Kinematic state estimation for a Mars rover

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SUMMARY

The Jet Propulsion Laboratory (JPL) is embarking on a series of exploration missions to Mars. Long-range rovers with several science instruments on board will traverse many kilometers on the surface. This paper describes some of the research underway at JPL as they pertain to navigation of these long-range rovers. A brief description is provided of the Sojourner rover which recently completed a short-range, near-lander Mars mission. A research rover called Rocky-7 under development at JPL implements a number of new technologies for long-range Mars traverses. The kinematic configuration of this vehicle is described as is tile sensor set. The paper then focuses on state estimation techniques used on-board Rocky-7. A brief review of roverrelated state-estimation work is presented and an Extended Kalman Filter framework is introduced. This includes process models for translation, small-angle/quaternion based rotations, gyro bias and wheel contact as well as observation models for a gyro, sun-sensor, and accelerometer. A novel slip constraint technique is developed to allow incorporation into the filter of the highly non-linear kinematics resulting from the rover's rocker-bogey suspension mechanism and wheel-to-ground contact point variations. Simulation results are discussed to allow detailed comparisons with ground-truth data and experimental results from Rocky-7 are presented.

KEYWORDS: Mars rover; Navigation; Estimation techniques; Process models; Kinematic configuration.

1. INTRODUCTION

Managed for the National Aeronautics and Space Administration (NASA) by the California Institute of Technology, the Jet Propulsion Laboratory (JPL) is the lead United States center for robotic exploration of the solar system. Recently a number of robotic missions to Mars have been undertaken. Mars Global Surveyor is orbiting Mars and sending very high resolution images of the red planet. Mars Pathfinder and its Sojourner Rover have finished their mission after a spectacular landing and the first use of a roving vehicle on Mars.

The Surveyor Program consists of a series of missions that will explore Mars and bring samples to Earth. Current plans call for launching two spacecraft at each launch opportunity every 26 months, typically an orbiter and a lander. The first of these is the '98 mission which will have a lander with a robotic arm. The robotic arm will be used to scoop soil and perform *in-situ* analysis. In '01 a lander with an arm and a copy of the Sojourner rover is expected to be used for near-lander science operations. The '03 and '05

missions will have long-range rovers with several science instruments on-board. These rovers will explore Mars by traversing many kilometers on the surface. In addition they will be drilling rocks and caching samples for return to Earth. The research described in this paper addresses some of the key navigation technologies required to field long-range rovers on Mars.

1.1. The Sojourner rover

We briefly summarize the capability of the current state-ofart in planetary rovers by describing the Sojourner rover. This rover has six wheels, each 13 centimeters in diameter and made of aluminum with stainless-steel treads and cleats. A rocker-bogey suspension system allows each wheel to move up and down independently of all the others. Sojourner is capable of scaling a rock of more than 20 centimeters height.

A laser striping system in conjunction with multiple monocular cameras is used to detect obstacles immediately in front of the rover. The vehicle is steered autonomously to avoid obstacles while continuing to attempt to achieve the commanded goal location. Tilt sensors allow the rover to be autonomously halted if it is dangerously close to tipping over.

While stopped, the rover updates its estimate of distance traveled and heading using the averaged wheel odometry and a single on-board heading gyro. This provides an estimate of progress to the goal location. The lander is also a crucial element for the navigation of the rover. Its stereo cameras obtain panoramic images of the area surrounding the rover which are then used by mission operators on Earth to periodically locate the rover as well as determine way-points for subsequent rover movements. As a consequence, rover operations are limited to the near vicinity of the lander.

1.2. Long-range rovers

Unlike the Sojourner rover, which traversed a total of 104 meters in the Pathfinder mission, long-range rovers could traverse to goals many kilometers from the lander. The scenario for long-range rover operations consists of traversing in the commanded direction to a goal region while periodically (e.g., every 100 m to 200 m) transmitting panoramic images to the Earth via a relay satellite. The operators on Earth provide new commands to either continue the traverse, select a new goal, or if the site is of interest to scientists, issue various site survey related commands.

All navigation is without the benefit of a lander and must therefore be highly autonomous. The rover must be able to

reliably traverse long distances, avoiding obstacles that are not visible to the operators on Earth in orbiter, lander descent, or lander panoramic images. At a given location, the rover must be able to reliably achieve science goals involving pointing and placement of science instruments on targets designated in returned imagery. Furthermore, the rover should be able to locate itself within the area of operations without the benefit of an external view provided by a lander stereo camera.

In order to achieve these goals, the long-range rover research program at JPL has focussed on:

- Implementation of a research rover vehicle called Rocky-7 (the name being derived from the continuing use of the Rocker/Bogey suspension system at JPL).
- Utilizing stereo-imaging to obtain a dense range map of the rover surroundings for use ill obstacle avoidance.
- Integrating a celestial sensor (*i.e.*, a sun sensor) to determine the rover's orientation.
- Developing a rover-attached deployable mast with mounted stereo-cameras to obtain high-vantage panoramic images of the area surrounding the rover.
- Developing advanced path-planning techniques in conjunction with mast imagery derived obstacle maps to navigate dense rock fields.
- Utilizing mast imagery derived range maps and range map matching techniques to periodically re-establish the location of the rover within a work area.
- Integrating image feature extraction and tracking methods to serve as a means of visual odometry during rover motion.
- Establishing a state estimation framework to incorporate the information from all of the sensors.

Of these items the last one, namely the state estimation framework, forms the focus of this paper. A key objective is a full state (position and attitude) estimator based on a sensor set consisting of a sun-sensor, gyros, a 3-axis accelerometer and the vehicle kinematic sensors. Of some interest here is to achieve this with a reduction in the electro-mechanical complexity of the vehicle by reducing the number of gyros to a single heading rate sensor instead of a full 3-axis implementation.

