

LE2: Cylindrical shell bending patch test

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This problem provides evidence that Abaqus can reproduce the result from the benchmark defined by NAFEMS and cited as the reference solution.

This page discusses:

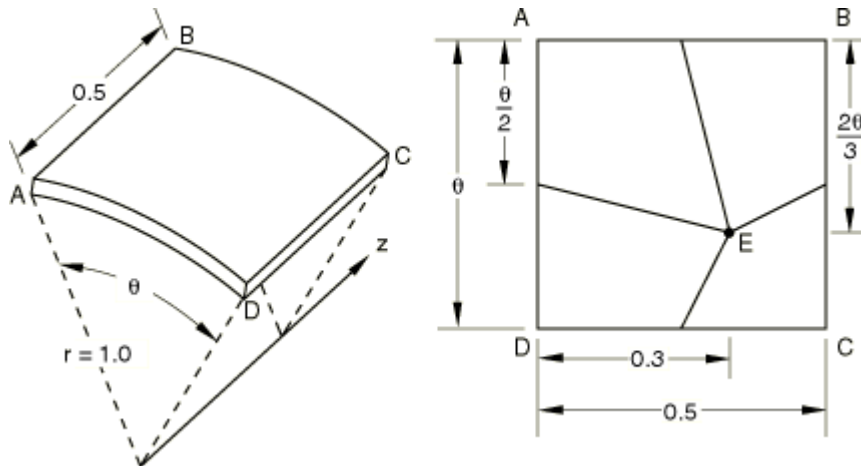
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ProductsAbaqus/StandardAbaqus/Explicit

Elements tested

- S3
- S3R
- S3RS
- S4
- S4R
- S4R5
- S4RS
- S4RSW
- S8R
- S8R5
- S9R5
- STRI3
- STRI65
- SC6R
- SC8R

Problem description



Model:

Sector of cylindrical shell with a thickness $t = 0.01$ m.

Material:

Linear elastic, Young's modulus = 210 GPa, Poisson's ratio = 0.3, density = 7800 kg/m^3 .

Boundary conditions:

Edge AB is clamped. Axial displacements are constrained along edges AD and BC.

Loading:

Uniform normal edge moment of 1000/unit length along edge DC. In the explicit dynamic analysis the loading is applied such that a quasi-static solution is obtained.

Reference solution

This is a test recommended by the National Agency for Finite Element Methods and Standards (U.K.): Test LE2 from NAFEMS publication TNSB, Rev. 3, "The Standard NAFEMS Benchmarks," October 1990.

Stress: Outer surface tangential stress at point E is 60 MPa.

Results and discussion

The results shown in [Table 1](#) through [Table 4](#) are interpolated from the integration points to the required nodal location. The values enclosed in parentheses are percentage differences with respect to the reference solution.

Table 1. Abaqus/Standard analysis, $\theta = 30^\circ$.

Element	Bottom Surface (MPa)	Top Surface (MPa)
S3/S3R	-44.3 (-26%)	40.6 (-32%)
S4	-63.2 (5%)	54.0 (-10%)
S4R	-58.0 (-3%)	58.0 (-3%)
S4R*	-55.0 (-8%)	55.2 (-8%)
S4R5	-58.6 (-2%)	58.6 (-2%)
S8R	-50.7 (-16%)	50.4 (-16%)
S8R5	-57.8 (-4%)	58.2 (-3%)
S9R5	-57.9 (-4%)	58.3 (-3%)
STRI3	-37.9 (-37%)	36.0 (-40%)
STRI65	-53.6 (-11%)	53.9 (-10%)
SC6R	-43.9 (-27%)	43.9 (-27%)
SC8R	-59.7 (-1%)	59.7 (-1%)
SC8R*	-54.8 (-9%)	-54.8 (-9%)

*Abaqus/Standard results with enhanced hourglass control.

Table 2. Abaqus/Explicit analysis, $\theta = 30^\circ$.

Element	Bottom Surface (MPa)	Top Surface (MPa)
S3R	-43.2 MPa (-28%)	39.7 MPa (-34%)
S3RS	-44.7 MPa (-26%)	42.2 MPa (-30%)
S4R	-58.2 MPa (-3%)	58.3 MPa (-2.8%)
S4RS	-57.0 MPa (-5%)	56.9 MPa (-5.2%)
S4RSW	-57.3 MPa (-4.5%)	57.4 MPa (-4.3%)

These results vary significantly from the target value since the mesh is too coarse to capture a curvature of $\theta = 30^\circ$. The mesh can be refined easily by reducing the arc angle to $\theta = 10^\circ$. The following results show that such mesh refinement greatly improves the accuracy of the results.

