

Algorithms in FAST v8

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1 Definitions and Nomenclature

Module Name	Abbreviation in Module	Abbreviation in this Document
ElastoDyn	ED	ED
AeroDyn	AD	AD
ServoDyn	SrvD	SrvD
SubDyn	SD	SD
HydroDyn	HydroDyn	HD
MAP	MAP	MAP
FEAMooring	FEAM	FEAM
InflowWind	IfW	IfW
IceFloe	IceFloe	IceF

Table 1: Abbreviations for modules in FAST v8

2 Initializations

3 Input-Output Relationships

3.1 Input-Output Solves (Option 2 Before 1)

This algorithm documents the procedure for the Input-Output solves in FAST, assuming all modules are in use. If an individual module is not in use during a particular simulation, the calls to that module's subroutines are omitted and the module's inputs and outputs are neither set nor used.

```

1: procedure CALCOUTPUTS_AND_SOLVEFORINPUTS()
2:    $y\_ED \leftarrow \text{ED\_CALCOUTPUT}(p\_ED, u\_ED, x\_ED, xd\_ED, z\_ED)$ 
3:
4:    $u\_AD \leftarrow \text{TRANSFEROUTPUTSTOINPUTS}(y\_ED)$ 
5:    $y\_AD \leftarrow \text{AD\_CALCOUTPUT}(p\_AD, u\_AD, x\_AD, xd\_AD, z\_AD)$ 
6:
7:    $u\_SrvD \leftarrow \text{TRANSFEROUTPUTSTOINPUTS}(y\_ED, y\_AD)$ 
8:    $y\_SrvD \leftarrow \text{SRVD\_CALCOUTPUT}(p\_SrvD, u\_SrvD,$ 
                                    $x\_SrvD, xd\_SrvD, z\_SrvD)$ 
9:
10:   $u\_ED(\text{not platform reference point}) \leftarrow \text{TRANSFEROUTPUTSTOINPUTS}(y\_SrvD, y\_AD)$ 
11:   $u\_HD \leftarrow \text{TRANSFERMESHMOTIONS}(y\_ED)$ 
12:   $u\_SD \leftarrow \text{TRANSFERMESHMOTIONS}(y\_ED)$ 
13:   $u\_MAP \leftarrow \text{TRANSFERMESHMOTIONS}(y\_ED)$ 
14:   $u\_FEAM \leftarrow \text{TRANSFERMESHMOTIONS}(y\_ED)$ 
15:
16:   $\text{ED\_HD\_SD\_MOORING\_ICE\_INPUTOUTPUTSOLVE}()$ 
17:
18:  If AeroDyn or ServoDyn had states to update, we should do this:
19:     $u\_AD \leftarrow \text{TRANSFEROUTPUTSTOINPUTS}(y\_ED)$ 
20:     $u\_SrvD \leftarrow \text{TRANSFEROUTPUTSTOINPUTS}(y\_ED, y\_AD)$ 
21:    However, they don't so we'll omit these steps for efficiency.
22: end procedure
```

Note that inputs to *ElastoDyn* before calling `CalcOutput()` in the first step are not set in `CalcOutputsAndSolveForInputs()`. Instead, the *ElastoDyn* inputs are set depending on where `CalcOutputsAndSolveForInputs()` is called:

- At time 0, the inputs are the initial guess from *ElastoDyn*;
- On the prediction step, the inputs are extrapolated values from the time history of *ElastoDyn* inputs;
- On the first correction step, the inputs are the values calculated in the prediction step;
- On subsequent correction steps, the inputs are the values calculated in the previous correction step.

3.2 Input-Output Solve for *HydroDyn*, *SubDyn*, *MAP*, *FEAMooring*, *IceFloe*, and the Platform Reference Point Mesh in *ElastoDyn*