The aim is to improve the precision of the odometry estimate by using the full kinematics of the rocker/bogey mechanisms of the rover as it traverses undulating/bumpy terrain. This is considerably more complex than a kinematically simpler vehicle operating on a fiat indoor environment. In addition to providing a backup to visionbased methods in regions of low visual content, improving the precision of the odometry allows vision-based estimation techniques to be improved. Visual feature tracking and range map matching methods can be layered on top of the baseline estimator with increased robustness to failure, reduced search/computation costs, and less frequent use of mast deployments and related imaging operations. The Kalman filtering framework adopted also provides a natural Bayesian means of combining any visually based motion estimates into the full state estimate. Another objective is to develop a sensing/estimation framework in which sensors



Fig. 1. Rocky-7 side view.

can be enabled/disabled based on monitoring of filter residuals, and mode switching between different estimator banks is triggered by observability conditions.

1.3. The Rocky-7 research platform

In this section we describe the Rocky-7 rover configuration shown in Figure 1, and detail the components relevant to the navigation of the vehicle.

Like Sojourner, the wheel diameter of Rocky-7 is 13 cm. The mobility system is a modification of the Rocker-Bogey design used in previous rovers at JPL. It consists of two rockers hinged to the sides of the main body. Each rocker has a steerable wheel at one end and a smaller bogey at the other end. Unlike its predecessors Rocky-3 and Rocky-4 (and the Sojourner flight rover) that have four steerable wheels, Rocky-7 has only two. Here too the emphasis is on investigating reduced number of actuators to minimize system electro-mechanical complexity.

The sun-sensor used on Rocky-7 is the Lockheed-Martin Wide Angle Sun Sensor (WASS) with a 160 degree field-ofview providing sun azimuth and elevation in its coordinate frame. It is mounted on the solar panel of Rocky-7. The rate gyro is the QRS-11 vibratory gyro from Systron-Donner, and the accelerometers are from Lucas-Shaevitz mounted in a 3-axis cage. Internal angles of the rover mechanism are read by potentiometers. Wheel drive and steering angles are read by optical encoders.

The rover is equipped with seven CCD cameras, two at each end, for the hazard avoidance system and three on the mast (2 for stereo, and I for close-up viewing). The hazard avoidance system uses passive stereo-vision with a pair of cameras with wide-angle lenses that allow detection of rocks and other hazards extending from near the rover wheels to about 1.5 meters distance.

1.4. Previous rover state estimation work

We briefly describe some representative related work in the area of mobile robot state estimation. Since Global Positioning System (GPS) based methods are not currently applicable to a vehicle on Mars, we have excluded them from our discussion.

A number of efforts have concentrated on the localization of the mobile robot based on external sellsor cues. For example, beacon-based localization of a mobile robot has been considered by Leonard¹ in which an Extended Kalman Filter (EKF) is employed to match environment observations to a map. Matthies² used a Kalman Filtering approach to track stereo vision features and obtain vehicle motion. Baumgartner and Skaar³ estimate a vehicle's position and orientation based on visual cues in discrete locations within a structured environment combined with sensed wheel estimates by an EKF. Olson⁴ utilizes range map matching to periodically localize a mobile vehicle.

Other efforts have aimed at fusing inertial navigation sensors with odometry. Borenstein and Feng⁵ develop a technique called gyrodometry where gyro data is only used when the gyro and odometry estimates differ, such as when traversing a bump. Barshan and Durrant-Whyte⁶ also use inertial sensors in an EKF to estimate position. Vaganay and Aldon⁷ utilize accelerometers and gyros for vehicle attitude estimation. Fuke and Krotkov⁸ utilize gyros and accelerometers together with odometry to estimate vehicle attitude estimates. Their filter allows the accelerometer signal to dominate at low frequencies and the gyro to dominate at higher frequencies. The combination of gyro and sun-sensor data for vehicle attitude estimation is addressed by Roumeliotis and Bekey.⁹

Kinematic techniques include an effort by Kim, et. al. 10 to extend a dead-reckoning formula for a two-wheeled mobile robot motion on a known curved surface. Slip modeling for a vehicle operating in the plane is considered by Madhavan,

et. al.¹¹ Here a random walk model for a slip angle parameter is introduced for estimating the motion of a truck with planar articulation elements.

As we see, many vehicle state estimation methods have been applied to vehicles with relatively simple kinematics and no slip, operating on mostly flat terrain, and using expensive external sensor cues such as that provided by vision. In this paper, we attempt to develop new techniques suitable for use on kinematically complex vehicles traversing highly uneven terrain, with explicit modeling of slip, and using a minimal complement of basic sensors.

2. ROVER KINEMATICS MODEL

In this section, we discuss the model used for rover state estimation. This consists of a simple state-space model to smoothly propagate rover position and attitude, a rover kinematic contact model to describe wheel interactions with the ground, and measurement models for the various sensors.

2.1. Coordinate frames and variables

Coordinate frames and variables are as defined in Figure 2. The unconstrained rover's degrees-of-freedom (dof's) are seen to be three translational, three rotational, three internal $(\gamma_0, \gamma_1, \gamma_2)$, two steering (λ_1, λ_2) , and six drive (ψ_1, \ldots, ψ_6) . Contact interactions at each wheel constrain these dof's to result in the rover typically having two translational dof's (x, y) and one angular dof (heading) when in full contact with the ground.

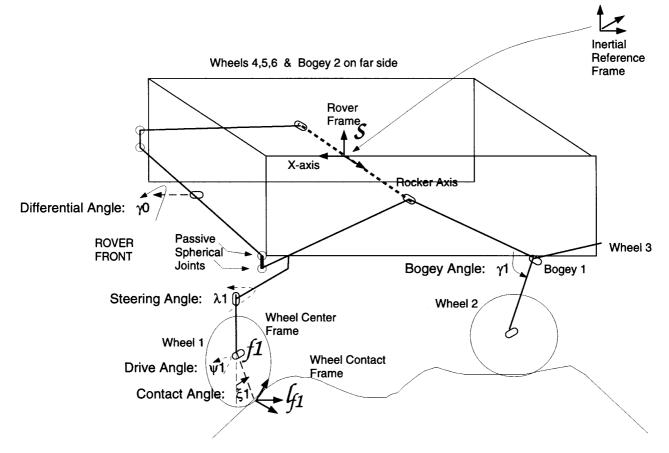


Fig. 2. Rocky-7 kinematics.

