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## 6. Solving the state equations

### 6.1 Discrete-time solution

- The general linear discrete-time state-space model is given by:

$$\begin{aligned}\mathbf{x}(k+1) &= \mathbf{A}\mathbf{x}(k) + \mathbf{B}u(k) \\ \mathbf{y}(k) &= \mathbf{C}\mathbf{x}(k) + \mathbf{D}u(k)\end{aligned}$$

- Here, we will restrict our attention to single-input single-output (SISO) systems, i.e. the input and output are scalar quantities:

$$\begin{aligned}\mathbf{x}(k+1) &= \mathbf{A}\mathbf{x}(k) + \mathbf{B}u(k) \\ y(k) &= \mathbf{C}\mathbf{x}(k) + \mathbf{D}u(k)\end{aligned}$$

where:

$$\mathbf{A} \in R^{n \times n}, \quad \mathbf{B} \in R^{n \times 1}, \quad \mathbf{C} \in R^{1 \times n}, \quad \mathbf{D} \in R^{1 \times 1}, \quad \mathbf{x}(k) \in R^n, \quad u(k), y(k) \in R^1$$

- The dynamics of the system are described by the state equation.
- Just like other difference/differential equations, it is solved by considering the unforced and forced responses separately.

#### Unforced response:

- This is where the input is set to zero for all time, i.e.:

$$u(k) = 0, \quad \forall k$$

- The state equation then becomes:

$$\mathbf{x}(k+1) = \mathbf{A}\mathbf{x}(k)$$

- Given an initial condition  $\mathbf{x}(0)$  we can calculate the response of the system as follows:

$$\begin{aligned}\mathbf{x}(1) &= \mathbf{A}\mathbf{x}(0) \\ \mathbf{x}(2) &= \mathbf{A}\mathbf{x}(1) = \mathbf{A}^2\mathbf{x}(0) \\ \mathbf{x}(3) &= \mathbf{A}\mathbf{x}(2) = \mathbf{A}^3\mathbf{x}(0) \\ &\text{etc...}\end{aligned}$$

- Therefore, in general:  $\mathbf{x}(k) = \mathbf{A}^k \mathbf{x}(0)$
- The matrix  $\mathbf{A}^k$  is known as the (*discrete-time*) **state transition matrix**.

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### Forced response:

- Now, consider the situation when the input is not zero:

$$\mathbf{x}(1) = \mathbf{A}\mathbf{x}(0) + \mathbf{B}u(0)$$

$$\begin{aligned}\mathbf{x}(2) &= \mathbf{A}\mathbf{x}(1) + \mathbf{B}u(1) \\ &= \mathbf{A}(\mathbf{A}\mathbf{x}(0) + \mathbf{B}u(0)) + \mathbf{B}u(1) \\ &= \mathbf{A}^2\mathbf{x}(0) + \mathbf{A}\mathbf{B}u(0) + \mathbf{B}u(1)\end{aligned}$$

$$\begin{aligned}\mathbf{x}(3) &= \mathbf{A}\mathbf{x}(2) + \mathbf{B}u(2) \\ &= \mathbf{A}(\mathbf{A}^2\mathbf{x}(0) + \mathbf{A}\mathbf{B}u(0) + \mathbf{B}u(1)) + \mathbf{B}u(2) \\ &= \mathbf{A}^3\mathbf{x}(0) + \mathbf{A}^2\mathbf{B}u(0) + \mathbf{A}\mathbf{B}u(1) + \mathbf{B}u(2)\end{aligned}$$

etc...

- In general, we get:

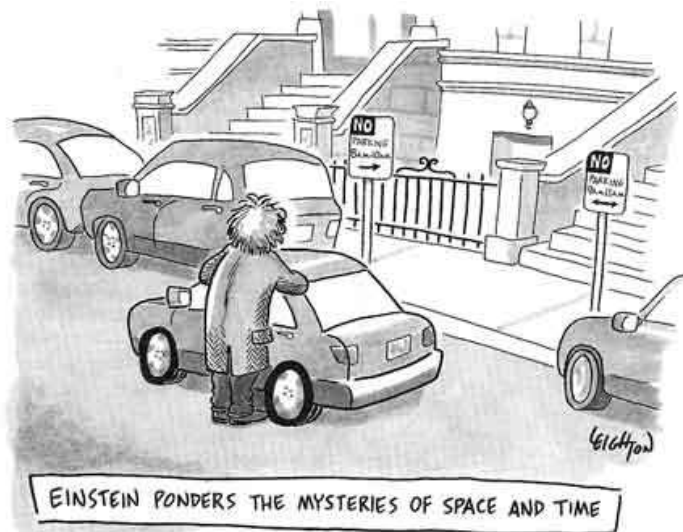
$$\begin{aligned}\mathbf{x}(k) &= \mathbf{A}^k\mathbf{x}(0) + \mathbf{A}^{k-1}\mathbf{B}u(0) + \mathbf{A}^{k-2}\mathbf{B}u(1) + \dots + \mathbf{A}^0\mathbf{B}u(k-1) \\ &= \underbrace{\mathbf{A}^k\mathbf{x}(0)}_{\text{unforced response}} + \underbrace{\sum_{i=1}^k \mathbf{A}^{k-i}\mathbf{B}u(i-1)}_{\text{forced response when } \mathbf{x}(0) = 0}\end{aligned}$$

### Output calculation:

- Once the state has been determined, the output is easily computed as:

$$\begin{aligned}y(k) &= \mathbf{C}\mathbf{x}(k) + \mathbf{D}u(k) \\ &= \mathbf{C}\mathbf{A}^k\mathbf{x}(0) + \sum_{i=1}^k (\mathbf{C}\mathbf{A}^{k-i}\mathbf{B}u(i-1)) + \mathbf{D}u(k)\end{aligned}$$

- Note, that  $\mathbf{D}$  is usually zero.



