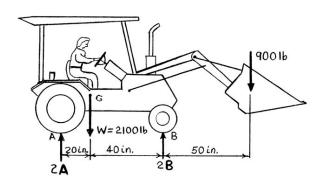


A 2100-lb tractor is used to lift 900 lb of gravel. Determine the reaction at each of the two (*a*) rear wheels *A*, (*b*) front wheels *B*.

# **SOLUTION**



(a) From f.b.d. of tractor

+) 
$$\Sigma M_B = 0$$
:  $(2100 \text{ lb})(40 \text{ in.}) - (2A)(60 \text{ in.}) - (900 \text{ lb})(50 \text{ in.}) = 0$ 

$$\therefore A = 325 \, \text{lb}$$

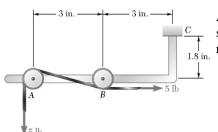
or 
$$\mathbf{A} = 325 \, \mathrm{lb} \, \uparrow \blacktriangleleft$$

(b) From f.b.d. of tractor

+) 
$$\Sigma M_A = 0$$
:  $(2B)(60 \text{ in.}) - (2100 \text{ lb})(20 \text{ in.}) - (900 \text{ lb})(110 \text{ in.}) = 0$ 

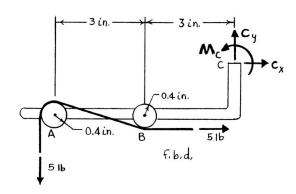
:. 
$$B = 1175 \, \text{lb}$$

or 
$$\mathbf{B} = 1175 \, \mathrm{lb} \, \uparrow \blacktriangleleft$$



A tension of 5 lb is maintained in a tape as it passes the support system shown. Knowing that the radius of each pulley is 0.4 in., determine the reaction at C.

## **SOLUTION**



From f.b.d. of system

$$\stackrel{+}{\longrightarrow} \Sigma F_x = 0$$
:  $C_x + (5 \text{ lb}) = 0$ 

$$\therefore C_x = -5 \text{ lb}$$

$$+ \uparrow \Sigma F_y = 0$$
:  $C_y - (5 \text{ lb}) = 0$ 

$$\therefore C_y = 5 \text{ lb}$$

Then

$$C = \sqrt{(C_x)^2 + (C_y)^2} = \sqrt{(5)^2 + (5)^2} = 7.0711 \text{ lb}$$

and

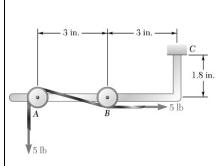
$$\theta = \tan^{-1}\left(\frac{+5}{-5}\right) = -45^{\circ}$$

or  $C = 7.07 \text{ lb} \ge 45.0^{\circ} \blacktriangleleft$ 

+) 
$$\Sigma M_C = 0$$
:  $M_C + (5 \text{ lb})(6.4 \text{ in.}) + (5 \text{ lb})(2.2 \text{ in.}) = 0$ 

$$M_C = -43.0 \, \text{lb} \cdot \text{in}$$

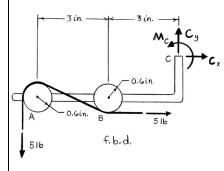
or 
$$\mathbf{M}_C = 43.0 \, \mathrm{lb \cdot in.}$$



Solve Problem 4.157 assuming that 0.6-in.-radius pulleys are used.

**P4.157** A tension of 5 lb is maintained in a tape as it passes the support system shown. Knowing that the radius of each pulley is 0.4 in., determine the reaction at C.

### **SOLUTION**



From f.b.d of system

$$\stackrel{+}{\longrightarrow} \Sigma F_x = 0$$
:  $C_x + (5 \text{ lb}) = 0$ 

$$\therefore C_x = -5 \text{ lb}$$

$$+ \int \Sigma F_{y} = 0$$
:  $C_{y} - (5 \text{ lb}) = 0$ 

$$\therefore C_{v} = 5 \text{ lb}$$

Then

$$C = \sqrt{(C_x)^2 + (C_y)^2} = \sqrt{(5)^2 + (5)^2} = 7.0711 \text{ lb}$$

and

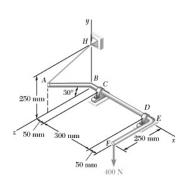
$$\theta = \tan^{-1}\left(\frac{5}{-5}\right) = -45.0^{\circ}$$

or 
$$C = 7.07 \text{ lb} \ge 45.0^{\circ} \blacktriangleleft$$

$$+$$
  $\Sigma M_C = 0$ :  $M_C + (5 \text{ lb})(6.6 \text{ in.}) + (5 \text{ lb})(2.4 \text{ in.}) = 0$ 

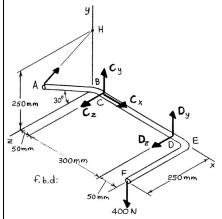
$$M_C = -45.0 \, \text{lb} \cdot \text{in}.$$

or 
$$\mathbf{M}_C = 45.0 \, \mathrm{lb \cdot in.}$$



The bent rod ABEF is supported by bearings at C and D and by wire AH. Knowing that portion AB of the rod is 250 mm long, determine (a) the tension in wire AH, (b) the reactions at C and D. Assume that the bearing at D does not exert any axial thrust.

## SOLUTION



(a) From f.b.d. of bent rod

$$\Sigma M_{CD} = 0$$
:  $\lambda_{CD} \cdot (\mathbf{r}_{H/B} \times \mathbf{T}) + \lambda_{CD} \cdot (\mathbf{r}_{F/E} \times \mathbf{F}) = 0$ 

where

$$\lambda_{CD} = \mathbf{i}$$

$$\mathbf{r}_{H/B} = (0.25 \,\mathrm{m})\mathbf{j}$$

$$\mathbf{T} = \mathbf{\lambda}_{AH} T$$

$$=\frac{\left(y_{AH}\right)\mathbf{j}-\left(z_{AH}\right)\mathbf{k}}{\sqrt{\left(y_{AH}\right)^{2}+\left(z_{AH}\right)^{2}}}T$$

$$y_{AH} = (0.25 \text{ m}) - (0.25 \text{ m})\sin 30^{\circ}$$

$$= 0.125 \text{ m}$$

$$z_{AH} = (0.25 \,\mathrm{m})\cos 30^{\circ}$$

$$= 0.21651 \,\mathrm{m}$$

$$T = \frac{T}{0.25} (0.125 \mathbf{j} - 0.21651 \mathbf{k})$$

$$\mathbf{r}_{F/E} = (0.25 \,\mathrm{m})\mathbf{k}$$

$$\mathbf{F} = -400 \text{ N j}$$

$$\begin{vmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0.125 & -0.21651 \end{vmatrix} (0.25) \left(\frac{T}{0.25}\right) + \begin{vmatrix} 1 & 0 & 0 \\ 0 & 0 & 1 \\ 0 & -1 & 0 \end{vmatrix} (0.25) (400 \text{ N}) = 0$$

$$-0.21651T + 0.25(400 \text{ N}) = 0$$

$$T = 461.88 \text{ N}$$

or  $T = 462 \text{ N} \blacktriangleleft$ 

#### **PROBLEM 4.159 CONTINUED**

(b) From f.b.d. of bent rod

$$\Sigma F_x = 0$$
:  $C_x = 0$ 

$$\Sigma M_{D(z-\text{axis})} = 0$$
:  $-[(461.88 \text{ N})\sin 30^\circ](0.35 \text{ m}) - C_y(0.3 \text{ m})$ 

