#### PHY 293 Lecture Notes

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**PHY293** 

The up-to-date version of this document can be found at https://github.com/HaysonC/skulenotes

## Chapter 1

## Waves

#### 1.1 Harmonic Oscillators

#### 1.1.1 Governing Equations of Harmonic Oscillators

This subsection collects the baseline ODEs for simple, damped, and driven oscillators to set notation used later.

**Types of Harmonic Oscillators** There are three types of harmonic oscillators: simple, damped, and driven harmonic oscillators. Consider a simple one dimensional harmonic oscillator, they are defined by the following differential equations:

**Definition 1.1.1.1** (Simple Harmonic Oscillator). A simple harmonic oscillator is described by Hooke's law:

$$m\frac{d^2x}{dt^2} + kx = 0\tag{1.1}$$

where k is the spring constant, m is the mass, and x is the displacement from equilibrium.

**Definition 1.1.1.2** (Damped Harmonic Oscillator). A damped harmonic oscillator is described by the following differential equation, by adding a damping term proportional to  $\dot{x}$  to the simple harmonic oscillator equation:

$$m\frac{d^2x}{dt^2} + b\frac{dx}{dt} + kx = 0 ag{1.2}$$

where b is the damping coefficient.

**Note on damping parameter conventions.** Different texts use different symbols and normalizations:

- Our notes use  $\gamma = \frac{b}{m}$ , so the ODE reads  $\ddot{x} + \gamma \dot{x} + \omega_0^2 x = 0$  and the decay envelope is  $e^{-\gamma t/2}$ .
- Many texts define  $\beta = \frac{b}{2m}$  and may call it  $\gamma$  instead. In that convention the ODE is  $\ddot{x} + 2\beta \dot{x} + \omega_0^2 x = 0$  with envelope  $e^{-\beta t}$ .

Equivalences:  $\beta = \frac{\gamma}{2}$  and our underdamped condition  $\gamma < 2\omega_0$  corresponds to  $\beta < \omega_0$  in the  $2\beta$  convention. When comparing formulas, check which definition is being used and replace  $\gamma \leftrightarrow 2\beta$  accordingly.

**Definition 1.1.1.3** (Driven Harmonic Oscillator). A driven harmonic oscillator is described by the following differential equation, which includes an external driving force F(t):

$$m\frac{d^2x}{dt^2} + b\frac{dx}{dt} + kx = F(t)$$
(1.3)

#### 1.1.2 Simple Harmonic Motion

We analyze the undamped solution forms, relate constants to initial conditions, and derive period/frequency relations.

**Definition 1.1.2.1** (Simple Harmonic Motion). You should have learned Hooke's law and Newton's second law, which give us the equation of motion for a simple harmonic oscillator. The same as equation (1.1), which can be rewritten as:

$$F = m\ddot{x} = -kx \tag{1.4}$$

By setting  $\omega^2 = \frac{k}{m}$ , a general solution can be written as:

$$x(t) = x_0 + A_1 \cos(\omega t) + A_2 \sin(\omega t) \tag{1.5}$$

where  $A_1$  and  $A_2$  are constants determined by the IVP,  $\omega$  is the angular frequency, and  $\phi$  is the phase constant.  $x_0$  is the equilibrium position (often set to 0). The unknown constants can be determined by knowing  $x, \dot{x}$  at specific times.

**Definition 1.1.2.2** (Period, Frequency, and Angular Frequency). The period T is the time it takes for one complete cycle of the motion, given by:

$$T = 2\pi \sqrt{\frac{m}{k}} \tag{1.6}$$

The frequency f is the number of cycles per unit time, given by:

$$f = \frac{1}{T} = \frac{1}{2\pi} \sqrt{\frac{k}{m}} \tag{1.7}$$

The angular frequency  $\omega$  is related to the frequency by:

$$\omega = 2\pi f = \sqrt{\frac{k}{m}} \tag{1.8}$$

**Example 1.1.2.3.** A simple harmonic oscillator consisting of mass m = 11.0 kg attached to a spring with spring constant  $k = 201 \text{ N m}^{-1}$ . At time t = 0 s the oscillator is at position x(0) = -0.207 m and has velocity  $v(0) = -1.33 \text{ m s}^{-1}$ . Determine all coefficients of the equation describing the position x(t) of the oscillator as a function of time, assuming the offset is zero.

