

# Finite Difference Method applied to DML simulation. Part 2b.

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## Abstract

Simulation of Distributed Mode Loudspeaker (DML) thanks to the Finite Difference Method (FDM)

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**Disclaimer** : this paper is written in the context of DIY DML building. This document is not written in the context of any academic or scientific work. Its content is reviewed only by the feedback it can get while posting it in audio DIY forum like [diyAudio](#).

The pdf format of the paper is directly extracted from a python script See Github [py2pdf](#) for more information about this method.

## 1 Introduction

The part 2a is the application of part 1 in a Python script up to the calculation of the system matrix.

In this part 2b, the focus is on the modes (frequency and space shape) and the comparison of the results of the script with some plate situations simulated with Elmer or having exact solutions.

[Elmer](#) is a free multi-platform FEM solver. More details about the simulation files used below.

The tentative to use formulas to extend the simply supported plate case comes from [1], is applied in the function `modes()` below and compare in the results to the `elmer` output.

## 2 Eigenfrequencies

In part 1, the partial derivative equation (PDE) of the vibration of a plate was shown.

For an isotropic homogeneous material, the governing equation is :

$$D\left(\frac{\partial^4 w}{\partial x^4} + 2\frac{\partial^4 w}{\partial x^2 \partial y^2} + \frac{\partial^4 w}{\partial y^4}\right) = -q(x, y, t) - \mu \frac{\partial^2 w}{\partial t^2} \quad (1)$$

The PDE in the case of an orthotropic material was also shown (refer to part 1).

The calculation of the eigenfrequencies is a specific case where this equation is solved.

Looking for the natural resonance frequencies, the external load is assumed to be null :  $q(x, y, t) = 0$ .

The second assumption is to separate the plate deviation  $w$  in a product of 2 functions :

$$w(x, y, t) = W(x, y) \cos(\omega t)$$

$w$  being in the time a cosine function, its second derivative over the time is also a cosine so the time “disappears” from the equation which becomes :

$$D\left(\frac{\partial^4 W}{\partial x^4} + 2\frac{\partial^4 W}{\partial x^2 \partial y^2} + \frac{\partial^4 W}{\partial y^4}\right) = \mu \omega^2 W \quad (2)$$

Which is with the finite difference method (see also part 1) FDM in the form :

$$M_{sys} W = \mu \omega^2 W \quad (3)$$

The modes are the eigenfrequencies of the  $M_{sys}$  and the mode shapes are the eigenvectors

This [web page](#) explains the concept of eigenvalue and of eigenvector; how `numpy` can extract them.

## 3 The testing plate

See figure 1 for the plate representation in the FDM script.

## 4 The script results

The plate is  $L_x = 0.4$  m by  $L_y = 0.6$  m

The plate thickness is  $h = 0.015$  m

The mesh is  $N_x = 41$  cells by  $N_y = 61$  cells

with a grid  $dx = 10$  mm by  $dy = 10$  mm

Young modulus  $E = 10$  MPa, Density  $\rho = 25.0$  kg/m<sup>3</sup>

Bending stiffness  $B = 3.09$  Nm, Areal density  $\mu = 0.375$  kg/m<sup>2</sup>

Case 0

Boundary conditions ['C [north]', 'C [west]', 'C [south]', 'C [east]']

Boundary coeff. North= 1.0 West= 1.0 South= 1.0 East= 1.0

m	n	elmer	FDM	err%	formula	err%
1	1	76	76	0	76	0
1	2	118	118	0	118	0
2	1	186	187	0	189	1
1	3	188	189	0	189	0
2	2	225	226	0	229	1
1	4	285	286	0	288	1
2	3	290	292	0	296	2
3	1	353	354	0	359	1

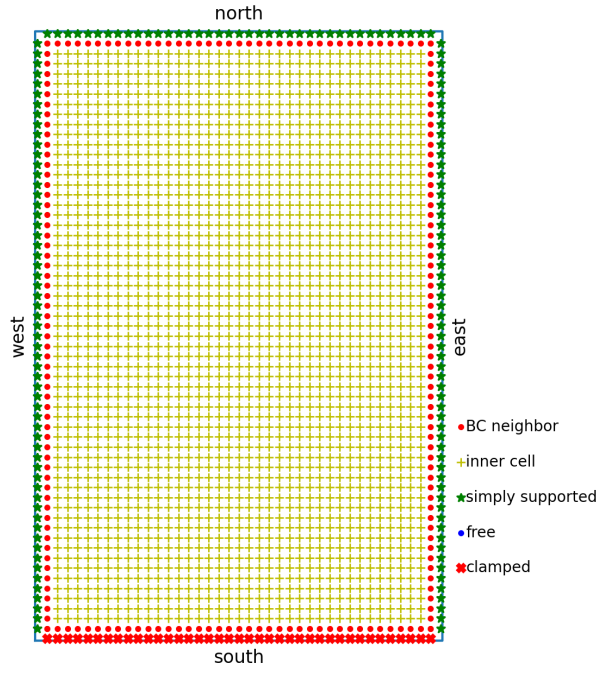


Figure 1: Testing plate

2	4	383	385	0	390	1
3	2	389	392	0	397	2
1	5	407	408	0	412	1
3	3	452	456	0	462	2
2	5	502	504	0	510	1
3	4	542	546	0	555	2
1	6	554	554	0	562	1
4	1	573	574	0	585	2
4	2	609	612	0	622	2
2	6	647	649	0	657	1
3	5	658	662	0	673	2
4	3	670	675	0	687	2
1	7	725	724	0	738	1
4	4	757	763	0	778	2
3	6	799	804	0	818	2
2	7	816	817	0	831	1
5	1	848	847	0	868	2