- **Example 6.1:** Determine the output  $y(k)$  for the system:

$$\mathbf{A} = \begin{bmatrix} 0.5 & 0 \\ 0.25 & 0.25 \end{bmatrix}, \quad \mathbf{B} = \begin{bmatrix} 1 \\ 1 \end{bmatrix}, \quad \mathbf{C} = [0.5 \quad 0], \quad \mathbf{D} = [0]$$

given that the input is  $u(k) = \delta(k)$  and  $\mathbf{x}(0) = [0 \ 0]^T$ .

**Solution:**

$$y(k) = [0.5 \quad 0] \begin{bmatrix} 0.5 & 0 \\ 0.25 & 0.25 \end{bmatrix}^k \begin{bmatrix} 0 \\ 0 \end{bmatrix} + \sum_{i=1}^k [0.5 \quad 0] \begin{bmatrix} 0.5 & 0 \\ 0.25 & 0.25 \end{bmatrix}^{k-i} \begin{bmatrix} 1 \\ 1 \end{bmatrix} \delta(i-1)$$

$\delta(i-1) = 1$  when  $i = 1$   
 $= 0 \ \forall \ i \neq 1$

$$\Rightarrow y(k) = [0.5 \quad 0] \begin{bmatrix} 0.5 & 0 \\ 0.25 & 0.25 \end{bmatrix}^{k-1} \begin{bmatrix} 1 \\ 1 \end{bmatrix} = \mathbf{C} \mathbf{A}^{k-1} \mathbf{B}$$

$$y(1) = [0.5 \quad 0] \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} \begin{bmatrix} 1 \\ 1 \end{bmatrix} = 0.5$$

$$y(2) = [0.5 \quad 0] \begin{bmatrix} 0.5 & 0 \\ 0.25 & 0.25 \end{bmatrix} \begin{bmatrix} 1 \\ 1 \end{bmatrix} = [0.5 \quad 0] \begin{bmatrix} 0.5 \\ 0.5 \end{bmatrix} = 0.25$$

$$y(3) = [0.5 \quad 0] \begin{bmatrix} 0.5 & 0 \\ 0.25 & 0.25 \end{bmatrix} \begin{bmatrix} 0.5 & 0 \\ 0.25 & 0.25 \end{bmatrix} \begin{bmatrix} 1 \\ 1 \end{bmatrix} = [0.5 \quad 0] \begin{bmatrix} 0.25 \\ 0.25 \end{bmatrix} = 0.125$$

In general:

$$y(k) = [0.5 \quad 0] \begin{bmatrix} 0.5^{k-1} \\ 0.5^{k-1} \end{bmatrix} = 0.5(0.5)^{k-1} = 0.5^k$$

$\swarrow$   $\mathbf{C}$                        $\nwarrow$   $\mathbf{A}^{k-1} \mathbf{B} = \mathbf{x}(k)$

- In this example we have succeeded in obtaining the solution of  $y(k)$  (and  $\mathbf{x}(k)$ ) in **closed form**.
- This implies that the solution is in terms of  $k$  only and, as a result, it is easy to work out answers for any value of  $k$ .

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## Notes:

- It is worth noting that we are fortunate in being able to get the general ‘*closed-form solution*’ for the elements of the state vector.
- The special case of zero initial conditions and (unit) impulse input helps to simplify things considerably.
- More computationally attractive methods are needed for calculating the transition matrix for general problems.
- We would like a solution that is easy to implement using computers.
- Two popular methods exist – one uses canonical state transformations (to diagonalise  $\mathbf{A}$ ), the other uses the Cayley-Hamilton theorem.
- In this module, we will only consider the method of state transformation (and, in particular, using the modal matrix).
- *Aside – the Cayley-Hamilton theorem states that an  $n \times n$  matrix  $\mathbf{A}$  satisfies its own characteristic equation:*

$$p(\lambda) = |\lambda \mathbf{I} - \mathbf{A}| = \lambda^n + a_{n-1}\lambda^{n-1} + \dots + a_1\lambda + a_0 = 0$$

*In other words, if we replace  $\lambda$  with  $\mathbf{A}$ , then  $p(\mathbf{A}) = 0$ .*

*This theorem is the basis of an alternative method used to calculate the state transition matrix – please refer to a suitable text book for detail.*



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## 6.2 State transformations

- State-space representations are not unique. There are many selections of state variables which can describe a system.
- State variables can be real or fictitious.
- Some state representations lead to computationally attractive forms.
- Thus, by using the appropriate state transformation, we can obtain a state-space representation that leads to significantly easier computation.
- Consider the following state transformation:

$$\mathbf{x}(k) = \mathbf{T}\mathbf{z}(k)$$

- Here,  $\mathbf{T}$  is any constant non-singular  $n \times n$  matrix. Since  $\mathbf{T}$  is constant, we can write:

$$\mathbf{x}(k+1) = \mathbf{T}\mathbf{z}(k+1)$$

- Substituting these expressions into the state equation  $\mathbf{x}(k+1) = \mathbf{A}\mathbf{x}(k) + \mathbf{B}u(k)$  gives:

$$\mathbf{T}\mathbf{z}(k+1) = \mathbf{A}\mathbf{T}\mathbf{z}(k) + \mathbf{B}u(k)$$

