

# Numerical Simulation of Heat Transfer in Vertical Fuel Bundles of 37-element Canadian SCWR

Huirui Han<sup>1</sup>, Chao Zhang<sup>1</sup> and Jin Jiang<sup>2</sup>

<sup>1</sup>*Department of Mechanical and Materials Engineering, The University of Western Ontario  
1151 Richmond Street, London, ON, Canada, N6A5B9*

<sup>2</sup>*Department of Electrical and Computer Engineering, The University of Western Ontario  
1151 Richmond Street, London, ON, Canada, N6A5B9*

Email: hhan55@uwo.ca

## ABSTRACT

Supercritical water cooled reactor was proposed as one of the Generation IV nuclear systems. Although many researches for heat transfer of supercritical water in circle channels have been conducted, there is still lack of prediction of heat transfer in transient process of tube bundles used in supercritical nuclear reactors. Besides, since there are multiple fuel rods, supercritical water is external flow not internal flow as in circle channels. This will cause the difference of fluid flow and heat transfer on cladding surface. In this work, a 37-element Canadian SCWR vertically arranged fuel bundle is simulated numerically. The thermo-hydraulic behaviors of the supercritical water in the SCWR fuel bundle are predicted. For nuclear power plant safety, the cladding surface temperature is a very important parameter. The results in this work show that the maximum cladding temperature is lower than the limit when the load is 100%. In the 3D transient process simulation, a step increase is introduced and test how long the system will return to steady state. This can provide references for future control system design.

## NOMENCLATURE

### Abbreviations

3D	three dimensional
CFD	Computational Fluid Dynamics
RSM	Reynolds Stress Model
SCWR	Supercritical Water-Cooled Reactor

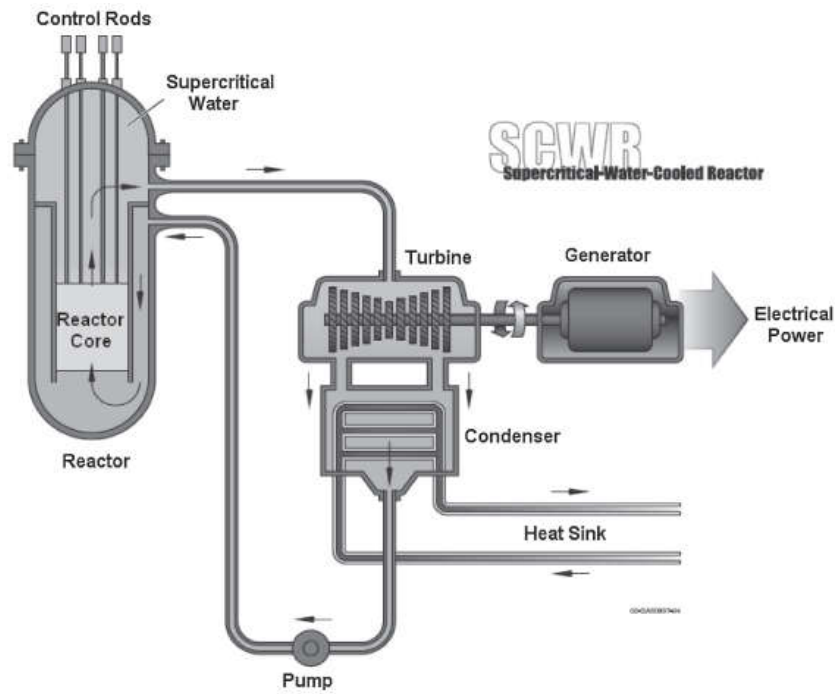
### General symbols

$C_{ij}$	convection term
$C_p$	specific heat capacity, J/kg·K

$D_{T,ij}$	turbulent diffusion
$D_{L,ij}$	molecular diffusion
$G_{ij}$	buoyancy production
$p$	pressure, Pa
$P_{ij}$	stress production
$Pr_t$	turbulent Prandtl number
$S$	heat sources
$T$	temperature, K
$u$	velocity, m/s
$\rho$	density, $kg / m^3$
$\mu$	dynamic viscosity, $Pa \cdot s$
$\lambda$	thermal conductivity, $W / m \cdot K$
$\mu_t$	turbulent viscosity, $Pa \cdot s$
$\phi_{ij}$	pressure strain
$\varepsilon_{ij}$	dissipation

