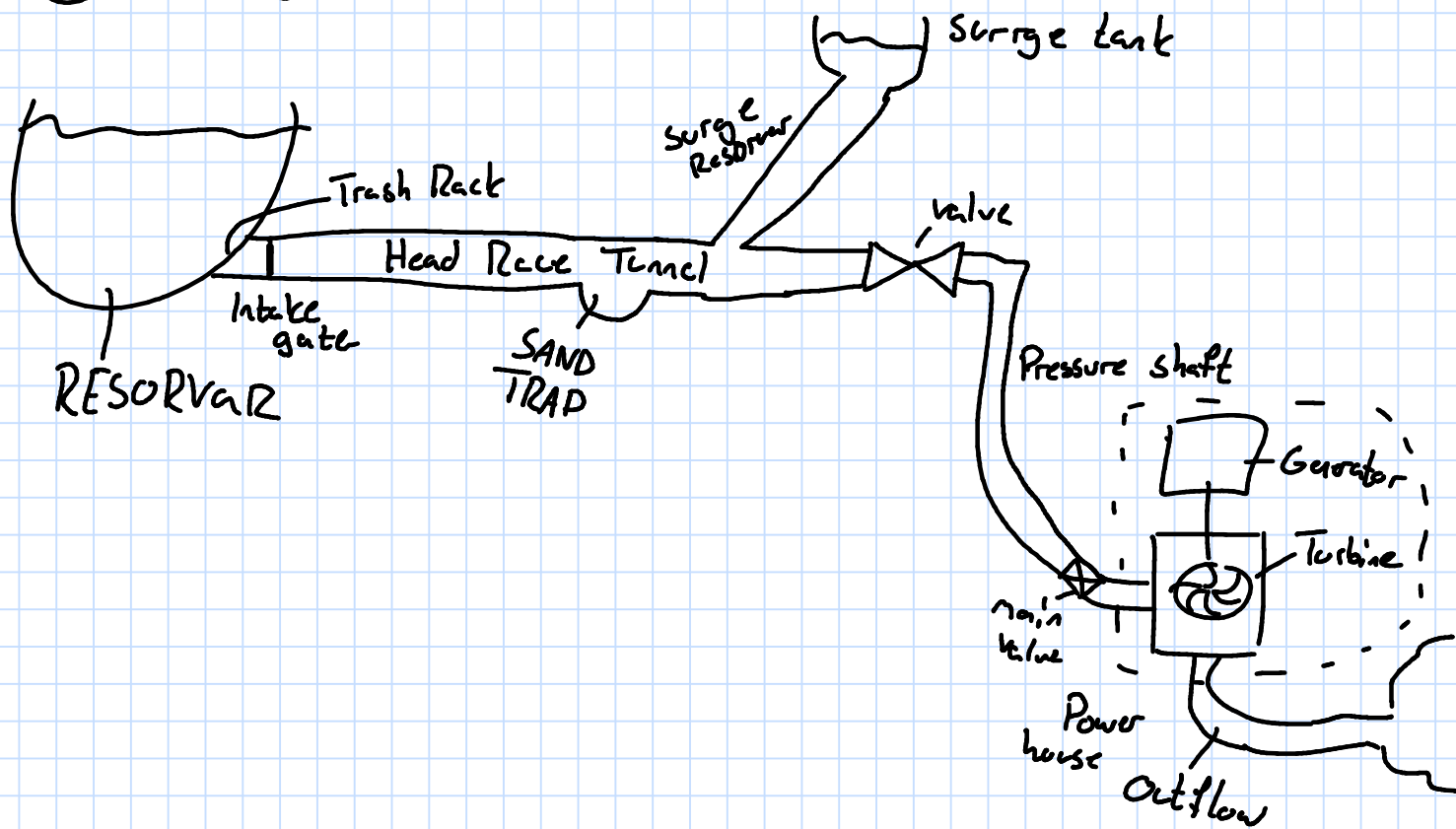
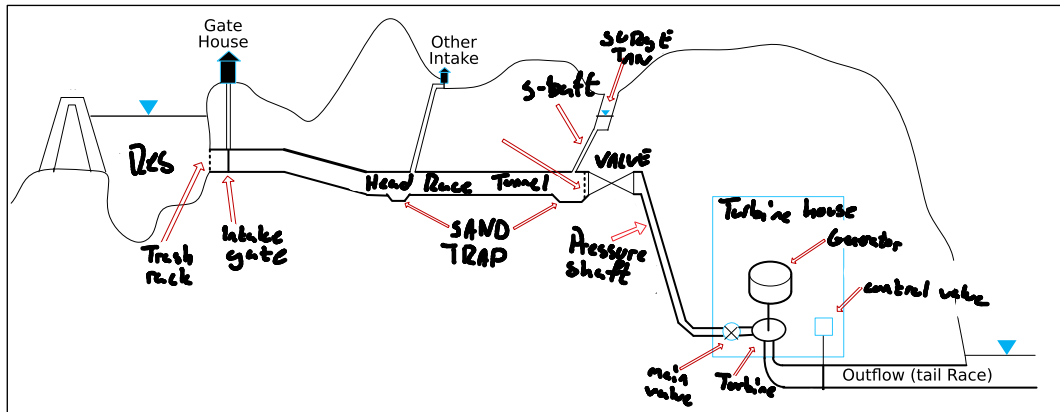


