



ESAME DI MECCANICA I - Corso di Laurea in Ing. Biomedica

ESAME DI MECCANICA TEORICA ED APPLICATA - Corso di Laurea in Ing. Robotica e dell'automazione

COGNOME _____ NOME _____ MATRICOLA _____ CDL _____

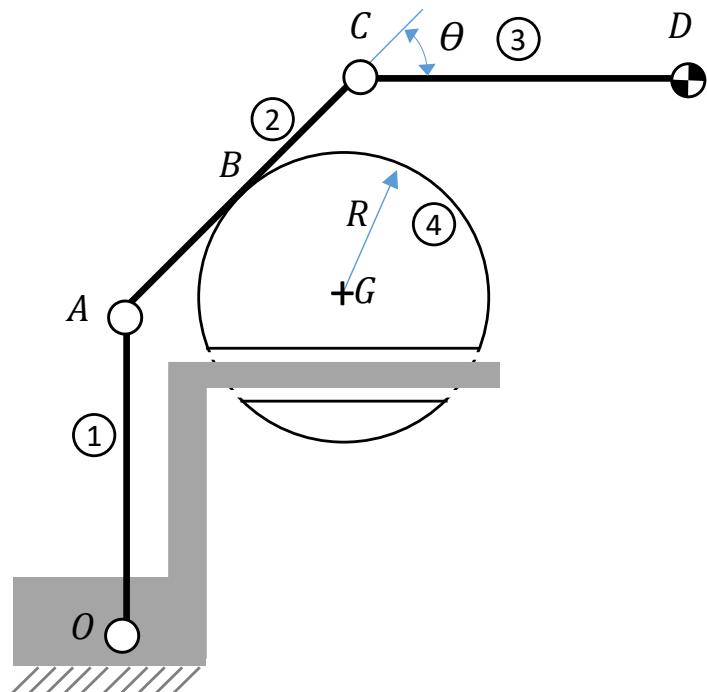
Esercizio 1

Si consideri il meccanismo in figura, costituito da 4 corpi: si noti che risulta essere formato da due sotto-mecchanismi: 1-2-3 e 1-2-4. Sia nota la configurazione del meccanismo nell'atto di moto considerato, e la velocità del corpo 1, ω oraria e costante nel tempo:

- 1) Fare l'analisi geometrica dei vincoli. Valutare il tipo di rotolamento in B per avere un 1 gdl.
- 2) Definire il moto assoluto di tutti i corpi e il moto relativo dei corpi dei corpi collegati.
- 3) Che tipo di meccanismo è quello costituito da 1-2-3?
- 4) Risolvere il problema delle velocità dei sotto-mecchanismi 1-2-3 e 1-2-4:
 - Scrivere le eq.ni. di chiusura in forma vettoriale e scalare, in forma parametrica.
 - Risolvere graficamente il problema delle velocità.
- 5) Valutare numericamente la velocità di G del corpo 4.
- 6) Valutare tutti i centri delle velocità assoluti e il centro delle velocità del moto relativo 2-4.
- 7) Scrivere le eq.ni di chiusura delle accelerazioni dei meccanismi 1-2-3 e 1-2-4

Dati:

$\theta = 45^\circ$, $OA = CD = l = 2.5 \text{ m}$,
 $AB = BC = AC/2$, $AC = l\sqrt{2}$,
 $R = 1.8 \text{ m}$, $|\omega| = 4.5 \text{ rad/s}$.



Esercizio 2

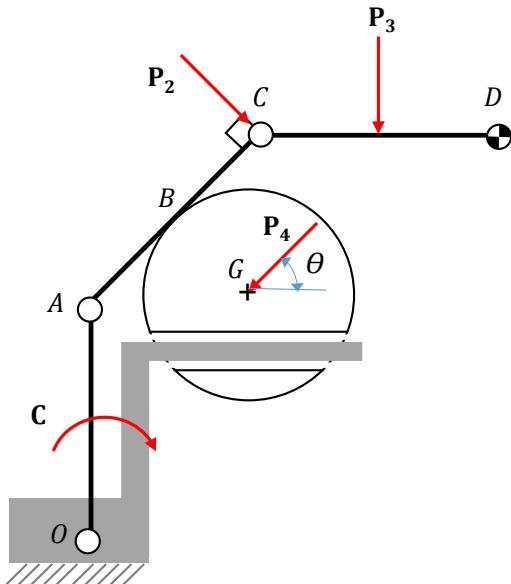
Si consideri il meccanismo dell'esercizio precedente a cui siano applicate 3 forze note \mathbf{P}_2 , \mathbf{P}_3 e \mathbf{P}_4 la cui azione è equilibrata da un momento \mathbf{C} applicato al corpo 1, incognito. Le masse di tutti i corpi sono trascurabili.

