

# Control Systems HW5

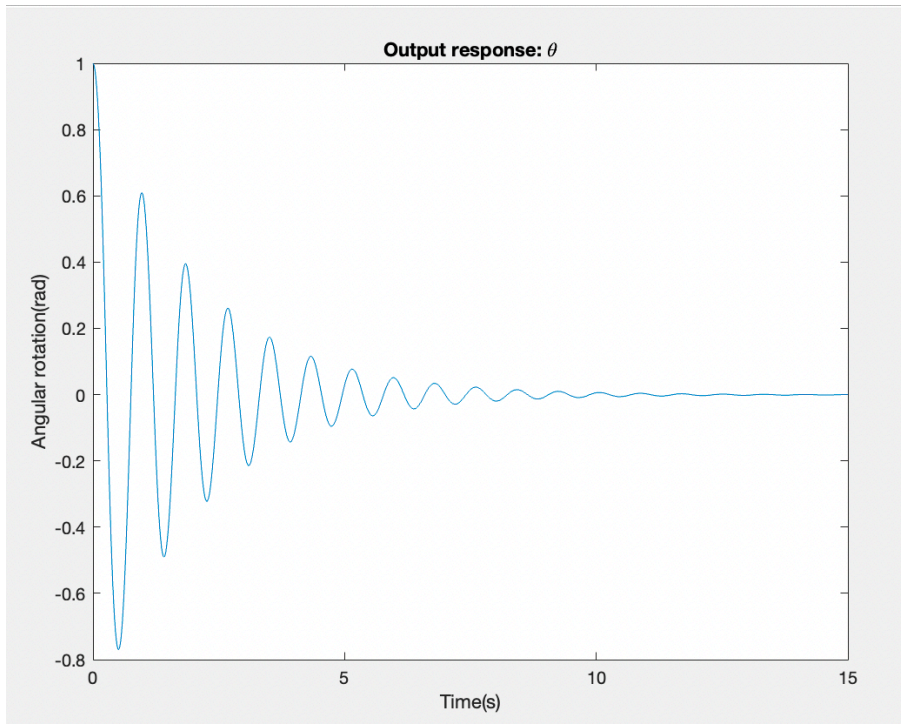
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1. Simulations by script (use the function of ode45)

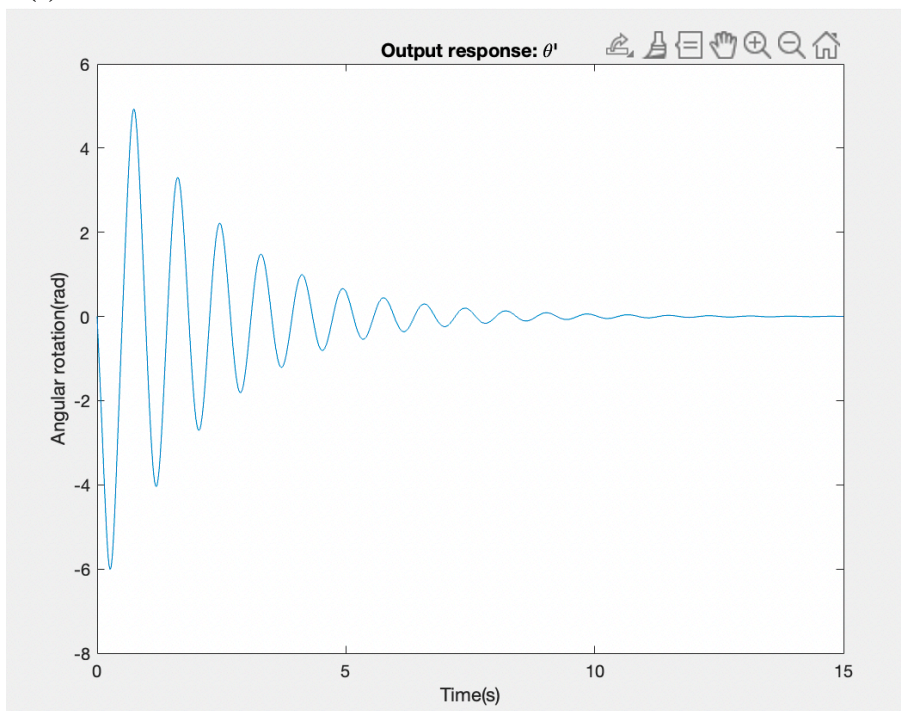
Assume that  $M = 2.5 \text{ kg}$ ,  $m = 10 \text{ kg}$ ,  $l = 1 \text{ m}$  and  $g = 9.81 \text{ m/s}^2$

$K_P = -7000$ ,  $K_I = -100$ ,  $K_D = -100$

$\theta(t)$  v.s  $t$  :



$\dot{\theta}(t)$  v.s  $t$  :



