

Equation of Motion:
Lagrangian, Principle of Least Action, and E-L Equation

- Larangian:
Under the constraint of
1)Space and time are homogenous, 2)time is isotropic, the Larangian for a system is given as

L = T - U(r), where { T = sum_{a=1}^N 1/2 m_a q_a^2 sum of KE
U: potential energy

E-L equation
For a given functional,

S = integral_{t_1}^{t_2} L(q, q-dot, t) dt

we could optimize it using the Euler-Lagrange equation,
d/dt (dL/dq-dot) - dL/dq = 0

where each EL equation and its solution corresponds to a degree of freedom.
Upon applying the EI equation to a generalized lagrangian, we reveal Newton's second law

d/dt (d(1/2 m v^2 - U(r))/dv) = d(1/2 m q-dot^2 - U(r))/dr
=> m v-dot = -dU/dq = F-dot(force)

- coordinate transformation:
In cartesian coordinates, L = 1/2 m (x-dot^2 + y-dot^2 + z-dot^2) - U
In cylindrical coordinates, L = 1/2 m (r-dot^2 + r^2 theta-dot^2 + z-dot^2) - U
In spherical coordinates, L = 1/2 m (r-dot^2 + r^2 theta-dot^2 + r^2 sin^2(theta) phi-dot^2) - U
Note that when taking partial differentiations, we treat each variable and its derivative as two independent variables. Don't ask why... We are doing physics here

Conservation Laws:
Energy, Momentum, COM, and Angular Momentum

- Energy:
Energy is defined as the following, and when the Lagrangian is homogeneity time, the energy is conserved.

E = sum_i q_i dL/dq_i-dot - L

considering L = T - U, we have E = T + U

- Total energy is also given as
E = 1/2 mu V^2 + E_i
where E_i is internal energy, and mu being the total mass

General momentum:
conservation of general momentum is from the following conservation

dL/dq_j-dot = 0 => p_j = dL/dq_j-dot

where q_j is a cyclic coordinate, i.e. L is independent of q_j

Total momentum
total momentum is defined as the following, and considering the homogeneity of space, the momentum is conserved in a closed system.
If the total momentum of a mechanical system in a given frame of reference is 0, then the said system is at rest relative to that frame. For simplicity's sake, we want to chose our frame of reference in which the total momentum is zero.

P = sum_a dL/dq_a-dot = sum_a m_a v_a
force is also given by F_j = dL/dq_j-dot
sum of all forces in a closed system is 0

- Center of Mass
Center of mass is defined so that, the velocity of the system as a whole, V = P/(sum m_a) is the time derivative of the center of mass. R = sum_a m_a r_a / (sum m_a).

Conservation of angular momentum
Angular momentum caracterizes the rotation of the system, and considering the isotropy of space, the angular momentum is conserved in a closed system.

L-dot = sum_a r_a x p_a is conserved in a closed system

- Angular momentum can be found by differentiating the lagrangian with respect to angular velocity, along the rotation axis z:

L_z = dL/dphi_a-dot (10)

Integration of the equations of motion: Connetcting Energy with motion

- Motion in 1 dimension
For a system with DOF=1, and with dL/dv = 0 (lagrangian independent of time, i.e. energy conserved), we can write the lagrangian and total energy as

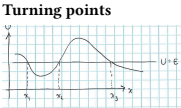
L = 1/2 m x-dot^2 - U(x), (11)

E = 1/2 m x-dot^2 + U(x) (12)

Equation 12 is a differential equation of position and time. Solving this ODE for time gives:

t = sqrt(m/2) integral dx / sqrt(E - U(x)) + C (13)

when given U(x), and by plugging it into Equation 12, we can solve for x(t) by substitution. Tricks on sub: when U(x) is of order 1, use u-sub; when it's of order 2, use trig-sub.



For a given potential function U(x), the turning points are the points where the potential energy is equal to the total energy, i.e. U(x) = E. At turning points, the system is either just about to move, or just about to stop.
Only motion where potential is less or equal to total energy is allowed.
Bounded motion: [x_1, x_2]; unbounded motion: x > x_3

Unbounded Motion:
When there is a potential well, the system could go into periodic motion with potential energy moving back and forth in the well, and position between x_1, x_2. We find period by doubling Equation 12:

T(E) = sqrt(2m) integral_{x_1(E)}^{x_2(E)} dx / sqrt(E - U(x)) (14)

where we represent x_1(E), x_2(E) in terms of E.
When given U(x), we can solve for x_1(E), x_2(E), and then plugging in to Equation 14, we can solve for period by integration via subsitution.

Simple Pendulum in polar coord's has the following:
T = 1/2 m l^2 theta-dot^2
U = mgl(1 - cos(theta))

It's period is given by Equation 14. Solving it gives us
T(E) = 4 sqrt(l/g) integral_0^{pi/2} du / sqrt(1 - k^2 sin^2(u)) (16)
where k = sin(theta_0/2), sin u = 1/k sin(theta_0/2)

Equation 16 can be simplified by small angle approx into
T(E) = 2pi sqrt(l/g) (1 + (theta_0^2/16)) (17)

Effective DOF=1 system
When the lagrangian is of the form L = f(x) - g(x), we can see it as a system with effective potential U_eff(x) = g(x), and effective kenetic energy T_eff(x) = f(x). The effective energy is therefore E = T_eff + U_eff.

Two body problem

- Problem setup
The two body problem considers two interacting masses with an interacting potential U(r_1, r_2) = g(|r_1 - r_2|). The lagrangian is given by

L = 1/2 m_1 r_1-dot^2 + 1/2 m_2 r_2-dot^2 - U(|r_1 - r_2|) (18)

- Reduced mass mu = (m_1 m_2)/(m_1 + m_2) = m_1 m_2 / M;
Center of mass R = (m_1 r_1 + m_2 r_2)/(M);
relative positon r = r_1 - r_2

- Putting the two body system into relative coordinates, and represent masses with reduced mass and COM, we have the following lagrangian:

L = 1/2 M R-dot^2 + 1/2 mu r-dot^2 - U(r) (20)

where the first term involves only the COM motion, and the second term involves only the relative motion.
By choosing our frame with the COM at rest and the total momentum zero, our problem is simplified to an effective one body problem with DOF = 2, given by

L = 1/2 mu r-dot^2 - U(r) (21)

- Conservation of Angular Momentum
Angular momentum is defined as L = r x mu r-dot, and is conserved here.
Knowing r-dot . L = 0, the motion is in the plane perpendicular to L. We can use polar coordinates to describe the motion,

L = 1/2 mu (r-dot^2 + r^2 theta-dot^2) - U(r) (22)

Using EL equation on Equation 22, we get
d/dt (dL/dtheta-dot) - dL/dtheta = 0
=> L_z = mu r^2 theta-dot = constant
(conservation of angular momentum on z-axis)

2 body problem in gravitational field
L = 1/2 m_1 r_1-dot^2 + 1/2 m_2 r_2-dot^2 - [m_1 g z_1 + m_2 g z_2 + U(r)]
= [1/2 M R-dot^2 - M g Z] + [1/2 mu r-dot^2 - U(r)] (24)

where Z is the vertical coordinate of the CM position, Z = (m_1 z_1 + m_2 z_2)/M

Kepler's second Law
We calculate the differential of area swept by particle in polar coordinates,

dA = 1/2 r^2 dphi
=> dA/dt = 1/2 mu L_z
L_z = 2 mu A-dot(constant)

This is the Kepler's second law, which states that the area swept by the radius in a given time is constant.

- EOM for two body system
The total energy:

E = T + U = 1/2 mu r-dot^2 + 1/2 mu r^2 phi-dot^2 + U(r) (26)
= 1/2 mu r-dot^2 + U(r) + L_z^2 / (2 mu r^2) (Notice L_z = mu r^2 phi-dot)

solving this ODE by integration gives
t(r) = integral sqrt(2/mu) [E - U(r) - L_z^2 / (2 mu r^2)] dr + C (27)

Also from L_z = mu r^2 phi-dot, by integrating with respect to time, we get
phi(t) = L_z / mu integral dt / r^2(t) + C' (28)

Equation 28 and Equation 26 describe the relative motion of the two body system in terms of constants {E, L_z, C, C'}

- Shape of orbit
Equation 26 skipped a step,

dr/dt = sqrt(2/mu) [E - U(r) - L_z^2 / (2 mu r^2)] (29)

this equation, combined with our beloved
L_z = mu r^2 phi-dot => dphi = L_z / mu r^2 dt (30)

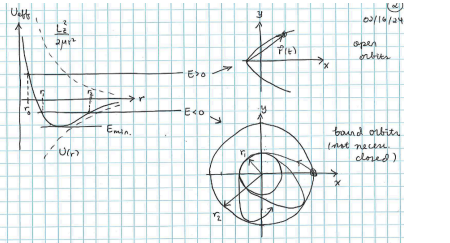
we get the equation of orbit:
dphi = L_z / (sqrt(2 mu) r^2 sqrt(E - U(r) - L_z^2 / (2 mu r^2))) dr
=> phi = L_z / (sqrt(2 mu) integral dr / (r^2 sqrt(E - U(r) - L_z^2 / (2 mu r^2))) + C (31)

Effective potential and shape of orbit (Only for Attractive Potential)

U_eff = U(r) + L_z^2 / (2 mu r^2); E = 1/2 mu r-dot^2 + U_eff(r) (32)

- When r -> inf, U_eff -> U(r), and when r -> 0, U_eff -> centrifrugal potential L_z^2 / (2 mu r^2).