To solve for  $A_1$  and  $A_2$ , while we assume  $x_0 = 0$ , we can use the initial conditions:

$$x(0) = A_1 \cos(0) + A_2 \sin(0) = A_1 = -0.207 \text{ m}$$
  
 $v(0) = -A_1 \omega \sin(0) + A_2 \omega \cos(0) = A_2 \omega = -1.33 \text{ m s}^{-1}$ 

We can find  $\omega$  from the given m and k:

$$\omega = \sqrt{\frac{k}{m}} = \sqrt{\frac{201 \text{ N m}^{-1}}{11.0 \text{ kg}}} \approx 4.28 \text{ rad s}^{-1}$$

Therefore, we can solve for  $A_2$ :

$$A_2 = \frac{v(0)}{\omega} = \frac{-1.33 \text{ m s}^{-1}}{4.28 \text{ rad s}^{-1}} \approx -0.311 \text{ m}$$

Thus, the equation describing the position x(t) of the oscillator as a function of time is:

$$x(t) = -0.207\cos(4.28t) - 0.311\sin(4.28t)$$

**Theorem 1.1.2.4** (A Trigonometric Identity). We can also express the solution in a more compact form using a single cosine function with a phase shift:

$$x(t) = A\cos(\omega t + \phi) \tag{1.9}$$

where

$$A = \sqrt{A_1^2 + A_2^2},\tag{1.10a}$$

$$\phi = \arctan\left(\frac{-A_2}{A_1}\right) = \arctan\left(\frac{-v(0)/\omega}{x(0)}\right).$$
 (1.10b)

*Proof.* Let  $A = \sqrt{A_1^2 + A_2^2}$  and choose  $\phi$  such that

$$\cos(\phi) = \frac{A_1}{A}, \quad \sin(\phi) = -\frac{A_2}{A}.$$

Then, we can rewrite our original solution as

$$x(t) = A_1 \cos(\omega t) + A_2 \sin(\omega t)$$

$$= A \cos(\phi) \cos(\omega t) - A \sin(\phi) \sin(\omega t)$$

$$= A \left[\cos(\phi) \cos(\omega t) - \sin(\phi) \sin(\omega t)\right]$$

$$= A \cos(\omega t + \phi),$$

by the cosine addition formula.

**Example 1.1.2.5.** To determine the amplitude A and phase constant  $\phi$  for the oscillator in the previous example, we can use the values of  $A_1$  and  $A_2$  we found:

$$\begin{split} A &= \sqrt{(-0.207)^2 + (-0.311)^2} \approx 0.374 \text{ m} \\ \phi &= \arctan\!\left(\frac{-(-0.311)}{-0.207}\right) \approx 4.12 \text{ rad} \quad \text{(since $A_1 < 0$ and $A_2 < 0$)} \end{split}$$

Therefore, the equation describing the position x(t) of the oscillator as a function of time can also be written as:

$$x(t) = 0.374\cos(4.28t + 4.12)$$

**Definition 1.1.2.6** (The Energy of a Simple Harmonic Oscillator). The total mechanical energy E of a simple harmonic oscillator is the sum of its kinetic energy K and potential energy U.

$$E = K + U \tag{1.11}$$

First we consider the change of potential energy from a position  $x_i$  to  $x_f$ , assuming the path is along the spring or the curve C of the oscillator. The force exerted by the spring is given by Hooke's law, F = -kx. The change in potential energy can be simply parametized and calculated as follows:

$$\Delta U = \int_C F \cdot ds = -\int_{x_i}^{x_f} F \, dx = \int_{x_i}^{x_f} kx \, dx = \left[ \frac{1}{2} kx^2 \right]_{x_i}^{x_f} = \frac{1}{2} k(x_f^2 - x_i^2) \tag{1.12}$$

Therefore, the potential energy U at a position x (taking the reference point at x=0) is given by:

$$U(x) = \frac{1}{2}kx^2 (1.13)$$

The kinetic energy K of the oscillator is given by:

$$K = \frac{1}{2}m\dot{x}^2\tag{1.14}$$

Therefore, the total mechanical energy E of the simple harmonic oscillator is:

$$E = K + U = \frac{1}{2}m\dot{x}^2 + \frac{1}{2}kx^2 \tag{1.15}$$

The total mechanical energy E remains constant over time, as energy is conserved in the absence of non-conservative forces (like friction or air resistance).