- Finally, premultiplying by  $\mathbf{T}^{-1}$  gives us the new state equation:

$$\begin{aligned}\mathbf{z}(k+1) &= \mathbf{T}^{-1}\mathbf{A}\mathbf{T}\mathbf{z}(k) + \mathbf{T}^{-1}\mathbf{B}u(k) \\ &= \mathbf{A}_z\mathbf{z}(k) + \mathbf{B}_zu(k)\end{aligned}$$

- The new output equation is:

$$\begin{aligned}y(k) &= \mathbf{C}\mathbf{T}\mathbf{z}(k) + \mathbf{D}u(k) \\ &= \mathbf{C}_z\mathbf{z}(k) + \mathbf{D}u(k)\end{aligned}$$

- Note – the only condition on  $\mathbf{T}$  is that it must be non-singular (i.e. invertible).
- Hence, there are an infinite number of state representations for the system.
- **We are interested in finding a representation that leads to a diagonal  $\mathbf{A}$  matrix** as this will make our computation significantly easier.
- So the issue becomes one of finding a suitable  $\mathbf{T}$  so that  $\mathbf{A}_z = \mathbf{T}^{-1}\mathbf{A}\mathbf{T}$  is diagonal.

- Now consider a matrix  $A$  with distinct eigenvalues  $\lambda_1, \lambda_2, \dots, \lambda_n$  and let  $\mathbf{m}_1, \mathbf{m}_2, \dots, \mathbf{m}_n$  be the corresponding eigenvectors.

- The **modal matrix**,  $M$ , is formed from these eigenvectors:  $M = [\mathbf{m}_1 \quad \mathbf{m}_2 \quad \dots \quad \mathbf{m}_n]$

- Note that by definition,  $A\mathbf{m}_i = \lambda_i\mathbf{m}_i$ .

- Hence:

$$\begin{aligned} \mathbf{AM} &= A[\mathbf{m}_1 \quad \mathbf{m}_2 \quad \dots \quad \mathbf{m}_n] \\ &= [A\mathbf{m}_1 \quad A\mathbf{m}_2 \quad \dots \quad A\mathbf{m}_n] \\ &= [\lambda_1\mathbf{m}_1 \quad \lambda_2\mathbf{m}_2 \quad \dots \quad \lambda_n\mathbf{m}_n] \\ &= M\Lambda \end{aligned}$$

where:

$$\Lambda = \begin{bmatrix} \lambda_1 & 0 & \dots & 0 \\ 0 & \lambda_2 & \dots & 0 \\ \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & \dots & \lambda_n \end{bmatrix}$$

- Since:  $\mathbf{AM} = M\Lambda \Rightarrow \Lambda = M^{-1}\mathbf{AM}$
- Thus, we can conclude that in order to obtain a diagonal state matrix, **we choose the modal matrix  $M$  as the state transformation matrix.**
- A state transformation by the modal matrix produces a set of  $n$  independent first order difference equations:

$$\begin{aligned} z_1(k+1) &= \lambda_1 z_1(k) + b_1 u(k) \\ &\vdots \\ z_n(k+1) &= \lambda_n z_n(k) + b_n u(k) \end{aligned}$$

### 6.2.1 Determining the modal matrix

- Recall that eigenvalues are calculated as the roots of the **matrix characteristic equation**:

$$\det(\lambda I - A) = |\lambda I - A| = 0$$

- The eigenvectors are then determined either by:
  - solving  $A\mathbf{m}_i = \lambda_i \mathbf{m}_i$
  - or
  - evaluating the cofactors of a row of  $(\lambda I - A)$

- **Example 6.2:** Determine the modal matrix for  $\mathbf{A} = \begin{bmatrix} -1 & 2 \\ -1 & -4 \end{bmatrix}$

**Solution:** Eigenvalues:

$$\begin{aligned}
 |\lambda \mathbf{I} - \mathbf{A}| = 0 &\Rightarrow \left| \begin{bmatrix} \lambda & 0 \\ 0 & \lambda \end{bmatrix} - \begin{bmatrix} -1 & 2 \\ -1 & -4 \end{bmatrix} \right| = 0 \Rightarrow \begin{vmatrix} \lambda+1 & -2 \\ 1 & \lambda+4 \end{vmatrix} = 0 \\
 &\Rightarrow (\lambda+1)(\lambda+4) + 2 = 0 \\
 &\Rightarrow \lambda^2 + 5\lambda + 6 = 0 \\
 &\Rightarrow (\lambda+2)(\lambda+3) = 0 \\
 &\Rightarrow \lambda_1 = -2, \lambda_2 = -3
 \end{aligned}$$

