

## ABSTRACT

KASHI, ADITYA. Mesh Movement and Curved Mesh Generation. (Under the direction of Hong Luo.)

This report describes methods of mesh movement for computational fluid dynamics (CFD) problems. The methods described can be classified into two families - elasticity-based methods and interpolation methods. The former class includes lineal and torsional spring analogy methods and linear elasticity method and its variants. Interpolation methods described here include the ‘Delaunay graph’ mapping, interpolation by radial basis functions, and a combination of Delaunay graph mapping and radial basis function interpolation. The merits and disadvantages of these methods have been discussed, and some two-dimensional test cases have been presented. Further, a method for generating curved meshes from linear meshes is presented in this work. The method is designed to generate a curved mesh for any type of grid in a computationally inexpensive and numerically robust manner. In this method, high-order nodes are first placed in the linear mesh, the high-order boundary nodes are then relocated to the true boundary, and finally, the high-order interior nodes are moved using one of the mesh-movement methods, based on the motion of the high-order boundary nodes. The developed method is used to generate curved meshes for a number of complex geometries. The numerical experiments indicate that the RBF-interpolation method is much better than the prevalent linear-elasticity methods for generating high-order curved meshes judging by curved-mesh quality, computing cost, and ease of implementation.

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Mesh Movement and Curved Mesh Generation

by  
Aditya Kashi

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Hong Luo  
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## DEDICATION

To my parents.

## BIOGRAPHY

The author was born in Kasauli, HP, India and lived throughout his childhood in Delhi, India. He completed his Bachelor degree in Mechanical Engineering and integrated Masters degree in Biological sciences from Birla Institute of Technology and Science, Pilani, India. During this time, he completed internships related to computational engineering in Indian Institute of Science, Bangalore, and Centre for Fuel Cell Technology, International Advanced Research Centre for Powder Metallurgy and New Materials, Chennai, India. In August 2014, he began the MS Mechanical Engineering program at NC State University, and went on to start work at the Computational Fluid Dynamics Laboratory under the guidance of Professor Hong Luo.

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# Chapter 1

## Introduction

Moving meshes are required in various problems in computational fluid dynamics (CFD), such as aeroelasticity, aerodynamic shape optimization [8] and some kinds of multi-phase/multi-component flow simulations. Mesh movement may also be used in generating curved meshes for spatially high-order computational methods [12]. In general, good properties of a mesh movement scheme are as follows.

1. It should be robust. In many kinds of meshes, such as meshes required for viscous flow computations, even small boundary movements can invalidate the mesh. The scheme should be able to preserve mesh validity, at the very least.
2. Mesh quality should be preserved to a great extent. Elements should not become highly distorted after movement, as this can impact flow computations on the deformed mesh.
3. It should be computationally inexpensive. Many applications need very fast mesh movement schemes as they require the mesh to be moved many times during the simulation, sometimes every time step.

Some mesh movement schemes that we present here satisfy only one or two of the above criteria. Our aim is to find a scheme that satisfies all three criteria well.

## Chapter 2

# Review of mesh movement methods

### 2.1 Elasticity methods

Elasticity-based methods are among the oldest mesh-movement methods in use. These include lineal spring analogy, torsional spring analogy, and various kinds of elastic solid mechanics analogies. These methods model the mesh as some kind of solid entity, impose a boundary displacement and solve equilibrium equations to propagate the motion to the interior nodes.

#### 2.1.1 Spring analogies

Lineal spring analogy is the simplest method that can be used to move internal mesh points in response to movement of the boundary, invented by J.T. Batina. Each edge of the mesh is assumed to represent a lineal spring connecting the two nodes which make up the edge. The idea is that after imposing boundary movement, the mesh is allowed to “go to equilibrium”. At every mesh point, we have

$$\sum_j k_{ij}(\Delta \mathbf{r}_i - \Delta \mathbf{r}_j) = \mathbf{0} \quad \forall i \quad (2.1)$$

where  $i$  ranges over all nodes,  $j$  ranges over points surrounding node  $i$  and  $\Delta \mathbf{r}_i$  is the displacement of node  $i$ .  $k_{ij}$  is the stiffness of the spring between nodes  $i$  and  $j$ , which can be taken as

$$k_{ij} = \frac{1}{\|\mathbf{r}_i - \mathbf{r}_j\|}. \quad (2.2)$$

This method is not reliable as it generates invalid elements (having zero or negative value of the Jacobian determinant of the reference-to-physical element mapping) for even relatively small displacements, depending on the mesh. However, for the application it was developed, it sometimes gives acceptable results. It is used, for example, by Mavriplis *et. al.* [15] in an optimization problem involving inviscid flow only. It is also less computationally expensive

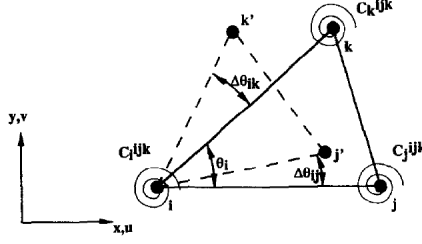


Figure 2.1: Movement of an element in torsion spring analogy

compared to other elasticity-based methods.

Note that separate problems are solved in each coordinate direction, which are uncoupled. It is the author's opinion that this is the main reason for its unreliability for many kinds of mesh and movement combinations. However, because of this, extension to 3D is trivial.