## 1. INTRODUCTION

In the Generation IV nuclear reactor concepts proposed in 2002, the Supercritical Water-Cooled Reactor (SCWR) is the only one that uses light water as coolant. CANDU series are heavy pressurized reactors. Compared with nowadays reactors, SCWR system configuration size is smaller. Also, since water is heated by reactor core to supercritical steam, then directly does work through turbine, thermal efficiency can be increased to 40% [2]. Typical Canadian SCWR is shown in Figure 1. Moderator for



**Figure1 [1]** Configuration of Supercritical Water-cooled Reactor System

**Table 1** Specifications of CANDU-SCWR

Spectrum	Thermal
Moderator	Heavy water
Coolant	Light water
Thermal Power	2540MW
Flow rate	1320kg/s
Number of channels	336
Electric power	1200MW
Efficiency	48%
Fuel	UO <sub>2</sub> /TH
Enrichment	4%
Inlet temperature	350°C
Outlet temperature	625°C
Cladding temperature	<850°C
Heated length	5m
Operating pressure	25MPa
Calandria diameter	4m

the reactor is heavy water and coolant is light water. The specifications for Canadian SCWR [3] is shown in Table 1. Because of the abrupt properties change of supercritical water around pseudocritical point, heat transfer deterioration is easy to happen. Since Canadian SCWR is still in conceptual design, there are no experimental data now for flow and heat transfer in fuel bundles. Numerical method is highly recommended to predict fluid flow and heat transfer of supercritical water in fuel bundles. This paper will focus on predicting thermo-hydraulic behavior of supercritical water in fuel bundles in transient process. Because of the specialty of supercritical water, heat transfer deterioration may occur on cladding surface. This will affect safety of whole reactor system. Azih et al. [4] studied the convective heat transfer of supercritical fluids without gravity, and obtained the effects of Reynolds number, boundary layer and other factors on heat transfer. Zeng et al. [5] also conducted computational fluid dynamics simulation in circle channel. The consideration of gravity and thermal expansion acceleration effect in turbulence model under subcritical simulation is put forward. This can be helpful for investigation of Canadian SCWR startup process. Zhang et al. [6] proposed a new supercritical fluid flow model to better deal with the physical instability properties of the supercritical fluid around the pseudo-critical point. Rohit [7] also simulated heat transfer of supercritical water in circle channels using different turbulence models. And the results are compared with experimental data to check fitting degree. However, these previous researchers mainly investigated heat transfer and fluid flow in single channel. This may ignore effects of interaction between fuel rods. And in fact, the supercritical water in fuel bundles is external flow around fuel rods, not internal flow in channels. Yina [8] simulated heat transfer and flow of supercritical water in a 37-element horizontal arranged SCWR in steady state. The nuclear power plant work procedures are complicated, it is hard to include all points in numerical simulation. However, there is still lack of 3D numerical simulation for supercritical water in vertical fuel bundles. In this work, we will investigate heat transfer of supercritical water in vertical fuel bundles with multi fuel rods in transient process.

## 2. GOVERNING EQUATIONS AND NUMERICAL MODELS

In this study, a 37-element fuel bundle [9] is chosen for numerical transient simulation. Fuel rods are

averagely distributed in the fuel bundle. The fuel bundle is vertically arranged and works under 100% load. Since the symmetry of fuel bundles, only a quarter of the fuel bundle is included in this simulation. And the heated length is reduced to 1m to save simulation time. The properties of supercritical water, such as density, specific heat, thermal conductivity, viscosity, are from Wagner [10]. The pseudocritical point region is about 384.9°C for 25MPa[11]. The governing equations for this three-dimensional flow and heat transfer are conservations of continuity, momentum, energy, which are shown in Reynolds averaged form [12]:

$$\frac{\partial \bar{\rho}}{\partial t} + \frac{\partial (\bar{\rho} u_i)}{\partial x_i} = 0 \quad (1)$$

