

# Advanced Robot Control and Learning

Prof. Sami Haddadin

# Modern Methods of Robot Control II

# Advanced Robot Control - Outline

Differential Geometry in Robotics

Basics of Task Space Modeling and Control

Modern Methods of Robot Control I

## **Modern Methods of Robot Control II**

- Impedance and Admittance Control
- Port-Hamiltonian Systems in Robotics
  - Motivation
  - Dirac Structure and Power Ports
  - Port-based Modeling of a Manipulator
  - Passivity Analysis

Linear Parametrization of Robot Dynamics

Robot Dynamics Identification

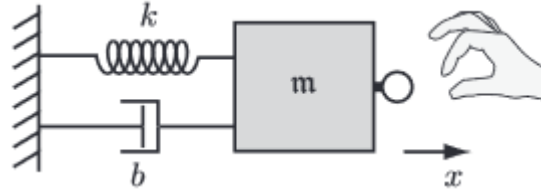
Bio-Inspired Robot Control

# References

- The content of this lecture is based on
  - Passivity-based Control and Estimation in Networked Robotics, *T. Hatanaka et al.*, Springer international publishing, 2015
  - Slides of the lecture Sensor-based Robotic Manipulation and Locomotion, *Prof. Dr. Alin Albu-Schäffer*, Technical University of Munich, Wintersemester 2018/19
  - M. W. Spong, S. Hutchinson, M. Vydiasagar: Robot Modelling and Control, John Wiley & Sons, 2006
  - Takegaki M., Arimoto S., 1981, “*A new feedback method for dynamic control of manipulators*”, Transactions ASME, Journal of Dynamic Systems, Measurement and Control, Vol. 103, pp. 119–125
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  - Slotine, J. J. E., & Li, W. (1991). *Applied nonlinear control* (Vol. 199, No. 1). Englewood Cliffs, NJ: Prentice hall.
  - Lynch, Kevin M., and Frank C. Park. *Modern Robotics*. Cambridge University Press, 2017.

# Impedance/Admittance Control

# Impedance & Admittance – Concept



- The robot end-effector is asked to render particular mass, spring, and damper properties.
- The dynamics for a one-dof robot rendering an impedance can be written as:

$$m\ddot{x} + b\dot{x} + kx = f$$

- Taking the Laplace transform yields:

$$(ms^2 + bs + k)X(s) = F(s)$$

- The impedance is defined by the transfer function from position perturbations to forces:  $Z(y) = \frac{F(s)}{X(s)}$
- The admittance is the inverse of the impedance:  $Y(s) = Z^{-1}(s) = \frac{X(s)}{F(s)}$

# Impedance & Admittance – Concept

- From electrical circuits:

$$U = RI + Li + \frac{1}{C} \int I \, dt$$

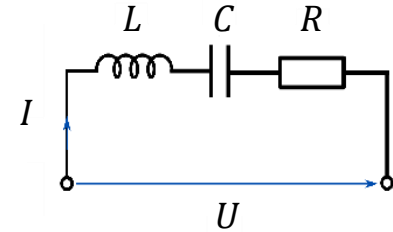
- Impedance:**  $Z(s) = \frac{U(s)}{I(s)}$  e.g.  $Z(s) = R + Ls + \frac{1}{Cs}$
- Admittance:**  $Y(s) = \frac{I(s)}{U(s)}$  e.g.  $Y(s) = \frac{1}{R + Ls + \frac{1}{Cs}}$

- Analogy between mechanical and electrical systems:

$$U = L\ddot{Q} + R\dot{Q} + \frac{1}{C}Q$$

$$F = M\ddot{X} + D\dot{X} + KX$$

- Impedance:**  $Z(s) = \frac{F(s)}{X(s)}$  Impedance causality: Input:  $X(s)$ , Output:  $F(s)$
- Admittance:**  $Y(s) = \frac{X(s)}{F(s)}$  Admittance causality: Input:  $F(s)$ , Output:  $X(s)$



$$U \leftrightarrow F, \quad I \leftrightarrow \dot{X}, \quad Q \leftrightarrow X,$$

$$R \leftrightarrow D, \quad L \leftrightarrow M, \quad \left(\frac{1}{C}\right) \leftrightarrow K$$

# Impedance-Control Algorithm

- The goal of impedance control is to implement the task-space behavior:  $M\ddot{x} + B\dot{x} + Kx = f_{ext}$
- The robot senses the endpoint motion  $x(t)$  and commands joint torques and forces to create  $-f_{ext}$ , the force to display to the user.
- Theoretically, an impedance-controlled robot should only be coupled to an admittance-type environment
- A good control law might be

$$\tau = J^T(\theta) \left( \underbrace{\tilde{\Lambda}(\theta)\ddot{x} + \tilde{\eta}(\theta, \dot{x})}_{\text{arm dynamics compensation}} - \underbrace{(M\ddot{x} + B\dot{x} + kx)}_{f_{ext}} \right)$$

