

Interaction Control

Introduction to interaction control methods

Joint Space Impedance Control

Task Space Impedance Control

PHYSICAL INTERACTION MOTIVATIONS

Applications

- robots that interact with humans (assistive) robots
- surface interaction tasks (polishing, deburring)
- MOBILE / LEGGED robots
- WEARABLE robots (exoskeletons)

objectives

- avoid applying too large forces (fragile objects)
- control interaction force (e.g. balancing, brushing)

PASSIVE METHODS

Physical springs are introduced between the robot and the environment, to control force (e.g. SEA)



- + more stable control

ACTIVE METHODS

use Feed-back control techniques

① Direct force control: explicit force feedback

⊖ (noisy) To directly regulate contact forces

② Indirect force control: control force and position at the same time

- COMPLIANCE CONTROL
- IMPEDANCE CONTROL

⊕ more flexible : I can change online the stiffness via software

⊖ The controller delay influences the maximum BANDWIDTH we can control the force.

• The most famous indirect methods are :

IMPEDANCE CONTROL : requires inner Torque loop

ADMITTANCE CONTROL : requires inner position loop

DIRECT FORCE CONTROL

IDEA:

- ① measure contact force f
- ② if $f < f^d \Rightarrow$ apply more force
- ③ if $f > f^d \Rightarrow$ apply less force

$$f^* = f^d + K_f (f^d - f) + \dots \text{integral}$$

e_f

$$\zeta = -J^T f^* + R$$

DYNAMICS:

$$M\ddot{q} + h = \zeta + J^T f$$

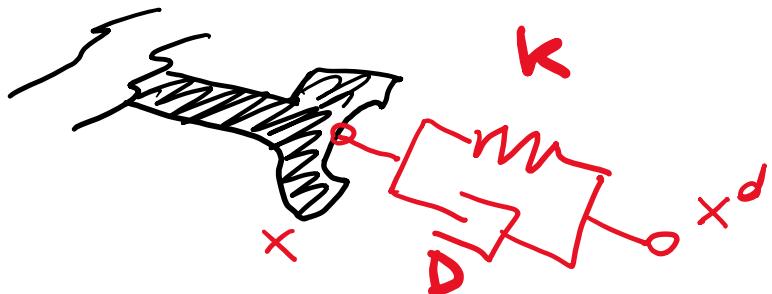
at steady state:

$$\cancel{g} - J^T f = -J^T f^* + \cancel{g} \Rightarrow f = f^d + K_f e_f$$
$$(1 + K_f) e_f = 0 \Rightarrow e_f = 0$$

IMPEDANCE CONTROL

IDEA: indirectly regulate forces by generating a motion that satisfy a dynamic relationship (impedance) between force and position. (e.g emulates SPRINGS / DAMPERS)

MECHANICAL IMPEDANCE



$$\frac{F(s)}{X(s)} = M s^2 + D s + K$$

(Laplace Domain)

- Mechanical impedance gives an idea on how a point of a system moves if you apply a force to it.

IMPEDANCE CONTROL IS IN THE MIDDLE ...

