Time-stepping and Krylov methods for large-scale instability problems

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Abstract ???

1 Introduction

2 Theoretical framework

Our attention is focused on the characterization of very high-dimensional nonlinear dynamical systems typically arising from the spatial discretization of partial differential equations such as the incompressible Navier-Stokes equations. In general, the resulting dynamical equations are written down as a system of first order differential equations

$$\dot{X}_j = \mathcal{F}_j(\left\{X_i(t); i=1,\cdots,n\right\},t)$$

where the integer n is the *dimension* of the system, and \dot{X}_j denotes the timederivative of X_j . Using the notation \mathbf{X} and $\boldsymbol{\mathcal{F}}$ for the sets $\{X_j, i=1,\cdots,n\}$ and $\{\mathcal{F}_j, i=1,\cdots,n\}$, this system can be compactly written as

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$$\dot{\mathbf{X}} = \mathcal{F}(\mathbf{X}, t),\tag{1}$$

where **X** is the $n \times 1$ state vector of the system and t is a continuous variable denoting time. Alternatively, accounting also for temporal discretization gives rise to a discrete-time dynamical system

$$X_{j,k+1} = \Phi_j(\{X_{i,k}; i = 1, \cdots, n\}, k)$$
 (2)

or formally

$$\mathbf{X}_{k+1} = \mathbf{\Phi}(\mathbf{X}_k, k) \tag{3}$$

where the index k now denotes the discrete time variable. If one uses first-order Euler extrapolation for the time discretization, the relation between \mathcal{F} and $\boldsymbol{\Phi}$ is given by

$$\boldsymbol{\Phi}(\mathbf{X}) = \mathbf{X} + \Delta t \boldsymbol{\mathcal{F}}(\mathbf{X}),$$

where Δt is the time-step and the explicit dependences on t and k have been dropped for the sake of simplicity.

In the rest of this section, the reader will be introduced to the concepts of fixed points and linear stability, two concepts required to characterize a number of properties of the system investigated. Particular attention will be paid to *modal* and *non-modal stability*, two approaches that have become increasingly popular in fluid dynamics over the past decades. Note that the concept of *nonlinear optimal perturbation*, which as raised a lot attention lately, is beyond the scope of the present contribution. For interested readers, please refer to the recent work by [1] and references therein.

Finally, while we will mostly use the continuous-time representation (1) when introducing the reader to the theoretical concepts exposed in this section, using the discrete-time representation (3) will prove more useful when discussing and implementing the different algorithms presented in §3.

2.1 Fixed points

Nonlinear dynamical systems described by Eq. (1) or Eq. (3) tend to admit a number of different equilibria forming the backbone of their phase space. These different equilibria can take the form of fixed points, periodic orbits or strange attractors for instance. In the rest of this work, our attention will be solely focused on fixed points.

For a continuous-time dynamical system described by Eq. (1), fixed points \mathbf{X}^* are solution to

$$\mathcal{F}(\mathbf{X}) = 0. \tag{4}$$

Conversely, fixed points X^* of a discrete-time dynamical system described by Eq. (3) are solution to

$$\boldsymbol{\Phi}(\mathbf{X}) = \mathbf{X}.\tag{5}$$

It must be emphasized that both Eq. (4) and Eq. (5) may admit multiple solutions. Such a multiplicity of fixed points can easily be illustrated by a simple Duffing oscillator given by

$$\dot{x} = y$$

$$\dot{y} = -\frac{1}{2}y + x - x^3.$$
(6)

Despite its apparent simplicity, this Duffing oscillator admits three fixed points, namely

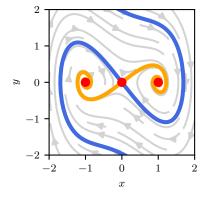
- a saddle at the origin $X^* = (0,0)$,
- two linearly stable spirals located at $X^* = (\pm 1, 0)$.

All of these fixed points, along with some trajectories, are depicted on figure 1 for the sake of illustration. Such a multiplicity of fixed points also occurs in dynamical systems as complex as the Navier-Stokes equations. Determining which of these fixed points is the most relevant one from a physical point of view is problem-dependent and left for the user to decide. Note however that computing these equilibrium points is a prerequisite to all of the analyses to be described in this chapter. Numerical methods to solve Eq. (4) or Eq. (5) will be discussed in §3.1.

