

Gear Drives

- Gears are defined as toothed wheels which transmit power & motion from one shaft to another by means of successive engagement of teeth
- Advantages compared to chain/belt drives:
 - i) It is positive drive & velocity ratio remains constant
 - ii) Centre distance b/w shafts is small, leads to compact construction
 - iii) Transmits very large power beyond range of chain/belt drives
 - iv) Can transmit even at low velocity, not possible in chain drives
 - v) Provision can be made in gearbox for gear shifting, changing velocity ratio over wide range.

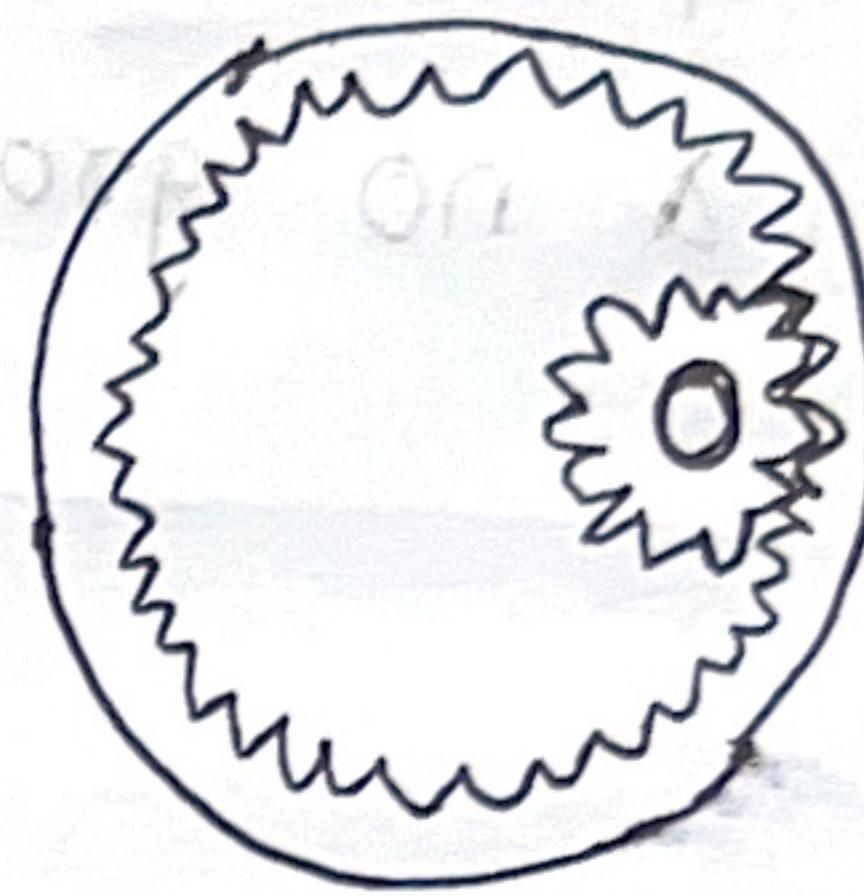
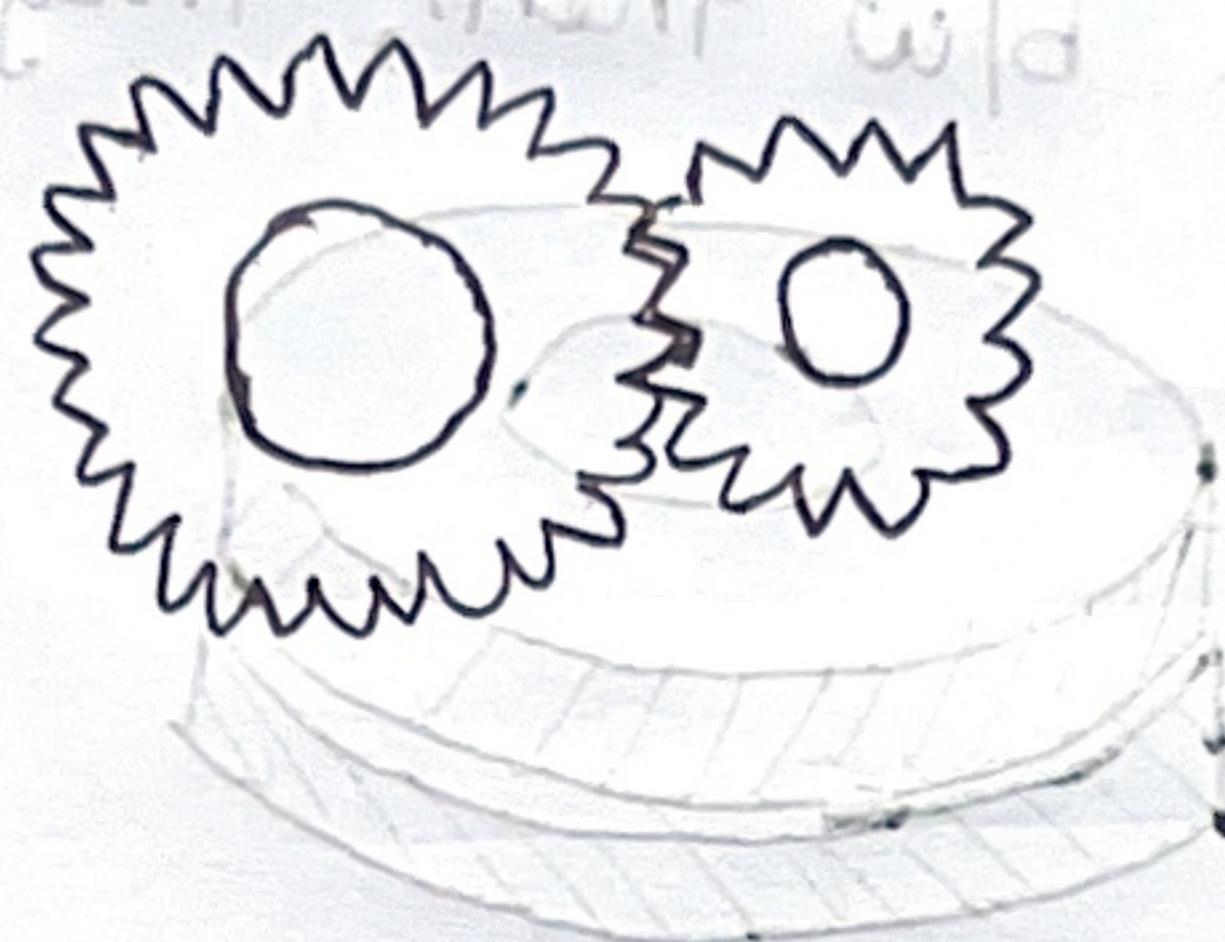
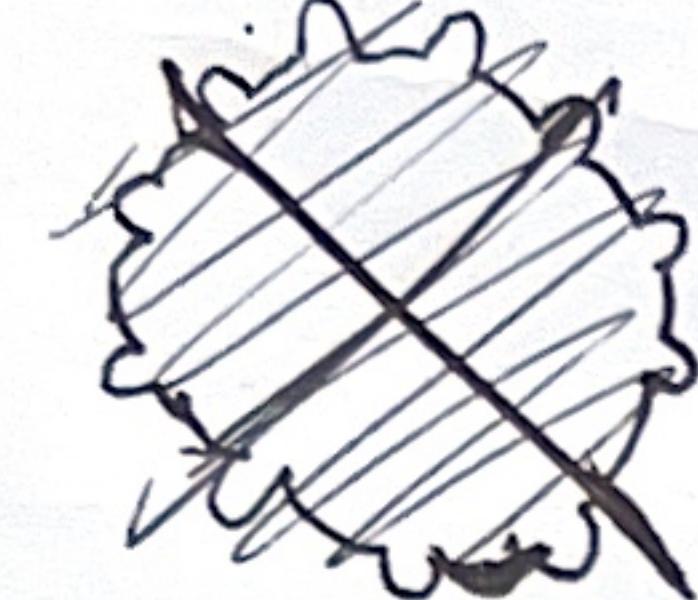
Gear Drives are **3** types:

- 1) Parallel shafts
 - i) Spur gears
 - ii) Helical gears
 - iii) Double helical / Herringbone Gears
- 2) Intersecting shafts
 - i) Straight Bevel Gears
 - ii) Spiral Bevel Gears
- 3) Skew shafts
 - i) Hypoid Gears
 - ii) Worm Gears

Spur Gears

- They have straight teeth \parallel to axes
 - At time of engagement of 2 gears, contact extends across entire width on a line \parallel to axes of rotation.
- Results in sudden application of load, high impact stress & excessive noise at high speeds

If gear has external teeth \Rightarrow shaft rotates in opposite direction

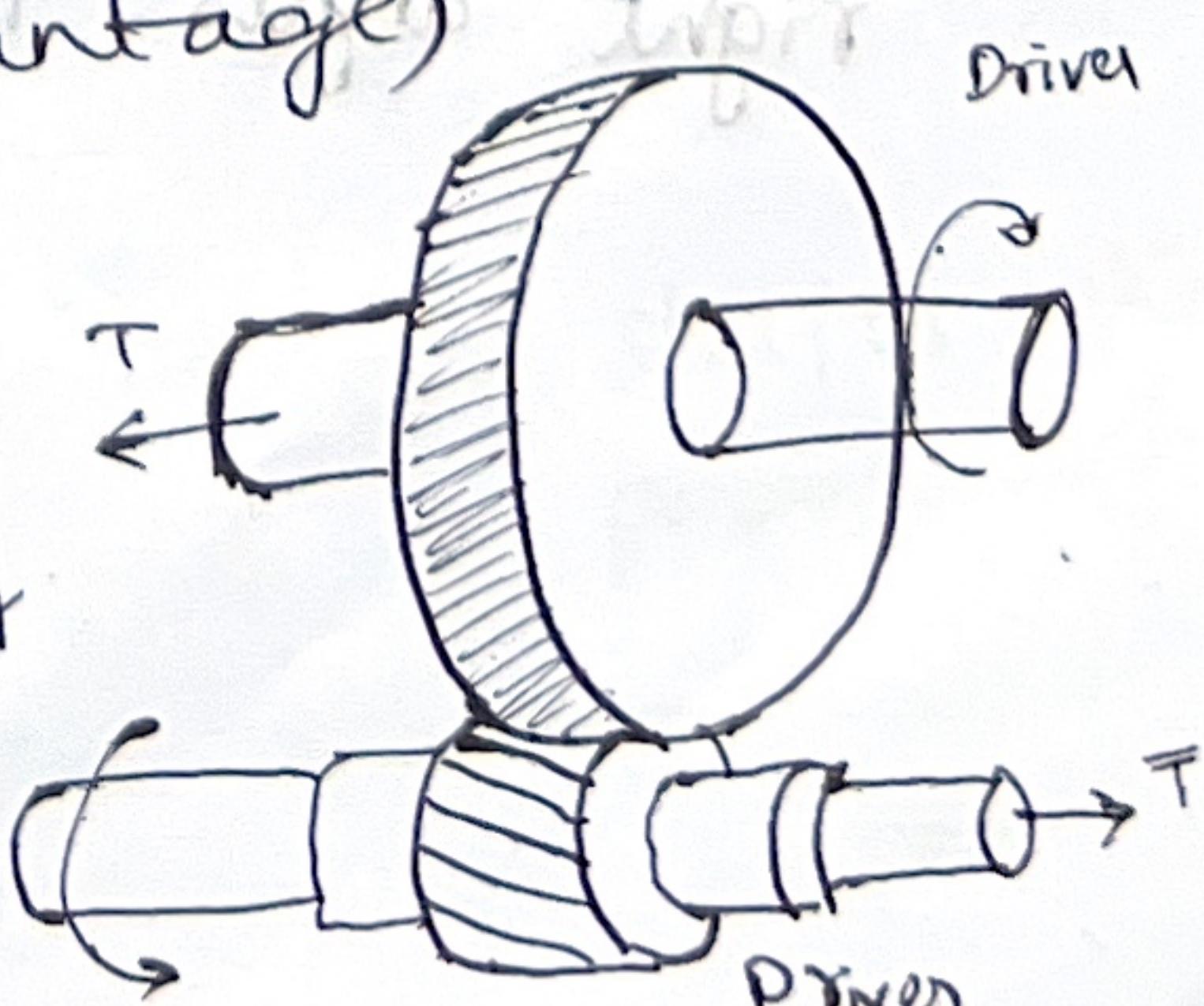


Helical Gears

- Teeth are curved, each being in helical shape
- 2 Mating gears have same helix angle but have teeth of opposite hands.
- At beginning of engagement, contact occurs at point of leading edge of curved path, but extends along diagonal line, as gears rotate.
- Low impact stress, so, noise reduced.

