of area =  $A \times y \times y$ .

$$\mathbf{I}_{\mathbf{z}} = \mathbf{A}_{\mathbf{y}}^{\mathbf{z}} + \mathbf{A}_{\mathbf{y}}^{\mathbf$$

where, Ix = moment of inertia



Ix - moment of inertia wirt x-axis

Ty - moment of inertia wir.t. y-axis

Ix - moment of intertia wiret x-axis

polar moment of intertia = Ip

polar MoI is used when a beam or member is twisting

for a continuous body,

$$I_{x} = \int y^{2} dA$$

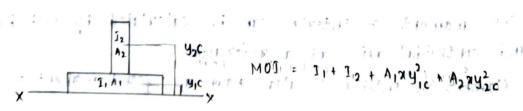


For a discontinuous body,

Individual Mos assert by adding area

$$=\frac{b_1 D_1^5}{12} + \frac{b_2 D_2^3}{12} + \dots$$

parallel axis theorem



1) Determine the Moment of interia for the rectangular section shown in figure

- (a) w.r.t centroidal axis-x
- (b) With base axis passing through base of the rectangle.

MOI is applied to area or cross section only but not applicable for line or surface.

(a) 
$$I_{x} = \int_{A} y^{2} dA$$

$$= \int_{A} h/a y^{2} \cdot b \cdot dy$$

$$= b \left[ \frac{y^3}{3} \right]_{-h/2}^{h/2} = \frac{b}{3} \left[ \frac{h^3}{8} + \frac{h^3}{8} \right]$$
$$= \frac{abh^3}{13.8} = \frac{bh^3}{12}$$

$$\begin{bmatrix} \varepsilon_{\Omega} + \frac{\varepsilon_{\Omega}}{\varepsilon} \end{bmatrix} \times \begin{bmatrix} \varepsilon_{\Omega} \\ \varepsilon \end{bmatrix} = \begin{bmatrix} \varepsilon_{\Omega} \\ \varepsilon \end{bmatrix} + \begin{bmatrix} \varepsilon_{\Omega} \\ \varepsilon \end{bmatrix} = \begin{bmatrix}$$

Parallel axis theorem, 
$$I_{xb} = I_x + A_x \left(\frac{h}{a}\right)^2$$
centroid.

$$= \frac{bh^3}{12} + bxhx \frac{hy}{y}$$
$$= \frac{bh^3}{12} + \frac{bh^3}{y} = \frac{bh^3}{3}$$

(6)

$$\frac{\text{dultiplication of the description of the desc$$

$$I_{y} = \int_{A} x^{2} dA = \int_{-b/2}^{b/2} x^{2} x dx x h = h \left[ \frac{x^{3}}{3} \right]_{-b/2}^{b/2}$$
$$= \frac{h}{3} \left[ \frac{b^{3}}{8} + \frac{b^{3}}{8} \right]$$
$$= \frac{hb^{3}}{12}$$

Solid circular beam of cross section

$$\frac{1}{12} = \int_{0}^{12} \frac{d^{2}}{dt} = \int_{0}^{12} \frac{d^{2}$$

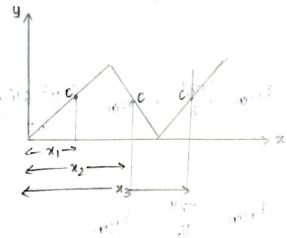
Polar moment of interia,  $I_p = I_x = I_x + I_y$ 

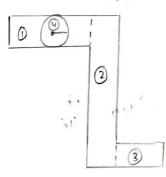
 $=\frac{19}{6 \mu_3} + \frac{19}{4 \mu_3}$ 

29/10/30/9

Line 
$$\Rightarrow$$
  $x_c = \frac{2x_{tr}}{\Sigma t_1}$ ,  $y_c = \frac{\Sigma y_{rt}}{\Sigma t_1}$ 

Volume 
$$\Rightarrow x_c = \frac{\sum x_i v_i}{\sum v_{i+1}}$$
,  $y_c = \frac{\sum y_i v_i}{\sum v_i}$ 

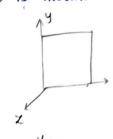


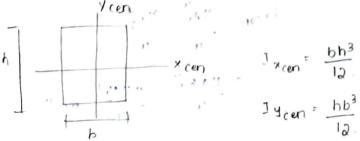


$$x_{c} = \frac{A_{1}x_{1} + A_{2}x_{2} + A_{3}x_{3} - A_{1}x_{1}}{A_{1} + A_{2} + A_{3} + A_{4}}$$

Moment of inertia:

It is meant only for areas.