① Tegn HPS vann veien



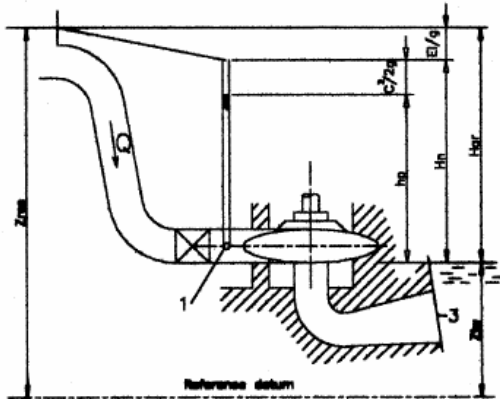
Overview

Typical high-head hydro power system



General theory

Gross quantities



Definition (Head and Energy)

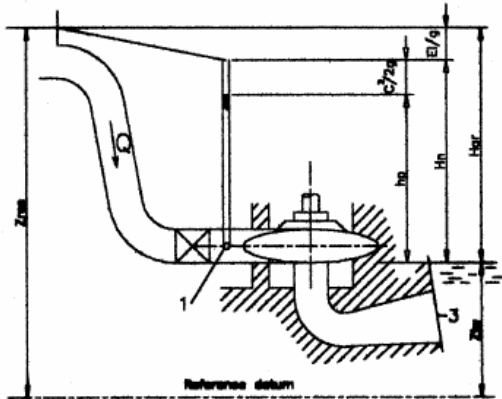
$$H_{gr} = z_{res} - z_{tw}$$

Definition (Power)

$$P_{gr} = \rho g H_{gr} Q \quad (1)$$

General theory

Net quantities



Definition (Head and Energy)

$$\begin{aligned} H_n &= H_{gr} - \frac{E}{g} = H_{gr} - H_z \\ &= h_p + \frac{c^2}{2g} \\ E_n &= H_n g \end{aligned}$$

Definition (Power)

$$P_n = \rho g H_n Q$$

(2)

Lös opgave:

$$Q = 1 \text{ m}^3/\text{s} \quad H = 100 \text{ m}$$

Hva er Power?

$$P = \rho g H Q = 1000 \cdot 9,81 \cdot 1 \cdot 100 = 981 \text{ kW} \approx 1 \text{ MW}$$

Turbine types

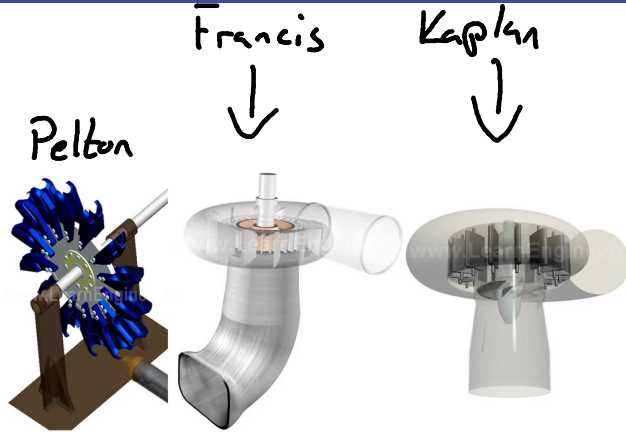


Figure: <https://youtu.be/koBLOKEZ3KU>

Hydraulic efficiency

Part 1

Definition (Total available power)

