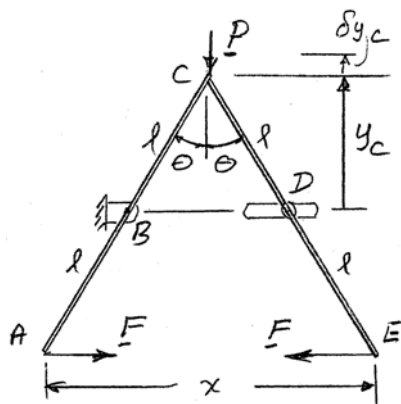


PROBLEM 10.31

Solve Problem 10.30 assuming that force **P** is moved to **C** and acts vertically downward.

SOLUTION



$$y_C = l \cos \theta, \quad \delta y_C = -l \sin \theta \delta \theta$$

Spring:

$$\text{Unstretched length} = 2l$$

$$x = 2(2l \sin \theta) = 4l \sin \theta$$

$$\delta x = 4l \cos \theta \delta \theta$$

$$F = k(x - 2l)$$

$$F = k(4l \sin \theta - 2l)$$

Virtual Work:

$$\delta U = 0: -P \delta y_C - F \delta x$$

$$-P(-l \sin \theta \delta \theta) - k(4l \sin \theta - 2l)(4l \cos \theta \delta \theta) = 0$$

$$P \sin \theta - 8kl(2 \sin \theta - 1) \cos \theta = 0$$

or

$$\frac{P}{8kl} = (2 \sin \theta - 1) \frac{\cos \theta}{\sin \theta}$$

With

$$l = 200 \text{ mm}, k = 300 \text{ N/m}, \text{ and } P = 160 \text{ N}$$

$$\frac{(160 \text{ N})}{8(300 \text{ N/m})(0.2)} = (2 \sin \theta - 1) \frac{\cos \theta}{\sin \theta}$$

or

$$(2 \sin \theta - 1) \frac{\cos \theta}{\sin \theta} = \frac{1}{3}$$

Solving numerically,

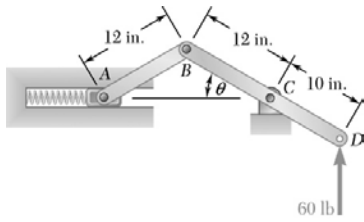
$$\theta = 39.65^\circ$$

and

$$\theta = 68.96^\circ$$

$$\theta = 39.7^\circ \blacktriangleleft$$

$$\text{and } \theta = 69.0^\circ \blacktriangleleft$$



PROBLEM 10.32

For the mechanism shown, block A can move freely in its guide and rests against a spring of constant 15 lb/in. that is undeformed when $\theta = 45^\circ$. For the loading shown, determine the value of θ corresponding to equilibrium.

SOLUTION

First note $y_D = 10 \sin \theta \text{ (in.)}$

Then $\delta y_D = 10 \cos \theta \delta \theta \text{ (in.)}$

Also $x_A = 2(12 \cos \theta) \text{ in.}$

Then $(x_A)_0 = (24 \text{ in.}) \cos 45^\circ$

and $\delta x_A = -24 \sin \theta \delta \theta \text{ (in.)}$

With $\delta \theta < 0$: Virtual Work: $\delta U = 0: (60 \text{ lb}) \delta y_D - F_{SP} |\delta x_A| = 0$

where

$$F_{SP} = k [x_A - (x_A)_0]$$

$$= (15 \text{ lb/in.}) (24 \cos \theta - 24 \cos 45^\circ) (\text{in.})$$

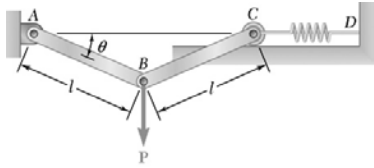
$$= (360 \text{ lb}) (\cos \theta - \cos 45^\circ)$$

Then $(60)(10 \cos \theta \delta \theta) - [360(\cos \theta - \cos 45^\circ)](24 \sin \theta \delta \theta) = 0$

or $5 - 72 \tan \theta (\cos \theta - \cos 45^\circ) = 0$

Solving numerically, $\theta = 15.03^\circ$ and $\theta = 36.9^\circ \blacktriangleleft$

