

a) The Max Von-Mises stress is  $312.5 \text{ MPa} < 615 \text{ MPa}$ . No yield  
 P1 Max displacement is  $2.1 \text{ mm}$  It's safe.

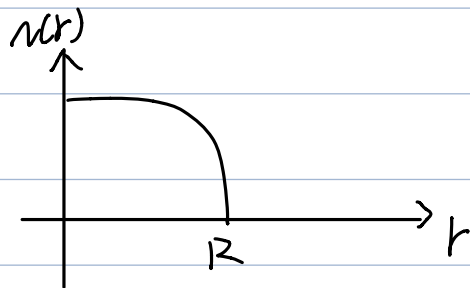
b) Peak shear stress on the leg is  $138 \text{ MPa}$

c)  $T = 125 \text{ N} \left( 0.015 - \frac{0.005}{2} - \frac{0.025}{2} \right) = 7.5 \text{ N}\cdot\text{m}$

$$T = 2 \int_A \phi dA \Rightarrow 7.5 = 2 \int_A \phi_0 v(x, y) dA$$

$$7.5 = 2 \phi_0 \int_0^R \int_0^{2\pi} v(r) r d\theta dr$$

$$\phi_0 = \frac{3.75}{\int_0^R \int_0^{2\pi} v(r) r d\theta dr}$$



Guess:  $v(r) = 1 - \left(\frac{r}{R}\right)^2 \Rightarrow \phi_0 = \frac{3.75}{\int_0^R \int_0^{2\pi} \left(1 - \left(\frac{r}{R}\right)^2\right) r d\theta dr}$

$$\phi_0 = \frac{7.5}{\pi R^2} \leftarrow R = 5.5 \text{ mm}$$

$$\phi_0 = 315679.23$$

$$\phi(r) = \phi_0 \cdot v(r) \Rightarrow \phi(r) = \phi_0 \left(1 - \frac{r^2}{R^2}\right)$$

In polar:  $x = r \cos \theta$   $y = r \sin \theta$

$$\phi(x, y) = \phi_0 \left(1 - \frac{x^2 + y^2}{R^2}\right) = \phi_0 - \frac{\phi_0 x^2}{R^2} + \frac{\phi_0 y^2}{R^2}$$

$$\frac{d\phi}{dx} = 6yz \quad \frac{d\phi}{dy} = 6xz \quad \begin{cases} \frac{\partial}{\partial x} \phi(x, y) = \gamma_{yz}(x, y) = -\frac{2\phi_0 x}{R^2} \\ \frac{\partial}{\partial y} \phi(x, y) = \gamma_{xz}(x, y) = -\frac{2\phi_0 y}{R^2} \end{cases}$$

$$R = \frac{55}{2} \Rightarrow \gamma_{yz} = -10107352320X$$

$$I_{xz} = -101017352320 \text{ y}$$

(d) The hand-calculation is more precise compare to the FEM, but we are using a circle to approach, it might not be that good.

$$P2 \quad B_e = \begin{bmatrix} \frac{\partial N}{\partial x} & 0 & 0 \\ 0 & \frac{\partial N}{\partial y} & 0 \\ 0 & 0 & \frac{\partial N}{\partial z} \\ \frac{\partial N}{\partial y} & \frac{\partial N}{\partial x} & 0 \\ 0 & \frac{\partial N}{\partial z} & \frac{\partial N}{\partial y} \\ \frac{\partial N}{\partial z} & 0 & \frac{\partial N}{\partial x} \end{bmatrix} \quad \begin{aligned} \frac{\partial N}{\partial x} &= -\frac{1}{L_x} \\ \frac{\partial N}{\partial y} &= -\frac{1}{L_y} \\ \frac{\partial N}{\partial z} &= -\frac{1}{L_z} \end{aligned}$$

$$N(x, y, z) = \left(1 - \frac{x}{L_x}\right) \left(1 - \frac{y}{L_y}\right) \left(1 - \frac{z}{L_z}\right)$$