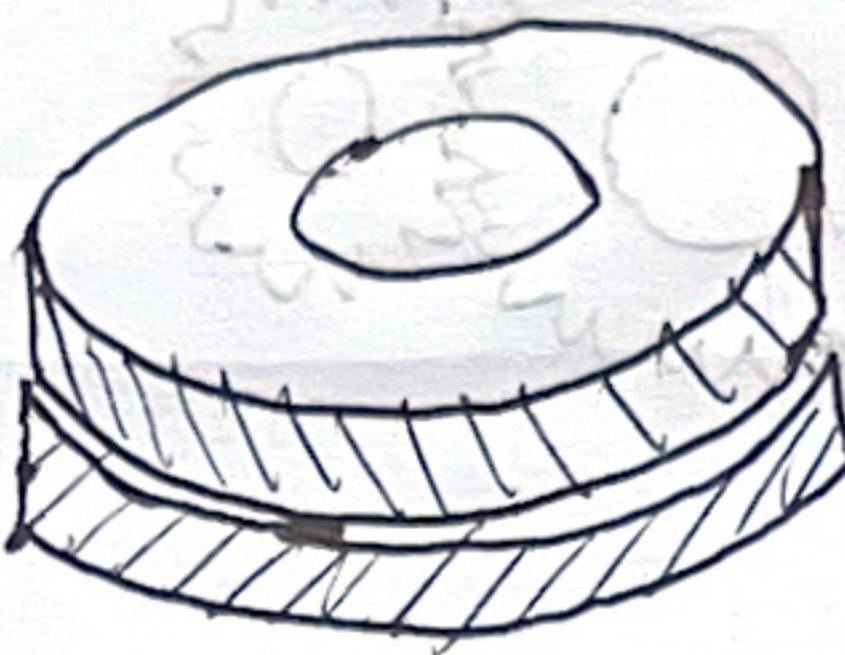
They can be used at higher velocities for greater load carrying cap.

- They have end thrust cuz of a force component along gear axis (disadvantage)



Double-Helical Gears

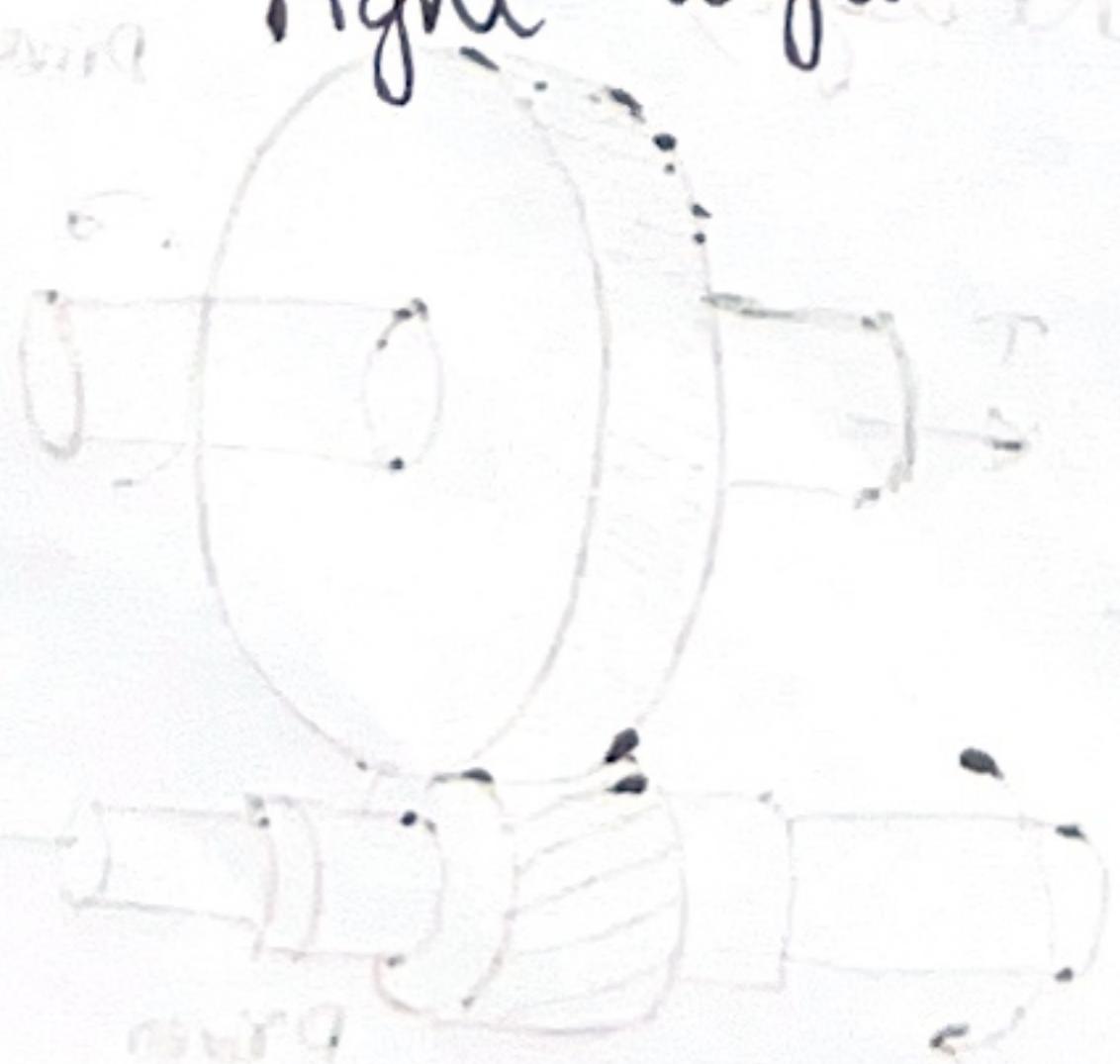
- Equivalent to a pair of helical gears secured together one right hand helix, other left hand helix.
- Both the teeths are separated by a groove.
No axial thrust as one cancels the other
can be run at high speeds & less noise & vibration
- If left & right gears meet at common apex,
& no groove b/w them, they are called herringbone gear.



