



# VAPOUR COMPRESSION SYSTEM, VCR

ME441, TFE Laboratory, UG Course + RAC Laboratory, RAC\_1 of 2+Vapour Compression Unit\_with\_Single\_Stage+R134a

*Thermal and Fluid Engineering Laboratory*



- **Aim**

- ✓ Understand the design and operation of a *Vapour Compression Refrigeration System*

- **Objective**

- ✓ *Verify Energy Balance in a Single Stage VCR System*  
@ various evaporator loads
- ✓ *Plot the variation of*
  - *Overall COP V/s Evaporator Temperature,  $t_{evp.sat}$*
  - *Refrigerant flow rate V/s Overall COP*
  - *Superheat V/s Evaporator Temperature,  $t_{evp.sat}$*
  - *Refrigeration Load & Power Input V/s Evaporator Temperature,  $t_{evp.sat}$*

- **Outline of the Presentation**

- ✓ Theory: Simple VCR Cycle and Components
- ✓ Discussion on P A Hilton experiment set up
- ✓ Experimental set up components and their working functions
- ✓ Discussion on the fundamental principle behind the single stage vapor compression
- ✓ Advantages and disadvantages
- ✓ The parameters to be measured
- ✓ Discussion on p-h diagram of R134a refrigerant and possible interpretations