$$K_{ii} = \frac{E}{(v+1)} \int_0^{L_z} \int_0^{L_y} \int_0^{L_x} \frac{v-1}{2v-1} \left( \left( \frac{\partial N}{\partial x} \right)^2 + \left( \frac{\partial N}{\partial y} \right)^2 + \left( \frac{\partial N}{\partial z} \right)^2 \right) dx dy dz$$

$$K_{ii} = \frac{E}{(v+1)} \int_0^{L_z} \int_0^{L_y} \int_0^{L_x} \frac{v-1}{2v-1} \left( \frac{1}{L_x^2} + \frac{1}{L_y^2} + \frac{1}{L_z^2} \right) dx dy dz$$

$$K_{ii} = \frac{E}{v+1} x y z \left( \frac{v-1}{2v-1} \left( \frac{1}{L_x^2} + \frac{1}{L_y^2} + \frac{1}{L_z^2} \right) \right) \Big|_0^{L_x, L_y, L_z}$$

$$K_{ii} = \frac{E L_x L_y L_z}{v+1} \left[ \frac{v-1}{2v-1} \left( \frac{1}{L_x^2} + \frac{1}{L_y^2} + \frac{1}{L_z^2} \right) \right]$$

$$K_{ii} = \frac{E}{v+1} \left[ \frac{v-1}{2v-1} \frac{L_y L_z}{L_x} + \frac{L_x L_z}{L_y} + \frac{L_x L_y}{L_z} \right]$$

P3

Kirchhoff-Love Plate theory

$$K \nabla^4 w = q(x, y) = q_0$$

$$B.C.s: \begin{cases} w(x=0) = 0 & w''(x=0) = 0 & (1) \\ w(x=L_x) = 0 & & (2) \\ \frac{\partial w}{\partial x}(x=\frac{L_x}{2}) = 0 & & (3) \\ \frac{\partial w}{\partial y}(x=0, x=L_x) = 0 & & \end{cases}$$

$$w'''' = q_0, \quad w''' = q_0 x + C_1, \quad w'' = \frac{q_0 x^2}{2} + C_1 x + C_2$$

$$w' = \frac{q_0 x^3}{6} + \frac{C_1 x^2}{2} + C_2 x + C_3$$

$$w = \frac{q_0 x^4}{24} + \frac{C_1 x^3}{6} + \frac{C_2 x^2}{2} + C_3 x + C_4$$

$$\text{From (1); } C_2 = 0, C_4 = 0$$

$$\text{From (2), } \frac{q_0 L_x^4}{24} + \frac{C_1 L_x^3}{6} + \frac{C_2 L_x^2}{2} + C_3 L_x = 0$$

$$\text{From (3), } \frac{q_0 L_x^3}{6(2^3)} + \frac{C_1 L_x^2}{2(2^2)} + C_2 \frac{L_x}{2} + C_3 = 0$$

$$C_1 = -\frac{q_0 L_x}{2}$$

$$0 = \frac{q_0 L_x^4}{24} + \frac{-q_0 L_x^4}{12} + C_3 L_x, \quad C_3 = \frac{q_0 L_x^3}{24}$$

$$\text{So, } w = \left( \frac{q_0 x^4}{24} + \frac{-q_0 L_x x^3}{12} + \frac{q_0 L_x^3 x}{24} \right)$$

D

Center:  $L_x = 2$  Center  $x = 1$

$$W = \frac{\left( \frac{500}{24} + \frac{-500(2)}{12} + \frac{500(2^3)(1)}{24} \right)}{\frac{2(29000)}{3(1-0.4^2)}} = 0.0045$$

Beam Theory.

$$EI W'''' = \frac{a(x)}{EI}, \quad W'''' = \frac{a(x)}{EI}, \quad W'''' = \frac{q_0 x + C_1}{EI}$$

