#### 1

# Control Systems

#### G V V Sharma\*

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Mason's Gain Formula

**Bode Plot** 

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2 Bode Plot

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8.1. Using Nichol's chart, find out whether each of the system below are stable or not

7 Compensators 8 Nichol's chart

$$G_1(s) = \frac{50}{s(s+3)(s+6)}$$
 (8.1.1)

$$H_1(s) = 1$$
 (8.1.2)

$$G_2(s) = \frac{9}{s^2(s+3)} \tag{8.1.3}$$

$$H_2(s) = (s+4)$$
 (8.1.4)

$$G_3(s) = \frac{20}{s(s+1)} \tag{8.1.5}$$

$$H_3(s) = \frac{s+3}{s+4} \tag{8.1.6}$$

$$G_4(s) = \frac{100(s+5)}{s(s^2+4)(s+3)}$$
(8.1.7)

$$H_4(s) = 1$$
 (8.1.8)

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3 SECOND ORDER SYSTEM
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Using Nichol's chart, find out the range of K for which the closed loop systems will be stable

$$G_5(s) = \frac{K}{s(s+6)} \tag{8.1.9}$$

$$H_5(s) = \frac{1}{s+9} \tag{8.1.10}$$

$$G_6(s) = \frac{K(s+2)(s+4)}{s^2 - 3s + 10}$$
 (8.1.11)

$$H_6(s) = \frac{1}{s} \tag{8.1.12}$$

$$G_7(s) = \frac{K}{(s+1)(s+3)}$$
 (8.1.13)

$$H_7(s) = \frac{s+5}{s+7} \tag{8.1.14}$$

For unity feedback (negative) systems given below, obtain closed loop frequency response using constant M and N circles.

$$G_8(s) = \frac{10}{s(s+1)(s+2)} \quad (8.1.15)$$

$$G_8(s) = \frac{10}{s(s+1)(s+2)}$$
 (8.1.15)  
$$G_9(s) = \frac{1000}{(s+3)(s+4)(s+5)(s+6)}$$
 (8.1.16)

$$G_0(s) = \frac{50(s+3)}{s(s+2)(s+4)} \quad (8.1.17)$$

For the above systems also estimate the percentage overshoot that can be expected when a step input is given to the system.

#### **Solution:**

Nichol's chart is the plot of gain and phase that is the magnitude(in dB) on the vertical axis and phase(in deg) on the horizontal axis for the given transfer function. It is called the gain phase plot. These plots are used for the stability analysis of the system.

There are four cases for finding the stability (r is magnitude in dB and  $\phi$  is phase in deg)

Case 1 : System with no unstable pole.

For the stable function T(s) whose Nichols plot intersects dB line at least one time. The system is stable if and only if one of the following holds

- 1) the steady gain | ko | is less than 0 dB
- 2) The Nichols plot  $T(j\omega)$  intersects the line segment C :=  $[(\phi,r) : r = 0 \text{ dB}, -180^{\circ} < \phi]$ <180° | for the function with the positive

steady gain ko and | ko | larger than dB.

Case 2: System with one unstable pole.

The system is stable if and only if the half part of the Nichols plot crosses the line segment C  $:= [(\phi, r) : r = 0 \text{ dB}, -180^{\circ} < \phi < 180^{\circ}], \text{ the}$ steady gain ko is negative and | ko | is larger than 0 dB.

Case 3: System with 2k unstable poles.

The feedback system is stable if and only if the plot intersects the line segment  $C := [(\phi,r)]$ r = 0 dB,  $-180^{\circ} < \phi < 180^{\circ}$ , the steady gain ko is positive and | ko | is larger than 0 dB

Case 4: System with 2k+1 unstable poles. The stability analysis of the system with 2k+1unstable poles is equivalent to that of the shifting Nichols plot of the function with one unstable pole.

Also, for the percentage overshoot(PO) we need the characteristic equation of the function so as to determine the damping ratio. The general characteristic equations is

$$s^2 + 2\zeta\omega s + \omega^2 \tag{8.1.18}$$

where  $\zeta$  is the damping ratio and  $\omega$  is the natural frequency.