- The task-space dynamics model  $\{\tilde{\Lambda}, \tilde{\eta}\}$  is expressed in terms of the coordinates



# Admittance -Control Algorithm

- The robot senses  $f_{ext}$  using a wrist force–torque sensor and controls its motions in response.
- In an admittance-control algorithm the force  $f_{ext}$  applied by the user is sensed by the wrist load cell, and the robot responds with an end-effector acceleration.
- A simple approach is to calculate the desired end-effector acceleration  $\ddot{x}_d$  according to:

$$M\ddot{x}_d + B\dot{x} + Kx = f_{ext}$$

- Solving with respect to  $\ddot{x}_d$ , we get:

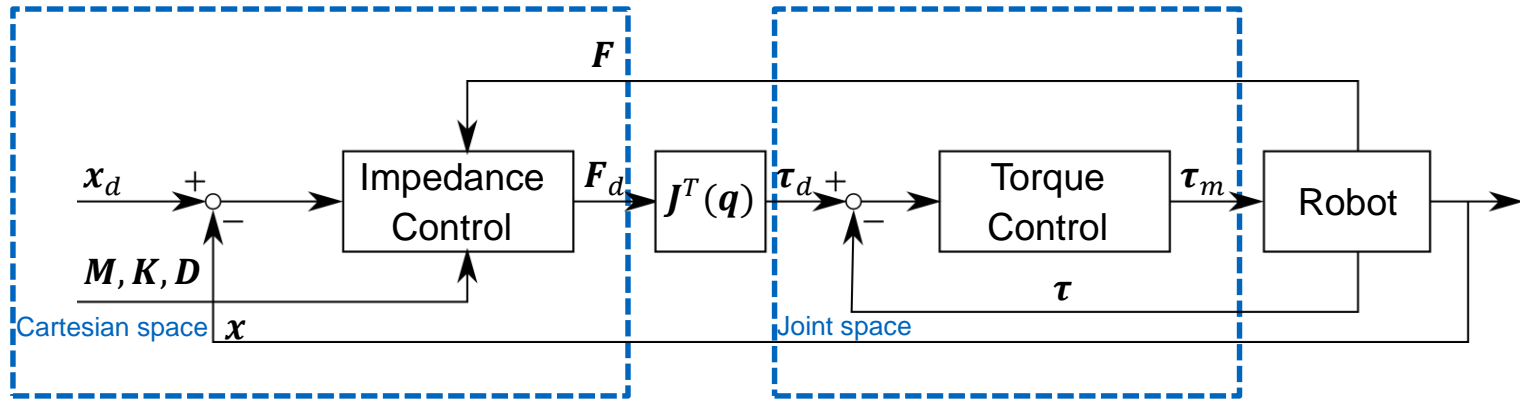
$$\ddot{x}_d = M^{-1}(f_{ext} - B\dot{x} - Kx)$$

- Considering  $\dot{x} = J(\theta)\dot{\theta}$ , the desired joint accelerations can be solved as:

$$\ddot{\theta}_d = J^{-1}(\theta)(\ddot{x}_d - \dot{J}(\theta)\dot{\theta})$$

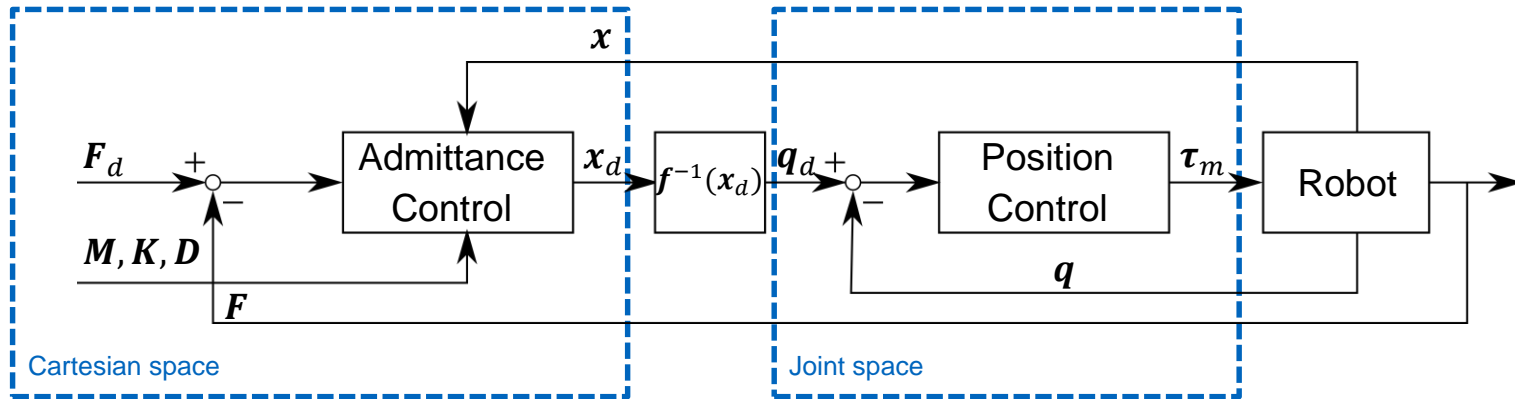
- Then, the inverse dynamics can be used to calculate the commanded joint forces and torques.