È richiesto di

- 1) Fare l'analisi fisica dei vincoli e valutare se il sistema è complessivamente o esternamente isostatico.

Applicando il PSE valutare in forma parametrica (lasciare la soluzione in funzione di l e R)

- 2) DCL definitivo di tutti i corpi per ogni sottocaso
- 3) Contributo a \mathbf{C} per ogni sottocaso
- 4) \mathbf{C} complessivo
- 5) Quanto vale \mathbf{C} se \mathbf{P}_2 raddoppia intensità? E se invece \mathbf{P}_4 dimezza intensità?

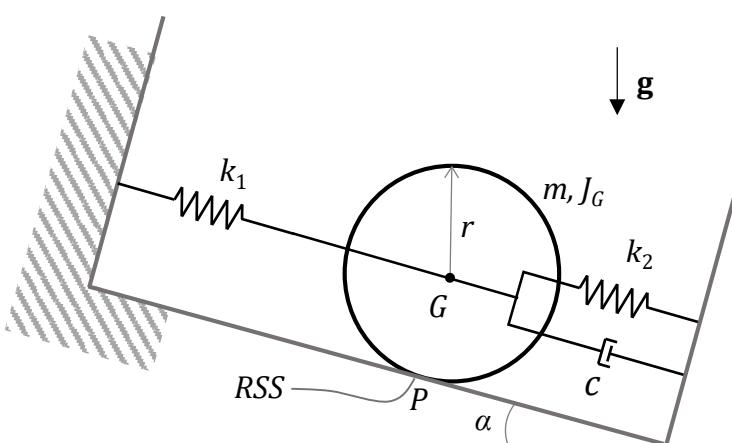


Esercizio 3

Il disco omogeneo mostrato in figura, collegato al telaio mediante due molle e uno smorzatore, rotola senza strisciare su un piano inclinato, soggetto ad oscillazioni indotte da posizione iniziale pari alla freccia statica. e velocità iniziale verso il basso di G pari a v_0 .

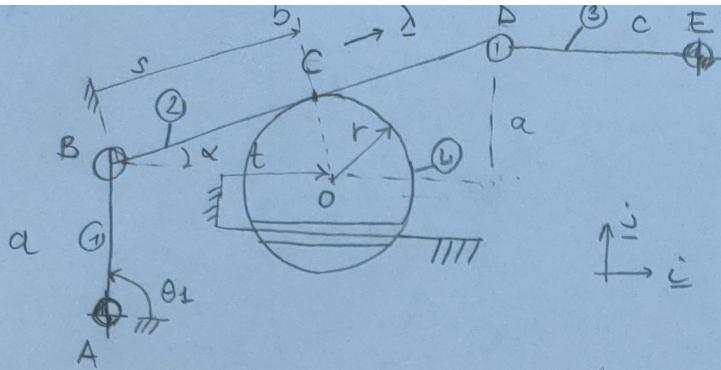
Si vuole studiare la dinamica del disco. Siamo in presenza di gravità.

- 1) Specificare in modo chiaro il sistema di riferimento, la coordinata lagrangiana e le eventuali equazioni di congruenza.
- 2) Rappresentare DCL del disco.
- 3) Valutare la freccia statica.
- 4) Valutare l'equazione del moto in forma canonica.
- 5) Valutare la pulsazione naturale, il fattore di smorzamento e la pulsazione smorzata in forma parametrica e numerica.
- 6) Trovare la legge oraria a regime (fare i passaggi) e rappresentarla graficamente.