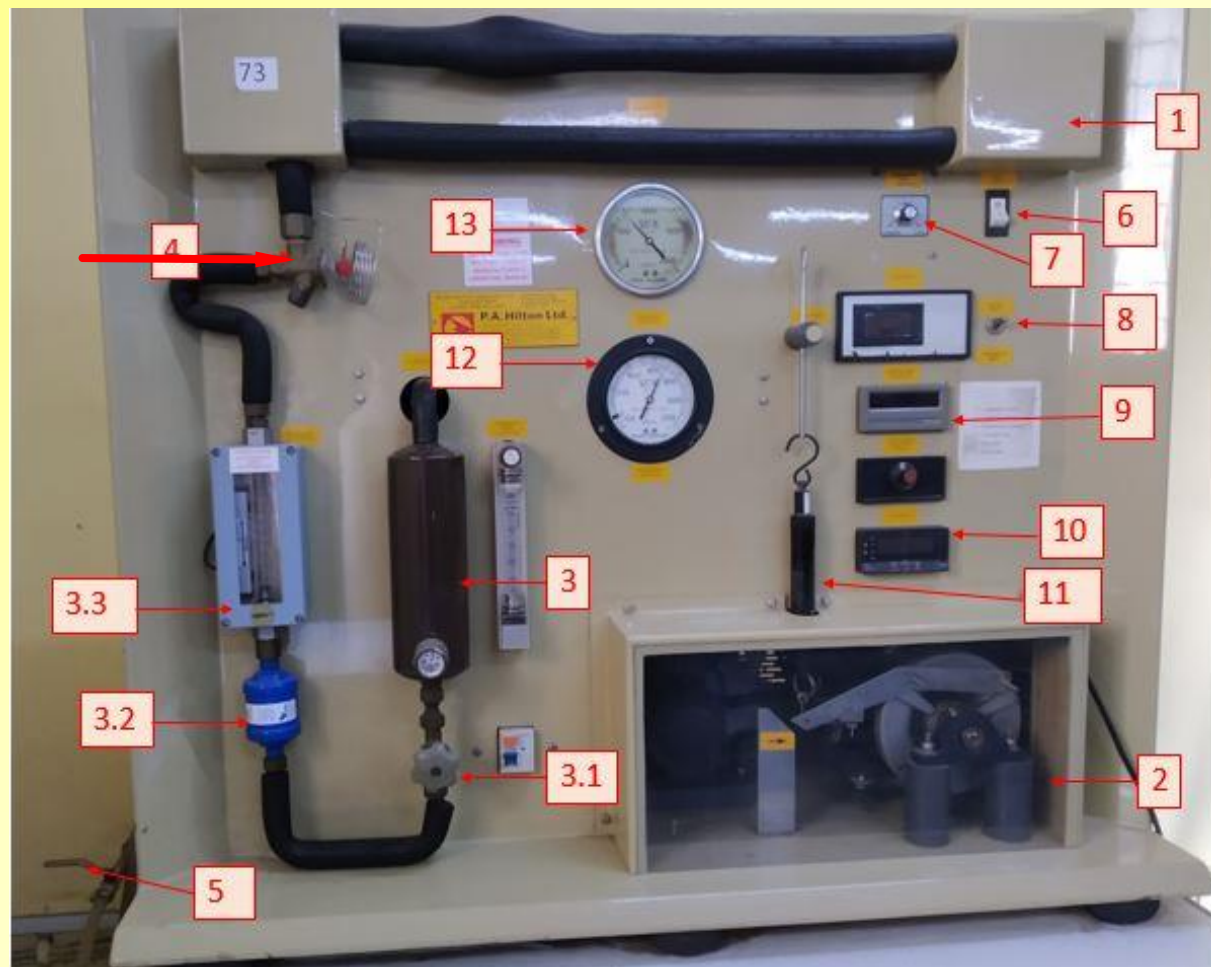
# APPARATUS

P A Hilton R714/28333 Refrigeration Laboratory Unit

## Component Description



- 1 *Evaporator with Electric Heater*
- 2 *Compressor with External Motor*
- 3 *Water Cooled Condenser with Sight Glass*
  - 3.1 *Isolation Valve*
  - 3.2 *Filter cum Dryer*
  - 3.3 *Rotameter for Refrigerant Liquid*
- 4 *Thermostatic Expansion Valve, TEV*
- 5 *Condenser Cooling Water Valve*
- 6 *Main Switch: Power Supply to the Unit*
- 7 *Evaporator Input Control: EVP Power*
- 8 *Watt Meter & Toggle Switch*  
*Enables Measuring Motor and EVP Power*
- 9 *Temperature Indicator & Section Switch:*  
*Enables Measuring Temperatures*
- 10 *Tachometer: Indicates Compressor Speed*
- 11 *Dynamometer: Enables Measuring Torque*
- 12 *Evaporator Pressure Gauge*
- 13 *Condenser Pressure Gauge*
- 14 *Rotameter for Condenser Cooling Water*





# PROCESS SCHEMATIC

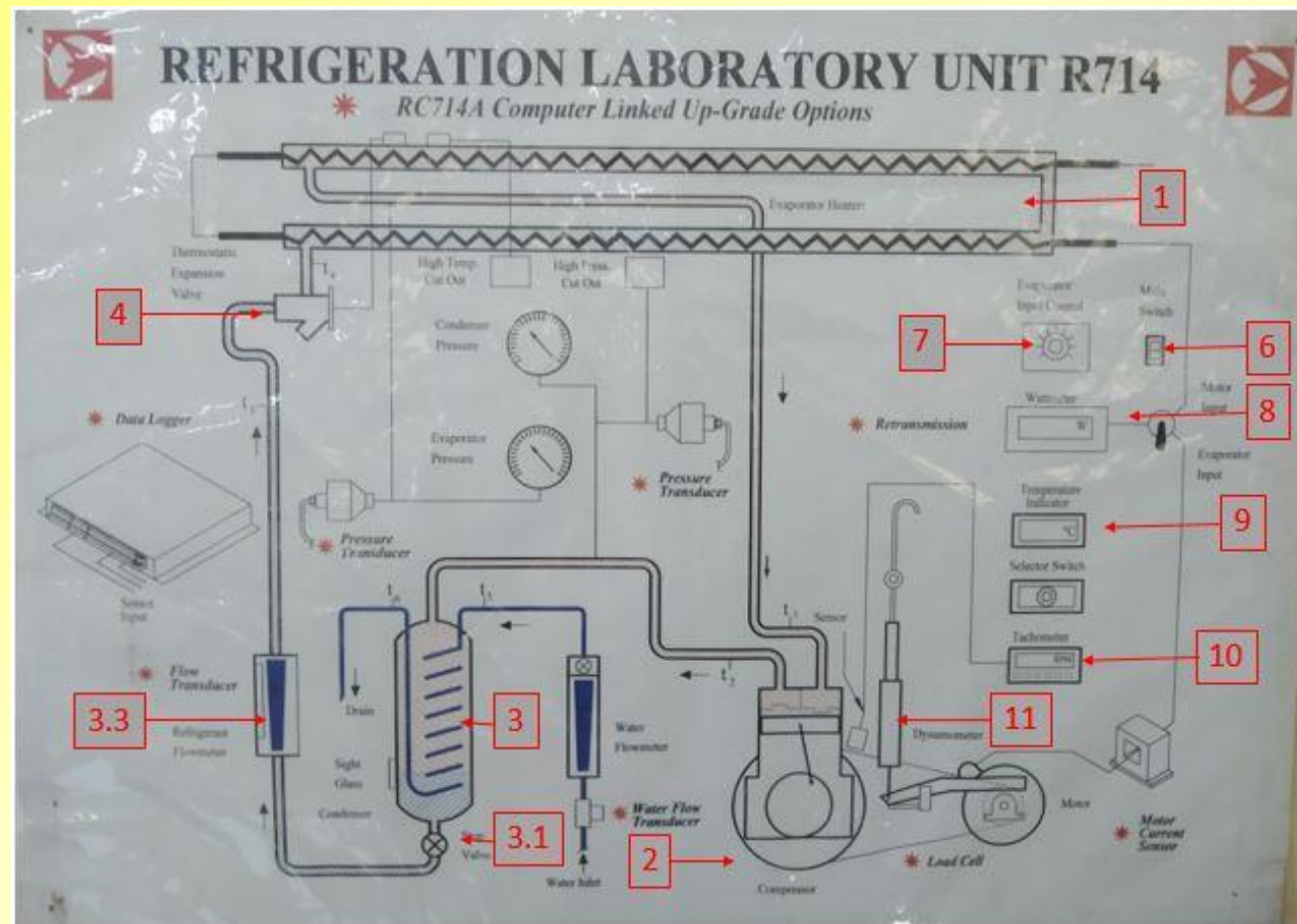
P A Hilton R714/28333 Refrigeration Laboratory Unit