The torsional spring analogy is an extension to the previous method, which adds torsional springs at each node in each element (see Farhat *et.al.* [4]). These torsional springs resist the relative angular motion between edges of an element. This leads to a substantially more robust movement. In the scheme of Farhat *et. al.*, the problem is divided into two parts, one related to lineal spring analogy and one to the torsional spring analogy. The lineal spring model is not the same as that used by Batina; in this case the lineal spring model also leads to a coupled system which is more robust than the uncoupled scheme. The torsional part adds even more resistance to invalid elements. As shown in figure 2.1, the torsional spring analogy adds torsion springs at each node of each element. The stiffness of a torsion spring associated with a node of a certain element is a function of the angle formed by the edges of that element meeting at the node.

While this method is significantly more robust than the lineal spring analogy of Batina, it is also much more computationally expensive. A fully coupled system of  $n_{dim}N_n$  equations must be solved, where  $N_n$  is the number of nodes that are to be moved in the mesh and  $n_{dim}$  is the dimension of the problem.

### 2.1.2 Elastic solid analogy ('Elasticity')

The deformable elastic solid analogy, employing the equations of (linear or non-linear) elasticity, is one of the widely-used mesh movement techniques. This class of schemes often lead to very robust mesh movement. The mesh is assumed to model a deformable solid body, which is then deformed according to the equations of solid mechanics, that is, linear or non-linear elasticity.

The simplest approach is linear elasticity.

$$\nabla \cdot \sigma = \mathbf{0} \quad \text{in } \Omega \quad (2.3)$$

Assuming an isotropic material, the constitutive relation can be taken as

$$\boldsymbol{\sigma} = 2\mu\boldsymbol{\epsilon} + \lambda(\text{tr}\boldsymbol{\epsilon})\mathbf{I} \quad (2.4)$$

where  $\boldsymbol{\sigma}$  is the stress and  $\boldsymbol{\epsilon}$  is the strain. Finally, the linear strain-displacement relation is given by

$$\boldsymbol{\epsilon} = \frac{1}{2}(\nabla\mathbf{u} + \nabla\mathbf{u}^T) \quad (2.5)$$

where  $\mathbf{u}$  is the displacement field, with Dirichlet boundary conditions (prescribed boundary displacement  $\mathbf{u}_b$ )

$$\mathbf{u} = \mathbf{u}_b \quad \text{on } \partial\Omega \quad (2.6)$$

The weak form of this set of equations, solved by Galerkin finite element method (FEM) leads to a linear system of size  $n_d N_n$ , where  $n_d$  is the dimension of the problem and  $N_n$  is the number of nodes to be moved.

While the basic linear elasticity scheme is adequate for some applications, it is not adequate for stretched, boundary-layer meshes for viscous flow, among others. The scheme is often modified by ‘stiffening’ the mesh appropriately, so as to attain some control over the propagation of deformation into the interior of the mesh, as done, for instance, by Hartmann and Leicht [7]. In their work, the material is stiffened based on the determinant of the local Jacobian matrix of the reference-to-physical mapping, that is to say smaller elements are stiffer than larger ones.

Non-linear elasticity is claimed to be a highly robust method for mesh movement by Persson and Peraire [12]. As long as the mesh is fine enough to resolve the displacements, they claim that element validity is guaranteed. In their work, for non-linear elasticity, the constitutive equation (2.4) and strain-displacement relation (2.5) are replaced by the ‘neo-Hookean’ constitutive model

$$\mathbf{P} = \mu((\mathbf{F}^T \mathbf{F})\mathbf{F}^{-T} - \mathbf{F}^{-T}) + \lambda(\ln \det \mathbf{F})\mathbf{F}^{-T} \quad (2.7)$$

where the deformation gradient  $\mathbf{F}$  is given by

$$\mathbf{F} = \frac{\partial \mathbf{x}}{\partial \mathbf{X}}.$$

Here,  $\mathbf{x}$  is the physical position vector of a point with coordinate  $\mathbf{X}$  in the reference configuration. The system is solved using Newton-GMRES iterations.

In our opinion, this method probably cannot be used in any unsteady moving-geometry simulations, as the cost will be prohibitive. However, it leads to good results for curved mesh generation, as can be seen in Persson and Peraire’s work.

The advantage of elasticity-based methods is that they can be made highly robust. Their disadvantage is that generally, the more robust the scheme, the more expensive it is. The cost

of the system of equations that needs to be solved usually scales with the total number of mesh nodes to be moved; this can get very expensive for 3D viscous meshes. Further, the implementation is usually not very easy; different cell types require different treatments (different basis functions, for instance, in case of elastic-solid approaches) and extension from 2D to 3D may not be trivial (as in case of torsional springs).

## 2.2 Interpolation Methods

Also called algebraic methods, these methods do not involve any physics-based modeling of the mesh. They are purely mathematical manipulations of the mesh. Unlike elasticity-based methods, schemes of this type tend to be fast and independent of mesh topology.

### 2.2.1 Delaunay graph mapping (DGM)

Developed by Liu, Qin and Xia [10], DGM is a fast method of mesh movement. It consists of the following steps.

