EE311FZ Control System Design — Project 2

Hanlin CAI (20122161) 20th Dec. 2022

System defined in transfer function form:

$$G(s) = \frac{2.3(s+18)}{(s+1)(s+3)(s+8)} \tag{1}$$

Performance specs:

$$\begin{cases} \text{Settling time: } T_s = 0.9 \\ \text{Peak overshoot: } PO \leq 10\% \\ \text{Control signal limits: } -24 \leq u \leq 24 \end{cases} \tag{2}$$

All the specs above have been explained in detail in the Producer 3.

Procedure 1 — Prior Analysis

First and foremost, as Equation (1) shown, the open-loop three order system is casual, which means that the output of the system is only related to present and past input, not to the future inputs. And we can verify as follows:

$$G(s) = \frac{2.3(s+18)}{(s+1)(s+3)(s+8)} = \frac{Y(s)}{X(s)}$$
(3)

Then, we utilize Inverse Laplace Transformation:

$$y(t) = \left(\frac{391}{280}e^{-t} - \frac{299}{120}e^{-5t} + \frac{23}{21}e^{-8t}\right) \cdot x(t) \tag{4}$$

A system is said to be causal if its output depends upon present and past inputs and does not depend upon future input. For non-causal system, the output depends upon future inputs also. As Equation (4) shown, we can know that the system is a **casual system**. Furthermore, we can convert given transfer function into standard form:

$$G(s) = \frac{2.3s + 41.4}{s^3 + 14s^2 + 53s + 40} \tag{5}$$

Moreover, we can use Nyquist plot to analyze the stability of the system.

The following Figure 1 shows the Nyquist plot of the given system. Since the Nyquist plot does not enclose (-1,0), we can get that:

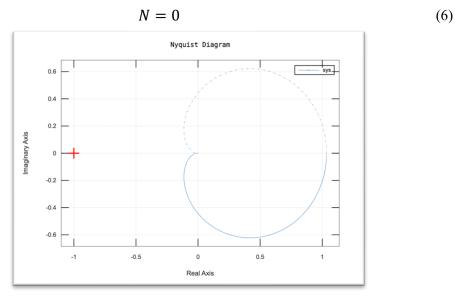


Figure 1 Nyquist plot of the given 3-order system

Considering the poles of the system, easy to get that

$$\begin{cases}
p_1 = -1 \\
p_2 = -3 \\
p_3 = -8
\end{cases}$$
(7)

All of the poles of the system meet p < 0, hence

$$P = 0 \tag{8}$$

We have already known that Z = P - N, so we can get

$$Z = 0 (9)$$

To summarize, since Z = 0, we can confirm the 3-order system is stable. The corresponding MATLAB program is shown in Table 1.

```
Table 1: MATLAB Program 1

This program is used to implement inverse Laplace transform and draw Nyquist plot.

%% Procedure 1 Prior Analysis
sqyms s;

TF1 = (2.3*s + 41.4)/( s^3 + 14*s^2 + 53*s + 40);

DE1 = ilaplace(TF1); % inverse
num1 = [2.3 , 41.4];
den1 = [1, 14, 53, 40];
sys1 = tf(num1,den1);
nyquist(sys1); % nyquist
```

Now, we have gained a basic knowledge of the system, let's move to next step.

Procedure 2 — State-space Description

The state-space description in controllable canonical form for a 3-order system can be written as follows,

$$y = [b_3 - a_3 b_0 \quad b_2 - a_2 b_0 \quad b_1 - a_1 b_0] \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + b_0 u$$
 (11)

As for the given transfer function,

Pc = inv([pc;pc*A;pc*A*A]);

$$G(s) = \frac{2.3s + 41.4}{s^3 + 14s^2 + 53s + 40} = \frac{Y(s)}{X(s)}$$
(12)

In this case, we can get the set of corresponding parameters,

$$\begin{cases}
a_1 = 14 \\
a_2 = 53 \\
a_3 = 40 \\
b_0 = 0 \\
b_1 = 0 \\
b_2 = 2.3
\end{cases}$$
(13)

Now, let's rearrange the Equations above, and we can get:

$$\begin{bmatrix} \dot{x} \\ \ddot{x} \\ \ddot{x} \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ -40 & -53 & -14 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} u \tag{14}$$

$$y = \begin{bmatrix} 41.4 & 2.3 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + 0u$$
 (15)

Also, we can utilize MATLAB to calculate directly, as shown in Table 2.

Table 2: MATLAB Program 2 This program is used to convert transfer function to state-space description. %% Procedure 2 State-space Description num2 = [2.3 , 41.4]; den2 = [1, 14, 53, 40]; [A,B,C,D] = tf2ss(num2,den2); sys2 = ss(A,B,C,D); Qc = ctrb(A,B); % controllability matrix pc = [0 0 1]*inv(Qc);

sys22 = ss2ss(sys2,inv(Pc)); % controllable canonical form

Procedure 3 — Performance Criteria

Considering the specification of a set of performance criteria, the following Table 3 illustrates the summary of the different performance criteria defined by (1) Transient performance specifications and (2) Limits on the control signal, based on Ref.^[1].

Table 3 Summary of the different performance criteria (By Hanlin CAI)

Criteria	Brief Description
Transient Perform	nance Specifications:
Delay time, t_d	The time required for the response to reach half the final value the very first time.
Rise time, t_r	The time required for the response to rise from 10% to 90%, 5% to 95%, or 0% to 100% of its final value. For underdamped $2^{\rm nd}$ -order systems, the 0% to 100% rise time is normally used. For overdamped systems, the 10% to 90% rise time is commonly used.
Peak time, t_p	The time required for the response to reach the first peak of the overshoot.
Overshoot, M_p	The maximum overshoot is the maximum peak value of the response curve measured from unity. The amount of the maximum overshoot directly indicates the relative stability of the system.
Settling time, t_s	The time required for the response curve to reach and stay within a range about the final value of size specified by absolute percentage of the final value (usually 2% or 5%).
Limits on the Cor	ntrol Signal:
Control limit, u	Control limits are used to detect signals in process data that indicate that a process is not in control and, therefore, not operating predictably. A value in excess of the control limit indicates a special cause is affecting the process.
Control Graph	The control chart is a graph used to study how a process changes over time. More detail can be accessed in Ref. [2].