2.2. Process model

The process model sets up the differential equations governing the system. It defines the state vector and associated state equations.

2.2.1 Translation state equation. A simple kinematic model is provided for the translation of the system.

$$\begin{bmatrix} I_{\dot{x}} \\ \dot{v}_{p} \end{bmatrix} = \begin{bmatrix} R(q)(v_{c} + v_{p}) \\ A_{v}v_{p} \end{bmatrix} + \begin{bmatrix} 0 \\ n_{v} \end{bmatrix}$$
 (1)

The translational position ${}^{J}x \in R^3$, of the rover frame is resolved in the inertial frame. However, the velocities v_c , $v_p \in R^3$, are resolved in the body frame. The body velocity v_c is given by the sum of two velocity terms $v_c = v_c + v_p$. The v_c is the prescribed velocity that the rover is attempting to follow as generated by the on-board controller. Typically this would be the v_c and v_c components of the body translational velocity. The v_c represents a perturbation velocity which is damped out by the damping matrix v_c and driven by an acceleration noise v_c . The matrix v_c is the rotation matrix of the rover attitude as a function of v_c , the attitude quaternion.

2.2.2 Attitude state equation. A purely kinematic model is also provided for the attitude of the system.

$$\begin{bmatrix} \dot{q} \\ \dot{\omega}_p \end{bmatrix} = \begin{bmatrix} \frac{1}{2} \Omega(\omega_c + \omega_p) q \\ A_\omega \omega_p \end{bmatrix} + \begin{bmatrix} 0 \\ n_\omega \end{bmatrix}$$
 (2)

All angular variables are resolved in body coordinate with $q \in R^4$ being the attitude quaternion. The total body angular velocity ω is the sum of two terms $\omega = \omega_c + \omega_p$, with ω_c , $\omega_p \in R^3$. The ω_c is the prescribed rate that the rover is attempting to follow as generated by the on-board controller. Typically this would be the z-axis component of the body angular velocity. The ω_p then represents a perturbation angular velocity which is damped out by the damping matrix A_ω and driven by an angular acceleration noise n_ω . The terms in quaternion evolution equation are given by:

$$\Omega(\boldsymbol{\omega}) \stackrel{\triangle}{=} \left[\begin{array}{cc} -\boldsymbol{\omega}^{\times} & \boldsymbol{\omega} \\ -\boldsymbol{\omega}^{T} & 0 \end{array} \right]; \quad \boldsymbol{\omega}^{\times} \stackrel{\triangle}{=} \left[\begin{array}{ccc} 0 & -\boldsymbol{\omega}_{3} & \boldsymbol{\omega}_{2} \\ \boldsymbol{\omega}_{3} & 0 & -\boldsymbol{\omega}_{1} \\ -\boldsymbol{\omega}_{2} & \boldsymbol{\omega}_{1} & 0 \end{array} \right] (3)$$

2.2.3 Small angle evolution. The basic idea is to define an intermediate local model to which the standard continuous-discrete Extended Kalman Filter (EKF) equations can be applied. Let the quaternion q be given in terms of angle/axis variables (Θ, u) , where $\Theta \in \mathcal{R}^1$ and unit vector $u \in \mathcal{R}^3$, as: 13

$$q = \begin{bmatrix} \epsilon \\ \eta \end{bmatrix} \tag{4a}$$

$$\epsilon = u \sin(\Theta/2) \tag{4b}$$

$$\eta = \cos(\Theta/2) \tag{4c}$$

Because the normalization constraint $q^Tq=1$ is not explicitly enforced in the estimator design, the quaternion q acts as an over-parameterized representation of the 3-axis attitude. In order to avoid the redundant state, a local vector angular variable $\theta \in \mathcal{R}^3$ is defined by a local linearization at the beginning of each propagation step. After the completion of the corresponding update step, the local angular variable is absorbed into the quaternion from the previous linearizations. The process is continued with subsequent linearizations. The corresponding covariance terms are maintained in terms of the local angular variable, which characterizes a ball of uncertainty about the attitude estimate.

For some $t = t_0$ and $q(t_0) = q_0$, a local angular variable θ is defined by the change of variables:

$$q(t) = q_0 \begin{bmatrix} \theta(t)/2 \\ 1 \end{bmatrix}$$
 (5)

which can be solved to give:

$$\theta(t) = 2Im\{q_0 * q\} \tag{6}$$

where " $Im\{.\}$ " is used to indicate the first three so-called imaginary components of the quaternion as given in (4a). Then from (2) and (3) the linearized equation for θ becomes:

$$\dot{\theta} = \omega - \frac{1}{2} \omega^{\times} \theta = (\omega_c + \omega_p) - \frac{1}{2} (\omega_c^{\times} + \omega_p^{\times}) \theta$$
 (7)

Note that this parameterization is linear in θ but preserves the nonlinear dependence of ω for this accuracy in θ . This is a locally valid nonlinear system which avoids difficulties associated with the redundancy of a quaternion representation of attitude. However, it is nonlinear and remains to be *further* linearized in the usual EKF sense to realize the propagation and update stages of the filter.

