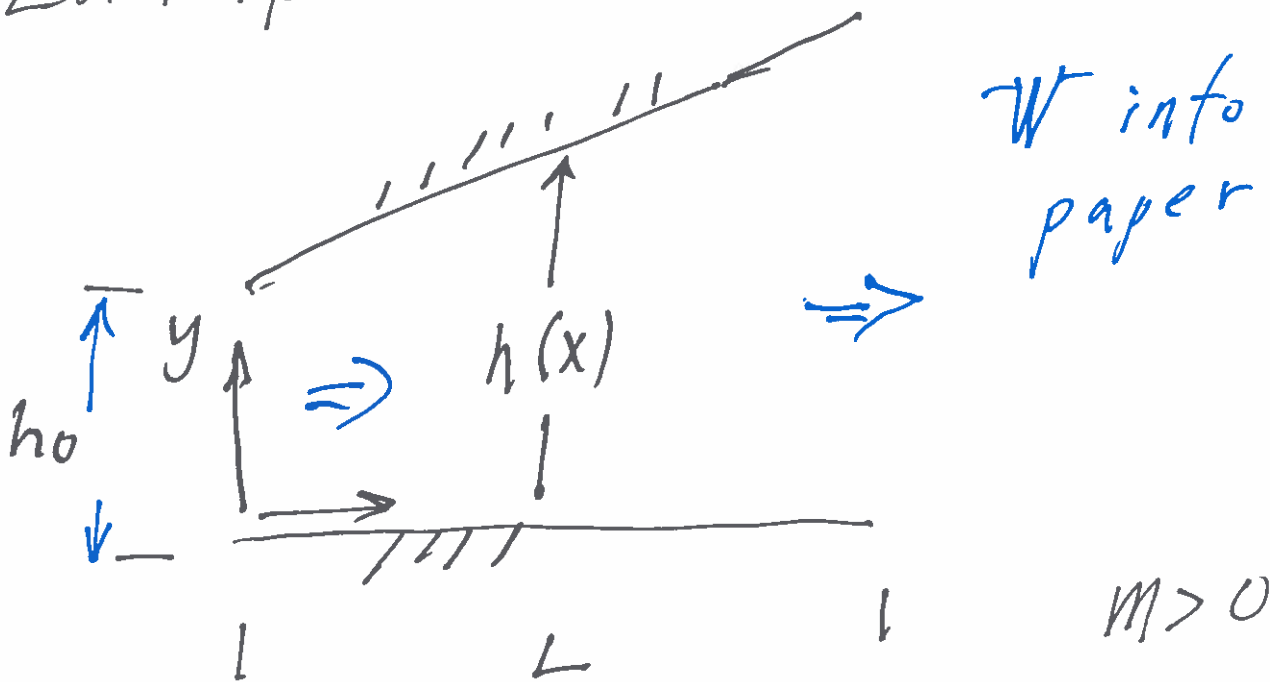


1

# Class # 6

Example - similar to HW



$m > 0$

given:  $h = h_0 \left( 1 + m \frac{x}{L} \right)$

$x=0 \quad h=h_0 \quad x=L \quad = h_0(1+m)$

$$\underline{\underline{\sigma}} = \begin{bmatrix} -p & \tau \\ \tau & -p \end{bmatrix}$$

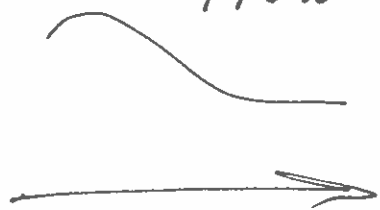
-2-

shear

Model for flow

$$\hat{\sigma} \approx -p \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} + \begin{bmatrix} 0 & \tau \\ \tau & 0 \end{bmatrix}$$

shear flow



$p$  = pressure

$\tau$  = shear stress

$\tau$  depends on motion

$$\dot{\gamma} \rightarrow 0 : \tau \rightarrow 0$$

$$\sigma_{xx} = -p = \sigma_{yy}$$

inward/compressive

This problem:

$$p = p_0 \quad \tau = \tau_0 \left( \frac{y}{h} \right)$$

$$\text{HW: } p = p(x) \quad \tau = \tau(x, y)$$

$$y=0 \quad \tau=0 \quad y=h(x) \quad \tau=\tau_0$$

-3-

Find  $\underline{F}$  on upper  
surface

Course

General:

stress model



$$\text{Diff} + \text{BC} \Rightarrow \underline{V} = \underline{\sigma}$$

Eq

$$\Rightarrow \underline{F}$$

this example just  $\underline{\sigma} \Rightarrow \underline{F}$

$$\underline{t} = \left. \frac{d\underline{F}}{dA} \right|_{\text{surface}} = \underline{n} \cdot \underline{\sigma}$$