#### 1.1.3 Damped Harmonic Motion

We solve the damped ODE, classify regimes (underdamped, critical, overdamped), and connect decay rates with parameters.

**Definition 1.1.3.1** (Damped Harmonic Motion). For small velocities, the drag force is approximately proportional to the velocity and acts in the opposite direction. This drag force can be modeled as  $F_d = -\gamma \dot{x}$ , where  $\gamma$  is the damping coefficient. Including this drag force in the equation

of motion for a harmonic oscillator leads to the damped harmonic oscillator equation (1.2). Which could be rewritten as:

$$\ddot{x} + \gamma \dot{x} + \omega_0^2 x = 0 \tag{1.16}$$

where  $\omega_0 = \sqrt{\frac{k}{m}}$  is the natural angular frequency of the undamped oscillator, and  $\gamma = \frac{b}{m}$  is the damping coefficient per unit mass.

To skip the math, lets assume a solution of the form  $x(t) = e^{i\omega t}$ , substituting into the differential equation gives us a formulation for  $\omega$ :

$$-\omega^2 - i\gamma\omega + \omega_0^2 = 0$$

$$\omega = -i\frac{\gamma}{2} \pm \sqrt{\omega_0^2 - \frac{\gamma^2}{4}}$$
(1.17)

Also, we can characterize the real and imaginary parts of  $\omega$  as:

$$\omega_r = \text{Re}(\omega) = \pm \sqrt{\omega_0^2 - \frac{\gamma^2}{4}}$$
 (1.18a)

$$\omega_i = \operatorname{Im}(\omega) = -\frac{\gamma}{2} \tag{1.18b}$$

The general solution for the damped harmonic oscillator can be written as:

$$x(t) = \exp(\omega_i t) \exp(-i\omega_r t) = \exp\left(-\frac{\gamma}{2}t\right) \exp(\mp i\sqrt{\omega_0^2 - \frac{\gamma^2}{4}}t)$$
 (1.19)

• No Damping ( $\gamma = 0$ ): The system behaves like a simple harmonic oscillator with angular frequency  $\omega_0$ . Given by:

$$z = \exp(-i\omega_0 t) \tag{1.20a}$$

• Underdamping  $(0 < \gamma < 2\omega_0)$ : The system oscillates with a gradually decreasing amplitude. The angular frequency of oscillation is given by  $\omega_r = \sqrt{\omega_0^2 - \frac{\gamma^2}{4}}$ . Given by:

$$z = \exp\left(-\frac{\gamma}{2}t\right) \exp(-i\omega_r t) \tag{1.20b}$$

The trigonometric form of the solution is:

$$x(t) = A_0 \exp\left(-\frac{\gamma}{2}t\right) \cos(\omega_r t + \phi)$$
 (1.20c)

where  $A_0$  and  $\phi$  are constants determined by the initial conditions From this, we can derive the following cases:

• Critical Damping ( $\gamma = 2\omega_0$ ): The system returns to equilibrium as quickly as possible without oscillating. Consider:

$$x(t) = e^{-\frac{\gamma}{2}t} f(t)$$

Inserting into the differential equation, we get:

$$\ddot{f} + \left(\omega_0^2 - \frac{\gamma^2}{4}\right)f = 0$$

Since  $\gamma = 2\omega_0$ , we have  $\omega_0^2 - \frac{\gamma^2}{4} = 0$ , leading to:

$$\ddot{f} = 0 \implies f(t) = A_1 t + A_2$$

Therefore, the general solution for the critically damped case is:

$$x(t) = (A_1t + A_2) \exp\left(-\frac{\gamma}{2}t\right) \tag{1.20d}$$

where  $A_1$  and  $A_2$  are constants determined by

• Overdamping ( $\gamma > 2\omega_0$ ): The system returns to equilibrium without oscillating, but more slowly than in the critically damped case. The solution is given by:

$$z = \exp\left(-\frac{\gamma}{2}t\right) \exp\left(\sqrt{\frac{\gamma^2}{4} - \omega_0^2}t\right)$$
 (1.20e)

So the general solution is (the solution is via a substitution of  $x(t) = e^{-\gamma t/2} f(t)$  into the differential equation, which resolves the ODE to a simple form):

$$x(t) = A_1 \exp\left[\left(-\frac{\gamma}{2} + \sqrt{\frac{\gamma^2}{4} - \omega_0^2}\right)t\right] + A_2 \exp\left[\left(-\frac{\gamma}{2} - \sqrt{\frac{\gamma^2}{4} - \omega_0^2}\right)t\right]$$
(1.20f)

where  $A_1$  and  $A_2$  are constants determined by the initial conditions.