2.2.4 Contact state equation.

$$[\dot{\xi}] = [A_{\xi}\xi] + [n_{\xi}] \tag{8}$$

The contact point vector $\xi = \{\xi_1, \xi_2, \xi_3, \xi_4, \xi_5, \xi_6\}$ is modeled very simply as a set of one parameter contacts about the equator of each wheel, with the nominal contact position $\xi_i = 0$. In reality there is an additional off-equatorial coordinate for the contact point at each wheel, a contact rotation angle, and two parameters that describe the point on the ground. However the one parameter model suffices to capture and couple the rotational and translational velocities. The dynamics of ξ encoded in A_{ξ} ensure that the contact point eventually returns to the nominal position and thereby

ensures correct steady-state behavior in the estimator when the rover travels on slopes of constant inclination.

There are issues regarding the observability of the contact points which are not discussed here. However, the experimental results indicate generally good observability for typical rover motions.

2.2.5 Gyro bias state equation.

$$[\dot{b}] = [n_b] \tag{9}$$

The gyro bias vector $b = \{b_1, b_2, b_3\}$ is modeled as a random-walk. Normally, the periodic measurement of all three attitude components by an absolute sensor such as 3-angle sun-sensor allows the gyro biases to be estimated while the rover is in motion. However, if only a single gyro is used in conjunction with a sun-sensor that only measures 2 attitude variables, then gyro bias observability is more complex, becoming a function of the sun-angle geometry and the kinds of attitude motions being undertaken by the rover. However, since the Mars rover operations call for the rover to periodically come to a stop (every few meters), bias can be simply estimated by averaging the gyro data during these times when the rover is stopped. Therefore we choose to not incorporate this model into the state estimator at this time.

2.3. Measurement and constraint model

The measurement models relate the state variables to the sensor data available from the system. Here we consider the standard complement of sensors available to the rover. We also address the nonlinear kinematics by means of slip related constraint that we treat in a manner similar to that of a measurement.

2.3.1 Accelerometer equation. The accelerometer readings is given by:

$$[a_m] = [a + \omega^{\times} v - R^T(q)g + \omega^{\times} \omega^{\times} L^b + \dot{\omega}^{\times} L^b] + [n_a] \quad (10)$$

Here all variables are resolved in body coordinates and the accelerometer frame is assumed to be aligned with the rover frame. The term L^b is the vector from the rover frame to the accelerometer frame origin. If the acceleration a is not modeled as a state, then the accelerometer data can be used to drive the process equation in the so-called *sensor integrating form* in lieu of the expression in Equation 1:

$$\dot{v}_p = a_m - \dot{v}_c + R^T(q)g - (\omega^{\times} v_p + \omega^{\times} v_c + \omega^{\times} \omega^{\times} L^b + \dot{\omega}^{\times} L^b)$$
 (11)

2.3.2 Gyro equation.

$$[\omega_m] = [B_{\varrho}\omega + b] + [n_{\varrho}] \tag{12}$$

Here the gyro measurement ω_m gives the angular rate ω in the body frame. If a full 3-axis gyro is present the B_g is an Identity matrix. However, if only a heading gyro is present, the matrix B_g implements the one-axis gyro measurement with:

$$B_{\sigma} \stackrel{\triangle}{=} [0\ 0\ 1\] \tag{13}$$

2.3.3 Sun-sensor equation. The sun-sensor design is a wide-angle lens which projects an image of the sky on a

two-dimensional position sensing device. The output currents of this device, specifically the ratio of currents for each dimension, provides the position of the centroid of the Sun's image from the edge of the device. With the edge currents designated as $\{I_1, I_2, I_3, I_4\}$ the centroid location is given by:

$$s_m(1) = \frac{(I_1 + I_4) - (I_2 + I_3)}{I_1 + I_2 + I_3 + I_4}$$
 (14a)

$$s_m(2) = \frac{(I_1 + I_2) - (I_3 + I_4)}{I_1 + I_2 + I_3 + I_4}$$
 (14b)

The sun-sensor has optics most easily described by a fisheye lens model. In the sun-sensor frame, the sun azimuth angle is given by $atan(s_m(2), -s_m(1))$, and elevation by $\pi/2(1-\sqrt{s_m(1)^2+s_m(2)^2})$. Equivalently, we choose to consider the output of the sensor to be a function of the two independent components of the unit vector to the sun as resolved in the sun-sensor frame. This is given by:

$$[s_m] = (2/\pi) \frac{\arcsin(\sqrt{s_x^2 + s_y^2})}{\sqrt{s_x^2 + s_y^2}} \begin{bmatrix} s_x \\ s_y \end{bmatrix} + [n_s]$$
 (15)

where:

$$\begin{bmatrix} s_x \\ s_y \end{bmatrix} = \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \end{bmatrix} R^T(q) s_v \tag{16}$$

The unit vector s_v respresents the sun-vector in inertial coordinates. This vector is rotated into the sun-sensor coordinate system (which is taken to be the rover frame) by the matrix $R(q) \in R^{3\times 3}$ corresponding to the quaternion q. The noise term n_s lumps the effects of electronics noise as well as calibration errors.

2.3.4 Kinematic slip equation. There is no simple analytical formulation of the inverse kinematics map that relates wheel rotations and contact interactions to the vehicle motion. Such a map must necessarily involve the dynamics of the vehicle and is too complicated for implementation in the filter. Instead we choose to embed the easily established forward kinematics within a *constraint* that is treated as a *measurement* in the filter. We shall see that this allows a natural implementation of the full kinematics of the vehicle. It exploits the ability of the Kalman filter to perform the appropriate least-squares averaging of the action of each kinematic chain in the rover.