POSITION CONTROL

control position no matter what force is applied

↑ accuracy
⊕

↑ force

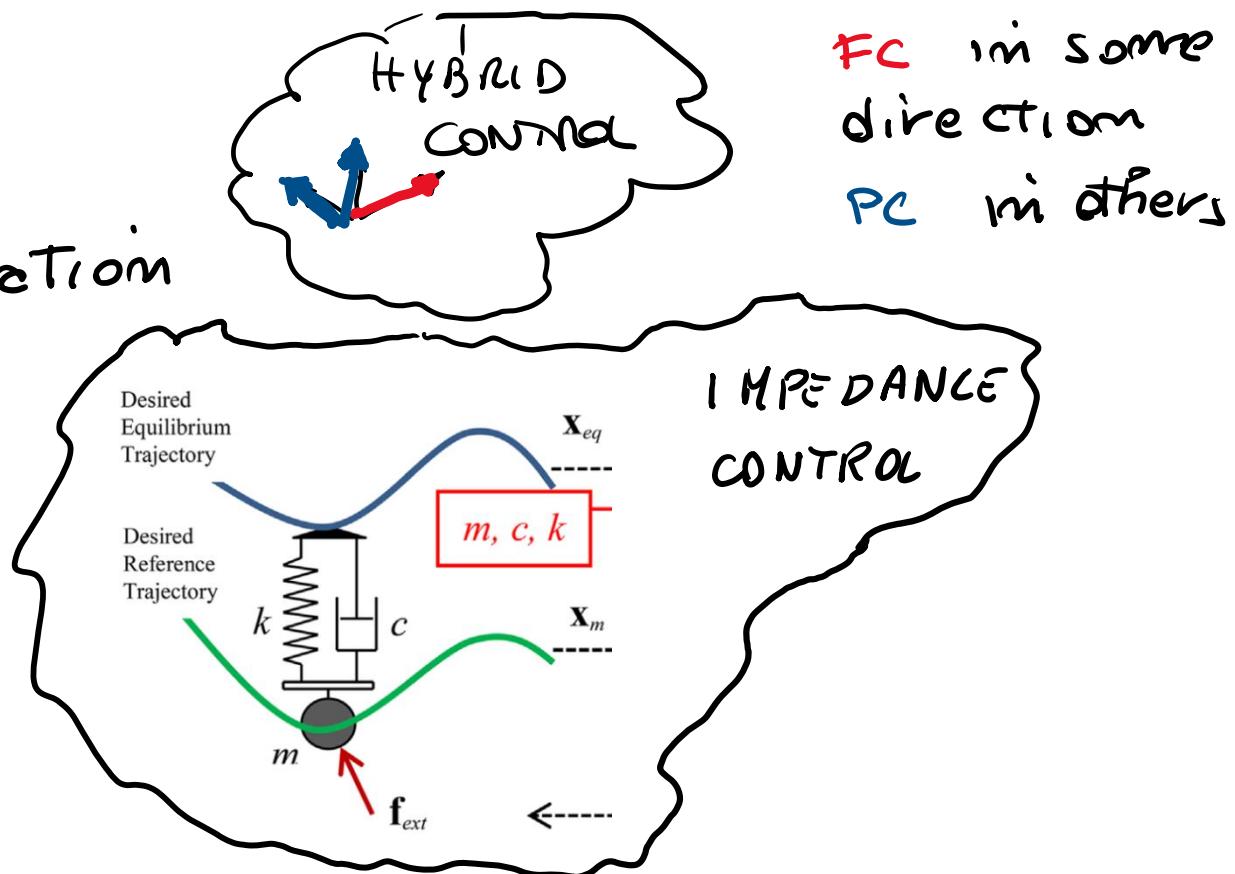
- actuator saturation
- break mechanical parts

FORCE CONTROL

control force no matter which position is achieved

FC in some direction

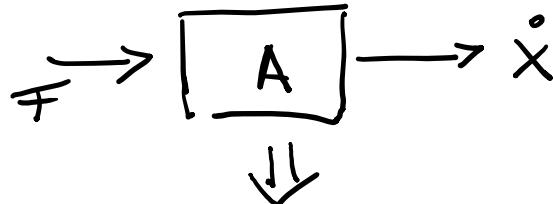
PC in others



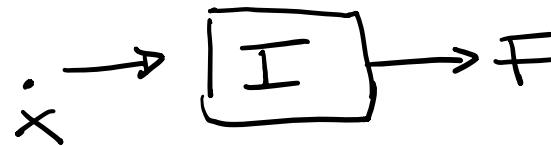
CAUSALITY

[Hogan 1985]

