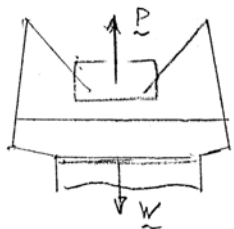


### PROBLEM 6.137

The drum lifter shown is used to lift a steel drum. Knowing that the mass of the drum and its contents is 240 kg, determine the forces exerted at  $F$  and  $H$  on member  $DFH$ .

### SOLUTION

**FBD System:**



$$\uparrow \Sigma F_y = 0;$$

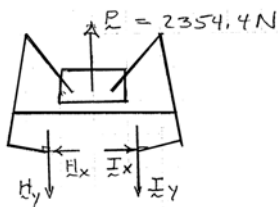
$$P - W = 0$$

$$P = W$$

$$= (240 \text{ kg})(9.81 \text{ m/s}^2)$$

$$P = 2354.4 \text{ N}$$

**FBD machine:**



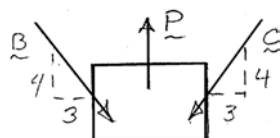
$$\text{Symmetry: } H_x = I_x$$

$$H_y = I_y$$

$$\uparrow \Sigma F_y = 0: P - 2H_y = 0$$

$$H_y = \frac{P}{2} = 1177.2 \text{ N}$$

**FBD ABC:**



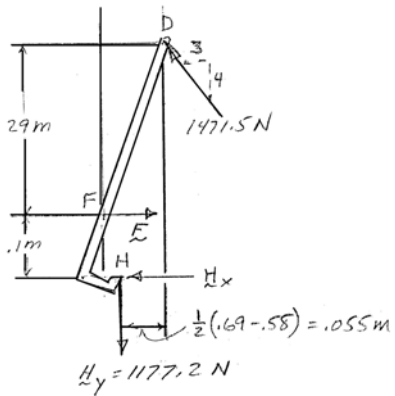
$$\text{Symmetry: } C = B$$

$$\Sigma F_y = 0: P - 2\frac{4}{5}B = 0$$

$$B = \frac{5}{8}P = 1471.5 \text{ N}$$

# PROBLEM 6.137 CONTINUED

FBD DFH:



$$\left( \sum M_H = 0: -(0.1 \text{ m})F + (0.39 \text{ m})\left(\frac{3}{5}1471.5 \text{ N}\right) \right.$$

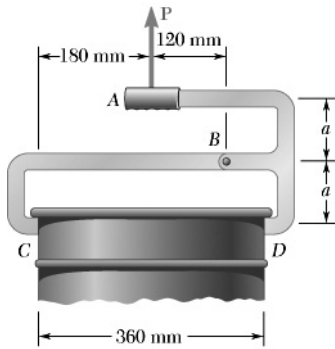
$$\left. - (0.055 \text{ m})\left(\frac{4}{5}1471.5 \text{ N}\right) = 0 \right.$$

$$F = 3973.05 \text{ N} \quad \mathbf{F} = 3.97 \text{ kN} \rightarrow \blacktriangleleft$$

$$\rightarrow \sum F_x = 0: 3973.05 \text{ N} - \frac{3}{5}1471.5 \text{ N} - H_x = 0$$

$$H_x = 3090.15 \text{ N}$$

$$\mathbf{H} = 3.31 \text{ kN} \nearrow 20.9^\circ \blacktriangleleft$$



### PROBLEM 6.138

A small barrel having a mass of 72 kg is lifted by a pair of tongs as shown. Knowing that  $a = 100$  mm, determine the forces exerted at  $B$  and  $D$  on tong  $ABD$ .