Eigenvectors – **method 1:**

$$\mathbf{A}\mathbf{m} = \lambda\mathbf{m} \Rightarrow \begin{bmatrix} -1 & 2 \\ -1 & -4 \end{bmatrix} \begin{bmatrix} m_1 \\ m_2 \end{bmatrix} = \lambda \begin{bmatrix} m_1 \\ m_2 \end{bmatrix} \Rightarrow \begin{cases} -m_1 + 2m_2 = \lambda m_1 \\ -m_1 - 4m_2 = \lambda m_2 \end{cases}$$

$$\text{Solving for } \lambda = -2 \Rightarrow \begin{cases} -m_1 + 2m_2 = -2m_1 \\ -m_1 - 4m_2 = -2m_2 \end{cases} \Rightarrow \begin{cases} m_1 = -2m_2 \\ m_1 = -2m_2 \end{cases} \Rightarrow \begin{bmatrix} -2 \\ 1 \end{bmatrix}$$

$$\text{Solving for } \lambda = -3 \Rightarrow \begin{cases} -m_1 + 2m_2 = -3m_1 \\ -m_1 - 4m_2 = -3m_2 \end{cases} \Rightarrow \begin{cases} m_1 = -m_2 \\ m_1 = -m_2 \end{cases} \Rightarrow \begin{bmatrix} 1 \\ -1 \end{bmatrix}$$

$$\Rightarrow \mathbf{M} = \begin{bmatrix} -2 & 1 \\ 1 & -1 \end{bmatrix}$$

Eigenvectors – **method 2 (cofactors)** – here, the eigenvectors are obtained by writing the cofactors of **any** row of  $(\lambda \mathbf{I} - \mathbf{A})$  in column format (note – make sure to allow for the correct sign, as shown below):

$$(\lambda \mathbf{I} - \mathbf{A}) = \begin{bmatrix} \lambda+1 & -2 \\ 1 & \lambda+4 \end{bmatrix} \leftrightarrow \begin{bmatrix} + & - \\ - & + \end{bmatrix}$$

$$\text{Take row 1, for example. The cofactor} = \begin{bmatrix} \lambda+4 \\ -1 \end{bmatrix}: \quad \lambda = -2 \Rightarrow \begin{bmatrix} 2 \\ -1 \end{bmatrix}, \quad \lambda = -3 \Rightarrow \begin{bmatrix} 1 \\ -1 \end{bmatrix}$$

$$\text{Take row 2. The cofactor} = \begin{bmatrix} 2 \\ \lambda+1 \end{bmatrix}: \quad \lambda = -2 \Rightarrow \begin{bmatrix} 2 \\ -1 \end{bmatrix}, \quad \lambda = -3 \Rightarrow \begin{bmatrix} 2 \\ -2 \end{bmatrix}$$

Here, each cofactor is an eigenvector and combining these for either row will give the modal matrix, as before.

- Note, in the last example, that the actual eigenvector values are not the same but this is not important.
- It is the relationship/ratio between each of the values in each eigenvector that is of key importance and this is the same in all cases.
- *This effectively relates to direction, and it is the direction itself that is important and not the actual distance in that direction!*

- **Example 6.3:** Determine the modal matrix for  $\mathbf{A} = \begin{bmatrix} 1 & 2 & 1 \\ 6 & -1 & 0 \\ -1 & -2 & -1 \end{bmatrix}$

**Solution:**

Eigenvalues:

$$\begin{aligned}
 |\lambda \mathbf{I} - \mathbf{A}| = 0 &\Rightarrow \begin{vmatrix} \lambda-1 & -2 & -1 \\ -6 & \lambda+1 & 0 \\ 1 & 2 & \lambda+1 \end{vmatrix} = 0 \\
 &\Rightarrow (\lambda-1)((\lambda+1)^2) - (-2)((-6)(\lambda+1)) + (-1)(-12 - (\lambda+1)) = 0 \\
 &\Rightarrow (\lambda-1)(\lambda^2 + 2\lambda + 1) - 12\lambda - 12 + 12 + \lambda + 1 = 0 \\
 &\Rightarrow (\lambda^3 + \lambda^2 - \lambda - 1) - 11\lambda + 1 = 0 \\
 &\Rightarrow \lambda^3 + \lambda^2 - 12\lambda = 0 \quad \Rightarrow \lambda(\lambda^2 + \lambda - 12) = 0 \\
 &\Rightarrow \lambda(\lambda-3)(\lambda+4) = 0 \quad \Rightarrow \lambda = 0, 3, -4
 \end{aligned}$$

Eigenvectors – using cofactor method:

$$(\lambda \mathbf{I} - \mathbf{A}) = \begin{bmatrix} \lambda-1 & -2 & -1 \\ -6 & \lambda+1 & 0 \\ 1 & 2 & \lambda+1 \end{bmatrix} \leftrightarrow \begin{bmatrix} + & - & + \\ - & + & - \\ + & - & + \end{bmatrix}$$

$$\text{Take row 1: cofactor} = \mathbf{M}(\lambda) = \begin{bmatrix} (\lambda+1)^2 \\ 6(\lambda+1) \\ -12 - (\lambda+1) \end{bmatrix} = \begin{bmatrix} (\lambda+1)^2 \\ 6(\lambda+1) \\ -13 - \lambda \end{bmatrix} :$$

$$\text{Hence: } \lambda = 0 \Rightarrow \begin{bmatrix} 1 \\ 6 \\ -13 \end{bmatrix}, \quad \lambda = 3 \Rightarrow \begin{bmatrix} 16 \\ 24 \\ -16 \end{bmatrix} = \begin{bmatrix} 2 \\ 3 \\ -2 \end{bmatrix}, \quad \lambda = -4 \Rightarrow \begin{bmatrix} 9 \\ -18 \\ -9 \end{bmatrix} = \begin{bmatrix} 1 \\ -2 \\ -1 \end{bmatrix}$$

Note – we have scaled the eigenvectors for convenience!



Finally:

$$\mathbf{M} = \begin{bmatrix} 1 & 2 & 1 \\ 6 & 3 & -2 \\ -13 & -2 & -1 \end{bmatrix} \quad \text{for} \quad \Lambda = \begin{bmatrix} 0 & 0 & 0 \\ 0 & 3 & 0 \\ 0 & 0 & -4 \end{bmatrix}$$

must be in the same order !

- By choosing  $\mathbf{T} = \mathbf{M}$  we get:

$$\mathbf{z}(k+1) = \mathbf{M}^{-1} \mathbf{A} \mathbf{M} \mathbf{z}(k) + \mathbf{M}^{-1} \mathbf{B} u(k) \quad \equiv \mathbf{A}_z \mathbf{z}(k) + \mathbf{B}_z u(k)$$

$$= \Lambda \mathbf{z}(k) + \mathbf{B}_z u(k)$$

- The state transition matrix for this system is given as:

$$\Lambda^k = \begin{bmatrix} \lambda_1^k & 0 & \cdots & 0 \\ 0 & \lambda_2^k & \cdots & 0 \\ \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & \cdots & \lambda_n^k \end{bmatrix}$$

- We can also easily obtain a closed form for the transition matrix  $\mathbf{A}^k$  of the original system  $\mathbf{x}(k+1) = \mathbf{A} \mathbf{x}(k) + \mathbf{B} u(k)$  as follows.

- Since:

$$\Lambda = \mathbf{M}^{-1} \mathbf{A} \mathbf{M} \Rightarrow \mathbf{A} = \mathbf{M} \Lambda \mathbf{M}^{-1}$$

- Then:

$$\mathbf{A}^k = (\mathbf{M} \Lambda \mathbf{M}^{-1})^k = \mathbf{M} \Lambda \mathbf{M}^{-1} \mathbf{M} \Lambda \mathbf{M}^{-1} \dots \mathbf{M} \Lambda \mathbf{M}^{-1} = \mathbf{M} \Lambda^k \mathbf{M}^{-1}$$

- Hence:

$$\boxed{\mathbf{A}^k = \mathbf{M} \Lambda^k \mathbf{M}^{-1}}$$

- **Example 6.4:** Determine the state transition matrix for the autonomous (i.e. no input) system:

$$\mathbf{x}(k+1) = \begin{bmatrix} -1 & 2 \\ -1 & -4 \end{bmatrix} \mathbf{x}(k)$$

and hence evaluate its response at  $k = 10$ , for the initial state  $\mathbf{x}(0) = [1 \ 0]^T$ .

**Solution:**

From example 6.2:

$$\mathbf{A} = \begin{bmatrix} -1 & 2 \\ -1 & -4 \end{bmatrix} \rightarrow \lambda = -2, -3 \rightarrow \mathbf{M} = \begin{bmatrix} -2 & 1 \\ 1 & -1 \end{bmatrix}$$

Now:  $\mathbf{x}(k) = \mathbf{A}^k \mathbf{x}(0)$

$$\begin{aligned} \mathbf{A}^k &= \mathbf{M} \mathbf{\Lambda}^k \mathbf{M}^{-1} \\ &= \begin{bmatrix} -2 & 1 \\ 1 & -1 \end{bmatrix} \begin{bmatrix} (-2)^k & 0 \\ 0 & (-3)^k \end{bmatrix} \begin{bmatrix} -2 & 1 \\ 1 & -1 \end{bmatrix}^{-1} \\ &= \begin{bmatrix} -2 & 1 \\ 1 & -1 \end{bmatrix} \begin{bmatrix} (-2)^k & 0 \\ 0 & (-3)^k \end{bmatrix} \left( \frac{1}{2-1} \right) \begin{bmatrix} -1 & -1 \\ -1 & -2 \end{bmatrix} \\ &= \begin{bmatrix} -2 & 1 \\ 1 & -1 \end{bmatrix} \begin{bmatrix} (-2)^k & 0 \\ 0 & (-3)^k \end{bmatrix} \begin{bmatrix} -1 & -1 \\ -1 & -2 \end{bmatrix} \\ &= \begin{bmatrix} -2 & 1 \\ 1 & -1 \end{bmatrix} \begin{bmatrix} -(-2)^k & -(-2)^k \\ -(-3)^k & -2(-3)^k \end{bmatrix} \end{aligned}$$

$$\Rightarrow \mathbf{A}^k = \begin{bmatrix} 2(-2)^k - (-3)^k & 2(-2)^k - 2(-3)^k \\ -(-2)^k + (-3)^k & (-2)^k + 2(-3)^k \end{bmatrix}$$

Hence:  $\mathbf{x}(k) = \mathbf{A}^k \begin{bmatrix} 1 \\ 0 \end{bmatrix} = \begin{bmatrix} 2(-2)^k - (-3)^k \\ -(-2)^k + (-3)^k \end{bmatrix}$

For  $k = 10$ :  $\mathbf{x}(10) = \begin{bmatrix} 2(-2)^{10} - (-3)^{10} \\ -(-2)^{10} + (-3)^{10} \end{bmatrix} = \begin{bmatrix} -57001 \\ 58025 \end{bmatrix}$

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## 6.3 Continuous-time solution

- The general linear SISO continuous-time state-space model is given by:

$$\begin{aligned}\dot{\mathbf{x}}(t) &= \mathbf{A}\mathbf{x}(t) + \mathbf{B}u(t) \\ y(t) &= \mathbf{C}\mathbf{x}(t) + \mathbf{D}u(t)\end{aligned}$$

- Once again, we will consider the unforced and forced responses separately.