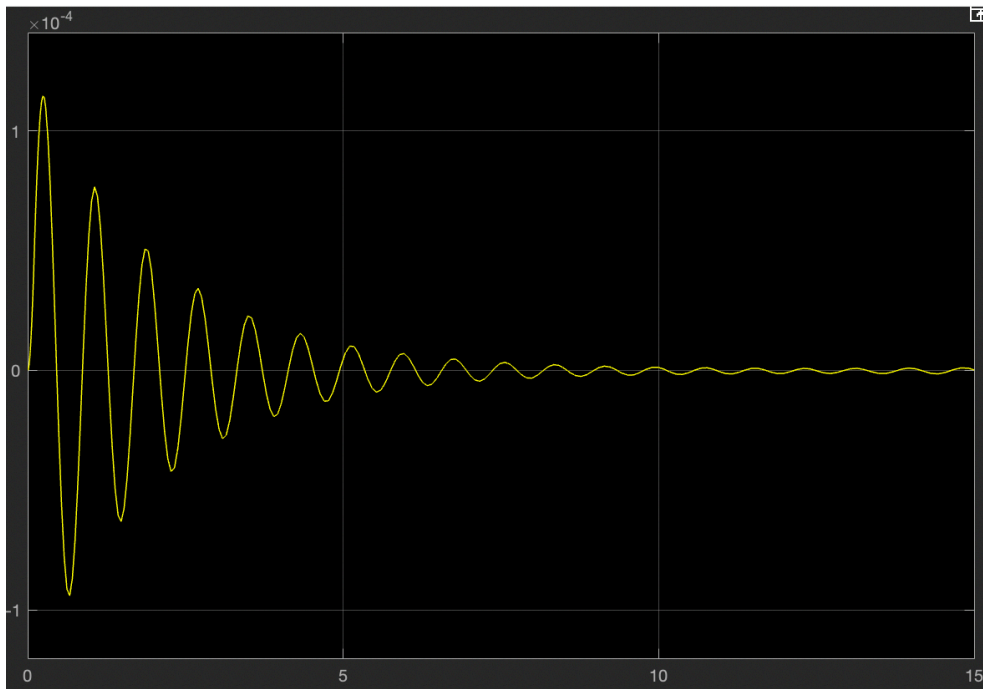
The above is the stable system since it satisfies that  $\theta(t) = 0$ ,  $\dot{\theta}(t) = 0$ , and we can also see that it approximately takes 9s to 10s to reach its steady state.

## 2. Simulation by Simulink

Assume that  $M = 2.5 \text{ kg}$ ,  $m = 10 \text{ kg}$ ,  $l = 1 \text{ m}$  and  $g = 9.81 \text{ m/s}^2$  (The same as part 1)

$K_P = -7000$ ,  $K_I = -100$ ,  $K_D = -100$

$\theta(t)$  v.s  $t$  :



The above system approximately takes 9s to 10s to reach its steady state.

### 3. Analysis

## Problem 1

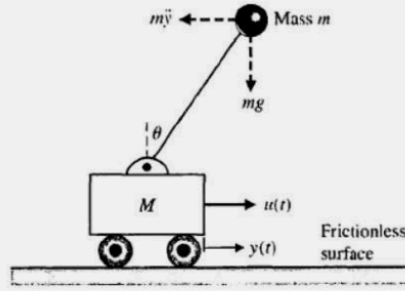


Figure 1 A cart and an inverted pendulum.

Force (horizontal):

$$(M + m)\ddot{y}(t) = u(t) - (mL\ddot{\theta}(t)\cos\theta(t) - mL\dot{\theta}^2(t)\sin\theta(t))$$

Torque:

$$mL^2\ddot{\theta}(t) = mgL\sin\theta(t) - mL\ddot{y}(t)\cos\theta(t)$$

Assume that  $M=100\text{Kg}$ ,  $m=10\text{Kg}$ ,  $L=1\text{m}$ ,  $g=9.81\text{m/s}^2$  and  $G_c(s) = K_P + \frac{K_I}{s} + sK_D$ ,

From the above, we obtain that

$$\ddot{y}(t) = \frac{1}{(M + m)} \left( u(t) - (mL\ddot{\theta}(t)\cos\theta(t) - mL\dot{\theta}^2(t)\sin\theta(t)) \right)$$

$$\ddot{\theta}(t) = \frac{1}{ML^2} (mgL\sin\theta(t) - mL\ddot{y}(t)\cos\theta(t))$$

And setting that  $x_1 = y$ ,  $x_2 = \dot{y}$ ,  $x_3 = \int \theta dt$ ,  $x_4 = \theta$ ,  $x_5 = \dot{\theta}$

So we can get the following relation:

$$\begin{cases} \dot{x}_1 = x_2 \\ \dot{x}_2 = \frac{u(t)}{(M + m)} - \frac{mL\dot{x}_5\cos x_4 - mLx_5^2\sin x_4}{(M + m)} \\ \dot{x}_3 = x_4 \\ \dot{x}_4 = x_5 \\ \dot{x}_5 = \frac{g}{L}\sin x_4 - \frac{\dot{x}_2}{L}\cos x_4 \end{cases}$$

and

$$U(s) = G_c(s)(R(s) - \theta(s)) - (PID \text{ control})$$

$$\begin{aligned} \Rightarrow u(t) &= -K_P\theta(t) - K_I \int \theta(t)dt - K_D \frac{d\theta(t)}{dt} \\ &= -K_P x_4 - K_I x_3 - K_D x_5 \end{aligned}$$