Since all the systems above are third order or fourth order systems we need to decompose them into first order and second order to find out the PO.

$$PO = \exp \frac{-\zeta \pi}{\sqrt{1-\zeta^2}} * 100$$
 (8.1.19)

- 8.2. Solve for stability
- 8.3. **Solution:** From (8.1.1) and (8.1.2),

$$G_1(s)H_1(s) = \frac{50}{s(s+3)(s+6)}$$
 (8.3.1)

codes/es17btech11009 1.py

The above code gives the following plot

The given system has no unstable poles so according to case 1, we could conclude that the plot satisfies the stability condition therefore, the system is stable.

The characteristic equation of the function

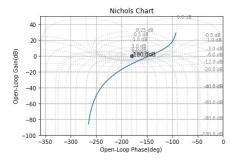


Fig. 8.3

(8.3.1) is,

$$s^2 + 9s + 18 = 0 (8.3.2)$$

From (8.1.18) and (8.1.19),

 $\zeta = 1.06$  and  $\omega = 4.24$ 

Since  $\zeta = 1.06 \approx 1$ . The system has no overshoots.

Percentage Overshoot = 0%

#### 8.4. From (8.1.3) and (8.1.4),

$$G_2(s)H_2(s) = \frac{9(s+4)}{s^2(s+3)}$$
 (8.4.1)

codes/es17btech11009 2.py

The above code gives the following plot

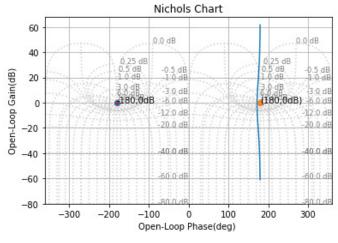


Fig. 8.4

The given system has no unstable poles so according to case 1, we could conclude that the plot satisfies the stability condition.

The characteristic equation of the function (8.4.1),

$$s^2(s+3) = 0 (8.4.2)$$

From (8.1.18) and (8.1.19),  $\zeta$  and  $\omega$  are not defined. Percentage overshoot cannot be estimated.

8.5. From (8.1.5) and (8.1.6),

$$G_3(s)H_3(s) = \frac{20(s+3)}{s(s+1)(s+4)}$$
 (8.5.1)

codes/es17btech11009 3.py

The above code gives the following plot

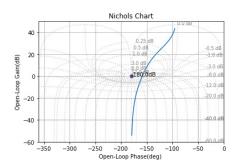


Fig. 8.5

The given system has no unstable poles so according to case 1, we could conclude that the plot satisfies the stability condition.

The characteristic equation of the function (8.5.1) is,

$$s^2 + 5s + 4 = 0 (8.5.2)$$

From (8.1.18) and (8.1.19),

 $\zeta = 1.25$  and  $\omega = 2$ 

Since  $\zeta = 1.25 \approx 1$ . The system has no overshoots.

Percentage overshoot = 0 %

#### 8.6. From (8.1.7) and (8.1.8),

$$G_4(s) H_4(s) = \frac{100(s+5)}{s(s^2+4)(s+3)}$$
 (8.6.1)

codes/es17btech11009 4.py

The above code gives the following plot The given system has no unstable poles so according to case 1, we could conclude that the plot satisfies the stability condition.

The characteristic equation of the function (8.6.1) is,

$$(s^2 + 3s)(s^2 + 4) = 0 (8.6.2)$$

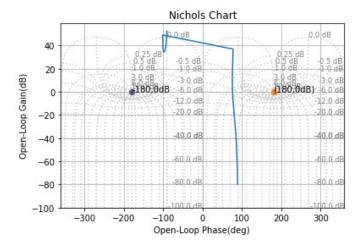


Fig. 8.6

From (8.1.18) and (8.1.19),  $\zeta = 0.75$  and  $\omega = 2$ Percentage Overshoot = 100 \*0.028 = 2.8 %

#### 8.7. Solve for range of K

#### 8.8. **Solution:**

From (8.1.9) and (8.1.10),

$$G_5(s)H_5(s) = \frac{k}{s(s+6)(s+9)}$$
 (8.8.1)

Clearly, The above system does not have unstable poles so we have to consider Case 1,

The above code gives the following plot for k = 10

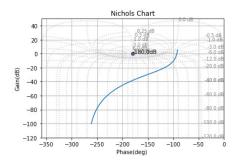


Fig. 8.8

By increasing k, we could see that the plot shifts in upward direction, so the range of k for which the system is stable is (5,650). In this range, the plot satisfies the conditions mentioned in Case 1 for stability.

