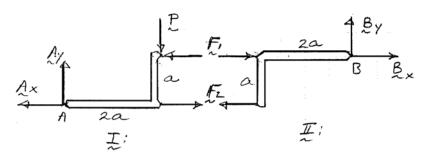
Each of the frames shown consists of two L-shaped members connected by two rigid links. For each frame, determine the reactions at the supports and indicate whether the frame is rigid.

SOLUTION

(a) member FBDs:



FBD I:
$$(\Sigma M_A = 0: aF_1 - 2aP = 0)$$
 $F_1 = 2P; \uparrow \Sigma F_y = 0: A_y - P = 0$ $A_y = P \uparrow$

FBD II:
$$(\Sigma M_B = 0: -aF_2 = 0)$$
 $F_2 = 0$

$$\longrightarrow \Sigma F_x = 0 : B_x + F_1 = 0, \ B_x = -F_1 = -2P \qquad \mathbf{B}_x = 2P \longrightarrow$$

FBD I:
$$\longrightarrow \Sigma F_x = 0$$
: $-A_x - F_1 + F_2 = 0$ $A_x = F_2 - F_1 = 0 - 2P$ $\mathbf{A}_x = 2P$

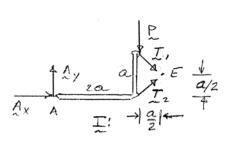
$$\uparrow \Sigma F_y = 0: B_y = 0 \qquad \text{so } \mathbf{B} = 2P \longrightarrow \blacktriangleleft$$

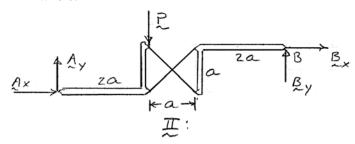
so $A = 2.24P \angle 26.6^{\circ} \blacktriangleleft$

frame is rigid

(b) **FBD left:**

FBD whole:





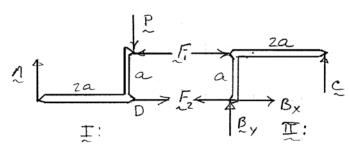
FBD I:
$$(\sum \Delta M_E = 0) : \frac{a}{2}P + \frac{a}{2}A_x - \frac{5a}{2}A_y = 0$$
 $A_x - 5A_y = -P$

FBD II:
$$(\Sigma M_B = 0: 3aP + aA_x - 5aA_y = 0)$$
 $A_x - 5A_y = -3P$

This is impossible unless P = 0 : not rigid

PROBLEM 6.115 CONTINUED

(c) member FBDs:



FBD I:
$$\Sigma F_y = 0: A - P = 0$$

$$\mathbf{A} = P \uparrow \blacktriangleleft$$

$$\sum M_D = 0 : aF_1 - 2aA = 0 \qquad F_1 = 2P$$

FBD II:
$$(\Sigma M_B = 0: 2aC - aF_1 = 0)$$
 $C = \frac{F_1}{2} = P$

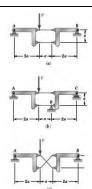
$$C = P \uparrow \blacktriangleleft$$

$$\longrightarrow \Sigma F_x = 0: F_1 - F_2 + B_x = 0$$
 $B_x = P - P = 0$

$$\uparrow \Sigma F_x = 0 \colon B_y + C = 0 \qquad B_y = -C = -P$$

 $\mathbf{B} = P$

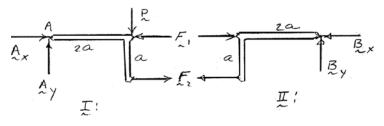
Frame is rigid



Each of the frames shown consists of two L-shaped members connected by two rigid links. For each frame, determine the reactions at the supports and indicate whether the frame is rigid.

SOLUTION

(a) member FBDs:

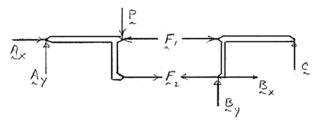


FBD II:
$$\Sigma F_y = 0$$
: $B_y = 0$ $\Sigma M_B = 0$: $aF_2 = 0$ $F_2 = 0$

FBD I:
$$(\Sigma M_A = 0: aF_2 - 2aP = 0)$$
 but $F_2 = 0$

so P = 0 not rigid for $P \neq 0$

(b) member FBDs:

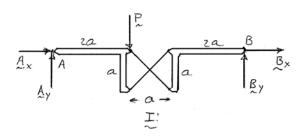


Note: 7 unknowns $(A_x, A_y, B_x, B_y, F_1, F_2, C)$ but only 6 independent equations.

