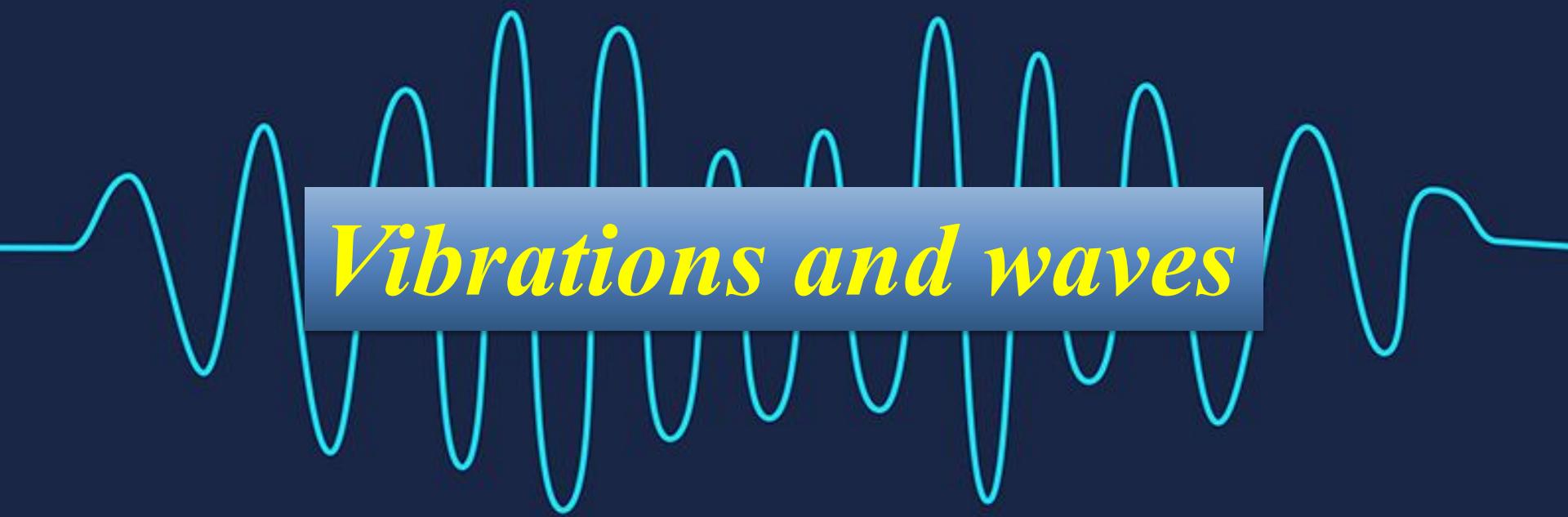


National Higher School of Autonomous Systems Technology

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## *Vibrations and waves*

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# Chapter 3 : Damped free vibration of single degree of freedom systems

$$\frac{d}{dt} \left( \frac{\partial(L)}{\partial \dot{q}} \right) - \frac{\partial(L)}{\partial q} + \frac{\partial D}{\partial \dot{q}} = 0$$

### *Contents*

**1-Definition**

**2- Applications**

**3- Differential equation**

**4- Solution of differential equation**

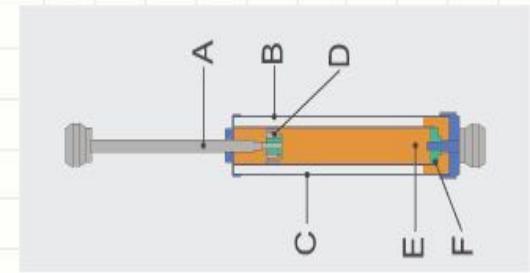
**5- Logarithmic decrement**

## Chapter 3 : Damped free vibration of single degree of freedom systems

### 1-Definition

#### Viscous damper

✓ A **viscous damper** the piston moves, inside a **cylinder** filled with **viscous fluid**, the fluid flows through small openings, creating **resistance**. This resistance **converts the motion energy into heat**, dissipating vibrations and helping the system return to equilibrium



## Chapter 3 : Damped free vibration of single degree of freedom systems

### 1-Definition

#### ■ Viscous damper

In the case of viscous friction, the friction force is proportional to the velocity but acts in the opposite direction.

