

A Guide to Design Disturbance Observer

Emre Sariyildiz¹

Department of System Design Engineering,
Keio University, Yokohama,
3-14-1 Hiyoshi, Kohoku-ku, Yokohama,
Kanagawa 223-8522, Japan
e-mail: emre@sum.sd.keio.ac.jp

Kouhei Ohnishi

Department of System Design Engineering,
Keio University, Yokohama,
3-14-1 Hiyoshi, Kohoku-ku, Yokohama,
Kanagawa 223-8522, Japan
e-mail: ohnishi@sd.keio.ac.jp

The goal of this paper is to clarify the robustness and performance constraints in the design of control systems based on disturbance observer (DOB). Although the bandwidth constraints of a DOB have long been very well-known by experiences and observations, they have not been formulated and clearly reported yet. In this regard, the Bode and Poisson integral formulas are utilized in the robustness analysis so that the bandwidth constraints of a DOB are derived analytically. In this paper, it is shown that the bandwidth of a DOB has upper and lower bounds to obtain a good robustness if the plant has nonminimum phase zero(s) and pole(s), respectively. Besides that the performance of a system can be improved by using a higher order disturbance observer (HODOB); however, the robustness may deteriorate, and the bandwidth constraints become more severe. New analysis and design methods, which provide good robustness and predefined performance criteria, are proposed for the DOB based robust control systems. The validity of the proposals is verified by simulation results. [DOI: 10.1115/1.4025801]

1 Introduction

A DOB, which was proposed by Ohnishi et al., is a robust control tool that is used to estimate external disturbances and system uncertainties [1–3]. The estimated disturbances, which include system uncertainties, are fed-back by using an inner feed-back loop so that the robustness of a system is achieved by using a DOB [2]. Performance goals of a system are achieved by using an outer feed-back loop controller that is designed independently by considering only the nominal plant model, since a DOB can nominalize the inner-loop [4,5]. This control structure is called as two-degrees-of-freedom control in the literature [5]. Although a DOB has been widely used in several motion control applications, e.g., robotics, industrial automation and automotive, in the last two decades, it has no systematic analysis and design methods [6–8]. Therefore, the performance and robustness of a DOB based control system highly depend on designers' own experiences.

A low pass filter (LPF) and the inverse of a nominal plant model are required to design a DOB. Although the LPF of a DOB is essential to satisfy causality in the inner-loop, it is one of the main robustness and performance limitation sources in the control systems based on DOB [9,10]. Besides that the inverse of a nominal plant model causes internal stability problem if the plant has nonminimum phase zero(s); therefore, a special consideration is required when a DOB is implemented to a nonminimum phase plant [11–13].

It is a well-known fact that a DOB can estimate disturbances precisely if they stay within the bandwidth of the DOB's LPF [14,15]. Therefore, its bandwidth is desired to set as high as possible to estimate disturbances in a wide frequency range, i.e., to improve the robustness and performance [2]. However, the bandwidth of a DOB is limited by the robustness of a system and noise, so it cannot be shaped freely [2,10]. The noise limitation is directly related to sampling rate and measurement plants and methodology; it puts an upper bound on the bandwidth of a DOB [2]. Several researches have been reported to increase the bandwidth of a DOB by suppressing the noise of measurement [16–18]. The robustness of a DOB based control system is directly related to the dynamic characteristics of the DOB's LPF and nominal plant. They also limit the bandwidth of a DOB; however, the

relation between the robustness of a system and the dynamic characteristics of the DOB's LPF and nominal plant has not been clearly reported yet [10,19,20]. Recently, it was shown by the authors that if a minimum-phase system has only real parametric uncertainties, then a DOB can guarantee the robustness of the system by increasing its bandwidth, and the stability margin of the system improves as the bandwidth of the DOB is increased [21]. However, it considers only the minimum-phase systems which have real parametric uncertainties when a first order DOB is used.

The main aim of this paper is to clarify the robustness constraints of DOB for a broad range of application area. The Bode integral formula is utilized so that the robustness of minimum-phase and time-delay systems are derived analytically; and the Poisson integral formula is used to derive the robustness constraints of systems with right half plane (RHP) zero(s) and pole(s) [22–26]. It is shown that right half plane (RHP) zero(s) and/or time-delay of a plant limit the bandwidth of a DOB, however, RHP pole(s) of a plant put(s) a lower bound on the bandwidth of a DOB to obtain a good robustness. Besides that increasing the order of a DOB improves the performance of a system by using the bandwidth of a DOB more effectively; however, the bandwidth constraints become more severe, and the robustness of a system deteriorates. New analysis and design methods are proposed by using the derived bandwidth constraints. The internal stability problem is solved by using an approximate minimum phase nominal plant model when a plant has nonminimum phase zero(s), and a performance controller is proposed for the conventional two-degrees-of-freedom control structure when a plant has RHP pole(s).

The rest of the paper is organized as follows: In Sec. 2, the conventional two-degrees-of-freedom control structure of a DOB based robust control system is presented briefly. In Sec. 3, the bandwidth constraints of a DOB are derived analytically by using the Bode and Poisson integral formulas. In Sec. 4, four case-studies are given. The paper ends with conclusion given in the last section.

2 Disturbance Observer

Figure 1 shows a general control block diagram for a DOB based robust control system. In this figure, $G(s)$ and $G_n(s)$ denote uncertain and nominal plant models, respectively; $Q(s)$ denotes the LPF of a DOB; $C(s)$ denotes the outer-loop controller; r , τ_{dis} , and ξ denote reference, disturbance, and noise external inputs, respectively; and $\hat{\tau}_{dis}$ denotes estimated disturbances,

¹Corresponding author.

Contributed by the Dynamic Systems Division of ASME for publication in the JOURNAL OF DYNAMIC SYSTEMS, MEASUREMENT, AND CONTROL. Manuscript received December 8, 2012; final manuscript received September 19, 2013; published online December 9, 2013. Assoc. Editor: Luis Alvarez.

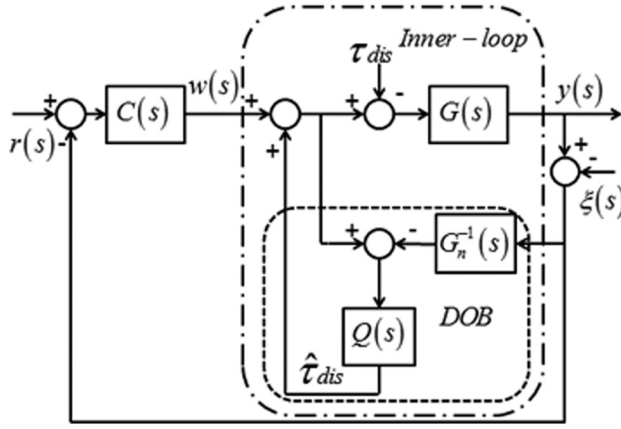


Fig. 1 A block diagram for a two-degrees-of-freedom DOB based robust control system

which includes external disturbances and system uncertainties. The open-loop, sensitivity and co-sensitivity transfer functions are derived from Fig. 1 as follows:

Inner Loop:

$$L_i(s) = \frac{G(s)Q(s)}{G_n(s)(1-Q(s))} \quad (1)$$

$$S_i(s) = \frac{G_n(s)(1-Q(s))}{G_n(s)(1-Q(s)) + G(s)Q(s)} \quad (2)$$

$$T_i(s) = \frac{G(s)Q(s)}{G_n(s)(1-Q(s)) + G(s)Q(s)} \quad (3)$$

Outer Loop:

$$L_o(s) = \frac{C(s)G(s)G_n(s)}{G_n(s)(1-Q(s)) + G(s)Q(s)} \quad (4)$$

$$S_o(s) = \frac{G_n(s)(1-Q(s)) + G(s)Q(s)}{G_n(s)(1-Q(s)) + G(s)Q(s) + G(s)G_n(s)C(s)} \quad (5)$$

$$T_o(s) = \frac{G(s)G_n(s)C(s)}{G_n(s)(1-Q(s)) + G(s)Q(s) + G(s)G_n(s)C(s)} \quad (6)$$

where L_\bullet , S_\bullet , and T_\bullet denote the open-loop, sensitivity, and co-sensitivity transfer functions, respectively [10].