## Component Description in Layout



HPL\_IITB

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# APPARATUS

P A Hilton R714/28333 Refrigeration Laboratory Unit  
*Important Parts: Evaporator, EVP*



HPL\_IITB

- *A Tubular Evaporator, 1, is Deployed in this Experimental Setup*

- ✓ Heat load to the evaporator is provided by an Electric Heater, within the evaporator, 1
- ✓ Additional heat load to the evaporating refrigerant is due to the heat gain through the insulation around the evaporator



- *Heat Load through the Electric Heater can be Varied by Turning Knob 7*

- ✓ This varies the power input to the electric heater and as a result the evaporator load



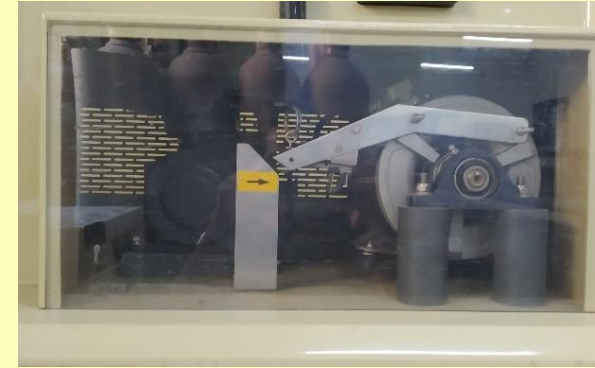
# APPARATUS

P A Hilton R714/28333 Refrigeration Laboratory Unit

## *Important Parts: Open Type Compressor with External Motor*



- *Power from the Motor is Transmitted to the Compressor Using a Belt Drive*
  - ✓ Diameter of the pulley mounted on the motor and the diameter of the pulley mounted on the compressor govern the compressor speed
    - *Speed of the motor and the pulley diameter ration (1.98 in the resent case) decides the speed of the compressor*
- *Compressor Power Measurement can be Enabled by Measuring the Following:*
  - ✓ Motor Speed, Torque on the Motor Shaft
- *Motor Speed is Measured Using Tachometer*
- *Torque on the Motor Shaft is Measured Using the Dynamometer*
  - ✓ Adjusting the sharp edge of the lever end of the dynamometer with the marked arrow next to it and then reading the tension in the force measuring gauge
  - ✓ Multiplying the force with the length of the arm gives the torque
  - ✓ Using the torque and motor rpm the power delivered by the motor can be calculated
- *Electrical Power Input to the Motor can be Read from the Wattmeter*