Each such forward kinematic chain has a component defined by sequence of links joining the rover frame to each wheel contact point, and a component given by the slip between the wheel and the ground. We introduce the notion of a slip measurement or constraint, t_{m_i} , that defines the relative 6-dof motion of the contact frame l_{f_i} (see Figure 2) on the wheel with respect to the ground. This slip is a function of the vehicle configuration, tile 6-dof vehicle velocity, the wheel-to-ground contact point location, and the joint rates associated with the kinematic chain emanating from the rover frame to the contact point.

$$t_{m_{i}} = B^{T} A d_{g_{sl_{s}}}^{-1} (\gamma, \lambda, \xi_{i}) \left(\begin{bmatrix} \upsilon_{c} + \upsilon_{p} \\ \omega_{c} + \omega_{p} \end{bmatrix} + J_{sf_{i}}^{s} (\gamma, \lambda) \begin{bmatrix} \dot{\gamma} \\ \dot{\lambda} \\ \dot{\psi} \end{bmatrix} \right) + [n_{t_{i}}]$$

$$(17)$$

The Ad_g^{-1} term is the Adjoint Operator [16] given by:

$$Ad_g^{-1} = \begin{bmatrix} R^T & -R^T p^{\times} \\ 0 & R^T \end{bmatrix}$$
 (18)

with

$$p^{\times} \stackrel{\triangle}{=} \begin{bmatrix} 0 & -p_3 & p_2 \\ p_3 & 0 & -p_1 \\ -p_2 & p_1 & 0 \end{bmatrix}$$
 (19)

The term $g_{sl_{ji}}$ represents the transformation from the rover frame to the contact point. The term $J_{sf_i}^s$ represents the Spatial Jacobian¹⁶ to each wheel center and is a function of the kinematics of the rover.

Here tile internal angles are represented by $\gamma = \{\gamma_0, \gamma_1, \gamma_2\}$, the steering angles by $\lambda = \{\lambda_1, \lambda_2\}$, the drive angles by $\psi = \{\psi_1, \psi_2, \psi_3, \psi_4, \psi_5, \psi_6\}$, and the contact point by ξ_i . The terms Ad_g^{-1} is a function of γ , λ , and ξ . However, it is not a function of the driving angles ψ because of rotational symmetry of the wheel. The term $J_{sf_i}^s$ is also not a function of the driving angles ψ and is only a function of γ and λ .

A variety of choices are possible for the *B* matrices. We choose to adopt the *B* matrix for pure rolling as given by:

$$B = \begin{pmatrix} 1 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 \end{pmatrix} \tag{20}$$

This slip constraint measurement can be decomposed into a known deterministic component and a component that is only known in a statistical sense. The deterministic component of the slip, indicated by a non-zero nominal value of t_m , is used to capture the effects of a known steering action. For example, a known rotational slip about the vertical is always present at each wheel to accommodate the yaw motion of the vehicle during a turn. Also, some transverse slip is introduced due to the nature of the nonsteered bogey wheels on a rover like Rocky-7. In this case, the bogey wheels have their rotation axis on two parallel lines longitudinally (x) offset from each other. As a consequence there is always some transverse slip even during an Ackerman based steering turn unless one is going on a straight line on fiat terrain. These deterministic slips are easy to calculate for Ackerman steered motions on flat ground and are used as approximations to the true deterministic slip even during motion over non-flat terrain.

Another deterministic slip constraint measurement can be derived from experiments. For example, over sandy terrain, a known rate of experimentally derived longitudinal (x) slip during traverses can be added as a non-zero t_m term.

A slip action that is only modeled statistically is due to tile wheel-ground interaction at each individual wheel. Consider the case when each wheel is driven by a control algorithm that attempts to maximize compliance of the wheel-ground rolling interaction, and also maximally coordinates the control effort across all the wheels. Then the slips at each wheel in the longitudinal (x) rolling direction are all independent of each other. In this case of *Uncorre*lated Slip Noise, the slip can be modeled as independent white noise with the corresponding correlation matrix having terms only on the diagonal. Now consider a rover in which each wheel is independently driven by a high-gain, "stiff" control algorithm. Then each wheel rotates to follow the set-point established by the controller and does not accommodate to any wheel-ground forces of interaction. The slips at each wheel are then just direct kinematic transformations of the deviation of the vehicle frame motion from its nominal path. The covariance matrix of this Correlated Slip Noise can be derived by transforming the process noise of the vehicle motion by the Jacobian from the vehicle frame to the wheel-to-ground contact frames, and will contain many off-diagonal terms.

Another statistically modeled slip action is due to the terrain curvature. As the vehicle makes progress on the terrain, the rocker-bogey mechanism mostly accommodates the vehicle to maintain contact over the terrain and a highly compliant wheel controller can zero out any rolling slip. However, there is usually some inevitable transverse slip at each wheel to accommodate the curvature changes of the surface at each wheel contact. This slip can be modeled as a zero-mean process with time constants and dynamics related to the rate of change of surface curvature parameters along the rover path.

In actual practice, the slip at the wheel is a combination of all of the above processes. We choose not to model all the statistically describable noise terms in all of their complexity. Instead we select a simple un-correlated slip model for our early implementations, with the covariance strengths determined by experiments. There may also be opportunities to develop a true slip related measurement using the motor current data on the drive motors but this too has not been explored in detail yet.

3. NONLINEAR STATE ESTIMATION

In this section, a nonlinear state estimation scheme is developed based on the process and measurement models defined in Section 2.2 and Section 2.3.

In this paper, from the various measurement models, we have chosen to focus on those implementing the heading gyro in conjunction with the sun-sensor. We have deliberately not considered the use of a 3-axis gyro because of our interest in minimizing the number of attitude sensors used. As the gyro and sun-sensor appear adequate in capturing attitude information we have chosen, at present, to not use the accelerometer measurements. However, they could serve the purpose of establishing long time-scale attitude varia-

tions by using a stochastic model of the forces on the rover.¹⁷ The accelerometers could also be used to determine the high-frequency positional variations of the rover using techniques similar to the Complementary Filter Method.¹⁸ However, we have chosen not to focus on this usage of the accelerometer in order to keep the focus on the nonlinear kinematic aspects of the rover state estimation problem.

We choose to model the various joint angles and corresponding rates as largely deterministic so as to avoid having to introduce these as additional states in the filter. Each of these joint variables is independent of the others and is directly observable from measurements with only some simple smoothing and differencing to derive the corresponding angles and rates.