# Cartesian Controller with Impedance Causality



- Inner loop: compliance
- Outer loop: high stiffness

# Cartesian Controller with Admittance Causality



- Inner loop: high stiffness
- Outer loop: compliance

# Considerations – Cartesian Space vs. Joint Space

- For the non-redundant, non-singular robot, the same controllers can be applied as in joint space
- Robot model in  $SE(3)$  space

$$\mathbf{M}_C(\mathbf{q})\ddot{\mathbf{x}} + \mathbf{C}_C(\mathbf{q}, \dot{\mathbf{q}})\dot{\mathbf{x}} + \mathbf{F}_g(\mathbf{q}) = \mathbf{F}_{in}$$

- PD controller with gravity compensation

$$\begin{aligned}\mathbf{F}_{in} &= \mathbf{K}_P(\mathbf{x}_d - \mathbf{x}) - \mathbf{K}_D\dot{\mathbf{x}} \\ \boldsymbol{\tau}_{in} &= \mathbf{J}^T(\mathbf{q})\mathbf{F}_{in} + \mathbf{g}(\mathbf{q})\end{aligned}$$

- Closed loop dynamics

$$\Rightarrow \mathbf{M}_C(\mathbf{q})\ddot{\mathbf{x}} + \mathbf{C}_C(\mathbf{q}, \dot{\mathbf{q}})\dot{\mathbf{x}} + \mathbf{K}_P(\mathbf{x}_d - \mathbf{x}) - \mathbf{K}_D\dot{\mathbf{x}} = 0$$

- Differences to joint control:
  - All statements are only *local*, not global
  - In case of redundant robot, convergence of coordinate  $\mathbf{x}$  does not imply stability of the whole system (Null space motion)

# Chapter 5: Port-Hamiltonian Systems in Robotics

# Motivation – Interconnection by Energy

- **Energy**
  - Transferable property of an object
  - Can be described in an equivalent nature in different domains
- Each system consists of subsystems, with their own network of interconnected atomic elements (Ports) that deal with a specific energy type
- Interconnection through exchange of energy per unit time: **Power**
- Power  $P$  as the dual product (i.e.  $\langle \varepsilon | \zeta \rangle = \varepsilon^T \zeta$ ) of *Power-conjugate variable*, namely **effort** and **flow**

# Power-conjugate variables

- Belonging to the dual spaces<sup>1</sup> (e.g.  $\mathcal{F} = \mathbb{R}^k, \mathcal{E} := \mathcal{F}^* = (\mathbb{R}^k)^*$ ), *flow* ( $f \in \mathcal{F}$ ) and *effort* ( $e \in \mathcal{E}$ ) permit a power description in a unified way
- They can be defined for each subsystem according to its related domain
- Examples

Energy Domain	Effort	Flow
Kinetic – translational	Force	Linear velocity
Kinetic – rotational	Torque	Angular velocity
Electrical	Voltage	Current
Magnetic	Current	Voltage
Hydraulic	Pressure	Volume flow
Thermal	Temperature	Entropy flow

<sup>1</sup> See first lecture on Differential Geometry in Robotics, tangent and cotangent spaces

# Dirac-structure and Power-ports



# Dirac Structure

- On the space  $\mathcal{F} \times \mathcal{F}^*$  a Dirac structure is a subspace  $\mathcal{D}$  such that
  - i.  $\langle e|f \rangle = 0, \forall (f, e) \in \mathcal{D}$
  - ii.  $\dim \mathcal{D} = \dim \mathcal{F}$
- First condition  $\Rightarrow$  the power-conservation property of the physical systems
- Second condition  $\Rightarrow$  the impossibility in controlling of both effort and flow<sup>1</sup>
- Belonging to the dual spaces<sup>1</sup> (e.g.  $\mathcal{F} = \mathbb{R}^k, \mathcal{E} := \mathcal{F}^* = (\mathbb{R}^k)^*$ ), *flow* ( $f \in \mathcal{F}$ ) and *effort* ( $e \in \mathcal{E}$ ) permit a power description in a unified way
- They can be defined for each subsystem according to its related domain
- Dirac structure as a power-conserving interconnection of atomic elements (i.e. *ports*)
- The subspace  $\mathcal{D}$  is called the space of *port variables*