Straight Bevel Gears

- Teeth are straight, radial to point of intersection of shaft axes & vary in cross-section through out the length.
- Usually used to connect shafts at right angles which run at lower speeds.

Gears of same size & connecting 2 shafts at right angles to each other are called mitre gears



Spiral Bevel Gears

- Teeth of bevel gears are inclined at an angle to face of bevel, they are called spiral/helical bevels.
- Smoother in action & quieter because of gradual load application & low impact stress.
- But axial thrust exist, calling for stronger bearing & supporting assemblies
- used to drive the differential of an automobile.

Hypoid Gears

- They are a type of spiral bevel gears but they have axes that are non-intersecting / parallel
- They are hyperbolic rather than regular conical geometry
- Mostly used in automotive industry

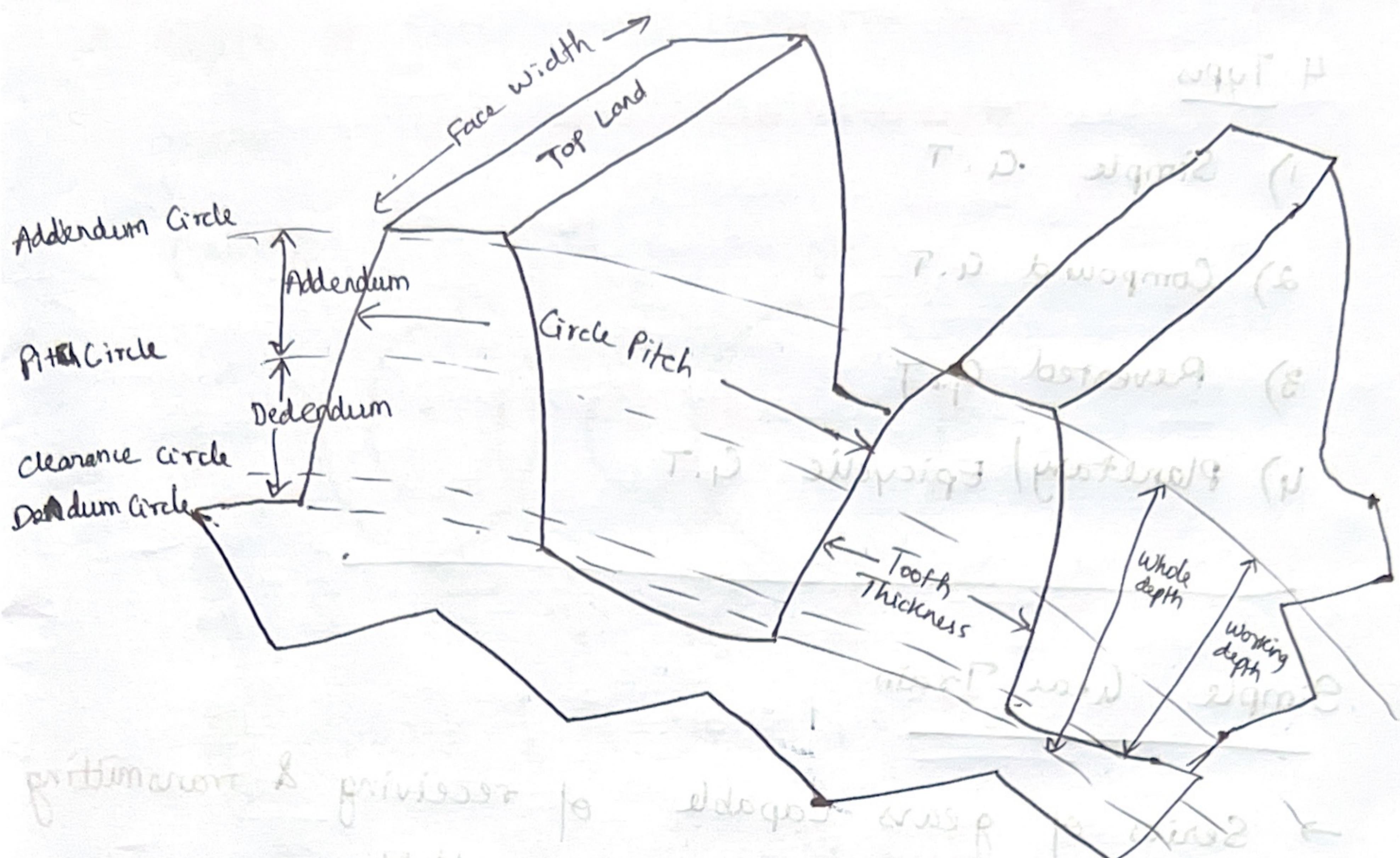
Worm Gears

- consist of a worm & worm wheel
- Worm is a form of threaded screw which meshes with matching wheel
- Worm gear drives are used for shafts which have axes that aren't parallel or perpendicular
- They are characterized by high speed reduction ratio

Spur Rack & Pinion

- It is a special case of spur gears with ∞ diameter
- It converts rotary motion to translatory motion or vice versa
- ex: used in lathe which rack transmits motion to saddle

Gear Terminology



$$\text{Circular pitch} \Rightarrow P = \frac{\pi d}{T}$$

d: pitch diameter

T: no. of teeth

$$\text{Diametral pitch} \Rightarrow P = \frac{T}{d}$$

$$\text{Module} \Rightarrow m = \frac{d}{T}$$

$$\text{Velocity ratio} \Rightarrow VR = \frac{\omega_2}{\omega_1} = \frac{N_2}{N_1} = \frac{d_1}{d_2} = \frac{T_1}{T_2}$$

ω : angular velocity (rad/s)

N: $\omega T = \text{N}$ (rpm)