3 Bandwidth Constraints of Disturbance Observer

In this section, the Bode and Poisson integral formulas are utilized to derive the bandwidth constraints of a DOB analytically. A multiplicative unstructured uncertainty is used to define the uncertain plant model given by

$$G(s) = G_n(s)(1 + \Delta W(s)) \exp(-\tau s) \quad (7)$$

where $G(s)$ and $G_n(s)$ denote the uncertain and nominal plant models, respectively; $W(s)$ denotes the multiplicative unstructured uncertainty weighting function; and τ denotes delay time. Without losing the generality, a first order approximation of the weighting function is described by using

$$W(s) = \frac{w_T^{-1}s + e_{\min}}{w_T^{-1}e_{\max}^{-1}s + 1} \quad (8)$$

where e_{\min} and e_{\max} denote the minimum and maximum modeling errors, respectively; and w_T is the frequency in which the nominal plant model starts to be a bad indicator for the uncertain plant

[27,28]. It is assumed that $-e_{\max}^{-1} < \Delta < 1$ instead of $|\Delta| < 1$ so that a RHP zero is not added due to uncertainty. The n th order LPF of a DOB is defined by using

$$Q(s) = \frac{g_0}{s^n + g_{n-1}s^{n-1} + \dots + g_1s + g_0} \quad (9)$$

If Eqs. (1)–(3) are rewritten in terms of the LPF, plant uncertainty and time-delay, then

$$\begin{aligned} L_i(s) &= \frac{Q(s)}{(1-Q(s))} (1 + \Delta W(s)) e^{-\tau s} \\ S_i(s) &= \frac{(1-Q(s))}{(1-Q(s)) + Q(s)(1 + \Delta W(s)) e^{-\tau s}} \\ T_i(s) &= \frac{Q(s)(1 + \Delta W(s)) e^{-\tau s}}{(1-Q(s)) + Q(s)(1 + \Delta W(s)) e^{-\tau s}} \end{aligned} \quad (10)$$

The bandwidth constraints of a DOB are derived analytically as follows:

3.1 Minimum-phase Plant. LEMMA 1. Let us consider the plant model given in Eq. (7) and assume that the uncertain plant is minimum-phase and the order of DOB is one. Then, it can be shown that the inner-loop is strictly robust if $\Delta > 0$, and its robustness can be guaranteed for a wide range of DOB's bandwidth if $\Delta < 0$. However, if a HODOB is used and/or the plant includes time delay, then the robustness of inner-loop cannot be guaranteed for a wide range of DOB's bandwidth even if $\Delta > 0$, and the bandwidth of a DOB becomes limited.

Proof. Let us first consider a minimum-phase plant and rewrite the open loop transfer function given in Eq. (10) by using a first order DOB

$$L_i(s) = g_0(1 + \Delta e_{\max}) \frac{s + w_T e_{\max} \left(\frac{1 + \Delta e_{\min}}{1 + \Delta e_{\max}} \right)}{s(s + w_T e_{\max})} \quad (11)$$

Equation (11) shows that the Nyquist plot of the inner-loop gets into the unit circle that is shown in Fig. 2 if $e_{\min} > e_{\max}$, which contradicts with the error assumption, when $\Delta > 0$. Thus, the first part of the Lemma 1, i.e., strict robustness, is satisfied.

Although the Nyquist plot of the inner-loop gets into the unit circle, i.e., strict robustness is lost, when $\Delta < 0$, the robust stability can be guaranteed for a wide range of DOB's bandwidth as it can be seen from Eq. (11).

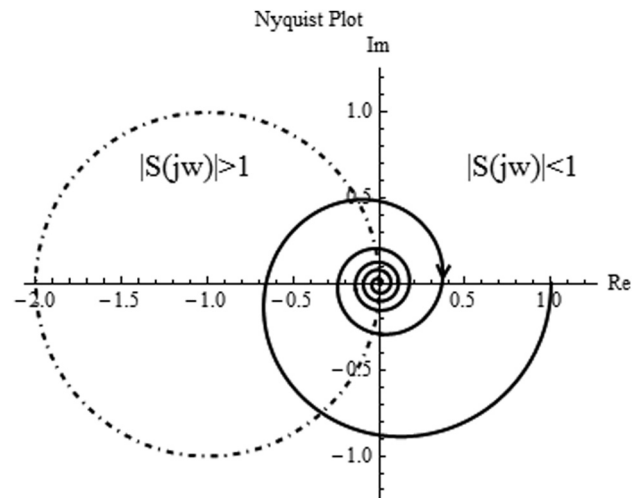


Fig. 2 Nyquist plot of inner-loop when a plant has time-delay and DOB is first order

However, if a HODOB is used instead of a DOB, then the robustness cannot be guaranteed for a wide range of DOB's bandwidth even if $\Delta > 0$. It can be easily shown by using a HODOB, e.g., a second order DOB. In that case, the robustness of a system can be guaranteed for a limited bandwidth of a HODOB.

Figure 2 shows the Nyquist plot of the inner-loop when a DOB is implemented to a time-delay plant. It indicates that the robustness cannot be guaranteed for a wide range of DOB's bandwidth if a plant includes time-delay. Hence, the proof of the Lemma 1 is completed.

The Lemma 1 gives us a basic insight into the robustness of a DOB; however, further analysis is required for a HODOB. The Horowitz's integral formula, given in Eq. (12), can be used to analyze the robustness of a HODOB [22,24].

$$\int_0^\infty (\log(|S_i(jw)|) - \log(|S_i(j\infty)|))dw = -\frac{\pi}{2} \text{Res}(\log(S_i(s))) \quad (12)$$

Since the relative degree of $L_i(s)$ is always higher than one when a HODOB is used, Eq. (12) can be simplified, and the Bode integral formula is obtained as follows: [22,24]

$$\int_0^\infty \log(|S_i(jw)|)dw = 0 \quad (13)$$

The robustness of a system depends on the magnitude of sensitivity function peak, which is defined by $\sup(|S(jw)|)$ where $\sup(*)$ denotes supremum of $*$. However, it is not an easy task to determine $\sup(|S(jw)|)$ by using Eq. (13) due to the infinite integral range. Although, from a mathematical point of view, Eq. (13) can be balanced with a small peak in a wide frequency range, control systems cannot exhibit this response due to uncertainties, digital control implementations, and so on. Besides that the Lemma 1 shows that $|S_i(jw)|$ has a peak if a HODOB is used. The Lemma 2 bounds the integral range of Eq. (13) when a HODOB is used.