Moreover, according to the Ref.^[2, 3], the following Table 4 contrasts the control limits and the specification limits.

Table 4 Difference between control limits and specification limits

Control Limits	Specification Limits
Voice of the process	Voice of the Dr. Siyuan Zhan
Calculated from Data	Defined by the customer
Appear on control charts	Appear on histograms
Apply to subgroups	Apply to items
Guide for process actions	Separate good items from bad
What the process is doing	What we want the process to do

Procedure 4 — Controllers Design

A. PID Controllers

(a) Open-loop Z-N method

First and foremost, we can use the open-loop ZN method as following steps:

- 1. Use MATLAB Simulink to simulate the given system;
- 2. Draw the step response diagram of the system;
- 3. Use for-loop to determine the max slope (point) of the step response curve;
- 4. Analyze the parameters of the fitted linear equation;
- 5. Utilize the ZN table to obtain the corresponding parameters of PID controller.

Figure 2 illustrates the Simulink diagram of the given system. Now, we use Simulink to capture the datasets of the system step response.

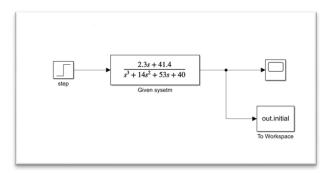


Figure 2 Simulink diagram of the system

Figure 3 shows the step response curve of the system, with the max slope point and the relevant fitted linear equation plot. Table 5 illustrates the relevant MATLAB program.

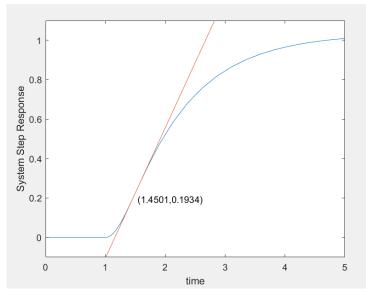


Figure 3 System step response with max slope point

Table 5: MATLAB Program 3

This program is used to find the max slope point of the system step response.

```
%% Procedure 4-1 Open-loop ZN method
% The data sets were obtained by Simulink (Figure).
time=[0;0.24000000000000;0.480000000000;0.7200000000000;0.960
00000000000;0.9999999999993;1;1.00000000001;1.05618173432533;
1.10626211619295;1.17458836333233;1.25586817150269;1.34644068723767
;1.45010569333239;1.57091185431932;1.71585866478841;1.8982108321579
7;2.13821083215797;2.37821083215797;2.61821083215797;2.858210832157
97;3.09821083215797;3.33821083215797;3.57821083215797;3.81821083215
797;4.05821083215797;4.29821083215797;4.53821083215797;4.7782108321
5797;5.01821083215797;5.25821083215797;5.49821083215797;5.738210832
15797;5.97821083215797;6.21821083215797;6.45821083215798;6.69821083
215798;6.93821083215798;7.17821083215798;7.41821083215798;7.6582108
3215798;7.89821083215798;8.13821083215798;8.37821083215798;8.618210
83215798;8.85821083215798;9.09821083215798;9.33821083215798;9.57821
083215798; 9.81821083215798; 10.0582108321580; 10.2982108321580; 10.538
2108321580;10.7782108321580;11.0182108321580;11.2582108321580;11.49
82108321580;11.7382108321580;11.9782108321580;12];
out=[0;0;0;0;0;0;0;2.34046557054457e-28;0.00380987847702979;0.01377
44245517423;0.0365675780509246;0.0747908249387998;0.127030268191934
;0.193449674806863;0.273276101769470;0.365919903393339;0.4717215567
80243;0.589268880116967;0.683568065541877;0.758308028425809;0.81727
1725373752;0.863706224494938;0.900248674980520;0.928998768131951;0.
951615842328969; 0.969407502894202; 0.983403052623937; 0.9944123830154
39;1.00307264187492;1.00988504699441;1.01524387619794;1.01945928122
106;1.02277523662095;1.02538365976076;1.02743551824683;1.0290495674
2882;1.03031922358530;1.03131797057083;1.03210361283773;1.032721620
98237;1.03320776344559;1.03359017668171;1.03389099361135;1.03412762
460792;1.03431376515727;1.03446018851098;1.03457536920959;1.0346659
7356330;1.03473724547788;1.03479330995601;1.03483741183999;1.034872
10361339;1.03489939313105;1.03492085982764;1.03493774613059;1.03495
102936801;1.03496147833347;1.03496969778147;1.03497616342877;1.0349
7667719028];
plot(time,out)
hold on;
temp = length(time) -1;
for i=1:temp
slope(i) = (out(i+1) - out(i)) / (time(i+1) - time(i));
[maxSlope,flag] = max(slope); % Find the max slope
t = time(flag)
                   % time = 1.4501
y = out(flag)
                   % \text{ out} = 0.1934
k = maxSlope
                   % \max k = 0.6608
b = y - maxSlope*t % b = -0.7648
x = 1:3;
tangent = (k*x + b);
plot(tangent);
text(t,y," (1.4501,0.1934)");
axis([0 5 -0.1 1.1]);
xlabel("time");
ylabel("System Step Response");
```

As shown in the Figure 3, the linear equation is fitted as

$$y = 0.6608 * t - 0.7648 \tag{16}$$

Hence, we can obtain that:

$$\begin{cases}
R = 0.6608 \\
L = 1.1574 \\
t_i = 2.3148 \\
t_d = 0.57869
\end{cases}$$
(17)

Then, we can obtain k_p , k_i , k_d as follows,

$$\begin{cases} k_p = \frac{1.2}{RL} = 1.5690 \\ k_i = \frac{k_p}{t_i} = \frac{k_p}{2L} = 0.67782 \\ k_d = k_p \cdot t_d = k_p \cdot 0.5L = 0.90798 \end{cases}$$
 (18)

Table 6 Open-loop ZN table

Controller	K_p	T_i	T_d
Р	T/L	∞	0
PI	0.9T/L	L/0.3	0
PID	1.2T/L = 1.569	2L	0.5L

Finally, as the Table 6 shown, the PID controller of open-loop ZN method is

$$G_c(s) = k_p \cdot \left(1 + \frac{1}{t_s s} + t_d s\right) = 1.569 \cdot (1 + 0.57869 \cdot s + \frac{1}{2.3148 \cdot s})$$
(19)

(b) Closed-loop Z-N method

Now, we utilize closed-loop Z-N method through following steps:

- 1. Set $k_d = k_i = 0$, hence, $G_c(s) = k_p$;
- 2. Then, find out $k_{p0} = k_{cr}$ (when system is marginally stable);
- 3. Lastly, using ZN table to obtain corresponding k_p , k_i , k_d .