<sup>1</sup> See fourth lecture on Modern Methods of Robot Control, Impedance control

# Internal Ports

## Energy Storage Port ( $\mathcal{S}$ ):

- Energy storage is describable by a finite-dimensional state space manifold  $\mathcal{H}$  with coordinates  $x$ , together with the energy function (namely Hamiltonian)

$$H: \mathcal{H} \rightarrow \mathbb{R}$$

- Examples:

Energy Domain	State
Kinetic – translational	Linear momentum ( $P$ )
Kinetic – rotational	Angular momentum ( $L$ )
Potential – translational	Linear displacement ( $x$ )
Potential – rotational	Angular displacement ( $\theta$ )
Electrical	Charge ( $Q$ )
Magnetic	Magnetic flux linkage ( $\lambda$ )

- For the related port variables  $(f_S, e_S)$ , which are interconnected to the energy storage of the system we have

$$P_S = e_S^T f_S = \left( \frac{\partial H}{\partial x} \right)^T (-\dot{x}) = -\dot{H}$$

## Energy Dissipation Port ( $\mathcal{R}$ ):

- Corresponds to internal energy dissipation (due to friction, resistance, etc.)
- Port variables are denoted by  $(f_R, e_R)$
- Static resistive relation  $R(f_R, e_R) = 0$ , with the property of  $\langle \varepsilon | f \rangle \leq 0 \forall (f_R, e_R)$

- Without the presence of additional external ports, Dirac structure of port-Hamiltonian system satisfies power balance

$$e_S^T f_S + e_R^T f_R = 0$$

- Hence

$$\frac{d}{dt} H = -e_S^T f_S = e_R^T f_R \leq 0$$

# External Ports

## Interaction Port ( $\mathcal{I}$ ):

- Corresponding with the interaction of the system with the environment
- Assumed to be known and described by the port variables  $(f_I, e_I)$

## Control Port ( $\mathcal{C}$ ):

- Corresponding to the control action driving the system
- Assumed to be known and to be described by the port variables  $(f_C, e_C)$
- The presence of sources may be included in this port

# All Ports Together

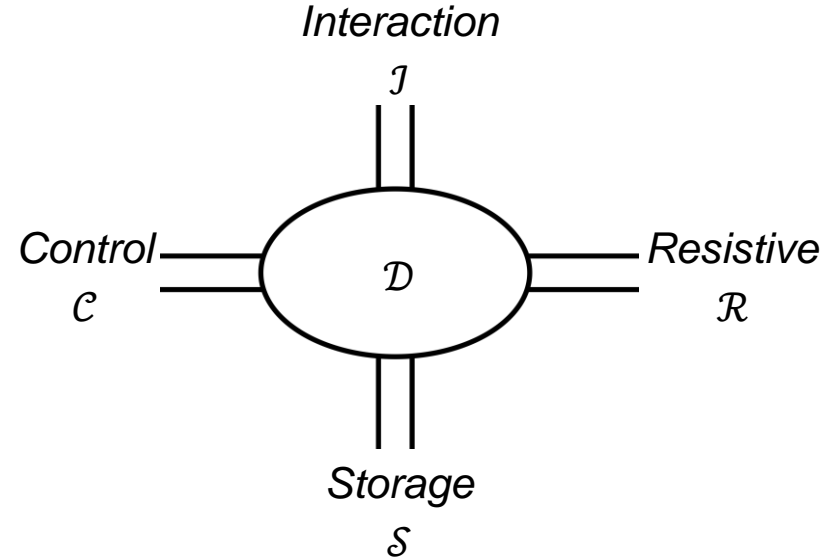
- Dirac structure  $\mathcal{D}$  of the port-Hamiltonian system, satisfies the power balance

$$e_S^T f_S + e_R^T f_R + e_C^T f_C + e_I^T f_I = 0$$

- Hence, for the energy balance

$$\frac{d}{dt}H = e_R^T f_R + e_C^T f_C + e_I^T f_I$$

- The variation of internal energy is equal to the sum of the power provided by the external ports (interaction and control) and the power dissipated by the resistive port



# Port-Hamiltonian Representation

- Multiple ways to represent Dirac structures resulation
- Kernel Representation:

Dirac structure  $\mathcal{D} \subset \mathcal{F} \times \mathcal{E}$ , with  $\mathcal{E} = \mathcal{F}^*$

$$\mathcal{D} = \{(f, e) \in \mathcal{F} \times \mathcal{F}^* | Ff + Ee = 0\}$$

for linear maps  $F: \mathcal{F} \rightarrow \mathcal{V}$  and  $E: \mathcal{E} \rightarrow \mathcal{V}$  satifying

- $EF^* + FE^* = 0$
- $\text{rank}(F + E) = \dim \mathcal{F}$

where  $\mathcal{V}$  is a linear space with same dimensions as  $\mathcal{F}$  with  $F^*: \mathcal{V}^* \rightarrow \mathcal{E}$  and  $E^*: \mathcal{V}^* \rightarrow (\mathcal{F}^*)^* = \mathcal{F}$  are adjoint maps of  $F$  and  $E$

- And for a system with all four mentioned ports:

$$\mathcal{D} = \{(f_S, e_S, f_R, e_R, f_C, e_C, f_I, e_I) \in \mathcal{F} \times \mathcal{F}^* | F_S f_S + E_S e_S + F_R f_R + E_R e_R + F_C f_C + E_C e_C + F_I f_I + E_I e_I = 0\}$$

# Port-Hamiltonian Representation

- By a proper substitution for the matrices of Kernel representation one will reach to the well-known port-Hamiltonian system representation

$$\begin{cases} \dot{x} = [J(x) - R(x)] \frac{\partial H}{\partial x}(x) + G(x)u_C + K(x)u_I \\ y_C = G^T(x) \frac{\partial H}{\partial x} \\ y_I = K^T(x) \frac{\partial H}{\partial x} \end{cases}$$

$(H, x)$ : Storage port Hamiltonian and states

$J(x)$ : Skew-symmetric matrix (assuring the power-conservation)

$R(x)$ : Symmetric positive-definite matrix (representing the dissipation port)

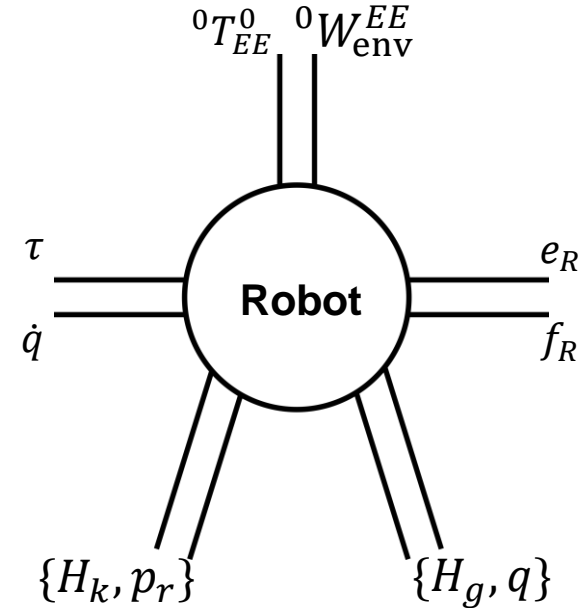
$u_I, u_C, y_I, y_C$ : the input and output port-variables of external ports

$G, K$ : Mapping matrices for the external ports

# Port-based Modeling of a Manipulator

# Modeling of a Robot

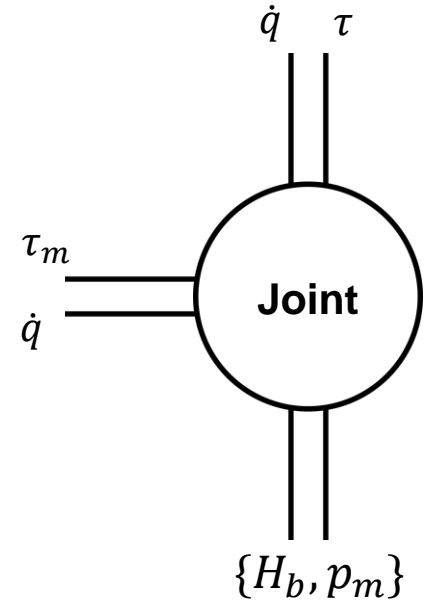
- Storage port – Gravitational**  $\{H_g, q\}$   
 State: Joint positions:  $x_g = q$   
 Hamiltonian: Robot's gravitational energy:  $H_g = U(q)$
- Storage port – Kinetic**  $\{H_k, p_r\}$   
 State: Joint generalized momenta:  $x_k = p_r = M(q)\dot{q}$  ( $M$ : mass matrix)  
 Hamiltonian: Robot's kinetic energy:  $H_k = \frac{1}{2} p_r^T M^{-1}(q) p_r$
- Dissipative port**  $(f_R, e_R)$   
 Flow: Joint velocities:  $f_R = \dot{q}$   
 Effort: Dissipative joint torques:  $e_R = -D(p, q)\dot{q}$  ( $D$ : symmetric positive matrix)
- Control port**  $(\dot{q}, \tau)$   
 Flow: Joint velocities:  $f_C = \dot{q}$   
 Effort: Input joint torques:  $e_C = \tau$
- Interaction port**  ${}^0T_{EE}^0, {}^0W_{env}^{EE}$   
 Flow: Robot's end-effector twist with regard to and described in the base frame:  $f_I = {}^0T_{EE}^0$   
 Effort: Wrench acting on the robot's end-effector described in the base frame:  $e_I = {}^0W_{env}^{EE}$