Next, we have to rearrange the equations of  $\dot{x}_2$  and  $\dot{x}_5$  since the equations are mixed with the derivative part of  $x_2$  and  $x_5$ . After arrangement, we obtain that

$$\dot{x}_2 = \frac{1}{M + m(1 - \cos^2 x_4)} (u(t) - mg \cos x_4 \sin x_4 + mLx_5^2 \sin x_4)$$

$$\dot{x}_5 = \frac{1}{M + m(1 - \cos^2 x_4)} \left( \frac{(M + m)g \sin x_4}{L} - \frac{u(t)\cos x_4}{L} - mx_5^2 \cos x_4 \sin x_4 \right)$$

By plotting the state-space parameters ( $\dot{x}_1, \dot{x}_2, \dot{x}_3, \dot{x}_4, \dot{x}_5$ ) obtained above using MATLAB function ode45, we can get the output response of  $\theta(t)$  and  $\dot{\theta}(t)$ . To stabilize the system, we have to check if poles of the system transfer function are all in the left-half plain, and use PID controller to adjust parameters ( $K_P, K_I, K_D$ ) in an efficient way. However, the equations are too complex to solve in this part, so the characteristic function has to be found, which is based on some assumptions. Next, Problem 2 will mention it in detail.

## Problem 2

2) Use Simulink to simulate the output response in s-domain.

Assume that  $\theta(t) \rightarrow 0^\circ \Rightarrow \sin \theta(t) \approx \theta(t), \cos \theta(t) \approx 1, \dot{\theta}(t) \approx 0,$

$$(M + m)\ddot{y}(t) + mL\ddot{\theta}(t) = u(t)$$

$$\ddot{y}(t) + L\ddot{\theta}(t) - g\theta(t) = 0$$

$$G(s) = \frac{-1/ML}{s^2 - \frac{(M+m)g}{ML}}$$

Applying PID control to the inverted pendulum.

Assume that  $T_d(s) = 1(T_d(t) = \delta(t))$

By the above assumptions, we get  $G(s)$ . The following uses MATLAB to find the characteristic equation:

```
%% Characteristic equation
clc; clear all; close all;

syms s KP KI KD
m=10;
M=100;
L=1;
g=9.81;

G = (-1/(M*L))/(s^2-(M+m)*g/(M*L));
Gc = (KD*s^2+KP*s+KI)/s;
(1+G*Gc)
```

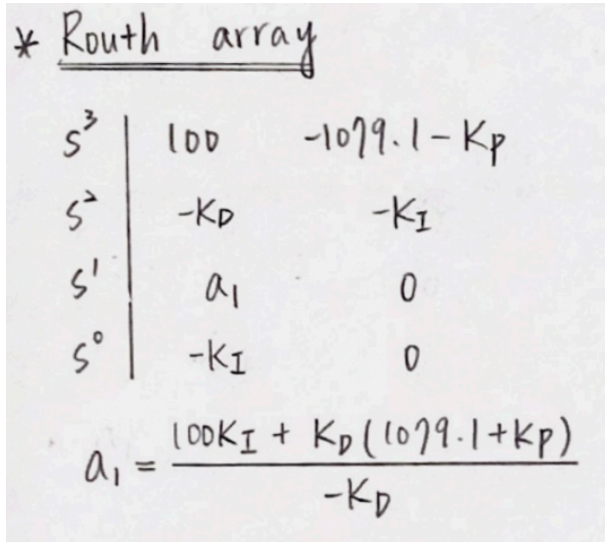
ans =

$$1 - (K_D s^2 + K_P s + K_I) / (100 s (s^2 - 10791/1000))$$

So the characteristic equation is

$$100s^3 - K_D s^2 - (K_P + 1079.1)s - K_I = 0$$

By using Routh-Hurwitz stability criterion,



\* Routh array

$s^3$	100	$-1079.1 - K_P$
$s^2$	$-K_D$	$-K_I$
$s^1$	$a_1$	0
$s^0$	$-K_I$	0

$$a_1 = \frac{100K_I + K_D(1079.1 + K_P)}{-K_D}$$

For a stable system, it must be satisfied that

$$-K_D > 0, -K_I > 0, a_1 = \frac{100K_I + K_D(1079.1 + K_P)}{-K_D} > 0$$

$$\Rightarrow K_D < 0, K_I < 0, K_P < -1079.1 - \frac{100K_I}{K_D}$$

We choose  $K_I = K_D = -100$ ,  $K_P = -7000$  to get a stable system.

It is obvious to see that the simulations by ode45 in part 1 and by Simulink in part 2 basically come to a stable system and almost have the same results. This homework gives the basic idea about how to use PID controller to get a stable system, and how to tune the parameters in a systematic way.