### Unforced response:

- This is where the input is set to zero for all time, i.e.:  $u(t) = 0, \forall t$
- The state equation then becomes:  $\dot{\mathbf{x}}(t) = \mathbf{A}\mathbf{x}(t)$
- Solving this first order differential equation gives:  $\mathbf{x}(t) = e^{\mathbf{A}t} \mathbf{x}(0)$
- In this case, the matrix  $e^{\mathbf{A}t}$  is known as the (*continuous-time*) **state transition matrix**.

### Forced response:

- Now, consider the situation when the input is not zero:  $\dot{\mathbf{x}}(t) = \mathbf{A}\mathbf{x}(t) + \mathbf{B}u(t)$
- The solution for this can be written as:  $\mathbf{x}(t) = e^{\mathbf{A}t} \mathbf{x}(0) + \int_0^t e^{\mathbf{A}(t-\tau)} \mathbf{B}u(\tau) d\tau$

### Output calculation:

- Once the state has been determined, the output is easily computed as:

$$\begin{aligned}y(t) &= \mathbf{C}\mathbf{x}(t) + \mathbf{D}u(t) \\ &= \mathbf{C}e^{\mathbf{A}t} \mathbf{x}(0) + \int_{t_0}^t \mathbf{C}e^{\mathbf{A}(t-\tau)} \mathbf{B}u(\tau) d\tau + \mathbf{D}u(t)\end{aligned}$$

- Aside note – these equations are similar in form to those obtained for the discrete-time state-space models but  $\sum \rightarrow \int$  and  $\mathbf{A}^k \rightarrow e^{\mathbf{A}t}$ .*
- Just as in the discrete case, we need efficient ways of computing the state transition matrix.
- The Modal matrix and Cayley-Hamilton methods described for  $\mathbf{A}^k$  can also be applied to  $e^{\mathbf{A}t}$  with some minor modifications.

### 6.3.1 The Modal matrix (or state transformation) method

- Consider  $e^{At}$  when  $\mathbf{A} = \begin{bmatrix} \lambda_1 & 0 \\ 0 & \lambda_2 \end{bmatrix}$ .

- Expanding  $e^{At}$  as a power series gives:

$$\begin{aligned} \exp\left(\begin{bmatrix} \lambda_1 & 0 \\ 0 & \lambda_2 \end{bmatrix} t\right) &= \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} + \begin{bmatrix} \lambda_1 & 0 \\ 0 & \lambda_2 \end{bmatrix} t + \begin{bmatrix} \lambda_1 & 0 \\ 0 & \lambda_2 \end{bmatrix}^2 \frac{t^2}{2!} + \begin{bmatrix} \lambda_1 & 0 \\ 0 & \lambda_2 \end{bmatrix}^3 \frac{t^3}{3!} + \dots \\ &= \begin{bmatrix} 1 + \lambda_1 t + \lambda_1^2 \frac{t^2}{2!} + \lambda_1^3 \frac{t^3}{3!} + \dots & 0 \\ 0 & 1 + \lambda_2 t + \lambda_2^2 \frac{t^2}{2!} + \lambda_2^3 \frac{t^3}{3!} + \dots \end{bmatrix} \\ &= \begin{bmatrix} e^{\lambda_1 t} & 0 \\ 0 & e^{\lambda_2 t} \end{bmatrix} \end{aligned}$$

- Thus, when  $A$  is a diagonal matrix, calculating  $e^{At}$  is straightforward.

- Consider:  $\dot{\mathbf{x}}(t) = \mathbf{A}\mathbf{x}(t)$

- Let  $\mathbf{x}(t) = \mathbf{M}\mathbf{z}(t)$ , where  $\mathbf{M}$  is the modal matrix for  $\mathbf{A}$ .

- This gives:  $\dot{\mathbf{z}}(t) = \mathbf{M}^{-1} \mathbf{A} \mathbf{M} \mathbf{z}(t) = \mathbf{\Omega} \mathbf{z}(t)$

- Solving this equation, we obtain:  $\mathbf{z}(t) = e^{\mathbf{\Omega} t} \mathbf{z}(0)$

- Note:  $\mathbf{\Omega} = \mathbf{M}^{-1} \mathbf{A} \mathbf{M} = \begin{bmatrix} \lambda_1 & & 0 \\ & \ddots & \\ 0 & & \lambda_n \end{bmatrix}$

- But  $\mathbf{z}(t) = \mathbf{M}^{-1} \mathbf{x}(t)$ , hence:  $\mathbf{x}(t) = \mathbf{M} \mathbf{z}(t) = \mathbf{M} e^{\mathbf{\Omega} t} \mathbf{z}(0) = \mathbf{M} e^{\mathbf{\Omega} t} \mathbf{M}^{-1} \mathbf{x}(0)$

- Previously, we noted that  $\mathbf{x}(t) = e^{At} \mathbf{x}(0)$ , hence:  $\mathbf{M} e^{\mathbf{\Omega} t} \mathbf{M}^{-1} \mathbf{x}(0) = e^{At} \mathbf{x}(0)$

- Thus, the state transition matrix  $e^{At}$  can be computed as:

$$e^{At} = \mathbf{M} e^{\mathbf{\Omega} t} \mathbf{M}^{-1}$$

- Note the similarity with the discrete-time equivalent.