$$\vec{F}_\alpha = - \alpha \vec{v}$$

Where  $\alpha$  is the coefficient of viscous friction and  $\vec{v}$  is the velocity of the particle

➤ Dissipation function  $D = \frac{1}{2} \alpha \dot{q}^2$

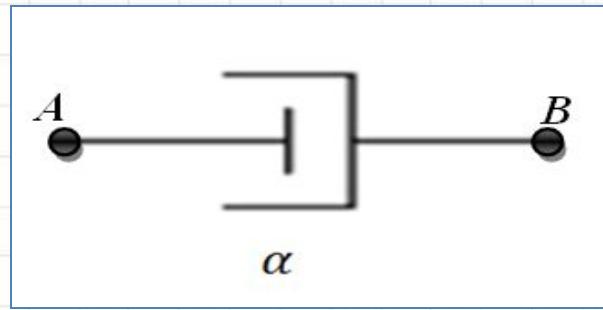
$$f = - \frac{\partial D}{\partial \dot{q}}$$

## Chapter 3 : Damped free vibration of single degree of freedom systems

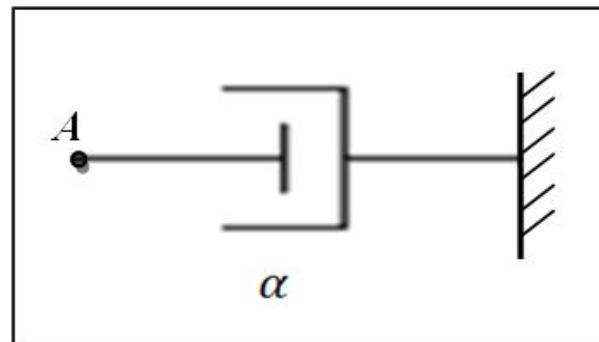
### 1-Definition

#### □ Viscous damper

$$D = \frac{1}{2} \alpha (\vec{v}_A - \vec{v}_B)^2$$



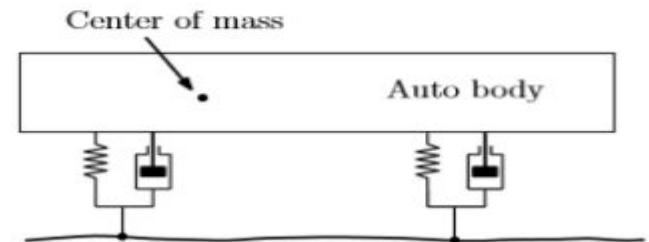
$$D = \frac{1}{2} \alpha v_A^2$$



## 2- Applications

### Example 1

The **suspension** is a system in vehicles designed to **absorb shocks** caused by the road. It makes driving more comfortable and improves vehicle **stability** by keeping the wheels in contact with the ground.

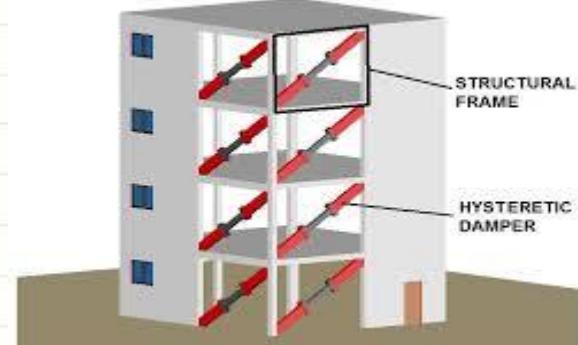
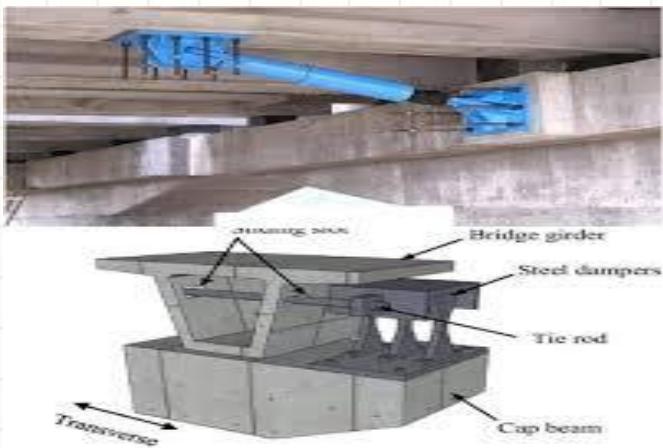


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### 2- Applications

Buildings and bridges: Dampers are installed to reduce vibrations caused by wind or earthquakes, thus protecting the structures.