The Kalman filter state is denoted by $X = (\theta, \omega_p, x, \upsilon_p, \xi)^T$. Note that we are using the small-angle process model defined by Equation 7. The measurement vector is denoted by: $Y = (\omega_m, s_m, t_m)^T$.

The overall nonlinear state estimation scheme for propagating the state X, covariance P, and associated quaternion state q can be outlined as follows:

- (i) For given $X_{k-1}(+)$, $P_{k-1}(+)$, $q_{k-1}(+)$ re-parameterize in terms of the local quaternion state $q_0 = q_{k-1}(+)$ to get state and measurement equations.
- (ii) Implement standard continuous-discrete EKF algorithm (cf. [12], p. 188).
 - (i) *Propagate* state and covariance from t_{k-1} to t_k : $X_{k-1}(+) \rightarrow X_k(-)$; $P_{k-1}(+) \rightarrow P_k(-)$.
 - (ii) *Update* state and covariance at t_k :

$$X_k(-) \longrightarrow X_k(+); \quad P_k(-) \longrightarrow P_k(+).$$

(iii) *Update* associated quaternion state and re-initialize local angular variable at t_k :

$$q_k(+) = q_{k-1}(+) \begin{bmatrix} \theta_k(+)/2 \\ 1 \end{bmatrix}$$
 (21a)

$$\theta_t(+) = 0 \tag{21b}$$

(iv) Proceed to next stage of propagation and updating $(k \leftarrow k+1)$.

4. EXPERIMENTS

Two sources of data are available for validating the state estimation algorithms. The first is test data from Rocky-7 operated in the JPL Mars Yard, a 15×25 meter outdoor test area that closely simulates Mars-like terrain constructed on the basis of statistical analysis of images taken by Viking Landers I and II. The second source is a high-fidelity kinematic simulator of the rover contact kinematics. ¹⁹ This simulator solves for the contact configuration for a set of closely spaced points along a specified rover path. It then uses the contact geometry and surface parameters of the wheel and ground to derive wheel motion. As such the results correspond closely to the case of highly compliant control algorithm at each wheel minimizing slip in the rolling direction.

In all cases the slip measurement/constraint knowledge made available to the estimator is taken to be nominally zero for the translational motions. In case of turning motions, a non-zero slip value is calculated for the bogey wheels based upon the nominal steering rate. Of course, the statistical models of the slip "noise" allow the estimator to optimally utilize (or discount) the value of the nominal slip knowledge.

We report results for three test setups. The first two are for a simulated Rocky-7 rover where extensive comparisons to the ground-truth data is possible. Then we show some experimental results on the Rocky-7 system.

4.1. Filter operation with simulated data

Here we demonstrate the tracking of rover position and attitude from a combination of sun-sensor, heading gyro and the wheel sensors. Two test cases are simulated. The first one corresponds to the case of a rover moving in a straight line over an undulating terrain. The second test case corresponds to a turn-in-place maneuver over the same terrain. In both experiments the Sun is assumed to be along the vector $[-1, 1, 1]\sqrt{3}$. The surface is modeled as:

$$z = 0.2 \left(\frac{0.25}{e^{5(6.25(-0.5+x)^2 + (6.25(-0.2+y)^2)}} - \frac{0.25}{e^{4(6.25(-0.5+x)^2 + 6.25(0.2+y)^2)}} + 0.5 \sin(0.75 xy) \right)$$
(22)

4.1.1 Straight-line motion. Figure 3 shows the rover at the start of a 1 meter straight-line motion over the undulating surface

Attitude sensor data is shown in Figure 4. Drive joint rates for the wheels on the left side of the vehicle are shown in Figure 5. As expected, the simulation indicates a variation in the drive rate as the wheel controllers attempt to maintain a zero slip condition. Drive joint rates on the right side of the vehicle exhibit a similar behavior and are not shown. As the steering angles are fixed during this motion they too are not plotted. Internal angles corresponding to the differential and the two bogeys are shown in Figure 6.

For the straight-line motion, the ground-truth attitude and the estimated values are shown in Figure 7. The ground-truth position and the estimated values are shown in Figure 8.

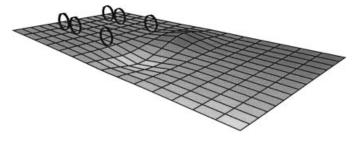


Fig. 3. Rocky-7 straight-line motion over undulating terrain.

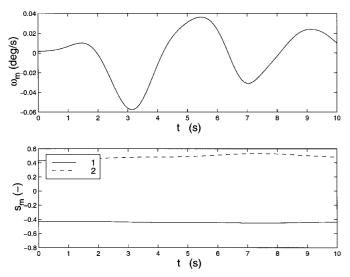


Fig. 4. Heading gyro and sun-sensor data.

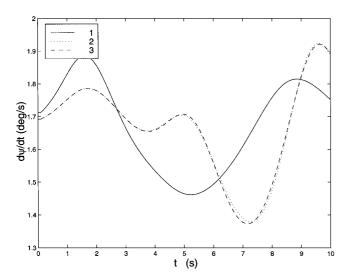


Fig. 5. Left drive train rates.

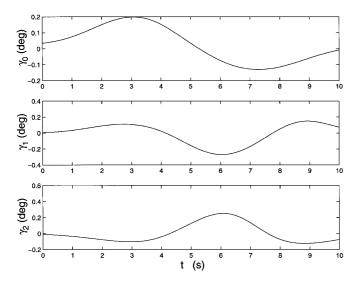


Fig. 6. Internal joint angles.

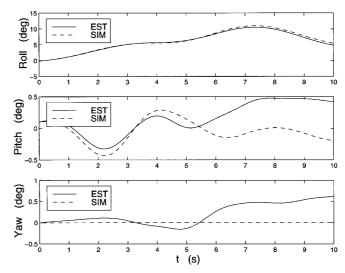


Fig. 7. Ground truth and estimate for attitude.