Mean error 0.36 %

Case 1

Boundary conditions ['S [north]', 'S [west]', 'S [south]', 'S [east]']

Boundary coeff. North= -1.0 West= -1.0 South= -1.0 East= -1.0

m	n	elmer	FDM	err%	formula	err%
1	1	40	40	0	40	0
1	2	77	78	1	78	1
2	1	124	125	0	125	0
1	3	140	140	0	140	0
2	2	161	162	0	162	0
2	3	223	225	0	225	0
1	4	227	227	0	228	0
3	1	265	265	0	266	0
3	2	302	302	0	303	0
2	4	310	312	0	313	0
1	5	340	339	0	341	0
3	3	363	364	0	366	0
2	5	423	423	0	425	0

3	4	449	452	0	454	1
4	1	463	459	0	463	0
1	6	477	475	0	479	0
4	2	499	497	0	501	0
4	6	560	559	0	901	60
4	6	560	559	0	901	60
3	5	561	563	0	566	0
1	7	640	635	0	641	0
4	4	645	646	0	651	0
3	6	699	699	0	704	0
5	1	717	708	-1	717	0
2	7	723	719	0	726	0

Mean error 0.03 %

Case 2

Boundary conditions ['S [north]', 'S [west]', 'C [south]', 'C [east]']

Boundary coeff. North= -1.0 West= -1.0 South= 1.0 East= 1.0

m	n	elmer	FDM	err%	formula	err%
1	1	56	56	0	57	1
1	2	96	96	0	97	1
2	1	154	154	0	155	0
1	3	163	163	0	164	0
2	2	191	192	0	194	1
1	4	255	255	0	257	0
2	3	255	257	0	259	1
3	1	308	308	0	311	0
3	2	344	345	0	348	1
2	4	345	347	0	350	1
1	5	373	372	0	376	0
3	3	406	409	0	413	1
2	5	461	463	0	467	1
3	4	494	497	0	503	1
1	6	515	514	0	520	0
4	1	517	515	0	522	0
4	2	553	553	0	560	1
2	6	603	603	0	609	0
3	5	608	612	0	618	1
4	3	614	616	0	623	1
1	7	682	679	0	689	1
4	4	700	703	0	713	1
3	6	748	750	0	760	1
2	7	769	767	0	777	1
5	1	781	776	0	790	1

Mean error 0.13 %

Case 3

Boundary conditions ['S [north]', 'C [west]', 'S [south]', 'C [east]']

Boundary coeff. North= -1.0 West= 1.0 South= -1.0 East= 1.0

m	n	elmer	FDM	err%	formula	err%
1	1	71	71	0	48	-32
1	2	99	100	1	100	1
1	3	155	155	0	177	14
2	1	183	184	0	129	-29
2	2	213	214	0	176	-17
1	4	239	239	0	278	16
2	3	266	268	0	253	-4
1	5	346	348	0	404	16
2	4	349	348	0	355	1
3	1	350	352	0	269	-23
3	2	381	383	0	313	-17
3	3	434	438	0	386	-11
2	5	452	454	0	482	6
1	6	485	482	0	555	14

3	4	512	517	0	487	-4
4	1	571	573	0	466	-18
2	6	585	585	0	634	8
4	2	603	605	0	508	-15
3	5	617	620	0	615	0
1	7	646	641	0	731	13
4	3	656	661	0	579	-11
4	4	734	740	0	678	-7
2	7	744	741	0	810	8
3	6	747	749	0	767	2
1	8	832	823	-1	932	12

Mean error 0.24 %

Case 4

Boundary conditions ['S [north]', 'S [west]', 'C [south]', 'S [east]']

Boundary coeff. North= -1.0 West= -1.0 South= 1.0 East= -1.0

m	n	elmer	FDM	err%	formula	err%
1	1	44	44	0	53	20
1	2	88	88	0	86	-2
2	1	126	127	0	153	21
1	3	157	157	0	146	-7
2	2	168	169	0	187	11
2	3	236	238	0	246	4
1	4	250	251	0	233	-6
3	1	267	266	0	309	15
3	2	307	307	0	344	12
2	4	330	331	0	329	0
1	5	369	369	0	345	-6
3	3	373	374	0	403	8
2	5	448	449	0	439	-2
4	1	464	460	0	521	12
3	4	465	467	0	486	4
4	2	503	501	0	556	10
1	6	512	511	0	482	-5
4	3	567	567	0	616	8
3	5	583	585	0	595	2
2	6	591	592	0	575	-2
4	4	658	659	0	699	6
1	7	679	676	0	645	-5
5	1	718	708	-1	789	9
3	6	726	727	0	729	0
5	2	756	748	-1	825	9

Mean error 0.0 %

## 5 Computational efficiency

The time to get the result depend of course of the computer. For the one used for this test, the results from Elmer were obtains much more faster than with this FDM script which let suppose there are important possibilities of improvements but which need skills in coding.

The possibilities offered by the FDM script like adding lumped elements remain.

The execution time increases as the cube of the number of cells.

Possible heuristic :  $t_{exec} = cte + N_{cells}^3/K$

See figure 2

## 6 FDM performances

With a 20mm grid, the precision for the 5 test cases are : -1.7, -.9, -1.2, -1.5, -1% over the 16th first modes.

With a 10mm grid, the precision for the 5 test cases are : .36, .21, .18, .36, .13% over the 16th first modes.