#### 1.1.4 Energy and Quality Factor

We study how energy decays under light damping, define the time constant and quality factor Q, and relate them to response.

**Definiton 1.1.4.1** (Energy of a Very Light Damping). Consider a very lightly damped harmonic oscillator, where  $\gamma \ll \omega_0$ . In this case, the angular frequency of oscillation  $\omega_r$  can be approximated as:

$$\omega_r \approx \omega_0 \left( 1 - \frac{\gamma^2}{8\omega_0^2} \right) \approx \omega_0$$

So the motion of the lightly damped oscillator can be approximated as:

$$x(t) \approx A_0 e^{-\frac{\gamma}{2}t} \cos(\omega_0 t + \phi)$$

Then, we can calculate the velocity of the oscillator:

$$\dot{x}(t) = -\frac{\gamma}{2} A_0 e^{-\frac{\gamma}{2}t} \cos(\omega_0 t + \phi) - A_0 \omega_0 e^{-\frac{\gamma}{2}t} \sin(\omega_0 t + \phi)$$
$$= A_0 \omega_0 e^{-\frac{\gamma}{2}t} \left( -\frac{\gamma}{2\omega_0} \cos(\omega_0 t + \phi) - \sin(\omega_0 t + \phi) \right)$$

The total mechanical energy E(t) of the lightly damped oscillator is given by:

$$E(t) = \frac{1}{2}m\dot{x}^2 + \frac{1}{2}kx^2$$

$$= \frac{1}{2}m\left[A_0\omega_0e^{-\frac{\gamma}{2}t}\left(-\frac{\gamma}{2\omega_0}\cos(\omega_0t + \phi) - \sin(\omega_0t + \phi)\right)\right]^2 + \frac{1}{2}k\left[A_0e^{-\frac{\gamma}{2}t}\cos(\omega_0t + \phi)\right]^2$$

$$= \frac{1}{2}mA_0^2\omega_0^2e^{-\gamma t}\left[\left(-\frac{\gamma}{2\omega_0}\cos(\omega_0t + \phi) - \sin(\omega_0t + \phi)\right)^2 + \cos^2(\omega_0t + \phi)\right]$$

$$= \frac{1}{2}mA_0^2\omega_0^2e^{-\gamma t}\left[\sin^2(\omega_0t + \phi) + \cos^2(\omega_0t + \phi) + \frac{\gamma^2}{4\omega_0^2}\cos^2(\omega_0t + \phi) + \frac{\gamma}{\omega_0}\sin(\omega_0t + \phi)\cos(\omega_0t + \phi)\right]$$

$$\approx \frac{1}{2}mA_0^2\omega_0^2e^{-\gamma t}\left[1 + \frac{\gamma^2}{4\omega_0^2}\cos^2(\omega_0t + \phi)\right] \quad \text{(neglecting the small term } \frac{\gamma}{\omega_0}\sin(\omega_0t + \phi)\cos(\omega_0t + \phi)\right]$$

$$\approx \frac{1}{2}mA_0^2\omega_0^2e^{-\gamma t} \quad \text{(since } \frac{\gamma^2}{4\omega_0^2} \text{ is very small)}$$

$$= E_0e^{-\gamma t} \quad \text{where } E_0 = \frac{1}{2}mA_0^2\omega_0^2 \text{ is the initial energy at } t = 0$$

We can also define the time constant  $\tau$  as the time it takes for the energy to decrease to  $\frac{1}{e}$  of its initial value:

$$\tau = \frac{1}{\gamma} \tag{1.21}$$

So we have, for very light damping:

$$E(t) = E_0 e^{-\gamma t} = E_0 e^{-\frac{t}{\tau}} \tag{1.22}$$

**Definiton 1.1.4.2** (Rate of Energy Loss). Taking the time derivative of the total mechanical energy E(t):

$$\frac{dE}{dt} = \frac{d}{dt} \left( \frac{1}{2} m \dot{x}^2 + \frac{1}{2} k x^2 \right)$$
$$= (ma + kx) \dot{x}$$