LEMMA 2. Let us assume that $L_i(s)$ satisfies

$$|L_i(s)| \leq \frac{M}{w^{k+1}} = \delta \leq \frac{1}{2} \forall w \geq w_\gamma \quad (14)$$

where $M \geq \lim_{s \rightarrow \infty} \sup |s^{k+1} L_i(s)|$ and $k+1$ is the order of DOB. Then, $S_i(s)$ satisfies

$$\left| \int_{w_\beta}^\infty \log(|S_i(jw)|)dw \right| \leq \frac{3\delta}{2k} w_\gamma$$

Proof. Equation (14) holds if a HODOB is used. Let us consider the relation given by [30]

$$\text{if } |L_i(s)| \leq \frac{1}{2}, \text{ then } |\log(1 + L_i(s))| \leq \frac{3}{2} |L_i(s)| \leq \frac{3}{2} \delta \quad (16)$$

If Eq. (14) is put into Eq. (16), then

$$\begin{aligned} \left| \int_{w_\beta}^\infty \log(|S_i(jw)|)dw \right| &\leq \int_{w_\gamma}^\infty |\log(S_i(jw))|dw \\ &= \int_{w_\gamma}^\infty |\log(1 + L_i(jw))|dw \\ &\leq \frac{3}{2} \int_{w_\gamma}^\infty \frac{M}{w^{1+k}} dw = \frac{3\delta}{2k} w_\gamma \end{aligned} \quad (17)$$

To derive the robustness constraints of a HODOB, the performance and robustness requirements are determined in a predefined frequency range by shaping the sensitivity transfer function. Then, the robustness of a system is analyzed, and the constraints are derived by using the Theorem 1 as follows:

THEOREM 1. Let us assume that a minimum phase plant is defined by using Eq. (7). Let us also assume that $S_i(s)$ satisfies $|S_i(jw)| \leq \alpha < 1, \forall w \leq w_\beta < w_\gamma$. If a DOB is used, then the system has a good robustness in a wide frequency range, yet its performance is limited by the dynamic characteristics of the DOB. However, if a HODOB is used, then the LPF of a HODOB should satisfy the following inequalities to obtain a good robustness and predefined performance criterion

$$|Q(jw)| \geq \frac{1 - \alpha}{1 + \alpha |\Delta W(jw)|}, \quad \forall w < \psi w_\gamma \text{ and } \frac{|1 - Q(j\psi w_\gamma)|}{|1 + \Delta Q W(j\psi w_\gamma)|} \geq \alpha \quad (18)$$

where $\psi = \sup_{w \in [w_\beta, w_\gamma]} \log(|S_i(jw)|) + 3\delta/2k / (\sup_{w \in [w_\beta, w_\gamma]} \log(|S_i(jw)|) + \log(\alpha^{-1}))$ in which $|L_i(jw)| \leq \delta \leq 1/2, \forall w \geq w_\gamma$.

Proof. The Lemma 1 proves the robustness of a DOB. Therefore, a HODOB can be considered directly. Let us rewrite Eq. (13) by using

$$\int_0^{w_\beta} \log(|S_i(jw)|)dw + \int_{w_\beta}^{w_\gamma} \log(|S_i(jw)|)dw + \int_{w_\gamma}^\infty \log(|S_i(jw)|)dw = 0 \quad (19)$$

If the sensitivity constraints given in the Theorem 1 and Lemma 2 are applied into Eq. (19), then

$$\log(\alpha) \int_0^{w_\beta} dw + \sup_{w \in [w_\beta, w_\gamma]} \log(|S_i(jw)|) \int_{w_\beta}^{w_\gamma} dw + \frac{3\delta}{2k} w_\gamma \geq 0 \quad (20)$$

which can be easily rewritten as follows:

$$\sup_{w \in [w_\beta, w_\gamma]} \log(|S_i(jw)|) \geq \log(\alpha^{-1}) \frac{w_\beta}{w_\gamma - w_\beta} - \frac{3\delta}{2k} \frac{w_\gamma}{w_\gamma - w_\beta} \quad (21)$$

$$\frac{w_\beta}{w_\gamma} \leq \psi = \frac{\sup_{w \in [w_\beta, w_\gamma]} \log(|S_i(jw)|) + \frac{3\delta}{2k}}{\sup_{w \in [w_\beta, w_\gamma]} \log(|S_i(jw)|) + \log(\alpha^{-1})} \quad (22)$$

where Eq. (22) is the function ψ given in the Theorem 1. If the sensitivity constraint given in the Theorem 1 is applied into Eq. (10), then

$$\frac{|1 - Q(jw)|}{|1 + \Delta Q W(jw)|} \leq \alpha < 1, \quad \forall w \leq w_\beta < w_\gamma \quad (23)$$

If Eq. (22) is applied into Eq. (23), then Eq. (18) is derived.

As the order of a DOB, which is defined by $k+1$, is increased, the difference between the frequencies w_γ and w_β decreases. Therefore, increasing k causes higher sensitivity peak as derived in Eq. (21).

Equation (23) shows that α^{-1} and w_β increase as the bandwidth of a HODOB is increased. Therefore, the peak of $|S_i(jw)|$ becomes higher with the increasing bandwidth of a DOB as derived in Eq. (21).

Equation (18) provides a new design tool to obtain a good robustness and predefined performance criterion, which are determined by α and w_β . If the LPF of a HODOB satisfies (18), then the robustness and performance goals of a system can be achieved. However, Eq. (18) includes conservatism due to sectionally constant sensitivity bound defined by $|S_i(jw)| \leq \alpha < 1, \forall w \leq w_\beta < w_\gamma$. It can be lessened by using more realistic sensitivity bounds [29,30].

3.2 Plant With Time-Delay. The Lemma 3 is used to bound the integral range of Eq. (13) when a plant has time-delay.

Lemma 3. Let us assume that $L_i(s)$ includes time-delay and satisfies

$$|L_i(s)| = |\tilde{L}_i(s)e^{-s\tau}| \leq \frac{M}{R^k} e^{-R\tau \cos(\theta)} \leq \delta \left(\frac{R}{|s|}\right)^k \quad \forall |s| \in S(R) \quad (24)$$

where $M \geq \lim_{s \rightarrow \infty} \sup |s^k L(s)|$; k is the order of DOB; $s = Re^{j\theta}$; and $S(R) = \{s : \text{Re}(s) \geq 0 \text{ and } |s| \geq R\}$. Then, $S_i(s)$ satisfies

$$\left| \int_R^\infty \log(|S_i(jw)|) dw \right| \leq \frac{3\pi}{4\tau} \delta \quad (25)$$

Proof. Similar to the Lemma 2 [25].

The bandwidth constraints of a DOB due to time-delay are derived by using the Theorem 2 as follows:

THEOREM 2. Let us assume that a plant is defined by using Eq. (7) where $G_n(s)$ is minimum-phase. Let us also assume that $S_i(s)$ satisfies $|S_i(jw)| \leq \alpha < 1, \forall w \leq w_B$. Then, the LPF of a DOB should satisfy the following inequalities to obtain a good robustness and predefined performance criterion

$$\begin{aligned} \frac{|Q(jw)|}{|(1-Q(jw))|} &\geq \frac{1-\alpha}{\alpha|1+\Delta W(jw)|}, \quad \forall w \leq \psi R \text{ and} \\ &\times \frac{|1-Q(j\psi R)|}{|1-Q(j\psi R) + Q(j\psi R)(1+\Delta W(j\psi R))e^{-j\tau\psi R}|} \\ &\geq \alpha \end{aligned} \quad (26)$$

If the order of a DOB is one, then

$$\begin{aligned} g &\geq \frac{(1-\alpha)w}{\alpha|1+\Delta W(jw)|}, \quad \forall w \leq \psi R \text{ and} \\ &\times \frac{\psi R}{|j\psi R + g(1+\Delta W(j\psi R))e^{-j\tau\psi R}|} \geq \alpha \end{aligned} \quad (27)$$

where $(\psi = \sup_{w \in [w_B, R]} \log(|S_i(jw)|) + 3\pi/4\tau R\delta) / (\sup_{w \in [w_B, R]} \log(|S_i(jw)|) + \log(\alpha^{-1}))$ in which $|L_i(s)| \leq \delta(R/|s|)^k \forall |s| \in S(R)$.