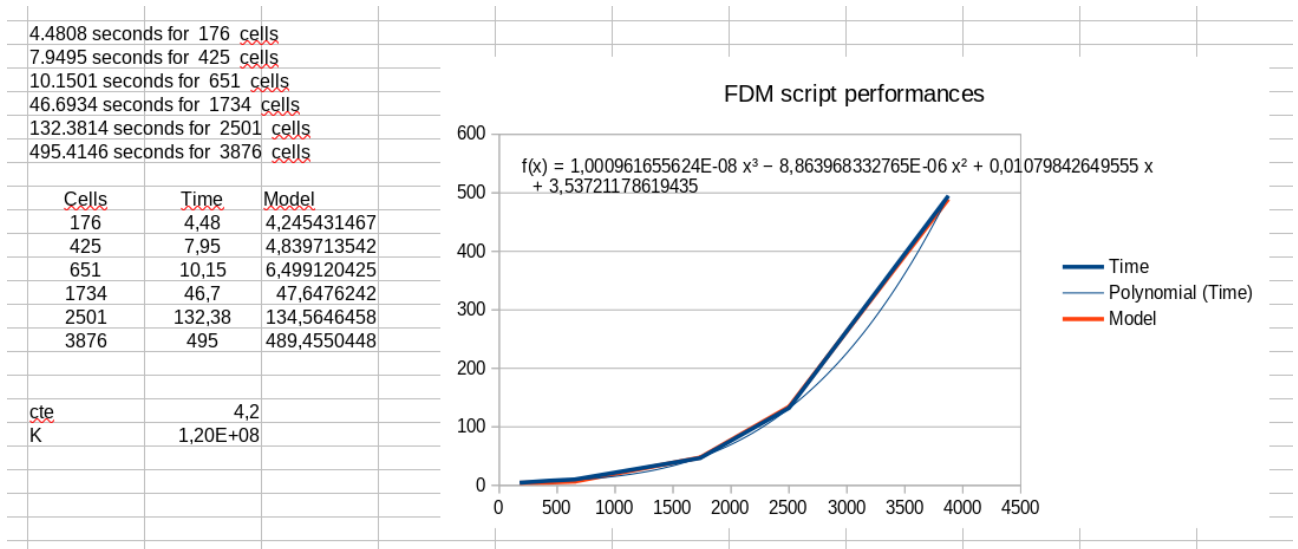


Figure 2: FDM performance

The paper shows a 10mm grid over the 25th first modes.

The formula tested gives good results for SSSS and CCCC but not so good in the mix of conditions which was not expected reading the original paper where the limitations are given when the free edge condition is used. As the elmer simulation is working in parallel of the FDM, there is less interest in a reference from a formula. The possible explanation of the deviation even if it is a bug is kept out of the scope of this test run.

## 7 Mode shapes

The FDM script extracts the mode shape. Below is an example presented in a colored 2D. Other views are possible with matplotlib.

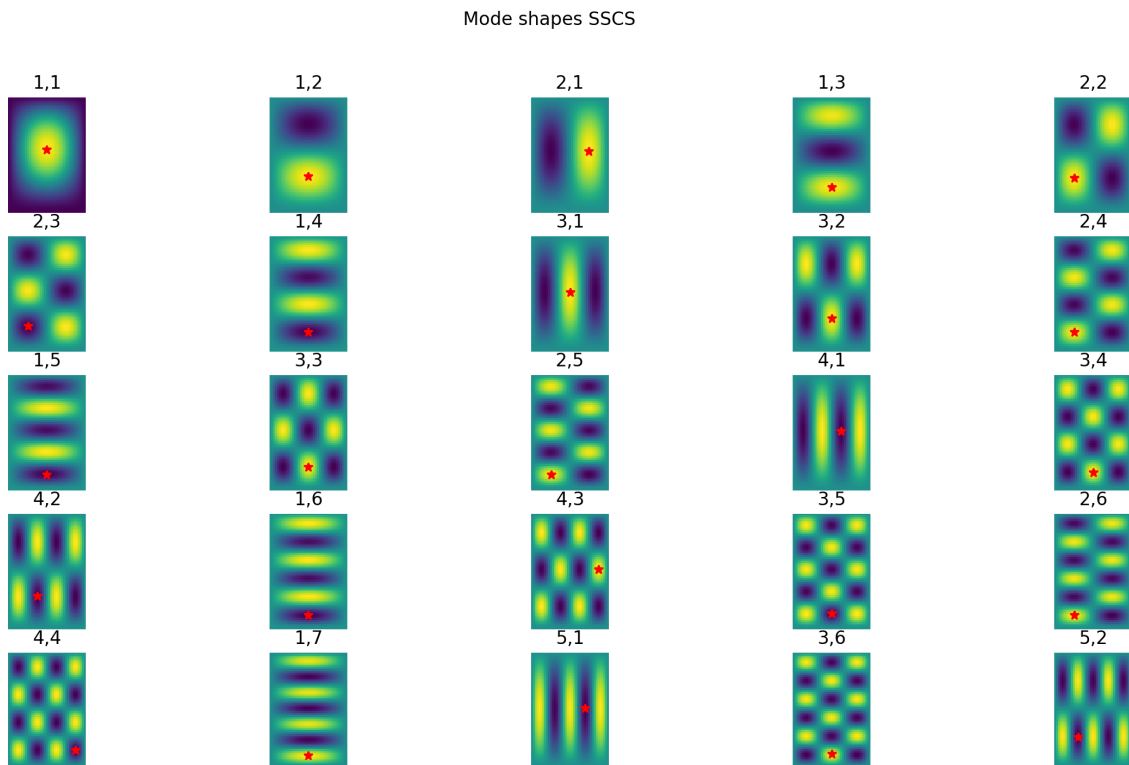


Figure 3: FDM performance

See figure 3

## 8 Elmer

### 8.1 Geometry (rectangular\_plate.geo)

```
// Inputs
Lx = 0.4; //m
Ly = 0.6; //m
gridsize = Lx / 20;

// All numbering counterclockwise from bottom-left corner
Point(1) = {-Lx/2, -Ly/2, 0, gridsize};
Point(2) = {Lx/2, -Ly/2, 0, gridsize};
Point(3) = {Lx/2, Ly/2, 0, gridsize};
Point(4) = {-Lx/2, Ly/2, 0, gridsize};
Line(1) = {1, 2};           // bottom line
Line(2) = {2, 3};           // right line
Line(3) = {3, 4};           // top line
Line(4) = {4, 1};           // left line
Line Loop(5) = {1, 2, 3, 4};
// the order of lines in Line Loop is used again in surfaceVector[]
Plane Surface(6) = {5};
```

The rectangular\_plate.msh is obtain from the gmsh application.