The characteristic equation D(s) of the given system is defined by

$$D(s) = 1 + k_p \cdot OLTF = 0 \tag{20}$$

$$s^{3} + 14 \cdot s^{2} + (53 + 2.3 \cdot k_{p})s + (40 + 41.4 \cdot k_{p}) = 0$$

$$7/28$$
(21)

Table 7 Routh table

D(s)	$s^3 + 14 \cdot s^2 + (53 + 2.3 \cdot k_p)s + (40 + 41.4 \cdot k_p)$				
s^3	1	$53 + 2.3 \cdot k_p$			
s^2	14	$40 + 41.4 \cdot k_p$			
s^1	$(-702+9.2k_p)/(-14)$	/			
s^0	-14	/			

Table 7 illustrates the Routh table of the characteristic equation above. Now, we let

$$(-702 + 9.2k_p)/14 = 0 (22)$$

Hence, $k_{p0} = k_{cr} = 76.3$, then we consider that

$$14s^2 + 40 + 41.4 \cdot k_p = 0 \tag{23}$$

We can easily get the set of the related parameters

$$\begin{cases} s = 15.12j \\ \omega_c = 15.12 \text{ rad/sec} \\ t_c = \frac{2\pi}{\omega_c} = 0.4156 \text{ sec} \end{cases}$$
 (24)

Then, we can obtain k_p , k_i , k_d as follows,

$$\begin{cases} k_p = 0.6k_{cr} = 45.78 \\ k_i = \frac{k_p}{t_i} = \frac{k_p}{\frac{t_c}{2}} = 220.3 \\ k_d = k_p \cdot t_d = k_p \cdot (t_c/8) = 2.378 \end{cases}$$
 (25)

Table 8 Closed-loop ZN table

Controller	K_p	T_i	T_d
Р	$0.5k_{cr}$	∞	0
PI	$0.45k_{cr}$	$1.2P_{cr}$	0
PID	$0.6k_{cr} = 45.78$	0.5 <i>P_{cr}</i>	$0.125P_{cr}$

In this case, as the Table 8 shown, the PID controller of closed-loop ZN method is

$$G_c(s) = \frac{\left(k_d \cdot s^2 + k_p \cdot s + k_i\right)}{s} = 45.78 + 2.378s + \frac{220.3}{s}$$
 (26)

(c) Performance Index (ITAE) method

Utilizing ITAE method, we should follow these steps:

- 1. Determine the closed-loop transfer function;
- 2. Obtain the corresponding ω_c ;
- 3. Obtain the related parameters of optimum 4-order PID controller;
- 4. Determine the pre-filter P(s).

The transfer function of the open loop 3-order system is given,

$$G(s) = \frac{2.3s + 41.4}{s^3 + 14s^2 + 53s + 40} \tag{27}$$

The closed-loop transfer function is defined as:

$$G_{CL}(s) = \frac{G_c(s) \cdot G(s)}{1 + G_c(s) \cdot G(s)}$$
(28)

where

$$G_c(s) = \frac{\left(k_d \cdot s^2 + k_p \cdot s + k_i\right)}{s}$$
 (29)

Referring to the equation above, we can get the closed-loop transfer function:

$$G_{CL}(s) = \frac{2.3k_d s^3 + (2.3k_p + 41.4k_d)s^2 + (2.3k_i + 41.4k_p)s + 41.4k_i}{s^4 + (2.3k_d + 14)s^3 + (2.3k_p + 41.4k_d + 53)s^2 + (2.3k_i + 41.4k_p + 40)s + 41.4k_i}$$
(30)

Recalling that we have the performance specs

$$\begin{cases} \text{Settling time: } T_s = 0.9 \\ \text{Peak overshoot: } PO \leq 10\% \\ \text{Control signal limits: } -24 \leq u \leq 24 \end{cases} \tag{31}$$

which means,

$$\begin{cases} T_s = 0.9 = \frac{4}{\zeta \omega_n} \\ M_p = e^{-\left(\zeta/\sqrt{1-\zeta^2}\right)\pi} \le 10\% \end{cases}$$
 (32)

Then, we can obtain the suitable value that

$$\begin{cases} \zeta = 0.59121 \\ \omega_n = 7.5175 \end{cases} \tag{33}$$

According to the EE311FZ PID Control, the optimum 4-order system is given as

$$T(s) = \frac{\omega_n^4}{s^4 + 2.1\omega_n s^3 + 3.4\omega_n^2 s^2 + 2.7\omega_n^3 s + \omega_n^4}$$
(34)

Contrast Equation (30) with Equation (34), we can approximate that

$$\begin{cases} 2.1\omega_n \approx 2.3k_d + 14\\ 3.4\omega_n^2 = 2.3k_p + 41.4k_d + 53\\ 2.7\omega_n^3 = 2.3k_i + 41.4k_p + 40\\ \omega_n^4 = 41.4k_i \end{cases} \tag{35}$$

Now, easy to obtain

$$\begin{cases}
k_p = 22.455 \\
k_i = 77.142 \\
k_d = 2.1135
\end{cases}$$
(36)

Finally, we can design the related pre-filter P(s) as,

$$P(s) \cdot G_{CL}(s) = T(s) \tag{37}$$

$$P(s) = \frac{\omega_n^4}{2.3k_d s^3 + (2.3k_p + 41.4k_d)s^2 + (2.3k_i + 41.4k_p)s + 41.4k_i}$$
(38)

Where,

$$P(s) = \frac{3193.6973}{4.8611s^3 + 139.15s^2 + 1107.064s + 3193.6788}$$
(39)

(d) Manual method

Using manual method means that we need to reduce computational complexity as much as possible. In a nutshell, we can follow the widely accepted method below:

- 1. Similar to closed-loop ZN method, firstly, we set $k_i = k_d = 0$, then $G_c(s) = k_p$;
- 2. Increasing k_p , until the system becomes marginally stable;
- 3. Increasing k_i , until the output error comes to eliminated;
- 4. Set a suitable value of k_i to make the system becomes more stable and possess less oscillations;
- 5. Finally, fine-tune the parameters to make the system more compliant.