T: no. of teeth

d: pitch diameter

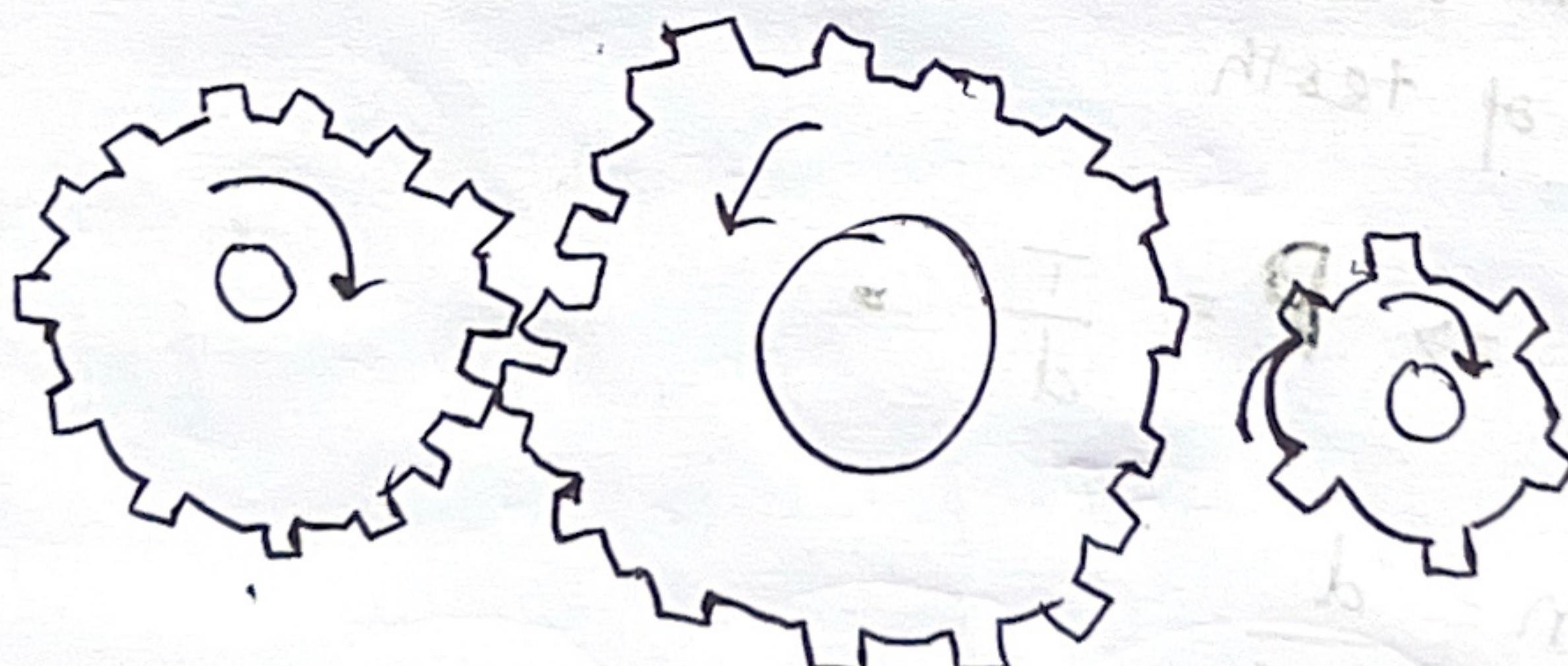
Gear Trains

4 Types

- 1) Simple G.T
- 2) Compound G.T
- 3) Reverted G.T.
- 4) Planetary / Epicyclic G.T

Simple Gear Train

- Series of gears capable of receiving & transmitting motion from one gear to another
- 1 shaft, 1 gear



$$\frac{N_2}{N_1} = \frac{T_1}{T_2}$$

$$\frac{N_3}{N_2} = \frac{T_2}{T_3}$$

$$\Rightarrow \frac{N_2}{N_1} \times \frac{N_3}{N_2} = \frac{T_1}{T_2} \times \frac{T_2}{T_3}$$

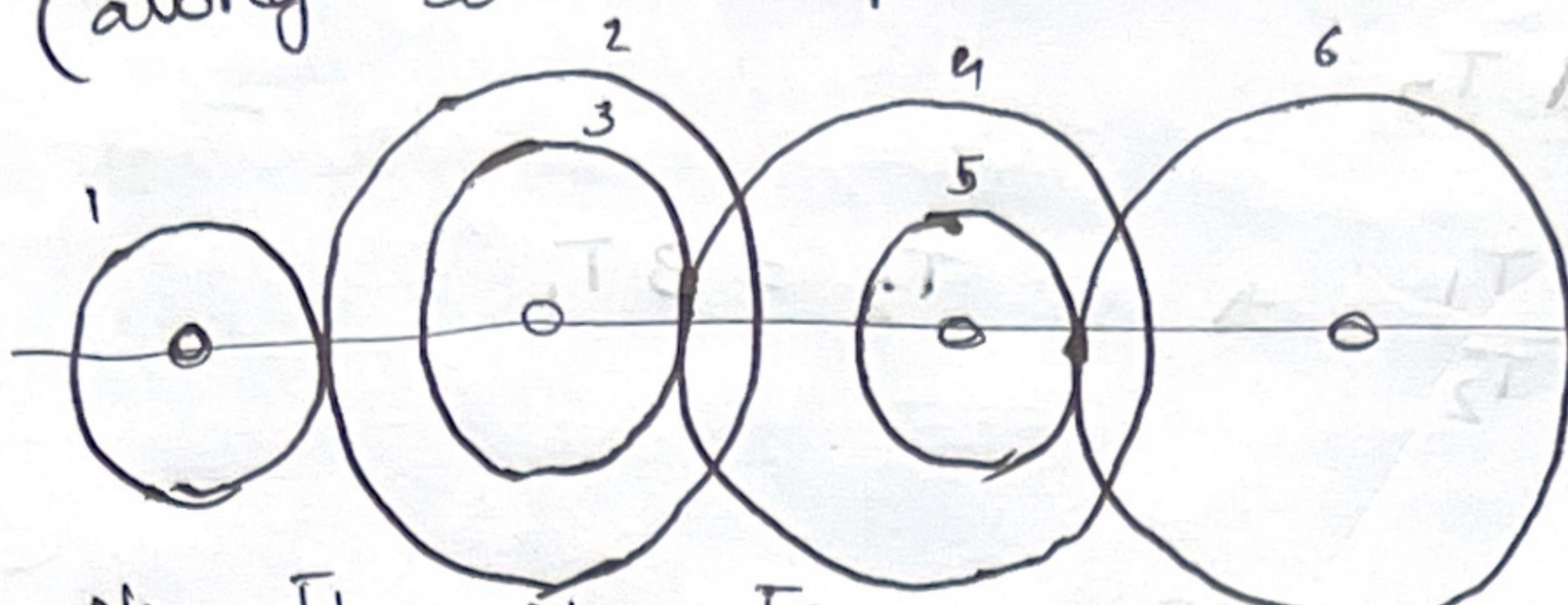
$$\frac{N_3}{N_1} = \frac{T_1}{T_3}$$

Intermediate gears have no effect on speed ratio & called idlers

Compound Gear Train

- Series of gears are connected such that 2 or more gears rotate with same angular velocity