## Chapter 3 : Damped free vibration of single degree of freedom systems

### 2- Differential equation:

Consider the single degree of freedom model with viscous damping shown in Figure. The viscous damper with coefficient  $\alpha$ .

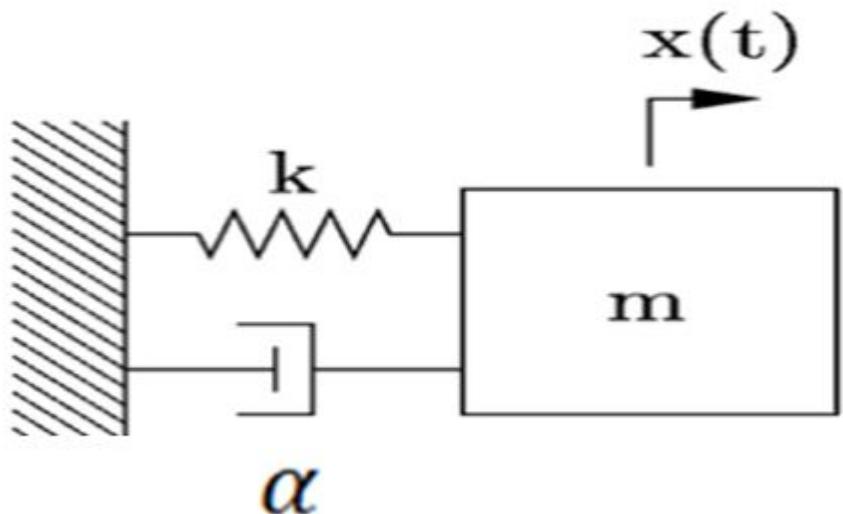
- Using Lagrange's equation:

$$T = \frac{1}{2} m \dot{x}^2 , \quad U = \frac{1}{2} k x^2$$

- Dissipation function :  $D = \frac{1}{2} \alpha \dot{x}^2$
- Lagrangian of the system:

$$L = T - U \Rightarrow L = \frac{1}{2} m \dot{x}^2 - \frac{1}{2} k x^2$$

$$\frac{d}{dt} \left( \frac{\partial(L)}{\partial \dot{q}} \right) - \frac{\partial(L)}{\partial q} + \frac{\partial D}{\partial \dot{q}} = 0 \Rightarrow \frac{d}{dt} \left( \frac{\partial(L)}{\partial \dot{x}} \right) - \frac{\partial(L)}{\partial x} + \frac{\partial D}{\partial \dot{x}} = 0$$



## Chapter 3 : Damped free vibration of single degree of freedom systems

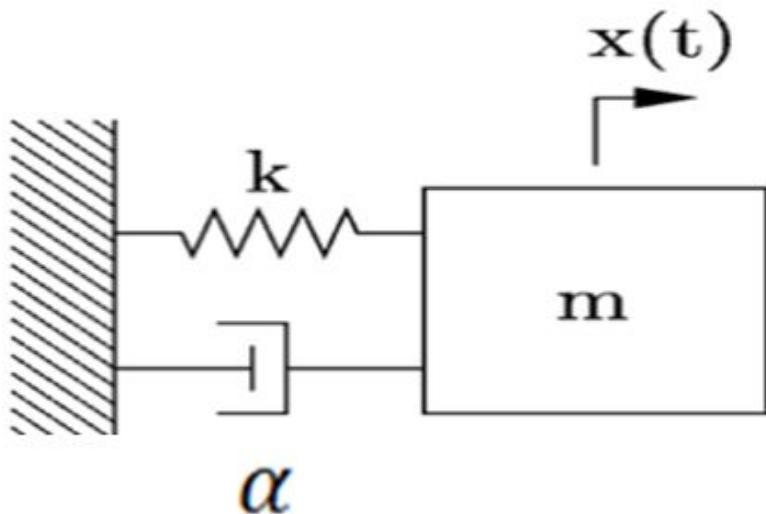
### 2- Differential equation:

$$\Rightarrow m\ddot{x} + \alpha\dot{x} + kx = 0 \quad \Rightarrow \ddot{x} + \frac{\alpha}{m}\dot{x} + \frac{k}{m}x = 0 \quad \Rightarrow \ddot{x} + 2\delta\dot{x} + \omega_n^2x = 0$$

Where:

- $\omega_n = \sqrt{\frac{k}{m}}$  The natural frequency of the system.
- $2\delta = \frac{\alpha}{m} \Rightarrow \delta = \frac{\alpha}{2m}$

$\delta$ : Damping factor or damping ratio



## Chapter 3 : Damped free vibration of single degree of freedom systems

### 3- Solution of differential equation :

- ✓ The equation of motion here is like that of an undamped system, with the addition of an extra term to account for the damping in this model  
It is a second-order linear differential equation with constant coefficients.

$$\ddot{x} + 2\delta\dot{x} + w_n^2 x = 0$$

The characteristic equation of this equation is written as:

$$r^2 + 2\delta r + w_n^2 = 0$$

- ✓ We distinguish **three regimes** based on the sign of the reduced discriminant:

$$\Delta = \sqrt{\delta^2 - w_n^2}$$

## Chapter 3 : Damped free vibration of single degree of freedom systems

### 3- Solution of differential equation :

#### 1- Overdamped response $\Delta > 0$ :

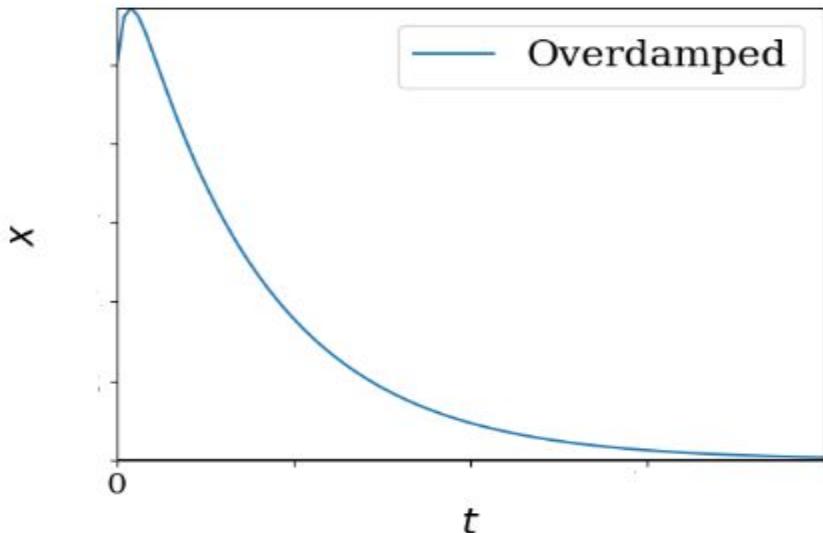
$$\Delta > 0 \Rightarrow \delta > w_n$$

- ✓ The characteristic equation has two real solutions

The solution is written as:

$$x(t) = e^{-\delta t} (Ae^{-(\sqrt{(\delta^2 - w_n^2)}t)} + Be^{\sqrt{(\delta^2 - w_n^2)}t})$$

- ✓ In an **overdamped** system, the return to equilibrium occurs **without oscillation** and more **slowly**, due to **significant damping**