### SOLUTION

Notes: From FBD whole, by inspection,

FBB  $ABD$ :

$$P = W = mg = (72 \text{ kg})(9.81 \text{ m/s}^2)$$

$$P = 706.32 \text{ N}$$

$BC$  is a two-force member:  $B_x = 3B_y$

$$\left( \sum M_D = 0: (0.1 \text{ m})B_x + (0.06 \text{ m})B_y - (0.18 \text{ m})P = 0 \right.$$

$$3B_y + 0.6B_y = 1.8P$$

$$B_y = \frac{P}{2} = 353.16 \text{ N}$$

$$B_x = \frac{3P}{2} = 1059.48 \text{ N}$$

$$\text{and } \mathbf{B} = 1.117 \text{ kN } \nearrow 18.43^\circ \blacktriangleleft$$

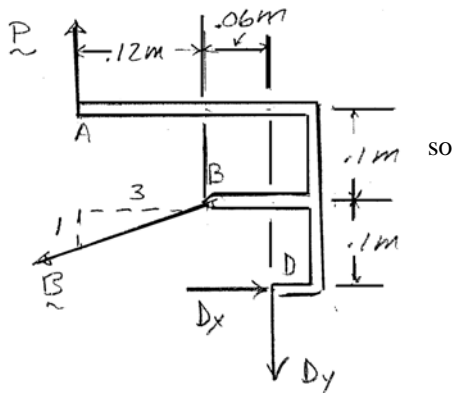
$$\rightarrow \sum F_x = 0: -B_x + D_x = 0$$

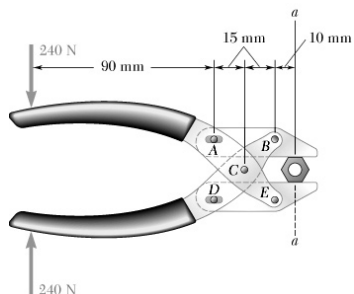
$$D_x = \frac{3P}{2} = 1059.48 \text{ N}$$

$$\uparrow \sum F_y = 0: P - B_y - D_y = 0$$

$$D_y = P - \frac{P}{2} = 353.16 \text{ N}$$

$$\text{so } \mathbf{D} = 1.117 \text{ kN } \searrow 18.43^\circ \blacktriangleleft$$



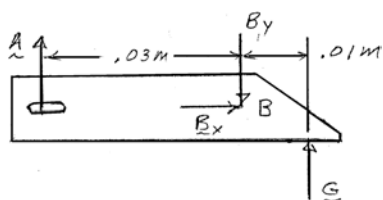


### PROBLEM 6.139

Determine the magnitude of the ripping forces exerted along line  $aa$  on the nut when two 240-N forces are applied to the handles as shown. Assume that pins  $A$  and  $D$  slide freely in slots cut in the jaws.

### SOLUTION

FBD jaw AB:



$$\rightarrow \Sigma F_x = 0: B_x = 0$$

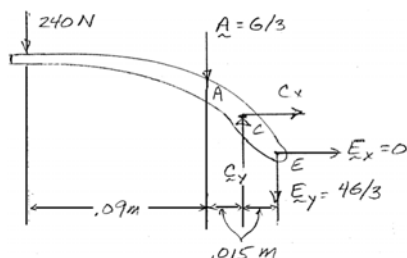
$$\curvearrowleft \Sigma M_B = 0: (0.01 \text{ m})G - (0.03 \text{ m})A = 0$$

$$A = \frac{G}{3}$$

$$\uparrow \Sigma F_y = 0: A + G - B_y = 0$$

$$B_y = A + G = \frac{4G}{3}$$

FBD handle ACE:

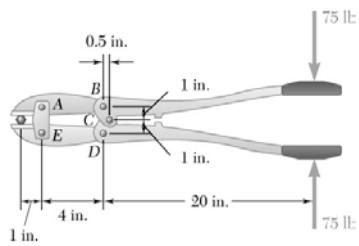


By symmetry and FBD jaw  $DE$ :  $D = A = \frac{G}{3}$ ,  $E_x = B_x = 0$ ,

$$E_y = B_y = \frac{4G}{3}$$

$$\curvearrowleft \Sigma M_C = 0: (0.105 \text{ m})(240 \text{ N}) + (0.015 \text{ m})\frac{G}{3} - (0.015 \text{ m})\frac{4G}{3} = 0$$

$$G = 1680 \text{ N} \blacktriangleleft$$

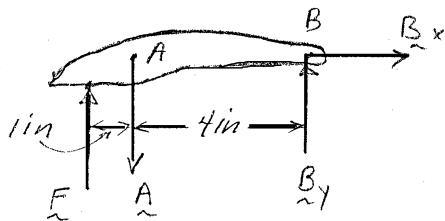


### PROBLEM 6.140

In using the bolt cutter shown, a worker applies two 75-lb forces to the handles. Determine the magnitude of the forces exerted by the cutter on the bolt.

