

Class #8

Exam #1, Mon Oct 5

Fluid Model - all observers
calculate same stress

$$\underline{\underline{\sigma}} = -p \underline{\underline{\delta}} + \underline{\underline{T}}$$

determined
by model, extra stress
deviatoric stress

want linear

2-D

$$\underline{\underline{Q}} = \underbrace{\text{sym part}}_{\text{deformation}} + \underbrace{\text{anti sym part}}_{\text{rotation}}$$

$$\underline{\underline{\nabla V}} = \begin{bmatrix} \frac{\partial V_x}{\partial x} & \frac{\partial V_y}{\partial x} \\ \frac{\partial V_x}{\partial y} & \frac{\partial V_y}{\partial y} \end{bmatrix}$$

velocity gradient

transpose

$$(\underline{\underline{\nabla V}})^T = \begin{bmatrix} \frac{\partial V_x}{\partial x} & \frac{\partial V_x}{\partial y} \\ \frac{\partial V_y}{\partial x} & \frac{\partial V_y}{\partial y} \end{bmatrix}$$

$$\text{sym part} = \frac{1}{2} (\underline{\underline{\nabla V}} + \underline{\underline{\nabla V}}^T)$$

$$\text{anti-sym part} = \frac{1}{2} (\underline{\underline{\nabla V}} - \underline{\underline{\nabla V}}^T)$$

$$\underline{\underline{\nabla V}} = \text{sym} + \text{anti-sym}$$

-3-

$$\dot{\gamma} = \text{sym} = \begin{bmatrix} \frac{\partial v_x}{\partial x} & \frac{\partial v_x}{\partial y} + \frac{\partial v_y}{\partial x} \\ \frac{\partial v_y}{\partial x} + \frac{\partial v_x}{\partial y} & 2 \frac{\partial v_y}{\partial y} \end{bmatrix}$$

rate
deformation
stretching
strain tensor

$$\dot{\omega} = \text{anti} = \begin{bmatrix} 0 & -\frac{\partial v_x}{\partial y} + \frac{\partial v_y}{\partial x} \\ \frac{\partial v_x}{\partial y} - \frac{\partial v_y}{\partial x} & 0 \end{bmatrix} \cdot \begin{matrix} \text{spin} \\ \text{tensor} \\ \text{vorticity} \\ \text{tensor} \end{matrix}$$

$$\dot{\omega} = \text{vorticity vector} = \omega_z \hat{e}_z$$

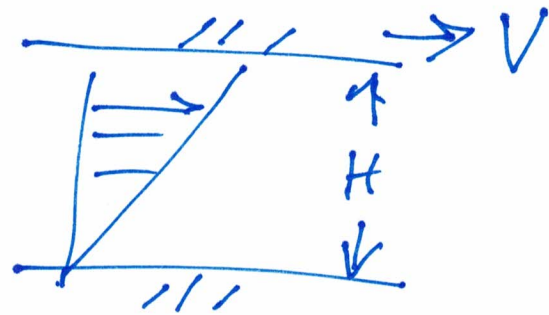
$$\omega_z = \frac{\partial v_y}{\partial x} - \frac{\partial v_x}{\partial y}$$

$$\nabla \underline{v} = \underbrace{\frac{1}{2} \dot{\gamma}}_{\text{deformation}} + \underbrace{\frac{1}{2} \dot{\omega}}_{\text{rotation}}$$

-30-

Simple shear flow
Couette flow

$$V_x = V \frac{y}{H} = \frac{V_y}{H}$$

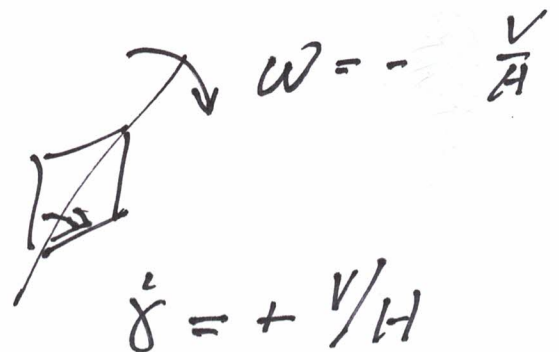


$$\dot{\gamma} = \begin{bmatrix} 0 & V/H \\ V/H & 0 \end{bmatrix}$$



fluid
element

$$\underline{\underline{\omega}} = \begin{bmatrix} 0 & -V/H \\ +V/H & 0 \end{bmatrix}$$



$$\underline{\underline{\omega}} = \nabla \times \underline{V}$$

$$= -\frac{V}{H}$$

$$\underline{V} = \frac{\partial}{\partial x} \hat{e}_x + \frac{\partial}{\partial y} \hat{e}_y \hat{j}$$

$$\underline{V} = V_x \hat{i} + V_y \hat{j}$$

-4-

Model:

$\underbrace{\nabla \mathbf{v}}_{\dot{\gamma}}$ - rotation
 $\dot{\gamma}$ deformation

Fluid Model (Newtonian)

$$\boxed{\tau = \mu \dot{\gamma}}$$

$\frac{N}{m^2}$

μ = Viscosity

$$\frac{N \cdot s}{m^2}$$

$$\dot{\gamma} = \frac{v}{h} = \frac{m/s}{m} = \frac{1}{s}$$

air }
water }

very low
viscosity

Let's say so far

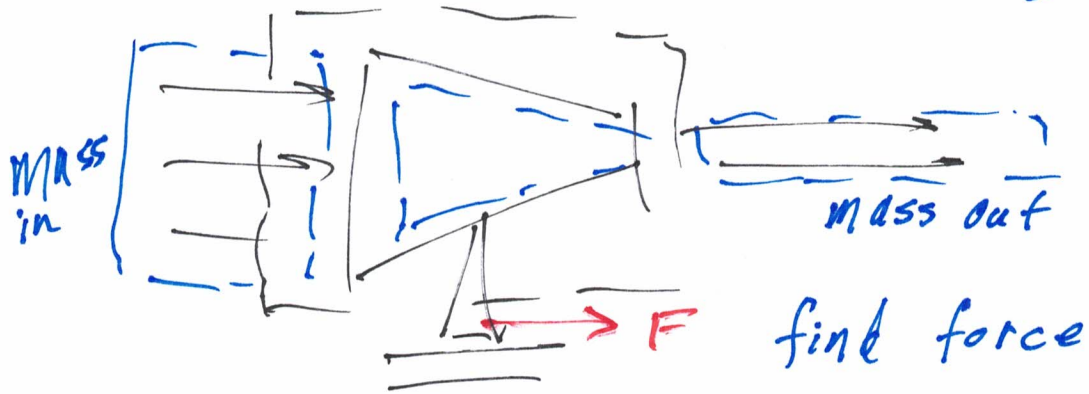
given $\mathbf{v} \Rightarrow \tau \Rightarrow \mathbf{F}$

most of
course

$\mathbf{F} \Rightarrow \tau \Rightarrow \mathbf{v}$

5-

Global Approach



Control volume - any volume of interest

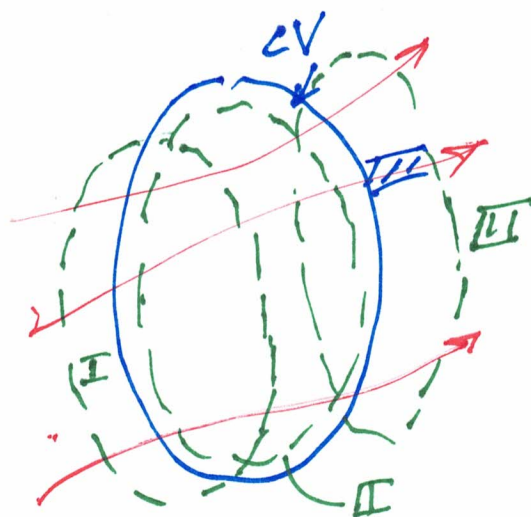
V = volume

— change size
shape
move

Most commonly CV - fixed

-6-

Reynolds Transport Theorem



$$\left. \frac{d\theta}{dt} \right|_{\text{mass}} = \left. \frac{d\Phi}{dt} \right|_{\text{CV}} + \dot{\Phi}_{\text{out}} - \Phi_{\text{in}}$$

--- => mass ————— volume

Φ = any extensive property
mass, momentum, ~~energy~~

convection

θ = generation of Φ in Π

θ = mass

$$\frac{d\theta}{dt} = 0$$

θ = momentum = $m \underline{V}$

$$\frac{d\theta}{dt} = \frac{d}{dt} m \underline{V} = m \underline{a} = \underline{F}$$