$$P_n = \rho g H_n Q \quad (3)$$

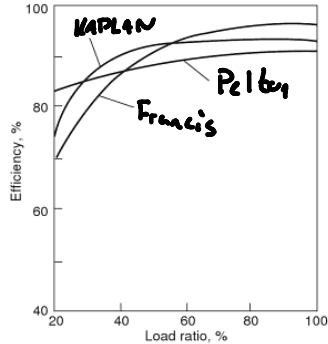
Definition (Power of the runner)

$$P_r = \rho Q (U_1 C_{u1} - U_2 C_{u2}) \quad (4)$$

Definition (Hydraulic efficiency)

$$\eta = \frac{P_r}{P_n} = \frac{\rho Q (U_1 C_{u1} - U_2 C_{u2})}{\rho g H_n Q} = \frac{U_1 C_{u1} - U_2 C_{u2}}{g H_n} \quad (5)$$

Turbine efficiencies



Variation of hydraulic efficiency for various types of turbine over a range of loading, at constant speed and constant head.

Turbine Classification

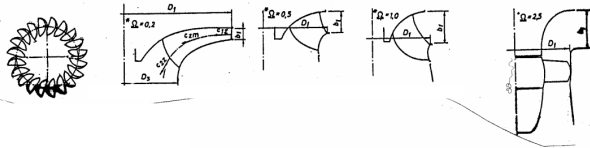
Speed Number and Specific Speed

Speed Number

$$\frac{n_q}{89} = \Omega = \frac{\omega}{\sqrt{Q}}$$

Pelton --- Francis --- KAPLAN

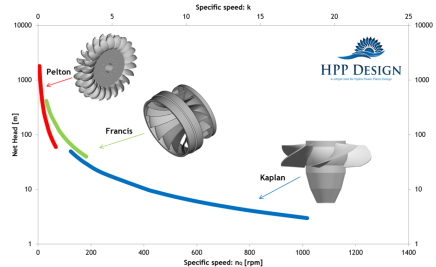
0.1 0.2 0.5 1.0 2.0
Speed number Ω



Specific Speed

$$n_q = \frac{\omega \sqrt{Q}}{\sqrt{H_n^3}}$$

$$k = \frac{\omega \sqrt{Q}}{\sqrt{g H_n^3}}$$



Lecture Outline

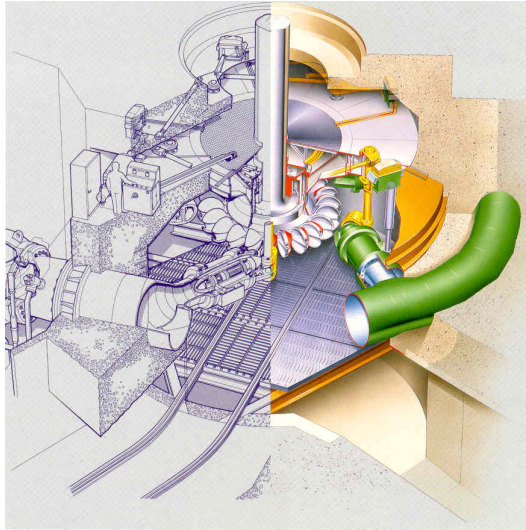
Slides

- Design of a Pelton turbine
 - Calculation of the different parameters
 - Two example calculation

Black board

- Introduction of a “Control Volume”
- In- and extensive properties
- General formulation of change of properties
- Lumped parameter model

Design of Pelton turbines



Design of Pelton turbines

Speed number range for an **ideal** Pelton turbine

Minium speed number

For the diameter $D = 10 \cdot d_s$ and one nozzle $z = 1$:

$$\underline{\Omega} = \frac{d_s}{D} \sqrt{\frac{\pi \cdot z}{4}} = \frac{1}{10} \sqrt{\frac{\pi \cdot 1}{4}} = 0.09$$

The minimum speed number for a Pelton turbine with one nozzle is:

$$\underline{\Omega}_{\min} = 0.09$$

Maximum speed number

For the diameter $D = 10 \cdot d_s$ and six nozzles $z = 6$:

$$\underline{\Omega} = \frac{d_s}{D} \sqrt{\frac{\pi \cdot z}{4}} = \frac{1}{10} \sqrt{\frac{\pi \cdot 6}{4}} = 0.22$$