Dati:

$$m = 15 \text{ kg}, r = 0.1 \text{ cm}, k_1 = 12 \text{ N/m}, \\ k_2 = 8 \text{ N/m}, c = 10 \text{ N m/s}, v_0 = 2 \text{ m/s}, \\ \alpha = 45^\circ$$



$$gdk = 2 \times 3 - 4 \times 2 - 1 \times 2 - x = 1$$

CER CP

$$12 - 8 - 2 - x = 1$$

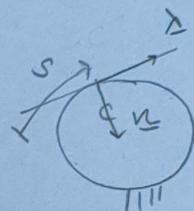
$$x = 1 \quad RCS$$

$$\underline{\omega}_1 = \dot{\theta}_1 \underline{k} \quad \text{con } \dot{\theta}_1 < 0 \quad \text{nata}$$

↓
2 meccanismi
debolmente accoppiati

NB

$$\underline{\omega}_{c2} \neq \underline{\omega}_{c4} \Rightarrow \underline{\omega}_{24}^{\text{rel}} \neq 0 \quad \underline{\omega}_{42}^{\text{rel}} = \dot{s} \underline{\lambda}$$



$$\sum \underline{\omega}_{ci}^{\text{ass}} = \underline{\omega}_c^{\text{rel}} + \underline{\omega}_c^{\text{tr}} = -\dot{s} \underline{\lambda} + \underline{\omega}_{B2} + \dot{\theta}_2 K_1 \vec{BC}$$

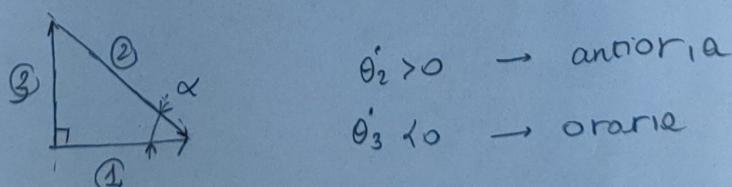
$$= \dot{s} \underline{\lambda} + \dot{\theta}_1 K_1 \vec{AB} + \dot{\theta}_2 K_1 \vec{BC} = t \underline{z}$$

? ? ?

Risolv il problema omologato

$$\underline{\omega}_{B1} = \underline{\omega}_{B2} \quad \dot{\theta}_1 \underline{k} \wedge \vec{AB} = \dot{\theta}_2 \underline{k} \wedge \vec{DB} + \dot{\theta}_3 \underline{l} \wedge \vec{ED}$$

(1) (2) (3)



$\dot{\theta}_2 > 0 \rightarrow$ antioraria
 $\dot{\theta}_3 < 0 \rightarrow$ oraria

$$|\dot{\theta}_1| a = |\dot{\theta}_3| a \Rightarrow |\dot{\theta}_1| = |\dot{\theta}_3|$$

$$\dot{\theta}_1 = \dot{\theta}_3 < 0 \quad \underline{\omega}_3 \downarrow$$

$$\dot{\theta}_2 b \cos \alpha = |\dot{\theta}_1| a$$

$$\dot{\theta}_2 = \frac{a}{b \cos \alpha} |\dot{\theta}_1| \quad \dot{\theta}_2 = \frac{\pi \alpha}{2} \cdot \frac{2}{\pi} |\dot{\theta}_1| \quad \underline{\omega}_2 \uparrow$$

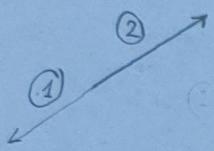
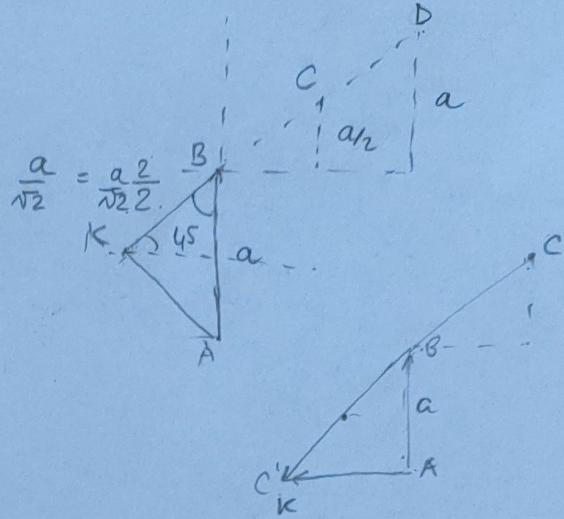
$$b = \frac{a}{\cos \alpha} = \frac{a^2}{\sqrt{2}}$$

