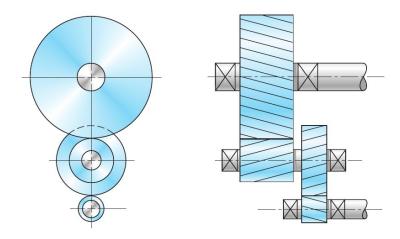
ME423 - Tutorial 8/ HW 8 - Due 22nd November

You have to submit all the problems by 11.00 pm, 22/11/2024 [Moodle].

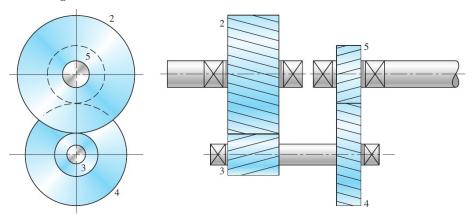
1. As a rough guideline a reduction ratio (speed ratio, gear ratio) of pair of gears should not exceed 10:1. Note reduction ratio = number of teeth on the driven gear/number of teeth of the driver gear. To obtain higher reduction ratios, one uses a compounded gear train. A two-stage compounded gear train is shown below:



a. Select the appropriate number of teeth (20° pressure angle) of the a two-stage compounded gear train to obtain a reduction ratio of 30:1 ($\pm 1\%$). Assume that in each stage a reduction of approximately $\sqrt{30}$ is desired. Also, assume that the number of teeth on the pinion is 16. Note that the minimum number of teeth on a pinion to avoid interference for given speed ratio and pressure angle is

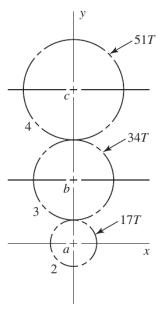
interference for given speed ratio and pressure angle is
$$N_p = \frac{2}{(1+2m_G)\sin^2\phi} \left(m_G + \sqrt{m_G^2 + (1+2m_G)\sin^2\phi}\right), \ m_G = N_G/N_P$$

- b. Select the appropriate number of teeth of the a two-stage compounded gear train to obtain an **exact** reduction ratio of 30:1.
- 2. It is sometimes desirable for the input shaft and the output shaft of a two-stage compound gear train to be in-line, as shown in Figure. This configuration is called a **compound reverted gear train**. This requires the distances between the shafts to be the same for both stages of the train.



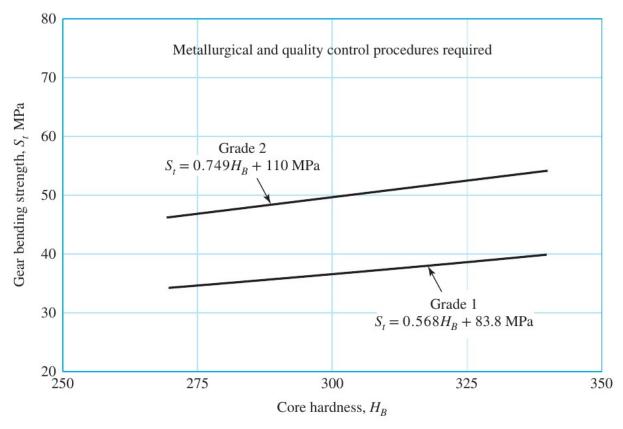
Select the appropriate number of teeth of a compound reverted gear train to obtain an **exact** reduction ratio of 30:1.

3. Shaft a in the figure has a power input of 75 kW at a speed of 1000 rev/min in the counterclockwise direction. The gears have a module of 5 mm and a 20° pressure angle. Gear 3 is an idler.



- (a) Find the force F_{3b} that gear 3 exerts against shaft b.
- (b) Find the torque T_{4c} that gear 4 exerts on shaft c.
- 4. A 17-tooth 20° pressure angle spur pinion rotates at 1800 rev/min and transmits 2980 watts to a 52-tooth disk gear. The module is 2.5 mm, the face width 38 mm, and the quality standard is No. 6. The pinion is a grade 1 steel with a hardness of 240 Brinell tooth surface and through-hardened core. The gear is steel, through-hardened also, grade 1 material, with a Brinell hardness of 200, tooth surface and core. Assume $J_P = 0.30$, $J_G = 0.40$, I = 0.121, $(K_S)_P = (K_S)_G = 1$, $C_H = 1$, $(C_f)_P = (C_g)_G = 1$. The loading is smooth because of motor and load. Assume a pinion life of 10^8 cycles and a reliability of 0.90.
- (a) Find the factor of safety of the gears in bending.
- (b) Find the factor of safety of the gears in wear.

Estimate the uncorrected bending fatigue strength using the following figure



Estimate the uncorrected surface-fatigue strength using the following figure:

