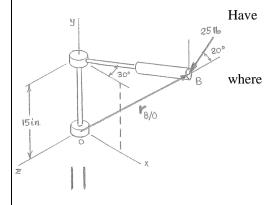


A 25-lb force acting in a vertical plane parallel to the yz plane is applied to the 8-in.-long horizontal handle AB of a socket wrench. Replace the force with an equivalent force-couple system at the origin O of the coordinate system.

SOLUTION



y

 $\Sigma \mathbf{F}$: $\mathbf{P}_B = \mathbf{F}$

$$\mathbf{P}_B = 25 \text{ lb} \Big[-(\sin 20^\circ) \mathbf{j} + (\cos 20^\circ) \mathbf{k} \Big]$$
$$= -(8.5505 \text{ lb}) \mathbf{j} + (23.492 \text{ lb}) \mathbf{k}$$

or
$$\mathbf{F} = -(8.55 \text{ lb})\mathbf{j} + (23.5 \text{ lb})\mathbf{k} \blacktriangleleft$$

Have

$$\Sigma \mathbf{M}_O$$
: $\mathbf{r}_{B/O} \times \mathbf{P}_B = \mathbf{M}_O$

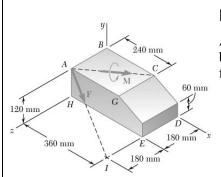
where

$$\mathbf{r}_{B/O} = \left[\left(8\cos 30^{\circ} \right) \mathbf{i} + \left(15 \right) \mathbf{j} - \left(8\sin 30^{\circ} \right) \mathbf{k} \right] \text{ in.}$$
$$= \left(6.9282 \text{ in.} \right) \mathbf{i} + \left(15 \text{ in.} \right) \mathbf{j} - \left(4 \text{ in.} \right) \mathbf{k}$$

∴
$$\begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 6.9282 & 15 & -4 \\ 0 & -8.5505 & 23.492 \end{vmatrix}$$
 lb·in. = \mathbf{M}_O

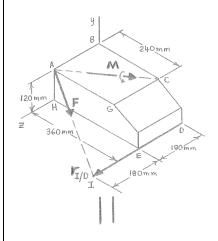
$$\mathbf{M}_O = [(318.18)\mathbf{i} - (162.757)\mathbf{j} - (59.240)\mathbf{k}]$$
lb·in.

or
$$\mathbf{M}_O = (318 \text{ lb·in.})\mathbf{i} - (162.8 \text{ lb·in.})\mathbf{j} - (59.2 \text{ lb·in.})\mathbf{k} \blacktriangleleft$$



A 315-N force **F** and 70-N·m couple **M** are applied to corner A of the block shown. Replace the given force-couple system with an equivalent force-couple system at corner D.

SOLUTION



Have

$$\Sigma \mathbf{F} : \quad \mathbf{F} = \mathbf{F}_{D}$$

$$= \lambda_{AI} F$$

$$= \frac{(0.360 \text{ m})\mathbf{i} - (0.120 \text{ m})\mathbf{j} + (0.180 \text{ m})\mathbf{k}}{0.420 \text{ m}} (315 \text{ N})$$

$$= (750 \text{ N})(0.360\mathbf{i} - 0.120\mathbf{j} + 0.180\mathbf{k})$$
or $\mathbf{F}_{D} = (270 \text{ N})\mathbf{i} - (90.0 \text{ N})\mathbf{j} + (135.0 \text{ N})\mathbf{k} \blacktriangleleft$

Have

$$\Sigma \mathbf{M}_D$$
: $\mathbf{M} + \mathbf{r}_{I/D} \times \mathbf{F} = \mathbf{M}_D$

where

$$\mathbf{M} = \lambda_{AC} M$$

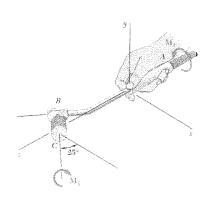
$$= \frac{(0.240 \text{ m})\mathbf{i} - (0.180 \text{ m})\mathbf{k}}{0.300 \text{ m}} (70.0 \text{ N} \cdot \text{m})$$

$$= (70.0 \text{ N} \cdot \text{m})(0.800\mathbf{i} - 0.600\mathbf{k})$$

$$\mathbf{r}_{I/D} = (0.360 \text{ m})\mathbf{k}$$

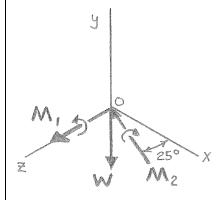
$$\therefore \mathbf{M}_{D} = (70.0 \text{ N} \cdot \text{m})(0.8\mathbf{i} - 0.6\mathbf{k}) + \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 0 & 0 & 0.36 \\ 0.36 & -0.12 & 0.18 \end{vmatrix} (750 \text{ N} \cdot \text{m})$$

$$= (56.0 \text{ N} \cdot \text{m})\mathbf{i} - (42.0 \text{ N} \cdot \text{m})\mathbf{k} + [(32.4 \text{ N} \cdot \text{m})\mathbf{i} + (97.2 \text{ N} \cdot \text{m})\mathbf{j}]$$
or $\mathbf{M}_{D} = (88.4 \text{ N} \cdot \text{m})\mathbf{i} + (97.2 \text{ N} \cdot \text{m})\mathbf{j} - (42.0 \text{ N} \cdot \text{m})\mathbf{k} \blacktriangleleft$



The handpiece of a miniature industrial grinder weighs 2.4 N, and its center of gravity is located on the y axis. The head of the handpiece is offset in the xz plane in such a way that line BC forms an angle of 25° with the x direction. Show that the weight of the handpiece and the two couples \mathbf{M}_1 and \mathbf{M}_2 can be replaced with a single equivalent force. Further assuming that $M_1 = 0.068 \,\mathrm{N\cdot m}$ and $M_2 = 0.065 \,\mathrm{N\cdot m}$, determine (a) the magnitude and the direction of the equivalent force, (b) the point where its line of action intersects the xz plane.

SOLUTION



First assume that the given force **W** and couples M_1 and M_2 act at the origin.

Now
$$\mathbf{W} = -W_{\mathbf{j}}$$

and
$$\mathbf{M} = \mathbf{M}_1 + \mathbf{M}_2 = -(M_2 \cos 25^\circ)\mathbf{i} + (M_1 - M_2 \sin 25^\circ)\mathbf{k}$$

Note that since **W** and **M** are perpendicular, it follows that they can be replaced with a single equivalent force.

(a) Have
$$F = \mathbf{W}$$
 or $\mathbf{F} = -W_{\mathbf{j}} = -(2.4 \text{ N})\mathbf{j}$

or
$$\mathbf{F} = -(2.40 \text{ N})\mathbf{j} \blacktriangleleft$$

(b) Assume that the line of action of **F** passes through point P(x, 0, z).

Then for equivalence

$$\mathbf{M} = \mathbf{r}_{P/O} \times \mathbf{F}$$

where

$$\mathbf{r}_{P/O} = x\mathbf{i} + z\mathbf{k}$$

$$\therefore -(M_2\cos 25^\circ)\mathbf{i} + (M_1 - M_2\sin 25^\circ)\mathbf{k}$$

$$= \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ x & 0 & z \\ 0 & -W & 0 \end{vmatrix} = (Wz)\mathbf{i} - (Wx)\mathbf{k}$$

PROBLEM 3.96 CONTINUED

Equating the i and k coefficients,

$$z = \frac{-M_z \cos 25^\circ}{W}$$
 and $x = -\left(\frac{M_1 - M_2 \sin 25^\circ}{W}\right)$