- by graphing the effective potential, and given constraint of total energy E, we can analyze the shape of the orbit:



- when E > 0, the orbit is unbounded, open orbit, hyperbola.
when E < 0, the orbit is bounded into a potential well, although not necessarily closed.
when E = E_min, the orbit is circular, F = -mu v^2 / r

The Kepler Problem: a special case of the two body problem conditions

U(r) = -alpha/r; U_eff = -alpha/r + L_z^2 / (2 mu r^2) (33)

Conic section orbits
We can proof that the orbit is a conic section given by

r(phi) = p / (1 + e cos(phi)) (34)
where { p = L_z^2 / mu alpha
e = sqrt(1 + 2 E L_z^2 / mu alpha^2)

Classifications of orbits based on energy of system E
When E > 0, e > 1, the orbit is unbounded, open orbit, hyperbola.

(x-c)^2/a^2 - y^2/b^2 = 1
{ a = p/(e^2-1), b = p/sqrt(e^2-1), c = ae
r_min = p/(1+e)

- when E = 0, e = 1, the orbit is parabola.
y^2 = p^2 - 2xrp,
r_min = p/2
when E < 0, e < 1, the orbit is closed, ellipse.

(x+c)^2/a^2 + y^2/b^2 = 1,
{ a = p/(1-e^2), b = p/sqrt(1-e^2), c = ae
r_min = p/(1+e); r_max = p/(1-e)

- When E = E_min, f = mu alpha^2 / (2 L_z^2), e = 0, orbit is circular. r(phi) = p = constant

More Kepler: Period, Kepler's third law

Orbit of each body
recall Equation 19, we can exrees the orbit of each body as such after some algebra:

r_1 = (m_2 / (m_1 + m_2)) r; r_2 = -(m_1 / (m_1 + m_2)) r (39)

- when m_1 = m_2 => r_1 = r_2 = -r/2, COM inside r_1 cap r_2
when m_1 >> m_2 => r_1 = r, r_2 = 0, m_1 is at rest, m_2 orbits m_1

- Period of orbit
L_z = 2 mu A-dot areal vel. is constant
Integrating A over a period,

A = integral_0^T A-dot dt = L_z T / (2 mu) (40)

Since area swept over a period is the area of the ellipse, we have

pi ab = L_z T / (2 mu), letting: b = sqrt(pa), p = L_z^2 / mu alpha
=> T = (2 pi a^3 / 2) sqrt(mu / alpha) (41)

Conservation of Laplace-Runge-Lenz vector
A = v x L - (alpha r)/r is conserved, and is perpendicular to the orbit plane. We can use it to verify : conic sections, eccentricity, and period.

• conserved quantity: $\vec{A} \cdot \vec{L} = 0, \frac{A}{\alpha} = \sqrt{1 + \frac{2EL^2}{\mu\alpha^2}}$

Orbital Transfer

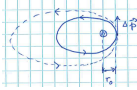
Instantaneous Change in velocity

$$\begin{aligned} (E, L_x) &\rightarrow (E', L'_x) \\ \Rightarrow (e, p) &\rightarrow (e', p') \end{aligned} \tag{42}$$

if thrust occur when satellite is at angle φ_0 , orbit orientation can change:

$$r(\varphi_0) = \frac{p}{1 + e \cos \varphi_0} = \frac{p'}{1 + e' \cos(\varphi_0 - \delta)} \tag{43}$$

Tangential thrust at perigee



at $\varphi = 0$, let $v = v_{\text{init}}$, $v' = v_{\text{right after}}$, $\lambda = v' / v$

$$\begin{aligned} L_x = \mu r_0 v &\Rightarrow L'_x = \mu r_0 v' = \lambda L_x \\ p' &= \lambda^2 p \end{aligned} \tag{44}$$

From Equation 43,

$$\frac{p}{1 + e} = \lambda^2 \frac{p}{1 - e'} \Rightarrow e' = \lambda^2(1 + e) - 1 \tag{45}$$

if $\lambda > 1$, $e' > e$, the satellite is in a higher, more elliptical orbit. Unbound if λ big enough

if $\lambda < 1$, $e' < e$, the satellite is in a lower orbit.

changing between circular orbits

- changing from R to R' , two thrusts(λ_1, λ_2) are needed. There is also an intermediate orbit

$$\begin{aligned} r(\varphi) &= p' / (1 + e' \cos \varphi), \\ \text{where } p' &= \lambda_1^2 p, e' = \lambda_1^2 - 1 \end{aligned} \tag{46}$$

changed from indermetiade to final,

$$\begin{aligned} r(\varphi = \pi) &= R' = \lambda_2^2 R / (2 - \lambda_1^2) \\ \Rightarrow \lambda_1 &= \sqrt{\frac{2R'}{R + R'}} \end{aligned} \tag{47}$$

final orbit:

$$\begin{aligned} r(\varphi) &= R'; e'' = 0, p'' = R' \\ \Rightarrow p'' &= \lambda_2^2 p' = p' / (1 - e') \\ \Rightarrow \lambda_2 &= \sqrt{\frac{R + R'}{2R'}} \end{aligned} \tag{48}$$

Small Oscillations

- Motion near a point of stable equilibrium.

DOF= 1 (one dimension)

- For a system of DOF = 1, with potential $U(q)$:

- stable equilibrium** at $U(q)_{\min}$, upward parabola, where $F = -\frac{dU}{dq} = 0$
- restoring force for small displacements $q - q_0$ is $F = -\frac{d^2U(q-q_0)}{dq}$

- Unstable equilibrium** at $U(q)_{\max}$, downward parabola, where $F = -\frac{dU}{dq} = 0$ as well.

- Consider small deviation from point of stable equilibrium, we use taylor expansion to show that it is really a small displacement. that is,

$$\begin{aligned} U &\approx U(q_0) + \frac{dU(q_0)}{dq}(q - q_0) + \frac{d^2U(q_0)}{2dq^2}(q - q_0)^2 \\ \text{while } \frac{dU(q_0)}{dq}(q - q_0) &= 0 \end{aligned} \tag{49}$$

letting $x = q - q_0$, we have

$$\begin{aligned} \left\{ \begin{aligned} U(x) &= U(q_0) + \left(\frac{1}{2}\right) \frac{d^2U(q_0)}{dq^2} x^2 \\ \text{putting into the form of } U(x) &= U(x_0) + \left(\frac{1}{2}\right) kx^2. \end{aligned} \right. \\ \Rightarrow \quad \quad \quad \boxed{k = \frac{d^2U(q_0)}{dq^2} > 0} \end{aligned} \tag{50}$$

we get KE, while choosing $U(q_0) = 0$:

$$\begin{aligned} T &= \frac{1}{2} a(q)^2 \dot{q}^2 = \frac{1}{2} a(q_0 + x) \dot{x}^2 \approx \frac{1}{2} m \dot{x}^2, \quad m = a(q_0) \\ \boxed{L &= T - U = \frac{1}{2} m \dot{x}^2 - \frac{1}{2} kx^2} \end{aligned} \tag{51}$$

EOM for DOF = 1 small Oscillations

using EL on Equation 51, we can get the EOM for one dimensional small Oscillations:

$$\begin{aligned} m\ddot{x} &= -kx \\ \Rightarrow \ddot{x} + \omega_0^2 x &= 0, \text{ where } \quad \boxed{\omega_0 = \sqrt{\frac{k}{m}} \text{ freq of osc.}} \end{aligned} \tag{52}$$

by magic of ODE, EOM reduces down to:

$$\begin{aligned} \boxed{x(t) = C_1 \cos(\omega_0 t) + C_2 \sin(\omega_0 t)} \\ \text{where } C_1, C_2 \text{ are constants} \end{aligned} \tag{53}$$

by trig magic, this could also be written as

$$\begin{aligned} x(t) &= a \cos(\omega_0 t + \alpha), \\ \text{where } \begin{cases} a = \sqrt{C_1^2 + C_2^2} & \text{amplitude of oscillation} \\ \omega_0 & \text{frequency of oscillation} \\ \tan \alpha = C_2 / C_1 & \text{phase at } t=0 \end{cases} \end{aligned} \tag{54}$$

energy for 1D small Oscillation

checking $\frac{dE}{dt} = 0 \Rightarrow$ energy-conservation:

$$\begin{aligned} E &= T + U = \frac{1}{2} m \dot{x}^2 + \frac{1}{2} kx^2 \\ &= \frac{1}{2} m a^2 \omega_0^2 [\text{constant}] \end{aligned} \tag{55}$$