ADMITTANCES



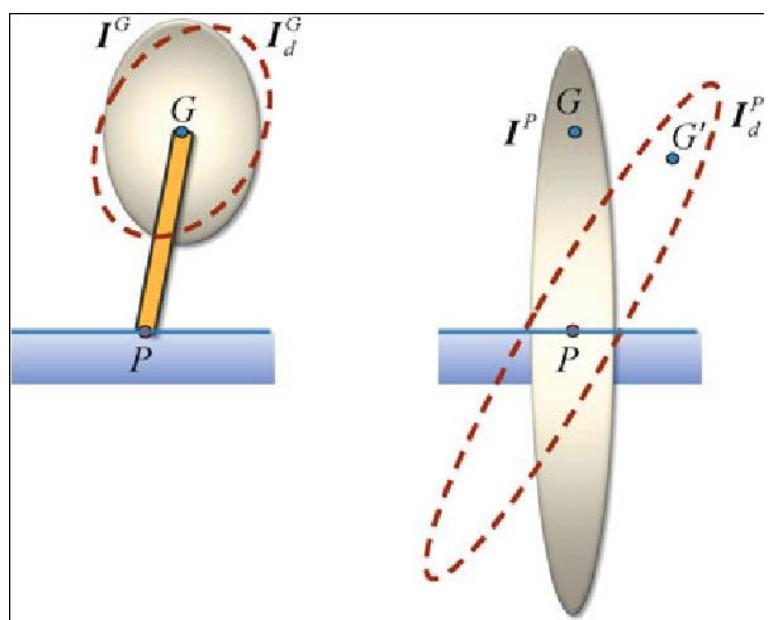
IMPEDANCES



- inertial objects accept force as input and give motion as output (ADMITTANCES)
- To interact with the environment the robot must behave as an **impedance** (cannot connect 2 admittances together)
- Idea: control robot motion, give a "disturbance" response for deviations from that motion, that has the form of an impedance

INERTIA SHAPING FEATURE

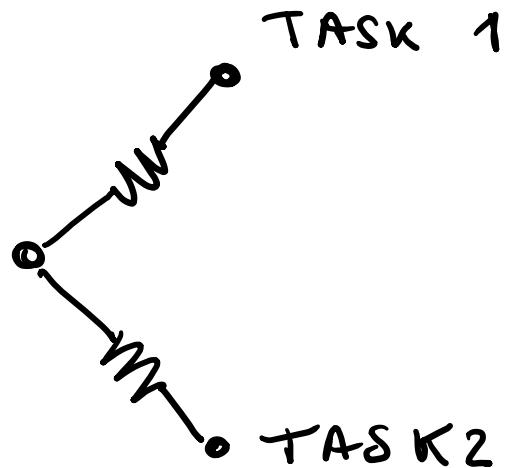
- mask the true inertia of the manipulator and (differently from op. sp. control) impose a desired one at the END-EFFECTOR (I cannot change manipulator inertia but I can change the apparent one at the end effector)



e.g. I can make it
configuration
independent

SUPERPOSITION OF IMPEDANCES

Because the desired impedances are linear we can superimpose their effects



- each impedance can represent one task
- The behaviour will be a compromise between the tasks (if they are conflicting)

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JOINT SPACE IMPEDANCE CONTROL

Desired impedance:

$$(1) M_\theta \ddot{q} + D_\theta (\dot{q}^d - \dot{q}) + K_\theta (q^d - q) = \mathbf{f}_{ext}$$

Dynamics:

$$M \ddot{q} + R = \mathcal{Z} + J^T F_{ext}$$

- choose \mathcal{Z} to behave as desired dynamics.

$$(1) \ddot{q} = M_\theta^{-1} [\mathcal{Z}_{ext} - D_\theta (\underbrace{\dot{q}^d - q}_{\dot{e}}) - K_\theta (\underbrace{q^d - q}_e)]$$

$$\boxed{\mathcal{Z} = R - J^T F_{ext} + M M_\theta^{-1} [\mathcal{Z}_{ext} - D_\theta \dot{e} - K_\theta e]}$$