For a undamped harmonic oscillator, ma + kx = 0, so  $\frac{dE}{dt} = 0$ , indicating that the total mechanical energy is conserved and obeys Hooke's law completely. However, for a damped harmonic oscillator,  $ma + kx = -b\dot{x}$ , leading to:

$$\frac{dE}{dt} = -b\dot{x}^2\tag{1.23}$$

**Definition 1.1.4.3** (Quality Factor (Q-Factor)). The quality factor Q is a dimensionless parameter that characterizes the damping of a harmonic oscillator. It is defined as:

$$Q = \frac{\omega}{\gamma} = \omega \tau \tag{1.24a}$$

And for very light damping, we can approximate  $\omega \approx \omega_0$ , leading to:

$$Q \approx \frac{\omega_0}{\gamma} = \omega_0 \tau \tag{1.24b}$$

This allows us to rewrite the equation of a damped harmonic oscillator as:

$$\ddot{x} + \frac{\omega_0}{Q}\dot{x} + \omega_0^2 x = 0 \tag{1.25}$$

and:

$$\omega = \omega_0 \sqrt{1 - \frac{1}{4Q^2}} \tag{1.26}$$

We can also consider the ratio between the energy at one time and the energy one period later:

$$\frac{E(t+T)}{E(t)} = \frac{E_0 e^{-\gamma(t+T)}}{E_0 e^{-\gamma t}}$$
$$= e^{-\gamma T} \approx 1 - \gamma T$$
$$\frac{E(t+T) - E(t)}{E(t)} \approx -\gamma T = -\frac{2\pi}{\omega_0} \gamma = -\frac{2\pi}{Q}$$

**Example 1.1.4.4.** What is the number of radians through which the damped system oscillates as its energy decreases to 1/e of its initial value?

We have

$$\frac{E}{E_0} = e^{-\gamma t} = \frac{1}{e} \implies \gamma t = 1 \implies t = \frac{1}{\gamma} = \tau$$

So the number of radians is

$$\theta = \omega \tau = \frac{\omega}{\gamma} = Q$$

#### 1.1.5 Undamped Forced Oscillations

We examine steady-state response to sinusoidal driving without losses and identify resonance behavior.

**Definition 1.1.5.1** (Undamped Forced Oscillations). Consider a driver force acting on the mass-spring system; the ODE becomes:

$$m\ddot{x} + kx = \eta \tag{1.27}$$

where  $\eta = F_0 \cos(\omega t)$  is the driving force with amplitude  $F_0$  and angular frequency  $\omega$ . The steady-state particular solution is:

$$x(t) = A(\omega)\cos(\omega t - \delta) \tag{1.28}$$

To derive  $A(\omega)$  and  $\delta$ , use the equation of motion:

$$A(\omega)(-\omega^2 + \omega_0^2)\cos(\delta) = \omega_0^2 a$$

where  $a = \frac{F_0}{k}$  is the static displacement of the mass when the driving force is constant. We also have:

$$A(\omega)(-\omega^2 + \omega_0^2)\sin(\delta) = 0$$

Now, consider the case when  $\delta = 0$ , we have:

$$A(\omega) = \frac{\omega_0^2 a}{\omega_0^2 - \omega^2} \tag{1.29}$$

and the case where  $\delta = \pi$ , we have:

$$A(\omega) = -\frac{\omega_0^2 a}{\omega_0^2 - \omega^2} \tag{1.30}$$

Observe, when  $\omega \approx \omega_0$ , the amplitude  $A(\omega)$  becomes very large, indicating resonance. At resonance, the system oscillates with maximum amplitude, which can lead to significant energy transfer from the driving force to the oscillator.

#### 1.1.6 Damped Forced Oscillations

We add damping to the driven case, derive the frequency response amplitude and phase, and study bandwidth and Q.