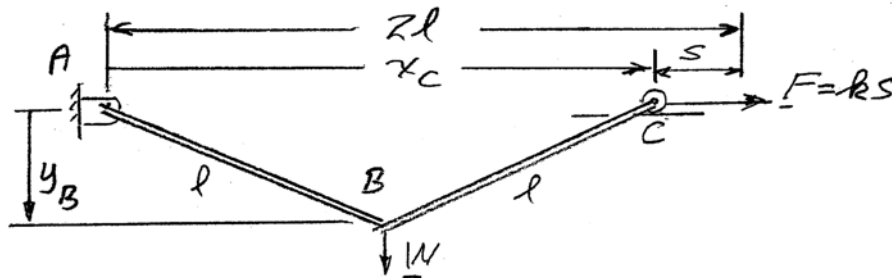
PROBLEM 10.33 AND 10.34



10.33: A force \mathbf{P} of magnitude 150 lb is applied to the linkage at B . The constant of the spring is 12.5 lb/in., and the spring is unstretched when AB and BC are horizontal. Neglecting the weight of the linkage and knowing that $l = 15$ in., determine the value of θ corresponding to equilibrium.

10.34: A vertical force \mathbf{P} is applied to the linkage at B . The constant of the spring is k , and the spring is unstretched when AB and BC are horizontal. Neglecting the weight of the linkage, derive an equation in θ , P , l , and k that must be satisfied when the linkage is in equilibrium.

SOLUTION



$$x_C = 2l \cos \theta \quad \delta x_C = -2l \sin \theta \delta \theta$$

$$y_B = l \sin \theta \quad \delta y_B = l \cos \theta \delta \theta$$

$$F = ks = k(2l - x_C) = 2kl(1 - \cos \theta)$$

Virtual Work:

$$\delta U = 0: F \delta x_C + W \delta y_B = 0$$

$$2kl(1 - \cos \theta)(-2l \sin \theta \delta \theta) + W(l \cos \theta \delta \theta) = 0$$

$$4kl^2(1 - \cos \theta) \sin \theta = Wl \cos \theta$$

or

$$(1 - \cos \theta) \tan \theta = \frac{W}{4kl}$$

Problem 10.33: Given:

$$l = 0.3 \text{ m}, \quad W = 600 \text{ N}, \quad k = 2500 \text{ N/m}$$

Then

$$(1 - \cos \theta) \tan \theta = \frac{600 \text{ N}}{4(2500 \text{ N/m})(0.3 \text{ m})}$$

or

$$(1 - \cos \theta) \tan \theta = 0.2$$

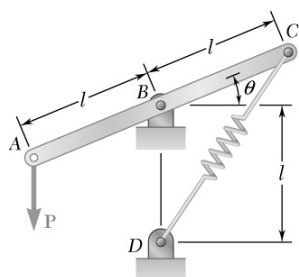
Solving numerically,

$$\theta = 40.22^\circ$$

$$\theta = 40.2^\circ \quad \blacktriangleleft$$

Problem 10.34: From above

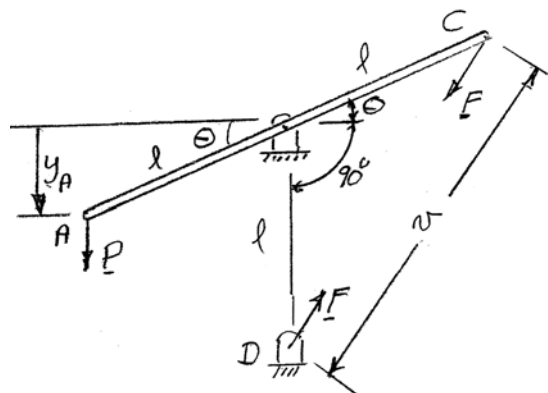
$$(1 - \cos \theta) \tan \theta = \frac{W}{4kl} \quad \blacktriangleleft$$



PROBLEM 10.35

Knowing that the constant of spring CD is k and that the spring is unstretched when rod ABC is horizontal, determine the value of θ corresponding to equilibrium for the data indicated.
 $P = 150 \text{ lb}$, $l = 30 \text{ in.}$, $k = 40 \text{ lb/in.}$

SOLUTION



$$y_A = l \sin \theta$$

$$\delta y_A = l \cos \theta \delta \theta$$

Spring: $v = CD$

Unstretched when $\theta = 0$

so that $v_0 = \sqrt{2}l$

For θ :

$$v = 2l \sin \left(\frac{90^\circ + \theta}{2} \right)$$

$$\delta v = l \cos \left(45^\circ + \frac{\theta}{2} \right) \delta \theta$$

Stretched length:

$$s = v - v_0 = 2l \sin \left(45^\circ + \frac{\theta}{2} \right) - \sqrt{2}l$$

Then

$$F = ks = kl \left[2 \sin \left(45^\circ + \frac{\theta}{2} \right) - \sqrt{2} \right]$$