$$\frac{\partial (\bar{\rho} u_i)}{\partial t} + \frac{\partial (\bar{\rho} u_i u_j)}{\partial x_j} = -\frac{\partial \bar{p}}{\partial x_i} + \frac{\partial}{\partial x_j} (\mu \frac{\partial \bar{u}_i}{\partial x_j} - \overline{\rho u_i u_j}) \quad (2)$$

$$\frac{\partial (\bar{\rho} C_p T)}{\partial t} + \frac{\partial (\bar{u}_i \bar{\rho} C_p T)}{\partial x_i} = \frac{\partial}{\partial x_i} [(\lambda + \frac{C_p \mu_t}{Pr_t}) \frac{\partial T}{\partial x_i}] + S \quad (3)$$

Zhang found the numerical results calculated by Reynolds Stress Model agree well with experimental data for vertical tubes[13]. Thus, the Reynolds stress model (RSM) with enhancement wall function is used for solving the Reynolds stress term in this simulation. Thermal and full buoyancy effects, viscous heating are all included. The transport equation of RSM is as follows[12]:

$$\begin{aligned} & \underbrace{\frac{\partial}{\partial t} (\overline{\rho u_i u_j})}_{\text{Local Time Derivative}} + \underbrace{\frac{\partial}{\partial x_k} (\overline{\rho u_k u_i u_j})}_{C_{ij}} = \\ & - \underbrace{\frac{\partial}{\partial x_k} [\overline{\rho u_i u_j u_k} + \overline{p'(\delta_{kj} u_i + \delta_{ik} u_j)}]}_{D_{T,ij}} + \underbrace{\frac{\partial}{\partial x_k} [\mu \frac{\partial}{\partial x_k} (\overline{u_i u_j})]}_{D_{L,ij}} \\ & - \underbrace{\overline{\rho(u_i u_k \frac{\partial u_j}{\partial x_k} + u_j u_k \frac{\partial u_i}{\partial x_k})}}_{P_{ij}} - \underbrace{\overline{\rho \beta (g_j u_i \theta + g_j u_i \theta)}}_{G_{ij}} \\ & + \underbrace{\overline{p'(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i})}}_{\phi_{ij}} - 2 \underbrace{\overline{\mu \frac{\partial u_i}{\partial x_k} \frac{\partial u_j}{\partial x_k}}}_{\varepsilon_{ij}} + \underbrace{S_{user}}_{\text{User defined source term}} \end{aligned} \quad (4)$$

The simulation parameters for the fuel bundle are shown in Table 2 [8]. The computational domain is in Figure 2 and the mesh is in Figure3. ANSYS Fluent 15.0 is used to solve equations. The SIMPLEC scheme is selected for pressure correction, and QUICK method is considered for conducting spatial discretization. The convergence criteria for continuum, momentum, and turbulence parameters are all  $10^{-6}$ , but for energy equation is  $10^{-9}$ . The heat flux of fuel rod cladding surface is  $10^6 W/m^2$  averagely. The reference pressure is 25MPa. 334290 nodes give grid independence for this simulation. Time step size is 0.01s and the max iteration for each time step is 20. Boundary conditions are as follows:

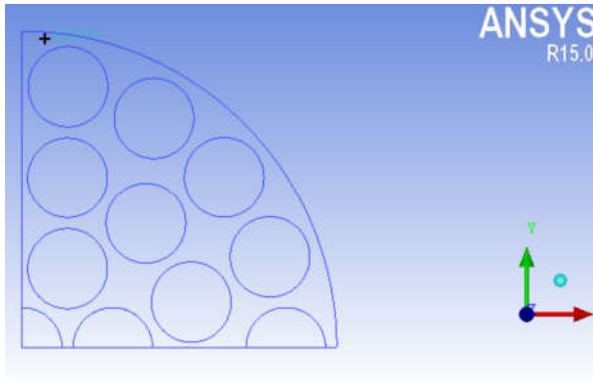
**Inlet:** Here, velocity-inlet is chosen for the inlet type, inlet velocity is 1.18m/s and inlet temperature is 623.15K. Turbulence intensity is 6%, and hydraulic diameter is 7.416mm based on Table 2;