**Definition 1.1.6.1** (Damped Forced Oscillations). Consider a damped harmonic oscillator subjected to an external driving force. It is described by the following differential equation:

$$m\ddot{x} + b\dot{x} + kx = F_0 \cos(\omega t) \tag{1.31}$$

where  $F_0$  is the amplitude of the driving force,  $\omega$  is the angular frequency of the driving force, b is the damping coefficient, m is the mass, and k is the spring constant. The general solution to this non-homogeneous differential equation is given by:

$$x(t) = A(\omega)\cos(\omega t - \delta) \tag{1.32}$$

and the amplitude  $A(\omega)$  is given by:

$$A(\omega) = \frac{\omega_0^2 a}{\sqrt{\omega^2 \gamma^2 + (\omega^2 - \omega_0^2)^2}}$$
 (1.33)

We can derive the following 4 cases:

- 1. No Damping  $(\gamma = 0)$ : In this case, the amplitude  $A(\omega)$  simplifies to the undamped case we discussed earlier.
- 2. Low-Frequency Limit ( $\omega \ll \omega_0$ ): In this limit, the amplitude  $A(\omega)$  approaches the static displacement  $a = \frac{F_0}{k}$ . This means that at very low frequencies, the system behaves like a static spring, and the mass is displaced by an amount proportional to the applied force.
- 3. **High-Frequency Limit** ( $\omega \gg \omega_0$ ): In this limit, the amplitude  $A(\omega)$  decreases with increasing frequency, following the relation  $A(\omega) \approx \frac{\omega_0^2 a}{\omega^2}$ . This indicates that at very high frequencies, the mass cannot respond quickly enough to the rapidly oscillating driving force, resulting in a smaller amplitude of oscillation.

4. **Resonance** ( $\omega \approx \omega_0$ ): At resonance, the amplitude  $A(\omega)$  reaches its maximum value but it does not become infinite due to the presence of damping:

$$A_{\text{max}} = \frac{\omega_0^2 a}{\omega_0 \gamma} = \frac{\omega_0 a}{\gamma} = Qa \tag{1.34}$$

We also have the phase shift  $\delta$  given by:

$$\tan(\delta) = \frac{\omega \gamma}{\omega_0^2 - \omega^2} \tag{1.35}$$

The phase shift  $\delta$  indicates how much the oscillation of the mass lags behind the driving force. The behavior of  $\delta$  can be summarized as follows:

- At low frequencies ( $\omega \ll \omega_0$ ),  $\delta$  approaches 0, meaning the mass oscillates in phase with the driving force.
- At resonance  $(\omega = \omega_0)$ ,  $\delta = \frac{\pi}{2}$ , indicating that the mass oscillates a quarter cycle behind the driving force.
- At high frequencies ( $\omega \gg \omega_0$ ),  $\delta$  approaches  $\pi$ , meaning the mass oscillates out of phase with the driving force.

**Definition 1.1.6.2** (Power absorbed during forced oscillations). We can calculate the velocity of the oscillator:

$$\dot{x}(t) = -\omega A(\omega)\sin(\omega t - \delta) = -v_0\sin(\omega t - \delta) \tag{1.36}$$

where  $v_0 = \omega A(\omega)$  is the maximum speed of the oscillator. Energy is lost at the following rate:

$$P(t) = bv(t)^2 (1.37)$$

Substituteing the expression for v(t) into the power equation gives:

$$P(t) = bv_0^2 \sin^2(\omega t - \delta)$$

The average power  $\langle P \rangle$  over one complete cycle of the driving force is given by:

$$\bar{P}(\omega) = \frac{bv_0^2}{2} = \frac{\omega^2 F_0^2 \gamma}{2m \left[ (\omega^2 - \omega_0^2)^2 + \omega^2 \gamma^2 \right]}$$
(1.38)

If the driving frequency  $\omega$  is close to  $\omega_0$ , use  $\omega^2 - \omega_0^2 \approx 2\omega_0\Delta\omega$  with  $\Delta\omega = \omega - \omega_0$  to get the Lorentzian form:

$$\bar{P}(\omega) \approx \frac{\omega_0 F_0^2}{2m\gamma \left[1 + \left(\frac{2\Delta\omega}{\gamma}\right)^2\right]}$$
 (1.39)

The maximum average power occurs at resonance:

$$\bar{P}_{\text{max}} = \frac{\omega_0 F_0^2}{2m\gamma} \tag{1.40}$$

Average Power vs Driving Frequency for Different Quality Factors, Power normalized to  $P_{\rm max}=1$ 