The characteristic equation of the function (8.8.1) is,

$$s^2 + 15s + 54 = 0 (8.8.2)$$

From (8.1.18) and (8.1.19),

 $\zeta = 1.02 \text{ and } \omega = 7.348$ 

Since  $\zeta = 1.02 \approx 1$ . The system has no overshoots.

Percentage overshoot = 0%

8.9. From (8.1.11) and (8.1.12),

$$G_6(s) H_6(s) = \frac{K(s+2)(s+4)}{s(s^2 - 3s + 10)}$$
 (8.9.1)

The above system has 2 unstable poles which are (1.5 + 2.75j) and (1.5 - 2.75j) respectively, so it falls in Case 3,

The above code gives the following plot for k = 2

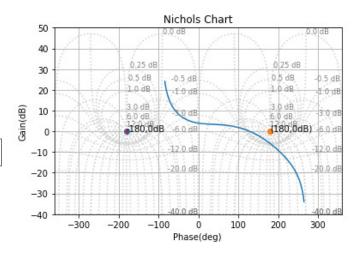


Fig. 8.9

By increasing k, we could see that the plot shifts in upward direction, so the range of k for which the system is stable is (0.125,4). In this range, the plot satisfies the conditions mentioned in Case 3 for stability.

The characteristic equation of the function (8.9.1) is,

$$s^2 - 3s + 10 = 0 (8.9.2)$$

From (8.1.18) and (8.1.19),

 $\zeta = 0.47 \text{ and } \omega = 3.16$ 

Percentage overshoot = 0.18 \* 100 = 18%

8.10. From (8.1.13) and (8.1.14),

$$G_7(s) H_7(s) = \frac{k}{(s+1)(s+3)(s+5)(s+7)}$$
(8.10.1)

Clearly, The above system does not have unstable poles so we have to consider Case 1,

The above code gives the following plot for k = 200

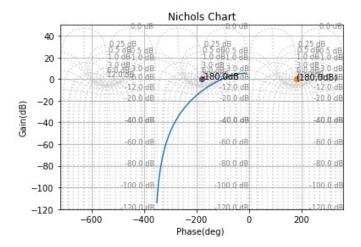


Fig. 8.10

By increasing k, we could see that the plot shifts in upward direction, so the range of k for which the system is stable is (100,600).

The characteristic equation of the function (8.10.1)is,

$$(s^2 + 4s + 3)(s^2 + 12s + 35) = 0$$
 (8.10.2)

Case 1:

$$(s^2 + 4s + 3) = 0 (8.10.3)$$

Case 2:

$$(s^2 + 12s + 35) = 0 (8.10.4)$$

From Case 1 and Case 2 and referring to (8.1.18) and (8.1.19),

 $\zeta \approx 1$  .Therefore the system has no overshoots. Percentage Overshoot = 0%

## 8.11. Closed loop frequency response

Constant M and N circles:

**Solution:** M circles are called constant magnitude Loci and N circles are called as

constant phase angle Loci. These are helpful in determining the closed-loop frequency response of unity negative feedback systems.

**Mcircle(Constant-Magnitude Loci):** Let  $G(j\omega)$  be complex quantity it can be written as

$$G(j\omega) = x + jy \tag{8.11.1}$$

where x,y are real quantities. Let M be magnitude of closed loop transfer function.

$$M = \left| \frac{x + jY}{1 + x + jy} \right|$$
 (8.11.2)

$$M^2 = \frac{x^2 + y^2}{(1+x)^2 + y^2}$$
 (8.11.3)

Hence,

$$X^{2}(1-M^{2}) - 2M^{2}X - M^{2} + (1-M^{2})Y^{2} = 0$$
(8.11.4)

If M = 1, then from Equation (8.11.3), we obtain  $x = \frac{-1}{2}$  This is the equation of a straight line parallel to the Y axis and passing through the point  $\left(\frac{-1}{2}, 0\right)$ .

