

# Control Systems EE2S21 – Lecture 1

Bart De Schutter

DCSC

Today:

**Organizational details**

**History of feedback control**

**Mathematical basics**

# Instructional Staff

## Lecturer:

- Bart De Schutter  
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## Teaching assistant:

- Tomas Pippia  
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# Preliminaries I

Announcements/important info/downloads via Brightspace (please enroll!):

- Slides of lectures
- Assignments (practicum and homework)
- Matlab files
- Old exams, etc.

# Preliminaries II

## Course Material:

### Feedback Control of Dynamic Systems

G.F. Franklin,  
J.D. Powell, and  
A. Emami-Naeini

*Seventh Edition, Prentice Hall, 2014.*

[Available from student organization  
(ETV)]



# Teaching objectives

- Getting familiar with basic systems and control concepts
- Getting insight in wide range of multi-disciplinary applications of control
- Being able to compute dynamic responses and analyze stability
- Being able to design stabilizing controllers
- Being able to design PID controllers
- Acquiring basic knowledge required for advanced control courses and/or interaction with control system designers

# Schedule

- Lectures
  - Mondays 15.30–17.30, room Ampere
  - Thursdays 10.30–12.30, room Aula B
  - **No lectures** on March 19 and 22
- Instructions/extra lectures:
  - Feb. 27, 15.30-17.30, room Ampere
  - Mar. 6, 15.30-17.30, room Ampere
  - Mar. 27, 15.30-17.30, room Ampere
  - Apr. 3, 15.30-17.30, room Ampere
- Simulink practicum (only for “real” EE2S21 students)
  - Mar. 8, Apr. 5: group A
  - Mar. 9, Apr. 6: group B
- Exam: Tuesday, Apr. 18, 13.30-16.30

## Simulink practicum

- For “real” EE2S21 students, not for WB2230 students / “schakelstudenten WB” (see Brightspace page of WB2230 for details & options)
- Practicum is done in teams of two
- Full-out assignment report to be handed in at end of practicum or to be sent in after weekend
- Pass or fail
- Participating in and passing *both* Simulink practicums is **obligatory** before you can start EPO-4
- Even if you passed the Simulink practicums in a previous year, you will have to redo them.



# Homework assignments

- 4 homeworks in total
  - Individual assignments
  - *Voluntary* but highly recommended
  - Up to 1 bonus point can be earned (0.25 per homework)
  - Bonus point is valid for 1 year, for exam and for resit
- 
- Assignment will be published on Brightspace
  - 1 week to complete
  - Fill out by hand, hand in hardcopy → hard constraint!
  - Hand in *before* start of lecture → hard constraint!

# Overview of the course

- Feb. 12: Introduction
- Feb. 15, 19, 22: Dynamical models
- Feb. 26: Systems response and transfer functions
- Feb. 27: Linearization, stability, controllability
- Mar. 1: Observability, realization
- Mar. 5: Pole placement
- Mar. 6: Feedback control
- Mar. 12: PID
- Mar. 15: Pole placement and root locus
- Mar. 26: Frequency domain specs – Bode plots
- Mar. 27: Nyquist stability criterion

## Overview of the course (continued)

- Mar. 29: Lead compensator via loop shaping
- Apr. 3: Lag compensator & PID
- Apr. 9: Digital control implementation
- Apr. 9, 12: Exercises and questions

# Contents of the Lecture

- Preliminaries
- **History of Control**
  - To introduce a number of generic concepts of feedback control
  - To lift a tip of the veil of the scope of applications
  - To indicate that we are standing on the shoulders of giants
- Mathematical background knowledge
- Summary

# Can you ride/control this one?

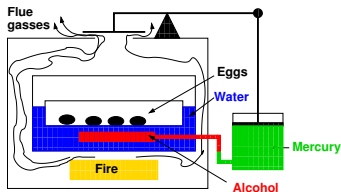


Rear-wheel steering of bike

Problem is in the “Dynamics”

How to design and evaluate a Dynamic control problem?

# Historical Landmarks in Feedback Control

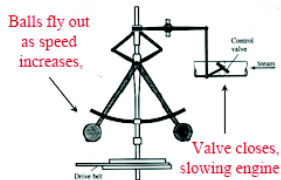


## Pre-1700

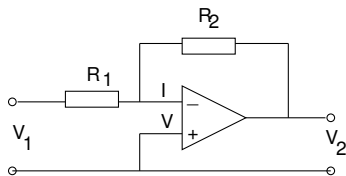
- Water clock ( $\sim 300$  BC, Alexandria), float valves
- Egg incubator (Drebbel, 1624) - temperature control

## Watt Governor (1788)

- Regulate speed of steam engines
- Reduce effects of variations in load (disturbance rejection)
- Stability analysis of unstable engine due to lubrication (J.C. Maxwell, 1868).