Damped 1D oscillation, and Complex representation

- when there is damping (friction, resistance, etc) $F_{\text{ric}} = -\beta \dot{x}$, the EOM becomes:

$$\begin{aligned} \ddot{x} + 2\gamma \dot{x} + \omega_0^2 x &= 0, \\ \text{where } 2\gamma &= \frac{\beta}{m}, \omega_0 = \sqrt{\frac{k}{m}} \end{aligned} \tag{56}$$

with ansatz $x(t) = e^{rt}$, $\dot{x} = r e^{rt}$, $\ddot{x} = r^2 e^{rt}$, the solution to Equation 56 is:

$$\begin{aligned} r^2 + 2\gamma r + \omega_0^2 &= 0, \\ \text{which has solution } r_+, r_- &= -\gamma \pm \sqrt{\gamma^2 - \omega_0^2} \\ \Rightarrow x(t) &= C_1 e^{r_+ t} + C_2 e^{r_- t}, \end{aligned} \tag{57}$$

notice the r subscripts here: r_+, r_-

underdamped, overdamped, and critically damped

Recall from your ODE class...

Equation 57 has the following 3 cases, each with different physical interpretation:

$$\begin{aligned} 1. \text{ underdamped:} \quad \quad \quad \left\{ \begin{aligned} r_{\pm} &= -\gamma \pm i \sqrt{\omega_0^2 - \gamma^2} \\ &= -\gamma \pm i\omega \\ &= \sqrt{\omega_0^2 - \gamma^2} \end{aligned} \right. \\ \gamma < \omega_0 \Rightarrow 2 \text{ complex roots:} \end{aligned} \tag{58}$$

The EOM is thus a linear combination of two complex exponentials:

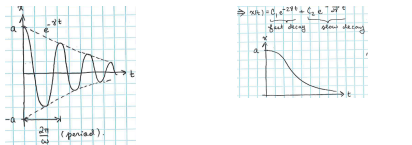
$$\begin{aligned} x(t) &= e^{-\gamma t} (C_1 e^{i\omega t} + C_2 e^{-i\omega t}) \\ &= e^{-\gamma t} (A \cos(\omega t) + B \sin(\omega t)) \\ \text{-- where } \begin{cases} A = C_1 + C_2 \\ B = i(C_1 - C_2) \end{cases} \\ &= ae^{-\gamma t} \cos(\omega t + \alpha) \\ a, \alpha &\text{ are constants} \end{aligned} \tag{59}$$

“The solution is a damped oscillation with frequency ω , and amplitude exponentially decaying with time.”

$$\begin{aligned} 2. \text{ Overdamped} \quad \quad \quad \gamma > \omega \Rightarrow x(t) = \\ c_1 e^{-\gamma + \sqrt{\gamma^2 - \omega^2} t} + c_2 e^{-\gamma - \sqrt{\gamma^2 - \omega^2} t} \end{aligned} \tag{60}$$

$$\begin{aligned} \text{when } \gamma \gg \omega_0, \Rightarrow \left\{ \begin{aligned} \gamma + \sqrt{\gamma^2 - \omega_0^2} &\approx 2\gamma \\ \gamma - \sqrt{\gamma^2 - \omega^2} &= \frac{\omega^2}{2\gamma} \end{aligned} \right. \\ x(t) &= c_1 e^{-2\gamma t} + c_2 e^{-(\omega_0^2/2\gamma)t} \end{aligned} \tag{61}$$

$$\begin{aligned} 3. \text{ Critically damped} \quad \quad \quad \gamma = \omega_0 \Rightarrow x(t) &= c_1 e^{-\gamma t} + c_2 t e^{-\gamma t} \end{aligned} \tag{62}$$



Forced Oscillations

When external force (F) is applied to the system, the largrangian becomes

$$\begin{aligned} L &= \frac{1}{2} m \dot{x}^2 - \frac{1}{2} kx^2 + F(t)x \\ \text{EL} \Rightarrow \ddot{x} + \omega_0^2 x &= \frac{F(t)}{m}, \text{ where } \omega_0 = \sqrt{\frac{k}{m}} \end{aligned} \tag{63}$$

- Example: Simple pendulum with moving pivot

$$\begin{aligned} \begin{cases} \dot{x} = X + l \sin \varphi \\ y = l \cos \varphi \end{cases} \Rightarrow \begin{cases} \dot{x} = \dot{X} + l \dot{\varphi} \cos \varphi \\ \dot{y} = -l \dot{\varphi} \sin \varphi \end{cases} \\ \Rightarrow L = T - U \end{aligned} \tag{64}$$

$$L = \frac{1}{2} m l^2 \dot{\varphi}^2 - mgl(1 - \cos \varphi) - ml\dot{X} \sin \varphi$$

$$\text{Expand ab. } \varphi = 0 \Rightarrow L = \frac{1}{2} m l^2 \dot{\varphi}^2 - \frac{1}{2} mgl\varphi^2 - ml\dot{X}\varphi$$

$$\text{EL} \Rightarrow \quad \ddot{\varphi} + \omega_0^2 \varphi = -\frac{\ddot{X}}{l}, \text{ where } \omega_0 = \sqrt{\frac{g}{l}}$$

reintroducing damping via external forcing

$$\ddot{x} + 2\gamma \dot{x} + \omega_0^2 x = f(t), f(t) = \frac{F(t)}{m} \tag{66}$$

When damping $f(t) = f_0 \cos(\Omega t)$, solution via complex number:

$$\begin{aligned} \ddot{z} + 2\gamma \dot{z} + \omega_0^2 z &= f_0 e^{i\Omega t} \\ \text{ansatz } z(t) &= z_0 e^{i\Omega t} \Rightarrow z_0 = \frac{f_0}{\omega_0^2 + 2i\gamma\Omega + \Omega_0^2} \\ \boxed{z_0 = a(\Omega) \cos(\Omega t + \delta(\Omega))} f_0 &\text{ is a particular solution, where } \\ \left\{ \begin{aligned} a(\Omega) &= \frac{1}{\sqrt{(\omega_0^2 - \Omega^2)^2 + (2\gamma\Omega)^2}} \\ \delta(\Omega) &= \arctan\left(2\gamma \frac{\Omega}{\omega_0^2 - \Omega^2}\right) \end{aligned} \right. \end{aligned} \tag{67}$$

We can study the properties of the system by looking at the amplitude and phase of the solution.

- Amplitude:

$$a(\Omega) = \frac{1}{\sqrt{(\omega_0^2 - \Omega^2)^2 + (2\gamma\Omega)^2}} \tag{68}$$

, when $\gamma \ll \omega_0$, response strongest and amplitude largest when $\omega_r = \omega_0$.



- Phase lag: $\tan \delta(\Omega) = 2\gamma \frac{\Omega}{\Omega^2 - \omega_0^2}$
in phase as $\Omega \rightarrow 0$, and out of phase as $\Omega \rightarrow \omega_0$.
- Genral solution to sinusoidal forcing:

$$\begin{aligned} x(t) &= a(\Omega) f_0 \cos(\Omega t + \delta(\Omega)) + a_0 e^{-\gamma t} \cos(\omega t + \alpha) \\ &\xrightarrow{t \gg \frac{1}{\gamma}} a(\Omega) f_0 \cos(\Omega t + \delta(\Omega)) \end{aligned} \tag{69}$$

Forgets initial condition after time.