Firstly, let's set $k_i = k_d = 0$, and generally increase the value of k_p until the system gets into marginally stable. Then, we decrease k_p until the oscillation gets small enough. Repeat this process several times, finally, we have got a general value of $k_p = 22.9$.

The second step is to increase k_i , until the output error comes to eliminated, which means the steady output equal to 1. In this step, we should be careful to the response time (t_r, t_s) of the system and make them as lower as possible. At the end, we make the value of $k_i = 109$.

Third, considering the k_d , we can make the system more stable and realizable, through utilizing the differentiation element of PID controller. Now, we fine-tune k_d to decrease the oscillations. Finally, we get a value of $k_d = 3.20$. Now, we have got all of the three parameters of PID controller as follows:

$$\begin{cases}
k_p = 22.9 \\
k_i = 109 \\
k_d = 3.20
\end{cases}$$
(40)

But we have not finished yet. Now, we need to simulate the system using **MATLAB Simulink**, to see whether the system is stable (Figure 4). Unfortunately, at this moment, although the response time and steady-state error have been good, the system overshoot is too large, which may lead to unsatisfactory impacts.

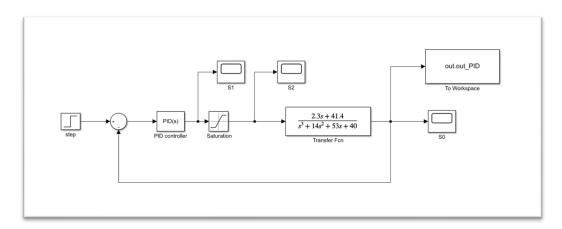


Figure 4 Simulink diagram of PID controller

In this case, according to the Ref. [4], we can try to decrease the value of k_i , while increases k_p and k_d (sightly). Through a hundred times of fine tuning. Ultimately, we have obtained a satisfactory set of PID parameters as follows:

$$\begin{cases}
k_p = 32.8 \\
k_i = 62.9 \\
k_d = 4.32
\end{cases}$$
(41)

Therefore, the corresponding PID controller is

$$G_c(s) = \frac{\left(k_d \cdot s^2 + k_p \cdot s + k_i\right)}{s} = 32.8 + 4.32s + \frac{62.9}{s}$$
(42)

B. State Feedback Controller Design

(a) Controllability and Observability

The transfer function of the 3-order system is given:

$$G(s) = \frac{2.3s + 41.4}{s^3 + 14s^2 + 53s + 40} \tag{43}$$

And we have obtained the corresponding state-space function:

$$y = \begin{bmatrix} 41.4 & 2.3 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}$$
 (45)

which means that

$$\begin{cases}
A = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ -40 & -53 & -14 \end{bmatrix} & B = \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} \\
C = \begin{bmatrix} 41.4 & 2.3 & 0 \end{bmatrix} & D = 0
\end{cases}$$
(46)

Now, to design the State Observer, firstly we need to make sure the given system is observable and controllable. According to the Ref. [1], we calculate that:

$$\begin{cases}
M = [B AB A^{2}B] = \begin{bmatrix}
0 & 0 & 1 \\
0 & 1 & -14 \\
1 & -14 & 143
\end{bmatrix} \\
N = \begin{bmatrix}
C \\
CA \\
CA^{2}
\end{bmatrix} = \begin{bmatrix}
41.4 & 2.3 & 0 \\
0 & 41.4 & 2.3 \\
-92.0 & -121.9 & 9.2
\end{bmatrix}$$
(47)

Using Octave online, we can easily determine that

$$\begin{cases} rank(M) = 3 \\ rank(N) = 3 \end{cases}$$
(48)

Because det(M) = 3 and rank(N) = 3, thus the system is controllable and observable.

(b) State Observer Design

According to Ref.^[1], the mathematical model of the observer is defined as

$$\dot{\tilde{\mathbf{x}}} = \mathbf{A}\tilde{\mathbf{x}} + \mathbf{B}u + \mathbf{K}_e(y - \mathbf{C}\tilde{\mathbf{x}})
= (\mathbf{A} - \mathbf{K}_e\mathbf{C})\tilde{\mathbf{x}} + \mathbf{B}u + \mathbf{K}_ey$$
(49)

In the equation above, where $\tilde{\mathbf{x}}$ is the estimated state and $\mathbf{C}\tilde{\mathbf{x}}$ is the estimated output. The inputs to the observer are the output y and the control input u. Matrix \mathbf{K}_e , which is called the observer gain matrix, is a weighting matrix to the correction term involving the difference between the measured output y and the estimated output $\mathbf{C}\tilde{\mathbf{x}}$. Also, the observer error equation is defined as:

$$\dot{\mathbf{x}} - \dot{\tilde{\mathbf{x}}} = \mathbf{A}\mathbf{x} - \mathbf{A}\tilde{\mathbf{x}} - \mathbf{K}_e(\mathbf{C}\mathbf{x} - \mathbf{C}\tilde{\mathbf{x}})$$

$$= (\mathbf{A} - \mathbf{K}_e\mathbf{C}) \cdot (\mathbf{x} - \tilde{\mathbf{x}})$$
(50)