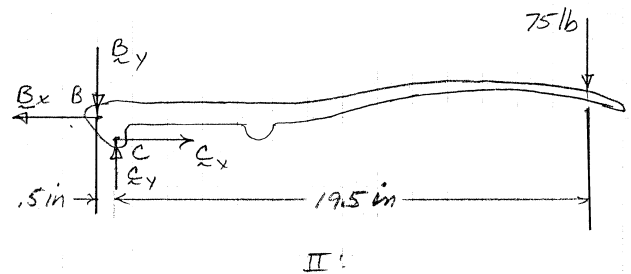
### SOLUTION

FBD Cutter AB:



$$\text{FBD I: } \rightarrow \Sigma F_x = 0: B_x = 0$$

FBD handle BC:



$$\text{FBD II: } \curvearrowleft \Sigma M_C = 0: (0.5 \text{ in.}) B_y - (19.5 \text{ in.}) 75 \text{ lb} = 0$$

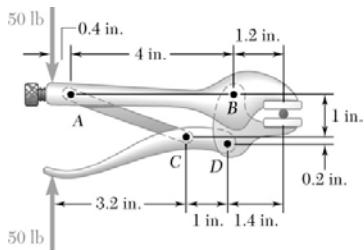
$$B_y = 2925 \text{ lb}$$

Then

$$\text{FBD I: } \curvearrowleft \Sigma M_A = 0: (4 \text{ in.}) B_y - (1 \text{ in.}) F = 0 \quad F = 4B_y$$

$$F = 11700 \text{ lb} = 11.70 \text{ kips} \blacktriangleleft$$

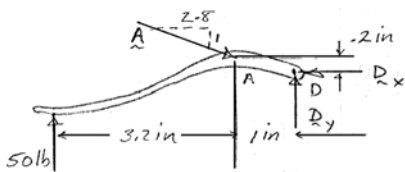
### PROBLEM 6.141



Determine the magnitude of the gripping forces produced when two 50-lb forces are applied as shown.

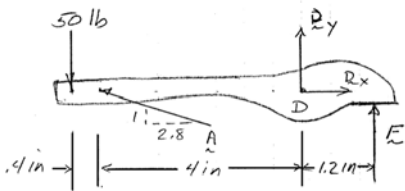
### SOLUTION

**FBD handle CD:**



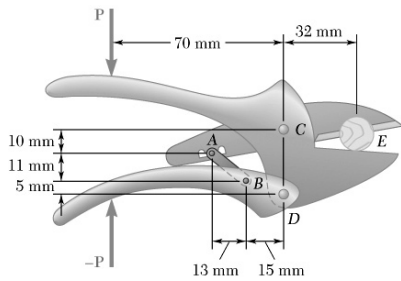
$$\begin{aligned} \sum M_D = 0: & -(4.2 \text{ in.})(50 \text{ lb}) - (0.2 \text{ in.}) \frac{2.8}{\sqrt{8.84}} A \\ & + (1 \text{ in.}) \left( \frac{1}{\sqrt{8.84}} A \right) = 0 \\ A = & 477.27 \sqrt{8.84} \text{ lb} \end{aligned}$$

**FBD handle AD:**



$$\begin{aligned} \sum M_D = 0: & (4.4 \text{ in.})(50 \text{ lb}) - (4 \text{ in.}) \frac{1}{\sqrt{8.84}} (477.27 \sqrt{8.84} \text{ lb}) \\ & + (1.2 \text{ in.}) F = 0 \end{aligned}$$

$$F = 1.408 \text{ kips} \blacktriangleleft$$

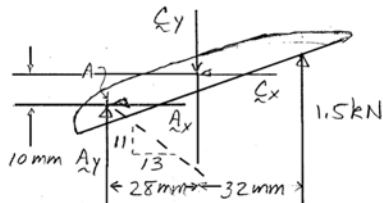


### PROBLEM 6.142

The compound-lever pruning shears shown can be adjusted by placing pin A at various ratchet positions on blade ACE. Knowing that 1.5-kN vertical forces are required to complete the pruning of a small branch, determine the magnitude  $P$  of the forces that must be applied to the handles when the shears are adjusted as shown.