We can compare the performance of the kinematic estimator with one based upon dead-reckoning. The averaged wheel odometery obtained by integrating the speed (as indicated at each wheel) results in an estimate of distance traveled as 1.0204 m, i.e. approximately a 2% error over the actual distance traveled. For this straight-line motion one could correct this estimate by using the instantaneous vehicle pitch angle to project the wheel-derived traversal speeds prior to integration. However, this results in negligible improvement since the rocker-bogey mechanism keeps the pitch angles below 0.5 degrees for the entire duration of the motion. With the kinematic estimator we can correct for the effects of the nonlinear internal kinematics and the variations in contact angles at all of the wheels. The distance traveled is then estimated to be 1.0036 m indicating a much improved 0.3% error over the distance traveled.

We note the cross-track error in $\hat{x}(2)$ reaches a maximum of about 14 mm. This is a result of a one-directional transverse slip active for the portion of the motion shown. Over a larger section of terrain this slip component would average out to zero resulting in the filter reporting only a small cross-track error on average.

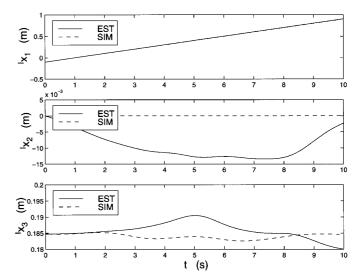


Fig. 8. Ground-truth and estimate for position.

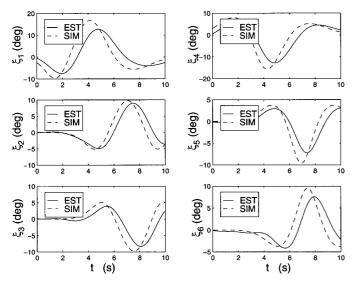


Fig. 9. Ground truth and estimate for contact points.

The ground-truth contact points and the estimated values are shown in Figure 9. We also observe that the tracking of the contact points is quite accurate, although with some lag.

The corresponding covariances are shown in Figure 10 and Figure 11. As expected there is a growth of the positional covariance with time.

4.1.2 Turn-in-place motion. Figure 12 shows the rover at the start of a 360 degree turn motion over the undulating surface.

The data and results are qualitatively very similar to that obtained for the straight-line motion case. Here we highlight only those aspects which differ from the straight-line case.

For the turn-in-place motion, the ground-truth attitude and the estimated values are shown in Figure 13 and indicate excellent attitude tracking from the sun-sensor and gyro data.

The ground-truth position and the estimated values are shown in Figure 14. We note that as the various slip noise values tend to average out to zero as the rover returns to its

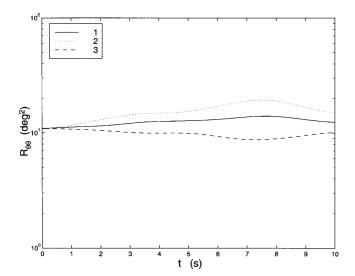


Fig. 10. Covariance of attitude estimate in terms of small angles.

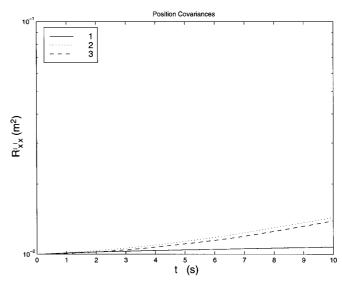


Fig. 11. Covariance of position estimate.

starting point, the estimates also recover from any bias shown during the motion.

The performance of the ground-truth contact point estimation is very similar to that obtained in the straight-line and is not shown here. There were, however, somewhat larger tracking errors for bogey wheels 3 and 6. These appear to be caused by the substantial slip in the bogey wheels due to the tight turning radius and the approximations inherent in the thin-wheel model used for the estimator.

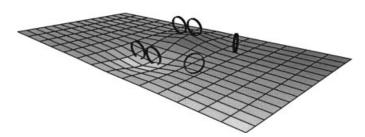


Fig. 12. Rocky 7 turn-in-place motion over undulating terrain.

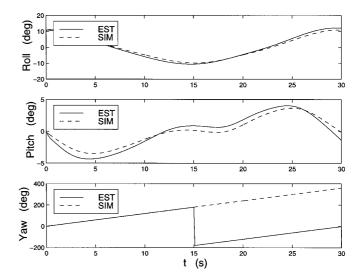


Fig. 13. Ground truth and estimate for attitude.

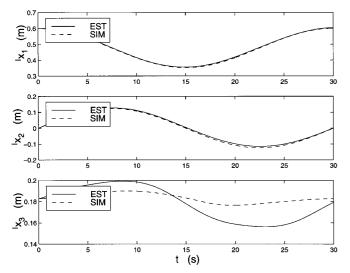


Fig. 14. Ground-truth and estimate for position.

The covariances for position shows a similar growth as in the straight-line motion case. Figure 15 shows the covariance of the angle estimate and it is interesting to note the periodic dips in two of the components. This is a consequence of the fact that the sun-sensor provide highquality information about absolute rotation for only two out of three attitude components – it is unable to provide any information about rotations about the sun-vector itself. The corresponding angular error covariance matrix is therefore "flattened" in two of three angular directions, with the main uncertainty axis lying along the line of the sun-vector. As the rover rotates, this highly stretched covariance is projected onto the local body coordinate frame that is fixed to the rotating rover. For particular yaw angles, corresponding to when the x or y axis of the body frame is perpendicular to the sun-vector, the projections of this covariance matrix are very small.