Proof. Similar to the Theorem 1. The Lemma 3 is used instead of the Lemma 2.

Equation (26) provides a new design tool to obtain a good robustness and predefined performance criterion when a plant has time-delay. It shows that the bandwidth of a DOB is limited due to time-delay. However, the proposed design tool also includes conservatism, since the sensitivity bounds are not realistic.

Some comments are required to determine R . Its smallest value, which satisfies the constraint given in the Theorem 2, should be used to lessen the peak of $|S_i(jw)|$.

$$\sup_{s \in S(R)} (L_i(s)) = \max \left\{ \sup_{w \geq R} (|L_i(jw)|), \sup_{0 \leq \theta \leq \pi/2} (|L_i(Re^{j\theta})|) \right\} \quad (28)$$

Equation (28) shows that the Theorem 2 holds even if $\sup_{s \in S(R)} (L_i(s)) \geq \delta$, which can be used to lessen the peak of $|S_i(jw)|$ [25]. The sensitivity peak can be lessened if the following inequality holds:

$$\begin{aligned} &\frac{w_T e_{\min}}{w^2 + (w_T e_{\min})^2} - \frac{w_T e_{\max}}{w^2 + (w_T e_{\max})^2} \\ &> \begin{cases} \tau, & \text{1st order DOB} \\ \tau + \frac{g_1}{s+g_1}, & \text{2nd order DOB} \\ \tau + \frac{g_1 g_2 + g_2 w^2}{g_2^2 w^2 + (g_1 - w^2)^2}, & \text{3rd order DOB} \\ \vdots & \end{cases} \end{aligned} \quad (29)$$

As it is expected from the Theorem 1, Eq. (29) shows that the peak of $|S_i(jw)|$ increases as the order of a DOB is increased.

3.3 Nonminimum Phase Plant

3.3.1 Performance Limitations. It is a well-known fact that RHP zero(s) and pole(s) of open loop transfer functions cause undershoot and overshoot in the step responses of the closed loop systems, respectively. To achieve good performance, the following inequalities should be held:

$$w_B \leq \frac{2.1991 z_{\text{RHP}}}{\log\left(1 - \frac{0.9}{y_{\text{undershoot}}}\right)} \quad (30)$$

$$w_B \geq \frac{2.1991 p_{\text{RHP}}}{\log(10(y_{\text{overshoot}} - 0.9))} \quad (31)$$

where $w_B, z_{\text{RHP}}, p_{\text{RHP}}, y_{\text{undershoot}}$ and $y_{\text{overshoot}}$, denote the bandwidth, RHP zero, RHP pole, infimum and supremum of the step response, respectively [27,28].

3.3.2 The Poisson's Integral Formulas. The Poisson's integral formulas are used to derive the bandwidth constraints of a DOB analytically when a plant has RHP pole(s) and/or zero(s).

Poisson's Integral Formulas: Assume that an open-loop transfer function $L(s)$ has a RHP zero/(pole) at $z_{\text{RHP}} = \sigma_z + jw_z / (p_{\text{RHP}} = \sigma_p + jw_p)$. Let $S(s)/(T(s))$ be the sensitivity/(co-sensitivity) transfer function defined by $(1+L(s))^{-1} / (L(s)(1+L(s))^{-1})$. Then, it can be shown that the sensitivity/(co-sensitivity) transfer function satisfies

$$\begin{aligned} \int_{-\infty}^{\infty} \log(|S(jw)|) \frac{\sigma_z}{\sigma_z^2 + (w_z - w)^2} dw &= \pi \log(|B_S^{-1}(z_{\text{RHP}})|) \\ \int_{-\infty}^{\infty} \log(|T(jw)|) \frac{\sigma_p}{\sigma_p^2 + (w_p - w)^2} dw &= \pi \log(|B_T^{-1}(p_{\text{RHP}})|) \end{aligned} \quad (32)$$

where $L(s) = \tilde{L}(s)B_S^{-1}(s)B_T(s)$; $\tilde{L}(s)$ is a minimum-phase transfer function; $B_S(s) = \prod_{i=1}^k ((p_i - s)/(\bar{p}_i + s))$ and $B_T(s) = \prod_{i=1}^l ((z_i - s)/(\bar{z}_i + s))$ are Blaschke products [29,30]. The integral ranges of Eqs. (32) and (33) are bounded by $W_{pi}(w) = ((\sigma_x)/(\sigma_x^2 + (w_x - w)^2))$.

3.3.3 Plant With RHP Zero(s). An approximate nominal plant model is used to solve the internal stability problem when a plant has RHP zero(s). It is defined by

$$G(s) = G_n(s)(1 + \Delta W_T(s)) = \hat{G}_n(s)r_{\text{err}}(s)(1 + \Delta W_T(s)) \quad (34)$$

where $\hat{G}_n(s)$ is the approximate nominal plant model, which has stable inverse; and $r_{\text{err}}(s) = G_n(s)(\hat{G}_n(s))^{-1}$ [31]. Then, the open-loop transfer functions are defined as follows:

Inner Loop:

$$L_i(s) = \frac{r_{\text{err}}(s)(1 + \Delta W_T(s))Q(s)}{1 - Q(s)} \quad (35)$$

Outer Loop:

$$L_o(s) = \frac{C(s)G(s)}{1 - Q(s) + r_{\text{err}}(s)(1 + \Delta W_T(s))Q(s)} \quad (36)$$

The bandwidth constraints of a DOB due to RHP zero(s) are derived by using the Theorem 3 as follows:

THEOREM 3. Let us assume that a plant, which has a RHP zero at z_{RHP} , is defined by using Eq. (7) when $\tau = 0$; and $S_i(s)$ and $T_i(s)$ are the sensitivity and co-sensitivity transfer functions of the inner-loop, respectively. Let us also assume that the frequency responses of the sensitivity and co-sensitivity transfer functions satisfy $|S_i(jw)| \leq \alpha_\beta, \forall w \leq w_\beta$ and $|T_i(jw)| \leq \alpha_\gamma, \forall w \geq w_\gamma$. Then,

the LPF of a DOB should satisfy the following constraints to obtain a good robustness and predefined performance criteria

$$\frac{|Q(jw)|}{|1-Q(jw)|} \geq \frac{1-\alpha_\beta}{\alpha_\beta |r_{\text{err}}(jw)(1+\Delta W_T(jw))|}, \quad \forall w < z_{\text{RHP}}\psi_1 \text{ and} \\ \times \frac{|1-Q(jz_{\text{RHP}}\psi_1)|}{|1-Q(jz_{\text{RHP}}\psi_1)+r_{\text{err}}Q(jz_{\text{RHP}}\psi_1)(1+\Delta W_T(jz_{\text{RHP}}\psi_1))|} \geq \alpha_\beta$$