# Modeling of a Joint

- Storage port – Rotational kinetic**  $\{H_b, p_m\}$   
 State: Motor generalized momenta:  $x_b = p_m = b\dot{q}$  ( $b$ : motor inertia)  
 Hamiltonian: Robot's kinetic energy:  $H_b = \frac{1}{2}p_m^T b^{-1}p_m$
- Control port**  $(\dot{q}, \tau_m)$   
 Flow: Joint velocities:  $f_C = \dot{q}$   
 Effort: Input motor torques:  $e_C = \tau_m$
- Interaction port**  $(\dot{q}, \tau)$   
 Flow: Joint velocities:  $f_C = \dot{q}$   
 Effort: Input joint torques:  $e_C = \tau$



# Modeling of Controller

- **Storage port – Proportional term**  $\{H_s, (q_d - q)\}$

State: Position error:  $x_s = q_d - q$

Hamiltonian: Energy stored in the controller due to the position error:

$$H_s = \frac{1}{2} x_s^T K_P x_s \quad (K_P: \text{controller proportional gain matrix})$$

- **Dissipative port**  $(\dot{q}, -K_D \dot{q})$

Flow: Joint velocities:  $f_R = \dot{q}$

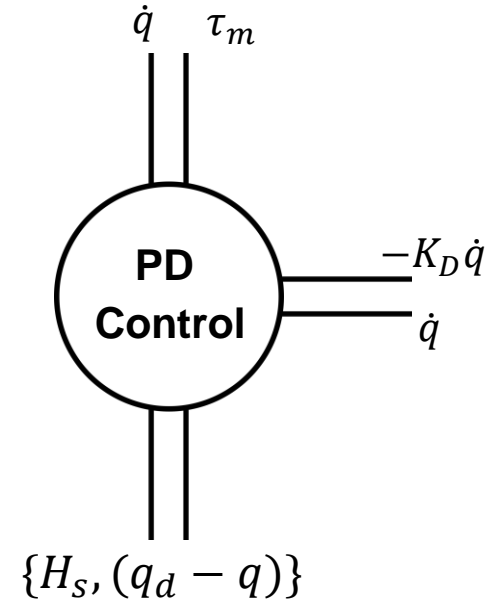
Effort: Dissipative torques due to damping effect:

$$e_R = -K_D \dot{q} \quad (K_D: \text{controller damping gain matrix})$$

- **Interaction port**  $(\dot{q}, \tau_m)$

Flow: Joint velocities:  $f_C = \dot{q}$

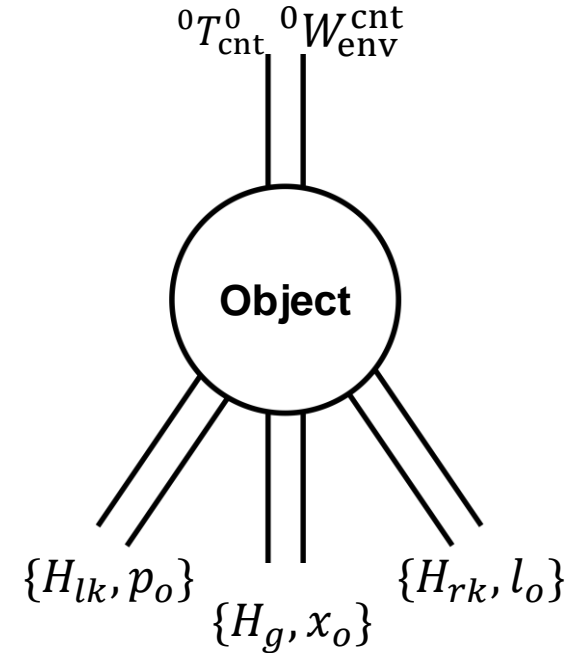
Effort: Input joint torques:  $e_C = \tau_m$