# Historical Landmarks in Feedback Control



## Emergence of Control (1920-1945)

- Black's use of negative feedback to reduce uncertainty (robustness)
- Mathematical foundation (fundamental similarities between different systems were noticed)

## A second Wave (1960s)

- Application driven developments (Sputnik (1957), etc.)
- Use of Digital computers
- Theoretical break-throughs (Kalman (1960), etc.)



# Modern Engineering Applications (DCSC)

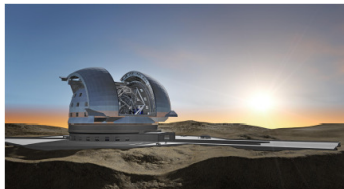
## Micro-systems

- Optical Communication
- Swarms of Nano-satellites
- Smart Structures



## Smart Optics

- Lithography
- Micro-, Nanoscopy
- Astronomy



## Robotics

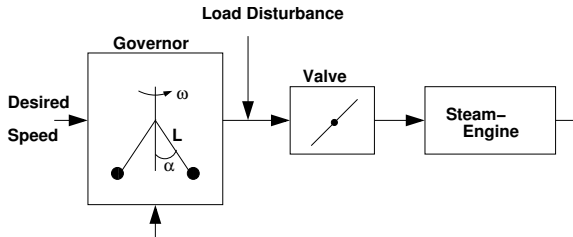
- Home Robotics
- Humanoids
- Rescue Robots





# Magic of Feedback (1)

## Schematic Representation of Watt's Governor

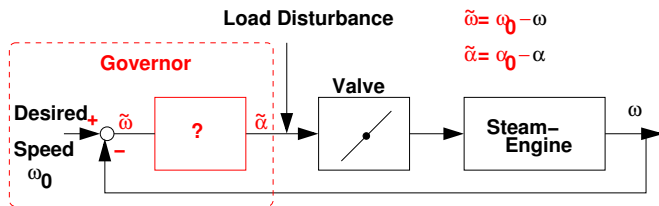


## Key Components

- **Sensor:** measuring the speed of the engine ( $\omega$ )
- **Actuator:** valve determining the steam input to the engine
- **Calculator:** relationship between sensor and actuator ( $\omega$  versus  $\alpha$ )

# Feedback Law of Watt's Governor

## Block Scheme Representation:

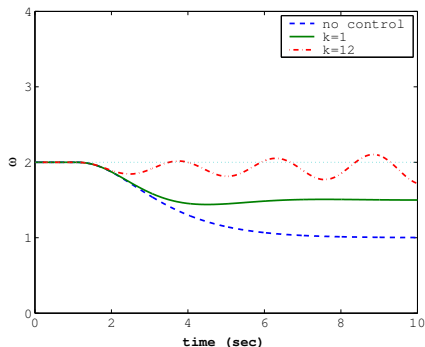


Static Analysis:  $\cos(\alpha) = \frac{g}{L\omega^2} \Rightarrow -\sin \alpha_0 \partial \alpha = -\frac{2g}{\omega_0^3 L} \partial \omega$


## Watt's Governor is Proportional (Negative) Feedback

- **Negative** Feedback: when steam engine runs harder, the valves reduces the steam input (inversion mechanism required)
- **Proportional** Feedback:  $? \approx \tilde{\alpha} = \mathbf{k}\tilde{\omega}$
- **Tuning controller:** The size of the controller gain  $\mathbf{k}$  by changing the rod length  $L$ .

# Simulation results with Watt's Governor



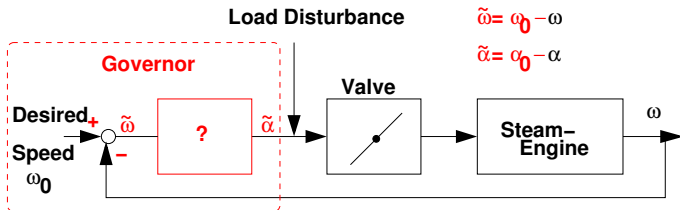
## Summary Observations:

- (Proportional) **Feedback** influences system dynamics
- **Increasing the proportional gain** can:
  - reduce the steady state error
  -  **destabilize** the system.

Feedback Design is Making Trade-offs

# Siemens modification of Watt's governor

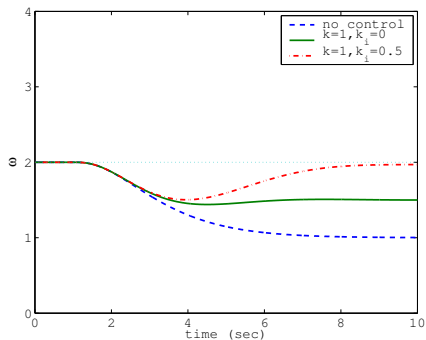
## Block Scheme Representation of Watt's Governor:



Siemens modification of Watt's Governor is Proportional *plus Integral* Feedback.