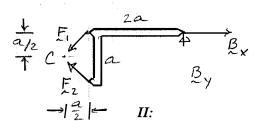
System is statically indeterminate ◀

System is, however, rigid ◀

(c) **FBD whole:**



FBD right:



PROBLEM 6.116 CONTINUED

FBD I:
$$\left(\sum \Sigma M_A = 0: 5aB_y - 2aP = 0\right)$$
 $\mathbf{B}_y = \frac{2}{5}P^{\dagger}$ $\left(\sum F_y = 0: A_y - P + \frac{2}{5}P = 0\right)$ $\mathbf{A}_y = \frac{3}{5}P^{\dagger}$

FBD II:
$$(\Sigma M_c = 0): \frac{a}{2}B_x - \frac{5a}{2}B_y = 0$$
 $B_x = 5B_y$ $B_x = 2P$

$$B_{\rm r} = 5B_{\rm v}$$

$$\mathbf{B}_{x} = 2P \longrightarrow$$

FBD I:
$$\Sigma F_x = 0$$
: $A_x + B_x = 0$ $A_x = -B_x$ $A_x = 2P$

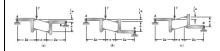
$$A_{r} = -B_{r}$$

$$\mathbf{A}_{r} = 2P \leftarrow$$

$$A = 2.09P \ge 16.70^{\circ} \blacktriangleleft$$

$$\mathbf{B} = 2.04P \angle 11.31^{\circ} \blacktriangleleft$$

System is rigid ◀

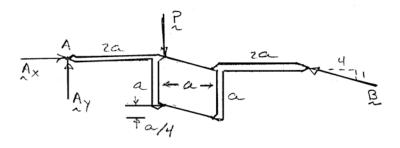


Each of the frames shown consists of two L-shaped members connected by two rigid links. For each frame, determine the reactions at the supports and indicate whether the frame is rigid.

SOLUTION

Note: In all three cases, the right member has only three forces acting, two of which are parallel. Thus the third force, at *B*, must be parallel to the link forces.

(a) **FBD whole:**



$$\sum M_A = 0$$
: $-2aP - \frac{a}{4} \frac{4}{\sqrt{17}} B + 5a \frac{1}{\sqrt{17}} B = 0$ $B = 2.06P$

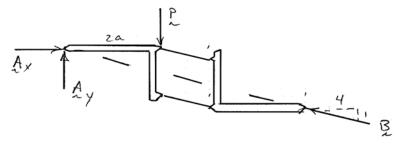
$$\mathbf{B} = 2.06P \ge 14.04^{\circ} \blacktriangleleft$$

$$\uparrow \Sigma F_y = 0$$
: $A_y - P + \frac{1}{\sqrt{17}}B = 0$ $A_y = \frac{P}{2} \uparrow$

$$A = 2.06P \angle 14.04^{\circ} \blacktriangleleft$$

rigid ◀

(b) **FBD whole:**



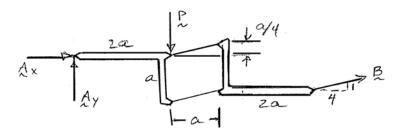
Since **B** passes through A, $(\Sigma M_A = 2aP = 0)$ only if P = 0

 \therefore no equilibrium if $P \neq 0$

not rigid ◀

PROBLEM 6.117 CONTINUED

(c) **FBD whole:**



$$\sum M_A = 0: \ 5a \frac{1}{\sqrt{17}}B + \frac{3a}{4} \frac{4}{\sqrt{17}}B - 2aP = 0 \qquad B = \frac{\sqrt{17}}{4}P \qquad \mathbf{B} = 1.031P \angle 14.04^{\circ} \blacktriangleleft$$

$$B = \frac{\sqrt{17}}{4}P$$

$$\mathbf{B} = 1.031P \ \angle 14.04^{\circ} \blacktriangleleft$$

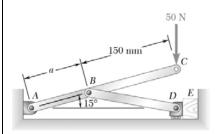
$$\longrightarrow \Sigma F_x = 0: A_x + \frac{4}{\sqrt{17}}B = 0 \qquad A_x = -P$$

↑
$$\Sigma F_y = 0$$
: $A_y - P + \frac{1}{\sqrt{17}}B = 0$ $A_y = P - \frac{P}{4} = \frac{3P}{4}$ $A = 1.250P \ge 36.9^{\circ}$ ◀

$$A_{y} = P - \frac{P}{4} = \frac{3P}{4}$$

$$A = 1.250P \ge 36.9^{\circ} \blacktriangleleft$$

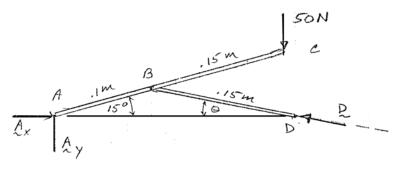
System is rigid ◀



A 50-N force directed vertically downward is applied to the toggle vise at C. Knowing that link BD is 150 mm long and that a = 100 mm, determine the horizontal force exerted on block E.