# Modeling of Object

- Storage port – Gravitational**  $\{H_g, x_o\}$   
 State: Object's center of mass position:  $x_g = x_o$   
 Hamiltonian: Object's gravitational energy:  $H_g = mg^T x_o$  ( $m$ : object's mass,  $g$ : gravity vector)
- Storage port – Linear Kinetic**  $\{H_{lk}, p_o\}$   
 State: Object's linear momentum:  $x_{lk} = p_o = m\dot{x}_o$   
 Hamiltonian: Object's linear kinetic energy:  $H_{lk} = \frac{1}{2m} p_o^T p_o$
- Storage port – Angular Kinetic**  $\{H_{rk}, l_o\}$   
 State: Object's rotational momentum:  $x_{rk} = l_o = I_o \omega_o$   
 Hamiltonian: Object's rotational kinetic energy:  

$$H_{rk} = \frac{1}{2} l_o^T I_o^{-1} l_o \quad (I_o: \text{inertia tensor}, \omega_o: \text{angular velocity of object})$$
- Interaction port**  ${}^0T_{\text{cnt}}^0, {}^0W_{\text{env}}^{\text{cnt}}$   
 Flow: Object's contact point twist with regard to and described in the base frame:  $f_I = {}^0T_{\text{cnt}}^0$   
 Effort: Wrench acting on the contact point of the object described in the base frame:  
 $e_I = {}^0W_{\text{env}}^{\text{cnt}}$



# Modeling of Contact

- Storage port – Elastic**  $\{H_s, x_s\}$

State: Contact elasticity's deflection:  $x_s = x_{\text{cnt},1} - x_{\text{cnt},2}$  ( $x_{\text{cnt},i}$ : the contact point's pose from side  $i$ )

Hamiltonian: Stored energy in contact's elastic element:

$$H_s = \frac{1}{2} x_s^T K_{\text{cnt}} x_s \quad (K_{\text{cnt}}: \text{the contact elasticity matrix})$$

- Dissipative port**  $(f_R, e_R)$

Flow: Relative twists on the damping element of the contact: Joint  $f_R = {}^0T_{\text{cnt},1}^0 - {}^0T_{\text{cnt},2}^0$

Effort: Wrench due to the contact's damping effect:

$$e_R = -D_{\text{cnt}} f_R \quad (D_{\text{cnt}}: \text{symmetric positive matrix})$$

- Interaction port** (to one side)  ${}^0T_{\text{cnt},1}^0, {}^0W_{\text{cnt}}^{\text{cnt},1}$

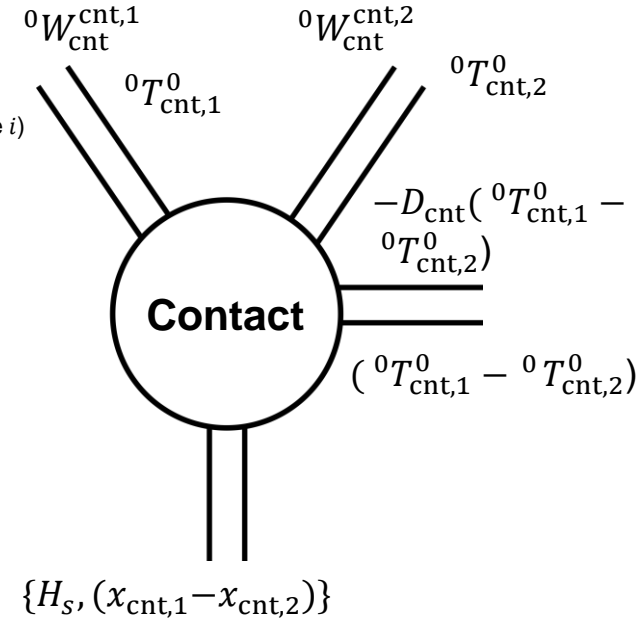
Flow: Twist on one side of the contact are:  $f_I = {}^0T_{\text{cnt},1}^0$

Effort: Wrench acting on the contact area from one side:  $e_I = {}^0W_{\text{cnt}}^{\text{cnt},1}$

- Interaction port** (to another side)  ${}^0T_{\text{cnt},2}^0, {}^0W_{\text{cnt}}^{\text{cnt},2}$

Flow: Twist on another side of the contact are:  $f_I = {}^0T_{\text{cnt},2}^0$

Effort: Wrench acting on the contact area from another side:  $e_I = {}^0W_{\text{cnt}}^{\text{cnt},2}$