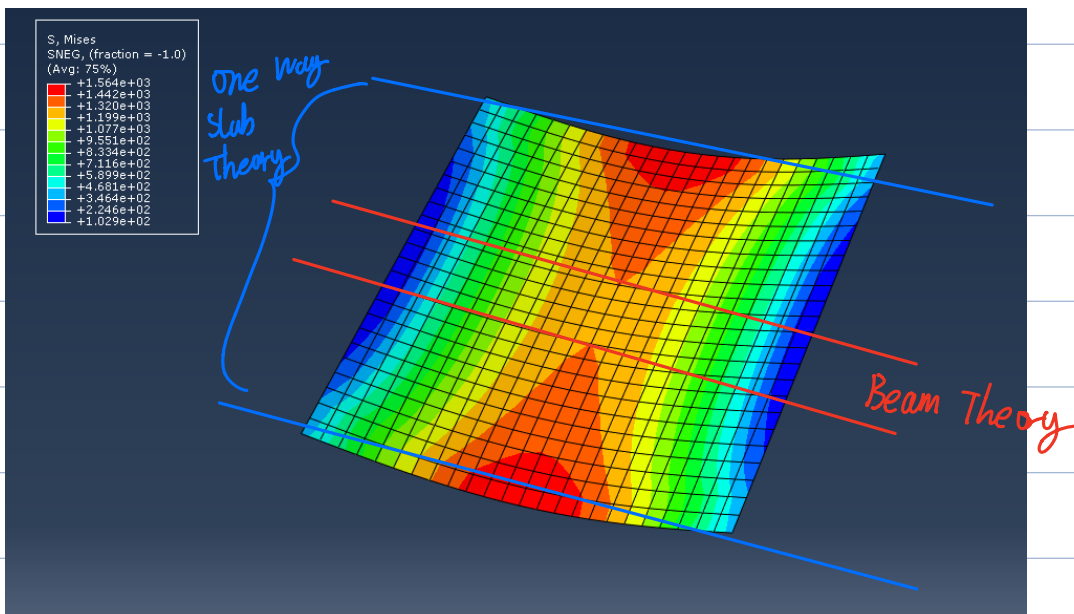
$$W'' = \frac{q_0 x^2}{2} + C_1 x + C_2, \quad W' = \frac{q_0 x^3}{6} + \frac{C_1 x^2}{2} + C_2 x + C_3$$

$$W = \frac{q_0 x^4}{24} + \frac{C_1 x^3}{6} + \frac{C_2 x^2}{2} + C_3 x + C_4, \quad \text{BCs are the same}$$

$C_1, C_2, C_3, C_4$  are the same

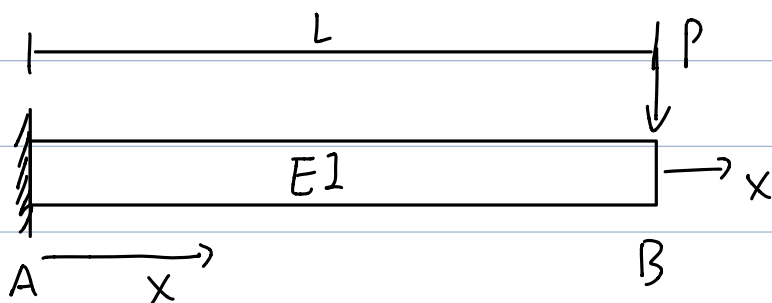
$$W = \frac{\left( \frac{q_0 x^4}{24} - \frac{q_0 L x^3}{12} + \frac{q_0 L x^3}{24} x \right)}{EI}$$

$$W = \frac{\frac{500(1)}{24} - \frac{500(2)}{12} + \frac{500(2)}{24}}{\frac{29000}{24(1)^3}} = \frac{270.833}{58000} = 0.0046$$