### 8.2 Mesh

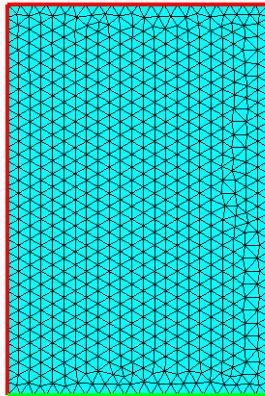


Figure 4: plate mesh

See figure 4

### 8.3 Solver (case.sif)

```
Header
  CHECK KEYWORDS Warn
  Mesh DB "." "."
  Include Path ""
  Results Directory ""
End

Simulation
  Max Output Level = 5
  Coordinate System = Cartesian
  Coordinate Mapping(3) = 1 2 3
  Simulation Type = Steady state
  Steady State Max Iterations = 1
  Output Intervals = 1
  Timestepping Method = BDF
```

```

BDF Order = 1
Solver Input File = case.sif
Post File = case.vtu
End

```

```

Constants
Gravity(4) = 0 -1 0 9.82
Stefan Boltzmann = 5.67e-08
Permittivity of Vacuum = 8.8542e-12
Boltzmann Constant = 1.3807e-23
Unit Charge = 1.602e-19
End

```

```

Body 1
Target Bodies(1) = 1
Name = "Body 1"
Equation = 1
Material = 1
End

```

```

Solver 1
Equation = Elastic Plates
Variable = -dofs 3 Deflection
Eigen System Values = 25
Eigen Analysis = True
Eigen System Select = Smallest magnitude
Procedure = "Smitc" "SmitcSolver"
Exec Solver = Always
Stabilize = True
Bubbles = False
Lumped Mass Matrix = False
Optimize Bandwidth = True
Steady State Convergence Tolerance = 1.0e-5
Nonlinear System Convergence Tolerance = 1.0e-7
Nonlinear System Max Iterations = 1
Nonlinear System Newton After Iterations = 3
Nonlinear System Newton After Tolerance = 1.0e-3
Nonlinear System Relaxation Factor = 1
Linear System Solver = Direct
Linear System Direct Method = Umfpack
End

```

```

Equation 1
Name = "Plate Equation"
Active Solvers(1) = 1
End

```

```

Material 1
Name = "EPS"
Porosity Model = Always saturated
Tension = 0.0
Youngs modulus = 10e6
Thickness = 0.015
Poisson ratio = 0.
Density = 25
End

```

```

Boundary Condition 1
Target Boundaries(2) = 3 4
Name = "fixed"
Deflection 2 = 0

```



```

    Deflection 3 = 0
    Deflection 1 = 0
End

Boundary Condition 2
    Target Boundaries(2) = 1 2
    Name = "Simply supported"
    Deflection 1 = 0
End

```

The Elmer results are stored in the `elmer()` function below to be displayed along with the script results.

## 9 Python script code

### 9.1 Import modules

```

import os # for py2pdf
import subprocess # for py2pdf with bash with alias
import libpy2pdf as p2p
import matplotlib.pyplot as plt
import numpy as np
from scipy import linalg
# from scipy.sparse.linalg import eigs
import time # for script time performance
import psutil # for script memory alloc check

```

### 9.2 Function mode()

```

def modes(B, aa, bb, BB, mumu, mm, nn): # called to check FDM results
    # return the mode frequency of
    # a plate size aa in x by bb in y, material stiffness BB, surface density mumu
    # for the mode mm, nn
    # under the boundary conditions B, B being something like ['SS', 'SS', 'C', 'SS']
    # extension of the Hazell Mitchell approach in R Masso thesis
    k1 = 1/np.pi/2/aa**2*(BB/mumu)**.5 # to transform dimension less lambda in frequency
    r = mm*bb/nn/aa # modal parameter
    dm = 1/(1/r**2 + 2) + 1/60/r # edge correction in m, x direction
    dn = 1/(r**2 + 2) + r/60 # edge correction in n, y direction
    km = 0 # edge coeff in m, x, set to 0 by default (SS condition)
    kn = 0 # edge coeff in n, y, set to 0 by default (SS condition)
    # add 1/2 each time an edge is clamped, either in m or n
    # corrections in one direction m or n : SS => 0, CS or SC => 1/2, CC => 1
    if B[0]=='C': # North
        km = km + 0.5
    if B[1]=='C': # West
        kn = kn + 0.5
    if B[2]=='C': # South
        km = km + 0.5
    if B[3]=='C': # East
        kn = kn + 0.5
    # corrected modes
    mc = mm + km*dm
    nc = nn + kn*dn
    lbda = np.pi**2*(mc**2 + nc**2*(aa/bb)**2) # dimensionless modal frequency parameter
    ff = k1*lbda # frequency Hz to return
    return ff