Then, defined the difference between $\dot{\mathbf{x}}$ and $\dot{\mathbf{x}}$ as the error vector \mathbf{e} , so

$$\begin{cases} \mathbf{e} = \mathbf{x} - \tilde{\mathbf{x}} \\ \dot{\mathbf{e}} = (\mathbf{A} - \mathbf{K}_e \mathbf{C}) \mathbf{e} \end{cases}$$
 (51)

We see that the dynamic behavior of the error vector is determined by the eigenvalues of matrix $\mathbf{A} - \mathbf{K}_e \mathbf{C}$. That is, if matrix $\mathbf{A} - \mathbf{K}_e \mathbf{C}$ is a stable matrix, the error vector will converge to zero for any initial error vector $\mathbf{e}(0)$. So far, we have obtained the value of \mathbf{A} , \mathbf{B} , \mathbf{C} . Therefore, the problem of designing a full-order observer becomes that of determining the observer gain matrix \mathbf{K}_e , such that the error dynamics \mathbf{e} defined by Equation above are asymptotically stable with sufficient speed of response.

Overall, the design of the full-order observer becomes that of determining an appropriate K_e such that $A - K_eC$ has desired eigenvalues.

Recall that the closed-loop transfer function of the given system is

$$G_{CL}(s) = \frac{G(s)}{1 + G(s)} = \frac{2.3s + 41.4}{s^3 + 14s^2 + 55.3s + 81.4}$$
 (52)

Figure 3 shows the Pole-zero map of $G_{CL}(s)$. And the poles of $G_{CL}(s)$ is given as:

$$\begin{cases}
p_1 = -8.7369 \\
p_2 = -2.6316 + 1.5465 j \\
p_3 = -2.6316 - 1.5465 j
\end{cases}$$
(53)

As shown in the following Figure 5, the two complex poles p_2 , p_3 are the predominant poles while the real pole p_1 will be the non-dominant pole

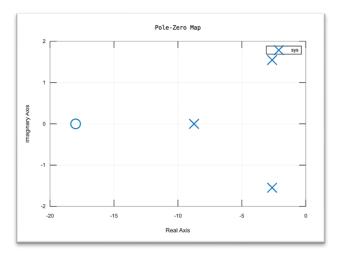


Figure 5 Pole-zero map of $G_{CL}(s)$

In this case, the closed-loop transfer function $G_{CL}(s)$ can be simplified to:

$$G_{CL}(s) \approx \frac{2.3s + 41.4}{(s + 2.6316 - 1.5465j)(s + 2.6316 + 1.5465j)}$$
 (54)

Recall the standard 2-order transfer function is defined as

$$G_{std2}(s) = \frac{K\omega n^2}{\left(s + \xi\omega n - \omega n\sqrt{\xi^2 - 1}\right)\left(s + \xi\omega n + \omega n\sqrt{\xi^2 - 1}\right)}$$

$$G_{std2}(s) = \frac{K\omega n^2}{s^2 + 2\xi\omega n + \omega n^2}$$
(55)

Comparing the Equations above, we can get

$$\begin{cases}
\zeta = 0.8621 \\
\omega_n = 3.0525
\end{cases}$$
(56)

Before pole placement, the system possesses

$$\begin{cases} T_s = \frac{4}{\zeta \omega_n} = 1.5200 \text{ s} \\ M_p = e^{-\left(\zeta/\sqrt{1-\zeta^2}\right)\pi} \le 10\% \end{cases}$$
 (57)

However, at this moment, the settling time does not meet $T_s \le 0.9$. Thus, the system needs to conduct pole placement. Through analyzing the design specs above, we can obtain the desired placement as follows:

$$\begin{cases}
p_1 = -4.4446 + 6.0635 \, j \\
p_2 = -4.4446 - 6.0635 \, j
\end{cases}$$
(58)

According to Ref. [1, 5], the desired poles for observer should be twice as the poles of closed-loop system. Therefore, the poles placement for the observer should be

$$\begin{cases} p_{o_1} = -8.8892 + 12.127 j \\ p_{o_2} = -8.8892 - 12.127 j \end{cases}$$
(59)

Once we get the desired poles for the observer, we can obtain the \mathbf{K}_e through MATLAB program, as shown in the following Table 9.

$$\mathbf{K}_{e} = \begin{bmatrix} 0.0155 \\ 5.1623 \\ -26.314 \end{bmatrix} \tag{60}$$

```
Table 9: MATLAB Program 4

This program is used to obtain the observer gain matrix.

%% Q4-(b) State Obverser Design
clear; clc;
A = [0 1 0;0 0 1;-40 -53 -14];
B = [0;0;1];
C = [41.4 2.3 0];
% pole1 = [-4.4446 + 6.0635j,-4.4446-6.0635j,-8.7369];
% K = acker(A',B,pole1)
pole2 = [-8.8892 + 12.127j,-8.8892-12.127j,-8.7369];
Ke = acker(A',C',pole2)
```

Recall the mathematical model of the observer,

$$\dot{\tilde{\mathbf{x}}} = \mathbf{A}\tilde{\mathbf{x}} + \mathbf{B}u + \mathbf{K}_e(y - \mathbf{C}\tilde{\mathbf{x}})
= (\mathbf{A} - \mathbf{K}_e\mathbf{C})\tilde{\mathbf{x}} + \mathbf{B}u + \mathbf{K}_ey$$
(61)

Now we have already got all of the parameters, let's rearrange the Equation as follows

$$\begin{bmatrix} \dot{\tilde{x}} \\ \ddot{\tilde{x}} \\ \ddot{\tilde{x}} \end{bmatrix} = \begin{bmatrix} -0.642 & 0.964 & 0 \\ -213.719 & -11.873 & 1 \\ 1049.404 & 7.522 & -14 \end{bmatrix} \begin{bmatrix} \widetilde{x_1} \\ \widetilde{x_2} \\ \widetilde{x_3} \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} u + \begin{bmatrix} 0.0155 \\ 5.1623 \\ -26.314 \end{bmatrix} y$$
(62)

(c) Feedback Controller Design

After implementing the observer, the next step is to design the feedback controller. Referring to Ref.^[1] (heavily), we shall use the following design procedure:

- 1. Derive a state-space model of the plant;
- 2. Choose the desired closed-loop poles for pole placement. Choose the desired observer poles;
- 3. Determine the state feedback gain matrix \mathbf{K} and the observer gain matrix \mathbf{K}_e .
- 4. Using the gain matrices \mathbf{K} and \mathbf{K}_e obtained in step 3, derive the transfer function of the observer controller. If it is a stable controller, check the response to the given initial condition. If the response is not acceptable, adjust the closed-loop pole location and/or observer pole location until an acceptable response is obtained.

