

## Chapter 4 : Flow Of Incompressible Fluids In Conduit And Thin Layers

### Important Terms ⇒

- ★ **Friction factor** : Ratio of wall shear stress to the product the density and velocity head.
- ★ **Hydraulically Smooth** : When further smoothening brings about no further reduction in the friction factor for a given Reynolds number.
- ★ **Moody Diagram** : 1. Log-log plot of friction factor vs Reynolds number.  
2. The frictional characteristics of round pipes(both smooth and rough) are summarized by a friction factor chart.
- ★ **Vena Contracta** : The cross-section of minimum area at which the jet changes from a contraction to an expansion is called vena contracta.
- ★ **Couette Flow** : It is the flow of a viscous fluid in the space between two surfaces, one of which is moving tangentially relative to the other. The relative motion of the surfaces imposes a shear stress on the fluid and induces flow.

## Important Formulas $\Rightarrow$

### 1. SHEAR-STRESS DISTRIBUTION IN A CYLINDRICAL TUBE :

$$\frac{dp}{dL} + \frac{2\tau}{r} = 0 \qquad \frac{\tau_w}{r_w} = \frac{\tau}{r}$$

### 2. RELATION BETWEEN SKIN FRICTION AND WALL SHEAR :

$$h_{fs} = \frac{2 \tau_w}{\rho r_w} \Delta L = \frac{4 \tau_w}{\rho D} \Delta L$$

### 3. Friction Factor :

$$f \equiv \frac{\tau_w}{\rho \bar{V}^2 / 2g_c} = \frac{2g_c \tau_w}{\rho \bar{V}^2}$$

### 4. RELATIONS BETWEEN SKIN-FRICTION PARAMETERS :

$$h_{fs} = \frac{2 \tau_w}{\rho r_w} \Delta L = \frac{\Delta p_s}{\rho} = 4f \frac{\Delta L}{D} \frac{\bar{V}^2}{2g_c}$$

$$f = \frac{\Delta p_s g_c D}{2 \Delta L \rho \bar{V}^2}$$

### 5. LAMINAR FLOW OF NEWTONIAN FLUIDS :

$$u = \frac{\tau_w g_c}{2r_w \mu} (r_w^2 - r^2) \qquad , \qquad \frac{u}{u_{\max}} = 1 - \left( \frac{r}{r_w} \right)^2$$

**6. Average Velocity :**

$$\bar{V} = \frac{\tau_w g_c}{r_w^3 \mu} \int_0^{r_w} (r_w^2 - r^2) r \, dr = \frac{\tau_w g_c r_w}{4\mu}, \quad \frac{\bar{V}}{u_{\max}} = 0.5$$

**7. HAGEN-POISEUILLE EQUATION :** Main use is in the experimental measurement of viscosity by measuring pressure drop and volumetric flow rate.

$$\Delta p_s = \frac{32 \Delta L \bar{V} \mu}{g_c D^2}, \quad f = \frac{16 \mu}{D \bar{V} \rho} = \frac{16}{N_{\text{Re}}}$$

**8. Velocity Distribution for Turbulent Flow :**

$$u^* \equiv \bar{V} \sqrt{\frac{f}{2}} = \sqrt{\frac{\tau_w g_c}{\rho}}$$

$$u^+ \equiv \frac{u}{u^*}$$

$$y^+ \equiv \frac{y u^* \rho}{\mu} = \frac{y}{\mu} \sqrt{\tau_w g_c \rho}$$

where  $u^*$  = friction velocity

$u^+$  = velocity quotient, dimensionless

$y^+$  = distance, dimensionless

$y$  = distance from wall of tube

**9. UNIVERSAL VELOCITY-DISTRIBUTION EQUATIONS :**

- Viscous sublayer :  $u^+ = y^+$
- Buffer layer :  $u^+ = 5.00 \ln y^+ - 3.05$
- Turbulent layer :  $u^+ = 2.5 \ln y^+ + 5.5$

for the viscous sublayer:  $y^+ < 5$

for the buffer zone:  $5 < y^+ < 30$

for the turbulent core:  $30 < y^+$

**10. Average velocity :**

$$\bar{V} = \frac{5\nu^2}{r_w^2 u^*} \int_0^{y_c^+} \left( 0.4u_c^+ + \ln \frac{y^+}{y_c^+} \right) (y_c^+ - y^+) dy^+, \quad \frac{\bar{V}}{u_{\max}} = \frac{1}{1 + (3.75\sqrt{f/2})}$$

**11. Von-Karman equation :**

$$\frac{1}{\sqrt{f/2}} = 2.5 \ln \left( N_{\text{Re}} \sqrt{\frac{f}{8}} \right) + 1.75$$

**12. Hydraulic Radius :**

$$r_H \equiv \frac{S}{L_p}$$

where  $S$  = cross-sectional area of channel

$L_p$  = perimeter of channel in contact with fluid

**13. Darcy-Weisbach equation :**

$$h_{fs} = \frac{2}{\rho} \frac{\tau_w}{r_w} \Delta L = \frac{\Delta p_s}{\rho} = 4f \frac{\Delta L}{D} \frac{\bar{V}^2}{2g_c}$$