- A Delaunay triangulation is constructed from the boundary points of the mesh. This triangulation is referred to as the Delaunay graph.
- Each internal mesh point is located in the Delaunay graph, and its barycentric coordinates are calculated with respect to the Delaunay element it lies in.
- Using the prescribed boundary displacements, the Delaunay graph is moved.
- Using the calculated barycentric coordinates, the interior mesh points are mapped back to the deformed Delaunay elements keeping the barycentric coordinates constant, thus moving the mesh.

Thus, no linear system needs to be solved. The bulk of the work is in computing the Delaunay triangulation and traversing it, which can be done efficiently. This method results in a valid mesh as long as the boundary displacement does not invalidate the Delaunay graph itself. If the Delaunay graph becomes invalid, the displacement must be carried out in several steps. The Delaunay graph is re-computed each step, and thus much more boundary movement is possible without invalidating the sequence of Delaunay graphs.

In our experiments, Delaunay graph mapping is not very robust when it comes to rotational motion, or otherwise very large deformation (see the results section).

### 2.2.2 Radial basis functions

Radial basis functions (RBFs) are radially-symmetric functions. They are used as a basis for the displacement field in the moving mesh [3]. Considering  $n_b$  boundary points in the mesh, we express the displacement as

$$\mathbf{s}(\mathbf{x}) = \sum_{j=1}^{n_b} \mathbf{a}_j \phi(\|\mathbf{x} - \mathbf{x}_{bj}\|) \quad (2.8)$$

where  $\mathbf{s}$  represents the displacement field,  $\mathbf{x}$  is the position vector of any point in the domain,  $\mathbf{x}_{bj}$  is the position of the  $j$ th boundary point,  $\phi$  is a radial basis function,  $\phi(\|\mathbf{x} - \mathbf{x}_{bj}\|)$  is the  $j$ th basis function for the displacement field, and  $\mathbf{a}_j$  is the  $j$ th coordinate or coefficient in that basis.

Since we know the displacements of the boundary nodes, we can solve for the coefficients  $\mathbf{a}_j$  using

$$\mathbf{s}(\mathbf{x}_{ib}) = \sum_{j=1}^{n_b} \mathbf{a}_j \phi(\|\mathbf{x}_{ib} - \mathbf{x}_{bj}\|). \quad (2.9)$$

In each coordinate direction, this leads to a system of  $n_b$  equations in  $n_b$  unknowns:

$$\mathbf{A}\mathbf{a}_k = \mathbf{s}_k \quad (2.10)$$

where the  $(i, j)$  component of  $\mathbf{A}$  is  $\phi(\|\mathbf{x}_{bi} - \mathbf{x}_{bj}\|)$ ,  $\mathbf{a}_k \in \mathbb{R}^{n_b}$  is the coefficient vector in the  $k$ th direction and  $\mathbf{s}_k \in \mathbb{R}^{n_b}$  is the vector of boundary displacements in the  $k$ th coordinate direction.

In our implementation, we assemble  $\mathbf{A}$  as a sparse matrix by evaluating the radial basis function between every pair of boundary points. This process can be parallelized easily.

Next, we need to evaluate equation (2.8) for each interior point, which requires one evaluation of the RBF for each interior point (in each direction). This too, can be parallelized easily.

The quality of the final mesh depends on which RBF is used. Many kinds of RBFs have been used in literature [3, 13]. In this work, we mainly use Wendland's C2 function

$$\phi(x) = (1 - x)^4(4x + 1), \quad (2.11)$$

or rather, a modification,

$$\phi(x) = \begin{cases} \left(1 - \frac{x}{r_s}\right)^4 \left(4\frac{x}{r_s} + 1\right) & x < r_s \\ 0 & x \geq r_s \end{cases} \quad (2.12)$$

where  $r_s$  is a real number called the ‘support radius’. This modified function has a compact

support. In light of this, we see the matrix  $\mathbf{A}$  is sparse if the support radius is less than the characteristic dimension of the domain. For curved mesh generation, this support radius can be kept quite small relative to the whole domain, giving us a very sparse linear system which is solved quite fast.

The scaling of the argument by the support radius serves to make the value of the RBF 1.0 at the boundary. This means that points very close to the boundary will deform just like the boundary itself. This results in an essentially rigid motion of points very close to the boundary, which causes the near-boundary elements to retain their quality after the mesh movement. The support radius is chosen big enough to accommodate the expected deformation of the boundary, but small enough that the linear system is sparse. An algorithm to determine a good support radius will be very useful in this regard.

### 2.2.3 Interpolation using radial basis function on the Delaunay graph (DGRBF)

This method proposed by Wang, Qin and Zhao [16] combines both the Delaunay graph mapping and interpolation by RBF. The general scheme of the mesh movement is the same as in case of DGM, but with the important difference that interpolation is not done using barycentric coordinates of the nodes with respect to the Delaunay graph elements. Here, the interpolation is done via radial basis functions in each Delaunay graph element. The displacement of a node with initial position  $\mathbf{x}$  is given by

$$\mathbf{s}(\mathbf{x}) = \sum_{j=1}^{n_t} \mathbf{a}_j \phi(\|\mathbf{x} - \mathbf{x}_{tj}\|) \quad (2.13)$$

where  $\mathbf{x}_{tj}$  are positions of nodes of the Delaunay simplex that contains the node at  $\mathbf{x}$ , and  $n_t$  is the number of nodes in that simplex (3 in 2D and 4 in 3D). We need to solve for the  $\mathbf{a}_j$ , the RBF coefficients, by solving a  $3 \times 3$  system in 2D or a  $4 \times 4$  system in 3D, in each Delaunay element.