The maximum speed number for a Pelton turbine with one nozzle is:

$$\underline{\Omega}_{\max} = 0.22$$

Active power and frequency control

Frequency response

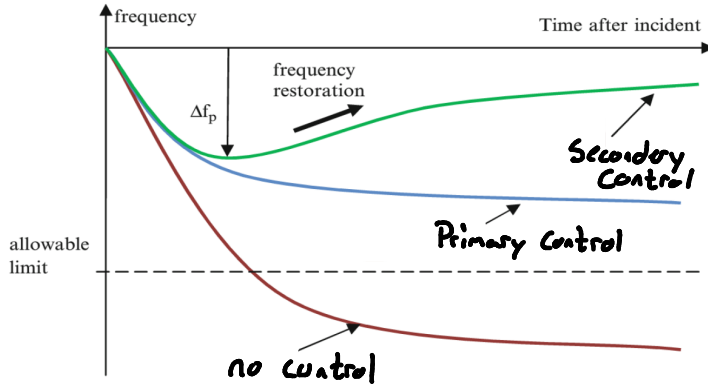


Figure: Planned frequency profile in response to large and sudden loss of generation

Active power and frequency

Primary Control

Droop: slope of frequency-power characteristic

$$R_{gi} = \frac{\Delta f / f_r}{\Delta P_{gi} / P_{gi,r}}$$

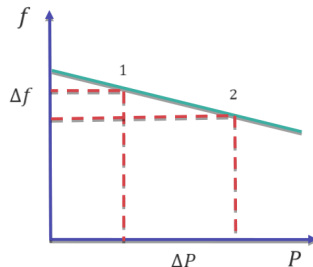
R_{gi} regulation of generator

Δf change in f

f_r nominal f

ΔP_{gi} change in Power

$P_{gi,r}$ nominal Power

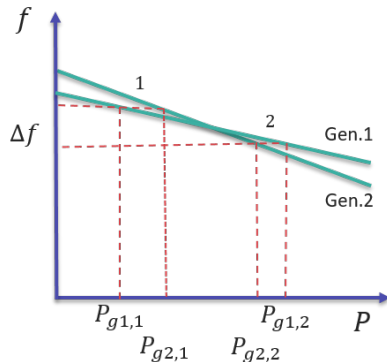


Active power and frequency: Primary Control

Two generators in parallel

Example

- $P_{g1r} = P_{g2r} = 200 \text{ MW}$, $R_{g1} = 0.02$, $R_{g2} = 0.06$
- Balance (1): $f_r = 50 \text{ Hz}$
- Sudden increase in of load $\Delta P \rightarrow$ frequency drop Δf
- Power distributed over two generators:
$$\frac{\Delta P_{g1}}{\Delta P_{g2}} = \frac{R_{g2}}{R_{g1}} = 3$$
- Power increase in ratio 3:1 (75% G_1 and 25% G_2)



Active power and frequency: Primary Control

Many generators in parallel

Network Power-Frequency Characteristic

$$\lambda = \frac{\Delta P}{\Delta f} \quad (\text{bias factor})$$

$$\Delta P = \sum \Delta P_{gi} = \left[-\frac{1}{R_g} \frac{P_{gi}}{f_r} \Delta f \right]$$

$$\Rightarrow \lambda =$$

Active power and frequency: Primary Control

Many generators in parallel

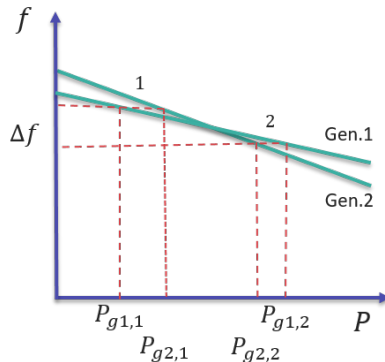
Example

- $P_{g1r} = P_{g2r} = 200 \text{ MW}$, $R_{g1} = 0.02$, $R_{g2} = 0.06$

$$\Rightarrow \lambda = \frac{1}{50} \cdot \left(\frac{200}{0.02} + \frac{200}{0.06} \right) = 266,67$$

$$\text{Since } \lambda : \Delta P = \frac{266}{2} = 133$$
$$\Delta f = \frac{133}{266} = 0.5$$

$$\Delta P = 133,3 \text{ MW} \Rightarrow \Delta f = 0.5 \text{ Hz}$$



Active power and frequency: Primary Control

Self-regulation

Definition of Self-regulation

Motors cause the system load to be slightly frequency dependent.