- Power obsorbed by oscillation
 $p = F\dot{x} = m f \dot{x}$
Avg power of oscillation

$$\begin{aligned} P_{\text{avg}} &= \frac{1}{T} \int_0^T m f \dot{x} dt = -\frac{1}{2} m f_0 a(\Omega) \Omega \sin \delta(\Omega) \\ \text{simplifies to } P_{\text{avg}}(\Omega) &= \gamma m f_0^2 \Omega^2 a_{\text{r}}^2(\Omega) \end{aligned} \tag{70}$$

Absorption around resonance frequency $\Omega = \omega_0 + \varepsilon$ is maximum:

$$P = \frac{\gamma m f_0^2}{4(\varepsilon^2 + \gamma^2)} \approx \frac{m f_0^2}{4\gamma} \tag{71}$$

Oscillations DOF>1

For a system with n DOF: $q = (q_1, q_2, \dots, q_n)$, PE = $U(q)$

- Stable equilibrium $\frac{\partial U(q)}{\partial q_i}|_{q=0}$

Example: Oscillation with 2 mass and 3 springs

$$\begin{aligned} L &= \frac{1}{2} m \dot{x}_1 + \frac{1}{2} m \dot{x}_2 - \frac{1}{2} kx_1^2 \\ &\quad - \frac{1}{2} kx_2^2 - \frac{1}{2} k'(x_1 - x_2)^2 \end{aligned}$$

EOM:

$$\begin{aligned} M \cdot \ddot{x} &= -K\ddot{x} \quad , \text{ where } M = \begin{pmatrix} m & 0 \\ 0 & m \end{pmatrix}, \\ \ddot{x} &= \begin{pmatrix} x_1 \\ x_2 \end{pmatrix}, K = \begin{pmatrix} k + k' & -k' \\ -k' & k + k' \end{pmatrix} \end{aligned} \tag{72}$$

ansatz: $\ddot{x} = \text{Re}[\tilde{a} e^{i\omega t}]$ Then the EOM eq becomes solving the eigenvalue problem:

$$\begin{aligned} \det(\omega^2 M - K) &= 0 \\ \Rightarrow \left\{ \begin{aligned} \omega^2 &= \frac{k}{m} \\ \omega^2 &= \frac{k+2k'}{m} \end{aligned} \right. \left\{ \begin{aligned} \vec{x}_- &= a_- \begin{pmatrix} 1 \\ 1 \end{pmatrix} \cos(\omega_- t + \delta_-) \\ \vec{x}_+ &= a_+ \begin{pmatrix} 1 \\ -1 \end{pmatrix} \cos(\omega_+ t + \delta_+) \end{aligned} \right. \end{aligned} \tag{73}$$

with constants $a_-, a_+, \delta_-, \delta_+$.

New Coords

$$\begin{aligned} \left\{ \begin{aligned} Q_1 &= \sqrt{\frac{m}{2}}(x_1 + x_2) \\ Q_2 &= \sqrt{\frac{m}{2}}(x_1 - x_2) \end{aligned} \right. \\ \Rightarrow L &= \frac{1}{2} (\dot{Q}_1^2 + \dot{Q}_2^2) - \frac{1}{2} (\omega_-^2 Q_1^2 + \omega_+^2 Q_2^2) \\ &\xrightarrow{\text{E.L.}} \dot{Q}_1 = -\omega_- Q_1, \dot{Q}_2 = -\omega_+ Q_2 \end{aligned} \tag{74}$$

Decoupled oscillators with coords Q_1, Q_2 .

General Coords

for general coords q_i , let $x_i = q_i - q_i^{(0)}$

$$\begin{aligned} U &= \frac{1}{2} \sum_{i,j} k_{ij} x_i x_j, \quad k_{ij} = k_{ji} = \frac{\partial^2 U(q)}{\partial q_i \partial q_j} \text{ symm mat} \\ T &= \frac{1}{2} \sum_{i,j} m_{ij} \dot{x}_i \dot{x}_j, \quad m_{ij} = m_{ji} = a_{ij}(q^{(0)}) \end{aligned} \tag{75}$$

the largrangian, in Matix form:

$$L = \frac{1}{2} \dot{\vec{x}}^T \cdot M \cdot \dot{\vec{x}} - \frac{1}{2} \vec{x}^T \cdot K \vec{x} \xrightarrow{\text{E.L.}} (\omega^2 M - K) \cdot \vec{a} = 0 \tag{76}$$

$\Rightarrow \det(\omega^2 M - K) = 0$ Solving the det for omega gives the normal freq (Eigenvalues)of system ω_n^2 , plug in Evalue into Equation 76 for eigenvec(normal modes) \vec{a}^n of system.

- General motion

$$x_i(t) = \sum_a a_a^i \text{Re}[C_a e^{i\omega_a t}] \tag{77}$$

- EXAMPLE: Normal freq is given

$$\begin{aligned} \omega &= \{0, \sqrt{2}\omega_0, \sqrt{3}\omega_0\}. \\ \omega &= \sqrt{2}\omega_0 \Rightarrow a_1 = -a_3 = -a_2 = ae^{i\delta} \Rightarrow \\ \vec{\theta} &= a(1 \quad -1 \quad -1)^T \cos(\sqrt{2}\omega_0 t + \delta) \\ \omega &= \sqrt{3}\omega_0 \Rightarrow a_1 = 0, a_2 = -a_3 = ae^{i\delta} \Rightarrow \\ \vec{\theta} &= a(0 \quad 1 \quad -1)^T \cos(\sqrt{3}\omega_0 t + \delta) \end{aligned} \tag{78}$$

- EXAMPLE: double pendulum

$$\begin{cases} \dot{x}_1 = l_1 \sin \varphi_1 & y_1 = -l_1 \cos \varphi_1 \\ \dot{x}_2 = l_1 \sin \varphi_1 + l_2 \sin \varphi_2 & y_2 = l_1 \cos \varphi_1 + l_2 \cos \varphi_2 \end{cases} \tag{79}$$

$$\begin{aligned} \Rightarrow T &= \frac{1}{2} m_1 l_1 \dot{\varphi}^2 + \frac{1}{2} m_2 (l_1^2 \dot{\varphi}_1^2 + l_2^2 \dot{\varphi}_2^2 \\ &\quad + 2l_1 l_2 \dot{\varphi}_1 \dot{\varphi}_2 \cos(\varphi_1 - \varphi_2)) \end{aligned} \tag{80}$$

$$U = -m_1 g l_1 \cos \varphi_1 - m_2 g (l_1 \cos \varphi_1 + l_2 \cos \varphi_2)$$

using $\cos \varphi \approx 1 - \frac{\varphi^2}{2}$

$$\begin{aligned} L &= \frac{1}{2} (\dot{\varphi}_1 \quad \varphi_2) \begin{pmatrix} (m_1 + m_2) l_1^2 & m_2 l_1 l_2 \\ m_2 l_1 l_2 & m_2 l_2^2 \end{pmatrix} (\varphi_1 \quad \varphi_2) \\ -\frac{1}{2} (\varphi_1 \quad \varphi_2) \begin{pmatrix} (m_1 + m_2) l_1 g & 0 \\ 0 & m_2 g l_2 \end{pmatrix} (\varphi_1 \quad \varphi_2) \\ &= \frac{1}{2} \dot{\vec{\varphi}}^T M \cdot \dot{\vec{\varphi}} - \frac{1}{2} \vec{\varphi}^T K \vec{\varphi} \end{aligned} \tag{81}$$

$$\text{When } m_1 = m_2 = m, \quad l_1 = l_2 = l \Rightarrow \quad M = ml^2 \begin{pmatrix} 2 & 1 \\ 1 & 1 \end{pmatrix}, K = mgl \begin{pmatrix} 2 & 0 \\ 0 & 1 \end{pmatrix}$$

$$\det((\omega^2 M - K)) = 0 \Rightarrow \omega^2 = \left(2 \pm \sqrt{2}\omega_0^2\right)$$

$$\begin{pmatrix} a_1^- \\ a_2^- \end{pmatrix} = C_- \cdot \begin{pmatrix} 1 \\ \sqrt{2} \end{pmatrix}, \quad \begin{pmatrix} a_1^+ \\ a_2^+ \end{pmatrix} = C_+ \cdot \begin{pmatrix} 1 \\ -\sqrt{2} \end{pmatrix} \tag{82}$$

Normal Coords

$\{x_i\} = \{Q_\alpha\}$, where $x_i = \sum_{\alpha=1}^n A_{i\alpha} Q_\alpha \Rightarrow$

$$\begin{aligned} \sum_j (\omega_\alpha^2 m_{ij} - k_{ij} A_{j\alpha}) &= 0 \\ \Rightarrow L &= \frac{1}{2} \sum_{\alpha=1}^n (\dot{Q}_\alpha^2 - \omega_\alpha^2 Q_\alpha^2) \xrightarrow{\text{E.L.}} \dot{Q}_\alpha + \omega_\alpha^2 Q_\alpha = 0 \end{aligned}$$

Motion of Rigid Body

- Example: rotor

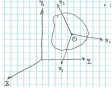
rotation with constraint $|\vec{r}_i - \vec{r}_j| = r_{ij}$,COM coords are useful here

$$\left\{ \begin{array}{l} \vec{r} = \vec{r}_1 - \vec{r}_2 \\ \vec{R} = \frac{m_1 \vec{r}_1 + m_2 \vec{r}_2}{m_1 + m_2} \end{array} \right\} \Rightarrow \left\{ \begin{array}{l} \vec{r}_1 = \vec{R} + m_2 \vec{r} / M \\ \vec{r}_2 = \vec{R} - m_1 \vec{r} / M \end{array} \right.$$

$$L = \frac{1}{2} M \dot{\vec{R}}^2 + \mu \dot{\vec{r}}^2, \quad \mu = m_1 \frac{m_2}{m_1 + m_2}$$

$$\stackrel{\text{polar}}{\Rightarrow} L = \frac{1}{2} M \dot{\vec{R}}^2 + \frac{1}{2} \mu a^2 (\dot{\theta}^2 + \dot{\varphi}^2 \sin^2 \theta)$$

frames of reference

$$(XYZ) \stackrel{R(\theta,\varphi,\psi)}{\Rightarrow} (x_1,x_2,x_3)$$


Velocity of pt in body: $\vec{v} = \vec{V} + \vec{\Omega} \times \vec{r}$, where V is Translational vel, Omega is angular vel, r is position vector.