**Outlet:** Outflow is selected;

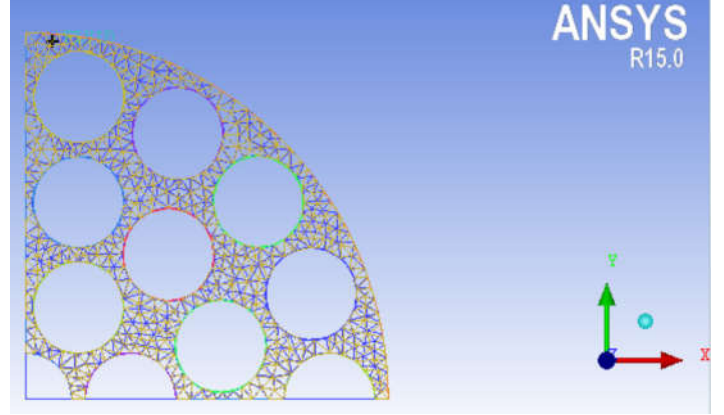
**Walls:** For the walls, here we consider they are smooth walls with no-slip conditions.

**Table 2** Parameters for the fuel bundle simulation

Fuel Rod Diameter/mm	13.081
Fuel Bundle Diameter/mm	103.378
Length/mm	1000
First Ring Diameter/mm	29.769
Second Ring Diameter/mm	57.506
Third Ring Diameter/mm	86.614