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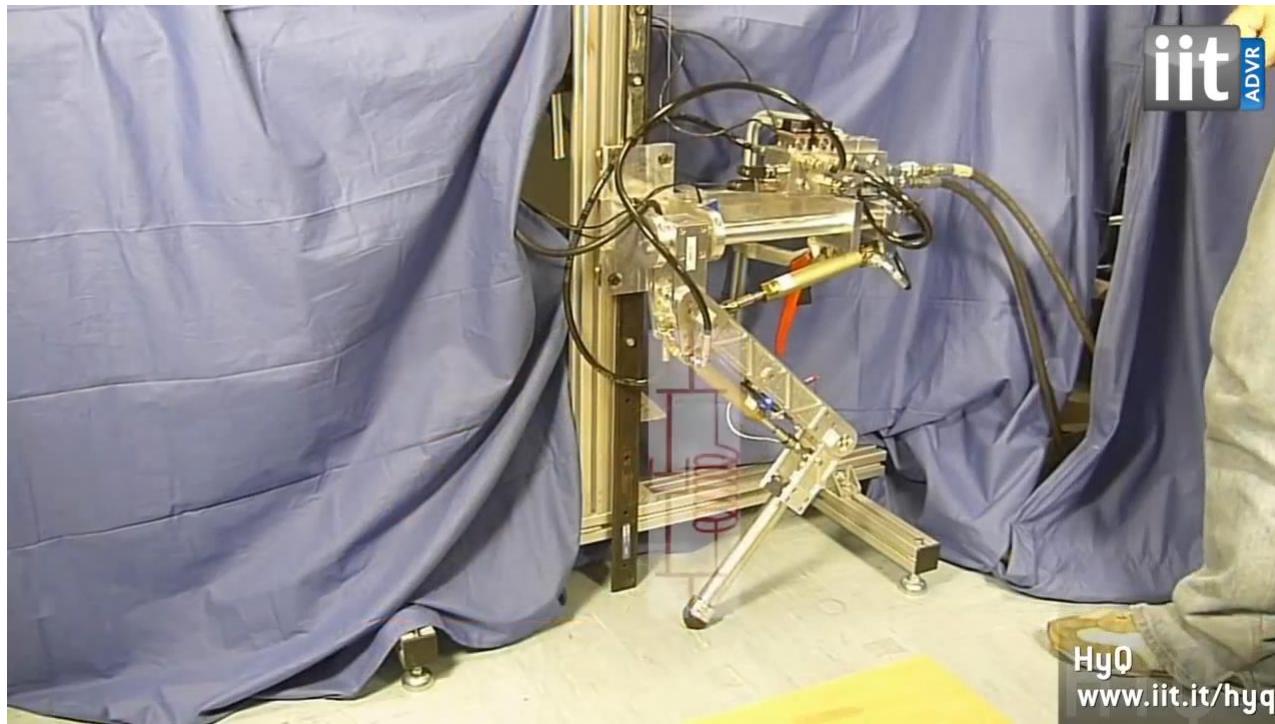
Task Space Impedance Control

TASK SPACE IMPEDANCE CONTROL

Desired impedance:

$$(1) \quad M_x \ddot{x} + K_x(x^d - x) + D_x(\dot{x}^d - \dot{x}) \leq F_{ext}$$

↳ desired inertia at end-effector



Dynamics:

$$(1) M\ddot{q} + R = \bar{z} + J^T F_{ext}$$

$$\boxed{\bar{z} = M\ddot{x}^d + R - J^T F_{ext}}$$

where :

$$(1) \rightarrow \ddot{x}^d = M_x^{-1} \left[F_{ext} - K_x (x^d - x) - D_x (\dot{x}^d - \dot{x}) \right]$$
$$\ddot{q}^d = J^{-1} (\ddot{x}^d - J\dot{q}) = J^{-1} \ddot{x}^d - J^{-1} J \dot{q}$$

$$(3) \boxed{\bar{z} = M J^{-1} M_x^{-1} (F_{ext} - K_x (x^d - x) - D_x (\dot{x}^d - \dot{x})) - M J^{-1} J \dot{q} + R - J^T F_{ext}}$$

⊖ exact cancellation is impossible

⊖ $M J^{-1} M_x^{-1}$ big forces if M_x is small

(-) requires Torque sensor at the joints and force sensor at the end-effector to measure F_{ext}