Largrangian for Rigid Body

$$T = \frac{1}{2} M V^2 + \frac{1}{2} \sum_a m_a \left[\Omega_a^2 r_a^2 - (\vec{\Omega} \cdot \vec{r}_a)^2 \right]$$

$$T_{\text{translational}} + T_{\text{rotational}}$$

consider rotation,

$$\Omega^2 = \sum_i \Omega_i^2, \quad \vec{\Omega} \cdot \vec{r}_a = \sum_i \Omega_i x_{a,i}$$

$$\Rightarrow T_{\text{rot}} = \frac{1}{2} \sum_{i,j} \Omega_i \Omega_j I_{i,j}, \quad I_{i,j} = \sum_a m_a (x_{ij} r_a^2 - x_{a,i} x_{a,j})$$

$$\Rightarrow L = \frac{1}{2} M V^2 + \frac{1}{2} \sum_{i,j} I_{i,j} \Omega_i \Omega_j - U$$

Inertial Tensor

• Discrete

$$I = \left(\begin{array}{ccc} \sum m (y^2 + z^2) & - \sum m x y & - \sum m x z \\ - \sum m x y & \sum m (x^2 + z^2) & - \sum m y z \\ - \sum m x z & - \sum m y z & \sum m (x^2 + y^2) \end{array} \right)$$

• Continuous

$$I_{ij} = \int \rho(x) (\delta_{ij} r^2 - x_i x_j) \, dV$$

$$I_{xx} = \int \rho(x) (y^2 + z^2) \, dV, I_{xy} = I_{yx} = - \int \rho(x) x y \, dV$$

$$I_{yy} = \int \rho(x) (x^2 + z^2) \, dV, I_{yz} = I_{zy} = - \int \rho(x) y z \, dV$$

$$I_{zz} = \int \rho(x) (x^2 + y^2) \, dV, I_{zx} = I_{xz} = - \int \rho(x) z x \, dV$$

example:

$$\begin{aligned} I_{zz} &= \int \left[b^2 \hat{y}^2 + c^2 \hat{z}^2 \right] a b c \, d\hat{x} \, d\hat{y} \, d\hat{z} \\ &= a b c \int \left(b^2 \hat{y}^2 + c^2 \hat{z}^2 \right) d\hat{x} \, d\hat{y} \, d\hat{z} \end{aligned}$$

Transform into spherical coord :

$$\begin{aligned} I_{zz} &= a b c \int \left[b^2 r^2 \sin^2 \theta \sin^2 \phi + c^2 r^2 \cos^2 \theta \right] r^3 \sin \theta \, dr \, d\theta \, d\phi \\ &= a b c \int \left[b^2 \int_0^{2\pi} \sin^2 \phi \, d\phi \int_0^\pi \sin^2 \theta \, d\theta \int_0^R r^4 \, dr \right. \\ &\quad \left. + c^2 \int_0^{2\pi} d\phi \int_0^\pi \cos^2 \theta \sin \theta \, d\theta \int_0^R r^4 \, dr \right] \end{aligned}$$

$$\approx \frac{1}{5} a b c \left[\frac{b^2}{3} + \frac{c^2}{3} \right]$$

• Example: coplanar system principal axis: Z $\Rightarrow I_{13} = I_{23} = 0$
 $I_3 = I_1 + I_2$

Principle axis and principal moments of inertia

In the principal frame:

$$T_{\text{rot}} = \frac{1}{2} (I_1 \Omega_1^2 + I_2 \Omega_2^2 + I_3 \Omega_3^2)$$

$$\begin{array}{l} \bullet \text{ spherical top } I_1 = I_2 = I_3 \\ \bullet \text{ Symmetric top } I_1 = I_2 \neq I_3 \\ \bullet \text{ Asymmetric top } I_1 \neq I_2 \neq I_3 \\ \bullet \text{ EExample:} \end{array}$$

$$\det (I - \lambda 1) = 0 \Rightarrow \lambda \text{ prncp. mom.}$$

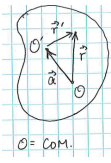
$$\vec{v} = \text{eigenvec.} = \text{prncp. axis}$$

• EExample: continuous with axis of symmetry $\rho(\vec{r}) = \rho = (r, x_3) \Rightarrow I_{ij} = \int \rho(\vec{r}) (\vec{r}^2 \delta - x_i x_j) \, dV$

Parallel axis theorem

when changing Origin diff. from COM(O),

$$I_{ij} =$$

$$I'_{ij} = M (a^2 \delta_{ij} - a_i a_j)$$


For a cube, when finding I at corner, first find I at COM, and

$$I'_{xx} = I_{xx} + M (b^2 + c^2) = \frac{4}{3} M (b^2) + c^2$$

$$I'_{yy} = I_{yy} + M (a^2 + c^2) = \frac{4}{3} M (a^2 + c^2)$$

$$I'_{zz} = I_{zz} + M (a^2 + b^2) = \frac{4}{3} M (a^2 + b^2)$$

$$\begin{aligned} I_{xx} &= - \int d\vec{r} \, g(\vec{r}) \, x_1 x_2 \\ &= - \int d\vec{r} \, x_2 \, r \sin \theta \, d\theta \, d\phi \, g(r, \theta, \phi) \, r \cos \theta \sin \theta \\ &= - \int d\theta_2 \, r \sin \theta \, g(r, \theta, \phi) \, r \, x_2 \int_0^{2\pi} d\phi \cos \theta \\ &= 0 \end{aligned}$$

$$\begin{aligned} I_{xx} &= 0, \quad I_{xy} = a b c \int_0^{2\pi} d\phi \int_0^\pi d\theta \int_0^R dr \, r^3 \sin \theta \cos \theta \sin \theta \cos \theta \\ &\Rightarrow I_{xx} = \left(\frac{I_{xx}}{I_{xx}} \frac{0}{0} \frac{0}{0} \right), \quad x_1, x_2, x_3 = \text{principal axes} \end{aligned}$$

$$\begin{aligned} I_{xx} &= \int d\vec{r} \, g(\vec{r}) \, x_1 x_1 \\ &= - \int d\vec{r} \, x_2 \, r \sin \theta \, d\theta \, d\phi \, g(r, \theta, \phi) \, r^2 \cos^2 \theta \sin \theta \\ &= - \int d\theta_2 \, r \sin \theta \, d\theta \, g(r, \theta, \phi) \, r^2 \int_0^{2\pi} d\phi \cos^2 \theta \sin \theta \\ &\Rightarrow I_{xx} = \left(\frac{I_{xx}}{I_{xx}} \frac{0}{0} \frac{0}{0} \right), \quad x_1, x_2, x_3 = \text{principal axes} \end{aligned}$$

$$\begin{aligned} I_{xx} + I_{yy} &= \int d\vec{r} \, g(\vec{r}) \, (x_1^2 + x_2^2) \\ &= \int d\vec{r} \, x_2 \, r \sin \theta \, d\theta \, d\phi \, g(r, \theta, \phi) \, r^2 \int_0^{2\pi} d\phi (\sin^2 \theta + \cos^2 \theta) \\ &\Rightarrow I_{xx} + I_{yy} = I_{zz} \end{aligned}$$

$$\Rightarrow I_{xx} = \left(\frac{I_{xx}}{I_{xx}} \frac{0}{0} \frac{0}{0} \right) \quad \text{--- axis that is axes in } x_1, x_2, \text{ plane}$$

$$\Rightarrow I_{xx} = \left(\frac{I_{xx}}{I_{xx}} \frac{0}{0} \frac{0}{0} \right) \quad \text{--- axis that is axes in } x_1, x_2, \text{ plane}$$

Angular momentum of a rigid body

\vec{L} in non-inertial frame

$$\vec{L} = \sum m (\vec{r} \times \vec{v}) = \sum m \left[\Omega r^2 - \vec{r} (\vec{\Omega} \cdot \vec{r}) \right]$$

$$L_i = \begin{matrix} & I_{ij} \Omega_j \end{matrix} \quad \vec{L} = I * \vec{\Omega}$$

If $(x_1 x_2 x_3)$ are principal axis, $L_1 = I_1 \Omega_1, L_2 = I_2 \Omega_2, L_3 = I_3 \Omega_3$

Free motion of a rigid body

angular momentum is conserved if no external torque. Motion in inertial COM frame is simpler.