$$-(400 \text{ N})(0.05 \text{ m}) = 0$$

$$\therefore C_y = -336.10 \text{ N}$$

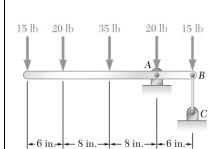
$$\Sigma M_{D(y-\text{axis})} = 0$$
:  $C_z(0.3 \text{ m}) - [(461.88 \text{ N})\cos 30^\circ](0.35 \text{ m}) = 0$   
 $\therefore C_z = 466.67 \text{ N}$ 

$$\Sigma F_y = 0$$
:  $D_y - 336.10 \text{ N} + (461.88 \text{ N}) \sin 30^\circ - 400 \text{ N} = 0$   
 $\therefore D_y = 505.16 \text{ N}$ 

$$\Sigma F_z = 0$$
:  $D_z + 466.67 \text{ N} - (461.88 \text{ N})\cos 30^\circ = 0$   
 $\therefore D_z = -66.670 \text{ N}$ 

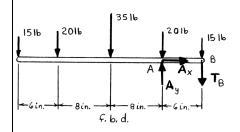
or 
$$\mathbf{D} = (505 \text{ N})\mathbf{j} - (66.7 \text{ N})\mathbf{k} \blacktriangleleft$$

or  $C = -(336 \text{ N})\mathbf{j} + (467 \text{ N})\mathbf{k} \blacktriangleleft$ 



For the beam and loading shown, determine (a) the reaction at A, (b) the tension in cable BC.

## **SOLUTION**



(a) From f.b.d of beam

$$\xrightarrow{+} \Sigma F_x = 0: \quad A_x = 0$$

$$+ \sum M_B = 0: \quad (15 \text{ lb})(28 \text{ in.}) + (20 \text{ lb})(22 \text{ in.}) + (35 \text{ lb})(14 \text{ in.})$$

$$+ (20 \text{ lb})(6 \text{ in.}) - A_y(6 \text{ in.}) = 0$$

$$\therefore A_y = 245 \text{ lb}$$

or 
$$\mathbf{A} = 245 \, \mathrm{lb} \, \uparrow \blacktriangleleft$$

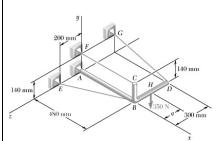
(b) From f.b.d of beam

+) 
$$\Sigma M_A = 0$$
:  $(15 \text{ lb})(22 \text{ in.}) + (20 \text{ lb})(16 \text{ in.}) + (35 \text{ lb})(8 \text{ in.})$   
 $-(15 \text{ lb})(6 \text{ in.}) - T_B(6 \text{ in.}) = 0$   
 $\therefore T_B = 140.0 \text{ lb}$ 

or 
$$T_B = 140.0 \, \text{lb} \, \blacktriangleleft$$

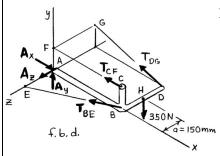
Check:

$$+\uparrow \Sigma F_y = 0$$
:  $-15 \text{ lb} - 20 \text{ lb} - 35 \text{ lb} - 20 \text{ lb}$   
 $-15 \text{ lb} - 140 \text{ lb} + 245 \text{ lb} = 0$ ?  
 $245 \text{ lb} - 245 \text{ lb} = 0 \text{ ok}$ 



Frame ABCD is supported by a ball-and-socket joint at A and by three cables. For a=150 mm, determine the tension in each cable and the reaction at A.

#### **SOLUTION**



First note

$$\mathbf{T}_{DG} = \lambda_{DG} T_{DG} = \frac{-(0.48 \text{ m})\mathbf{i} + (0.14 \text{ m})\mathbf{j}}{\sqrt{(0.48)^2 + (0.14)^2} \text{ m}} T_{DG}$$

$$= \frac{-0.48\mathbf{i} + 0.14\mathbf{j}}{0.50} T_{DG}$$

$$= \frac{T_{DG}}{25} (24\mathbf{i} + 7\mathbf{j})$$

$$\mathbf{T}_{BE} = \lambda_{BE} T_{BE} = \frac{-(0.48 \text{ m})\mathbf{i} + (0.2 \text{ m})\mathbf{k}}{\sqrt{(0.48)^2 + (0.2)^2} \text{ m}} T_{BE}$$