$$\mathbf{s}(\mathbf{x}_{ti}) = \sum_{j=1}^{n_t} \mathbf{a}_j \phi(\|\mathbf{x}_{ti} - \mathbf{x}_{tj}\|). \quad (2.14)$$

DGRBF method provides flexibility in choosing the RBF and the support radius, and with a good choice of these, robust mesh movement can be achieved for translational motion and possibly some other kinds of deformation. It has been observed in [16] and in our tests, that interpolation of displacements by DGRBF often does not give very good results for large rotational deformations. The remedy for this is to interpolate the rotational angles instead of the displacements directly, and then calculate the displacements at each interior node using these



interpolated rotation angles using the rotation matrix. For example, in 2D,

$$x_{new} = (x - x_0) \cos a_z - (y - y_0) \sin a_z + x_0 \quad (2.15)$$

$$y_{new} = (x - x_0) \sin a_z + (y - y_0) \cos a_z + y_0 \quad (2.16)$$

where  $(x_0, y_0)$  is the center of rotation, and  $a_z$  is the interpolated rotation angle at the node in question. For interpolation of the angles, the same formulae (2.13) and (2.14) are used, with the rotation angles replacing the displacements. Angle interpolation can be combined with displacement interpolation for problems involving both translation and rotation. This angle interpolation scheme, and its combination with the displacement interpolation scheme, is referred to as ‘DGRBF2’ in [16].

## 2.3 Mesh quality

In order to judge the effectiveness of mesh-movement methods, we need to measure the quality of the deformed mesh. Some mesh quality measures have been derived for linear 2D and 3D elements by Knupp [9]. These include the relative size, shape and size-shape metrics for triangles and tetrahedra, and size, shape, skew, size-shape and size-skew metrics for quadrangles and hexahedra.

The size metric distinguishes elements having very small or very large volume with respect to some reference volume. The shape metric identifies elements that have unequal edges or unequal angles between edges; it is independent of size of the element. Finally, the skew metric is a measure of unequal angles between edges of the element only, irrespective of lengths of edges and volume of element; it is different from shape in case of non-simplicial (like quadrangular and hexahedral) elements. While dealing with quadrangles, we use the skew metric.

All metrics are normalized so that they are 1.0 for ideal elements with regard to the characteristic they measure, and they are 0.0 for degenerate elements.

## Chapter 3

# Curved mesh generation

### 3.1 Introduction

With the rising popularity of high-order computational methods in the aerospace community, robust techniques to obtain a high-order representation of the boundary have become important. One such technique is to produce high-order meshes, otherwise called curvilinear or curved meshes. Another technique in this regard is isogeometric analysis [2] where CAD data is directly used in analysis. In this work, we deal with curved unstructured mesh generation, using only the linear mesh data.

The problem of curved mesh generation involves at least one issue - the placement of the boundary “high-order” nodes. Some kind of description of a high-order boundary is required, and the optimal placement of high-order boundary nodes must be decided.

Additionally, we usually need to deal with one more issue - maintaining the quality of elements near the boundary after placing the high-order boundary nodes. The deformation of the initially polygonal/polyhedral boundary to get a curved boundary usually causes the quality of boundary elements to get reduced. In cases with a viscous mesh, these elements may even become invalid [12, 14]. Thus, the deformation of the boundary must be propagated into the domain, to displace interior nodes upto at least some distance from the boundary.

In our knowledge, there are, broadly speaking, three types of methods to achieve movement of the interior nodes. The first type uses models based on solid mechanics to try to achieve valid mesh movement. Examples of this type are lineal spring analogy [1], torsion spring analogy [4], linear-elasticity (and variants thereof)[7] and non-linear elasticity [12]. Each of these requires solution of a system of equations involving the same number of variables as mesh points to be moved. The linear-elasticity based models require solution of Poisson-type partial differential equations.

The second type of technique to move the interior nodes involves *interpolation* of the bound-

ary displacement to interior nodes. Several methods exist; some examples are radial basis function (RBF) interpolation [3], ‘explicit’ interpolation [11], and ‘Delaunay graph mapping’ [10]. These methods do not involve any physics-based modeling of the mesh.

Finally, researchers have “untangled” the near-boundary elements, and otherwise improved the quality of the mesh, by optimization processes. The cost function is some kind of curved-mesh quality measure. One prominent example of this is in the Gmsh meshing software [14].

We have tried several methods from the solid mechanics and interpolation classes of mesh-movement methods. Here, we mainly discuss the RBF method and linear elasticity.

In the rest of the sections, we discuss the methodology used to generate 2D unstructured quadratic meshes using only the linear mesh, and then present examples to demonstrate the effectiveness of our method. We also compare the quality of meshes given by the RBF interpolation method and the linear-elasticity method.

## 3.2 Method

We initially preprocess the linear mesh to generate a straight-faceted high-order mesh. To do this for 2D meshes, we place extra nodes at equally-spaced positions in each facet (edge in case of 2D) of the mesh, and if required, extra nodes are also placed within the elements. Nodes of this intermediate mesh are later displaced to obtain the curved mesh. For a 2D quadratic mesh made up of triangular elements, we need a high-order node at the midpoint of every edge.