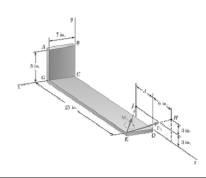
(b) For
$$W = 2.4 \text{ N}, M_1 = 0.068 \text{ N} \cdot \text{m}, M_2 = 0.065 \text{ N} \cdot \text{m}$$

$$x = \frac{0.068 - 0.065\sin 25^{\circ}}{-2.4} = -0.0168874 \text{ m}$$

or x = -16.89 mm

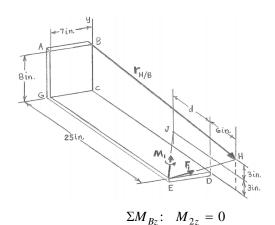
$$z = \frac{-0.065\cos 25^{\circ}}{2.4} = -0.024546 \text{ m}$$

or z = -24.5 mm



A 20-lb force \mathbf{F}_1 and a 40-lb·ft couple \mathbf{M}_1 are applied to corner E of the bent plate shown. If \mathbf{F}_1 and \mathbf{M}_1 are to be replaced with an equivalent force-couple system $(\mathbf{F}_2, \mathbf{M}_2)$ at corner B and if $(M_2)_z = 0$, determine (a) the distance d, (b) \mathbf{F}_2 and \mathbf{M}_2 .

SOLUTION



(a) Have

 $\mathbf{k} \cdot \left(\mathbf{r}_{H/B} \times \mathbf{F}_{1}\right) + M_{1z} = 0 \tag{1}$

where

$$\mathbf{r}_{H/B} = (31 \text{ in.})\mathbf{i} - (2 \text{ in.})\mathbf{j}$$

$$\mathbf{F}_{1} = \boldsymbol{\lambda}_{EH} F_{1}$$

$$= \frac{(6 \text{ in.})\mathbf{i} + (6 \text{ in.})\mathbf{j} - (7 \text{ in.})\mathbf{k}}{11.0 \text{ in.}} (20 \text{ lb})$$

$$= \frac{20 \text{ lb}}{11.0} (6\mathbf{i} + 6\mathbf{j} - 7\mathbf{k})$$

$$M_{1z} = \mathbf{k} \cdot \mathbf{M}_{1}$$

$$\mathbf{M}_{1} = \boldsymbol{\lambda}_{EI} M_{1}$$

$$\mathbf{M}_{1} = \lambda_{EJ} M_{1}$$

$$= \frac{-d\mathbf{i} + (3 \text{ in.})\mathbf{j} - (7 \text{ in.})\mathbf{k}}{\sqrt{d^{2} + 58 \text{ in.}}} (480 \text{ lb·in.})$$

Then from Equation (1),

$$\begin{vmatrix} 0 & 0 & 1 \\ 31 & -2 & 0 \\ 6 & 6 & -7 \end{vmatrix} \frac{20 \text{ lb} \cdot \text{in.}}{11.0} + \frac{(-7)(480 \text{ lb} \cdot \text{in.})}{\sqrt{d^2 + 58}} = 0$$

PROBLEM 3.97 CONTINUED

Solving for d, Equation (1) reduces to

$$\frac{20 \text{ lb} \cdot \text{in.}}{11.0} \left(186 + 12 \right) - \frac{3360 \text{ lb} \cdot \text{in.}}{\sqrt{d^2 + 58}} = 0$$

d = 5.3955 in.

From which

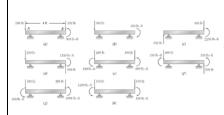
or d = 5.40 in.

(b)
$$\mathbf{F}_2 = \mathbf{F}_1 = \frac{20 \text{ lb}}{11.0} (6\mathbf{i} + 6\mathbf{j} - 7\mathbf{k})$$
$$= (10.9091\mathbf{i} + 10.9091\mathbf{j} - 12.7273\mathbf{k})\text{lb}$$

or
$$\mathbf{F}_2 = (10.91 \text{ lb})\mathbf{i} + (10.91 \text{ lb})\mathbf{j} - (12.73 \text{ lb})\mathbf{k} \blacktriangleleft$$