(37) where

$$\frac{|r_{\text{err}}Q(jw)(1+\Delta W_T(jw))|}{|1-Q(jw)|} \leq \frac{\alpha_\gamma}{1-\alpha_\gamma}, \quad \forall w > z_{\text{RHP}}\psi_2 \text{ and} \\ \times \frac{|r_{\text{err}}Q(jz_{\text{RHP}}\psi_2)(1+\Delta W_T(jz_{\text{RHP}}\psi_2))|}{|1-Q(jz_{\text{RHP}}\psi_2)+r_{\text{err}}Q(jz_{\text{RHP}}\psi_2)(1+\Delta W_T(jz_{\text{RHP}}\psi_2))|} \geq \alpha_\gamma$$

(38)

$$\psi_1 = \tan \left(\frac{\log(1+\alpha_\gamma)(\pi-2\vartheta(w_\gamma)) + 2\log\left(\max_{w_\beta \leq w \leq w_\gamma}(|S(jw)|)\right)\vartheta(w_\gamma)}{2\left(\log\left(\max_{w_\beta \leq w \leq w_\gamma}(|S(jw)|)\right) + \log(\alpha_\beta^{-1})\right)} - \frac{\pi \log(|B_S^{-1}(z_{\text{RHP}})|)}{2\left(\log\left(\max_{w_\beta \leq w \leq w_\gamma}(|S(jw)|)\right) + \log(\alpha_\beta^{-1})\right)} \right) \\ \psi_2 = \tan \left(\frac{\log((1+\alpha_\gamma)^{-1})\pi + 2\left(\log(\alpha_\beta^{-1}) + \log\left(\max_{w_\beta \leq w \leq w_\gamma}(|S(jw)|)\right)\right)\vartheta(w_\beta)}{2\left(\log\left(\max_{w_\beta \leq w \leq w_\gamma}(|S(jw)|)\right) + \log((1+\alpha_\gamma)^{-1})\right)} + \frac{\pi \log(|B_S^{-1}(z_{\text{RHP}})|)}{2\left(\log\left(\max_{w_\beta \leq w \leq w_\gamma}(|S(jw)|)\right) + \log(\alpha_\beta^{-1})\right)} \right) \\ \vartheta(w_\bullet) = \int_0^{w_\bullet} W dw = \int_0^{w_\bullet} \frac{z_{\text{RHP}}}{z_{\text{RHP}}^2 + w^2} dw = \arctan\left(\frac{w_\bullet}{z_{\text{RHP}}}\right)$$

Proof. If $|T_i(jw)| \leq \alpha_\gamma, \forall w \geq w_\gamma$, then $|S_i(jw)| \leq 1 + \alpha_\gamma, \forall w \geq w_\gamma$. If the sensitivity constraints are applied into Eq. (32), then

$$\log(1+\alpha_\gamma) \left(\int_{-\infty}^{-w_\gamma} W(w) dw + \int_{w_\gamma}^{\infty} W(w) dw \right) \\ + \log(\alpha_\beta) \int_{-w_\beta}^{w_\beta} W(w) dw + \log\left(\max_{w_\beta \leq w \leq w_\gamma}(|S_i(jw)|)\right) \\ \times \left(\int_{-w_\gamma}^{-w_\beta} W(w) dw + \int_{w_\beta}^{w_\gamma} W(w) dw \right) \geq \pi \log(|B_S^{-1}(z_{\text{RHP}})|)$$

(39)

which can be transformed into

$$\log\left(\max_{w_\beta \leq w \leq w_\gamma}(|S_i(jw)|)\right) \geq \log(\alpha_\beta^{-1}) \frac{2\vartheta(w_\beta)}{2(\vartheta(w_\gamma) - \vartheta(w_\beta))} \\ + \log((1+\alpha_\gamma)^{-1}) \frac{\pi - 2\vartheta(w_\gamma)}{2(\vartheta(w_\gamma) - \vartheta(w_\beta))} \\ + |B_S^{-1}(z_{\text{RHP}})| \frac{\pi}{2(\vartheta(w_\gamma) - \vartheta(w_\beta))}$$

(40)

$$\vartheta(w_\beta) \leq \frac{\log(1+\alpha_\gamma)(\pi-2\vartheta(w_\gamma)) + 2\log\left(\max_{w_\beta \leq w \leq w_\gamma}(|S_i(jw)|)\right)\vartheta(w_\gamma)}{2\left(\log\left(\max_{w_\beta \leq w \leq w_\gamma}(|S_i(jw)|)\right) + \log(\alpha_\beta^{-1})\right)} \\ - \frac{\pi \log(|B_S^{-1}(z_{\text{RHP}})|)}{2\left(\log\left(\max_{w_\beta \leq w \leq w_\gamma}(|S_i(jw)|)\right) + \log(\alpha_\beta^{-1})\right)}$$

(41)

$$\vartheta(w_\gamma) \geq \frac{\log((1+\alpha_\gamma)^{-1})\pi + 2\left(\log(\alpha_\beta^{-1}) + \log\left(\max_{w_\beta \leq w \leq w_\gamma}(|S_i(jw)|)\right)\right)\vartheta(w_\beta)}{2\left(\log\left(\max_{w_\beta \leq w \leq w_\gamma}(|S_i(jw)|)\right) + \log((1+\alpha_\gamma)^{-1})\right)} + \frac{\pi \log(|B_S^{-1}(z_{\text{RHP}})|)}{2\left(\log\left(\max_{w_\beta \leq w \leq w_\gamma}(|S_i(jw)|)\right) + \log(\alpha_\beta^{-1})\right)}$$

(42)

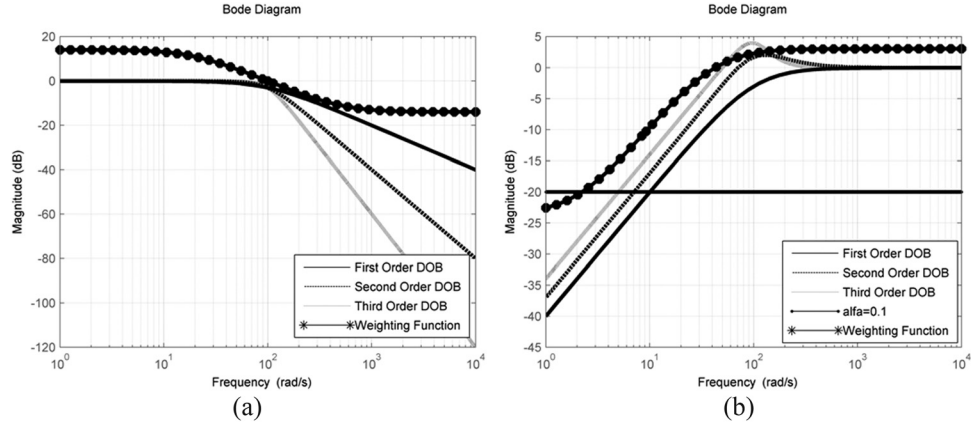


Fig. 4 Frequency responses of the inner-loop sensitivity and co-sensitivity transfer functions when the bandwidth of DOB is 100 rad/s