Since only the linear mesh is taken as input, a high-order boundary representation first needs to be reconstructed from the piecewise linear  $\mathbf{C}^0$  boundary. For 2D meshes, we use a cubic spline reconstruction to get a smooth ( $\mathbf{C}^2$ ) parametric curve describing the boundary. All boundary nodes are used as spline control points, and we get a cubic function between every two consecutive boundary nodes. At the control points, we require the two spline curves meeting there to share a common tangent and curvature, thus enforcing  $\mathbf{C}^2$  continuity. Corners are detected by comparing the normal vectors of the two facets sharing a point. If the dot product between the two normal vectors is below a particular threshold, the point is considered as a corner and is not smoothed over by the reconstruction procedure. Alternatively, the user can list all the corner points as input. The spline reconstruction requires the solution of a symmetric positive definite linear system, of size equal to the number of boundary nodes in the curved part of the boundary, in order to calculate the cubic spline coefficients. The system(s) can be solved quickly using a conjugate gradient (CG) solver.

Once we have the smooth reconstructed boundary, we calculate the final positions of the boundary nodes in the curved mesh. This is currently done by simply moving the high-order boundary nodes, originally at regular intervals on the boundary facet, to corresponding intervals in parameter space on the cubic spline curve associated with that facet. This method is found

to be quite robust for the meshes tested. Thus, the boundary displacements are obtained.

To propagate the boundary motion to the interior of the mesh, we favor interpolation by radial basis functions (RBFs) [3]. We have also used linear elasticity to propagate the mesh movement to the interior. The isotropic linear elasticity variational formulation, as given by Gockenbach [6], is implemented in a P2 Galerkin FEM code for generation of quadratic meshes. In this case however, the size of the linear system to be solved is proportional to the total number of nodes in the mesh.

Though the RBF method requires the solution of a linear system of size proportional to only the number of boundary points, extra computation is required to evaluate equation (2.8) at each of the interior nodes to compute their displacements. In comparison, elasticity-based methods directly give us the displacements.

# Chapter 4

## Results

### 4.1 Mesh movement

We present a case of rotation of an interior object inside a far-field boundary using interpolation methods. We show results for a 60-degree rotation in case of DGM (fig. 4.2), RBF (fig. 4.3) and DG-RBF2 (fig. 4.4) methods. Note that DGRBF2 refers to the interpolation of rotation angles by DGRBF. The case has been referred from Wang *et. al.* [16]. Figure 4.1 shows the un-deformed mesh.

We note that the RBF and DGRBF2 methods can handle the rotation case quite well. The stretched, fine elements near the inner boundaries are well-preserved, while the elements further in the interior, which are larger and can thus take more deformation, are distorted somewhat. Though RBF gives slightly better mesh quality than DGRBF2 for this amount of rotation, RBF takes 1.926 seconds of total CPU time (using conjugate gradient solver), while DGRBF2 takes only 0.08 seconds of total CPU time. Further, shown in figure 4.5 is a case of extreme rotation of 120 degrees, which DGRBF2 solves while maintaining mesh validity. None of the other methods are able to do this. RBF has been found to give valid meshes upto about 110 degrees of rotation.

While solving certain problems with the pure RBF method, we find that the linear system to be solved is sometimes quite ill-conditioned, depending upon the mesh. However, in some of the cases that we worked on (such as curved mesh generation of hybrid meshes for viscous simulations of airfoils), sparse direct methods are capable of solving the system efficiently and effectively. This is likely to hold true in general as the systems to be solved will usually not be very large (having size of the number of boundary nodes).