$$\begin{split} \mathbf{M}_2 &= \mathbf{r}_{H/B} \times \mathbf{F}_1 + \mathbf{M}_1 \\ &= \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 31 & -2 & 0 \\ 6 & 6 & -7 \end{vmatrix} \frac{20 \text{ lb} \cdot \text{in.}}{11.0} + \frac{\left(-5.3955\right)\mathbf{i} + 3\mathbf{j} - 7\mathbf{k}}{9.3333} (480 \text{ lb} \cdot \text{in.}) \\ &= \left(25.455\mathbf{i} + 394.55\mathbf{j} + 360\mathbf{k}\right) \text{lb} \cdot \text{in.} \\ &+ \left(-277.48\mathbf{i} + 154.285\mathbf{j} - 360\mathbf{k}\right) \text{lb} \cdot \text{in.} \\ &\mathbf{M}_2 = -\left(252.03 \text{ lb} \cdot \text{in.}\right) \mathbf{i} + \left(548.84 \text{ lb} \cdot \text{in.}\right) \mathbf{j} \end{split}$$

or
$$\mathbf{M}_2 = -(21.0 \text{ lb} \cdot \text{ft})\mathbf{i} + (45.7 \text{ lb} \cdot \text{ft})\mathbf{j} \blacktriangleleft$$



A 4-ft-long beam is subjected to a variety of loadings. (a) Replace each loading with an equivalent force-couple system at end A of the beam. (b) Which of the loadings are equivalent?

SOLUTION ΣF_{v} : $-200 \text{ lb} - 100 \text{ lb} = R_{a}$ (a) Have (*a*) or $\mathbf{R}_a = 300 \text{ lb} \downarrow \blacktriangleleft$ ΣM_A : 900 lb·ft – (100 lb)(4 ft) = M_a and or $\mathbf{M}_a = 500 \text{ lb} \cdot \text{ft}$ (b) Have ΣF_{v} : $-300 \text{ lb} = R_{b}$ or $\mathbf{R}_h = 300 \, \mathrm{lb} \, \downarrow \, \blacktriangleleft$ ΣM_A : $-450 \text{ lb} \cdot \text{ft} = M_b$ and or $\mathbf{M}_b = 450 \text{ lb} \cdot \text{ft}$ ΣF_{v} : 150 lb - 450 lb = R_{c} (c) Have or $\mathbf{R}_c = 300 \text{ lb} \downarrow \blacktriangleleft$ and ΣM_A : 2250 lb·ft - (450 lb)(4 ft) = M_c or $\mathbf{M}_c = 450 \text{ lb} \cdot \text{ft}$ (d) Have ΣF_{v} : $-200 \text{ lb} + 400 \text{ lb} = R_{d}$ or $\mathbf{R}_d = 200 \text{ lb} \uparrow \blacktriangleleft$ ΣM_A : $(400 \text{ lb})(4 \text{ ft}) - 1150 \text{ lb} \cdot \text{ft} = M_d$ and or $\mathbf{M}_d = 450 \, \mathrm{lb \cdot ft}$ ΣF_{v} : $-200 \text{ lb} - 100 \text{ lb} = R_{e}$ (e) Have or $\mathbf{R}_{e} = 300 \, \mathrm{lb} \, \downarrow \, \blacktriangleleft$ ΣM_A : 100 lb·ft + 200 lb·ft - (100 lb)(4 ft) = M_e and or $\mathbf{M}_e = 100 \text{ lb} \cdot \text{ft}$