**Figure 2** Computational domain

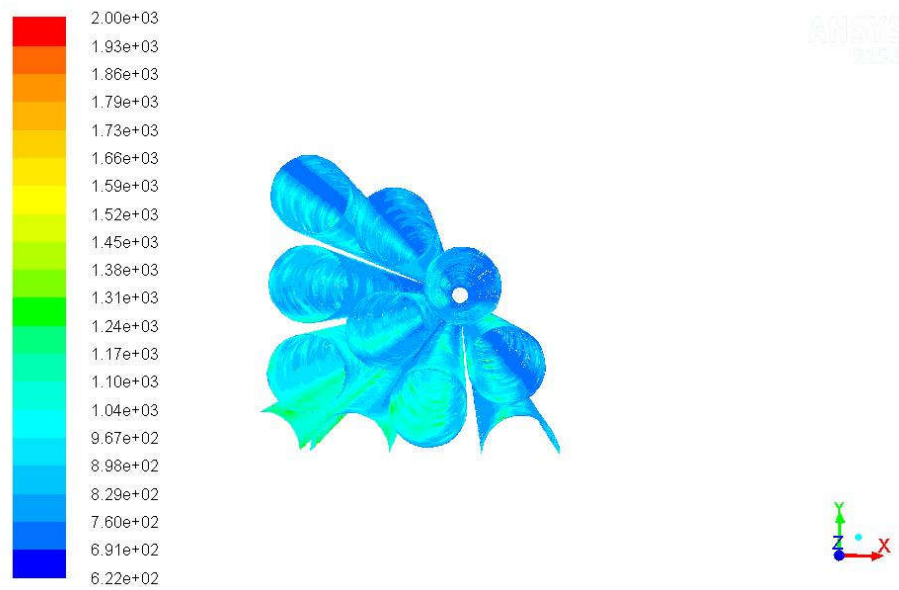


**Figure 3** Mesh

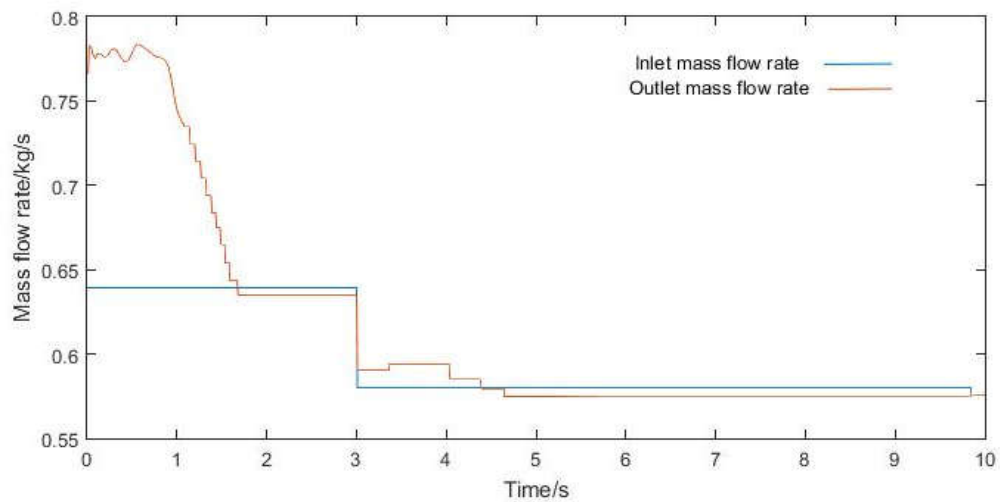
### 3. RESULTS AND DISCUSSIONS

Figure 4 shows the cladding surface temperature at  $t=3s$ . It is shown that the maximum value is below the limit when the load is 100%. Figure5 (a) and 5(b) show the inlet and outlet mass flow rate, outlet temperature of the fuel bundle variation with time when inlet mass flow rate decrease 10% at 3s. It can be seen that the flow is very unstable at first when supercritical water is heated around the pseudocritical point. Inlet and outlet mass flow rate tends to be uniform when  $t=2s$ . At the same time, the outlet temperature also increases to a peak value and then gradually remain stable before the step decrease inserted. A 10% step decrease of inlet mass flow rate is introduced when  $t=3s$ . Then the outlet mass flow rate also decrease and reach the inlet value quickly. The outlet temperature just increase about  $5^{\circ}C$  and then becomes stable again. In this 10s transient process, the system can return to a relatively steady state quickly after perturbances are introduced. Figure 5(c) shows the cross section view of the cladding surface temperature distribution of fuel rods at  $t=10s$ . For each fuel rod, the cladding surface temperature distribution is not uniform. Higher temperatures always appear on the location area between each two fuel rods. The maximum cladding temperature shown in Figure 5(c) can be up to about 1250K, which is higher than the design value in Table 1. This may cause safety accident in nuclear power plant, especially when the nuclear power unit is in shut-down process.

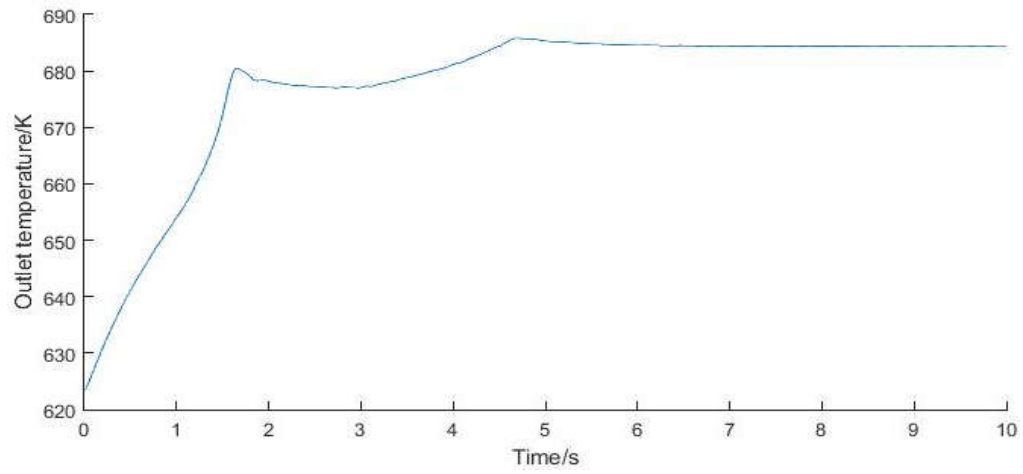
For another condition, the wall heat flux decrease 10% at  $t=3s$ , these three factors variations are shown in Figure 6(a) and 6(b). Trends are the same before 3s. However, when the 10% step decreased is inserted at



