

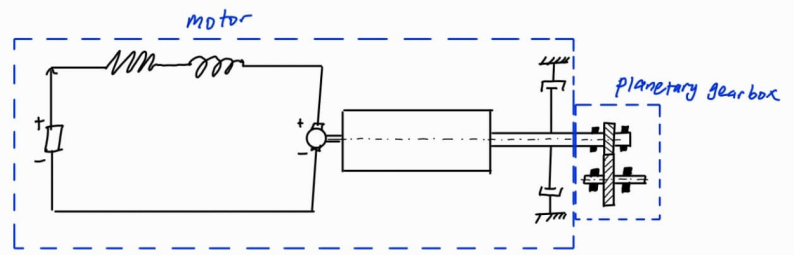
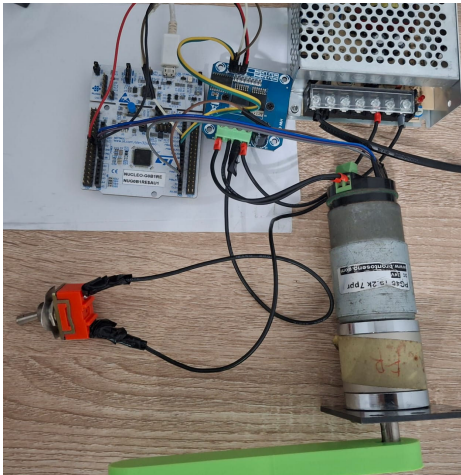
# Position Control of a DC Motor Using Feedback Linearization and LuGre Based Friction Extended State Observer

This project implements feedback linearization and a simple PI controller for precise position control of a brushed DC motor. Additionally, a LuGre-based observer is used to estimate and compensate for friction and external disturbances, enhancing control performance.

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## Research Equipment



In this research, i use a 70 Watt (output power) DC motor PG45 Brontoseno, BTS7960 motor driver, power supply (24 V and 5 A), and an STM32 Nucleo G0B1RE microcontroller

## BDC Motor Modelling

Documentation on modelling and parameter identification can be accessed through the following link below

<https://github.com/IrfNyafi/Modelling-and-Parameter-Identification-of-Brushed-DC-Motor.git>

$$\frac{d\omega(t)}{dt} = \rho(t)(-0.1628 |\omega(t)| + 11.04) - 0.7184 \omega(t) - 167037.21858901123 z - 4569.25 \frac{dz}{dt}$$

$$\frac{dz(t)}{dt} = \omega(t) - \frac{1336.29774871209 |\omega(t)| z}{0.1668 + 0.29525698321793 e^{-\left(\frac{\omega}{0.561797287214991}\right)^2}}$$

- $\rho(t)$  is the average voltage define as percentage duty cycle times the input supply voltage
- $z(t)$  is the internal LuGre friction state
- $\omega(t)$  is the motor speed

## Non-linear Luenberger Observer

let us consider the following model

$$\frac{d\omega(t)}{dt} = \rho(t)(a_1 |\omega(t)| + a_2) - \beta\omega(t) - \frac{\sigma_0}{J}z - \frac{\sigma_1}{J}\frac{dz}{dt} + f(t)$$

$$\frac{dz(t)}{dt} = \omega(t) - \frac{\sigma_0|\omega(t)|z}{T_c + \Delta T e^{-\left(\frac{\omega}{\omega_s}\right)^2}}$$

$f(t)$  is generalized disturbance which is the combination of internal disturbance (arises from modeling errors) and external disturbance (comes from external torque exerted by the environment). By the above model, we design the observer as follow

$$\frac{d\hat{z}(t)}{dt} = \omega(t) - \sigma_0 \frac{|\omega(t)|}{h(\omega)} \hat{z}(t) + K_1(\omega(t) - \hat{\omega}(t))$$

$$\frac{d\hat{\omega}(t)}{dt} = -\frac{\sigma_0}{J}\hat{z} - \frac{\sigma_1}{J}\frac{d\hat{z}(t)}{dt} - \frac{F_v}{J}\omega(t) + \hat{f}(t) + u(t) + \frac{K_2}{J}(\omega(t) - \hat{\omega}(t))$$

$$\frac{d\hat{f}(t)}{dt} = K_3(\omega(t) - \hat{\omega}(t))$$

Documentation of the observer can be found here, <https://github.com/IrfNyafi/Lugre-Friction-Model-Based-Observer-For-Friction-Compensation-of-DC-Motor-Motion-Control.git>