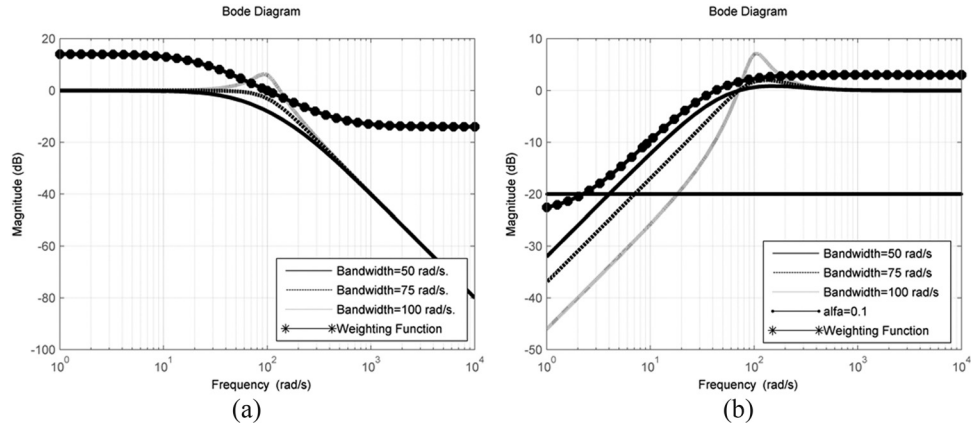


Fig. 5 Frequency responses of the inner-loop sensitivity and co-sensitivity transfer functions when a 2nd order DOB is used

where B_W denotes the bandwidth of DOB. Figure 5 shows the frequency responses of the inner-loop sensitivity and co-sensitivity transfer functions when a second order DOB has different bandwidth values. The performance and robustness constraints, which are different from Eq. (47) due to conservatism, are obtained directly from Fig. 5 as follows:

$$w_\beta \leq 15 \text{ rad/s and } B_W < 65 \text{ rad/s} \quad (48)$$

The weighting function of the sensitivity transfer function is $W_S(s) = ((0.707s + 30)/(s + 2))$.

4.2 Plant With Time-Delay. Let us consider the time-delay constraints by using the following plant model:

$$G_n(s) = \frac{s + 10}{s^2 + 5s + 10} \text{ and } G(s) = G_n(s)(1 + \Delta W_T(s))e^{-0.01s} \quad (49)$$

where $W_T(s) = (3s + 240)/(s + 600)$. The bandwidth constraint of DOB is obtained by using the Theorem 2 as follows:

The design parameters α and $\sup_{w \in [w_\beta, R]} \log(|S_i(jw)|)$ should be determined by considering the performance and robustness design criteria. Besides, δ and R , which depend on the order of DOB, should be determined. Let us assume that $\alpha = 0.1$, $\sup_{w \in [w_\beta, R]} \log(|S_i(jw)|) = 2$ and the order of DOB is one. Since a first order DOB is used, if we take $\delta = 0.1$, then $R \cong B_W \delta^{-1} = 10 B_W$, where B_W is the bandwidth of DOB. If Eq. (26) is used, then $\sup_{w \in [w_\beta, R]} \log(|S_i(jw)|) \leq 2$ is satisfied for a wide range of DOB's bandwidth. If a second order DOB is used,

then $\sup_{w \in [w_\beta, R]} \log(|S(jw)|) \leq \sqrt{2}$ is satisfied for $B_W \leq 500 \text{ rad/s}$. Figure 6 shows the frequency responses of inner-loop sensitivity and co-sensitivity transfer functions. The real bandwidth constraint, which is significantly different from the derived one due to conservatism, is obtained as $B_W \leq 70 \text{ rad/s}$. The sensitivity weighting function is same as given above.

4.3 Plant With a RHP Zero. Let us consider the RHP zero constraints by using the following plant model:

$$G_n(s) = \frac{-s + 50}{s^2 + 25s + 40} \text{ and } G(s) = G_n(s)(1 + \Delta W_T(s)) \quad (50)$$

where $W_T(s) = ((3.75s + 450)/(s + 1500))$ and $\hat{G}_n \cong ((s^2 + 200s + 20)/((4s + 0.4)(s^2 + 25s + 40)))$ [31]. It is assumed that $\alpha_\beta = 0.5$, $\alpha_\gamma = 0.2$, $\|S\|_\infty = 2$, and $w_\gamma = 2B_W$. The performance and robustness constraints are derived by using the Theorem 3 as follows:

$$6 \leq B_W \leq 55, \quad w_\beta \leq 35 \text{ rad/s} \quad (51)$$

Figure 7 shows the frequency responses of inner-loop sensitivity and co-sensitivity transfer functions when a first order DOB is used. The performance and bandwidth constraints, which are different from Eq. (51) due to conservatism, are obtained directly from Fig. 7 as follows:

$$12 \leq BW \leq 24 \text{ and } w_\beta \leq 15 \text{ rad/s} \quad (52)$$

The sensitivity weighting function and the outer loop controller, which satisfies Eq. (30), are used given by $W_S(s) = 0.5((s + 16)/(s + 2))$ and $C(s) = 1 + 0.1s + 4/s$, respectively.

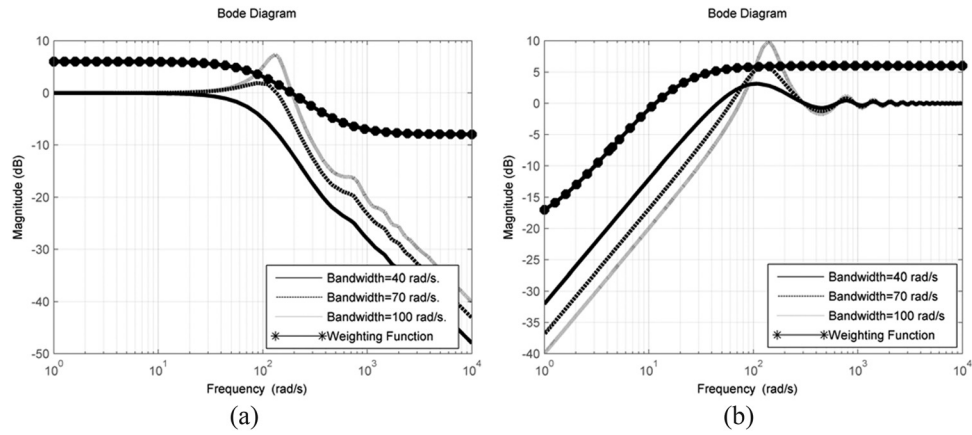


Fig. 6 Frequency responses of the inner-loop sensitivity and co-sensitivity transfer functions when the order of DOB is one

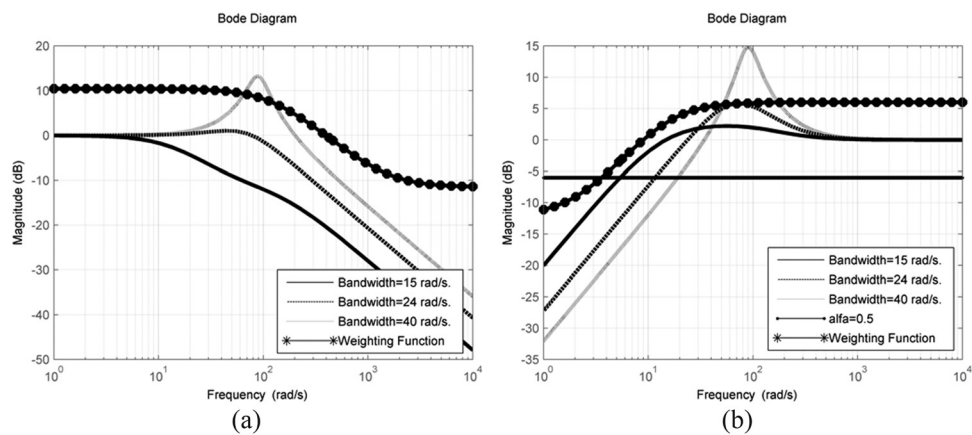


Fig. 7 Frequency responses of the inner-loop sensitivity and co-sensitivity transfer functions when the order of DOB is one