# APPARATUS

P A Hilton R714/28333 Refrigeration Laboratory Unit

**Important Parts: Condenser, CND, Water Cooled**



HPL\_IITB



- **Water Cooled Condenser, WCC**
  - ✓ Desuperheats, condenses and subcools compressed refrigerant exiting the compressor
- **Water Cooled Condensers Typically Offer Lower Condensing Temperatures as Compared to Air Cooled Condensers**
  - ✓ Cooling water inlet temperature is typically lower than ambient air temperature
  - ✓ Heat transfer coefficient with water is better than that with ambient air
- **Water Cooled Condensers Typically are More Compact**
  - ✓ Heat transfer coefficient with water is better than that with ambient air
  - ✓ In the present system the area of condenser coil,  $A_{he.cnd} = 0.075 \text{ m}^2$
- **Water Flow Through the WCC can be Measured Using the Rotameter**
  - ✓ Range of the rotameter is from 4 to 50 g/s & Least Count is 2 g/s
  - ✓ Water flow for the experiments are typically set at 30 g/s
    - *After setting a value the water flow may vary due to variation in tap water pressure*



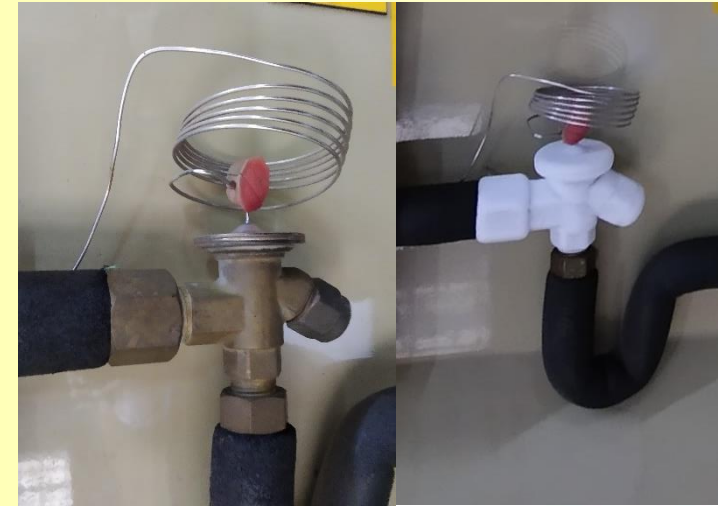
# APPARATUS

P A Hilton R714/28333 Refrigeration Laboratory Unit

## *Important Parts: Thermostatic Expansion Valve , TEV*



- ***TEV Regulates the Refrigerant Flow***
  - ✓ from the Condenser Outlet to the Evaporator Inlet
- ***TEV Regulates the Refrigerant Flow to Maintain Super Heat at the Outlet of the Evaporator or Inter of the Compressor***
  - ✓ Designed to keep the superheating of the refrigerant at a constant level
  - ✓ Amount of superheat can be set by setting the adjustment screw provided in the TEV
    - *Normally this is factory set and it is not recommended to be tampered with at site*
- ***Bulb Type Temperature Sensor Located at the Outlet of the Evaporator is Connected to the Valve by a Capillary***
  - ✓ Increase in superheat above the set value, increases the pressure in the sensor and the valve opens up to regulate the refrigerant flow
- ***TEV is Expected to Work as an Adiabatic Expansion Device***
  - ✓ It should be ideally insulated to avoid the frosting seen in the setup







# THEORY

ME 441 Lab Manual

## Vapour Compression Refrigeration System, VCRS



- ***VCRS is the Most Commonly Used Refrigeration System***
  - ✓ Refrigeration/cooling effect is achieved due to evaporation of the refrigerant in the evaporator
    - *Evaporation takes place at low temperatures and its vapour pressure is low*
  - ✓ This low pressure refrigerant is compressed in the compressor
    - *Higher pressure refrigerant at the outlet of the compressor is then condensed in the condenser*
  - ✓ Condensation at higher pressure takes place as higher temperature
    - *Enabling heat transfer to is the surrounding of suitable heat sink*
  - ✓ Condensed refrigerant from the outlet of the condenser is throttled through the Expansion Device and let in to the evaporator
- ***VCRS Works on the Vapour Compression Cycle (VCC) it Consist of Four Steps***
  - *Isobaric heat extraction in the evaporator*
  - *Isentropic compression*
  - *Isobaric heat rejection in condenser*
  - *Isenthalpic expansion in expansion device*
- ***VCC Introduces Two Irreversibilities as Compared to Carnot Cycle***
  - ✓ Irreversibility due to non-isothermal heat rejection & that due to isenthalpic throttling
- ***Due to These Irreversibilities, the Cooling Effect Reduces and Work Input Increases***
  - ✓ COP of the VCRS reduces as compared to Carnot cycle





