















Tan Jieqi

Task 1 Use a weather forecast website, and utilize the psychrometric chart and the formula we went through in the class to determine the absolute humidity, the wet-bulb temperature and the mass of water vapour in the air in Classroom A (Aula A) of Piacenza campus in the moment that you are solving this exercise (provide the inputs that you utilized)

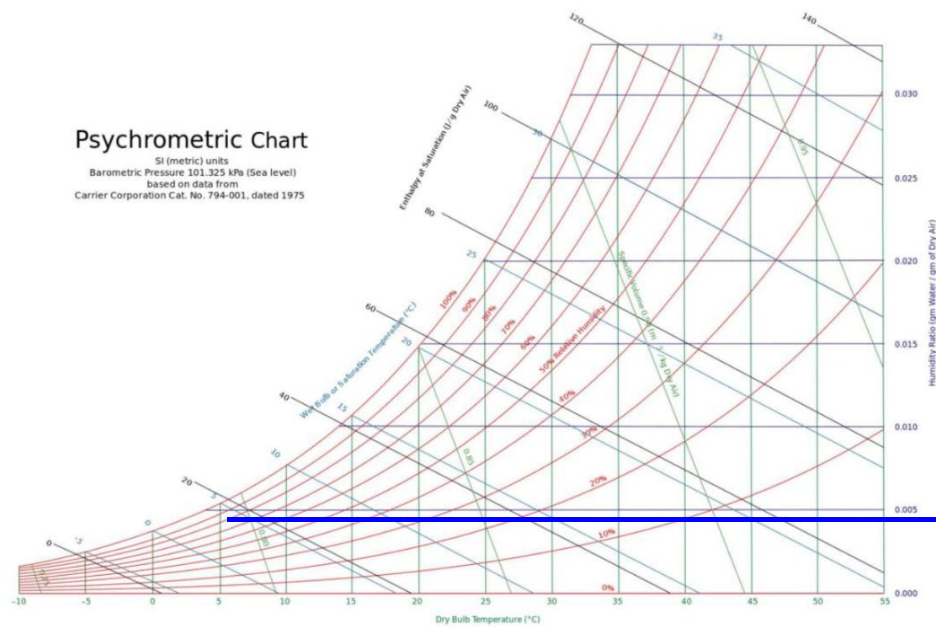
	05:00	07:00	10:00	14:00	18:00	7:00 pm	21:00
	 Cloud	 Cloud	 Cloud	 Cloud	 Partly Cloud	 Fog	 Fog
Effective temperature	9 ° C	9 ° C	9 ° C	11 ° C	9 ° C	9 ° C	10 ° C
Perceived temperature	9 ° C	9 ° C	9 ° C	11 ° C	8 ° C	8 ° C	10 ° C
Rainfall	0 mm	0 mm	0 mm	0 mm	0 mm	0 mm	0 mm
Humidity	96 %	96 %	97 %	95 %	99 %	99 %	97 %
Atmospheric pressure	1022 hPa	1021 hPa	1022 hPa	1019 hPa	1018 hPa	1018 hPa	1017 hPa
Wind intensity	1 km / h	2 km / h	3 km / h	6 km / h	6 km / h	6 km / h	4 km / h
Wind direction	 NEITHER	 NO	 N	 IS	 SELF	 SELF	 NEITHER
Probability of fog	0 %	0 %	1 %	0 %	0 %	53 %	55 %
Dew point	8 ° C	8 ° C	9 ° C	10 ° C	9 ° C	9 ° C	9 ° C
Clouds	100 %	100 %	97 %	100 %	50 %	84 %	98 %
Low clouds	99 %	100 %	96 %	99 %	36 %	68 %	66 %
Medium clouds	87 %	9 %	29 %	95 %	34 %	48 %	97 %
High clouds	95 %	66 %	9 %	1 %	0 %	0 %	0 %

$$P = 1.022 \text{ kPa}$$

$$\phi = 96\%$$

$$T = 9^\circ\text{C}$$

$$A = 10 \cdot 8 \cdot 5$$



As we can see in the chart:

$$\omega = 0.0045$$

$$T_{wb} = 5^{\circ}\text{C}$$

$$\text{when: } \omega = 0.0045 = 0.622 \frac{P_v}{P_a} = 0.622 \frac{P_v}{P - P_v}$$

$$P_v = 1.3584 \text{ kPa}$$

$$P_a = 101.456 \text{ kPa}$$

$$\text{Then } \rightarrow m = \frac{PV}{TR_{sp}}$$

$$m_a = \frac{P_a V_a}{TR_a} = 503.21 \text{ kg}$$

$$m_v = \frac{P_v V_a}{TR_v} = 4.19 \text{ kg}$$

Task 2 Utilize the same methodology we went through in the class and determine the sensible and latent load corresponding to internal gains, the ventilation, and the infiltration in a house with a *good* construction quality and with the same geometry as that of the example which is located in Brindisi, Italy