$$\text{se } b = \frac{2a}{\sqrt{2}} \quad \Rightarrow \quad = |\dot{\theta}_1| \quad \dot{\theta}_2 = -\dot{\theta}_1$$

$$\dot{s}\lambda + \dot{\theta}_1 \underline{K} \wedge \overrightarrow{AB} - \dot{\theta}_1 \underline{K} \wedge \overrightarrow{BC} = \dot{t} \underline{i}$$

$$\dot{s}\lambda + \dot{\theta}_1 \underline{K} \wedge (\overrightarrow{AB} - \overrightarrow{BC}) = \dot{t} \underline{i}$$

① ② ③



$$\begin{cases} \dot{t} = 0 \\ \dot{s} > 0 \end{cases}$$

$$\begin{pmatrix} -a_{12} \\ +a_{12} \end{pmatrix}$$

$$\dot{s} \begin{pmatrix} \cos \alpha \\ \sin \alpha \end{pmatrix} + \dot{\theta}_1 \underline{K} \wedge \left[\begin{pmatrix} 0 \\ a \end{pmatrix} - \begin{pmatrix} a_{12} \\ a_{12} \end{pmatrix} \right] = \begin{pmatrix} \dot{t} \\ 0 \end{pmatrix}$$

$$\dot{s} \begin{pmatrix} \frac{\sqrt{2}}{2} \\ \frac{\sqrt{2}}{2} \end{pmatrix} + \begin{vmatrix} i & j & \underline{K} \\ 0 & 0 & \dot{\theta}_1 \\ -a_{12} & a_{12} & 0 \end{vmatrix} = \begin{pmatrix} \dot{t} \\ 0 \end{pmatrix}$$

$$-j \dot{\theta}_1 \frac{a}{2} + \frac{a}{2} \dot{\theta}_1 \underline{i}$$

$$\begin{cases} \dot{s} \frac{\sqrt{2}}{2} - \frac{a}{2} \dot{\theta}_1 = \dot{t} \\ \dot{s} \frac{\sqrt{2}}{2} - \dot{\theta}_1 \frac{a}{2} = 0 \end{cases}$$

$$\begin{cases} \dot{s} = \dot{\theta}_1 \frac{a}{2} \frac{2}{\sqrt{2}} = \frac{a}{\sqrt{2}} \dot{\theta}_1 \\ \dot{\theta}_1 \frac{a}{\sqrt{2}} \frac{\sqrt{2}}{2} - \frac{a}{2} \dot{\theta}_1 = \dot{t} = 0 \end{cases} \quad \dot{s} < 0$$