# OBSERVATION TABLE

ME 441 Lab Manual

## Vapour Compression Refrigeration system, VCRS



Sr #	Measurements	Units	1	2	3	4	5	6
1	$p_c$	$\text{kN/m}^2$	800	850	950	970	975	1050
2	$p_e$	$\text{kN/m}^2$	100	125	175	200	225	250
3	$t_1$	$^{\circ}\text{C}$	7.0	-0.1	2.2	5.2	6.1	9.6
4	$t_2$	$^{\circ}\text{C}$	65.0	72.7	63.2	68.8	60.2	70.1
5	$t_3$	$^{\circ}\text{C}$	29.6	30.5	32.5	34.2	34.3	36.4
6	$t_4$	$^{\circ}\text{C}$	-29.0	-23.7	-15.3	-10.6	-8.2	-5.2
7	$t_5$	$^{\circ}\text{C}$	28.2	27.5	27.6	28	26.8	28.1
8	$t_6$	$^{\circ}\text{C}$	30.9	31.1	32.7	33.5	33.0	35.0
9	$m_w$	$\text{g/sec}$	30.0	30	32	32	32 0	34



# OBSERVATION TABLE

ME 441 Lab Manual

## Vapour Compression Refrigeration system, VCRS



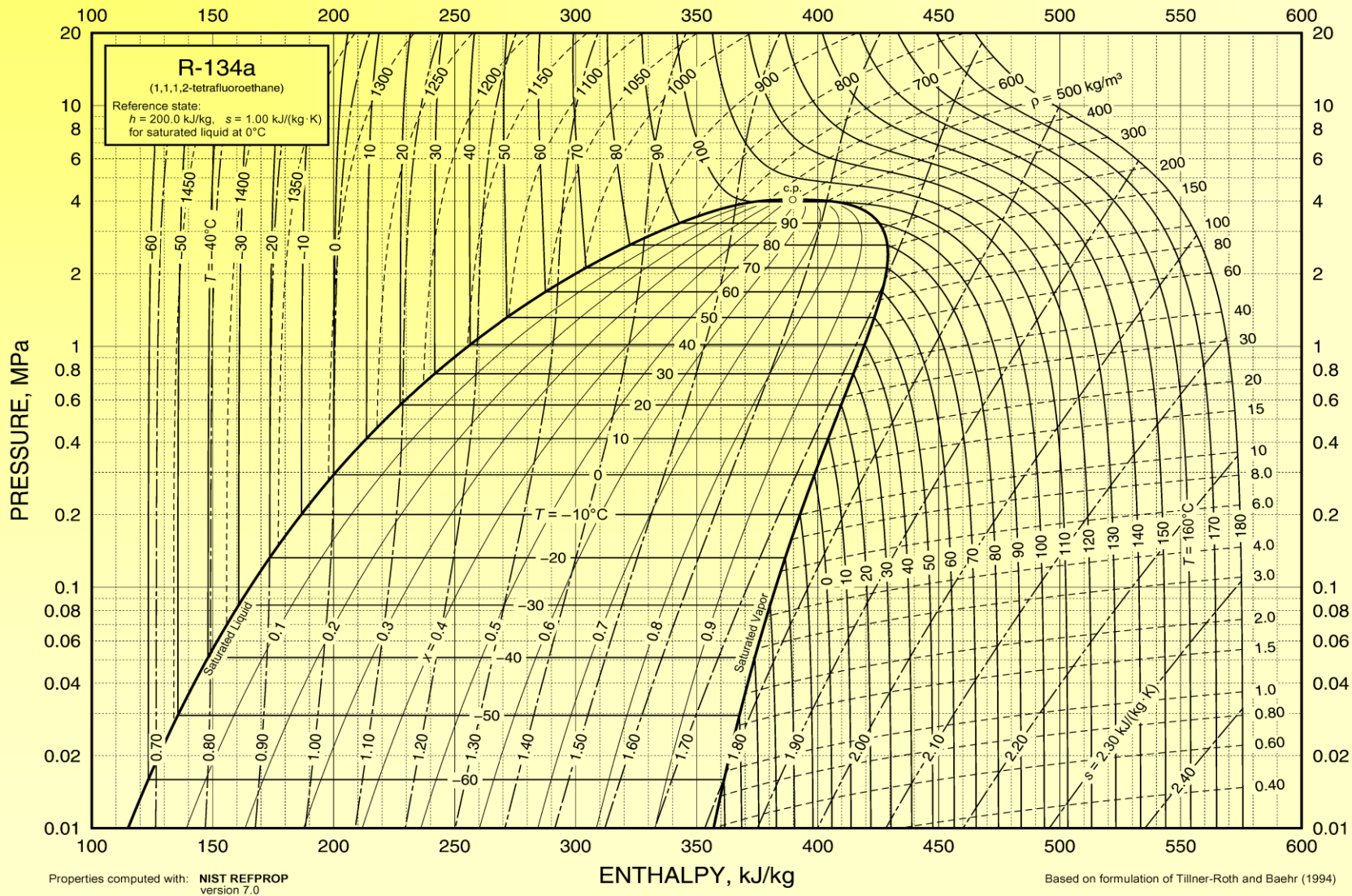
Sr #	Measurements	Units	1	2	3	4	5	6
10	$m_r$	g/s	1.9	2.2	3.2	3.8	4.2	4.5
11	$Q_{e_{el}}$	W	126	210	400	505	603	686
12	$Q_{m_{el}}$	W	343	373	424	451	458	493
13	F	N	10.0	10.5	12.0	12.5	13.0	14.0
14	$n_c$	rpm	742	743	743	743	743	743
15	$n_m$	rpm	1470	1472	1472	1473	1472	1473