7-

$$\underbrace{\frac{d\theta}{dt}}_{\text{mass}} = \underbrace{\frac{d\theta}{dt}}_{\text{cv}} + \dot{\Phi}_{\text{out}} - \dot{\Phi}_{\text{in}} \quad \Phi = \frac{\underline{F}}{m}$$

mass

$$0 = \frac{d}{dt} \int_V \rho dV + \dot{m}_{\text{out}} - \dot{m}_{\text{in}}$$

$$\dot{m} = \int_A \rho \underline{V} \cdot \underline{n} dA \quad \begin{array}{l} \text{mass} \\ \text{flow rate} \\ \text{kg/s} \end{array}$$

momentum

$$\underline{\dot{F}} = \underline{\dot{F}} = \frac{d}{dt} \int_{\text{cv}} (\rho \underline{V}) dV$$

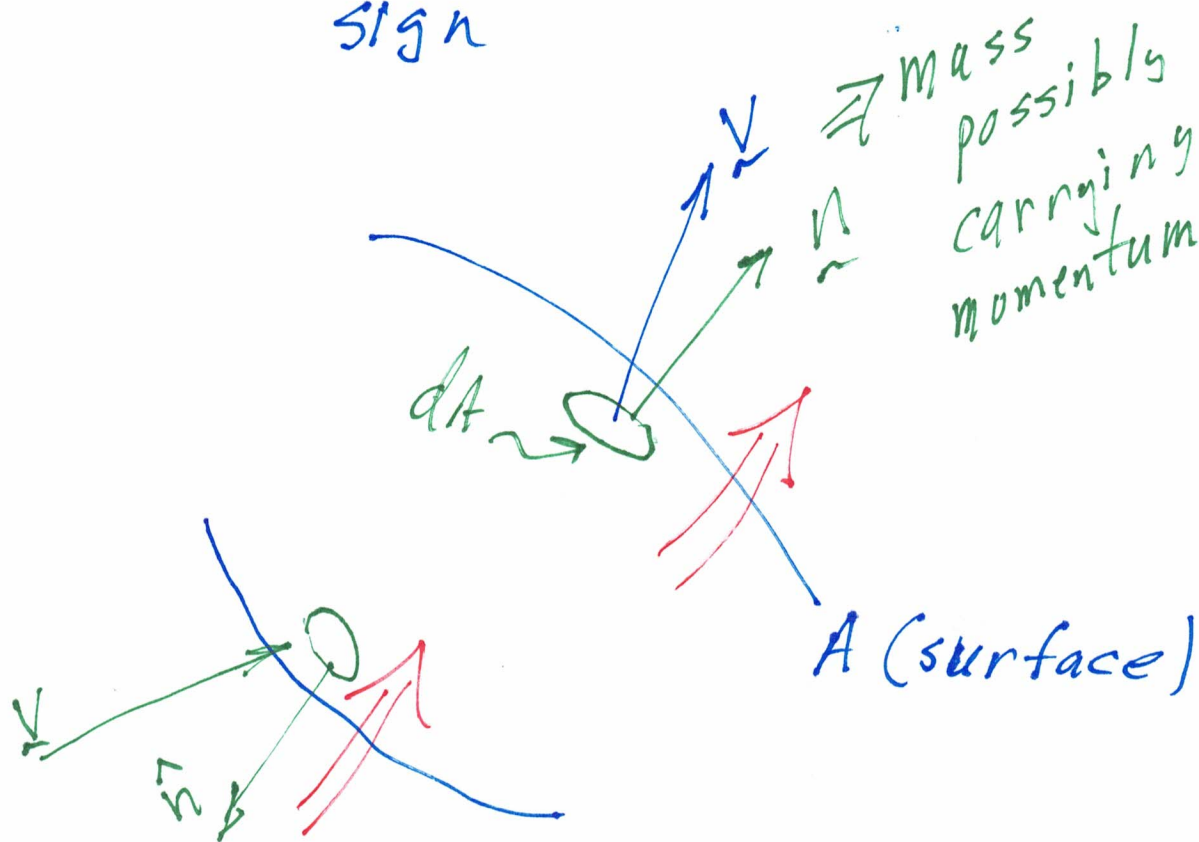
$$+ \dot{\underline{G}}_{\text{out}} - \dot{\underline{G}}_{\text{in}}$$

$\underline{\dot{G}} = \text{momentum flow rate}$

8.

$$\dot{G}_{\vec{n}} = \int_A \rho \vec{V} \cdot \vec{n} dA = \int_A \rho \vec{V} (\vec{V} \cdot \vec{n}) dA$$