## Control System Law: Modified PID Controller Using Feedback Linearization

let us define

$$u(t) = \rho(t)(a_1 |\omega(t)| + a_2) - \beta\omega(t) - \frac{\sigma_0}{J}z - \frac{\sigma_1}{J}\frac{dz}{dt} + f(t)$$

then our dynamic model can be written as second order dynamic model

$$\frac{d\dot{\theta}(t)}{dt} = u(t)$$

define the error dynamics

$$e(t) = \theta_d - \theta(t)$$

$$\dot{e}(t) = \dot{\theta}_d(t) - \dot{\theta}(t)$$

$$\ddot{e}(t) = \ddot{\theta}_d - \ddot{\theta}(t) = \ddot{\theta}_d - u(t)$$

we design the control input law  $u(t)$  using simple proportional integral controller. The writer choose the control input to be

$$u(t) = \ddot{\theta}_d + K_p e(t) + K_i \int_{t_0}^t e(\tau) d\tau + K_d \frac{de(t)}{dt}$$

substitute the proposed control law to the error dynamics, we will get

$$\ddot{e}(t) = \dot{\omega}_d(t) - \dot{\omega}(t) = \dot{\omega}_d(t) - \dot{\omega}_d(t) - K_p e(t) - K_i \int_{t_0}^{\tau} e(\tau) d\tau - K_d \frac{de(t)}{dt}$$

$$\ddot{e}(t) + K_p e(t) + K_i \int_{t_0}^{\tau} e(\tau) d\tau + K_d \frac{de(t)}{dt} = 0$$

similar dynamics

$$\frac{d}{dt} \ddot{e}(t) + K_d \ddot{e}(t) + K_p \dot{e}(t) + K_i e(t) = 0$$

based on that, the gain  $K_p$  and  $K_i$  is chosen to be Hurwitz to ensure stability. We can also use pole placement technique to make tuning easier. Let us define  $P_1$ ,  $P_2$ , and  $P_3$  as the desire error dynamic poles. Construct the desire characteristic equation

$$(s - P_1)(s - P_2)(s - P_3) = 0$$

$$s^3 - (P_1 + P_2 + P_3)s^2 + (P_1P_2 + P_1P_3 + P_2P_3)s - P_1P_2P_3 = 0$$

thus, we get  $K_p = (P_1P_2 + P_1P_3 + P_2P_3)$ ,  $K_i = -P_1P_2P_3$ , and  $K_d = -(P_1 + P_2 + P_3)$ .

We conclude the control law as the following:

$$u(t) = \dot{\omega}_d(t) + K_p e(t) + K_i \int_{t_0}^{\tau} e(\tau) d\tau + K_d \frac{de(t)}{dt} = \rho(t)(a_1 |\omega(t)| + a_2) - \beta \omega(t) - \frac{\sigma_0}{J} \hat{z} - \frac{\sigma_1}{J} \frac{d\hat{z}}{dt} + \hat{f}(t)$$

$$\rho(t) = \frac{\dot{\omega}_d(t) + K_p e(t) + K_i \int_{t_0}^{\tau} e(\tau) d\tau + K_d \frac{de(t)}{dt} + \beta \omega(t) + \frac{\sigma_0}{J} \hat{z} + \frac{\sigma_1}{J} \frac{d\hat{z}}{dt} - \hat{f}(t)}{(a_1 |\omega(t)| + a_2)}$$

the values of  $\hat{z}$ ,  $\frac{d\hat{z}}{dt}$ , and  $\hat{f}(t)$  are obtained from the nonlinear Luenberger observer

## Impedance Control

for this case, we can design

$$u(t) = \ddot{\theta}_d + \frac{1}{M_d} \left( K_{sd} e(t) + C_d \frac{de(t)}{dt} \right)$$

where

- $M_d$  is the desire apparent inersia
- $C_d$  is the desire apparent damping
- $K_{sd}$  is the desire apparent stiffness
- we define,  $e(t) = \theta_d(t) - \theta(t)$

remember, our system is

$$\frac{d\dot{\theta}(t)}{dt} = u(t)$$

substitute the propose control input  $u(t)$  to the system dynamic

$$\frac{d\dot{\theta}(t)}{dt} = u(t) = \ddot{\theta}_d + \frac{1}{M_d} \left( K_{sd} e(t) + C_d \frac{de(t)}{dt} \right)$$

$$-(\ddot{\theta}_d - \ddot{\theta}) = \frac{1}{M_d} \left( K_{sd} e(t) + C_d \frac{de(t)}{dt} \right)$$

$$M_d(\ddot{\theta}_d - \ddot{\theta}) + C_d(\dot{\theta}_d - \dot{\theta}) + K_{sd}(\theta_d - \theta) = 0$$

We conclude the control law as the following:

$$u(t) = \dot{\omega}_d(t) + K_p e(t) + K_d \frac{de(t)}{dt} = \rho(t)(a_1 |\omega(t)| + a_2) - \beta \omega(t) - \frac{\sigma_0}{J} \hat{z} - \frac{\sigma_1}{J} \frac{d\hat{z}}{dt} + \hat{f}(t)$$

$$\rho(t) = \frac{\dot{\omega}_d(t) + K_p e(t) + K_d \frac{de(t)}{dt} + \beta \omega(t) + \frac{\sigma_0}{J} \hat{z} + \frac{\sigma_1}{J} \frac{d\hat{z}}{dt} - \hat{f}(t)}{(a_1 |\omega(t)| + a_2)}$$

the values of  $\hat{z}$ ,  $\frac{d\hat{z}}{dt}$ , and  $\hat{f}(t)$  are obtained from the nonlinear Luenberger observer