```

### 9.3 Functions elmer()

```
def elmer(Bname, rank, rankmax):
    if rank > rankmax:
        rank = rankmax
    # SSSS
    if Bname == 'SSSS':
        mode = np.array([40.4, 77.6, 124.8, 140.1, 161.5, 223.4, 227.7, 265.8, 302.1,
                        310.6, 340.4, 363.3, 423.1, 449.9, 463.5, 477.9, 499.5, 560.1,
                        560.7, 561.8, 640.4, 645.9, 699., 717.8, 723.3])[rank]
    # SSCC
    if Bname == 'SSCC':
        mode = np.array([56.6, 96.4, 154.3, 163.1, 191.8, 255.3, 255.8, 308., 344.4,
                        345.9, 373., 406.6, 461.8, 494.6, 515.4, 517.2, 553., 603.1,
                        608.7, 614.1, 682.3, 700.4, 748.4, 769.4, 781.9])[rank]
    # SCSC
    if Bname == 'SCSC':
        mode = np.array([71., 99.5, 155.4, 183.7, 213.3, 239., 266.8, 346.5, 349.1,
                        350.7, 381.3, 434.8, 452.9, 485., 512.9, 571.9, 585.8, 603.,
                        617., 646.3, 656.9, 734.4, 744.8, 747.2, 832.6])[rank]
    # SSCS
    if Bname == 'SSCS':
        mode = np.array([44.2, 88.1, 126.8, 157.1, 168.5, 236.5, 250.9, 267.1, 307.,
                        330., 369.3, 373.3, 448.6, 464.4, 465.5, 503.1, 512.3, 567.8,
                        583.4, 591.8, 658.5, 679.7, 718.5, 726.2, 756.5])[rank]
    # CCCC
    if Bname == 'CCCC':
        mode = np.array([76.6, 118.2, 186.9, 188.4, 225.2, 285.1, 290.8, 353., 383.4,
                        389.8, 407.1, 452.8, 502.5, 542., 554., 573.6, 609.4, 647.,
                        658., 670.9, 725., 757.8, 799.7, 816.6, 848.2])[rank]
    return mode
```

### 9.4 Other functions

```
def fillcelltype(mat): # Solver : fill a 2D matrix according to the cell type
    mat[0,:] = 1 # cell type boundary
    mat[-1,:] = 1
    mat[:,0] = 1
    mat[:,-1] = 1
    mat[1,2:-2] = 2 # North, cells before boundary
    mat[-2,2:-2] = 4 #South, cells before boundary
    mat[2:-2,1] = 3 # West, cells before boundary
    mat[2:-2,-2] = 5 # East, cells before boundary
    mat[1,1] = 32 # cell type inner corner
    mat[1,-2] = 25
    mat[-2,1] = 34
    mat[-2,-2] = 45
    return mat

def index(kk,jj,n): # Solver : from 2D plate index to 1D system matrix index
    ii = kk + n*jj
    return ii

def SSSS(aa, bb, BB, mumu, mm, nn): # not called in the script
    # return the mode frequency of
    # a plate size aa in x by bb in y, material stiffness BB, surface density mumu
    # for the mode mm, nn
    # under the boundary conditions ['SS', 'SS', 'SS', 'SS'] simply supported plate
    k1 = np.pi/2/aa**2*(BB/mumu)**.5
```

```

k2 = (aa/bb)**2
ff = k1*(mm**2 + k2*nn**2)
return ff

def CCCC(aa, bb, BB, mumu, mm, nn): # not called in the script
    # return the mode frequency of
    # a plate size aa in x by bb in y, material stiffness BB, surface density mumu
    # for the mode mm, nn
    # under the boundary conditions ['C', 'C', 'C', 'C'] clamped plate
    k1 = np.pi/2*(BB/mumu)**.5
    dm = 1/((nn*aa/mm/bb)**2 + 2)
    dn = 1/((mm*bb/nn/nn)**2 + 2)
    ff = k1*((mm+dm)/aa)**2 + ((nn+dn)/bb)**2
    return ff

def bound(B): # return the mark style of the edge points according to the BC for plotting
    bstyle = ['g*', 'g*', 'g*', 'g*'] # by default all the edges are simply supported (green)
    for ii in range(0, 4):
        if B[ii]=='C': # clamped edge (red)
            bstyle[ii] = 'rX'
        if B[ii]=='F': # free edge (blue)
            bstyle[ii] = 'b.'
    return bstyle

def boundname(B):
    Bname = B[0] + B[1] + B[2] + B[3]
    return Bname

def boundcoeff(B): # return the value of the stencil AxD according to the BC
    bcoeff = np.ones([4]) # by default all the edges are clamped
    # bcoeff = [-1, -1, -1, -1] # by default all the edges are simply supported
    for ii in range(0, 4):
        if B[ii]=='S': # simply supported
            bcoeff[ii] = -1
        if B[ii]=='F': # free edge
            bcoeff[ii] = 0
    return bcoeff

def plot_plate(n1, n2, bstyle, btext, file2save): # plot the plate
    m = 0.2
    fig = plt.figure(figsize=(15, 8)) # create the figure
    plt.plot([-m, -m, n1+m-1, n1+m-1, -m], [-m, n2+m-1, n2+m-1, -m, -m]) # plate limit
    for i in range(1, n1-1):
        plt.plot(i, 0, bstyle[2])
        plt.plot(i, n2-1, bstyle[0])
    for j in range(1, n2-1):
        plt.plot(0, j, bstyle[1])
        plt.plot(n1-1, j, bstyle[3])
    for i in range(2, n1-2): # inner cells
        for j in range(2, n2-2):
            plt.plot(i, j, '+', color='y')
    for i in range(1, n1-1):
        plt.plot(i, 1, '.', color='r')
        plt.plot(i, n2-2, '.', color='r')
    for j in range(1, n2-1):
        plt.plot(1, j, '.', color='r')
        plt.plot(n1-2, j, '.', color='r')
    plt.text(0.5*(n1-1), n2, btext[0], ha='center', va='bottom', size='large')
    plt.text(-1, 0.5*(n2-1), btext[1], rotation='vertical', va='center', ha='right', size='large')
    plt.text(0.5*(n1-1), -1, btext[2], ha='center', va='top', size='large')
    plt.text(n1, 0.5*(n2-1), btext[3], rotation='vertical', va='center', ha='left', size='large')