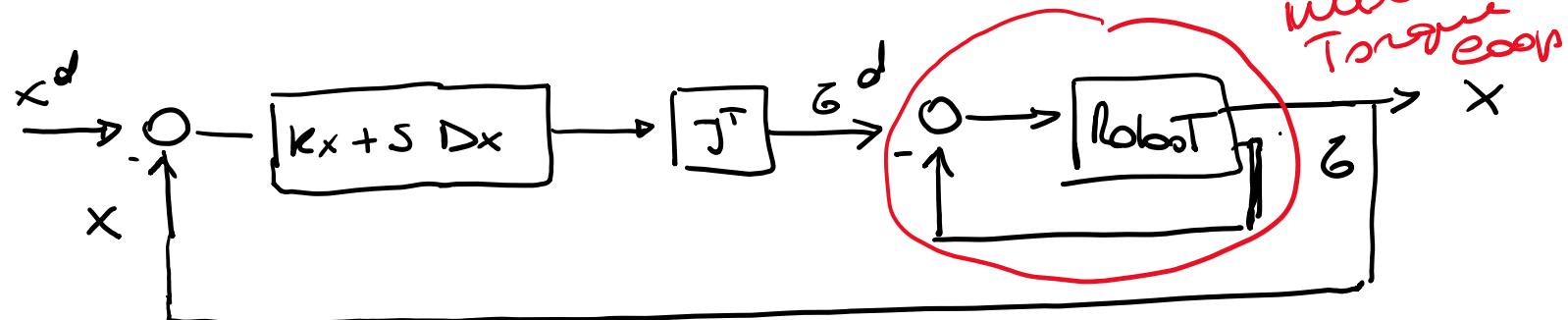
(+) plugging (3) into (2) we cancel the robot dynamics enforcing the desired one:

$$\ddot{x} = \ddot{x}^d \rightarrow M \ddot{x} = F_{ext} - k_x(x^d - x) - D_x(\dot{x}^d - \dot{x})$$

PERFECT IMPEDANCE EMULATION

\Rightarrow because of (-) is not so used in practice and only k_x, D_x terms are used:

$$(4) \quad \ddot{z} = J^T [k_x(x^d - x) + D_x(\dot{x}^d - \dot{x})]$$



- (4)
- equivalent to 2 PD control in Task space
 - no need of contact force sensor
 - OK if inertias are small
 - passive if $K_x > 0, D_x > 0$



stable when interacting
with passive environments

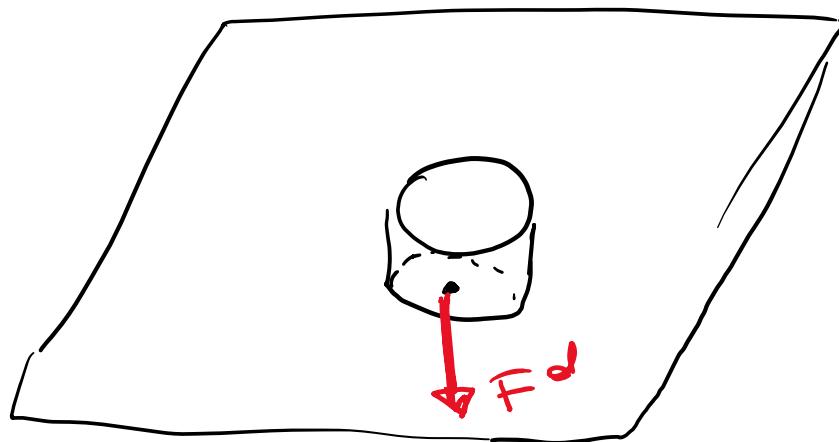
ADDING DESIRED FORCE

Desired impedance:

$$M_x \ddot{x} + K_x (x^d - x) + D_x (\dot{x}^d - \dot{x}) = F^d - F_{ext}$$

EXAMPLES :