- $$\tilde{\alpha} = k\tilde{\omega} + k_i \int_0^t \tilde{\omega}(\tau) d\tau$$
- What does it do?

# Simulation results with Watt's Governor



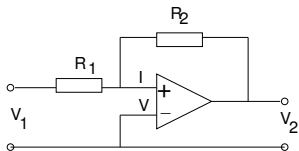
## Summary Observations:

- Integral action can remove steady-state errors.

Feedback Design is more than tuning a gain

## Magic of Feedback (2)

### The Armstrong Amplifier (1915):



Model ( $i \approx 0$ ):  $\frac{V_1 - V}{R_1} = \frac{V - V_2}{R_2}$

Let **open-loop gain** amplifier be

$$V_2 = G V$$

we obtain the input-output model:

$$\frac{V_2}{V_1} = G \frac{R_2}{R_1 + R_2 - G R_1}$$

Sensitivity of gain  $\frac{V_2}{V_1}$  with respect to changes in the passive components

**Example:** Consider

$$G = 5, R_1 = 24, 24.5, 25 \text{ k}\Omega, R_2 = 100 \text{ k}\Omega$$

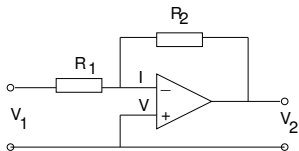
This changes the amplifier gain  $\frac{V_2}{V_1}$  from  $25G$ ,  $50G$  to  $\infty$ .

Explanation:

$$\frac{\partial \frac{V_2}{V_1}}{\partial R_1} = -G \frac{R_2(1 - G)}{(R_1 + R_2 - G R_1)^2}$$

# The Modern Break-through of Feedback

## The Black Amplifier (1927):



Model ( $i \approx 0$ ):  $\frac{V_1 - V}{R_1} = \frac{V - V_2}{R_2}$

Let **open-loop gain** amplifier be

$$V_2 = -GV$$

we obtain the input-output model:

$$\frac{V_2}{V_1} = -\frac{R_2}{R_1} \frac{1}{1 + \frac{1}{G} \left(1 + \frac{R_2}{R_1}\right)}$$

Sensitivity of gain  $\frac{V_2}{V_1}$  with respect to changes in the active component  $G$ , which typically vary from  $10^5$  to  $10^8$  can be neglected

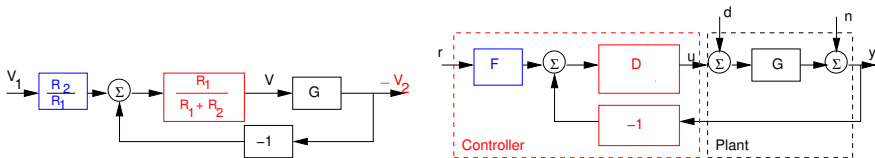
Explanation:

$$\frac{\partial \frac{V_2}{V_1}}{\partial G} = -\frac{R_2(R_1 + R_2)}{(GR_1 + (R_1 + R_2))^2}$$

# Value and Abstraction of Feedback Amplifier

IEEE LAMME MEDAL (1957): “... *the entire explosive extension of the area of control, both electrical and mechanical (fluid, heat, ...), grew out of the understanding of the feedback principle ...*”

## STANDARD FEEDBACK BLOCK-SCHEME

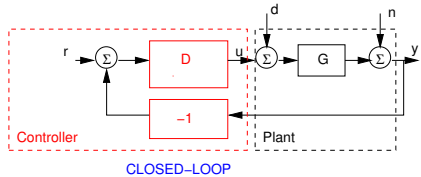
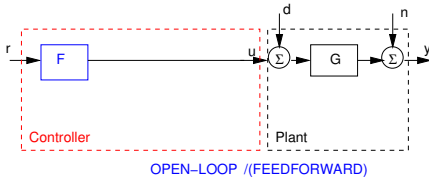


Convention: positive plant gain and negative feedback gain



# Generic Controller Architectures

Open- and Closed-loop configuration:

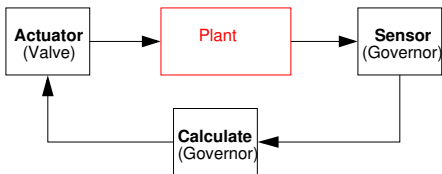


Reference Tracking


Disturbance reduction & robustness

# Summary of Magic of Feedback

Feedback consists of a **sensing, actuation AND calculation** element



Feedback has the potential to

- 1 **reduce** the effect of (load) disturbances
- 2 **change the dynamic response**
- 3  **destabilize** the system

Feedback involves making compromise (trade-off) between different performance criteria (1-2) and stability (3)