## Chapter 3 : Damped free vibration of single degree of freedom systems

### 3- Solution of differential equation :

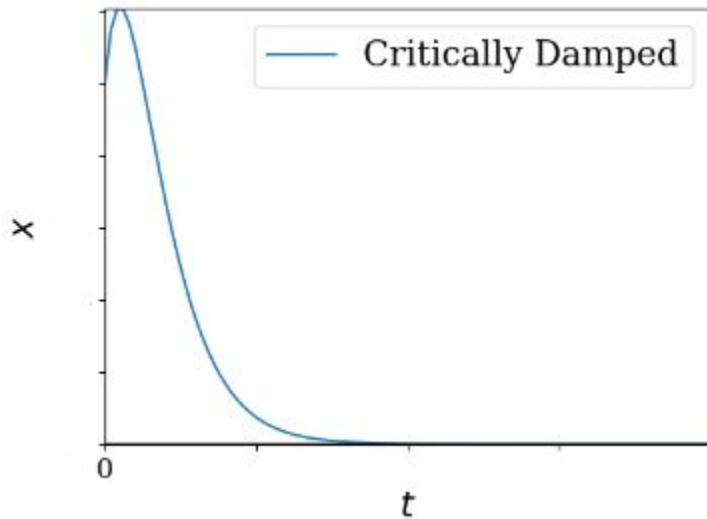
#### 2- Critically damped response $\Delta = 0$ :

$$\Delta = 0 \Rightarrow \delta = w_n$$

- ✓ The characteristic equation has double solution

The solution is written as:

$$x(t) = e^{-\delta t} (A + Bt)$$



- ✓ The **critically damped** system returns to its equilibrium state as **quickly** as possible **without oscillating**. In this regime, the damping is just sufficient to prevent oscillations, and the system reaches equilibrium in **minimal time**

## Chapter 3 : Damped free vibration of single degree of freedom systems

### 3- Solution of differential equation :

#### 3- Underdamped response $\Delta < 0$ :

$$\Delta < 0 \quad \Rightarrow \quad \delta < w_n$$

- ✓ The characteristic equation has two complex roots

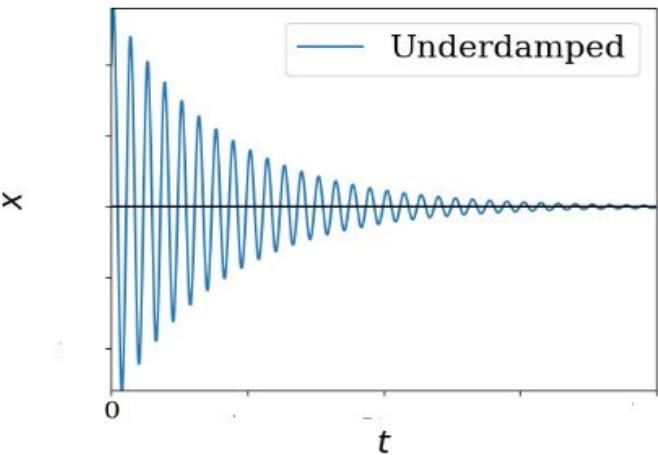
The solution is written as:

$$x(t) = A e^{-\delta t} \cos(w_d t + \varphi)$$

- ✓ Where  $w_d$  : pseudo frequency or damped natural frequency

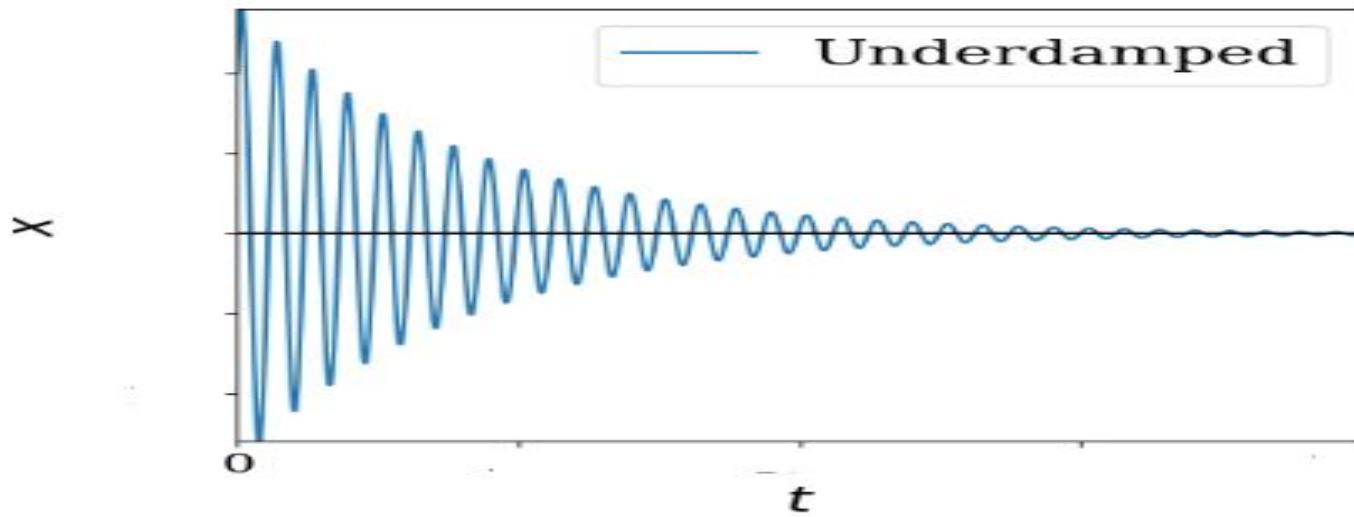
$$w_d = \sqrt{w_n^2 - \delta^2}$$

- ✓ The pseudo period or damped period  $T_d = \frac{2\pi}{w_d} \Rightarrow T_d = \frac{2\pi}{\sqrt{w_n^2 - \delta^2}}$



## Chapter 3 : Damped free vibration of single degree of freedom systems

### 3- Solution of differential equation :

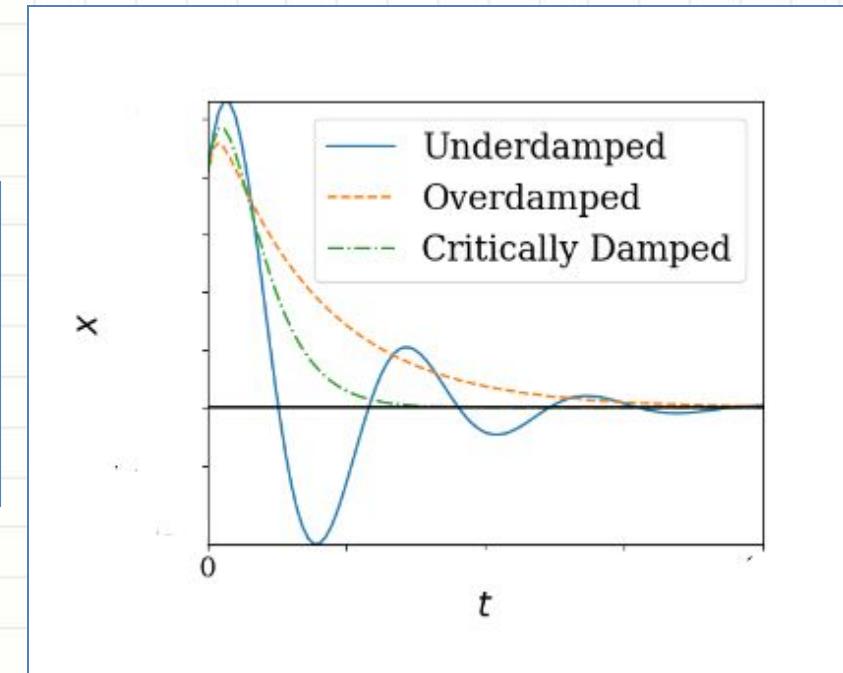


- ✓ In the **underdamped** regime, the system **oscillates** around its equilibrium position with an amplitude that gradually **decreases** over time due to damping

## Chapter 3 : Damped free vibration of single degree of freedom systems

### 3- Solution of differential equation :

✓ 
$$\begin{cases} T_n = \frac{2\pi}{w_n} \\ T_d = \frac{2\pi}{w_d} = \frac{2\pi}{\sqrt{w_n^2 - \delta^2}} \end{cases} \Rightarrow T_d > T_n$$