Based on the Procedure above, we have already obtained the desired gain matrix. Now we need to adjust the closed-loop pole location or observer pole location. Also, we utilize the **MATLAB Simulink** for system simulation, as shown in Figure 6.

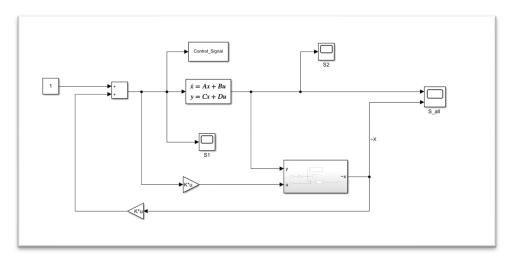


Figure 6 Simulation of the system

As for the pole placement, firstly we need to obtain the value of initial point x_i and u_i , where the expected value would be

$$\begin{cases} \mathbf{A}x_i + \mathbf{B}u_i = 0 \\ \mathbf{C}x_i = r \end{cases} \tag{63}$$

which is

$$\begin{bmatrix} x_{SS} \\ u_{SS} \end{bmatrix} = \begin{bmatrix} A & B \\ C & D \end{bmatrix}^{-1} \begin{bmatrix} 0 \\ r \end{bmatrix}$$
 (64)

Substituting the matrix above, we get that

$$\begin{bmatrix} x_i \\ u_i \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 & 0 \\ 0 & 0 & 1 & 0 \\ -40 & -53 & -14 & 1 \\ 41.4 & 2.3 & 0 & 0 \end{bmatrix}^{-1} \begin{bmatrix} 0 \\ 0 \\ 0 \\ 1 \end{bmatrix} = \begin{bmatrix} 0.024 \\ 0 \\ 0 \\ 0.966 \end{bmatrix}$$
(65)

So, we can know that

$$\begin{cases} x_i = 0.024 \cdot x \\ u_i = 0.966 \cdot u \end{cases} \tag{66}$$

Now, we can utilize the MATLAB function to obtain the state feedback gain matrix **K**, as shown in the following Table 10.

$$\mathbf{K} = \begin{bmatrix} 13.003 \\ -11.345 \\ 0 \end{bmatrix} \tag{67}$$

```
Table 10: MATLAB Program 5

This program is used to obtain the state feedback gain matrix.

%% Q4-(c) Feedback Controller Design
clear; clc;
A = [0 1 0;0 0 1;-40 -53 -14];
B = [0;0;1];
C = [41.4 2.3 0];
pole1 = [-4.4446 + 6.0635j,-4.4446-6.0635j,-8.7369];
K = acker(A',B,pole1)
pole2 = [-8.8892 + 12.127j,-8.8892-12.127j,-8.7369];
Ke = acker(A',C',pole2)
```

Then, through a hundred times of fine tuning. Ultimately, we obtained a satisfactory value of \mathbf{K}_e and \mathbf{K} as follows:

$$\begin{cases}
\mathbf{K}_{e} = \begin{bmatrix}
0.0155 \\
5.1623 \\
-26.314
\end{bmatrix} \\
\mathbf{K} = \begin{bmatrix}
-1.5020 \\
-11.345 \\
0
\end{bmatrix}$$
(68)

Now, we have finished all the designs of the controllers based on various methods. In the next section, we will compare the pros and cons of each method in detail.

Procedure 5-1 — Time-domain Simulink Simulation

In this section, we will simulate all the systems we design in Procedure 4 above.

(a) PID Controller

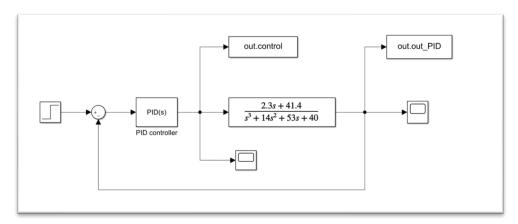


Figure 7 Block diagram for PID controller

(b) PID Controller with control signal limit

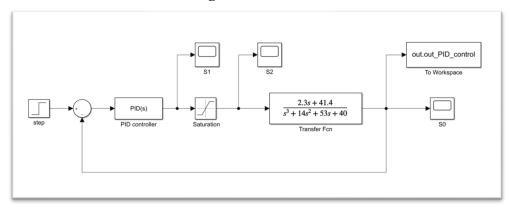


Figure 8 Block diagram for PID controller (control signal limit)

(c) ITAE-PID Controller

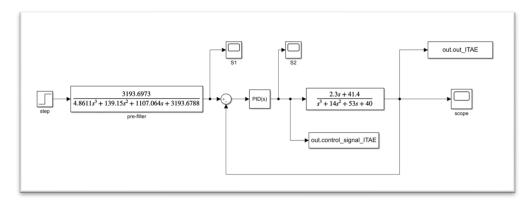


Figure 9 Block diagram for ITAE-PID controller

(d) ITAE-PID Controller with control signal limit

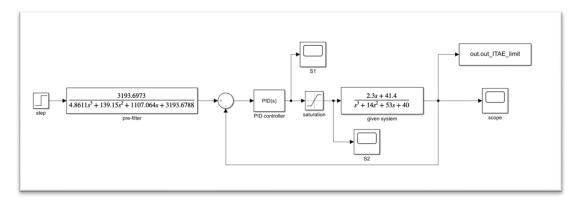


Figure 10 Block diagram for ITAE-PID controller (control signal limit)