# Key Steps in Feedback Control Design

- Model the process (using insights in the physics, etc.)
- Analyze the model (stability, response, etc.)
- Design a controller to fulfill given performance criteria

→ that is what we will do in this course

## Required background knowledge

- Roots of quadratic functions and higher-degree polynomials
- Linear differential equations
- Laplace transforms
- Partial fraction expansions
- Complex numbers
- Dynamics of electrical and mechanical systems

# Mathematical Background (Recap)

- ① Dynamical Models
  - Differential equations
  - Transfer functions
- ② Laplace Transform
- ③ Response to a sinusoid
- ④ Final Value Theorem (FVT)

# Basic System Analysis

Differential equation

$$\ddot{y}(t) + a_1\dot{y}(t) + a_2y(t) = bu(t)$$

with Laplace transform:  $Y(s) = \int_0^\infty y(t)e^{-st}dt$  ( $y(t) \xrightarrow{\mathcal{L}_-} Y(s)$ )

Transfer function (TF):

$$H(s) := \frac{Y(s)}{U(s)} = \frac{b}{s^2 + a_1s + a_2}$$

while  $y(t) = \int_{-\infty}^\infty h(\tau)u(t-\tau)d\tau$  with  $h(\tau) \xleftarrow{\mathcal{L}_-^{-1}} H(s)$  being the impulse response of the system.

# Input-output (i/o) relation through transfer function

For

$$u(t) = e^{\lambda t} \quad \lambda \in \mathbb{C}$$

then

$$\begin{aligned} y(t) &= \int_{-\infty}^{\infty} h(\tau) u(t - \tau) d\tau \\ &= \underbrace{\int_{-\infty}^{\infty} h(\tau) e^{-\lambda \tau} d\tau}_{H(s)|_{s=\lambda}} \cdot e^{\lambda t} \end{aligned}$$

so that

$$y(t) = H(s)|_{s=\lambda} \cdot u(t)$$

# Response of a system to a sinusoidal input

Let

$$u(t) = A \cos(\omega t) = \frac{A}{2} [e^{j\omega t} + e^{-j\omega t}]$$

then

$$y(t) = \frac{A}{2} [H(j\omega)e^{j\omega t} + H(-j\omega)e^{-j\omega t}]$$

If  $H(j\omega) = |H(j\omega)|e^{j\angle H(j\omega)} = M(\omega)e^{j\phi(\omega)}$  then

$$y(t) = A \cdot M \cdot \cos(\omega t + \phi)$$

$H(j\omega)$  is the **frequency response function (FRF)** (complex valued)



# Final value theorem

$$\lim_{t \rightarrow \infty} y(t) = \lim_{s \rightarrow 0} sY(s)$$

Consequence:

If  $u$  is a unit step,  $U(s) = \frac{1}{s}$ ,

$$\begin{aligned}\lim_{t \rightarrow \infty} y(t) &= \lim_{s \rightarrow 0} s \cdot G(s) \cdot \frac{1}{s} \\ &= \lim_{s \rightarrow 0} G(s)\end{aligned}$$

is DC-gain of  $G$

## Example FVT

Consider the transfer function of a laser positioning system in a copy machine:

$$Y(s) = \frac{5(s+100)}{s^2 + 60s + 500} R(s)$$

What is the final value of the output  $y(t)$  ( $\lim_{t \rightarrow \infty} y(t)$ ) when the input  $r(t)$  is a unit step?

$$\begin{aligned} \lim_{t \rightarrow \infty} y(t) &= \lim_{s \rightarrow 0} sY(s) \\ &= \lim_{s \rightarrow 0} s \frac{5(s+100)}{s^2 + 60s + 500} \frac{1}{s} \\ &= \boxed{1} \end{aligned}$$



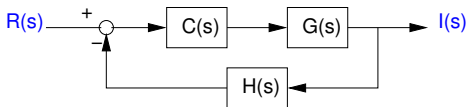
What is the final value of  $y(t)$  when it is the unit step response of the system with transfer function:

$$G(s) = \frac{K}{s-1}$$

Is it  $-K$ ?

# Calculation Transfer functions of a closed-loop system

Consider a block scheme representation of controlling a lamp's intensity by an opto-transistor feedback loop:



Determine the transfer function  $\frac{I(s)}{R(s)}$ ?

$$I(s) = G(s)C(s)(R(s) - H(s)I(s))$$
$$(1 + G(s)C(s)H(s))I(s) = G(s)C(s)R(s)$$
$$\frac{I(s)}{R(s)} = \left(1 + \boxed{G(s)C(s)H(s)}\right)^{-1} \boxed{G(s)C(s)}$$

**Rule for Negative Feedback SISO systems:** The direct-through path over one plus the loop gain (counterclockwise!)