$\underline{t}$  = traction vector  
stress vector or

$$F_x = \int_A t_x dA$$

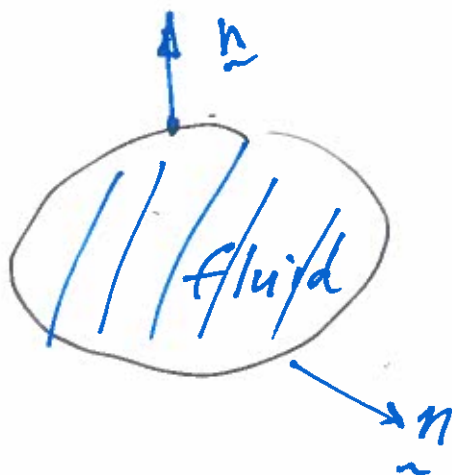
Surface

$$F_y = \int_A t_y dA$$

4-

$$\underline{F} = F_x \hat{e}_x + F_y \hat{e}_y$$

$\Rightarrow$  find  $\underline{n}$  outward unit normal to surface



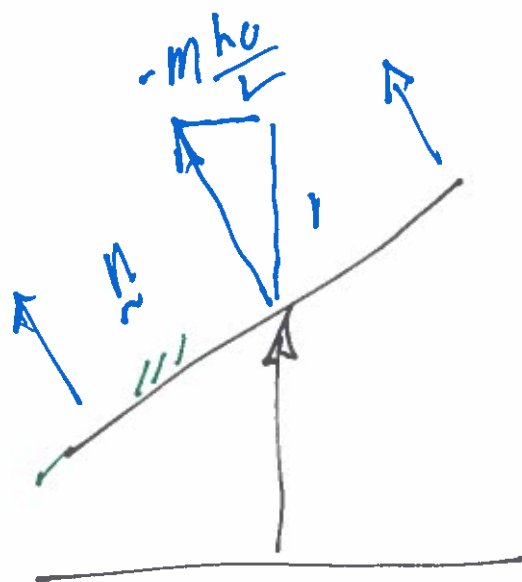
Surface defined:  $y = h(x)$

$$f = y - h(x) \quad f = 0 \text{ surface in question}$$

$$\underline{n} = \hat{n} = \frac{\underline{\nabla} f}{|\underline{\nabla} f|}$$

any vector  $\underline{a}$  unit vector  $= \frac{\underline{a}}{|\underline{a}|}$  in direction of  $\underline{a}$

-5-



$$h = h_0 \left( 1 + m \frac{x}{L} \right)$$

$$f = y - h_0 \left( 1 + m \frac{x}{L} \right)$$

$$\underline{\nabla} f = \frac{\partial f}{\partial x} \hat{e}_x + \frac{\partial f}{\partial y} \hat{e}_y$$

$$\frac{\partial f}{\partial x} = -h_0 \frac{m}{L} \quad \frac{\partial f}{\partial y} = 1$$

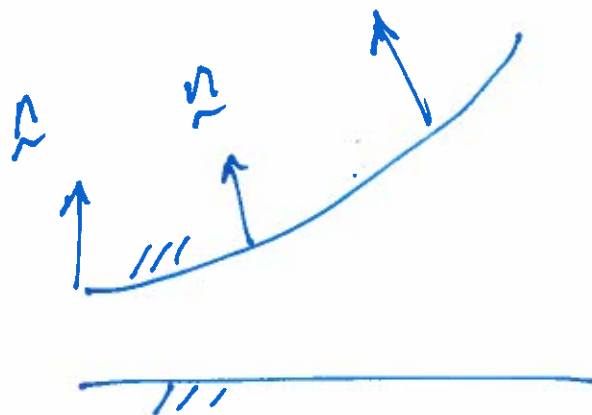
$$\underline{\nabla} f = -m \frac{h_0}{L} \hat{e}_x + \hat{e}_y$$

$$\underline{n} = \pm \frac{\underline{\nabla} f}{|\underline{\nabla} f|} = \frac{-m \frac{h_0}{L} \hat{e}_x + \hat{e}_y}{\sqrt{1 + \left( -\frac{h_0}{L} m \right)^2}}$$

= const (this case, not HW)

-6-

HW:



$\propto x^2$

$$\underline{\tilde{L}} = \underline{\tilde{L}} \cdot \underline{\tilde{\sigma}}$$

$$= \frac{1}{\sqrt{I}} \left( -m \frac{\hbar \omega}{L} \hat{e}_x + \hat{e}_y \right) \cdot \left( \sigma_{xx} \hat{e}_x \hat{e}_x + \sigma_{xy} \hat{e}_x \hat{e}_y + \dots \right) \cdot \left( -p \hat{e}_x \hat{e}_x + \tau \hat{e}_x \hat{e}_y + \tau \hat{e}_y \hat{e}_x - p \hat{e}_y \hat{e}_y \right)$$

$$\underline{\tilde{L}} = L_x \hat{e}_x + L_y \hat{e}_y =$$

$$= \left( +m \frac{\hbar \omega}{L} p \hat{e}_x - m \frac{\hbar \omega}{L} \tau \hat{e}_y \right.$$

$$\left. + \tau \hat{e}_x - p \hat{e}_y \right) \frac{1}{\sqrt{I}}$$

$$= \left[ \hat{e}_x \left( \frac{m \hbar \omega}{L} p + \tau \right) + \hat{e}_y \left( -\frac{m \hbar \omega}{L} \tau - p \right) \right] \frac{1}{\sqrt{I}}$$