(e) State-space Feedback Controller

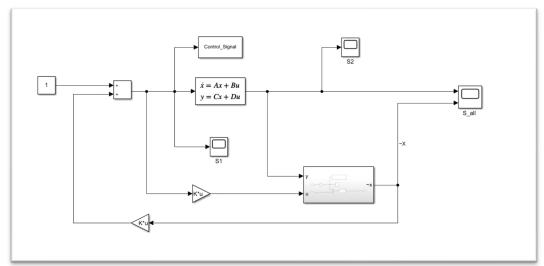


Figure 11 Block diagram for the State-space feedback controller

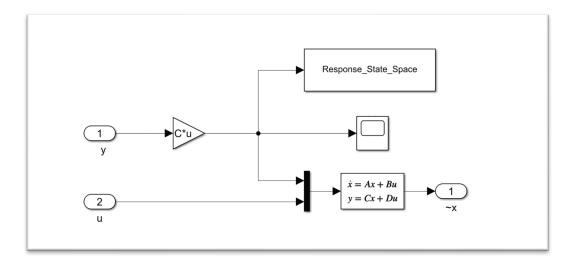


Figure 12 Subsystem of the state-space feedback controller (State-space Observer)

(f) State-space Feedback Controller with control signal limit

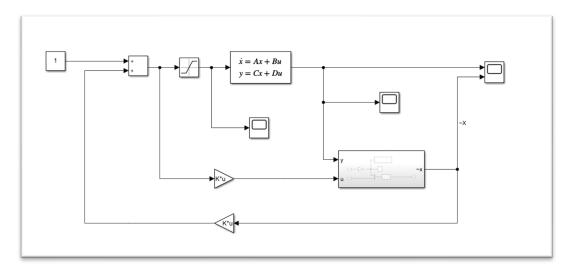


Figure 13 Block diagram for the State-space feedback controller (control signal limit)

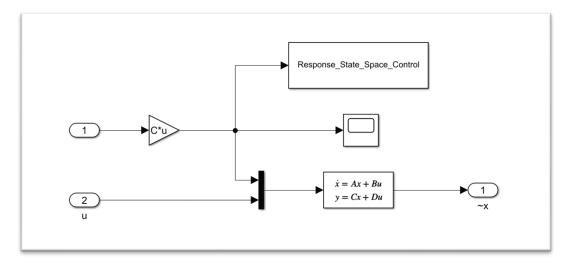


Figure 14 Subsystem of the state-space feedback controller (State-space Observer)

Procedure 5-2 — Time-domain Simulink Simulation

(1) PID Controller: open-loop ZN method

Recall that the PID controller of open-loop ZN method is

$$G_c(s) = 1.569 \cdot (1 + 0.57869 \cdot s + \frac{1}{2.3148 \cdot s})$$
 (69)

Figure 15 shows the output response of the system.

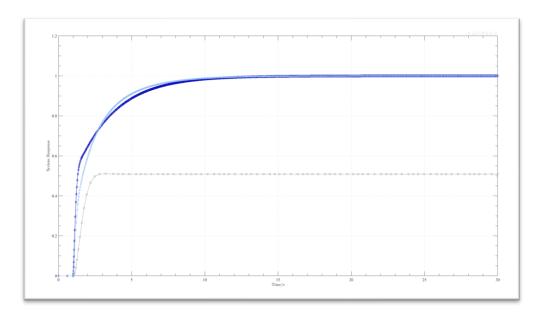


Figure 15 System output response (open-loop ZN method)

And Table 11 illustrates the performance specs of the system above.

Table 11 Performance specs

System	T_S	M_p	u_{max}	e_{ss}
Without limit	9.4568s	0%	/	0
With limit	8.6679s	0%	92.4	0

(2) PID Controller: closed-loop ZN method

Recall that the PID controller of open-loop ZN method is

$$G_c(s) = 45.78 + 2.378s + \frac{220.3}{s} \tag{70}$$

Figure 16 shows the output response of the system.

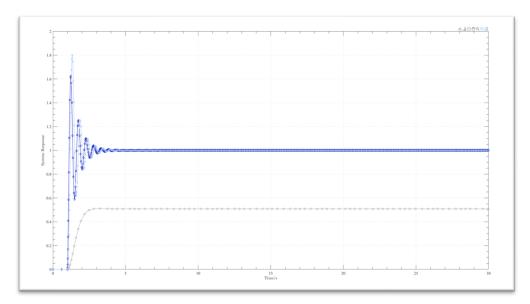


Figure 16 System output response (closed-loop ZN method)

And Table 12 illustrates the performance specs of the system above.

Table 12 Performance specs

System	T_{s}	M_p	u_{max}	$e_{\scriptscriptstyle SS}$
Without limit	3.1278s	62.9%	/	0
With limit	3.2601s	79.8%	283.3	0

(3) PID Controller: ITAE method

Recall that the PID controller of ITAE method is

$$G_c(s) = 22.455 + 2.1135s + \frac{77.142}{s}$$
 (71)

Figure 17 shows the output response of the system.

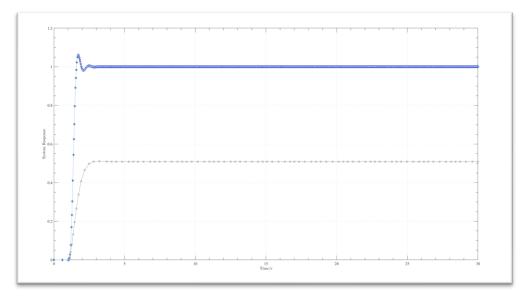


Figure 17 System output response (ITAE method)

And Table 13 illustrates the performance specs of the system above.

Table 13 Performance specs

System	T_{S}	M_p	u_{max}	$e_{\scriptscriptstyle SS}$
Without limit	1.9006s	6.14%	/	0
With limit	1.0906s	6.14%	6.013	0

(4) PID Controller: mutual method

Recall that the PID controller of ITAE method is

$$G_c(s) = 32.8 + 4.32s + \frac{62.9}{s}$$
 (72)

Figure 18 shows the output response of the system.