(along same shaft)



$$\frac{N_2}{N_1} = \frac{T_1}{T_2}$$

$$\frac{N_4}{N_3} = \frac{T_3}{T_4}$$

$$\frac{N_6}{N_5} = \frac{T_5}{T_6}$$

$$N_2 = N_3 ; N_4 = N_5 ; N ; N_6$$

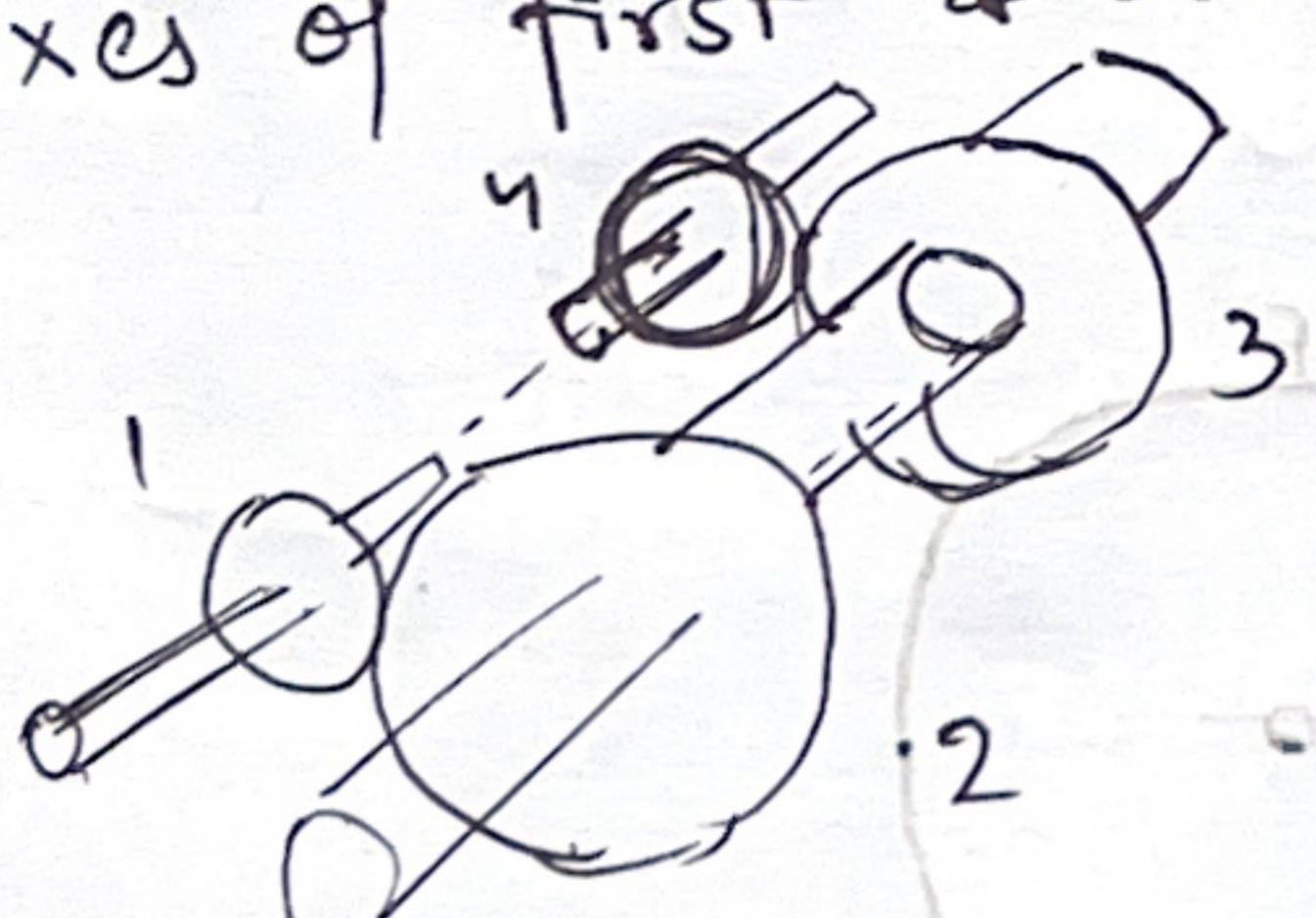
$$\frac{N_2}{N_1} \times \frac{N_4}{N_3} \times \frac{N_6}{N_5} = \frac{T_1}{T_2} \times \frac{T_3}{T_4} \times \frac{T_5}{T_6}$$

$$\frac{N_6}{N_5} = \frac{T_1 \times T_3 \times T_5}{T_2 \times T_4 \times T_6}$$

Reverted Gear Train

→ Axes of first & last wheels of compound gear coincide

- Axes of first & last wheels of compound gear coincide



→ used in clocks & simple lathes where back gear is used to give slow speed to chuck