- **Example 6.5 (a):** Determine the **state transition matrix** for the autonomous system:

$$\dot{\mathbf{x}} = \begin{bmatrix} 0 & 2 \\ -3 & -5 \end{bmatrix} \mathbf{x}$$

**Solution:**

Eigenvalues:  $\lambda = -2, -3$  (*... verify for yourself*)

Modal matrix:  $\mathbf{M} = \begin{bmatrix} -1 & -2 \\ 1 & 3 \end{bmatrix}$  (*... verify for yourself*)

Thus:  $\mathbf{M}^{-1} = \begin{bmatrix} -3 & -2 \\ 1 & 1 \end{bmatrix}$

Hence:

$$\begin{aligned} e^{\mathbf{A}t} &= \begin{bmatrix} -1 & -2 \\ 1 & 3 \end{bmatrix} e^{\begin{bmatrix} -2 & 0 \\ 0 & -3 \end{bmatrix}t} \begin{bmatrix} -3 & -2 \\ 1 & 1 \end{bmatrix} \\ &= \begin{bmatrix} -1 & -2 \\ 1 & 3 \end{bmatrix} \begin{bmatrix} e^{-2t} & 0 \\ 0 & e^{-3t} \end{bmatrix} \begin{bmatrix} -3 & -2 \\ 1 & 1 \end{bmatrix} \\ &= \begin{bmatrix} 3e^{-2t} - 2e^{-3t} & 2e^{-2t} - 2e^{-3t} \\ -3e^{-2t} + 3e^{-3t} & -2e^{-2t} + 3e^{-3t} \end{bmatrix} \end{aligned}$$

- **Example 6.5 (b):** Determine the **evolution of the state vector** for the system:

$$\dot{\mathbf{x}}(t) = \begin{bmatrix} 0 & 2 \\ -3 & -5 \end{bmatrix} \mathbf{x}(t) + \begin{bmatrix} 0 \\ 1 \end{bmatrix} u(t)$$

if the input  $u(t)$  is a step and the initial state  $\mathbf{x}(0) = [1 \ -1]^T$ .

**Solution:**

We need to determine:  $\mathbf{x}(t) = e^{\mathbf{A}t} \mathbf{x}(0) + \int_0^t e^{\mathbf{A}(t-\tau)} \mathbf{B}u(\tau) d\tau$

$$e^{\mathbf{A}t} \mathbf{x}(0) = \begin{bmatrix} 3e^{-2t} - 2e^{-3t} & 2e^{-2t} - 2e^{-3t} \\ -3e^{-2t} + 3e^{-3t} & -2e^{-2t} + 3e^{-3t} \end{bmatrix} \begin{bmatrix} 1 \\ -1 \end{bmatrix} = \begin{bmatrix} e^{-2t} \\ -e^{-2t} \end{bmatrix}$$

$$\begin{aligned}
& \int_0^t e^{A(t-\tau)} \mathbf{B}u(\tau) d\tau \rightarrow e^{At} \int_0^t e^{-A\tau} \mathbf{B}u(\tau) d\tau \\
& = e^{At} \int_0^t \begin{bmatrix} 3e^{2\tau} - 2e^{3\tau} & 2e^{2\tau} - 2e^{3\tau} \\ -3e^{2\tau} + 3e^{3\tau} & -2e^{2\tau} + 3e^{3\tau} \end{bmatrix} \begin{bmatrix} 0 \\ 1 \end{bmatrix} (1) d\tau \\
& = e^{At} \int_0^t \begin{bmatrix} 2e^{2\tau} - 2e^{3\tau} \\ -2e^{2\tau} + 3e^{3\tau} \end{bmatrix} d\tau \\
& = e^{At} \left( \begin{bmatrix} e^{2\tau} - \frac{2}{3}e^{3\tau} \\ -e^{2\tau} + e^{3\tau} \end{bmatrix} \bigg|_0^t \right) \\
& = e^{At} \left( \begin{bmatrix} e^{2t} - \frac{2}{3}e^{3t} - 1 + \frac{2}{3} \\ -e^{2t} + e^{3t} + 1 - 1 \end{bmatrix} \right) \\
& = \begin{bmatrix} 3e^{-2t} - 2e^{-3t} & 2e^{-2t} - 2e^{-3t} \\ -3e^{-2t} + 3e^{-3t} & -2e^{-2t} + 3e^{-3t} \end{bmatrix} \begin{bmatrix} e^{2t} - \frac{2}{3}e^{3t} - \frac{1}{3} \\ -e^{2t} + e^{3t} \end{bmatrix} \\
& = \begin{bmatrix} \left(3 - 2e^t - e^{-2t} - 2e^{-t} + \frac{4}{3} + \frac{2}{3}e^{-3t}\right) + \left(-2 + 2e^t + 2e^{-t} - 2\right) \\ \left(-3 + 2e^t + e^{-2t} + 3e^{-t} - 2 - e^{-3t}\right) + \left(2 - 2e^t - 3e^{-t} + 3\right) \end{bmatrix} \\
& = \begin{bmatrix} -e^{-2t} + \frac{2}{3}e^{-3t} + \frac{1}{3} \\ e^{-2t} - e^{-3t} \end{bmatrix}
\end{aligned}$$