$$\lambda = -\frac{\Delta P}{\Delta f} = \sum_i \left(\frac{1}{R_{gi}} \cdot \frac{P_{gi,r}}{f_r} \right) + \frac{\mu}{100} \cdot P$$
$$\mu = \frac{\Delta P' / P_r}{\Delta f} \cdot 100\%$$

μ the self regulation effect of the load P [%/Hz]

$\Delta P'$ the change in the power consumption of the loads because of the change in frequency Δf [MW]

P the value of the original load of the system plus the change ΔP at the original system frequency [Hz]

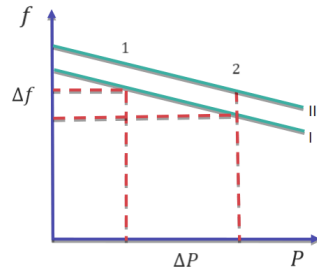
Active power and frequency: Secondary Control

Load Frequency Control (LFC)

Principle

1. Change setpoint to P_2

2.



Active power and frequency: Tertiary Control

Area Control Error (ACE)

Definition

$$ACE_i = (P_a - P_s) + \lambda (f_a - f_s)$$

ACE_i = Area control error

P_{a_i} = active Power

P_{s_i} = setpoint Power

λ_i = characteristic for controller

f_a = f active

f_s = setpoint

ACE should be zero for scheduled power exchange at nominal frequency.

Active power and frequency: Tertiary Control

Area control

Example

Example for three areas A, B, C:

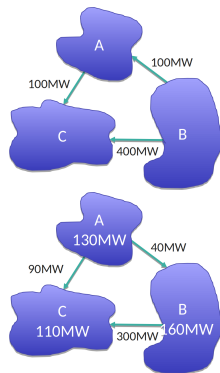
- Scheduled $B \rightarrow C = 400 \text{ MW}$, $B \rightarrow A = A \rightarrow C = 100 \text{ MW}$
- $\lambda_A = 13 \text{ GW/Hz}$, $\lambda_B = 16 \text{ GW/Hz}$, $\lambda_C = 11 \text{ GW/Hz}$
- System $\lambda = 40 \text{ GW/Hz}$
- Sudden drop in frequency $\Delta f = -0.01 \text{ Hz}$

$$A: (130 - 0) + 13 \cdot 10^3 (-0.01) = 0$$

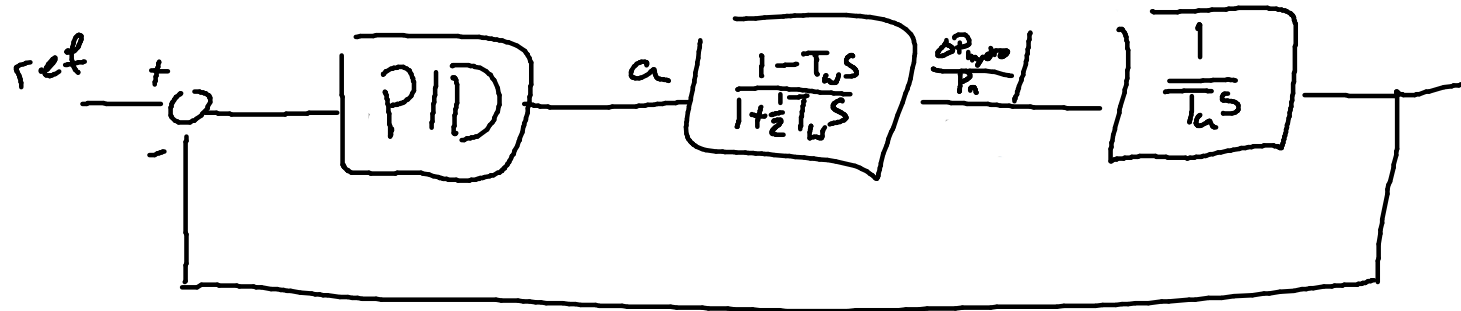
$$B: (260 - 500) + 16 \cdot 10^3 (-0.01) = -400 \text{ MW}$$

$$C: (-390 + 500) + 11 \cdot 10^3 (-0.01) = 0$$

- Generation in area B should be increased by 400 MW



The transfer function of the hydro power system can be represented by:



For the determining the time constants of the transfer-function based model we need to determine the following two time constants:

$$T_w = \frac{Q}{gH}$$

$$T_a = \frac{J \omega^2}{P_e}$$

We do not worry about the controller parameters since we are going to use a standard PID model from the standard library.