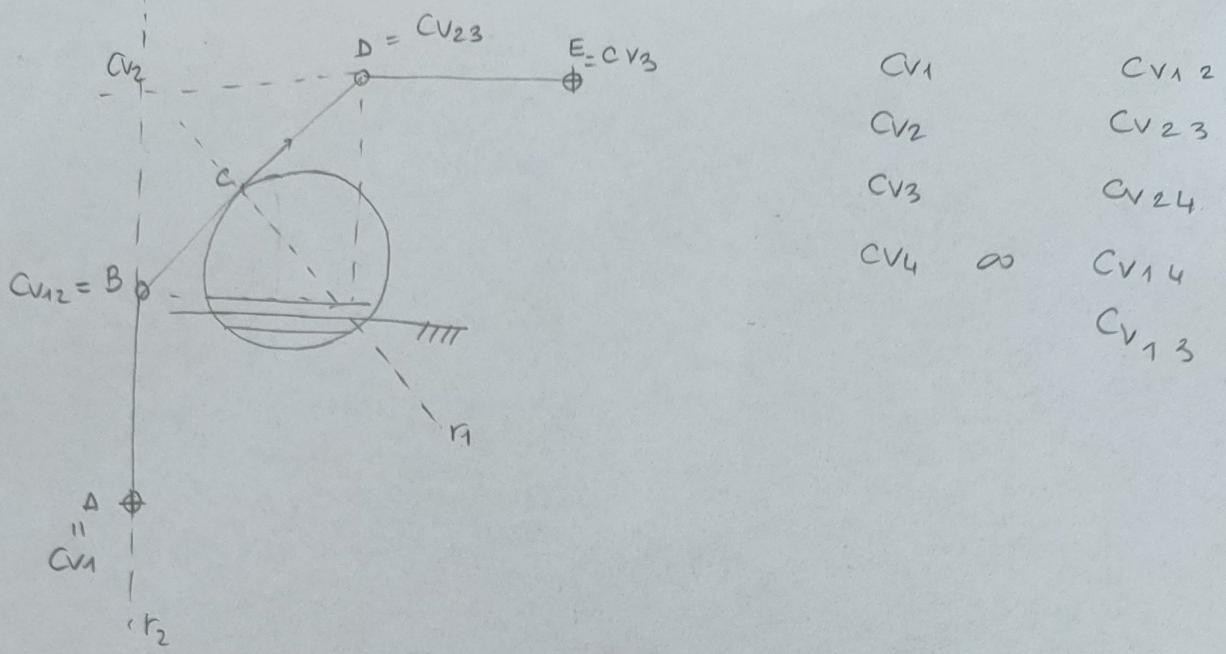
$$\underline{\omega}_{12} = (\dot{\theta}_1 - \dot{\theta}_2) \underline{K} = \dot{\theta}_1 - (-\dot{\theta}_2) \underline{K} = 2 \dot{\theta}_1 \underline{K}$$

$\rightarrow \underline{\omega}_{12}$

$$\underline{\omega}_{23} = (\dot{\theta}_2 - \dot{\theta}_3) \underline{K} = (-\dot{\theta}_1 - \dot{\theta}_1) \underline{K} = -2 \dot{\theta}_1 \underline{K}$$

$\rightarrow \underline{\omega}_{23}$

$$\underline{\omega}_{42} = \underline{\omega}_4 - \underline{\omega}_2 = -\underline{\omega}_2 = -\dot{\theta}_2 \underline{K} = \dot{\theta}_1 \underline{K}$$



$$CV_{24} : \underline{\alpha}_C = \dot{s}\underline{\lambda} \rightarrow r_1 \\ \text{prendiamo triploto: } CV_2 \quad CV_4 \quad CV_{24} \rightarrow r_2 \quad \left. \right\} CV_2 = CV_{24}$$

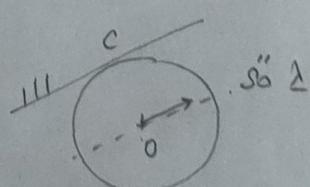
$$\underline{\alpha}_{B(1)} = \underline{\alpha}_{B(2)}$$

$$-\ddot{\theta}_1^2 \underline{k} \wedge \overrightarrow{AB} = \ddot{\theta}_2 \underline{k} \wedge \overrightarrow{DB} - \dot{\theta}_2^2 \overrightarrow{DB} + \ddot{\theta}_3 \underline{k} \wedge \overrightarrow{ED} - \dot{\theta}_3^2 \overrightarrow{ED}$$

$$\underline{\alpha}_{C(4)} = \underline{\epsilon}^i = \underline{\alpha}_{C(0)} - \sum \underline{\alpha}_{(2)} \quad || \\ \downarrow \\ \text{attenzione} \\ \text{non sappiamo} \\ \text{termine } \underline{\alpha}_{(2)}^{\text{rel}}$$

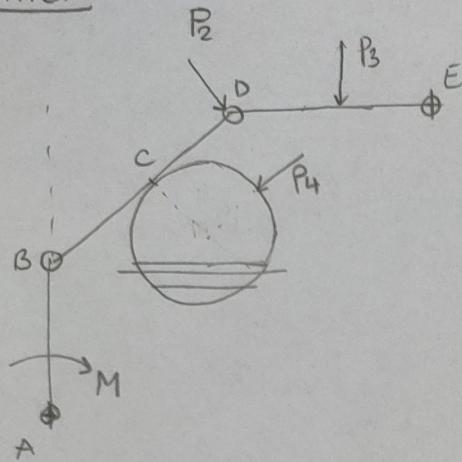
$$\rightarrow \frac{\ddot{s}^2 \underline{\lambda}}{?} + \frac{\dot{s}^2 \underline{n}}{\underline{P}_c ?}$$