Deburring , cleanup



DIFFERENCE WITH PD CONTROL

PD CONTROL

- control position

VS

IMPEDANCE CONTROL

- K_p, K_d have no physical meaning

$$u = K_p e + K_d \dot{e}$$

u = valve opening / voltage

- There is no inner loop

- do not care about disturbances

makes the robot appear as a physical system



- control relationship B/w position and force



- K, D have physical meaning of stiffness / damping



- There is a torque inner loop

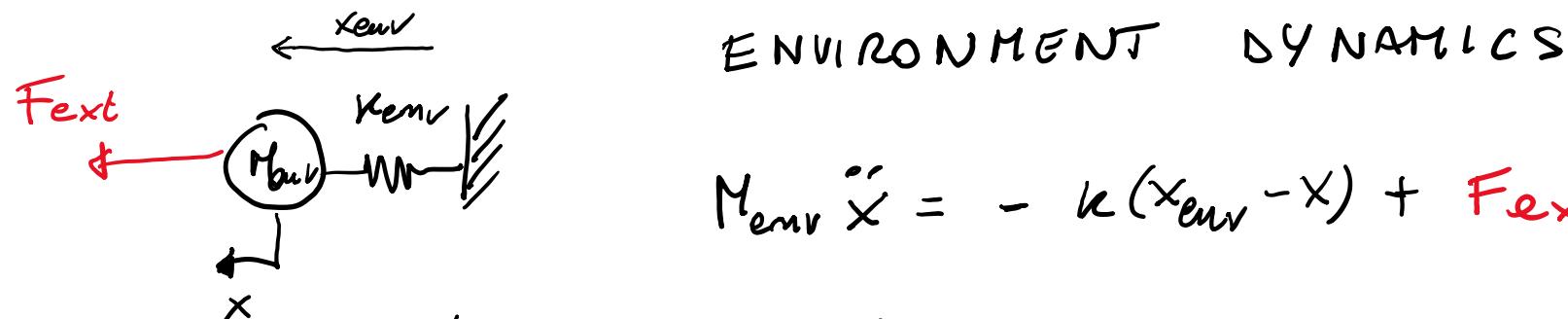


- gives a controlled response to a disturbance from environment

STABILITY IN CONTACT WITH ENVIRONMENT

①

STABILITY depends both on impedance parameters and on dynamics of environment



$$M_x \ddot{x} + K_x (\overset{\circ}{x} - x) + D_x (x^d - \dot{x}) = F_{\text{ext}}$$

CLOSED LOOP DYNAMICS:

$$(M_x + M_e) \ddot{x} + D_x \dot{x} + (K_x + K_{\text{env}}) x = k_x x^d + K_{\text{env}} x_{\text{env}} + D_x \dot{x}^d$$

\hookrightarrow is asymptotically stable if

M_x, D_x, K_x are closed positive definite

Passivity

Restatement of the
energy conservation principle

A passive system
cannot store more
energy than is
supplied to it from
the outside