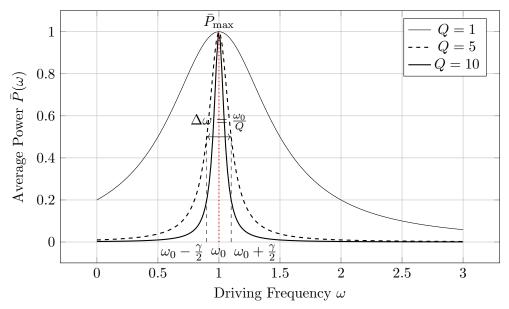


Figure 1.1: Average Power vs Driving Frequency for Different Quality Factors

Full Width Half Height The graph of  $\bar{P}(\omega)$  versus  $\omega$  has a peak at  $\omega = \omega_0$  with a maximum value of  $\bar{P}_{\text{max}}$ . The full width at half maximum (FWHM) is the width of the peak at half of its maximum height. The FWHM is given by:

$$\omega_{\text{FWHM}} = 2\Delta\omega = 2\gamma = \frac{2\omega_0}{Q} \tag{1.41}$$

To illustrate the graph, we can plot  $\bar{P}(\omega)$  versus  $\omega$  for different values of the quality factor Q:

#### 1.1.7 Simple Pendulum

Consider the arc length s of a pendulum bob, we have:

$$s = l\theta \quad m\ddot{s} = -mg\sin(\theta) \tag{1.42}$$

For small angles, we can approximate  $\sin(\theta) \approx \theta$ , leading to:

$$\ddot{\theta} + \frac{g}{l}\theta = 0 \tag{1.43}$$

And the ODE has solution:

$$\theta(t) = \theta_0 \cos(\omega t + \phi) \tag{1.44}$$

**Energy** For small angle  $\theta$ , the total mechanical energy E of the pendulum is given by:

$$E = K + U = \frac{1}{2}ml^2\dot{\theta}^2 + mgl(1 - \cos(\theta)) \approx \frac{1}{2}ml^2\dot{\theta}^2 + \frac{1}{2}mgl\theta^2$$
 (1.45)

This shows that the pendulum behaves like a simple harmonic oscillator with an effective spring constant  $k = \frac{mg}{l}$ .

**The physical pendulum** For a rigid body swinging about a pivot point, the equation of motion is given by:

$$I\ddot{\theta} = \tau \tag{1.46}$$

For simple rod of length L and mass m pivoted at one end, the moment of inertia about the pivot point is  $I = \frac{1}{3}mL^2$ . The torque due to gravity when the rod is displaced by an angle  $\theta$  from the vertical is  $\tau = -mg\frac{L}{2}\sin(\theta)$ . For small angles, we can approximate  $\sin(\theta) \approx \theta$ , leading to:

$$\ddot{\theta} + \frac{3g}{2L}\theta = 0 \tag{1.47}$$

The angular frequency of oscillation for the physical pendulum is:

$$\omega = \sqrt{\frac{3g}{2L}} \tag{1.48}$$

The LC Circuit An LC circuit consists of an inductor L and a capacitor C connected in series. The charge q(t) on the capacitor satisfies the differential equation:

$$L\frac{d^2q}{dt^2} + \frac{1}{C}q = 0 (1.49)$$

This is analogous to the equation of motion for a simple harmonic oscillator, with the angular frequency given by:

$$\omega = \frac{1}{\sqrt{LC}} \tag{1.50}$$

The RLC Circuit An RLC circuit consists of a resistor R, inductor L, and capacitor C connected in series. The charge q(t) on the capacitor satisfies the differential equation:

$$L\frac{d^2q}{dt^2} + R\frac{dq}{dt} + \frac{1}{C}q = 0 (1.51)$$

This is analogous to the resistor behaving as a damped harmonic oscillator, with the angular frequency given by:

$$\omega = \sqrt{\frac{1}{LC} - \left(\frac{R}{2L}\right)^2} \tag{1.52}$$

### 1.2 The Wave Equation

**Definition 1.2.0.1** (The Wave Equation). The wave equation is a second-order linear partial differential equation that describes the propagation of waves (sound, EM, water). In one dimension, it is:

$$\frac{\partial^2 u}{\partial t^2} - c^2 \frac{\partial^2 u}{\partial r^2} = 0 \tag{1.53}$$

where u(x,t) is the disturbance, c is the wave speed, x is position, and t is time.

## Chapter 2

# **Modern Physics**