P4

(a)



$$M(x) = (L-x)P \quad \dots (1)$$

$$EI v'' = M(x)$$

$$EI v'' = (L-x)P \quad \dots (2)$$

$$\int EI v'' dx = \int (L-x)P dx$$

↓

$$EI v' = \left(Lx - \frac{x^2}{2}\right)P + C_1 \quad \dots (3)$$

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$$EI v = \left(-\frac{x^3}{6} + \frac{Lx^2}{2}\right)P + C_1 x + C_2 \quad \dots (4)$$

$$\left. \begin{array}{l} v'(0)=0 \\ v(0)=0 \end{array} \right\} \text{BCs} \quad \text{From (3) } EI v' = \left(Lx - \frac{x^2}{2}\right)P + C_1 \Rightarrow C_1 = \frac{L^2 P}{2}$$

$$\text{From (4) } EI v = \left(-\frac{x^3}{6} + \frac{Lx^2}{2}\right)P + C_1 x + C_2 \Rightarrow C_2 = -\frac{L^3 P}{6}$$

$$EI v = \frac{1}{6}(L-x)^3 P + \frac{L^2 P}{2}x - \frac{L^3 P}{6} \Rightarrow V = \frac{P}{6EI} (3L-x)x^2$$

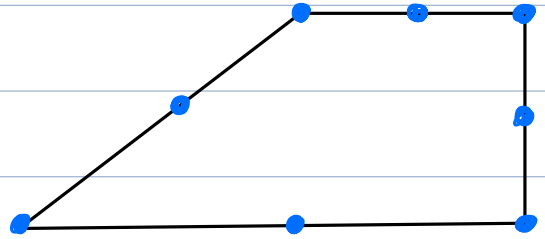
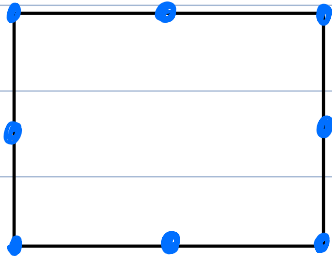
$$EI v' = -\frac{1}{2}(L-x)^2 P + \frac{L^2 P}{2} \Rightarrow V' = \frac{P}{2EI} (2L-x)x$$

$$V = \frac{100}{6(10^7)83.33} (3(100) - 100)100^2 = 0.04$$

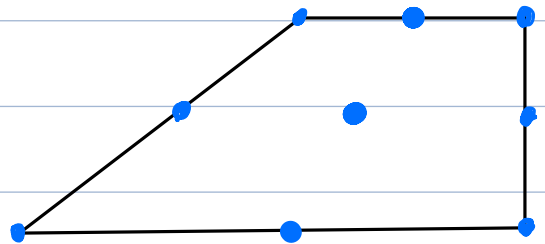
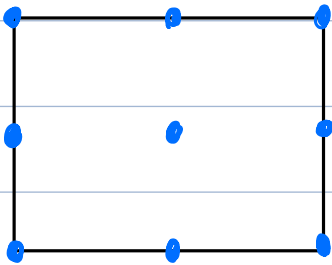
$$I = \frac{1(10)^3}{12} = 83.33$$

(b)

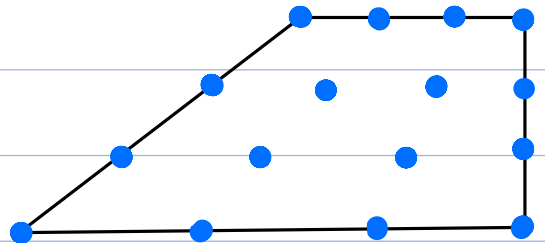
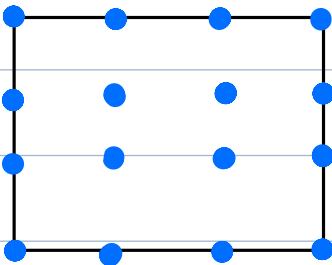
8-nodes



9-nodes



16-nodes



(c) Serendipity elements are rectangular elements with no interior nodes (8-nodes)

Lagrangian elements are those with interior nodes.  
(9-nodes, 16 nodes)