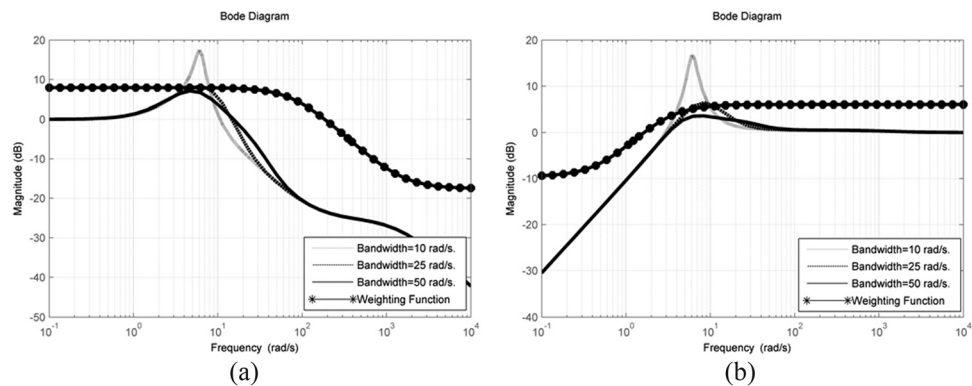


Fig. 8 Frequency responses of the outer-loop sensitivity and co-sensitivity transfer functions when the order of DOB is two

4.4 Plant With a RHP Pole. Let us consider the RHP pole constraints by using the following plant model:

$$G_n(s) = \frac{1}{s(s-5)} \text{ and } G(s) = G_n(s)(1 + \Delta W_T(s)) \quad (53)$$

where $W_T(s) = (7.5s + 600)/(s + 1500)$. Since the DOB is implemented in the inner-loop, the stabilizing outer loop control-

ler, $C_S(s) = 20 + 12s$, is designed by considering only the nominal plant model, and the robustness is achieved by using low control signals. The frequency responses of outer-loop sensitivity and co-sensitivity transfer functions can be seen in Fig. 8 when a second order DOB is used. It clearly shows that the robustness of the system improves as the bandwidth of DOB is increased. The lower bound on the bandwidth of DOB can be derived by using the Theorem 4 similarly; however, it also includes conservatism.

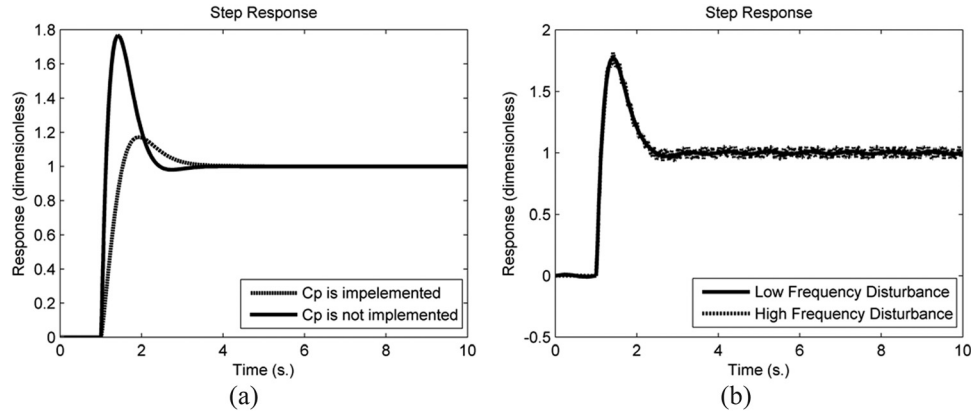


Fig. 9 Step responses of the unstable plant when it is controlled by using the proposed robust controller

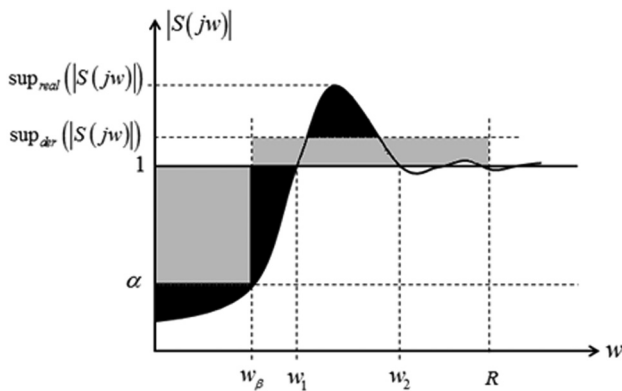


Fig. 10 A general frequency response of a sensitivity function

Figure 9(a) shows the performance improvement of the controller $C_P(s)$ which is designed as $C_P(s) = (s + 10)/(4s + 10)$. Figure 9(b) shows the step responses for different external disturbances. As it can be seen from the figure, the DOB cannot estimate high frequency disturbances precisely, so the robustness of the system deteriorates. There is a trade-off between the robustness and noise response to determine the bandwidth of DOB.

4.5 Comments on Conservatism. The source of conservatism, i.e., the approximate sensitivity/co-sensitivity bounds, can be seen in Fig. 10. In this figure, the grey areas are determined by the sectionally constant sensitivity bounds, and the black areas denote the errors which cannot be considered in the robustness analysis. It clearly shows that there is a significant difference between the areas, which are bounded by the real sensitivity function and its approximate bound.

Let us again consider the plant with time-delay example, which has the most severe conservative result, to obtain more accurate bandwidth constraint by decreasing the conservatism. If we consider the nominal plant model with time-delay, then the sensitivity function frequency response is derived as follows:

$$|S(jw)|^2 = \frac{w^2}{w^2 - 2gw \sin(w\tau) + g^2} \quad (54)$$

where g is the bandwidth of the first order DOB; and τ is delay time. The frequencies w_1 and w_2 , given in Fig. 10, can be obtained approximately as follows:

$$w_1 = \sqrt{\frac{3\tau - 1.73205\sqrt{\tau^2(3 - g_0\tau)}}{\tau^3}}, \quad (55)$$

$$w_2 = \sqrt{\frac{3\tau + 1.73205\sqrt{\tau^2(3 - g_0\tau)}}{\tau^3}}$$

The conservatism can be decreased by using w_1 and w_2 instead of R . If we use the sectionally constant sensitivity bound with w_1 and w_2 , then the bandwidth limitation of DOB, which is obtained as 70 rad/s in the second example, is derived as 95 rad/s. Hence, the conservatism can be lessened by considering more realistic sensitivity bounds.

5 Conclusion

This paper has been concerned with the problem of robustness and performance constraints in the design of DOB based control systems. The bandwidth constraints of DOB are derived analytically by using the Bode and Poisson integral formulas, and new analysis and design tools are proposed. The proposed tools include conservatism; however, it can be lessened by using a more realistic sensitivity/co-sensitivity bound, which increases the complexity of analysis, as shown in the paper. The experiences with the proposed tools showed us that the conservatism is not a severe problem, since the proposed tools give a deep insight into the design constraints of DOB based control systems. Therefore, the analysis and design tools are very useful, and they can be easily implemented into many different DOB based robust control problems.