PROBLEM 3.98 CONTINUED

(f) Have
$$\Sigma F_{y}$$
: $-400 \text{ lb} + 100 \text{ lb} = R_{f}$

or
$$\mathbf{R}_f = 300 \text{ lb} \downarrow \blacktriangleleft$$

and
$$\Sigma M_A$$
: -150 lb·ft + 150 lb·ft + (100 lb)(4 ft) = M_f

or
$$\mathbf{M}_f = 400 \, \mathrm{lb \cdot ft}$$

(g) Have
$$\Sigma F_y$$
: -100 lb - 400 lb = R_g

or
$$\mathbf{R}_g = 500 \, \mathrm{lb} \, \mathbf{\triangleleft}$$

and
$$\Sigma M_A$$
: 100 lb·ft + 2000 lb·ft - (400 lb)(4 ft) = M_g

or
$$\mathbf{M}_g = 500 \text{ lb} \cdot \text{ft}$$

(h) Have
$$\Sigma F_{v}$$
: -150 lb - 150 lb = R_{h}

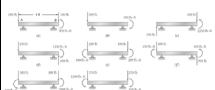
or
$$\mathbf{R}_h = 300 \text{ lb} \downarrow \blacktriangleleft$$

and
$$\Sigma M_A$$
: 1200 lb·ft - 150 lb·ft - (150 lb)(4 ft) = M_h

or
$$\mathbf{M}_h = 450 \, \mathrm{lb \cdot ft}$$

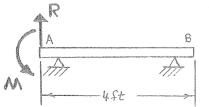
Therefore, loadings (c) and (h) are equivalent

(*b*)



A 4-ft-long beam is loaded as shown. Determine the loading of Problem 3.98 which is equivalent to this loading.

SOLUTION



Have

$$\Sigma \mathbf{F}_{y}$$
: -100 lb - 200 lb = R

or $\mathbf{R} = 300 \text{ lb} \downarrow$

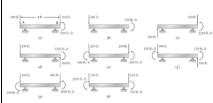
and

$$\Sigma M_A$$
: $-200 \text{ lb} \cdot \text{ft} + 1400 \text{ lb} \cdot \text{ft} - (200 \text{ lb})(4 \text{ ft}) = M$
or $\mathbf{M} = 400 \text{ lb} \cdot \text{ft}$

Equivalent to case (*f*) of Problem 3.98 ◀

Problem 3.98 Equivalent force-couples at A

case	R	M
(a)	300 lb ↓	500 lb·ft)
(b)	300 lb ↓	450 lb·ft
(c)	300 lb ↓	450 lb·ft)
(d)	200 lb †	450 lb·ft)
(e)	300 lb ↓	100 lb·ft)
(f)	300 lb ↓	400 lb·ft
(g)	500 lb ↓	500 lb·ft)
(h)	300 lb ↓	450 lb·ft)

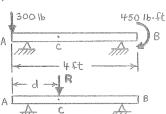


Determine the single equivalent force and the distance from point A to its line of action for the beam and loading of (a) Problem 3.98b, (b) Problem 3.98d, (c) Problem 3.98e.

Problem 3.98: A 4-ft-long beam is subjected to a variety of loadings. (a) Replace each loading with an equivalent force-couple system at end A of the beam. (b) Which of the loadings are equivalent?

SOLUTION

(a)



For equivalent single force at distance d from A

Have ΣF_{v} : -300 lb = R

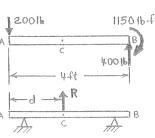
or $\mathbf{R} = 300 \text{ lb} \downarrow \blacktriangleleft$

and ΣM_C : $(300 \text{ lb})(d) - 450 \text{ lb} \cdot \text{ft} = 0$

or $d = 1.500 \text{ ft} \blacktriangleleft$

(b)

(c)



Have $\Sigma F_{y}: -200 \text{ lb} + 400 \text{ lb} = R$

or $\mathbf{R} = 200 \text{ lb} \uparrow \blacktriangleleft$

and ΣM_C : $(200 \text{ lb})(d) + (400 \text{ lb})(4-d) - 1150 \text{ lb} \cdot \text{ft} = 0$

or $d = 2.25 \text{ ft} \blacktriangleleft$

Have ΣF_{y} : -200 lb - 100 lb = R

or $\mathbf{R} = 300 \text{ lb} \, \mathbf{\blacksquare}$

and ΣM_C : 100 lb·ft + (200 lb)(d) - (100 lb)(4 - d) + 200 lb·ft = 0

or $d = 0.333 \text{ ft} \blacktriangleleft$