SOLUTION

FBD machine:



Note: $(0.1 \text{ m})\sin 15^\circ = (0.15 \text{ m})\sin \theta$

$$\theta = \sin^{-1}(0.17255) = 9.9359^{\circ}$$

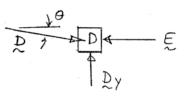
$$\widehat{AD} = (0.1 \text{ m})\cos 15^{\circ} + (0.15 \text{ m})\cos \theta$$

= 0.24434 m

 $\sum M_A = 0$: $(0.24434 \text{ m})D \sin \theta - (0.25 \text{ m})(\cos 15^\circ)(50 \text{ N}) = 0$

$$D = \frac{0.25 \cos 15^{\circ}}{(0.24434)(0.17255)} 50 \text{ N} = 286.38 \text{ N}$$

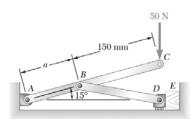
FBD part D:



$$\longrightarrow \Sigma F_x = 0: \ D\cos\theta - E = 0$$

$$E = D\cos\theta = 282.1\,\mathrm{N}$$

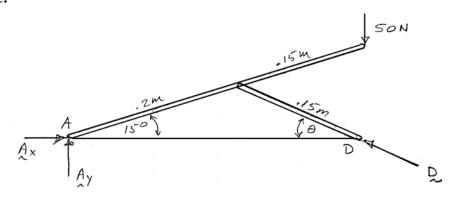
$$\mathbf{E}_{\mathrm{block}} = 282 \,\mathrm{N} \longrightarrow \blacktriangleleft$$



A 50-N force directed vertically downward is applied to the toggle vise at C. Knowing that link BD is 150 mm long and that a = 200 mm, determine the horizontal force exerted on block E.

SOLUTION

FBD machine:



Note:
$$(0.2 \text{ m})\sin 15^{\circ} = (0.15 \text{ m})\sin \theta$$

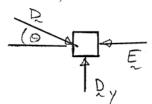
$$\theta = \sin^{-1}(0.3451) = 20.187^{\circ}$$

$$\widehat{AD} = (0.2 \text{ m})\cos 15^{\circ} + (0.15 \text{m})\cos \theta$$

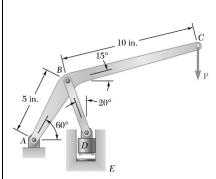
= 0.33397 m

$$(\Sigma M_A = 0: (0.33397 \text{ m})D \sin 20.187^\circ - (0.35\text{m})(\cos 15^\circ)(50 \text{ N}) = 0$$
 $D = 146.67 \text{ N}$

FBD part D:



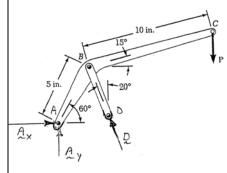
$$\mathbf{E}_{\text{on block}} = 137.7 \text{ N} \longrightarrow \blacktriangleleft$$



The press shown is used to emboss a small seal at E. Knowing that P = 60 lb, determine (a) the vertical component of the force exerted on the seal, (b) the reaction at A.

SOLUTION

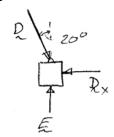
FBD machine:



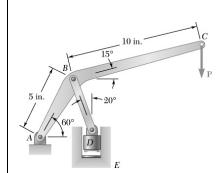
 $\sum M_A = 0: \left[(5 \text{ in.}) \cos 60^\circ \right] D \cos 20^\circ + \left[(5 \text{ in.}) \sin 60^\circ \right] D \sin 20^\circ$ $- \left[(5 \text{ in.}) \cos 60^\circ + (10 \text{ in.}) \cos 15^\circ \right] (60 \text{ lb}) = 0$ D = 190.473 lb $\Rightarrow \sum F_x = 0: A_x - D \sin 20^\circ = 0 \qquad A_x = 65.146 \text{ lb}$ $\uparrow \sum F_y = 0: A_y + D \cos 20^\circ - 60 \text{ lb} = 0 \qquad A_x = -118.99 \text{ lb}$

so **A** = 135.7 lb
$$\sqrt{}$$
 61.3° \triangleleft

FBD part D:



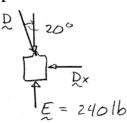
$$\uparrow \Sigma F_y = 0$$
: $E - D\cos 20^\circ = 0$ $E = (190.47 \text{ lb})\cos 20^\circ$ $= 178.98 \text{ lb}$ $\mathbf{E}_{\text{on seal}} = 179.0 \text{ lb}$



The press shown is used to emboss a small seal at E. Knowing that the vertical component of the force exerted on the seal must be 240 lb, determine (a) the required vertical force \mathbf{P} , (b) the corresponding reaction at A.

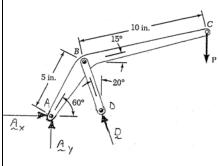
SOLUTION

FBD part D:



$$D = \frac{240 \text{ lb}}{\cos 20^{\circ}} = 255.40 \text{ lb}$$

FBD machine:



$$\left(\sum \Delta M_A = 0: \left[(5in.)\cos 60^\circ \right] D\cos 20^\circ + \left[(5in.)\cos 60^\circ \right] D\sin 20^\circ$$

$$-\left[\left(5 \text{ in.}\right) \cos 60^{\circ} + 10 \text{ in.}\right) \cos 15^{\circ}\right] P = 0$$

$$P = 80.453 \text{ lb}$$

$$(b) \qquad \sum F_x = 0 : A_x - D \sin 20^\circ = 0$$

$$A_x = 87.35 \text{ lb}$$

$$\Sigma F_y = 0: A_y + 240 \text{ lb} - 80.5 \text{ lb} = 0$$

$$A_{\rm y} = 159.5 \; {\rm lb}$$