Note $\vec{V} \cdot \vec{n}$ takes care of
sign



out: $\vec{V} \cdot \vec{n} > 0$ in $\vec{V} \cdot \vec{n} < 0$

-9-

mass

$$0 = \frac{d}{dt} \int_{CV} \rho dV + \int_{CS} \rho (\underline{V} \cdot \underline{n}) dA$$

mass in CV

control surface

$$0 = \frac{d}{dt} m_{CV} + \dot{m}_{out} - \dot{m}_{in}$$

mass flow out

(surface enclosing control volume)

momentum

$$\sum \underline{F} = \frac{d}{dt} \int_{CV} \rho \underline{V} dV + \int_{CS} \rho (\underline{V} \cdot \underline{n}) dA$$

$$= \frac{d \underline{G}_{CV}}{dt} + \underline{G}_{out} - \underline{G}_{in}$$

LHS:

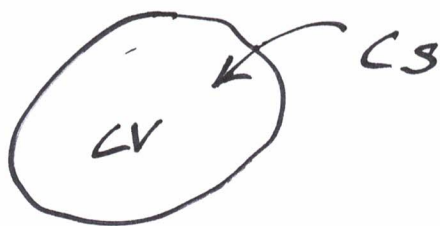
$$\text{mass} - \left. \frac{dm}{dt} \right|_{m=\text{const.}} = 0$$

$$\text{momentum} - \left. \frac{d \underline{mV}}{dt} \right|_{\underline{m}=\text{const.}} = \underline{F}$$

-10-

Divergence Theorem

changes $\int_{CV} \Leftrightarrow \int_{CS}$



$$d\vec{A} = \vec{n} dA$$

$$(1) \int_{CS} \rho \vec{V} \cdot \vec{n} dA = \int_{CV} \nabla \cdot (\rho \vec{V}) dV$$

flow out divergence inside

Back to Reynold Transport Th
for mass - fixed CV

$$0 = \frac{d}{dt} \int_{CV} \rho dV + \underbrace{\int_S \rho (\vec{V} \cdot \vec{n}) dA}_{\text{flow out}}$$

(1) for $\phi = 1$ $\int_{CV} \nabla \cdot \rho \vec{V} dV$

-12-

2D, incompressible

$$\frac{\partial V_x}{\partial x} + \frac{\partial V_y}{\partial y} = 0$$

know $V_x \Rightarrow$ Find V_y

$$V_x = \dots$$

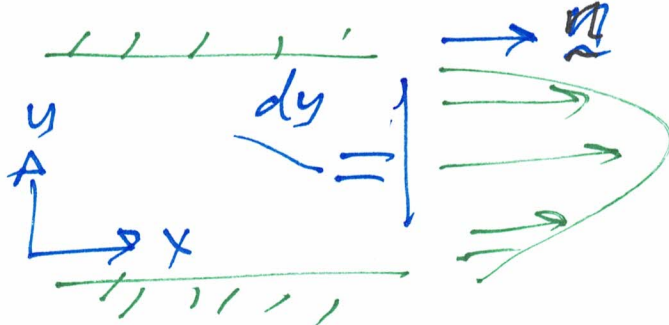
know V_y find V_x

$$\frac{\partial V_x}{\partial x} = - \frac{\partial V_y}{\partial y}$$

e.g. know V_y

$$V_x = \int \frac{\partial V_y}{\partial x} dx + \text{const} \quad \leftarrow \text{BC}$$

$$\dot{m} = \text{mass flow} = \int_A \rho (\underline{V} \cdot \underline{n}) dA$$



$$\dot{m} = \int_A \rho V_x dA$$

$$dA = W dy$$

W into paper

-11-

$$0 = \int_V \frac{\partial \rho}{\partial t} dV + \int_V (\underline{\nabla} \cdot \rho \underline{V}) dV$$

$$0 = \int_V \left(\frac{\partial \rho}{\partial t} + \underline{\nabla} \cdot \rho \underline{V} \right) dV$$

$$0 = \frac{\partial \rho}{\partial t} + \underline{\nabla} \cdot \rho \underline{V}$$

differential continuity equation

$$0 = \frac{d}{dt} \int_{CV} \rho dV + \int_{CS} \rho (\underline{V} \cdot \underline{n}) dA$$

global cont equation

$\rho = \text{density} = \text{const}$
incompressible

$$0 = \cancel{\frac{\partial \rho}{\partial t}} + \cancel{\rho (\underline{\nabla} \cdot \underline{V})} \Rightarrow \underline{\nabla} \cdot \underline{V} = 0$$

$\text{div} \underline{V} = 0$