• *ex motion of a symmetric top* $I_1 = I_2 = I_3 = I, \quad \vec{I} = I \begin{pmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{pmatrix}$

$\vec{L} = I \vec{\Omega} \Rightarrow \dot{\vec{L}} = 0 \Rightarrow \dot{\vec{\Omega}} = 0$ Uniform rotation about fixed axis parallel to \vec{L}
• *ex rigid rotor* $I_1 = I_2 = \sum m x_3^2, \quad I_3 = 0$

$\vec{L} = I \vec{\Omega}, \quad \vec{\Omega} \perp x_3$ by geometry We have $\vec{\Omega} = 0 \Rightarrow$ Motion is unif in plane perp to $\vec{\Omega}$ and that it stays in that plane.

• *ex asymmetric top* $I_1 = I_2 = I_\perp \neq I_3 \Rightarrow \vec{I} = \begin{pmatrix} I_\perp & 0 & 0 \\ 0 & I_\perp & 0 \\ 0 & 0 & I_3 \end{pmatrix}$ x_3 is symm. axis,

for any orthogonal axes

Rigid body EOM

$$\begin{cases} \dot{\vec{p}} = \vec{F} \\ \dot{\vec{L}} = \vec{K} \text{ torque} \end{cases} \quad (93)$$

Euler angles: ψ spin, θ nutation, φ precession



$(\theta \in [0, \pi], \varphi \in [0, 2\pi], \psi \in [0, 2\pi])$ in turns of rotation $R = R(\vec{z}, \varphi) R(\vec{X}, \theta) R(\vec{Z}, \psi)$

The lagrangian in Euler angles

• First: $T = \frac{1}{2} (I_1 \Omega_1^2 + I_2 \Omega_2^2 + I_3 \Omega_3^2)$

• Rotation in components:

$$\begin{aligned} \Omega_1 &= \dot{\varphi} \sin \theta \sin \psi + \dot{\theta} \cos \psi \\ \Omega_2 &= \dot{\varphi} \sin \theta \cos \psi - \dot{\theta} \sin \psi \\ \Omega_3 &= \dot{\varphi} \cos \theta + \dot{\psi} \end{aligned} \quad (94)$$

• $T = \frac{1}{2} I_1 (\dot{\varphi} \sin \theta \sin \psi + \dot{\theta} \cos \psi)^2 + \frac{1}{2} I_2 (\dot{\varphi} \sin \theta \cos \psi - \dot{\theta} \sin \psi)^2 + \frac{1}{2} I_3 (\dot{\varphi} \cos \theta + \dot{\psi})^2$

• $L(\theta, \varphi, \psi, \dot{\theta}, \dot{\varphi}, \dot{\psi}) = T - U$

Free motion of symmetric top in Euler angles

$$I_1 = I_2 = I_\perp \Rightarrow T = \frac{1}{2} I_\perp (\dot{\theta}^2 + \dot{\varphi}^2 \sin^2 \theta) + \frac{1}{2} I_3 (\dot{\varphi} \cos \theta + \dot{\psi})^2$$

$$\Omega_\perp = L_z / I_\perp, \quad \Omega_3 = L_z \cos \theta / I_3 \quad \text{E-L-}\Rightarrow$$

$$\begin{aligned} \theta : \frac{d}{dt} I_\perp \dot{\theta} &= I_\perp \sin \theta \cos \theta \, \dot{\varphi}^2 - I_3 \dot{\varphi} \sin \theta (\dot{\varphi} \cos \theta + \dot{\psi}) \\ \varphi : \frac{d}{dt} I_\perp \dot{\varphi} \sin^2 \theta + I_3 \cos \theta (\dot{\varphi} \cos \theta + \dot{\psi}) &= 0 \\ \psi : \frac{d}{dt} I_3 (\dot{\varphi} \cos \theta + \dot{\psi}) &= 0 \end{aligned} \quad (95)$$

choosing \vec{z} along the angular momentum, we have $L_3 = L_z \cos \theta = I_3 \Omega_3 = I_3 (\dot{\varphi} \cos \theta + \dot{\psi})$
 $\Rightarrow L_3 = \text{const} \Rightarrow \theta = \text{const} \quad \Omega_3 = \frac{L_z \cos \theta}{I_3} \quad \dot{\varphi} = \frac{L_z}{I_\perp \cos \theta} = \frac{L_z}{I_\perp} = \text{const}$
• *ex heavy symmetric top with one pt fixed* By parallel axis thm, I'_{ij} + $M(\vec{r}^2 \delta_{ij} - l_i l_j)$
 $\Rightarrow I'_1 = I_1 + M l^2, \quad I'_3 = I_3, \quad U = mgZ = Mgl \cos \theta$
 $\Rightarrow L = T - U = \frac{1}{2} I'_\perp (\dot{\theta}^2 + \dot{\varphi}^2 \sin^2 \theta) + \frac{1}{2} I_3 (\dot{\psi} + \dot{\varphi} \cos \theta)^2 = Mgl \cos \theta$
E-L :

$$\begin{aligned} L_z &= p_\varphi = (I_\perp \sin^2 \theta + I_3 \cos^2 \theta) \dot{\varphi} \quad \text{const} \\ L_3 &= p_\psi = I_3 (\dot{\psi} + \dot{\varphi} \cos \theta) \quad \text{const} \end{aligned} \quad (96)$$

Considering energy conservation

$$E = T + U \Rightarrow E = \frac{L_z^2}{2I_\perp} - Mgl = \frac{1}{2} I'_\perp \dot{\theta}^2 + \frac{1}{2I'_\perp} \frac{(L_z - L_3 \cos \theta)^2}{\sin^2 \theta} - Mgl(1 - \cos \theta)$$

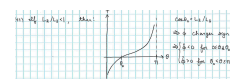
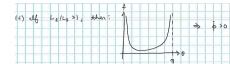
$$U_{\text{eff}}(\theta)$$

effective 1 dof problem. recognizing

$$\dot{\theta} = \frac{d\theta}{dt} \Rightarrow t = \int \frac{d\theta}{(\sqrt{2[E - U_{\text{eff}}(\theta)]/I'_\perp}} \quad (98)$$

Considering U_eff: when $\theta = 0, L_z = L_3$ when $\theta \approx 0 \Rightarrow U_{\text{eff}} \approx \left(\frac{L_z^2}{8I_\perp} - \frac{Mgl}{2} \right) \theta^2$
Motion about $\theta = 0$ is stable if $L_z^2 > 4I'_\perp Mgl \Rightarrow \Omega_3^2 > 4I'_\perp Mgl / I_3^2$, or stable if spinning ab. symm. axis is fast enough.

• Nutation: consider $\dot{\varphi} = \frac{L_3 (L_z / L_3 - \cos \theta)}{I_3 \sin^2 \theta} = \frac{L_z}{I_3} f(\theta)$



considering the sign and trends of $f(\theta)$ given constrains on theta, we can differentiate different nutation motion. If θ_0 in graph 2 is out of range, the nutation is smooth; if θ_0 is in range, the nutation is oscillatory(will change sign and spin in spiral.); if θ_0 is on the endpoint of our constrained range, the nutation is spiky and "not smooth" at points.

Euler equations

set body frame $(X, Y, Z) = (\hat{e}_1^b, \hat{e}_2^b, \hat{e}_3^b)$ space frame $(x_1, x_2, x_3) = (\hat{e}_1, \hat{e}_2, \hat{e}_3)$ Set any vector $\vec{A} = \sum A_i^b \hat{e}_i^b = \sum A_i \hat{e}_i$. By magic of vec analysis,

$$\left(\frac{d\vec{A}}{dt} \right)_{\text{Space}} = \left(\frac{d\vec{A}}{dt} \right)_{\text{Body}} + \vec{\Omega} \times \vec{A}_{\text{Space}} \quad (99)$$

When applied to $\left(\frac{d\vec{L}}{dt} \right)_{\text{Space}} = \vec{K} = \left(\frac{d\vec{L}}{dt} \right)_{\text{body}} + \vec{\Omega} \times \vec{L}$, recognizing $L_i = I_i \Omega_i$:

$$\begin{aligned} I_1 \dot{\Omega}_1 + (I_3 - I_2) \Omega_2 \Omega_3 &= K_1 \\ I_2 \dot{\Omega}_2 + (I_1 - I_3) \Omega_3 \Omega_1 &= K_2 \\ I_3 \dot{\Omega}_3 + (I_2 - I_1) \Omega_1 \Omega_2 &= K_3 \end{aligned} \quad (100)$$

$K_i = 0$ if \vec{L} is conserved on i axis.