$$\underline{\alpha}_{O(0)} = \underline{\epsilon}^i \\ \underline{\alpha}_{O(0)} = \underline{\alpha}_0^{\text{rel}} + \underline{\alpha}_0^{\text{tr}} + \underline{\alpha}_0^{\text{cor}} = 2 \underline{\omega}^{\text{tr}} \wedge \underline{\nu}^{\text{rel}} \\ \sum \underline{\alpha}_{(2)} = \underline{\alpha}_0^{\text{rel}} + \underline{\alpha}_0^{\text{tr}} + \underline{\alpha}_0^{\text{cor}} = 2 \underline{\omega}^{\text{tr}} \wedge \underline{\nu}^{\text{rel}} \\ \ddot{s}_0 \underline{\lambda} = -\dot{\theta}_1^2 \overrightarrow{AB} + \ddot{\theta}_2 \underline{k} \wedge \overrightarrow{BO} - \dot{\theta}_2^2 \overrightarrow{BO}$$



$$\underline{\nu}^{\text{rel}} = \underline{\omega}_0^{\text{rel}} + \underline{\omega}^{\text{rel}} \wedge \overrightarrow{CO} = \dot{s}\underline{\lambda} + \underline{\omega}_{42} \wedge \overrightarrow{CO} = \dot{s}\underline{\lambda} + \dot{\theta}_1 \underline{k} \wedge \overrightarrow{CO}$$

STATICA



AN. FÍSICA

$$\phi - 2 \text{ gols}$$

$$\frac{1}{\Gamma} - 2 \text{ gols}$$

$$O = \begin{pmatrix} R_D \\ R_C \end{pmatrix}$$

$$2 \times 4$$

$$2 \times 1$$

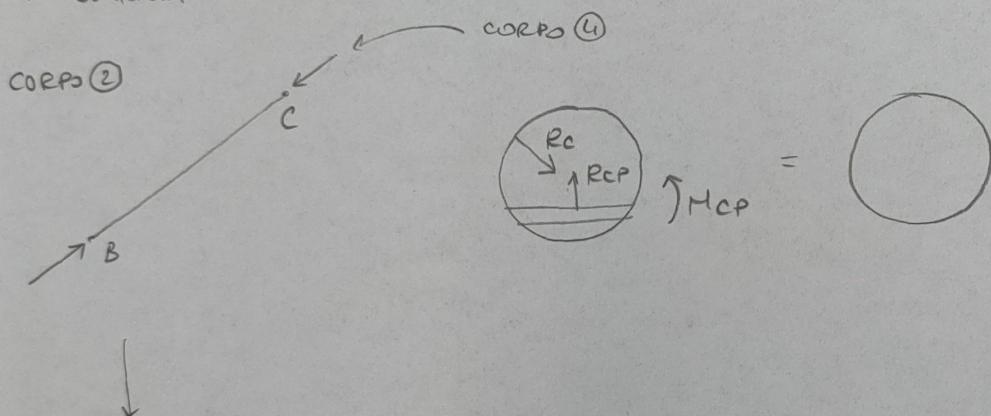
$$\frac{1}{(11)}$$

+ 1 INC. ATIVA

$$11 + 1 = 12 \text{ INC. VS } 12 \text{ EPNI}$$

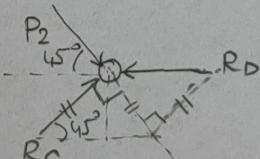
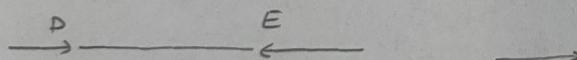
CASO P2

CORPO SCARICO:



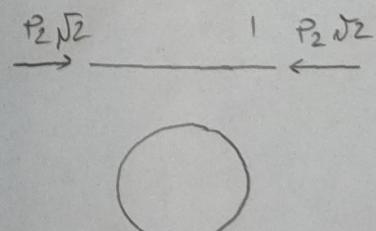
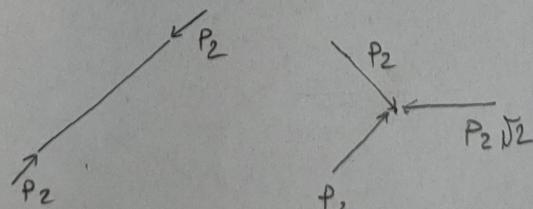
PERNS CARICO

$$P_2 = R_C = R_D$$



PCL DEF

$$P_2 \begin{cases} \rightarrow \\ \downarrow \\ \leftarrow \end{cases} M = P_2 a \frac{\sqrt{2}}{2}$$



$$M = P_2 a \cos \alpha = \frac{\sqrt{2}}{2} a P_2$$

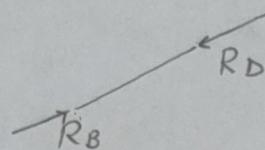
$$\left\{ \begin{array}{l} P_2 \frac{\sqrt{2}}{2} - R_C \frac{\sqrt{2}}{2} = 0 \\ P_2 \frac{\sqrt{2}}{2} + R_C \frac{\sqrt{2}}{2} = R_D \end{array} \right. \quad \begin{array}{l} P_2 = R_C \\ R_D = \sqrt{2} P_2 \end{array}$$

CASO P₃

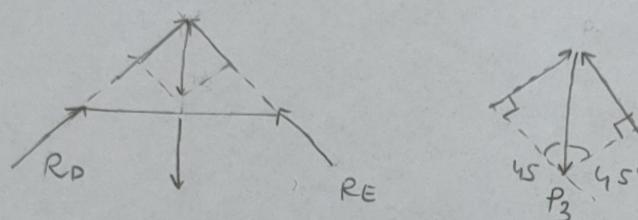
CORPO 4 SCARICO



CORPO 2 SCARICO



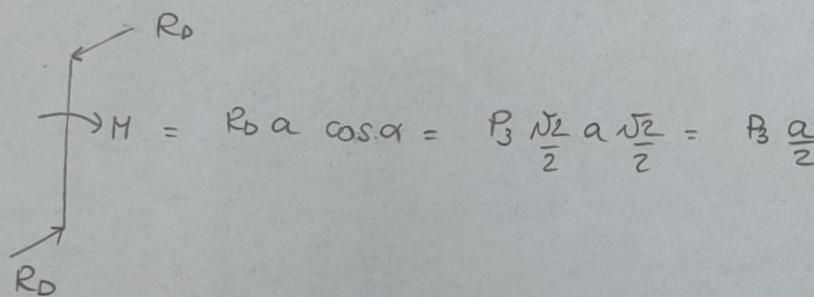
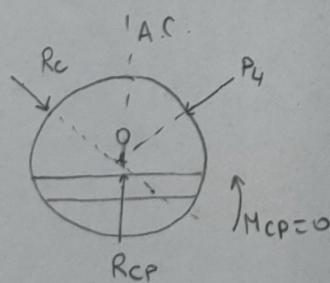
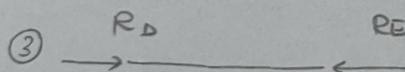
CORPO 3



$$R_D = P_3 \frac{\sqrt{2}}{2} = R_E$$

R

CORPO 1

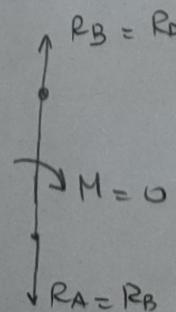
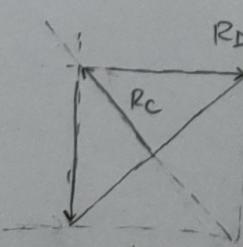
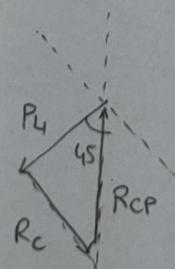
CASO P₄