$$(\chi_c, y_c) : \left(\frac{b}{3}, \frac{h}{3}\right)$$

$$\frac{y_c}{y_c} = \frac{\pi a^{\frac{1}{4}}}{y}$$

$$I_{x} = I_{y} = \frac{\pi \delta^{4}}{4}$$

$$I_{y cen} = \frac{\pi \delta^{4}}{8}, I_{x cen} = \frac{(4\pi^{2} - 64)^{44}}{13\pi}$$

$$1_{xcen} = \frac{\pi \delta^4}{16} = 1_{ycen}$$

$$1_{xcen} = \frac{hb^3}{36}, \quad 1_{ycen} = \frac{hb^3}{46}$$

(Isoceles 
$$\Delta L$$
)

$$1 \times cen = \frac{bh^3}{36} \quad 1 \times cen = \frac{hb^3}{36}$$

Composite area MOI
$$\frac{1}{x^{cen}} = \frac{\pi s_1^u - \pi s_2^u}{u} = \frac{\pi x \cdot 50^u}{u} = \frac{\pi x$$

$$= \frac{\pi \times 50^{4}}{4} - \frac{\pi \times 40^{4}}{4} \mod 4$$

$$= \frac{\pi}{4} \left[ 6050000 - 3560000 \right]$$

$$= \frac{\pi}{4} \left( \frac{c_{1}^{4}}{16} - \frac{d_{2}^{4}}{16} \right)$$

$$= \frac{\pi}{60} \left( \frac{d_{1}^{4}}{16} - \frac{d_{2}^{4}}{16} \right)$$

Pavallel axis theorem

1)

MOI with respect to centroid: Ycen

$$\int x \, can = 50 \times \frac{10^3}{12} + 50 \times 10 \times (50 - 5)^2 + 10 \times \frac{80^3}{12}$$

$$\frac{4.50 \times 10^{3}}{12}. (50-5)^{2} \times 50 \times 10$$

$$= 4166.66 + 4012500 + 426666.66 + 4166.66 + 132500$$

I y cen = 
$$\frac{10 \times 50^3}{12} + 50 \times 10 \times (0) + \frac{90 \times 10^3}{12} + \frac{10 \times 50^2}{12} = \frac{50 \times 10 \times 0}{12}$$

$$= 10 \times 50^{3} + 0 + \frac{60 \times 10^{3}}{12} + 10 \times 50^{3} + 0$$

$$I_{x} = I_{x_{1}} cen + A_{1}y_{c}^{2} + I_{x_{2}} cen + A_{2}y_{c_{2}}^{2} + I_{x_{3}} cen + A_{3}y_{c_{3}}^{2}$$

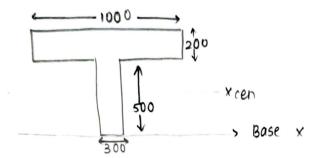
$$= \frac{50 \times 10^{3}}{10} + \frac{50 \times 10}{10} \left( \frac{10}{2} + 90 \right)^{2} + \frac{10 \times 80^{3}}{10} + \frac{80 \times 10}{80 \times 10} \left( \frac{80}{2} + 10 \right)^{2}$$

$$+ \frac{50 \times 10^{3}}{10} + 10 \times 50 \left( \frac{10}{2} \right)^{2}$$

= 4166.66 + 4512500 + 426666.66 +2000000 + 4166.66 +

= 6959999.98 mm4

2)



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(3)

Radius of Gyvation

$$91 = \sqrt{\frac{I}{A}}$$

91 - determines in which the structure buckles

$$\mathfrak{H}_{x} = \sqrt{\frac{J_{x}}{A}}$$

$$\mathfrak{H}_{y} = \sqrt{\frac{J_{y}}{A}}$$

If 9y < 91x the buckling happens with respect to y-axis. 91x < 91y the buckling happens with respect to x-axis

$$T_x = 3.8 \times 10^{-5} \text{ m}^{4}$$
  
 $T_y = 4.5 \times 10^{-6} \text{ m}^{4}$ 

ing phote 11.5 m² lasore nardites to such on

$$91x = \sqrt{\frac{1}{A}}$$

$$91y = \sqrt{\frac{1}{A}}$$

$$1011 = \sqrt{\frac{1}{A}}$$

$$= \sqrt{3.8 \times 10^{-5} \text{ m}^4} = \sqrt{4.5 \times 10^{-6} \text{ m}^4}$$

$$= \sqrt{3.533 \times 10^{-5} \text{ m}^6} = \sqrt{1.7320 \times 10^{-3} \text{ m}^4}$$

$$= 0.0050332 \text{ m}$$

$$= 0.0050332 \text{ m}$$

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The state of the s

in is not off as a quarter in the speciment