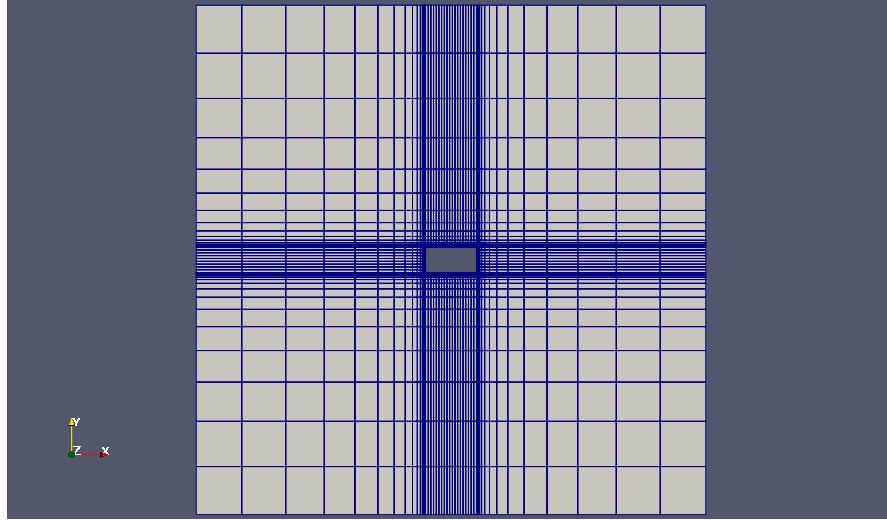


Figure 4.1: Original mesh

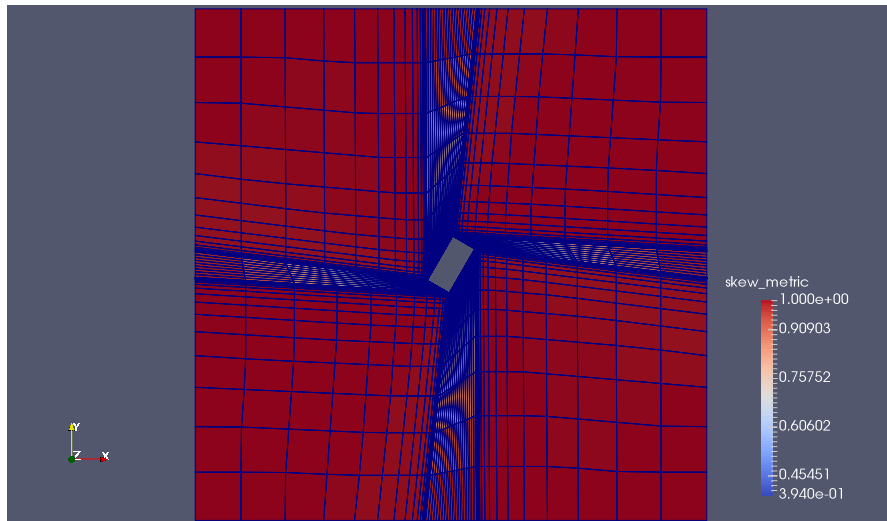


Figure 4.2: 60 degrees rotation by DGM

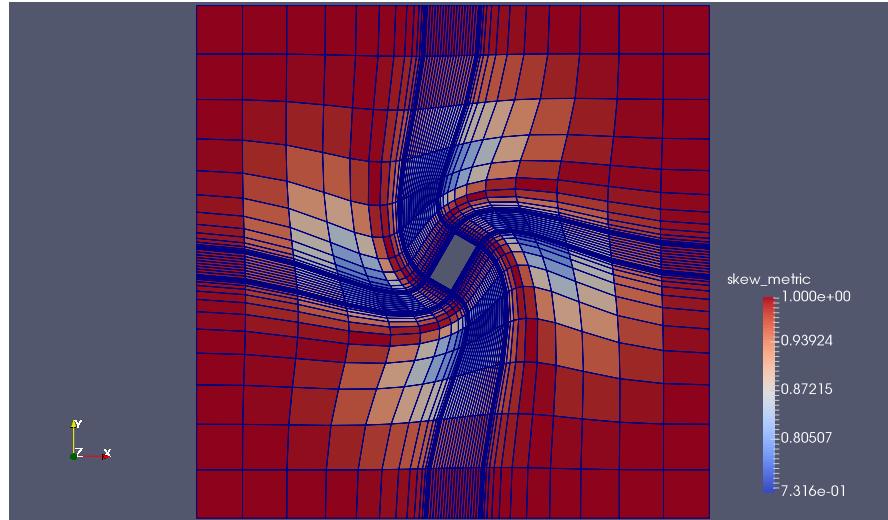


Figure 4.3: 60 degrees rotation by RBF

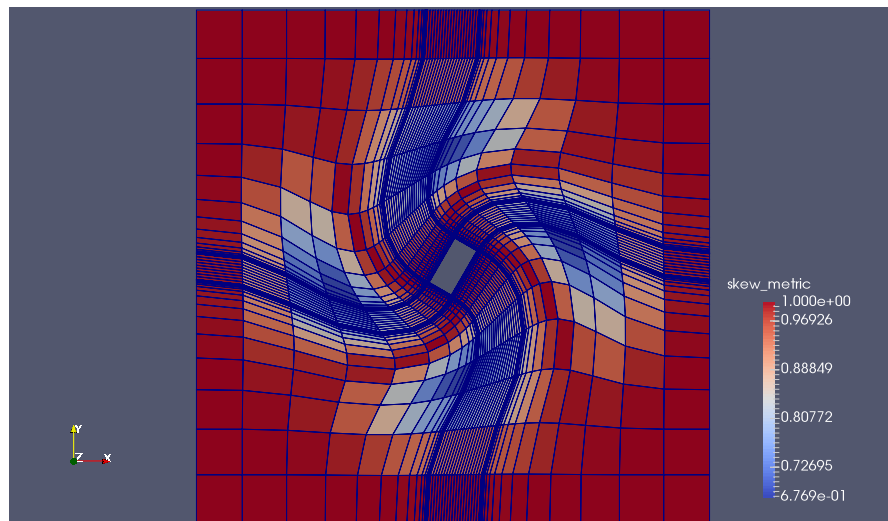


Figure 4.4: 60 degrees rotation by DGRBF2

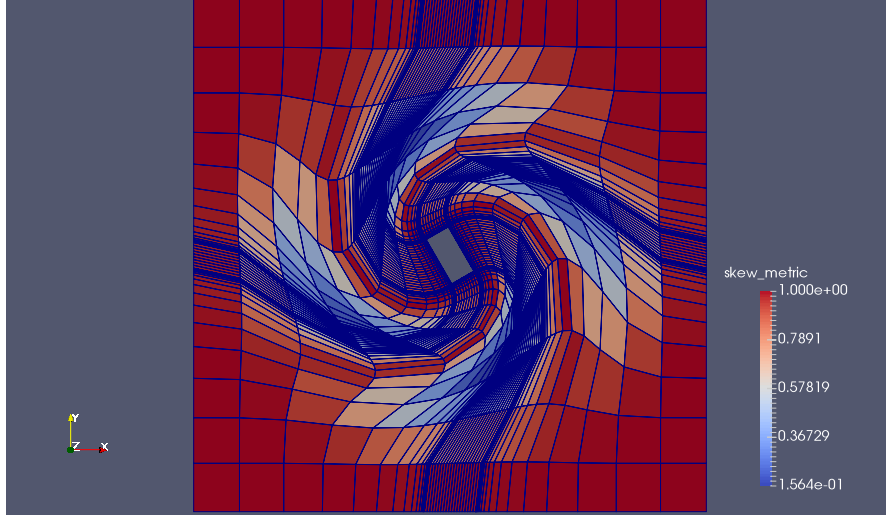


Figure 4.5: Large rotational motion carried out by DGRBF2

## 4.2 Generation of curved meshes

Curved meshes for various standard viscous flow test cases have been generated with the method described in the previous section. These include flow past NACA0012 airfoil, flow through a bump channel and flow past a multi-element airfoil. Out of these, the mesh for the multi-element airfoil case is a truly unstructured mesh; the others are structured meshes which we first convert to unstructured format.

The validity, and to some extent quality, of the generated mesh is established by computing the minimum scaled Jacobian determinant for each element. If the minimum value of the Jacobian determinant is zero or negative, the element is considered invalid. This is done as a post-processing step using the plugin ‘AnalyseCurvedMesh’ available in Gmsh [5]. The quantity computed by this plugin is given as

$$m_i = \frac{\inf_{\mathbf{x} \in \Omega_i} \det \mathbf{J}(\mathbf{x})}{\det \mathbf{J}_1}, \quad (4.1)$$

where  $\mathbf{J}$  is the Jacobian matrix of the transformation of a reference element to the curved physical element, and  $\mathbf{J}_1$  is the Jacobian of the transformation of the reference element to the linear (straight) physical element.  $\Omega_i$  represents the  $i$ th element.