### SOLUTION

**FBD cutter AC:**



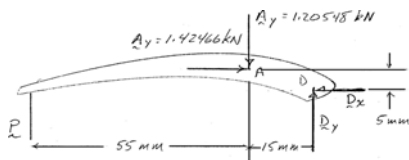
$$\sum M_C = 0: (32 \text{ mm})1.5 \text{ kN} - (28 \text{ mm})A_y - (10 \text{ mm})A_x = 0$$

$$10A_x + 28\left(\frac{11}{13}A_x\right) = 48 \text{ kN}$$

$$A_x = 1.42466 \text{ kN}$$

$$A_y = 1.20548 \text{ kN}$$

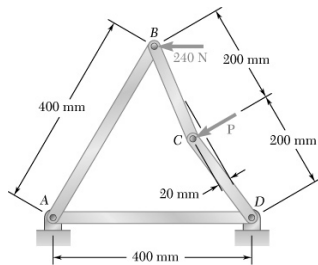
**FBD handle AD:**



$$\sum M_D = 0: (15 \text{ mm})(1.20548 \text{ kN}) - (5 \text{ mm})(1.42466 \text{ kN})$$

$$-(70 \text{ mm})P = 0$$

$$P = 0.1566 \text{ kN} = 156.6 \text{ N} \quad \blacktriangleleft$$



### PROBLEM 6.143

Determine the force **P** which must be applied to the toggle *BCD* To maintain equilibrium in the position shown.

### SOLUTION

**FBD joint B:**

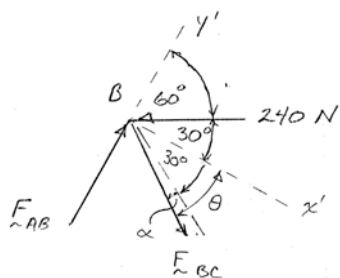
Note:

$$\theta = 30^\circ + \alpha$$

$$= 30^\circ + \tan^{-1} \frac{20}{200}$$

$$= 30^\circ + 5.711^\circ$$

$$= 35.711^\circ$$

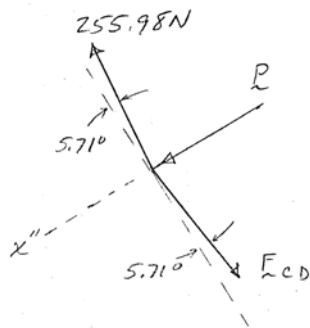


$$\sum F_{x'} = 0: F_{BC} \cos 35.711^\circ - (240 \text{ N}) \cos 30^\circ = 0$$

$$F_{BC} = 255.98 \text{ N T}$$

**FBD joint C:**

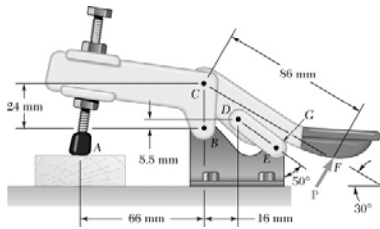
By symmetry:  $F_{CD} = 255.98 \text{ N}$