$$= \frac{-0.48\mathbf{i} + 0.2\mathbf{k}}{0.52} T_{BE}$$

$$= \frac{T_{BE}}{13} (-12\mathbf{j} + 5\mathbf{k})$$

From f.b.d. of frame ABCD

$$\Sigma M_x = 0$$
:  $\left(\frac{7}{25}T_{DG}\right)(0.3 \text{ m}) - (350 \text{ N})(0.15 \text{ m}) = 0$ 

or 
$$T_{DG} = 625 \text{ N} \blacktriangleleft$$

$$\Sigma M_y = 0$$
:  $\left(\frac{24}{25} \times 625 \text{ N}\right) (0.3 \text{ m}) - \left(\frac{5}{13} T_{BE}\right) (0.48 \text{ m}) = 0$ 

or 
$$T_{BE} = 975 \text{ N} \blacktriangleleft$$

$$\Sigma M_z = 0$$
:  $T_{CF} (0.14 \text{ m}) + \left(\frac{7}{25} \times 625 \text{ N}\right) (0.48 \text{ m})$   
 $- (350 \text{ N}) (0.48 \text{ m}) = 0$ 

or 
$$T_{CF} = 600 \text{ N} \blacktriangleleft$$

## **PROBLEM 4.161 CONTINUED**

$$\Sigma F_x = 0$$
:  $A_x + T_{CF} + (T_{BE})_x + (T_{DG})_x = 0$   
 $A_x - 600 \text{ N} - (\frac{12}{13} \times 975 \text{ N}) - (\frac{24}{25} \times 625 \text{ N}) = 0$ 

$$\therefore A_x = 2100 \text{ N}$$

$$\Sigma F_y = 0$$
:  $A_y + (T_{DG})_y - 350 \text{ N} = 0$ 

$$A_y + \left(\frac{7}{25} \times 625 \text{ N}\right) - 350 \text{ N} = 0$$

$$A_y = 175.0 \text{ N}$$

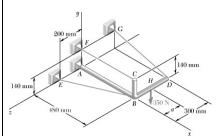
$$\Sigma F_z = 0: \quad A_z + (T_{BE})_z = 0$$

$$A_z + \left(\frac{5}{13} \times 975 \text{ N}\right) = 0$$

$$\therefore A_z = -375 \text{ N}$$

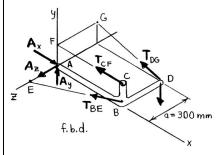
Therefore

$$A = (2100 \text{ N})\mathbf{i} + (175.0 \text{ N})\mathbf{j} - (375 \text{ N})\mathbf{k} \blacktriangleleft$$



Frame ABCD is supported by a ball-and-socket joint at A and by three cables. Knowing that the 350-N load is applied at D (a = 300 mm), determine the tension in each cable and the reaction at A.

#### **SOLUTION**



First note

$$\mathbf{T}_{DG} = \lambda_{DG} T_{DG} = \frac{-(0.48 \text{ m})\mathbf{i} + (0.14 \text{ m})\mathbf{j}}{\sqrt{(0.48)^2 + (0.14)^2} \text{ m}} T_{DG}$$

$$= \frac{-0.48\mathbf{i} + 0.14\mathbf{j}}{0.50} T_{DG}$$

$$= \frac{T_{DG}}{25} (24\mathbf{i} + 7\mathbf{j})$$

$$\mathbf{T}_{BE} = \lambda_{BE} T_{BE} = \frac{-(0.48 \text{ m})\mathbf{i} + (0.2 \text{ m})\mathbf{k}}{\sqrt{(0.48)^2 + (0.2)^2} \text{ m}} T_{BE}$$