```

```

# legend
lstep = n2//10 + 1
plt.plot(n1+1, lstep, 'rX'); plt.text(n1+1.5, lstep, "clamped", va = 'center')
plt.plot(n1+1, 1.5*lstep, 'b. '); plt.text(n1+1.5, 1.5*lstep, "free", va = 'center')
plt.plot(n1+1, 2*lstep, 'g*'); plt.text(n1+1.5, 2*lstep, "simply supported", va = 'center')
plt.plot(n1+1, 2.5*lstep, 'y+'); plt.text(n1+1.5, 2.5*lstep, "inner cell", va = 'center')
plt.plot(n1+1, 3*lstep, 'r. '); plt.text(n1+1.5, 3*lstep, "BC neighbor", va = 'center')
plt.axis('equal')
plt.axis('off')
fig.savefig(file2save, bbox_inches="tight", dpi = 200) # this is the key line to store the matplotl
# plt.title('DML FDM : the plat', fontsize=18) # title
plt.show()

def memory_usage():
    current_process = psutil.Process()
    memory = current_process.memory_info().rss
    amemory = str(int(memory / (1024 * 1024)))+ "MB"
    return amemory

def freemem():
    stats = psutil.virtual_memory() # returns a named tuple
    available = int(getattr(stats, 'available')/1024/1024)
    total = int(getattr(stats, 'total')/1024/1024)
    afreemem = str(available) + "MB free on total " + str(total) + "MB"
    return afreemem

def signchange(a3):
    s3= np.sign(a3)
    #print(s3)
    #s3[s3==0] = -1 # replace zeros with -1
    zero_crossings3 = np.diff(s3)
    zero_crossings3 = np.abs(zero_crossings3)
    #print(zero_crossings3)
    count = np.count_nonzero(zero_crossings3 == 2)+1
    #print(count)
    return count

```

## 9.5 Prepare py2pdf

```

# lines to be included for py2pdf export
scriptname = os.path.basename(__file__).split('.')[0] # get this script file name without extension
py2pdfdir = p2p.newoututdir(scriptname)
logfile = py2pdfdir + "/DML_FDM3b.txt" # define the logfile
p2p.clearlog(logfile) # clear the logfile (in case script is ran several times)

print(memory_usage(), "Mb used after import")
print(freemem())

```

## 9.6 Parameters

```

# Dimensions (in m otherwise mentionned)
Lxmm = 400; Lymm= 600 # use locally mm to avoid rounding problems
h = 0.015 # plate thickness

# Boundary conditions C = clamped, SS = simply supported, F = free (F not available for now)
Direction = ['north', 'west', 'south', 'east']
Bound_case = [['C', 'C', 'C', 'C'],
               ['S', 'S', 'S', 'S'],
               ['S', 'S', 'C', 'C'],
               ['S', 'C', 'S', 'C'],
               ['S', 'S', 'C', 'S']]

```

```

# Material
# do not change the values (used in Elmer for comparison)
Ex = 10e6; Ey = 10e6 # Young modulus x and y direction in Pa
rho = 25. # density in kg/m^3
nu = 0.3 # Poisson ration

B = Ex*h**3/12/(1-nu**2) # bending stiffness
mu = rho*h # areal density
# Mesh
dxmm = 10; dymm = dxmm
Nx = Lxmm//dxmm + 1; Ny = Lymm//dymm + 1 # number of points in the grid
Lx = Lxmm/1000; Ly = Lymm/1000
dx = dxmm/1000; dy= dymm/1000
# Expected results
m = 5; n = m # m by n modes are displayed
case2print = 4 # plot case2print

# Summary
p2p.print_twice(logfile, "The plate is Lx =", Lx,"m by Ly =", Ly, "m")
p2p.print_twice(logfile, "The plate thickness is h =", h,"m")
p2p.print_twice(logfile, "The mesh is Nx =", Nx,"cells by Ny =", Ny, "cells")
p2p.print_twice(logfile, "with a grid dx =", dxmm,"mm by dy =", dymm, " mm")
p2p.print_twice(logfile, "Young modulus E =", int(Ex/1e6), "MPa, Density rho =", rho, "kg/m^3")
p2p.print_twice(logfile, "Bending stiffness B =", int(B*100)/100, "Nm, Areal density μ =", mu, "kg/m^2")

```

## 9.7 System matrix filling process

```

tic = time.perf_counter() # optional, start of a timer for time performance check
# Fill the cell type matrix (inner, boundary...)
celltype = np.zeros([Ny, Nx])
celltype = fillcelltype(celltype)

# Fill the system matrix components
Jxrange = range(0,Nx); Kyrange = range(0, Ny)
Imat = np.zeros([Ny,Nx]) # for test
Ax = np.zeros([Ny*Nx,Ny*Nx]) # x direction
AxW = np.zeros([Ny*Nx,Ny*Nx]) # x direction for clamped or simply supported
AxS = np.zeros([Ny*Nx,Ny*Nx]) # x direction for clamped or simply supported
Axy = np.zeros([Ny*Nx,Ny*Nx]) # xy direction
Ay = np.zeros([Ny*Nx,Ny*Nx]) # y direction
AyN = np.zeros([Ny*Nx,Ny*Nx]) # y direction for clamped or simply supported
AyS = np.zeros([Ny*Nx,Ny*Nx]) # y direction for clamped or simply supported
BM = np.zeros([Ny*Nx,Ny*Nx]) # Boundaries
SysMat = np.zeros([Ny*Nx,Ny*Nx]) # System Matrix

for j in Jxrange:
    for k in Kyrange:
        i = k + Ny*j
        Imat[k,j]=i
        cell = celltype[k,j]
        if cell!=1:
            Axy[index(k,j,Ny),index(k,j,Ny)] = 4
            Axy[index(k,j,Ny),index(k-1,j-1,Ny)] = 1
            Axy[index(k,j,Ny),index(k+1,j+1,Ny)] = 1
            Axy[index(k,j,Ny),index(k-1,j+1,Ny)] = 1
            Axy[index(k,j,Ny),index(k+1,j-1,Ny)] = 1
            Axy[index(k,j,Ny),index(k+1,j,Ny)] = -2
            Axy[index(k,j,Ny),index(k-1,j,Ny)] = -2
            Axy[index(k,j,Ny),index(k,j+1,Ny)] = -2
            Axy[index(k,j,Ny),index(k,j-1,Ny)] = -2