This procedure implements Solve Option 1 for the accelerations and loads in *HydroDyn*, *SubDyn*, *MAP*, *FEAMooring*, and *ElastoDyn* (at its platform reference point mesh). The other input-output relationships for these modules are solved using Solve Option 2.

```

1: procedure ED_HD_SD_MOORING_ICE_INPUTOUTPUTSOLVE()
2:
3:    $y\_MAP \leftarrow \text{CALCOUTPUT}(p\_MAP, u\_MAP, x\_MAP, xd\_MAP, z\_MAP)$ 
4:    $y\_FEAM \leftarrow \text{CALCOUTPUT}(p\_FEAM, u\_FEAM, x\_FEAM, xd\_FEAM, z\_FEAM)$ 
5:    $y\_IceF \leftarrow \text{CALCOUTPUT}(p\_IceF, u\_IceF, x\_IceF, xd\_IceF, z\_IceF)$ 
6:
7:   ▷ Form  $u$  vector using loads and accelerations from  $u\_HD$ ,  $u\_SD$ , and
   platform reference input from  $u\_ED$ 
8:
9:    $u \leftarrow \text{U\_VEC}(u\_HD, u\_SD, u\_ED)$ 
10:   $k \leftarrow 0$ 
11:  loop   ▷ Solve for loads and accelerations (direct feed-through terms)
12:     $y\_ED \leftarrow \text{ED\_CALCOUTPUT}(p\_ED, u\_ED, x\_ED, xd\_ED, z\_ED)$ 
13:     $y\_SD \leftarrow \text{SD\_CALCOUTPUT}(p\_SD, u\_SD, x\_SD, xd\_SD, z\_SD)$ 
14:     $y\_HD \leftarrow \text{HD\_CALCOUTPUT}(p\_HD, u\_HD, x\_HD, xd\_HD, z\_HD)$ 
15:    if  $k \geq k\_max$  then
16:      exit loop
17:    end if
18:     $u\_MAP\_tmp \leftarrow \text{TRANSFERMESHMOTIONS}(y\_ED)$ 
19:     $u\_FEAM\_tmp \leftarrow \text{TRANSFERMESHMOTIONS}(y\_ED)$ 
20:     $u\_IceF\_tmp \leftarrow \text{TRANSFERMESHMOTIONS}(y\_SD)$ 
21:     $u\_HD\_tmp \leftarrow \text{TRANSFERMESHMOTIONS}(y\_ED, y\_SD)$ 
22:     $u\_SD\_tmp \leftarrow \text{TRANSFERMESHMOTIONS}(y\_ED)$ 
         $\cup \text{TRANSFERMESHLOADS}(y\_SD,$ 
                                    $y\_HD, u\_HD\_tmp,$ 
                                    $y\_IceF, u\_IceF\_tmp)$ 
23:     $u\_ED\_tmp \leftarrow \text{TRANSFERMESHLOADS}(y\_ED,$ 
                                    $y\_HD, u\_HD\_tmp,$ 
                                    $y\_SD, u\_SD\_tmp,$ 
                                    $y\_MAP, u\_MAP\_tmp,$ 
                                    $y\_FEAM, u\_FEAM\_tmp)$ 
24:
25:     $U\_Residual \leftarrow u - \text{U\_VEC}(u\_HD\_tmp, u\_SD\_tmp, u\_ED\_tmp)$ 
26:
27:    if last Jacobian was calculated at least  $DT\_UJac$  seconds ago then
28:      Calculate  $\frac{\partial U}{\partial u}$ 

```

```

29:     end if
30:     Solve  $\frac{\partial U}{\partial u} \Delta u = -U\_Residual$  for  $\Delta u$ 
31:
32:     if  $\|\Delta u\|_2 < \text{tolerance}$  then                                 $\triangleright$  To be implemented later
33:         exit loop
34:     end if
35:
36:      $u \leftarrow u + \Delta u$ 
37:     Transfer  $u$  to  $u\_HD$ ,  $u\_SD$ , and  $u\_ED$   $\triangleright$  loads and accelerations only
38:      $k = k + 1$ 
39: end loop
40:                                      $\triangleright$  Transfer non-acceleration fields to motion input meshes
41:
42:      $u\_HD(\text{not accelerations}) \leftarrow \text{TRANSFERMESHMOTIONS}(y\_ED, y\_SD)$ 
43:      $u\_SD(\text{not accelerations}) \leftarrow \text{TRANSFERMESHMOTIONS}(y\_ED)$ 
44:
45:      $u\_MAP \leftarrow \text{TRANSFERMESHMOTIONS}(y\_ED)$ 
46:      $u\_FEAM \leftarrow \text{TRANSFERMESHMOTIONS}(y\_ED)$ 
47:      $u\_IceF \leftarrow \text{TRANSFERMESHMOTIONS}(y\_SD)$ 
48: end procedure

```

3.3 Implementation of line2-to-line2 loads mapping

The inverse-lumping of loads is computed by a block matrix solve for the distributed forces and moments, using the following equation:

$$\begin{bmatrix} F^{DL} \\ M^{DL} \end{bmatrix} = \begin{bmatrix} A & 0 \\ B & A \end{bmatrix} \begin{bmatrix} F^D \\ M^D \end{bmatrix} \quad (1)$$

Because the forces do not depend on the moments, we first solve for the distributed forces, F^D :

$$[F^{DL}] = [A] [F^D] \quad (2)$$

We then use the known values to solve for the distributed moments, M^D :

$$[M^{DL}] = [B \quad A] \begin{bmatrix} F^D \\ M^D \end{bmatrix} = [B] [F^D] + [A] [M^D] \quad (3)$$

or

$$[M^{DL}] - [B] [F^D] = [A] [M^D] \quad (4)$$

Rather than store the matrix B , we directly perform the cross products that the matrix B represents. This makes the left-hand side of Equation 4 known, leaving us with one matrix solve. This solve uses the same matrix A used to obtain the distributed forces in Equation 2; A depends only on element connectivity (***bjj: check that this is true***). We use the LU factorization of matrix A so that the second solve does not introduce much additional overhead.