If  $M \neq 1$  Equation (8.11.4) can be written as

$$x^{2} + \frac{2M^{2}}{M^{2} - 1}x + \frac{M^{2}}{M^{2} - 1} + y^{2} = 0$$
 (8.11.5)

Simplifying,

$$\left(x + \frac{M^2}{M^2 - 1}\right)^2 + y^2 = \frac{M^2}{\left(M^2 - 1\right)^2}$$
 (8.11.6)

Equation (8.11.6) is the equation of a circle with center  $\left(-\frac{M^2}{M^2-1},0\right)$  and radius  $\left|\frac{M}{M^2-1}\right|$  Thus the intersection of Nquist plot with M

circle at a frequency( $\omega$ ) results as the magnitude of closed loop transfer function as M at frequency ( $\omega$ )

N Circles(Constant-Phase-Angle Loci): Finding Phase angle  $\alpha$  from (8.11.3) we get,

$$\alpha = \tan^{-1}\left(\frac{y}{x}\right) - \tan^{-1}\left(\frac{y}{1+x}\right) \quad (8.11.7)$$
Let  $\tan \alpha = N$  (8.11.8)
$$N = \tan\left(\tan^{-1}\left(\frac{y}{x}\right) - \tan^{-1}\left(\frac{y}{1+x}\right)\right) \quad (8.11.9)$$

Simplifying,

$$N = \frac{y}{x^2 + x + y^2} \tag{8.11.10}$$

Further Simplifying..

$$\left(x + \frac{1}{2}\right)^2 + \left(y - \frac{1}{2N}\right)^2 = \frac{1}{4} + \frac{1}{(2N)^2}$$
 (8.11.11)

Equation (8.11.11) is the equation of a circle with center at  $\left(\frac{-1}{2}, \frac{1}{2N}\right)$  and radius  $\sqrt{\frac{1}{4} + \frac{1}{(2N)^2}}$ Thus the intersection of Nquist plot with N circle at a frequency( $\omega$ ) results as the phase of closed loop transfer function as  $tan^{-1}(N)$ at frequency  $(\omega)$ 

#### 8.12. From (8.1.15),

$$G_8(s) = \frac{10}{s(s+1)(s+2)}$$
 (8.12.1)

**Solution:** The following code gives the nichol plot of (8.12.1)

codes/es17btech11009 8.py

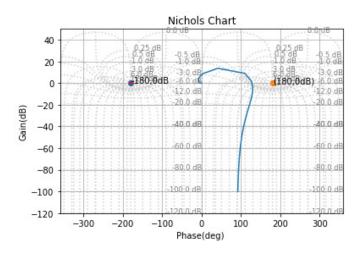
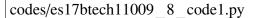


Fig. 8.12

The M and N circles of  $T(1\omega)$  in the gain phase plane are transformed into M and N contours 8.15. The points N and frequencies are listed in in rectangular co-ordinates. A point on the constant M loci in  $T(1\omega)$  plane is transferred to gain phase plane by drawing the vector directed from the origin of  $T(1\omega)$  plane to a particular point on M circle and then measuring the length in dB and angle in degree.

8.13. The M and N contours in rectangle coordinates look like the following. The following code plots Fig. ??



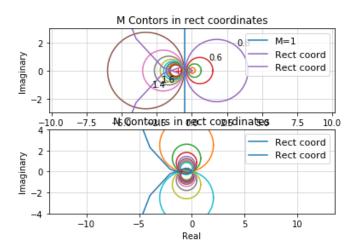


Fig. 8.13

8.14. To find the intersection of M and N contours in the rectangle co-ordinates at different frequencies.

> **Solution:** The following code finds intersection of M and N contours with rectangular coordinates at different frequencies

The points M and frequencies are listed in **Table 8.14** 

M in dB	M	ω
13.64	4.81	1.68
11.84	3.91	1.85
7.64	2.41	1.98
5.15	1.81	2.07
-4.29	0.61	2.61
-40	0.01	8.71

**TABLE 8.14** 

Table8.15

The constant N locus for given value of  $\alpha$ is not the entire circle but only an arc. This is beacuse tangent of angle remains same if  $+180^{\circ}$  or  $-180^{\circ}$  is added to the angle.