# Overall System

Now, connecting all the aforementioned sub-systems, we reach to one unified system with the internal ports of:

- The overall Hamiltonian function will be the summation of all Hamiltonians:

$$H_{tot} = H_{k,controller} + H_{k,robot} + H_{g,robot} + H_{s,contact} + H_{lk,object} + H_{rk,object} + H_{g,object}$$

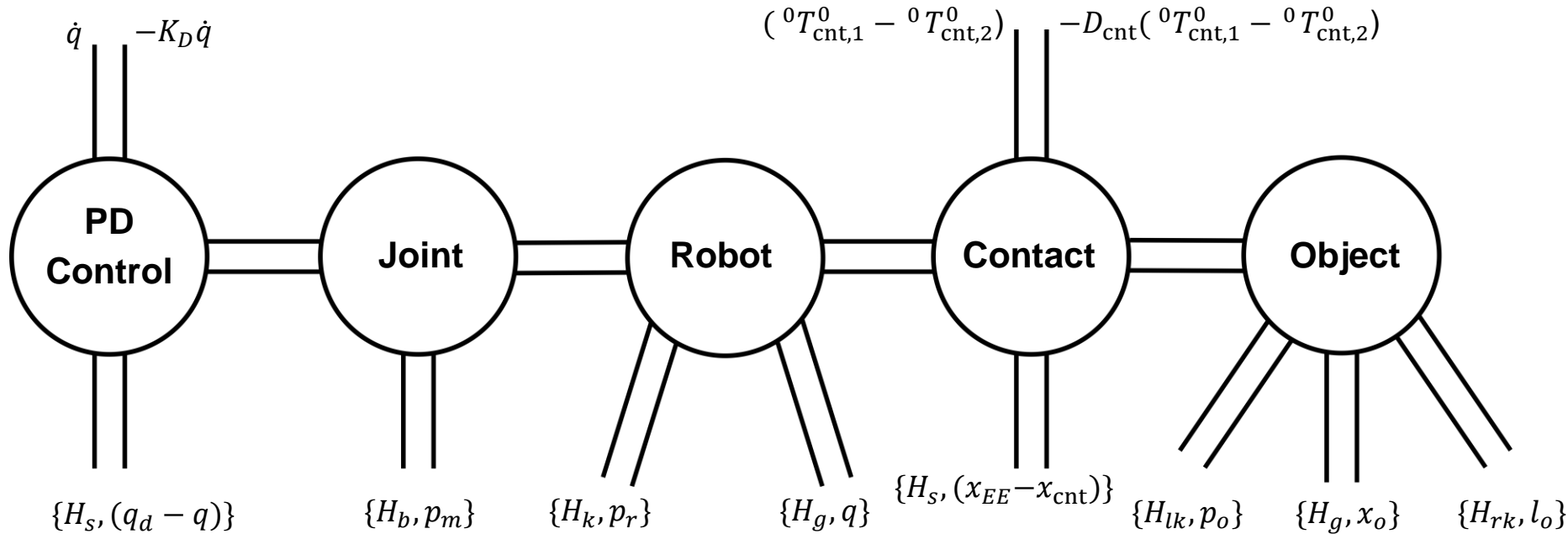
- Assumption: the dissipation port of the robot can be disregarded, and as a result there will be two dissipation ports in the system for the controller and contact

Possible Extensions:

- One can consider an elastic joint, for which there would be an extra storage port for the elasticity, and an extra dissipative port for the damping effect
- One can consider a rigid contact, for which  ${}^0T_{cnt,1}^0 = {}^0T_{cnt,2}^0$  and  ${}^0W_{cnt}^{cnt,1} = -{}^0W_{cnt}^{cnt,2}$ . This can be modelled as a constraint on the twists

After the proper connection, the whole system can be shown as a single port-based model.

# Overall System



# Passivity Analysis

# How to Prove Passivity by Port-based Modeling?

As mentioned before, for a single system with all kinds of ports, the fundamental property in port-based modeling is:

$$\frac{d}{dt}H = e_R^T f_R + e_C^T f_C + e_I^T f_I$$

For our system, this means:

$$\begin{aligned}\frac{d}{dt}H_{tot} &= e_{R,controller}^T \cdot f_{R,controller} + e_{R,contact}^T \cdot f_{R,contact} \\ \rightarrow \dot{H}_{tot} &= -\dot{q}^T K_D \dot{q} - \left( {}^0T_{cnt,1}^0 - {}^0T_{cnt,2}^0 \right)^T D_{cnt} \left( {}^0T_{cnt,1}^0 - {}^0T_{cnt,2}^0 \right) \leq 0\end{aligned}$$

Hence, the overall system is proven to be passive.