# p – h CHART

ASHRAE HBF 2017, Ch 30, P 30.17, Fig 8 + Safety Group A1, GWP-AR5 1300, ODP 0

***R-134a, HFC, 1,1,1,2-Tetrafluoroethane, CH<sub>2</sub>FCF<sub>3</sub>***







# CALCULATION

ME 441 Lab Manual

## Vapour Compression Refrigeration system, VCRS



- **Refrigeration Load**

$$Q_e = m_r (h_1 - h_4)$$

- **Shaft Power**

$$P_s = T\omega$$

$$\omega = 2\pi N_m / 60$$

$$T = F \times 0.165 \text{ (arm length for dynamometer)}$$

- **Friction Power**

$$P_f = 0.165 \times F_f \times \omega$$

$$F_f = 5N$$

- **Indicated Power**

$$P_i = P_s - P_f$$

- **COP Based on Electrical Power or Overall COP**

$$COP_{ep} = Q_e / Qm_{el}$$

- **COP Based on Shaft Power**

$$COP_{ep} = Q_e / P_s$$



# CALCULATION

ME 441 Lab Manual

## *Vapour Compression Refrigeration system, VCRS*



- *COP Based on Indicated Power*

$$COP_{ip} = Q_e / P_i$$

- *Volumetric efficiency*

$$\eta_{vol} = (m_r \times v_1) / V_{swept}$$

$$V_{swept} = 43.0 \text{ cm}^3/\text{rev} \times \text{rps of compressor}$$

- *Heat Balance*

$$W_{in} = P_i \times \eta_{vol}$$

$$Q_{cnd} = m_w C_p (t_6 - t_5)$$

$$W_{in} + Q_e = Q_{cnd}$$

**(Verify Energy Balance)**



# REFRIGERANTS

ASHRAE HBF 2013, Ch 29, 30



Refrigerant	Name	ODP	GWP <sub>100</sub>	<b>p<sub>r</sub></b>			<b>t<sub>r,critical</sub></b> °C	Issues
				at 7.2°C	at 54.4°C	critical		
				bar	bar	bar		
R-22	Dichlorofluoromethane	0.04	<b>1790</b>	6.3	21.3	50.0	96.1	High GWP
R-134a	Tetrafluoroethane	0	<b>1370</b>	3.8	14.6	40.6	101.1	High GWP
R-290	Propane	0	~20	5.9	19.1	42.5	96.7	Flammable
R-600	Butane	0	~20	1.3	5.5	38.0	152.0	Flammable
R-407C	R-32/125/134a (23/25/52)	0	<b>1700</b>	6.4	24.0	46.3	86.0	High GWP
R-410A	R-32/125 (50/50)	0	<b>2100</b>	10.0	34.0	49.0	71.4	High GWP
R-717	Ammonia	0	< 1	5.6	22.0	113.3	132.3	Toxic
R-744	Carbon Dioxide	0	1	42.1	<i>SH</i>	<b>73.8</b>	<b>31.0</b>	High p <sub>r</sub> , Low t <sub>critical</sub>
R-718	Water/Steam			0.010	0.140	220.6	373.3	Very low pressure, low density