Planetary / Epicyclic Gear Train

→ Gear Train having relative motion of axes

- Axis of at least 1 gear moves relative

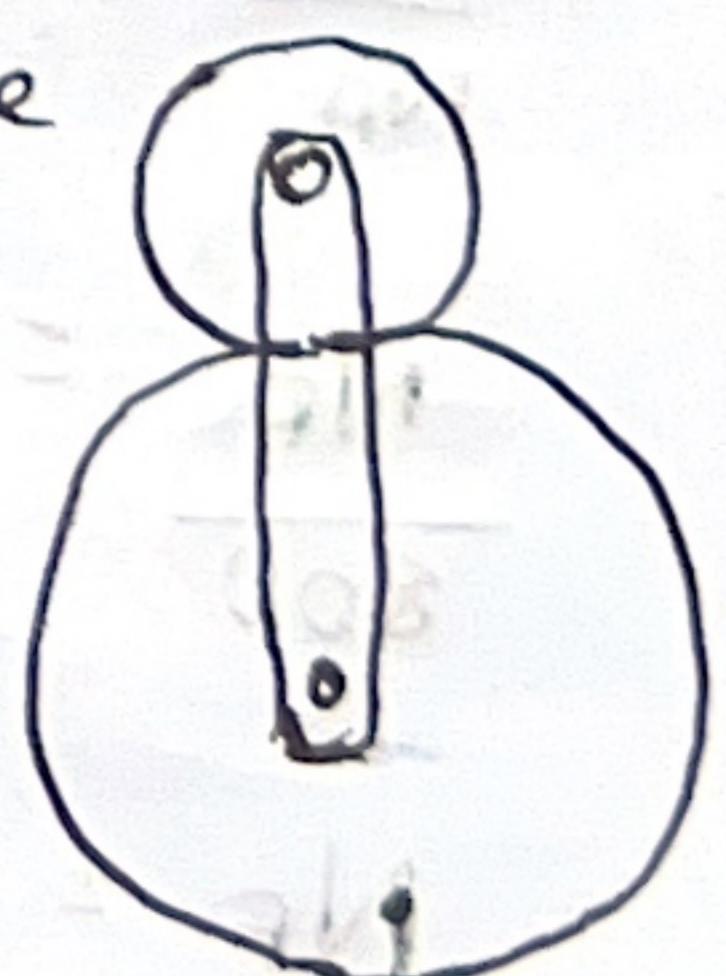
to frame

→ Usually wheel that rolls outside is called epicyclic

→ Large speed reductions are possible

→ Even more compact unit is possible

→ Used in transmission, computing devices etc.,



Q1. The following data relate to 2 meshing gears -

Velocity ratio - $1/3$

Module - 4 mm

(Centre distance - 200 mm)

Find T_1 & T_2

$$A. \frac{N_2}{N_1} = \frac{1}{3} = \frac{T_1}{T_2} \Rightarrow T_2 = 3T_1$$

$$C = \frac{d_1 + d_2}{2} = 200$$

$$= m(T_1 + T_2) = 200 \Rightarrow m(4T_1) = 400$$

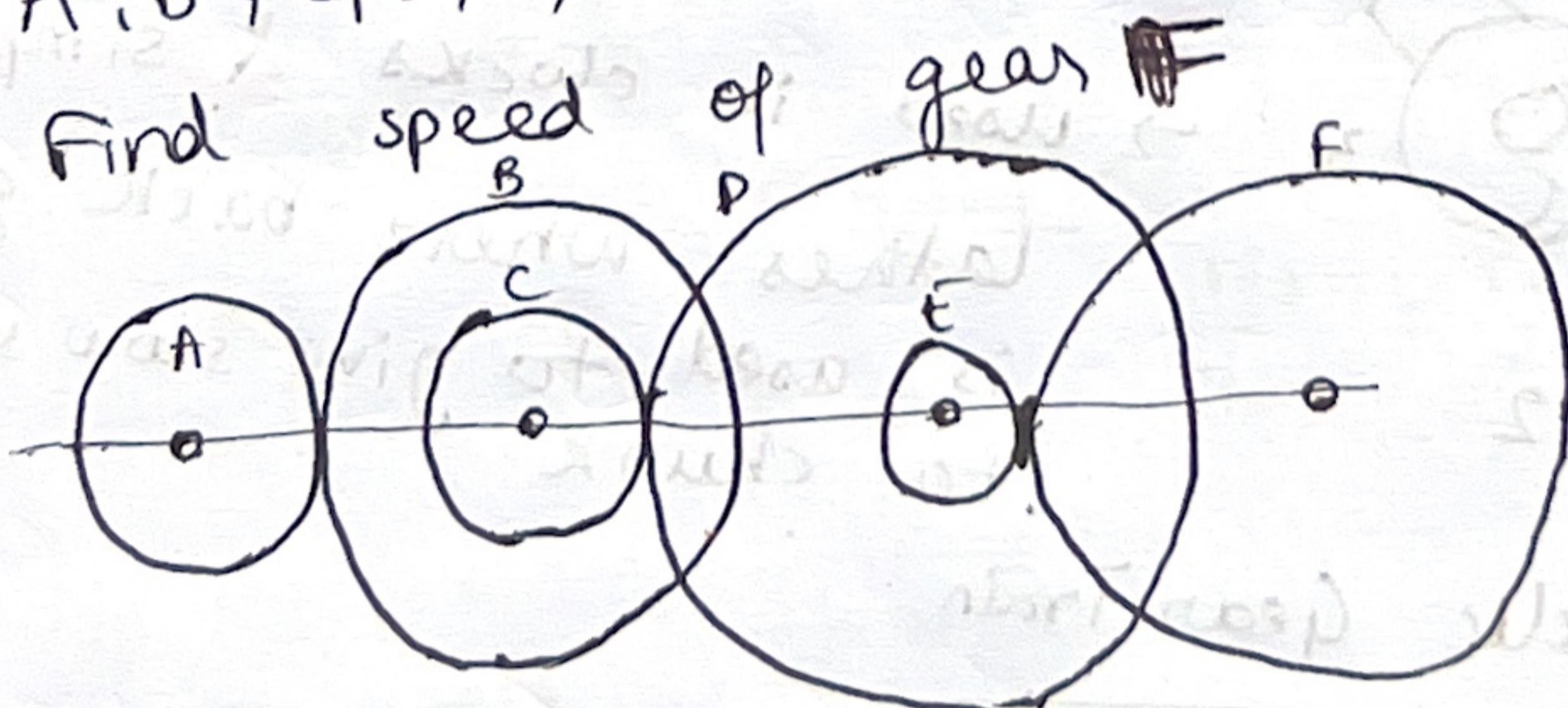
$$T_1 = \frac{400}{16} = 25$$

$$T_2 = 75$$

Q2. Compound Gear Train. Motor shaft rotating at 800 rpm connected to gear A. No of Teeth on

A, B, C, D, E, F are 24, 56, 30, 80, 32, 72.

