

B-bar METHOD

Why B-bar?

- The selective - reduced integration procedure can help us avoid volumetric locking for incompressible material, without the effect of "hour-glassing"
- While the method is very efficient for isotropic elastic materials, it may not be possible case to extend the SRI to the general case where, for example, the volumetric and deviatoric parts of the constitutive model are coupled
- For this reason, another method, called the B-bar method (Hughes, 1980), is commonly employed ~~for~~ in lieu of SRI
- The B-bar method is equivalent to SRI for isotropic elastic materials, but it can also be used for general linearly elastic or inelastic material
- Like selective reduce integration, the B-bar method works by treating the volumetric and deviatoric parts of the stiffness matrix separately.
- Instead of separating the volumetric integral into two parts, however, the B-bar method modifies the definition of the strain in the element.
- In this method, shear strain is calculated with full integration as normal FEM; however, volumetric strain is calculated with one order lower than the standard of FEM.

Formulation:

- According to small strain theory, total strain is the summation of shear strain and volumetric strain:

$$\epsilon_{ij} = e_{ij} + \frac{\epsilon_{KK}}{3} \delta_{ij}$$

in which e_{ij} is shear strain; $\epsilon_{KK} \delta_{ij}$: volumetric strain

- We replace this volumetric strain by:

$$w = \frac{1}{V_{el}} \int_{V_{el}} \epsilon_{KK} dV$$

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Formulation (continued)

- Total strain is rewritten:

$$\bar{\epsilon}_{ij} = \epsilon_{ij} + \frac{w}{2} \delta_{ij}$$

Plane strain \rightarrow 3D

- To implement this to FEM framework, a new \bar{B} matrix is defined so that: $\bar{\epsilon} = \bar{B} u^{el}$

Why \bar{B} , but B : N_1, N_2, N_3 only?

$$\frac{\partial \bar{N}^a}{\partial X_j} = \frac{1}{V^{el}} \int_{V^{el}} \frac{\partial N^a}{\partial X_j} dV$$

$$\bar{B} = \begin{bmatrix} \frac{\partial N_1}{\partial x_1} & 0 & \frac{\partial N_2}{\partial x_1} & 0 & \frac{\partial N_3}{\partial x_1} & 0 \\ 0 & \frac{\partial N_1}{\partial x_2} & 0 & \frac{\partial N_2}{\partial x_2} & 0 & \frac{\partial N_3}{\partial x_2} \\ \frac{\partial N_1}{\partial x_2} & \frac{\partial N_1}{\partial x_1} & \frac{\partial N_2}{\partial x_2} & \frac{\partial N_2}{\partial x_1} & \frac{\partial N_3}{\partial x_2} & \frac{\partial N_3}{\partial x_1} \end{bmatrix}$$

Plane strain 2D \rightarrow 2
3D : 3

$$\frac{1}{V^{el}} \begin{bmatrix} \frac{\partial N_1}{\partial x_1} & \frac{\partial N_1}{\partial x_2} & \frac{\partial N_2}{\partial x_1} & \frac{\partial N_2}{\partial x_2} & \frac{\partial N_3}{\partial x_1} & \frac{\partial N_3}{\partial x_2} \\ \frac{\partial N_1}{\partial x_1} & \frac{\partial N_1}{\partial x_2} & \frac{\partial N_2}{\partial x_1} & \frac{\partial N_2}{\partial x_2} & \frac{\partial N_3}{\partial x_1} & \frac{\partial N_3}{\partial x_2} \\ 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} + \frac{1}{V^{el}} \begin{bmatrix} \frac{\partial N_1}{\partial x_1} & \frac{\partial N_1}{\partial x_2} & \frac{\partial N_2}{\partial x_1} & \frac{\partial N_2}{\partial x_2} & \frac{\partial N_3}{\partial x_1} & \frac{\partial N_3}{\partial x_2} \\ \frac{\partial N_1}{\partial x_1} & \frac{\partial N_1}{\partial x_2} & \frac{\partial N_2}{\partial x_1} & \frac{\partial N_2}{\partial x_2} & \frac{\partial N_3}{\partial x_1} & \frac{\partial N_3}{\partial x_2} \\ 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}$$

- Finally, stiffness matrix is calculated as follow

$$K^{el} = \int_{V^{el}} \bar{B}^T D B dV$$

- Notice that ϵ_v in the element is everywhere equal to its mean value in B-bar method.

- In this study, we used 4-nodes isoparametric element in which ϵ_v is integrated with only 1 Gauss point, and shear strain is fully integrated with 4-Gauss points.
- In \bar{B} method, the value of \bar{B}_v at any location (ξ, η) in the interior of the element, is set equal to the value of the specific array at the origin of the element $\xi=0, \eta=0$