Coupled stability via passivity

Active System



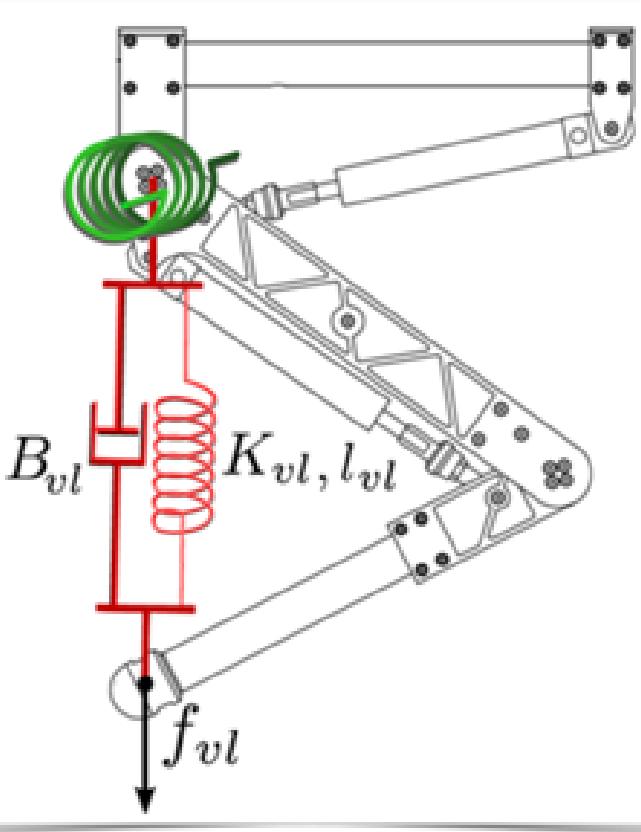
Passive
Elements

Stable
Interaction

Passive
Environment

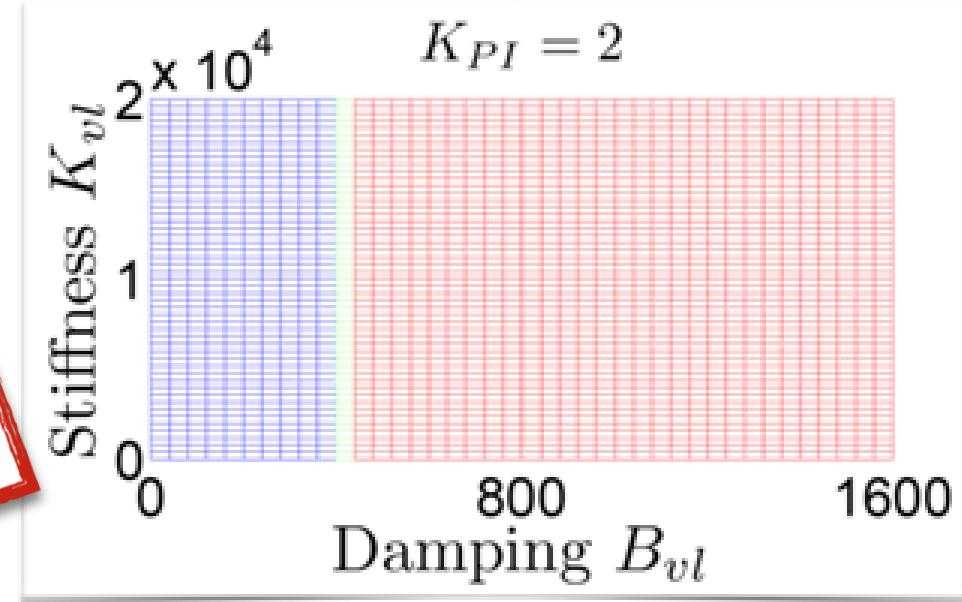
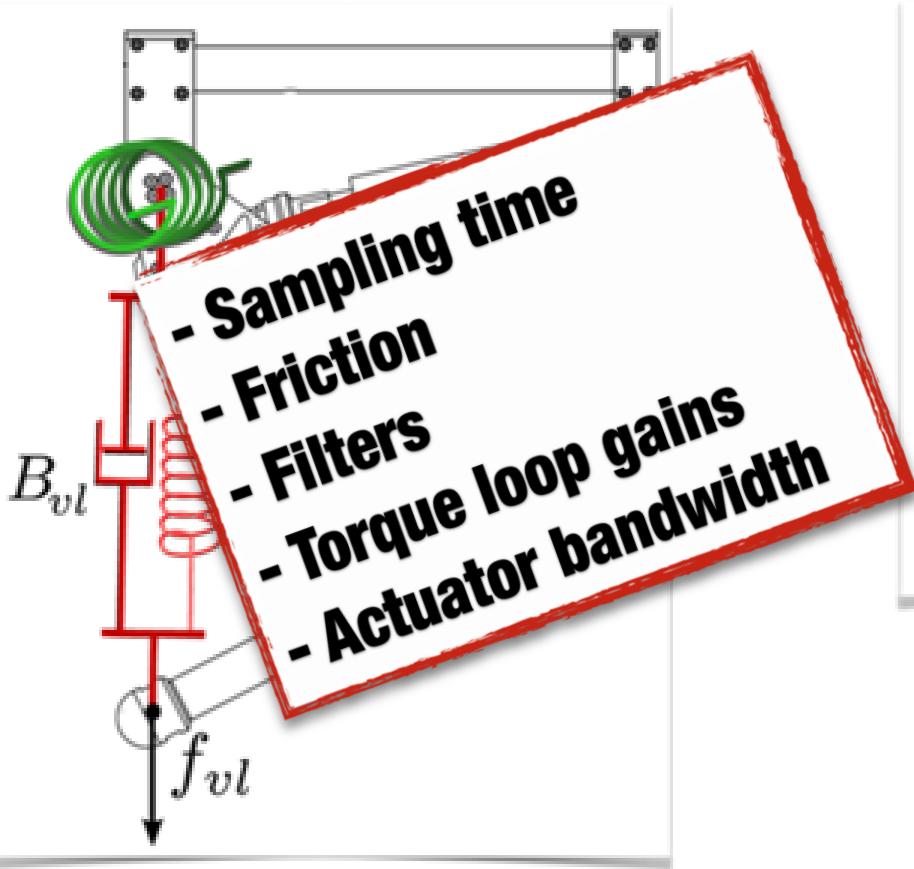


Z-Width



Range of **stiffness** and
damping that keeps
the system **passive**

Z-Width



- Passive range
- Stable, but not passive range
- Unstable range

[Boaventura et al. IROS, 2013]
[Focchi, PhD Thesis, 2013]

References

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