• *ex symmetric top* $I_1 = I_2 = I, \vec{K} = 0 \quad (\dot{\Omega}_1 + \frac{I_3 - I}{I} \Omega_2 \Omega_3 = 0; \Omega_2 + \frac{I_3 - I}{I} \Omega_3 \Omega_1 = 0; \Omega_3 = 0)$ let $\omega = ((I_3 - I) / (I)) \Omega_3 \Rightarrow$

$$\left(\Omega_1 = A \cos \omega t; \Omega_2 = -\frac{1}{\omega} \dot{\Omega}_1 = A \sin \omega t \right)$$

Motion in non-inertial frame

• Set non-inertial frame with velocity $\vec{V}(t), \vec{A} = \dot{\vec{V}}, \quad \vec{v} = \vec{v}' + \vec{V}(t)$ where \vec{v}' is velocity w.r.t. non-inertial frame.

lagrangian $L' = \frac{1}{2} m \vec{v}'^2 - m \vec{r}' \cdot \vec{A} - U$, using E-L eq: $m \vec{v}' = - \frac{\partial U}{\partial \vec{r}'} - m \vec{A}$

• *ex pendulum in acc. car* $m \vec{r}' = \vec{T} + m \vec{g} - m \vec{A}$,

finding equil. angle: $\vec{T} = -m(\vec{g} - \vec{A}) = -m \vec{g}_{\text{eff}}$, then use geometry between $\vec{g}, -\vec{A} \Rightarrow \tan \varphi_0 = \frac{A}{g}$. Oscillation freq. $\omega = \sqrt{g_{\text{eff}} / l}$

Motion in rotating frame

Set rotation with $\vec{\Omega}, L = \frac{1}{2} m v^2 + m \vec{v} \cdot (\vec{\Omega} \times \vec{r}) + \frac{1}{2} m (\vec{\Omega} \times \vec{r})^2 - m \vec{r} \cdot \vec{A} - U$

$$\text{Using E-L,}$$

$$\boxed{m \vec{v} = - \frac{\partial U}{\partial \vec{r}} - m \vec{A} + 2m(\vec{v} \times \vec{\Omega}) + m \vec{\Omega} \times (\vec{r} \times \vec{\Omega}) + m \vec{r} \times \dot{\vec{\Omega}}}$$

• Namely,

$$\begin{aligned} m \vec{v} &= - \frac{\partial U}{\partial \vec{r}} + \vec{F}_{\text{cor}} + \vec{F}_{\text{cent}} \\ \vec{F}_{\text{Cor}} &= 2m(\vec{v} \times \vec{\Omega}), \quad \vec{F}_{\text{cent}} = m \vec{\Omega} \times (\vec{r} \times \vec{\Omega}) = m(\vec{\Omega} \times \vec{r}) \times \vec{\Omega} \end{aligned} \quad (101)$$

• *ex free fall on earth, centrifugal force* $\vec{F} = \vec{g}_0 + m \Omega^2 R \sin \theta \hat{\rho} \Rightarrow \vec{g}_{\text{eff}} = \vec{g}_0 + \Omega^2 R \sin \theta \hat{\rho}$

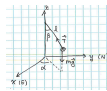
• *ex free fall, coriolis force* $\vec{v} = \vec{g} + 2\vec{v} \times \vec{\Omega}, \quad \vec{\Omega} = \Omega \sin \theta \hat{y} + \Omega \cos \theta \hat{z}$

In components,

$$\begin{aligned} \vec{v}_x &= 2\Omega (v_y \cos \theta - v_z \sin \theta) \\ \vec{v}_y &= -2\Omega v_x \cos \theta \\ \vec{v}_z &= 2\Omega v_x \sin \theta - g \end{aligned} \quad (102)$$

Free fall EOM: $\vec{R} = \int v \, dt$, consider $\vec{v} = \vec{v}_1 + \vec{v}_2 = -\vec{g} + 2\vec{v}_1 \times \vec{\Omega} + 2\vec{v}_2 \times \vec{\Omega}$ where approximately, $\vec{v}_2 = 2(\vec{v}_0 - g\vec{t}) \times \vec{\Omega}$. If no initial velocity, integrating velocity in x components gives, $x(t) = \frac{1}{3} g \Omega \left(\frac{2b}{g} \right)^{3/2} \sin \theta$

• *ex fouchaults pendulum* EOM

$$\begin{aligned} \vec{r} &= l \sin \beta \cos \alpha \hat{x} + l \sin \beta \sin \alpha \hat{y} + (l - l \cos \beta) \hat{z} \\ \vec{T} &= -T \sin \beta \cos \alpha \hat{x} - T \sin \beta \sin \alpha \hat{y} + T \cos \beta \hat{z} \\ \vec{\Omega} &= \Omega \sin \theta \hat{y} + \Omega \cos \theta \hat{z} \end{aligned}$$


$$\begin{cases} T = mg \\ m \ddot{x} = T_x + 2m \dot{x} \cdot (\dot{\vec{r}} \times \vec{\Omega}) = -\frac{mgx}{2m} + 2m \Omega \dot{y} \cos \theta \\ m \ddot{y} = -\frac{mgy}{2m} - 2m \Omega \dot{x} \cos \theta \end{cases} \quad (103)$$

$$\text{letting } \omega^2 = \frac{g}{2}, \quad \Omega \pm \omega \cos \theta, \quad \boxed{\eta = x + iy = e^{i\omega t}}$$

$$\begin{aligned} \ddot{x} + \omega^2 x &= 2\Omega_2 \dot{y}, \dot{y} + \omega^2 y = -2\Omega_2 \dot{x} \\ \gamma &= -\Omega_2 \pm \sqrt{\omega^2 - \Omega_2^2} \\ \eta(t) &= a e^{-i\Omega_2 t} \cos \omega t \\ \Rightarrow \begin{cases} x = a \cos \Omega_2 t \cos \omega t \\ y = a \sin \Omega_2 t \cos \omega t \end{cases} \end{aligned} \quad (104)$$

Hamiltonian Mechanics

$H(q, p, t) = \sum_{j=1}^n p_j \dot{q}_j - L(q, \dot{q}, t) \quad 1D: H = \frac{p^2}{2m} + U(x)$

• Hamilton's equation $\dot{q}_i = \frac{\partial H}{\partial p_i}, \quad \dot{p}_i = -\frac{\partial H}{\partial q_i}$

• *ex particle in polar*

$$L = T - U = \frac{1}{2} m (\dot{r}^2 + r^2 \dot{\varphi}^2) - U(r, \varphi) \Rightarrow p_r = \frac{\partial L}{\partial \dot{r}} = m \dot{r}, p_\varphi = \frac{\partial L}{\partial \dot{\varphi}} = m r^2 \dot{\varphi}$$

$$H = p_r \dot{r} + p_\varphi \dot{\varphi} - L = \frac{p_r^2}{2m} + \frac{p_\varphi^2}{2mr^2} \Rightarrow \dot{r} = \frac{\partial H}{\partial p_r} = \frac{p_r}{m}, \quad \dot{\varphi} = \frac{\partial H}{\partial p_\varphi} = \frac{p_\varphi}{mr^2}$$

$$\dot{p}_r = -\frac{\partial H}{\partial r} = \frac{p_\varphi^2}{mr^3} - \frac{\partial U}{\partial r}, \quad \dot{p}_\varphi = -\frac{\partial H}{\partial \varphi} = -\frac{\partial U}{\partial \varphi}$$

Phase space

• *ex harmonic oscillator* $H = \frac{p^2}{2m} + (\frac{1}{2}) m \omega^2 x^2, \quad \omega = \sqrt{\frac{k}{m}}$

$$\left\{ \dot{x} = \frac{\partial H}{\partial p} = \frac{p}{m}, \quad \dot{p} = -\frac{\partial H}{\partial x} = -m \omega^2 x \right\} \Rightarrow \left\{ \dot{q} = \frac{p}{m}, \quad \dot{p} = -m \omega^2 q \right\}$$

$q(t_0 + \delta t) = q(t_0) + \dot{q} \delta t = q_0 + \frac{p_0}{m} \delta t; \quad p(t_0 + \delta t) = p(t_0) + \dot{p} \delta t = p_0 - m \omega^2 q \delta t$ parametric ellipse in phase space.

Liouville's thm

volume of a region op phase space is conserved under time evolution, when boundary of volume and all pts inside move along their orbit for some amount of time.

Poisson bracket

Time evolution of an observable $A(q, p, t)$:

More generally, for $A(q,p,t),\quad B(q,p,t)$

$$\{A,B\}=\sum_i\frac{\partial A}{\partial q_i}\frac{\partial B}{\partial p_i}-\frac{\partial A}{\partial p_i}\frac{\partial B}{\partial q_i}\tag{109}$$

notice, $\{A,p_i\}=\frac{\partial A}{\partial q_i},\{A,q_i\}=-\frac{\partial A}{\partial p_i}$,

- When

$$\frac{dC}{dt}=\frac{\partial C}{\partial t}+\{C,H\}=0\tag{110}$$

then $C(q,p,t)$ is conserved.