8-

$$F_x = \int_A \tau_x dA \quad y=h$$

$$\tau_h = \tau(y=h)$$

$$dA = W dx$$

$$F_x = W \int_0^L \left( \frac{\rho h v}{L} p_v + \tau_v \right) dx \quad \frac{1}{\sqrt{1}}$$

$$= \frac{WL}{\sqrt{1}} \left( m \frac{h v}{L} p_v + \tau_v \right)$$

$$\tau = \tau_v \left( \frac{y}{h} \right)$$

$$\tau_h = \tau_v \left( \frac{h}{h} \right)$$

$$= \tau_v$$

(next page)

$$F_y = \dots$$

Any <sup>problem</sup> ~~case~~ - if you can write as def. integral

$$f = \int_0^d f(x; \underline{a, b, c}) dx$$

solved!

know.

a, b, c, d

-1-

$$\underline{\underline{t}} = \frac{1}{\sqrt{1}} \left[ \underline{\underline{t}}_x \hat{e}_x + \underline{\underline{t}}_y \hat{e}_y \right] = \underline{\underline{t}}(x, y)$$

stress vector any pt in  
flow

want  $\underline{\underline{t}}$  at  $y = h(x)$

$\underline{\underline{t}}(x, h(x))$  upper surface

$$y = h(x) \quad \tau = \tau_0 \left( \frac{y}{h} \right)$$

$$\text{at } y = h : \tau = \tau_0$$

at  $y = h(x)$

$$\underline{\underline{t}}_{y=h} = \left( \frac{\rho h_0}{L} p_0 + \tau_0 \right) \hat{e}_x + \left( -\frac{\rho h_0}{L} \tau - p \right) \hat{e}_y \cdot \frac{1}{\sqrt{1}}$$



- 8 - mathematica

$$\text{NIntegrate}[f(x), \{x, 0, L\}]$$

↑

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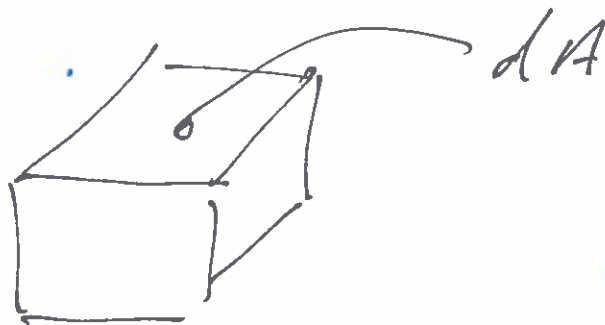
New materia

$$m_{\sim} a = \sum \vec{F}_{\sim}$$

$a$  of particle, not

point in space

$$\sum \vec{F}_{\sim} = \underbrace{F_{\sim \text{body}}}_{\text{gravity}} + \underbrace{F_{\sim \text{surface}}}_{\text{stress on surface}}$$



-9-

Model for stress

$$f_n(\underline{V})$$

$$\underline{\underline{\sigma}} = -p \underline{\underline{\delta}} + \underline{\underline{\tau}}$$

$\underline{\underline{\delta}}$  = Unit tensor = Kronecker delta

$$= \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

Want  $\underline{V} \rightarrow 0$   $\underline{\underline{\sigma}} = \begin{bmatrix} -p & 0 \\ 0 & -p \end{bmatrix}$   
 $\underline{\underline{\tau}} \rightarrow 0$

Model:  $\underline{\underline{\tau}} = 0$  inviscid flow

rather ok away from body/  
wall/surface

-10-

All observers calculate same stress

observers can translate rotate

$$\underline{\underline{\tau}} = k \underline{\underline{V}}$$

$$\underline{\underline{V}} = \underline{\underline{V}}_{\text{fluid}} - \underline{\underline{V}}_{\text{obs}}$$

$$\underline{\underline{V}}_{\text{obs}}$$

$$\tau_{xy} = \mu \frac{\partial v_x}{\partial y}$$

shear flow

linear viscosity

$$\text{use } \underline{\underline{\nabla}} \underline{\underline{V}} = \begin{pmatrix} \frac{\partial v_x}{\partial y} & \frac{\partial v_y}{\partial x} \\ \dots & \dots \end{pmatrix}$$

$$\underline{\underline{\tau}} = ? \mu \underline{\underline{\nabla}} \underline{\underline{V}} \text{ why not?}$$

rotating observers calculate different  $\underline{\underline{\nabla}} \underline{\underline{V}}$

$$\tau_{xx} = \mu \frac{\partial v_x}{\partial x}$$

$$\tau_{xy} = \mu \frac{\partial v_x}{\partial y} \text{ etc}$$