4.2. Filter operation with experimental data

Here we demonstrate the tracking of rover position and attitude from a combination of sun-sensor, heading gyro and

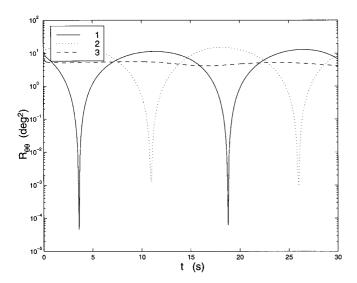


Fig. 15. Covariance of attitude estimates in terms of small angles.

the wheel sensors. The control algorithm used on the rover consists of individual high-gain controllers on each wheel. No wheel coordination is attempted based on any of the returned sensor data. The motion for this test set consists of a straight-line traverse on flat terrain with a single obstacle (a brick) encountered by the right wheels (wheels 4, 5, 6). The obstacle is successfully traversed with the rover coming to rest with a portion of the right bogeys (wheels 5, 6) still resting on the obstacle.

The obstacle causes two simultaneous effects. It results in a rover pitch-up as the wheels negotiate the obstacle as well as a change in the roll angle. Due to the momentum of the rover motion, there is minimal change in the rover heading. The data set is for a twenty second period with the filter operating at 10 Hz.

Sensor data from the attitude sensors is shown in Figures 16. As the steering angles are fixed during this motion they are not plotted. We also note that due to the nature of the control, all the drive wheels follow essentially the same commanded 5th-order spline trajectory regardless of the forces and torques experienced at each wheel, and so we have chosen not to plot them. Internal configuration angles are plotted in Figure 17.

In this experimental run, we did not have access to the prescribed motion values being generated by the controller. We therefore treat the entire body velocity of the rover as a perturbation and allow an increased process noise term in the filter to allow for its estimation. The estimated angles are is shown in Figure 18. We see that the estimator has correctly picked up the roll and pitch deflections induced by the obstacle.

The estimated positions and velocities arc shown in Figure 19. We see that the estimator has correctly picked up the z-deflection cause by the brick that results in an increase in vehicle height.

Estimated contact states are shown in Figure 20. We note that the contact angle variations are quite large under the right wheels as would be expected by the traversal of those wheels over the obstacle. Since the final configuration of the rover is such that the right-side bogey wheels are in the middle of traversing the obstacle, the corresponding contact

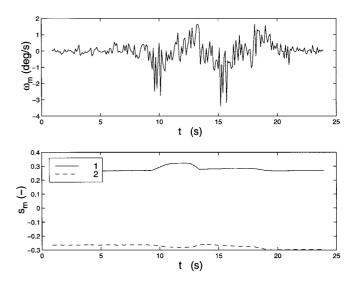


Fig. 16. Heading gyro and sun-sensor data.

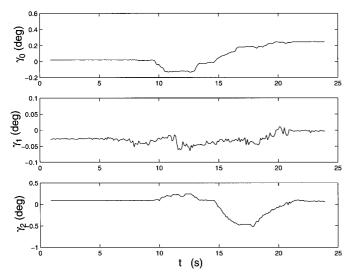


Fig. 17. Internal joint angles.

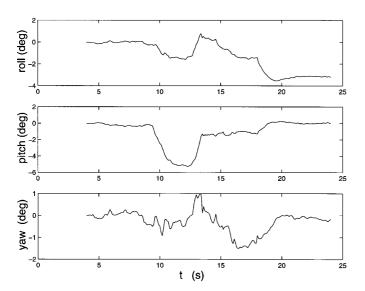


Fig. 18. Estimate for attitude.

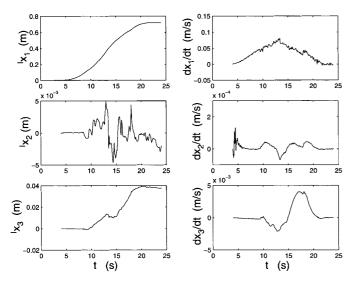


Fig. 19. Estimate for position and velocities.

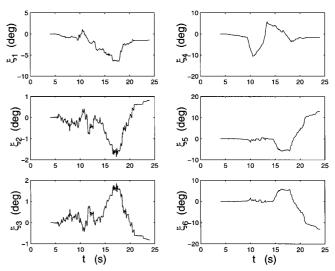


Fig. 20. Estimate for contact angles.

points are significantly displaced from zero. However, the contact point for the right front wheel returns to near zero as it proceeds on level ground after climbing over the obstacle. As expected the wheels on the left side of the vehicle experience very little change in contact angles.

The position and angular covariances are qualitatively similar to those obtained in the simulations and are therefore not plotted here. Ground-truth for this experiment is obtained by measuring tape and protractor at the end of the run. The estimated attitude values match the ground-truth values within the precision of the measurements taken. The estimated position values are within 1 cm of the ground-truth data and the estimated contact angles are within 5 degrees of the ground-truth.

5. CONCLUSIONS

We have demonstrated a kinematic estimator that successfully addresses the issues involved in roving a kinematically complex vehicle over non-flat terrain. By formulating a slip-constraint concept, we have been able to incorporate the non-linear kinematics, contact point and slip behavior of the vehicle into a Kalman filtering framework. The results represent a significant improvement over simpler odometry based methods and are very encouraging as evident by comparisons to to the ground-truth data obtained from simulations and experiments.

6. FUTURE WORK

In addition to more experiments to characterize and tune the system, we also expect to work on the following items:

- Using the data from the accelerometer sensor data to further improve the state estimate.
- Incorporating a smoother to post-process the sensor data every time the rover comes to one of its periodic halts. As we can obtain very precise attitude estimate when the rover is stopped using the accelerometer and sun-sensor, running the sensor data through the smoother will allow us to reconstruct many of the process disturbances and

reduce the covariance of both the attitude and position estimates.

- Using some results from nonlinear observability theory that allows one to switch optimally between gyro and sunsensor based attitude estimation together with simultaneous bias calibration of the gyro.
- Combining the kinematic estimator developed here with visual odometry and range-map matching techniques.

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