$$\sum F_{x''} = 0: P - 2(255.98 \text{ N}) \sin 5.711^\circ = 0$$

$$\mathbf{P} = 50.9 \text{ N } \nearrow 30.0^\circ \blacktriangleleft$$



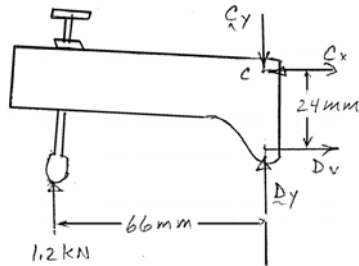


### PROBLEM 6.144

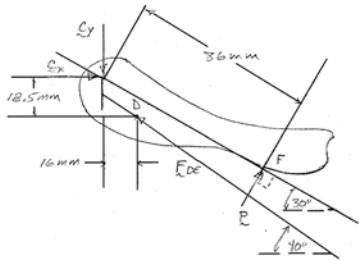
In the locked position shown, the toggle clamp exerts at  $A$  a vertical 1.2-kN force on the wooden block, and handle  $CF$  rests against the stop at  $G$ . Determine the force  $\mathbf{P}$  required to release the clamp. (Hint: To release the clamp, the forces of contact at  $G$  must be zero.)

### SOLUTION

#### FBD BC:



#### FBD CDF:



$$\left( \sum M_B = 0: (24 \text{ mm}) C_x - (66 \text{ mm})(1.2 \text{ kN}) = 0 \right.$$

$$C_x = 3.3 \text{ kN}$$

$$\rightarrow \sum F_x = 0: D_x - C_x = 0 \quad D_x = 3.3 \text{ kN}$$

$$\left( \sum M_C = 0: (86 \text{ mm}) P \right.$$

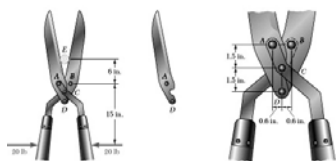
$$- [18.5 \text{ mm} - (16 \text{ mm}) \tan 40^\circ] (F_{DE} \cos 40^\circ) = 0$$

$$F_{DE} = 22.124 P$$

$$\rightarrow \sum F_x = 0: C_x - F_{DE} \cos 40^\circ + P \sin 30^\circ = 0$$

$$3.3 \text{ kN} - (22.124 P) \cos 40^\circ + P \sin 30^\circ = 0$$

$$\mathbf{P = 201 \text{ N} } \nearrow 60^\circ \blacktriangleleft$$



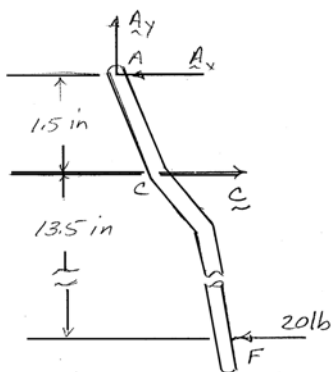
### PROBLEM 6.145

The garden shears shown consist of two blades and two handles. The two handles are connected by pin *C* and the two blades are connected by pin *D*. The left blade and the right handle are connected by pin *A*; the right blade and the left handle are connected by pin *B*. Determine the magnitude of the forces exerted on the small branch *E* when two 20-lb forces are applied to the handles as shown.

### SOLUTION

Note: By symmetry the vertical components of pin forces **C** and **D** are zero.

**FBD handle ACF:** (not to scale)

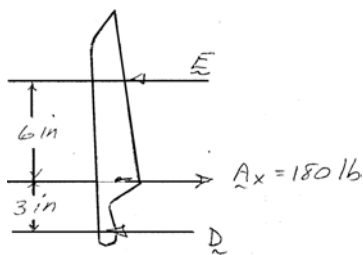


$$\uparrow \Sigma F_y = 0: A_y = 0$$

$$\curvearrowleft \Sigma M_C = 0: (13.5 \text{ in.})(20 \text{ lb}) - (1.5 \text{ in.})A_x = 0 \quad A_x = 180 \text{ lb}$$

$$\rightarrow \Sigma F_x = 0: C - A_x - 20 \text{ lb} = 0 \quad C = (180 + 20) \text{ lb} = 200 \text{ lb}$$

**FBD blade DE:**



$$\curvearrowleft \Sigma M_D = 0: (9 \text{ in.})E - (3 \text{ in.})(180 \text{ lb}) = 0$$

$$E = 60.0 \text{ lb} \blacktriangleleft$$