Virtual Work:

$$\delta U = 0: P \delta y_A - F \delta v = 0$$

$$Pl \cos \theta \delta \theta - kl \left[2 \sin \left(45^\circ + \frac{\theta}{2} \right) - \sqrt{2} \right] l \cos \left(45^\circ + \frac{\theta}{2} \right) \delta \theta = 0$$

or

$$\begin{aligned} \frac{P}{kl} &= \frac{1}{\cos \theta} \left[2 \sin \left(45^\circ + \frac{\theta}{2} \right) \cos \left(45^\circ + \frac{\theta}{2} \right) - \sqrt{2} \cos \left(45^\circ + \frac{\theta}{2} \right) \right] \\ &= \frac{1}{\cos \theta} \left[2 \sin \left(45^\circ + \frac{\theta}{2} \right) \cos \left(45^\circ + \frac{\theta}{2} \right) \cos \theta - \sqrt{2} \cos \left(45^\circ + \frac{\theta}{2} \right) \right] \\ &= 1 - \sqrt{2} \frac{\cos \left(45^\circ + \frac{\theta}{2} \right)}{\cos \theta} \end{aligned}$$

PROBLEM 10.35 CONTINUED

Now, with $P = 150$ lb, $l = 30$ in., and $k = 40$ lb/in.

$$\frac{(150 \text{ lb})}{(40 \text{ lb/in.})(30 \text{ in.})} = 1 - \sqrt{2} \frac{\cos\left(45^\circ + \frac{\theta}{2}\right)}{\cos \theta}$$

or

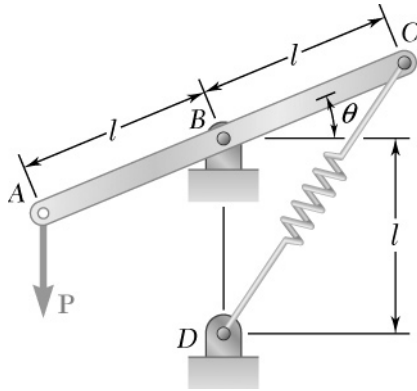
$$\frac{\cos\left(45^\circ + \frac{\theta}{2}\right)}{\cos \theta} = 0.61872$$

Solving numerically,

$$\theta = 17.825^\circ$$

$$\theta = 17.83^\circ \blacktriangleleft$$

PROBLEM 10.36



Knowing that the constant of spring CD is k and that the spring is unstretched when rod ABC is horizontal, determine the value of θ corresponding to equilibrium for the data indicated. $P = 600 \text{ N}$, $l = 800 \text{ mm}$, $k = 4 \text{ kN/m}$.

SOLUTION

From the analysis of Problem 10.35, we have

$$\frac{P}{kl} = 1 - \sqrt{2} \frac{\cos\left(45^\circ + \frac{\theta}{2}\right)}{\cos\theta}$$

With $P = 600 \text{ N}$, $l = 800 \text{ mm}$, and $k = 4 \text{ kN/m}$

$$\frac{(600 \text{ N})}{(4000 \text{ N/m})(0.8 \text{ m})} = 1 - \sqrt{2} \frac{\cos\left(45^\circ + \frac{\theta}{2}\right)}{\cos\theta}$$

or

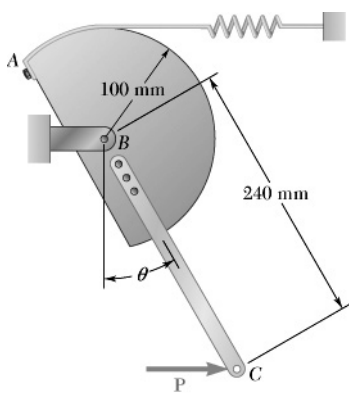
$$\frac{\cos\left(45^\circ + \frac{\theta}{2}\right)}{\cos\theta} = 0.57452$$

Solving numerically,

$$\theta = 30.98^\circ$$

$$\theta = 31.0^\circ \blacktriangleleft$$

PROBLEM 10.37



A horizontal force \mathbf{P} of magnitude 160 N is applied to the mechanism at C . The constant of the spring is $k = 1.8 \text{ kN/m}$, and the spring is unstretched when $\theta = 0$. Neglecting the mass of the mechanism, determine the value of θ corresponding to equilibrium.