Table 3. Abaqus/Standard analysis, $\theta = 10^\circ$.

Element	Bottom Surface (MPa)	Top Surface (MPa)
S3/S3R	-60.1 (0.2%)	59.9 (-0.2%)
S4	-60.5 (0.8%)	59.5 (-0.8%)
S4R	-60.0 (0%)	60.0 (0%)
S4R*	-60.0 (0%)	60.0 (0%)
S4R5	-60.0 (0%)	60.0 (0%)
S8R	-59.6 (-0.7%)	59.7 (-0.5%)
S8R5	-59.9 (-0.2%)	60.0 (0%)

Element	Bottom Surface (MPa)	Top Surface (MPa)
S9R5	−59.7 (−0.5%)	60.0 (0%)
STRI3	−60.8 (1.3%)	60.8 (1.3%)
STRI65	−59.6 (−0.7%)	59.7 (−0.5%)
SC6R	−60.2 (0.3%)	60.2 (0.3%)
SC8R	−60.2 (0.3%)	60.2 (0.3%)
SC8R*	−60.2 (0.3%)	60.2 (0.3%)

*Abaqus/Standard results with enhanced hourglass control.

Table 4. Abaqus/Explicit analysis, $\theta = 10^\circ$.

Element	Bottom Surface (MPa)	Top Surface (MPa)
S3R	−60.2 MPa (−0.3%)	59.9 MPa (−0.1%)
S3RS	−60.0 MPa (0%)	59.8 MPa (−0.3%)
S4R	−60.0 MPa (0%)	60.0 MPa (0%)
S4RS	−60.1 MPa (−0.1%)	60.1 MPa (−0.1%)
S4RSW	−60.0 MPa (0%)	59.9 MPa (−0.2%)

Input files

Abaqus/Standard input files

$\theta = 30^\circ$:

[nle2xf3c.inp](#)

S3/S3R elements.

[nle2xe4c.inp](#)

S4 elements.

[nle2xf4c.inp](#)

S4R elements.

[nle2xf4c_eh.inp](#)

S4R elements with enhanced hourglass control.

[nle2x54c.inp](#)

S4R5 elements.

[nle2x68c.inp](#)

S8R elements.

[nle2x58c.inp](#)

S8R5 elements.

[nle2x59c.inp](#)

S9R5 elements.

[nle2x63c.inp](#)

STRI3 elements.

[nle2x56c.inp](#)

STRI65 elements.

[nle2_std_sc6r_30.inp](#)

SC6R elements.

[nle2_std_sc8r_30.inp](#)

SC8R elements.

[nle2_std_sc8r_30_eh.inp](#)

SC8R elements with enhanced hourglass control.

$\theta = 10^\circ$:

[nle2xf3f.inp](#)

S3/S3R elements.

[nle2xe4f.inp](#)

S4 elements.

[nle2xf4f.inp](#)

S4R elements.

[nle2xf4f_eh.inp](#)

S4R elements with enhanced hourglass control.

[nle2x54f.inp](#)

S4R5 elements.

[nle2x68f.inp](#)

S8R elements.

[nle2x58f.inp](#)

S8R5 elements.

[nle2x59f.inp](#)

S9R5 elements.

[nle2x63f.inp](#)

STRI3 elements.

[nle2x56f.inp](#)

STRI65 elements.

[nle2_std_sc6r_10.inp](#)

SC6R elements.

[nle2_std_sc8r_10.inp](#)

SC8R elements.

[nle2_std_sc8r_10_eh.inp](#)

SC8R elements with enhanced hourglass control.

Abaqus/Explicit input files

$\theta = 30^\circ$:

[le2_s3r_c.inp](#)

S3R elements.

[le2_s3rs_c.inp](#)

S3RS elements.

[le2_s4r_c.inp](#)

S4R elements.

[le2_s4rs_c.inp](#)

S4RS elements.

[le2_s4rsw_c.inp](#)

S4RSW elements.

$\theta = 10^\circ$:

[le2_s3r_f.inp](#)

S3R elements.

[le2_s3rs_f.inp](#)

S3RS elements.

[le2_s4r_f.inp](#)

S4R elements.

[le2_s4rs_f.inp](#)

S4RS elements.

[le2_s4rsw_f.inp](#)

S4RSW elements.