The characteristic equation of the function (8.12.1) is,

$$s^2 + 3s + 2 = 0 (8.15.1)$$

α	N	ω
-78.69	-5	10.351
-77.10	-4.4	11.789
-75.2	-3.8	14.027
-66.5	-2.3	27.06
-63.4	-2.0	6.115
5.7	0.1	27.066

**TABLE 8.15** 

From (8.1.18) and (8.1.19),  $\zeta = 1.06 \approx 1$  and  $\omega = 1.414$ The system has no overshoots. Percentage Overshoot = 0%

8.16. From (8.1.16),

$$G_9(s) = \frac{1000}{(s+3)(s+4)(s+5)(s+6)}$$
(8.16.1)

**Solution:** The following code gives the nichols plot

codes/es17btech11009\_9.py

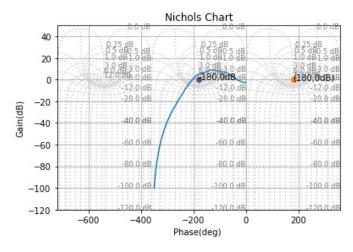


Fig. 8.16

8.17. The M and N contours in rectangle coordinates look like the following. The following code plots Fig. 8.17

8.18. The following code finds intersection of M and N contours with rectangular co-ordinates at different frequencies

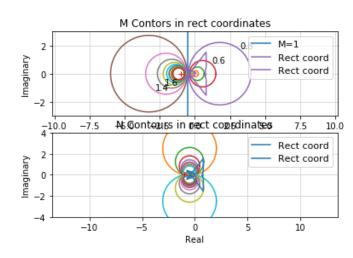


Fig. 8.17

codes/es17btech11009 9 code2.py

The points M and frequencies are listed in Table 8.18

M in dB	M	ω
9.57	3.01	4.67
5.15	1.81	5.01
-4.29	0.61	5.83
-10.17	0.31	6.7
-40	0.01	15.14

**TABLE 8.18** 

8.19. The points N and frequencies are listed in Table 8.19

α	N	ω
-78.69	-5.0	7.76
-77.1	-4.4	7.553
-74.05	-3.5	6.944
5.71	0.1	36.128
34.99	0.7	22.57

**TABLE 8.19** 

The characteristic equation of the function (8.16.1) is,

$$(s^2 + 7s + 12)(s^2 + 11s + 30) = 0$$
 (8.19.1)

Case 1:

$$s^2 + 7s + 12 = 0 (8.19.2)$$

Case 2:

$$s^2 + 11s + 30 = 0 (8.19.3)$$

From Case 1 and Case 2 and referring to (8.1.18) and (8.1.19),

 $\zeta=\approx 1$  . Therefore the system has no overshoots.

Percentage Overshoot = 0%

8.20. From (8.1.17),

$$G_0(s) = \frac{50(s+3)}{s(s+2)(s+4)}$$
 (8.20.1)

**Solution:** The following code gives the nichols plot

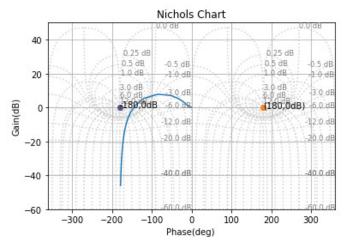


Fig. 8.20

8.21. The M and N contours in rectangle coordinates look like the following. The following code plots Fig. 8.21

8.22. The following code finds intersection of M and N contours with rectangular co-ordinates at different frequencies

The points M and frequencies are listed in Table 8.22

8.23. The points N and frequencies are listed in Table8.23

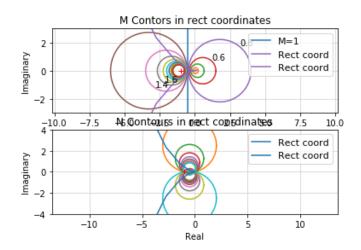


Fig. 8.21

M in dB	M	ω
7.64	2.41	6.34
6.48	2.11	7.58
-0.81	0.91	9.86
-10.17	0.31	14.33
-40	0.01	57.91

**TABLE 8.22** 

α	N	ω
-66.50	-2.30	6.40
-78.60	-5.00	6.72
-101.50	4.90	7.32
-158.19	0.4	11.68
-174.28	0.1	46.01

**TABLE 8.23** 

The characteristic equation of the function (8.20.1)is,

$$s^2 + 6s + 8 = 0 (8.23.1)$$

From (8.1.18) and (8.1.19),  $\zeta = 1.06 \approx 1$  and  $\omega = 2.828$  The system has no overshoots. Percentage Overshoot = 0%