SOLUTION

Have

$$s = r\theta \quad \delta s = r\delta\theta$$

$$F = ks = kr\theta$$

and

$$x_C = l \sin \theta$$

$$\delta x_C = l \cos \theta \delta \theta$$

Virtual Work:

$$\delta U = 0: P \delta x_C - F \delta s = 0$$

$$Pl \cos \theta \delta \theta - kr\theta(r\delta\theta) = 0$$

or

$$\frac{Pl}{kr^2} = \frac{\theta}{\cos \theta}$$

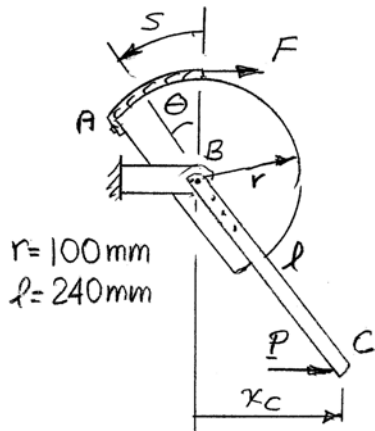
$$\frac{(160 \text{ N})(0.24 \text{ m})}{(1800 \text{ N/m})(0.1 \text{ m})^2} = \frac{\theta}{\cos \theta}$$

$$2.1333 = \frac{\theta}{\cos \theta}$$

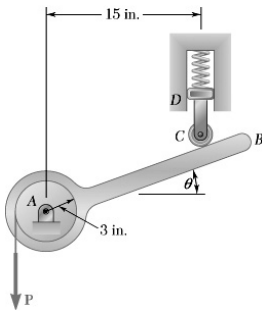
Solving numerically,

$$\theta = 1.054 \text{ rad} = 60.39^\circ$$

$$\theta = 60.4^\circ \blacktriangleleft$$

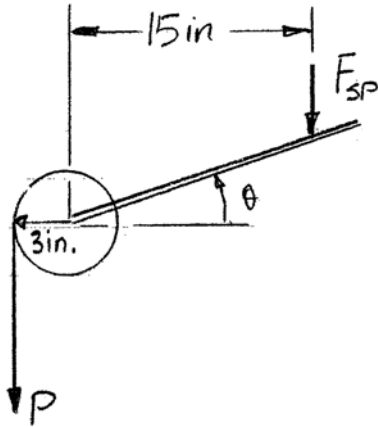


PROBLEM 10.38



A cord is wrapped around drum *A* which is attached to member *AB*. Block *D* can move freely in its guide and is fastened to link *CD*. Neglecting the weight of *AB* and knowing that the spring is of constant 4 lb/in. and is undeformed when $\theta = 0$, determine the value of θ corresponding to equilibrium when a downward force **P** of magnitude 96 lb is applied to the end of the cord.

SOLUTION



Have

$$y_C = 15 \tan \theta \text{ (in.)}$$

Then

$$\delta y_C = 15 \sec^2 \theta \delta \theta \text{ (in.)}$$

Virtual Work:

$$\delta U = 0: P \delta s_P - F_{SP} \delta y_C = 0$$

where

$$\delta s_P = (3 \text{ in.}) \delta \theta$$

and

$$F_{SP} = k y_C$$

$$= (4 \text{ lb/in.})(15 \text{ in.}) \tan \theta$$

$$= 60 \tan \theta \text{ (lb)}$$

$$\text{Then } (96 \text{ lb})(3 \text{ in.}) \delta \theta - [(60 \tan \theta) \text{ lb}][15 \sec^2 \theta \delta \theta \text{ in.}] = 0$$

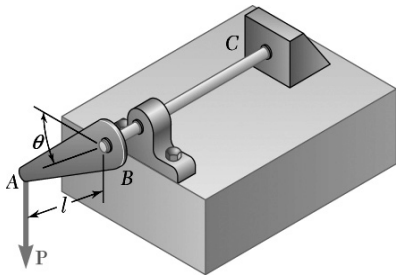
or

$$3.125 \tan \theta \sec^2 \theta = 1$$

Solving numerically,

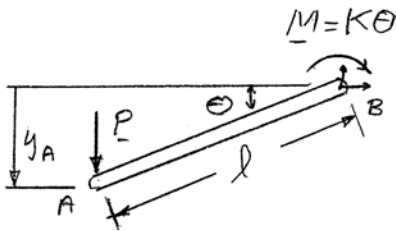
$$\theta = 16.41^\circ \blacktriangleleft$$

PROBLEM 10.39



The lever AB is attached to the horizontal shaft BC which passes through a bearing and is welded to a fixed support at C . The torsional spring constant of the shaft BC is K ; that is, a couple of magnitude K is required to rotate end B through 1 rad. Knowing that the shaft is untwisted when AB is horizontal, determine the value of θ corresponding to the position of equilibrium when $P = 400$ lb, $l = 10$ in., and $K = 150$ lb · ft/rad.