Acknowledgment

This research was supported in part by the Ministry of Education, Culture, Sports, Science and Technology of Japan under Grant-in-Aid for Scientific Research (S), 25220903, 2013.

References

- [1] Ohishi, K., Ohnishi, K., and Miyachi, K., 1983, "Torque-Speed Regulation of dc Motor Based on Load Torque Estimation," *Proc. IEEE IPEC-TOKYO*, 2, pp. 1209–1216.
- [2] Ohnishi, K., Shibata, M., and Murakami, T., 1996, "Motion Control for Advanced Mechatronics," *IEEE/ASME Trans. Mechatron.*, 1(1), pp. 56–67.
- [3] Schrijver, E., and Johannes, D. V., 2002, "Disturbance Observers for Rigid Mechanical Systems: Equivalence, Stability, and Design," *ASME J. Dyn. Syst., Meas., Control*, 124(4), pp. 539–548.
- [4] Ohishi, K., Miyazaki, T., and Nakamura, Y., 1995, "Two-Degrees-of-Freedom Speed Controller Based on Doubly Coprime Factorization and Speed Observer," 21st IEEE International Conference on Industrial Electronics, Control, and Instrumentation, Orlando, FL, 1, pp. 602–608.

- [5] Umeno, T., and Hori, Y., 1991, "Robust Speed Control of dc Servomotors Using Modern Two Degrees-Of-Freedom Controller Design," *IEEE Trans. Ind. Electron. Control Instrum.*, 38(5), pp. 363–368.
- [6] Guo, L., and Tomizuka, M., 1997, "High-Speed and High-Precision Motion Control With an Optimal Hybrid Feed Forward Controller," *IEEE/ASME Trans. Mechatron.*, 2(2), pp. 110–122.
- [7] Guvenç, B. A., Guvenç, L., and Karaman, S., 2010, "Robust MIMO Disturbance Observer Analysis and Design With Application to Active Car Steering," *Int. J. Robust Nonlinear Control*, 20(8), pp. 873–891.
- [8] Chan, S. P., 1991, "A Disturbance Observer for Robot Manipulators With Application to Electronic Components Assembly," *IEEE Trans. Ind. Electron. Control Instrum.*, 42(5), pp. 487–493.
- [9] Wang, C., and Tomizuka, M., 2004, "Design of Robustly Stable Disturbance Observers Based on Closed Loop Consideration Using H_∞ Optimization and its Applications to Motion Control Systems," American Control Conference (ACC), Boston, MA, 4, pp. 3764–3769.
- [10] Sariyildiz, E., and Ohnishi, K., 2013, "Analysis the Robustness of Control Systems Based on Disturbance Observer," *Int. J. Control*, 86, pp. 1733–1743.
- [11] Yang, W. C., and Tomizuka, M., 1994, "Disturbance Rejection Through an External Model for Non-Minimum Phase Systems," *ASME J. Dyn. Syst., Meas., Control*, 116(1), pp. 39–44.
- [12] Jo, N. H., Shim, H., and Son, Y. I., 2010, "Disturbance Observer for Non-minimum Phase Linear Systems," *Int. J. Control, Autom. Syst.*, 8(5), pp. 994–1002.
- [13] Chen, X., Zhai, G., and Fukuda, T., 2004, "An Approximate Inverse System for Non-Minimum-Phase Systems and Its Application to Disturbance Observer," *Syst. Control Lett.*, 52(3–4), pp. 193–207.
- [14] Katsura, S., Irie, K., and Ohishi, K., 2008, "Wideband Force Control by Position-Acceleration Integrated Disturbance Observer," *IEEE Trans. Ind. Electron. Control Instrum.*, 55(5), pp. 1699–1706.
- [15] Ishikawa, J., and Tomizuka, M., 1998, "Pivot Friction Compensation Using and Accelerometer and a Disturbance Observer for Hard Disk Drives," *IEEE/ASME Trans. Mechatron.*, 3(3), pp. 194–201.
- [16] Tsuji, T., Hashimoto, T., Kobayashi, H., Mizuochi, M., and Ohnishi, K., 2009, "A Wide-Range Velocity Measurement Method for Motion Control," *IEEE Trans. Ind. Electron. Control Instrum.*, 56(2), pp. 510–519.
- [17] Jeon, S., and Tomizuka, M., 2007, "Benefits of Acceleration Measurement in Velocity Estimation and Motion Control," *Control Eng. Practice*, 15(3), pp. 325–332.
- [18] Mitsantisuk, C., Ohishi, K., and Katsura, S., 2012, "Estimation of Action/Reaction Forces for the Bilateral Control Using Kalman Filter," *IEEE Trans. Ind. Electron.*, 59(11), pp. 4383–4393.
- [19] Choi, Y., Yang, K., Chung, W. K., Kim, H. R., and Suh, H., 2003, "On the Robustness and Performance of Disturbance Observers for Second-Order Systems," *IEEE Trans. Autom. Control*, 48(2), pp. 315–320.
- [20] Shima, H., and Jo, N. H., 2009, "An Almost Necessary and Sufficient Condition For Robust Stability of Closed-Loop Systems With Disturbance Observer," *Automatica*, 45(1), pp. 296–299.
- [21] Sariyildiz, E., and Ohnishi, K., 2013, "Bandwidth Constraints of Disturbance Observer in the Presence of Real Parametric Uncertainties," *Eur. J. Control*, 19(3), pp. 199–205.
- [22] Bode, H. W., 1945, *Network Analysis and Feedback Amplifier Design*, D. Van Nostrand Co., New York.
- [23] Stein, G., 2003, "Respect the Unstable," *IEEE Control Syst. Mag.*, 23(4), pp. 12–25.
- [24] Horowitz, I. M., 1963, *Synthesis of Feedback Systems*, Academic Press, New York.
- [25] Freudenberg, J. S., and Looze, D. P., 1987, "A Sensitivity Tradeoff for Plants With Time Delay," *IEEE Trans. Autom. Control*, 32(2), pp. 99–104.
- [26] Freudenberg, J. S., and Looze, D. P., 1985, "Right Half Plane Poles and Zeros and Design Tradeoffs in Feedback Systems," *IEEE Trans. Autom. Control*, 30(6), pp. 555–565.
- [27] Skogestad, S., and Postlethwaite, I., 2001, *Multivariable Feedback Control: Analysis and Design*, 2nd ed., John Wiley & Sons, New York.
- [28] Zhou, K., and Doyle, J. C., 1997, *Essentials of Robust Control*, Prentice-Hall, Upper Saddle River, NJ.
- [29] Middleton, R. H., and Goodwin, G. C., 1990, *Digital control and estimation. A unified approach*, Prentice-Hall, inc., Englewood Cliffs, NJ.
- [30] Seron, M. M., Braslavsky, J. H., and Goodwin, G. C., 1997, *Fundamental Limitations in Filtering and Control*, Springer-Verlag, London.
- [31] Sariyildiz, E., and Ohnishi, K., 2013, "A New Solution for the Robust Control Problem of Non-Minimum Phase Systems Using Disturbance Observer," *IEEE International Conference on Mechatronics (ICM)*, Vicenza, Italy, pp. 46–51.