Here, we present results of 2D curved mesh generation in case of a multi-element airfoil. The mesh has triangular elements. It is meant for viscous flow computations and thus has highly stretched, thin elements near the wing boundaries. In this case, if the interior nodes are not moved properly, invalid elements result. In the figures, we compare curved meshes



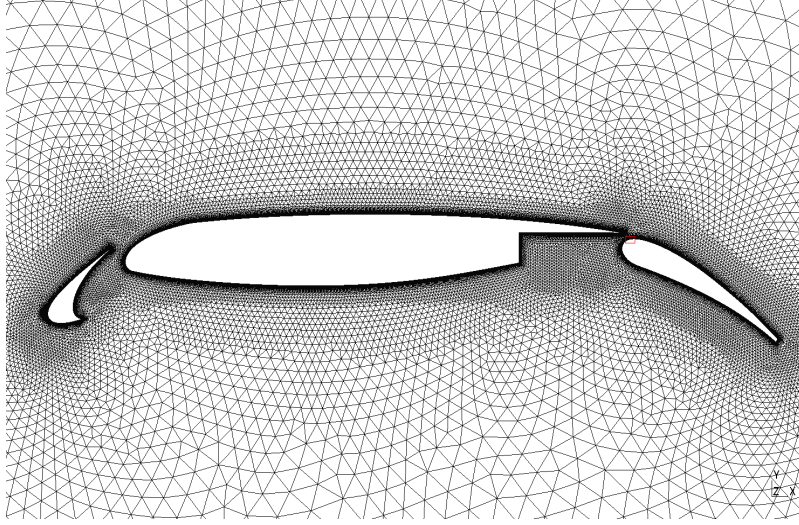


Figure 4.6: Mesh of multi-element airfoil. The little box shows approximately the region which is magnified in the figures that follow.

generated by isotropic linear elasticity and RBF interpolation. Both problems are solved using a Jacobi-preconditioned CG solver. Linear elasticity uses a Young's modulus  $E$  of  $1 \times 10^6$  Pa and a Poisson's ratio of 0.4, though it is observed that the particular values do not influence the outcome much. A number of values over the range 1.0 to  $1.0 \times 10^{10}$  were tried for  $E$ . RBF interpolation uses a support radius of 0.04 (for reference, the chord length of the wing is approximately 18.3).

In figures 4.7 and 4.8, the first mesh is tangled; even though the nodes are being moved, they are not being moved far enough. It is clear that simple linear elasticity is not enough

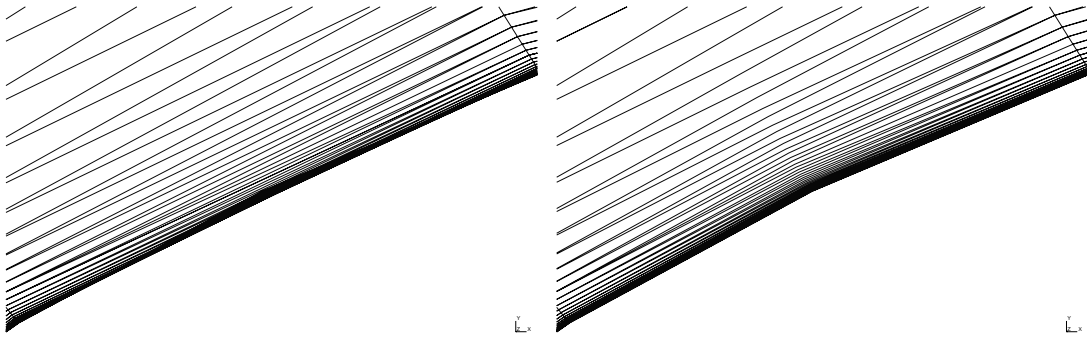


Figure 4.7: Portion of quadratic viscous mesh for multi-element airfoil, showing a boundary face in the flap, generated by linear elasticity (left) and RBF (right) methods.