## Chapter 3 : Damped free vibration of single degree of freedom systems

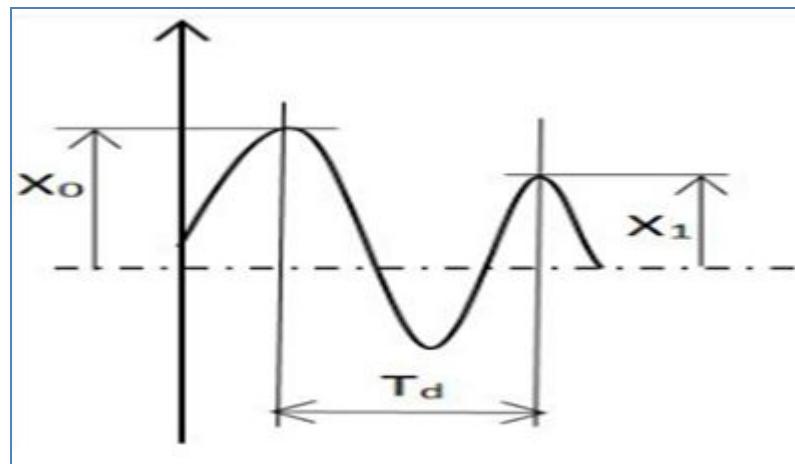
### 4-Logarithmic decrement

The **logarithmic decrement** is a measure used in system dynamics to quantify the decay of a damped oscillation. It represents the logarithmic ratio between two successive oscillation **amplitudes**. It indicates how **quickly** a system **loses energy** and dampens over time.

- In a general way

$$D = \frac{1}{n} \ln \frac{q(t)}{q(t+nT_d)}$$

- $q(t)$  is the amplitude at time  $t$
- $q(t + nT_d)$  is the amplitude after  $nT_d$  period



## Chapter 3 : Damped free vibration of single degree of freedom systems

### 4-Logarithmic decrement

#### ➤ Underdamped response

$$D = \frac{1}{n} \ln \frac{x(t)}{x(t+nT_d)} \Rightarrow D = \frac{1}{n} \ln \frac{Ae^{-\delta t} \cos(w_d t + \varphi)}{Ae^{-\delta(t+nT_d)} \cos(w_d(t+nT_d) + \varphi)}$$

$$\Rightarrow D = \frac{1}{n} \ln \frac{Ae^{-\delta t} \cos(w_d t + \varphi)}{Ae^{-\delta t} \cdot e^{-n\delta T_d} \cos(w_d t + \varphi)} \Rightarrow D = \frac{1}{n} \ln e^{n\delta T_d} \Rightarrow D = \delta T_d$$

## Chapter 3 : Damped free vibration of single degree of freedom systems

### 5-Quality factor

The quality factor  $Q$  of a system is a dimensionless measure of the damping rate of an oscillator.

- The lower the damping, the higher the quality of the system.
- There are three definitions of  $Q$ :

## Chapter 3 : Damped free vibration of single degree of freedom systems

### 5-Quality factor

#### 1. Related to energy:

$$Q = 2\pi \frac{E_{\text{Sys}}}{E_{\text{Diss}}}$$

Where

- $E_{\text{Sys}}$  is the maximum energy contained in the system
- $E_{\text{Diss}}$  is the energy dissipated by the system over a pseudo-period

## Chapter 3 : Damped free vibration of single degree of freedom systems

### 5-Quality factor

2-Related to differential equation of the system:

$$Q = \frac{w_n}{2\delta}$$

$$\overbrace{\phantom{...}}^Q$$

$$\ddot{x} + 2\delta\dot{x} + w_n^2 x = 0$$

- The quality factor is used to determine the type of transient regime in an oscillatory system (**Overdamped**, **Critically damped**, or **Underdamped**)

$$\ddot{x} + \frac{w_n}{Q} \dot{x} + w_n^2 x = 0$$

## Chapter 3 : Damped free vibration of single degree of freedom systems

### 5-Quality factor

**1-Overdamped response  $\Delta > 0$  :**

$$\Delta = \sqrt{\delta^2 - w_n^2}$$

$$\begin{aligned} \Delta > 0 &\Rightarrow \delta > w_n \Rightarrow \frac{\delta}{w_n} > 1 \\ \frac{w_n}{2\delta} < \frac{1}{2} &\Rightarrow Q < \frac{1}{2} \end{aligned}$$