BRINDISI, Italy

WMO#: 163200

Lat: 40.65N

Long: 17.95E

Elev: 10

StdP: 101.2

Time Zone: 1.00 (EUW)

Period: 86-10

WBAN: 99999

Annual Heating and Humidification Design Conditions

Coldest Month	Heating DB		Humidification DP/MCDB and HR						Coldest month WS/MCDB				MCWS/PCWD to 99.6% DB		
			99.6%			99%			0.4%		1%				
	99.6%	99%	DP	HR	MCDB	DP	HR	MCDB	WS	MCDB	WS	MCDB	MCWS	PCWD	
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)	
(1)	2	2.9	4.1	-5.1	2.5	7.2	-3.0	3.0	7.4	13.4	10.2	12.4	10.6	3.4	250

Annual Cooling, Dehumidification, and Enthalpy Design Conditions

Hottest Month	Hottest Month DB Range	Cooling WB/MCWB						Evaporation WB/MCDB						MCWS/PCWD to 0.4% DB	
		0.4%		1%		2%		0.4%		1%		2%			
		DB	MCWB	DB	MCWB	DB	MCWB	WB	MCDB	WB	MCDB	WB	MCDB	MCWS	PCWD
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)	(p)
(7)	8	7.1	32.8	23.6	31.1	24.3	29.9	24.3	27.2	29.7	26.3	29.0	25.6	28.3	4.2 180

	Dehumidification DP/MCDB and HR									Enthalpy/MCDB						Hours 8 to 4 & 12.8/20.6
	0.4%			1%			2%			0.4%		1%		2%		
	DP	HR	MCDB	DP	HR	MCDB	DP	HR	MCDB	Enth	MCDB	Enth	MCDB	Enth	MCDB	
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)	(p)	
(3)	26.3	21.8	29.2	25.4	20.7	28.5	24.7	19.7	27.9	86.0	30.1	82.2	29.1	78.5	28.3	1236

Extreme Annual Design Conditions

Extreme Annual WS			Extreme Max WB	Extreme Annual DB				n-Year Return Period Values of Extreme DB									
				Mean		Standard deviation		n=5 years		n=10 years		n=20 years		n=50 years			
1%	2.5%	5%		Min	Max	Min	Max	Min	Max	Min	Max	Min	Max	Min	Max	Min	Max
(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(l)	(m)	(n)	(o)	(p)	(q)	(r)
(4)	11.3	9.9	8.7	31.4	0.4	37.3	1.4	3.0	-0.6	39.4	-1.4	41.1	-2.2	42.8	-3.2	44.9	

$$Q_{igensible} = 136 + 2.2A_{cf} + 22N_{oc} = 620W$$

$$Q_{iglaten} = 20 + 0.22A_{cf} + 12N_{oc} = 88W$$

-->

$$A_{ul} = 1.4 \text{ cm}^2 / \text{m}^2$$

$$A = A_{wall} + A_{roof} = 344 \text{ m}^2$$

$$--> T_{cooling} = 24^\circ\text{C} --> \Delta T_{cooling} = 7.1K$$

$$--> T_{heating} = 20^\circ\text{C} --> \Delta T_{cooling} = 24.1K$$

$$DR = 7.1^\circ\text{C}$$

Then

$$IDF_{cooling} = 0.33 \frac{L}{s * \text{cm}^2}$$

$$IDF_{heating} = 0.073 \frac{L}{s * \text{cm}^2}$$

$$Q_{inf,heating} = A * IDF_{heating} = 35.15 \frac{L}{s}$$

$$Q_{inf,cooling} = A * IDF_{cooling} = 15.89 \frac{L}{s}$$

$$Q_v = 0.05A_{cf} + 3.5(N_{br} + 1) = 17 \frac{L}{s}$$

$$Q_{inf-v,heating} = Q_{inf,heating} + Q_v = 52.15 \frac{L}{s}$$

$$Q_{inf-v,cooling} = Q_{inf,cooling} + Q_v = 32.89 \frac{L}{s}$$

So

$$C_{sensible} = 1.23$$

$$C_{latent} = 3010$$

$$\Delta\omega_{cooling} = 0.0039$$

$$q_{inf-v,cooling, sensible} = C_{sensible} Q_{inf-v,cooling} \Delta T_{cooling} = 287.25W$$

$$q_{inf-v,cooling, latent} = C_{latent} Q_{inf-v,cooling} \Delta\omega_{cooling} = 386.13W$$

$$q_{inf-v,heating, sensible} = C_{sensible} Q_{inf-v,heating} \Delta T_{cooling} = 1546W$$