2.2 Linear stability analysis

Having computed a given fixed point \mathbf{X}^* of a continuous-time nonlinear dynamical system given by Eq. (1), one may ask whether it corresponds to a stable or unstable equilibrium of the system. Before pursuing, the very notion of *stability* needs to be explained. It is traditionally defined following the concept of Lyapunov stability. Having computed the equilibrium state \mathbf{X}^* , the system is perturbed around this state. If it returns back to the equilibrium point, the latter is deemed stable, otherwise, it

Fig. 1 Phase portrait of the unforced Duffing oscillator (6). The red dots denote the three fixed points admitted by the system. The blue (resp. orange) thick line depicts the stable (resp. unstable) manifold of the saddle point located at the origin. Grey lines highlight a few trajectories exhibited for different initial conditions.



is regarded as unstable. It has to be noted that, in the concept of Lyapunov stability, an infinite time horizon is allowed for the return to equilibrium.

The dynamics of a perturbation $\mathbf{x} = \mathbf{X} - \mathbf{X}^*$ are governed by

$$\dot{\mathbf{x}} = \mathcal{F}(\mathbf{X}^* + \mathbf{x}). \tag{7}$$

Assuming the perturbation \mathbf{x} is infinitesimal, $\mathcal{F}(\mathbf{X})$ can be approximated by its first-order Taylor expansion around $\mathbf{X} = \mathbf{X}^*$. Doing so, the governing equations for the perturbation \mathbf{x} simplify to

$$\dot{\mathbf{x}} = \mathcal{A}\mathbf{x},\tag{8}$$

where A is the $n \times n$ Jacobian matrix of F. Starting from an initial condition \mathbf{x}_0 , the perturbation at time t is given by

$$\mathbf{x}(t) = \exp\left(\mathbf{A}t\right)\mathbf{x}_0. \tag{9}$$

The operator $\mathcal{M}(t) = \exp{(\mathcal{A}t)}$ is known as the *exponential propagator*. Introducing the spectral decomposition of \mathcal{A}

$$\mathcal{A} = \mathcal{V} \Lambda \mathcal{V}^{-1}$$

Eq. (9) can be rewritten as

$$\mathbf{x}(t) = \mathbf{\mathcal{V}} \exp\left(\mathbf{\Lambda}t\right) \mathbf{\mathcal{V}}^{-1} \mathbf{x}_0, \tag{10}$$

where the ith column of \mathcal{V} is the eigenvector \mathbf{v}_i associated to the ith eigenvalue $\lambda_i = \mathbf{\Lambda}_{ii}$. Assuming that the eigenvalues of \mathcal{A} have been sorted by decreasing real part, it can easily be shown that

$$\lim_{t\to+\infty}\exp\left(\mathcal{A}t\right)\mathbf{x}_{0}=\lim_{t\to+\infty}\exp\left(\lambda_{1}t\right)\mathbf{v}_{1}.$$

The asymptotic fate of an initial perturbation \mathbf{x}_0 is thus entirely dictated by the real part of the leading eigenvalue λ_1 :

- if $\Re(\lambda_1) > 0$, a random initial perturbation \mathbf{x}_0 will eventually grow exponentially fast. Hence, the fixed point \mathbf{X}^* is deemed *linearly unstable*.
- If $\Re(\lambda_1) < 0$, the initial perturbation \mathbf{x}_0 will eventually decay exponentially rapidly. The fixed point \mathbf{X}^* is thus *linearly stable*.

The case $\Re(\lambda_1) = 0$ is peculiar. The fixed point \mathbf{X}^* is called *elliptic* and one cannot conclude about its stability solely by looking at the eigenvalues of \mathcal{A} . In this case, one needs to resort to *weakly non-linear analysis* which essentially looks at the properties of higher-order Taylor expansion of $\mathcal{F}(\mathbf{X})$. Once again, this is beyond the scope of the present chapter. Interested readers are referred to [?] for more details about such analyses.

Illustration

Let us illustrate the notion of linear stability on a simple example. For that purpose, we will consider the same linear dynamical system as in [2].

2.3 Non-modal stability analysis

2.3.1 Optimal perturbation

Formulation using Rayleigh quotient

Formulation using Lagrange multipliers

2.3.2 Resolvent analysis

Formulation using Rayleigh quotient

Formulation using Lagrange multipliers

2.4 Krylov methods for for solving linear systems

- 3 Numerical methods
- 3.1 Fixed points computation
- 3.1.1 Selective Frequency Damping
- 3.1.2 Newton-Krylov method
- 3.1.3 BoostConv
- 3.2 Modal stability analysis
- 3.2.1 Power Iteration method
- 3.2.2 Arnoldi decomposition
- 3.2.3 Krylov-Schur decomposition
- 3.3 Non-modal stability analysis
- 3.3.1 Optimal perturbation analysis
- 3.3.2 Resolvent analysis
- 4 Application to fluid dynamics
- 5 Conclusions and perspectives

References

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