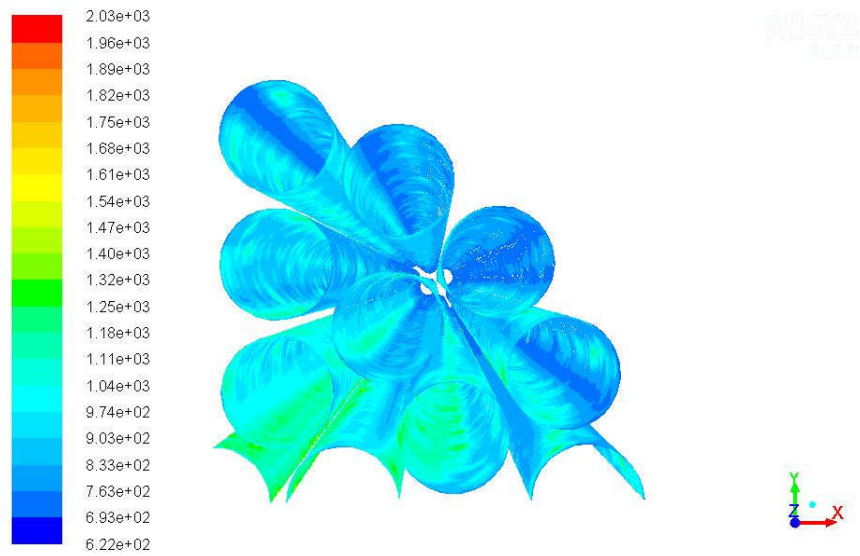
**Figure 4** Cladding surface distributions at  $t=3s$



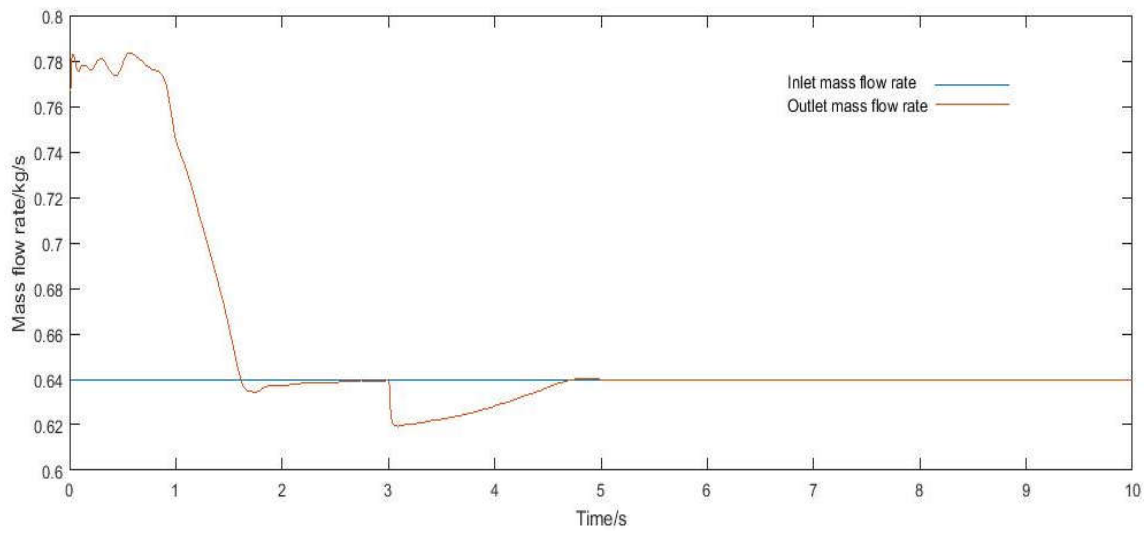
**Figure 5(a)** Inlet and outlet mass mass flow rate variation with time