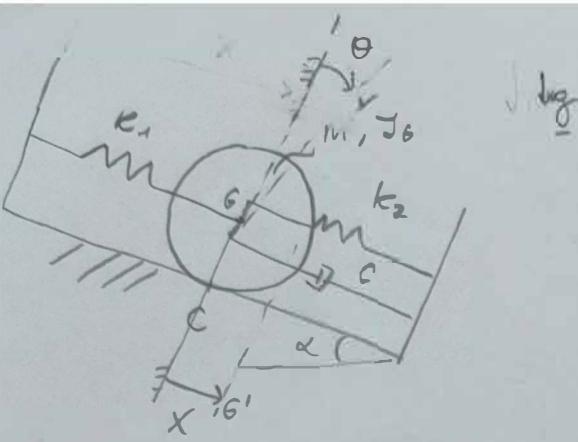
$$R_C = P_4$$

$$R_{Cp} = \frac{P_4}{\sin \alpha} = \frac{2}{\sqrt{2}} P_4 = \sqrt{2} P_4$$

$$M_{Cp} = 0$$

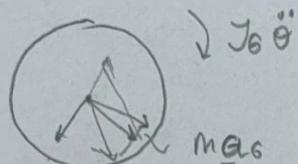
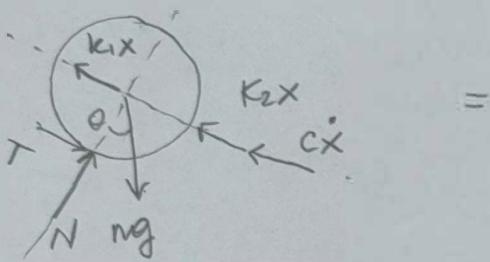
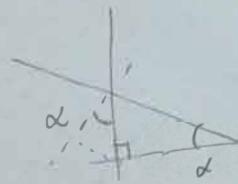
$$R_D = \frac{R_C 2}{\sqrt{2}} = P_4 \sqrt{2}$$





$$\alpha = 30^\circ$$

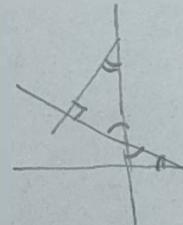
$$x = r\theta$$



Frecueza estatica

$$c) (k_1 + k_2) \tilde{x} r - mg \sin \alpha r = 0$$

$$\tilde{x} = \frac{mg \sin \alpha}{k_1 + k_2}$$



CONDIZ. DINAMICHE

$$c) (k_1 x + k_2 x) r + c \dot{x} r - mg \sin \alpha r = -J_0 \ddot{\theta} + (m \overline{CG}/r \underline{a}_c) \cdot K$$

$$\left(\frac{1}{2} m r^2 + m r^2 \right) \ddot{\theta} + (k_1 + k_2) x r \parallel + c r \dot{x} \parallel = m g \sin \alpha r$$

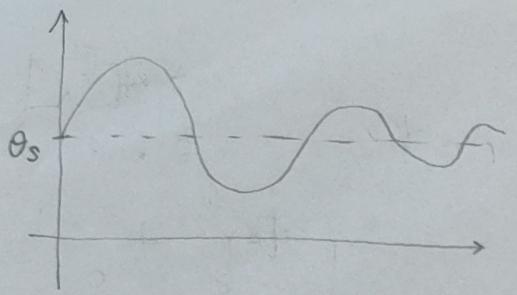
$$\frac{3}{2} m r^2 \ddot{\theta} + c r^2 \dot{\theta} + (k_1 + k_2) r^2 \theta = m g \sin \alpha r$$

$$\tilde{J} \ddot{\theta} + \tilde{c} \dot{\theta} + \tilde{k} \theta = M_0$$

L, forzante costante

\downarrow
soluz. particolare

$$\ddot{\theta} = \frac{\tilde{x}}{r} = \frac{mg \sin \alpha}{r(k_1 + k_2)}$$



$$\dot{\theta}_p = A \sin(\omega_n t + \varphi) e^{-\xi \omega_n t} + \dot{\theta}_s$$

$$\theta_p = A \sin(\omega_n t + \varphi) e^{-\xi \omega_n t} + \theta_s$$

$$\ddot{\theta}_p = A (-\xi \omega_n) \sin(\omega_n t + \varphi) e^{-\xi \omega_n t} + A \omega_n^2 \cos(\omega_n t + \varphi)$$

$$x_p(0) = x_s \rightarrow \varphi$$