Cononical transformation

consider transformation $q_i\rightarrow Q_i(q,t)$ the transformation is canonical iff the transformation leave the form of Hamilton's eq. unchanged.

$$\begin{cases} \dot{q}=\frac{\partial H}{\partial p} \Rightarrow \text{cases } \dot{Q}=\frac{\partial K}{\partial P}, \dot{P}=-\frac{\partial K}{\partial Q} \\ \dot{p}=-\frac{\partial H}{\partial q} \end{cases}\tag{111}$$

where $K(Q,P,t)$ new Hamiltonian.

Exerts from practice problems

constraints, small Oscillations

A particle of mass m moves without friction on the inside wall of an axially symmetric vessel given by $z=b^2(x^2+y^2)$

- KE in cylindrical coords:

$$\begin{aligned} T &= \frac{1}{2}m(\dot{\rho}^2+\rho^2\dot{\theta}^2+\dot{z}^2),\quad \dot{z}=b\dot{\rho}\rho\Rightarrow \\ L &= \frac{m}{2}\left[\dot{\rho}^2(1+b^2\rho^2)+\rho^2\dot{\theta}^2\right]-\frac{mgb}{2}\rho^2 \end{aligned}\tag{112}$$

E-L:

$$\begin{aligned} \ddot{\rho}(1+b^2\rho^2)+b^2\dot{\rho}^2\rho-\rho\dot{\theta}^2+gb\rho &=0 \\ m\rho^2\dot{\theta}=\text{const}\equiv M \quad \text{conserved angular momentum} \end{aligned}\tag{113}$$

- energy and angular momentum given z_0,b,g,m

$$E=\frac{m}{2}\left[\dot{\rho}^2(1+b^2\rho^2)+\rho^2\dot{\theta}^2\right]+\frac{mgb}{2}\rho^2\tag{114}$$

For a fixed z_0,ρ_0 is the equilibrium position, and $\dot{\rho}=0$, then

$$\begin{aligned} E &= \frac{m}{2}\rho_0^2\dot{\theta}^2+mgb\frac{\rho_0^2}{2} \\ \dot{\theta}^2 &= gb \\ \Rightarrow E &= 2mgz_0 \end{aligned}\tag{115}$$

plugging in $\dot{\theta},\rho=\rho_0$, we have $M=2mz_0\sqrt{\frac{g}{b}}$

- frequency of small oscillations about equilibrium perturbation: $\rho=\rho_0+\varepsilon$, neglecting anything with ε^2 , EOM of rho is

$$\ddot{\varepsilon}(1+b^2\rho_0^2)-\rho\dot{\theta}^2+gb\rho_0+gb\varepsilon=0\tag{116}$$

want to know $\rho\dot{\theta}^2$, can be found from θ EOM

$$\begin{aligned} \rho\dot{\theta}^2 &= \frac{M^2}{m^2\rho^3}=\frac{M^2}{m^2\rho_0^3}\left(\frac{1}{\left(1+\frac{\varepsilon}{\rho_0}\right)^3}\right)\approx\frac{M^2}{m^3\rho_0^4}\left(1-3\frac{\varepsilon}{\rho_0}\right) \\ &= b\rho_0g-3bg\varepsilon \end{aligned}\tag{117}$$

Plugging in to rho EOM, we have

$$\begin{aligned} \ddot{\varepsilon}(1+2bz_0)+4gb\varepsilon &=0 \\ \ddot{\varepsilon}=-\omega^2\varepsilon,\Omega^2 &= \frac{4gb}{1+2bz_0} \end{aligned}\tag{118}$$

Conservation laws

two particles of $\{m_1,q_1,\vec{r}_1\},\{m_2,q_2,\vec{r}_2$ in capacitor with $\vec{E}=E_0\hat{z}$, particles interact with $U(r_1,r_2)=\frac{k}{|\vec{r}_1-\vec{r}_2|}e^{-\frac{q_1q_2}{|\vec{r}_1-\vec{r}_2|}}$. List all conserved quantities and associate each with a specific symmetry of the problem.

- lagrangian $L=\frac{1}{2}m_1\dot{\vec{r}}_1^2+\frac{1}{2}m_2\dot{\vec{r}}_2^2-U+E_0(q_1z_1+q_2z_2)$. Setting $\vec{r}=(x,y,z)=\vec{r}_1-\vec{r}_2,\vec{R}=(X,Y,Z)=\frac{m_1\vec{r}_1+m_2\vec{r}_2}{M},\mu=\frac{m_1m_2}{M}$, we can have

$$L=\left[\frac{1}{2}M\dot{\vec{R}}^2+(q_1+q_2)E_0Z\right]+\left[\frac{1}{2}\mu\dot{\vec{r}}^2-U(r)+\frac{q_1m_2-q_2m_2}{M}E_0z\right]\tag{119}$$

Observe: momenta $P_x=\frac{\partial L}{\partial \dot{X}},P_y=\frac{\partial L}{\partial \dot{Y}}$ are conserved. Invariance under time translation gives conserved energy

$$E=\frac{\partial L}{\partial \dot{\vec{R}}}\dot{\vec{R}}+\frac{\partial L}{\partial \dot{\vec{r}}}\dot{\vec{r}}-L\tag{120}$$

Angular momentum $L_{\text{tot}}=\vec{r}_1\times\vec{p}_1+\vec{r}_2\times\vec{p}_2=M\vec{R}\times\dot{\vec{R}}+\mu\vec{r}\times\dot{\vec{r}}=\vec{R}\times\dot{\vec{P}}+\vec{r}\times\dot{\vec{p}}$. Invariance under rotation about $\hat{z}:R\rightarrow R+\varepsilon\hat{z}\times R,\quad r\rightarrow\vec{r}+\varepsilon\hat{z}\times\vec{r}$ gives conserved $L_z=\left(\vec{R}\times\dot{\vec{P}}\right)_z\quad L_z=|\vec{r}\times\dot{\vec{p}}|_z$.

Normal modes

A system of N particles with masses m_i moves around a circle of radius a , with position angle θ_i . Interaction potential $U=\frac{k}{2}\sum_1^N(\theta_{j+1}-\theta_j)^2$, with $\theta_{N+1}=\theta_1+2\pi$. lagrangian of system is $\frac{a^2}{2}\sum_1^Nm_j\dot{\theta}_j^2-U$

- show Lagrangian for particle i , show system in equilibrium when particles are equally spaced.

$$L=\frac{a^2}{2}\sum_1^Nm_j\dot{\theta}_j^2-\frac{k}{2}\sum_1^N(\theta_{j+1}-\theta_j)^2\tag{121}$$

E-L for $\theta_i:\quad a^2m_i\ddot{\theta}_i=k(\theta_{i+1}-\theta_i)-k(\theta_i-\theta_{i-1})=-k[2\theta_i-(\theta_{i+1}+\theta_{i-1})]$
When equally spaced, $\theta_i=\frac{2\pi i}{N}$, thus $\ddot{\theta}_i=0$ for all particles, thus equilibrium.

- show the system always has a normal mode of osc. with 0 freq.

$$\mathbb{M}\cdot\ddot{\vec{\theta}}=-\mathbb{K}\cdot\vec{\theta},\quad M_{ij}=a^2m_i\delta_{ij},\quad K_{ij}=k(2\delta_{i,j}-\delta_{i,j+1}-\delta_{i,j-1})\tag{122}$$

take anstaz subsitution $\vec{\theta}\rightarrow\vec{z}=\vec{b}e^{i\omega t}$ gives $\omega^2\mathbb{M}\cdot\vec{b}=\mathbb{K}\cdot\vec{b}$, where \vec{b} is a constant vec. Look for a 0 freq $\omega=0$, $\mathbb{K}\cdot\vec{b}=0$ holds, so $b_i=b$. let $b=\Theta(t)$, knowing $\dot{\Theta}=0$ recall our substitution, the time evo of $\theta_{i(t)}=\Theta_0+\Theta_1t$ i.e. trajectory is all masses rotating at same rate Θ_1

- find all normal modes when $N=2,M_1=km/a^2,m_2=2km/a^2$. Using standard normal mode analysis, for $N=2$,

Appendix

- Taylor expansion:

$$f(x)|_0\approx f(a)+f'(a)(x-a)+f''(a)\frac{(x-a)^2}{2}\tag{123}$$

- small angle approximation:

$$\sin(\theta)\approx\theta\quad\cos(\theta)\approx1-\frac{\theta^2}{2}\tag{124}$$