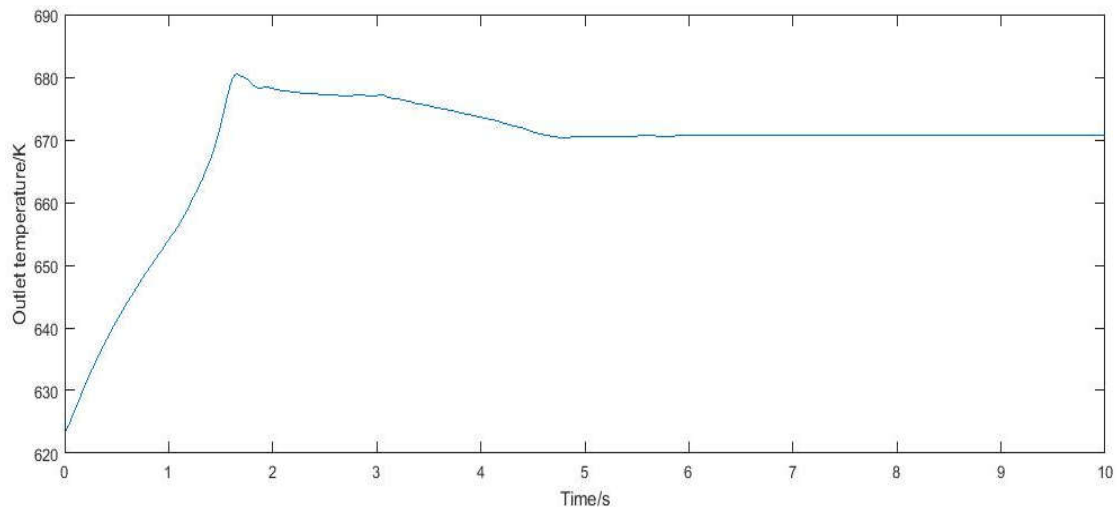
**Figure 5(b)** Outlet temperature of the fuel bundle variation with time



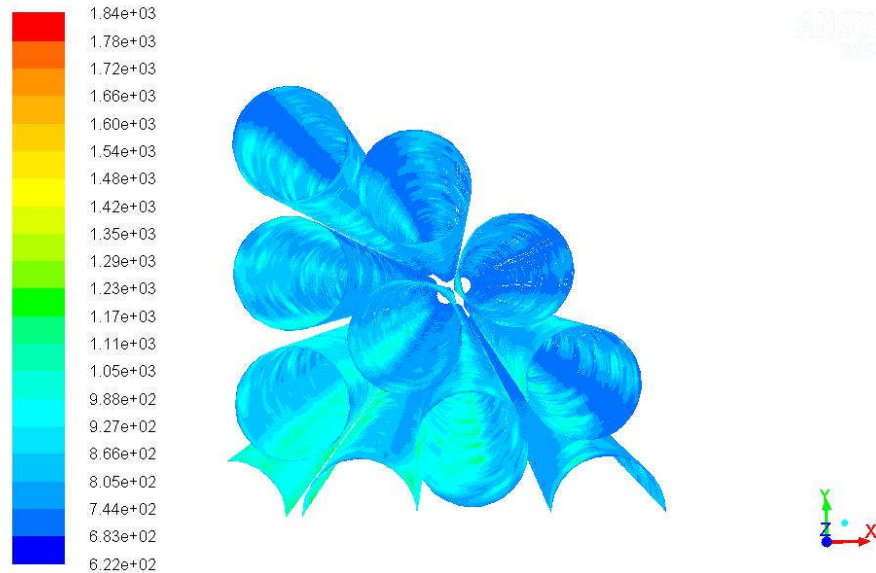
**Figure 5(c)** Cladding surface temperature(K) distributions



**Figure 6(a)** Inlet and outlet mass mass flow rate variation with time



**Figure 6(b)** Outlet temperature of the fuel bundle variation with time



**Figure 6(c)** Cladding surface temperature(K) distributions

3s, the outlet mass flow rate present a sharp decrease along with. This is mainly due to the sharp density change around the pseudocritical point when the heat flux decreases. Figure 6(c) shows the cladding surface temperature distributions at  $t=10s$  when heat flux decrease 10% at  $t=3s$ . Although the fuel rod cladding surface temperature is still uniform, the maximum value shown is below the design value.

#### 4. CONCLUSIONS

Transient flow and heat transfer behaviors of supercritical water in the fuel bundle perturbed under two conditions at the design point are investigated in this paper. The flow is not steady at the beginning because of the special property of supercritical water. When perturbances are inserted, the system can return to steady state in a short time. However, the cladding surface temperature distribution is not uniform for each fuel rod and the maximum value is above the design value when the system return to relatively stable. This should be controlled to make sure safety when a nuclear reactor is normally working. Since the Canadian SCWR is still in conceptual design, experimental data are needed in the future to validate simulation results.

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