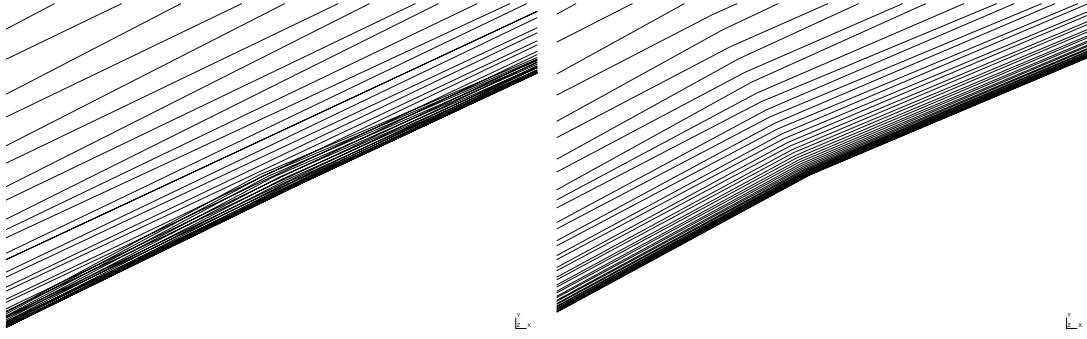


Figure 4.8: Portion of quadratic viscous mesh for multi-element airfoil, showing a boundary face in the flap, generated by linear elasticity (left) and RBF (right) methods. Both are zoomed to for a clearer view.

for curved mesh generation - we get negative Jacobians in several near-boundary elements for many curved boundary faces. However, the RBF method is satisfactory, with a scaled minimum Jacobian range of 0.64 to 1.0 throughout the mesh (figure 4.9). As explained earlier, we see that the elements very close to the boundary deform very little - they have scaled Jacobian nearly 1.0. As we go further from the boundary, the Jacobian drops to a minimum and then rises again to 1.0 at a distance that approximately equals the support radius.

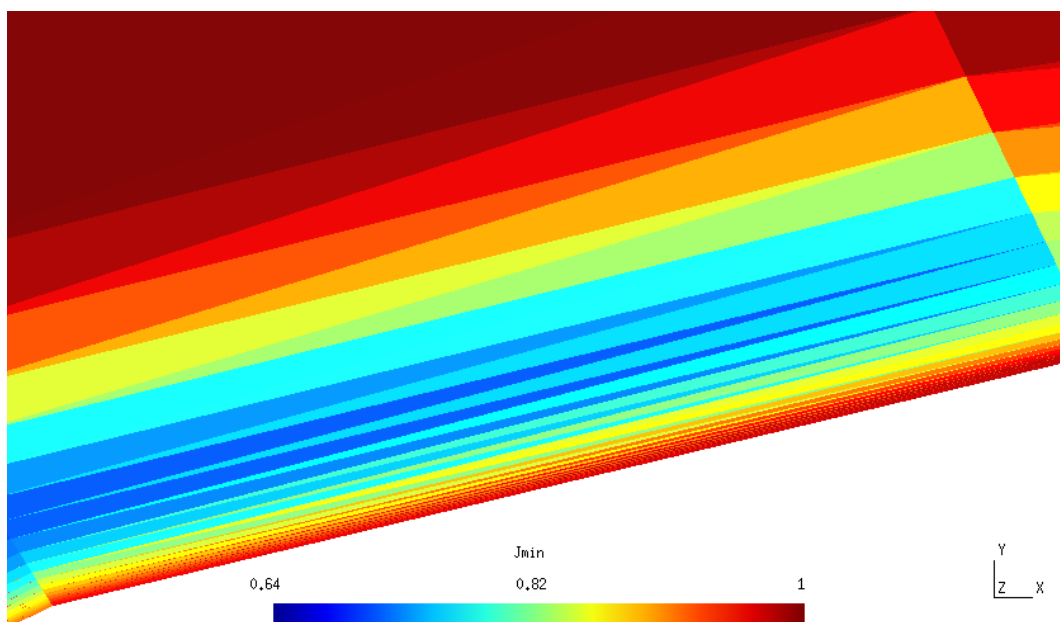


Figure 4.9: Minimum scaled Jacobian over each element for mesh generated by RBF interpolation. (The mesh is actually curved, though the graphics of Gmsh ignore that.)

## Chapter 5

# Conclusion

A lot of choice is available for mesh-movement techniques, but only a few will be good for a given application. We conclude that interpolation methods are generally well-suited for unsteady simulations that require mesh movement, as they are very fast. When deformations are relatively small and not highly rotational, DGM is a good choice while when larger and more general deformations are needed, DGRBF is best. The pure RBF method, while usually giving good robust results, is more expensive than other interpolation methods considered here.

RBF, and elastic solid analogy methods, are found to be effective for curved mesh generation, where computational cost is less of an issue. We have used RBF for curved mesh generation with good results for certain viscous flow cases.

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## APPENDIX