```

```

if cell==0:
    Ax[index(k,j,Ny),index(k,j,Ny)] = 6
    Ax[index(k,j,Ny),index(k,j-1,Ny)] = -4
    Ax[index(k,j,Ny),index(k,j+1,Ny)] = -4
    Ax[index(k,j,Ny),index(k,j-2,Ny)] = 1
    Ax[index(k,j,Ny),index(k,j+2,Ny)] = 1
    Ay[index(k,j,Ny),index(k,j,Ny)] = 6
    Ay[index(k,j,Ny),index(k-1,j,Ny)] = -4
    Ay[index(k,j,Ny),index(k+1,j,Ny)] = -4
    Ay[index(k,j,Ny),index(k-2,j,Ny)] = 1
    Ay[index(k,j,Ny),index(k+2,j,Ny)] = 1
if cell==2 or cell==4:
    Ax[index(k,j,Ny),index(k,j,Ny)] = 6
    Ax[index(k,j,Ny),index(k,j-1,Ny)] = -4
    Ax[index(k,j,Ny),index(k,j+1,Ny)] = -4
    Ax[index(k,j,Ny),index(k,j-2,Ny)] = 1
    Ax[index(k,j,Ny),index(k,j+2,Ny)] = 1
if cell==3 or cell==5:
    Ay[index(k,j,Ny),index(k,j,Ny)] = 6
    Ay[index(k,j,Ny),index(k-1,j,Ny)] = -4
    Ay[index(k,j,Ny),index(k+1,j,Ny)] = -4
    Ay[index(k,j,Ny),index(k-2,j,Ny)] = 1
    Ay[index(k,j,Ny),index(k+2,j,Ny)] = 1
if cell==3 or cell==32 or cell==34: #West
    Ax[index(k,j,Ny),index(k,j,Ny)] = 6
    Ax[index(k,j,Ny),index(k,j-1,Ny)] = -4
    Ax[index(k,j,Ny),index(k,j+1,Ny)] = -4
    AxW[index(k,j,Ny),index(k,j,Ny)] = 1
    Ax[index(k,j,Ny),index(k,j+2,Ny)] = 1
if cell==5 or cell==25 or cell==45: #East
    Ax[index(k,j,Ny),index(k,j,Ny)] = 6
    Ax[index(k,j,Ny),index(k,j-1,Ny)] = -4
    Ax[index(k,j,Ny),index(k,j+1,Ny)] = -4
    Ax[index(k,j,Ny),index(k,j-2,Ny)] = 1
    AxE[index(k,j,Ny),index(k,j,Ny)] = 1
if cell==2 or cell==32 or cell==25: #North
    Ay[index(k,j,Ny),index(k,j,Ny)] = 6
    Ay[index(k,j,Ny),index(k-1,j,Ny)] = -4
    Ay[index(k,j,Ny),index(k+1,j,Ny)] = -4
    AyN[index(k,j,Ny),index(k,j,Ny)] = 1
    Ay[index(k,j,Ny),index(k+2,j,Ny)] = 1
if cell==4 or cell==34 or cell==45: #South
    Ay[index(k,j,Ny),index(k,j,Ny)] = 6
    Ay[index(k,j,Ny),index(k-1,j,Ny)] = -4
    Ay[index(k,j,Ny),index(k+1,j,Ny)] = -4
    AyS[index(k,j,Ny),index(k,j,Ny)] = 1
    Ay[index(k,j,Ny),index(k-2,j,Ny)] = 1
if cell==1:
    BM[index(k,j,Ny),index(k,j,Ny)] = 1

```

## 9.8 Solver

```

# Case = 4
for Case in range(5):
    Bound_cond = Bound_case[Case]
    BC = boundcoeff(Bound_cond)
    BCname = boundname(Bound_cond)
    plotboundcond = ['', '', '', '']
    for i in range(4):
        plotboundcond[i] = Bound_cond[i] + " [" + Direction[i] + "]"