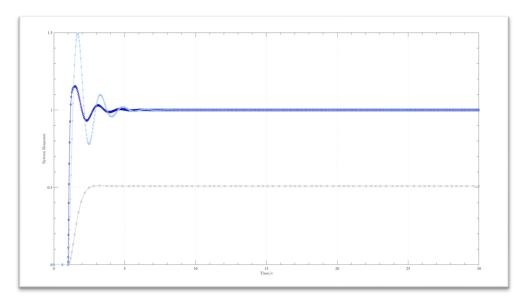


Figure 18 System output response (mutual method)

And Table 14 illustrates the performance specs of the system above.

Table 14 Performance specs

System	T_{S}	M_p	u_{max}	$e_{\scriptscriptstyle SS}$
Without limit	3.3485s	19.24%	/	0
With limit	4.3940s	49.21%	429.1	0

(5) State-space: Feedback Controller

Recall the gain matrices of the state feedback controller system is

$$\begin{cases}
\mathbf{K}_{e} = \begin{bmatrix}
0.0155 \\
5.1623 \\
-26.314
\end{bmatrix} \\
\mathbf{K} = \begin{bmatrix}
-1.5020 \\
-11.345 \\
0
\end{bmatrix}$$
(73)

Figure 19 shows the output response of the system above.

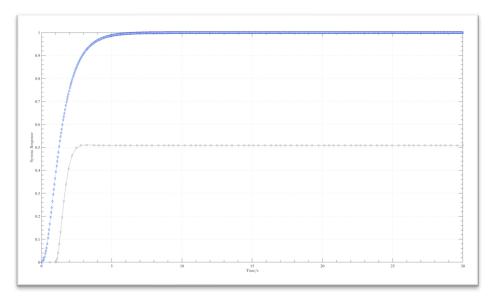


Figure 19 System output response (State feedback controller)

And Table 15 illustrates the performance specs of the system above.

Table 15 Performance specs

System	$T_{\mathcal{S}}$	M_p	u_{max}	$e_{\scriptscriptstyle SS}$
Without limit	4.4953s	0%	/	0.0003%
With limit	4.4953s	0%	1.002	0.0003%

So far, we have presented all the performance specs of the controller we designed. In the following section, we will summarize and compare all the methodologies.

Procedure 6 — Comparison of Methods

Congratulations! We have come a long way to get here. Now, we will compare the system performance of all the controllers we design above. The following Table 16 and Table 17 illustrate the performance specs of various methods. And the Table 18 shows the summary for the control signal limit of the different methods.

Table 16 Summary of various design methodologies (without control signal limit)

Without limit	T_r	T_{s}	M_p	$e_{\scriptscriptstyle SS}$
Open loop ZN	4.4358s	9.4568s	0%	0%
Closed loop ZN	0.6065s	3.1278s	62.9%	0%
ITAE	0.3321s	1.9006s	6.14%	0%
Manual	0.1349s	3.3485s	14.24%	0%
State Feedback	2.4987s	4.4953s	0%	0.00037%

Table 17 Summary of various design methodologies (with control signal limit)

Without limit	T_r	T_s	M_p	$e_{\scriptscriptstyle SS}$
Open loop ZN	3.5867s	8.6689s	0%	0%
Closed loop ZN	0.1408s	3.2601s	79.8%	0%
ITAE	0.3321s	1.9006s	6.14%	0%
Manual	0.2502s	4.3940s	49.12%	0%
State Feedback	2.4987s	4.4953s	0%	0.00037%

Table 18 Summary for the control signal limit of the different methods

Methods	Open-loop ZN	Closed- loop ZN	ITAE	Manual	State Feedback
u_{max}	92.4	283.3	6.013	397.1	1.002

Procedure 7 — Brief Conclusion

As compared in the Table 16-18 above, we can draw the conclusion as follows:

- 1. As for rise time t_r , ITAE controller performs better than any other controller. And the manual and closed-loop controllers share the rank 2^{nd} . The state feedback and open-loop controller do not perform well.
- 2. As for settling time t_s , ITAE controller outperform other controller, while open-loop controller possesses an unsatisfactory performance. And the other controller performs similarly.
- 3. Considering maximum overshoot M_P , obviously, open-loop controller and state feedback controller have the best performance. ITAE controller also perform well. However, the closed-loop and manual controller would not be the suitable option.
- 4. In the domain of steady state error e_{ss} , all five controllers perform satisfactorily.

As shown in the following Table 19, all the methodologies have been summed up.

Table 19 Summar	afa-iaa		
I anie 10 Silmmar	v ot various ('Antraller deglo	n methodologies
Table 19 Summa	y or various v	controller design	ii iiicuiouoiogico

Methodology	T_r	T_{s}	M_p	e_{ss}
Open loop ZN	5th	5th	1st √	1st ✓
Closed loop ZN	3rd	2nd	5th	1st ✓
ITAE (SOTA)	1st ✓	1st ✓	3rd	1st ✓
ITAE (SOTA) Manual	1st ✓ 2nd	1st √ 2nd	3rd 4th	1st √ 1st √

Acknowledgements

At the end, I gratefully acknowledge Dr. Siyuan Zhan for his generous guidance and support during this semester. And I hope to thank Mr. Jian Lang for his helpful suggestions. Also, I would like to thank the reviewer who carefully evaluates this report. Best wishes!

Hanlin CAI 20th Dec. 2022

References

- [1] Ogata K. Modern control engineering [M]. Prentice hall Upper Saddle River, NJ, 2010.
- [2] CONTROL CHART by Northwest Analytics Inc. Available online URL: https://asq.org/quality-resources/control-chart [Z]. 2022
- [3] Control Limits Vs. Specification Limits by Northwest Analytics Inc. Available online URL: https://www.nwasoft.com/resources/information-center/article/control-limits-vs-specification-limits [Z]. 2022
- [4] How to Tune a PID Controller. Available online URL: https://pidexplained.com/how-to-tune-a-pid-controller/ [Z]. 2022
- [5] Pole Placement Method. Available online URL: https://www.sciencedirect.com/topics/engineering/pole-placement-method [Z]. 2013