$$= \frac{-0.48\mathbf{i} + 0.2\mathbf{k}}{0.52} T_{BE}$$

$$= \frac{T_{BE}}{13} (-12\mathbf{i} + 5\mathbf{k})$$

From f.b.d of frame ABCD

$$\Sigma M_x = 0$$
:  $\left(\frac{7}{25}T_{DG}\right)(0.3 \text{ m}) - (350 \text{ N})(0.3 \text{ m}) = 0$ 

or 
$$T_{DG} = 1250 \text{ N} \blacktriangleleft$$

$$\Sigma M_y = 0$$
:  $\left(\frac{24}{25} \times 1250 \text{ N}\right) \left(0.3 \text{ m}\right) - \left(\frac{5}{13} T_{BE}\right) \left(0.48 \text{ m}\right) = 0$ 

or 
$$T_{BE} = 1950 \text{ N} \blacktriangleleft$$

$$\Sigma M_z = 0$$
:  $T_{CF} (0.14 \text{ m}) + \left(\frac{7}{25} \times 1250 \text{ N}\right) (0.48 \text{ m})$   
 $- (350 \text{ N}) (0.48 \text{ m}) = 0$ 

or 
$$T_{CF} = 0 \blacktriangleleft$$

## **PROBLEM 4.162 CONTINUED**

$$\Sigma F_x = 0$$
:  $A_x + T_{CF} + (T_{BE})_x + (T_{DG})_x = 0$   
 $A_x + 0 - (\frac{12}{13} \times 1950 \text{ N}) - (\frac{24}{25} \times 1250 \text{ N}) = 0$ 

$$\therefore A_x = 3000 \text{ N}$$

$$\Sigma F_y = 0$$
:  $A_y + (T_{DG})_y - 350 \text{ N} = 0$ 

$$A_y + \left(\frac{7}{25} \times 1250 \text{ N}\right) - 350 \text{ N} = 0$$

$$A_y = 0$$

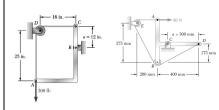
$$\Sigma F_z = 0: \quad A_z + \left(T_{BE}\right)_z = 0$$

$$A_z + \left(\frac{5}{13} \times 1950 \,\mathrm{N}\right) = 0$$

$$\therefore A_z = -750 \text{ N}$$

Therefore

$$A = (3000 \text{ N})i - (750 \text{ N})k \blacktriangleleft$$

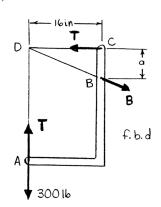


In the problems listed below, the rigid bodies considered were completely constrained and the reactions were statically determinate. For each of these rigid bodies it is possible to create an improper set of constraints by changing a dimension of the body. In each of the following problems determine the value of a which results in improper constraints. (a) Problem 4.81, (b) Problem 4.82.

## **SOLUTION**

(a)

T



(a) 
$$+ \sum M_B = 0$$
:  $(300 \text{ lb})(16 \text{ in.}) - T(16 \text{ in.}) + T(a) = 0$ 

or

$$T = \frac{(300 \text{ lb})(16 \text{ in.})}{(16 - a) \text{in.}}$$

 $\therefore$  T becomes infinite when

$$16 - a = 0$$

or  $a = 16.00 \, \text{in}$ .

400 mm f, b, d.

(b) 
$$+ \sum \Delta M_C = 0$$
:  $(T - 80 \text{ N})(0.2 \text{ m}) - \left(\frac{8}{17}T\right)(0.175 \text{ m})$ 

$$-\left(\frac{15}{17}T\right)(0.4 \text{ m} - a) = 0$$

$$0.2T - 16.0 - 0.82353T - 0.35294T + 0.88235Ta = 0$$

or

175 mm

$$T = \frac{16.0}{0.88235a - 0.23529}$$

 $\therefore$  T becomes infinite when

$$0.88235a - 0.23529 = 0$$

$$a = 0.26666 \,\mathrm{m}$$

or  $a = 267 \text{ mm} \blacktriangleleft$