SOLUTION



Have

$$y_A = l \sin \theta$$

$$\delta y_A = l \cos \theta \delta \theta$$

Virtual Work:

$$\delta U = 0: P \delta y_A - M \delta \theta = 0$$

$$Pl \cos \theta \delta \theta - K \theta \delta \theta = 0$$

or

$$\frac{\theta}{\cos \theta} = \frac{Pl}{K} \quad (1)$$

With $P = 400$ lb, $l = 10$ in., and $K = 150$ lb · ft/rad

$$\frac{\theta}{\cos \theta} = \frac{(400 \text{ lb}) \left(\frac{10 \text{ in.}}{12 \text{ in./ft}} \right)}{150 \text{ lb} \cdot \text{ft/rad}}$$

or

$$\frac{\theta}{\cos \theta} = 2.2222$$

Solving numerically,

$$\theta = 61.25^\circ$$

$$\theta = 61.2^\circ \blacktriangleleft$$

PROBLEM 10.40

Solve Problem 10.39 assuming that $P = 1.26$ kips, $l = 10$ in., and $K = 150$ lb · ft/rad. Obtain answers in each of the following quadrants: $0 < \theta < 90^\circ$, $270^\circ < \theta < 360^\circ$, and $360^\circ < \theta < 450^\circ$.

SOLUTION

Using Equation (1) of Problem 10.39 and

$$P = 1.26 \text{ kip}, l = 10 \text{ in.}, \text{ and } K = 150 \text{ lb} \cdot \text{ft/rad}$$

we have

$$\frac{\theta}{\cos \theta} = \frac{(1260 \text{ lb}) \left(\frac{10 \text{ in.}}{12 \text{ in./ft}} \right)}{150 \text{ lb} \cdot \text{ft/rad}}$$

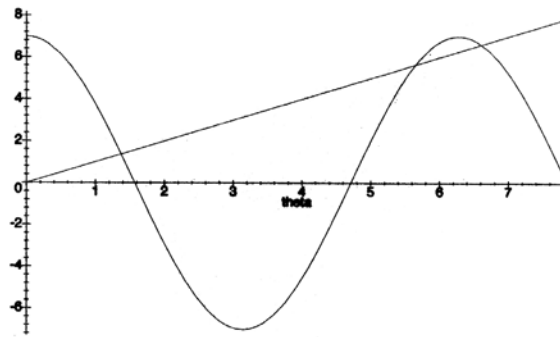
or

$$\frac{\theta}{\cos \theta} = 7 \quad \text{or} \quad \theta = 7 \cos \theta \quad (1)$$

The solutions to this equation can be shown graphically using any appropriate graphing tool, such as Maple, with the command: `plot({theta, 7 * cos(theta)}, t = 0..5 * Pi/2);`

Thus, we plot $y = \theta$ and $y = 7 \cos \theta$ in the range

$$0 \leq \theta \leq \frac{5\pi}{2}$$



We observe that there are three points of intersection, which implies that Equation (1) has three roots in the specified range of θ .

$$0 \leq \theta \leq 90^\circ \left(\frac{\pi}{2} \right); \quad \theta = 1.37333 \text{ rad}, \quad \theta = 78.69^\circ \quad \theta = 78.7^\circ \blacktriangleleft$$

$$270 \leq \theta \leq 360^\circ \left(\frac{3\pi}{2} \leq \theta \leq 2\pi \right); \quad \theta = 5.65222 \text{ rad}, \quad \theta = 323.85^\circ \quad \theta = 324^\circ \blacktriangleleft$$

$$360 \leq \theta \leq 450^\circ \left(2\pi \leq \theta \leq \frac{5\pi}{2} \right); \quad \theta = 6.61597 \text{ rad}, \quad \theta = 379.07^\circ \quad \theta = 379^\circ \blacktriangleleft$$