```

```

p2p.print_twice(logfile, "Case", Case)
p2p.print_twice(logfile, "Boundary conditions", plotboundcond)
p2p.print_twice(logfile, "Boundary coeff. North=", BC[0], " West=", BC[1], " South=", BC[2], " East=")
# Show the plate
boundline = bound(Bound_cond) # set the line style for plotting
if Case==case2print:
    plot_plate(Nx, Ny, boundline, Direction, py2pdfdir + "/plate.png") # show the mesh

# Here is the solver
# Compute the system matrix according to the boundary conditions
SysMat = Ax + BC[0]*AyN + BC[1]*AxW + BC[2]*AyS + BC[3]*AxE + Ay + 2*Axy
SysMat = (B*SysMat)/mu/dx**4 + BM

# Compute the eigenfrequencies and the mode shapes
vals,vecs = linalg.eig(SysMat)

toc = time.perf_counter() # to evaluate time performance of the algorithm
# show some performance info
print(memory_usage(), "used")
print(freemem())
print(f"Excecution time = {toc - tic:0.4f} seconds for ", Nx*Ny," cells")

# Clean the results by removing 1 values and complex from the eigenvalues
vals = vals[vals != 1.+0j] # remove the value 1.+0j
vals = np.real(vals)
modefreq = np.zeros([len(vals), 2]) # prepare an array for the eigenfrequencies
modefreq[:,1] = vals**0.5/2/np.pi # extract the frequencies
modefreq[:,0] = np.arange(0, len(vals), 1) # give an index
modefreq = modefreq[modefreq[:,1].argsort()] # sort
modefreq = modefreq.astype(int) # keep interger format to reduce the number of digit
...

## Script output printing
```python
results = np.zeros([m*n,8]) # prepare array result
results[:,3] = modefreq[:,1] # eigenfrequency in col 3
results[:,7] = modefreq[:,0] # mode shape number in col 7

for i in range(m*n):
    #results[i,6] = np.mean(vecs[:, int(results[i,7])]) # mode shape mean value
    results[i,2] = elmer(BCname, i, m*n-1) # elmer result in col 2

drivingpoint = np.zeros([Ny, Nx])
if Case == case2print:
    fig = plt.figure(figsize=(15, 8)) # create the figure

for i in range(m*n):
    modeshape = np.real(np.transpose(vecs[:, int(results[i,7])].reshape((Nx,Ny))))
    # modeshape is the 2D matrix showing the plate deviation in the plate coordinates
    ind = np.unravel_index(np.argmax(np.absolute(modeshape), axis=None), modeshape.shape)
    # ind returns the coordinates of the maximum sees in modeshape
    diry = modeshape[:,ind[1]]
    # diry is the 1D slice of modeshape in y direction at the maximum
    dirx = modeshape[ind[0],:]
    # dirx is the 1D slice of modeshape in x direction at the maximum
    # the mode numbers are obtains by checking how lany times dirx, diry signs change
    mx = signchange(dirx)
    ny = signchange(diry)
    results[i,0] = mx
    results[i,1] = ny
    results[i,5] = modes(Bound_cond, Lx, Ly, B, mu, mx, ny) # formula result in 5
    if results[i,7] != 0: # test of driving point similar to Zenker's paper

```

```

mode = np.absolute(modeshape)
drivingpoint = drivingpoint + (mode/np.max(mode))*0.3
if i > 0:
    drivingpoint = drivingpoint/np.max(drivingpoint)
if Case == case2print:
    plt.subplot(m,n,i+1)
    plt.imshow(modeshape)
    plt.plot(ind[1],ind[0], 'r*')
    # plt.title(str(int(results[i,0]))+" "+str(int(results[i,1])))
    plt.title(str(mx)+" "+str(ny))
    plt.axis('equal')
    plt.axis('off')

# results[:,6] = 100*results[:,6]/np.max(abs(results[:,6]))
# results[:,6] = results[:,6].astype(int)
error = 1000*(results[:,3].astype(int)/results[:,2].astype(int)-1)
results[:,4] = error.astype(int)/10
mean_error = int(100*np.average(results[:,4]))/100
results[:,4] = results[:,4].astype(int)

error = 1000*(results[:,5].astype(int)/results[:,2].astype(int)-1)
results[:,6] = error.astype(int)/10
results[:,6] = results[:,6].astype(int)

#print(results)
spacing = 10
legend = ["m", "n", "elmer", "FDM", "err%", "formula", "err%"]
aa = ""
for j in range(7):
    bb = legend[j]
    bb = (spacing - len(bb)-1)*" " + bb + " "
    aa = aa + bb
p2p.print_twice(logfile, aa)
for i in range(m*n):
    aa=""
    for j in range(7):
        bb = str(int(results[i, j]))
        bb = (spacing - len(bb)-1)*" " + bb + " "
        aa = aa + bb
    p2p.print_twice(logfile, aa)
p2p.print_twice(logfile, "Mean error", mean_error, "%")

if Case == case2print:
    plt.suptitle("Mode shapes " + Bound_cond[0] + Bound_cond[1] + Bound_cond[2] + Bound_cond[3])
    plt.show()
    fig.savefig(py2pdfdir + "/modeshape.png", bbox_inches="tight", dpi = 200) # this is the key line

'''
ind = np.unravel_index(np.argmax(np.absolute(drivingpoint)), axis=None, modeshape.shape)
plt.imshow(drivingpoint)
plt.plot(ind[1],ind[0], 'r*')
plt.show()
'''

```

## 9.9 py2pdf

```

print("start py2pdf")
cmd = "py2pdf " + scriptname # alias
subprocess.call(['/bin/bash', '-i', '-c', cmd]) # to launch the bash file (alias)

```



## References

- [1] R. Maso, “Application of boundary edge effect corrections to the vibration of beams and rectangular plates.” PhD thesis, Carleton University, 1993. Available: [https://curve.carleton.ca/system/files/etd/10817845-5151-47ac-9066-6fc2e360a6f2/etd\\_pdf/832572c0d80b3466b8f401a713c81266/maso-applicationofboundaryedgeeffectcorrections.pdf](https://curve.carleton.ca/system/files/etd/10817845-5151-47ac-9066-6fc2e360a6f2/etd_pdf/832572c0d80b3466b8f401a713c81266/maso-applicationofboundaryedgeeffectcorrections.pdf)
- [2] A. W. Leissa, *Vibration of plates*, vol. 160. Scientific and Technical Information Division, National Aeronautics and ..., 1969. Available: <https://ntrs.nasa.gov/api/citations/19700009156/downloads/19700009156.pdf>