Find speed of gear F

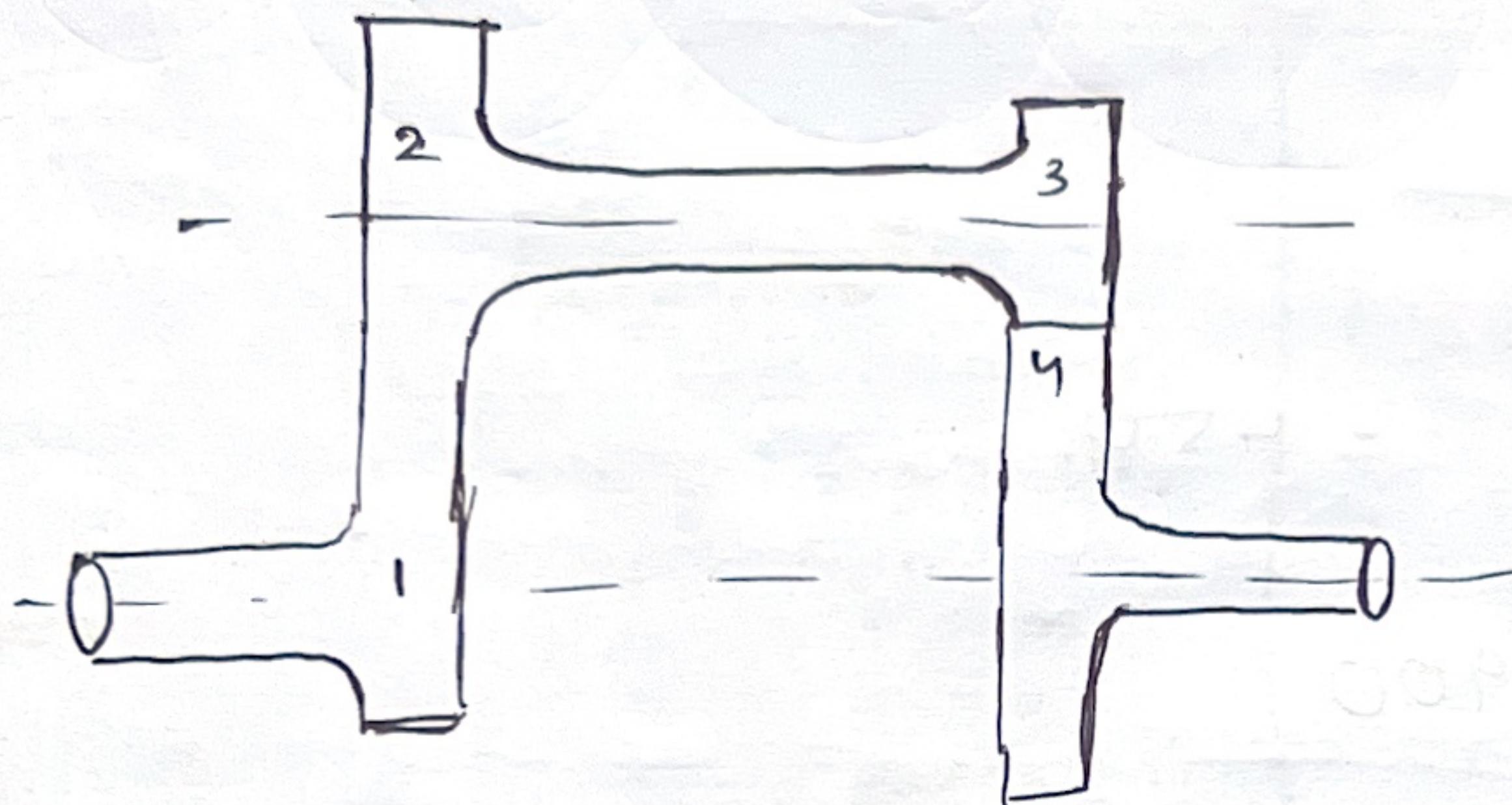


$$A. \frac{N_F}{N_A} = \frac{T_A}{T_B} \times \frac{T_C}{T_D} \times \frac{T_E}{T_F}$$

$$\frac{N_F}{800} = \frac{24}{56} \times \frac{30}{80} \times \frac{32}{72}$$

$$N_F = 57.143 \text{ rpm}$$

Q3. A reverted G.T. shown is used to provide speed ratio of 10. Module of gear 1 & 2 is 3.2mm & 3 & 4 is 2mm. Determine suitable no. of teeth for each gear. No gear has less than 20 teeth. Centre distance b/w shafts is 160 mm.



A.

$$\frac{N_1}{N_2} = 2.5 = \frac{T_2}{T_1}$$

$$\frac{N_3}{N_4} = 4 = \frac{T_4}{T_3}$$

$$r_1 + r_2 = 160 \Rightarrow \frac{m_1 T_1}{2} + \frac{m_2 T_2}{2} = 160$$

$$r_3 + r_4 = 160 \Rightarrow \frac{m_3 T_3}{2} + \frac{m_4 T_4}{2} = 160$$

$$3.2(T_1 + T_2) = \frac{100}{320}$$

$$T_1 + 2.5 T_1 = 100$$

$$3.5 T_1 = 100$$

$$T_1 = 28$$

$$T_2 = 72$$

$$2(T_3 + T_4) = 320$$

$$2(T_3 + 4T_3) = 320$$

$$T_3 = \frac{160}{5} = 32$$

$$T_4 = 128$$

$$\Rightarrow V.R = \frac{T_2}{T_1} \times \frac{T_4}{T_3} = \frac{72}{28} \times \frac{128}{32} = 10.29$$

Q. 2 Spur gears A & B connected to two 11th shafts are 450 mm apart. Gear A runs at double speed of gear B. Gear B runs at 150 rpm A.C.W. If circular pitch is given to be 20, calculate no. of teeth on gears A & B

$$A. P = \frac{\pi d}{T}$$

$$C = \frac{d_1 + d_2}{2} = 450$$

$$d_1 + d_2 = 900$$

$$20 = \frac{\pi \times d_1}{T_1} = \frac{\pi d_2}{T_2}$$

$$\frac{20T_1}{\pi} = d_1$$

$$d_2 = \frac{20T_2}{\pi}$$

$$\frac{N_1}{N_2} = 2 \Rightarrow N_1 = 300$$

$$\frac{T_2}{T_1} = 2 \Rightarrow T_2 = 2T_1$$

$$\frac{20}{\pi} (T_1 + T_2) = 900$$

$$\frac{20}{\pi} (T_1 + 2T_1) = 900$$

$$\frac{60}{\pi} T_1 = 900$$

$$T_1 = 47$$

$$T_2 = 94$$