**2- Critically damped response  $\Delta = 0$  :**

$$\delta = w_n \Rightarrow Q = \frac{1}{2}$$

**3- Underdamped response  $\Delta < 0$  :**

$$\delta < w_n \Rightarrow Q > \frac{1}{2}$$

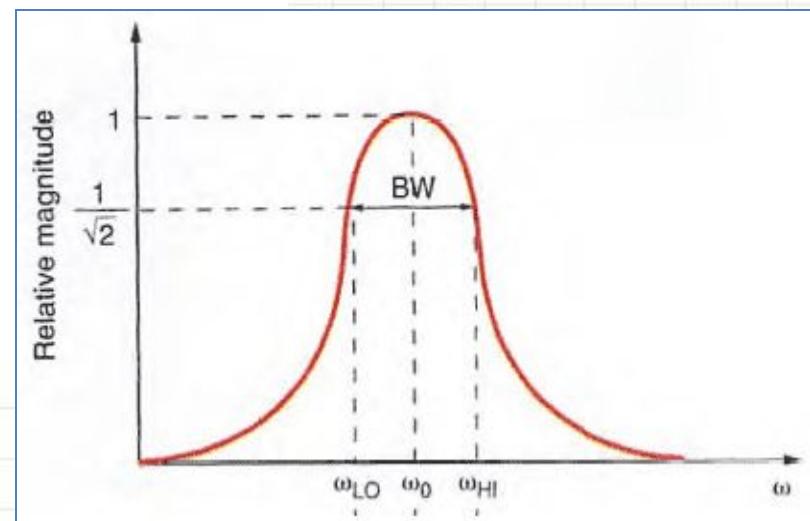
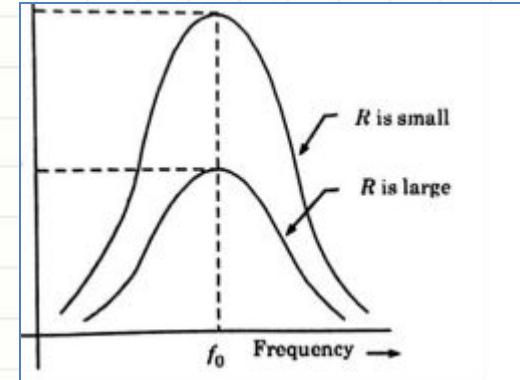
# Chapter 3 : Damped free vibration of single degree of freedom systems

## 5-Quality factor

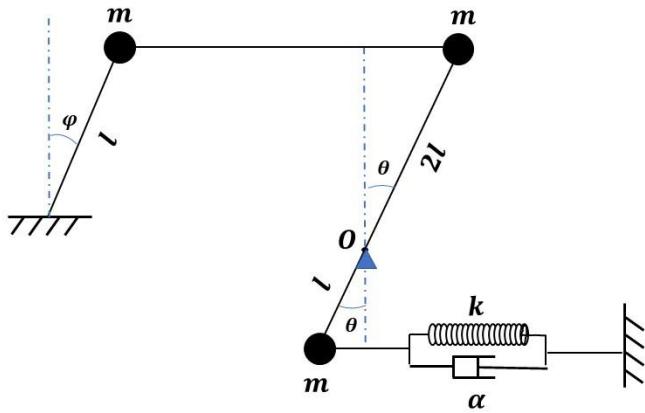
3-Related to the bandwidth :

$$Q = \frac{\omega_r}{B} \quad (\text{Chapter 4})$$

- The **bandwidth** is the range of frequencies over which a system responds effectively to vibrations. ( forced systems)
- The **higher** the quality factor, the **smaller** the bandwidth
- The higher the Q, the more selective the filter  
(Filter electronic, acoustic, optical....)



## Chapter 3 : Damped free vibration of single degree of freedom systems



1. Determine the potential energy of the system and the condition for oscillation in the absence of friction.
2. Determine the kinetic energy as well as the dissipation function, and deduce the equation of motion for the damped system.
3. What are the values of  $\alpha$  required to maintain the system in oscillation?

## Chapter 3 : Damped free vibration of single degree of freedom systems