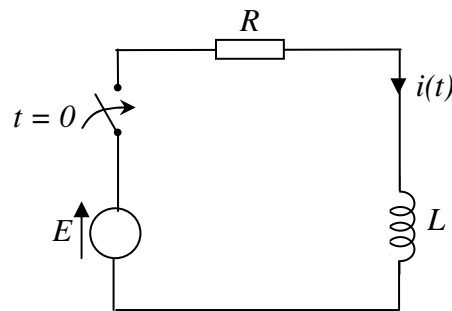
Hence:

$$\begin{aligned}
\mathbf{x}(t) &= e^{At} \mathbf{x}(0) + \int_0^t e^{A(t-\tau)} \mathbf{B}u(\tau) d\tau \\
\Rightarrow \mathbf{x}(t) &= \begin{bmatrix} e^{-2t} \\ -e^{-2t} \end{bmatrix} + \begin{bmatrix} -e^{-2t} + \frac{2}{3}e^{-3t} + \frac{1}{3} \\ e^{-2t} - e^{-3t} \end{bmatrix} \\
\Rightarrow \mathbf{x}(t) &= \begin{bmatrix} \frac{2}{3}e^{-3t} + \frac{1}{3} \\ -e^{-3t} \end{bmatrix}
\end{aligned}$$

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## 6.4 The *trilogy* of solutions

- In general linear systems can be represented using differential equations, transfer functions or state-space models.
- Each of these are solved in very different ways but will ultimately yield the same solution!
- By way of example, recall the series R-L circuit from EE214:



- The first order **differential equation model** for this simple circuit is given by:

$$L \frac{di}{dt} + Ri = E$$

- Solving this equation directly gives us the solution:

$$i(t) = \frac{E}{R} \left( 1 - e^{-\frac{R}{L}t} \right)$$

- Using Laplace Transforms, we obtain the **transfer function model** for this system as:

$$I(s) = \frac{E}{s(R + sL)}$$

- Solving this model using partial fractions and Inverse Laplace Transforms gives us the exact same solution, as expected, i.e.:

$$i(t) = \frac{E}{R} \left( 1 - e^{-\frac{R}{L}t} \right)$$

- Refer to EE214 notes for details of the above solutions.
- Here, we will complete the ‘trilogy’ of solutions, by obtaining a state space model for the R-L circuit and solving this model directly.

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### 6.4.1 The state-space model and solution for the series R-L circuit

- We know the first-order differential equation is:  $L \frac{di}{dt} + Ri = E$
- Here, we only have a **first-order** differential equation and hence only one state:  $x_1 = i$
- We now obtain an expression for  $\dot{x}_1$ :

$$\dot{x}_1 = \frac{di}{dt} = \frac{E}{L} - \frac{R}{L}i$$

- Rewriting this equation in terms of state  $x_1$ , and letting input  $u = E$ , we get:

$$\dot{x}_1 = \frac{-R}{L}x_1 + \frac{1}{L}u$$

- The output  $y = i(t) = x_1$ . Hence, our simple **state-space model** is expressed, in full, as:

$$[\dot{x}_1] = \left[ \frac{-R}{L} \right] [x_1] + \left[ \frac{1}{L} \right] [u]$$

$$[y] = [1] [x_1]$$

- Since  $y = x_1$ , to solve for the output we simply need to evaluate the expression:

$$\mathbf{x}(t) = e^{\mathbf{A}t} \mathbf{x}(0) + \int_0^t e^{\mathbf{A}(t-\tau)} \mathbf{B}u(\tau) d\tau$$

- Here, we have only one state and, also, the initial condition is  $x_1(0) = 0$ . Hence:

$$\begin{aligned} y = x(t) &= 0 + \int_0^t e^{-\frac{R}{L}(t-\tau)} \left( \frac{1}{L} \right) E d\tau \\ &= \frac{E}{L} e^{-\frac{R}{L}t} \int_0^t e^{\frac{R}{L}\tau} d\tau = \frac{E}{L} e^{-\frac{R}{L}t} \left( e^{\frac{R}{L}\tau} \left( \frac{L}{R} \right) \right)_0^t = \frac{E}{R} e^{-\frac{R}{L}t} \left( e^{\frac{R}{L}t} - 1 \right) \\ &= \frac{E}{R} \left( 1 - e^{-\frac{R}{L}t} \right) \end{aligned}$$

- Once again, we obtain the same solution.



- 
- Hence, we have shown that we can represent systems using three different models and the solutions to each of these models, as expected, are the same.
  - The key benefit of using state-space models is that the model and its solution scales easily for higher-order problems.
  - Differential equations, on the other hand, become more complex to solve for higher orders.
  - Transfer function models do not allow for initial conditions and can only be written for an individual input-output relationship.
  - State-space models can allow represent multi-input multi-output systems in the same model, caters for initial conditions and offers the same solution methodology irrespective of the order of the model.
  - In the next section, we will examine the issue of stability from the state-space model viewpoint.

