

Physical System Representation

PDE

↓ Transform, Graph, approximation

ODE's

Laplace
Transforms

Nested
Integrals

State
Space

$$G(s) = C(sI - A)^{-1}B + D$$

$G(s)$ Transfer
function

$$Y(s) = G(s) U(s)$$

inverse
Laplace Transform

Impulse
Response

$$y(t) = \int_0^t g(t-\tau) u(\tau) d\tau$$

$$= g(t) * u(t)$$

$$= C$$

first-order
system
representation

$$\dot{x} = Ax + Bu$$

$$y = Cx + Du$$

NOTE: Note the following

- ① $SS \rightarrow TF$ (on MATLAB): $ss2tf$
 $TF \rightarrow SS$ (on MATLAB): $tf2ss$
- ② ODE's & TF's are unique for a given system
- ③ SS Representations are not unique
- ④ We may go from transfer function to ODE

E.g: $G(s) = \frac{Y(s)}{U(s)} = \frac{10}{s^2 + 4s + 5} \Rightarrow (s^2 + 4s + 5)Y(s) = 10U(s)$

with all zero initial conditions in time, we have

$$\ddot{y}(t) + 4\dot{y}(t) + 5y = 10u$$

⑤ Discrete-Time System Representation

Given

$$\begin{aligned} \dot{x}(t) &= A x(t) + B u(t) \\ y(t) &= C x(t) + D u(t) \end{aligned} \quad \Leftarrow \text{Linear Time Invariant (LTI) System}$$

Suppose during interval Δt , $u(t)$ is constant (zero-order hold)
 $u_k \equiv u(t)$ for $t \in [k\Delta t, (k+1)\Delta t)$

then, $x_k = x(k\Delta t)$ will be

$$x_{k+1} = A_d x_k + B_d u_k$$

$$y_k = C_d x_k + D_d u_k$$

where $x_k = x(k\Delta t)$

$$A_d = e^{A\Delta t}$$

$$B_d = \int_0^{\Delta t} e^{A\tau} B d\tau$$

$$C_d = C$$

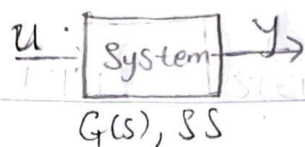
$$D_d = D$$

In fact,

$$\begin{bmatrix} A_d & B_d \\ 0 & I \end{bmatrix} = e^{\begin{bmatrix} A & B \\ 0 & 0 \end{bmatrix} \Delta t}$$

STABILITY OF LTI SYSTEMS

Stability



Bounded output $|y(t)| < \infty \forall t$
if $|u(t)| < \infty \forall t$

① $G(s)$ given \rightarrow find poles of $G(s)$ and confirm that they lie in the open LHP

② State Space given: $\dot{x}(t) = Ax(t) + Bu(t)$

$$x(t) = e^{At}x(0) + \int_0^t e^{A(t-\tau)}Bu(\tau)d\tau$$
$$= e^{At}x(0) + e^{At}B * u(t)$$

Assuming x is a scalar; $A=a$, $B=b$. For small changes in u to cause only small changes in x , e^{at} must always remain bounded. Hence the system is stable if e^{at} decays to zero or $a < 0$

For vector x , e^{At} must also decay to zero: that happens iff all $\text{eig}(A)$ lie in the Open Left Hand Plane

NOTE Poles of $G(s) \equiv$ Eigenvalues of A

Hint:

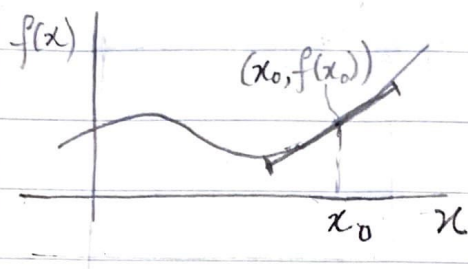
$$(sI - A)^{-1} = \mathcal{L}(e^{At})$$

\downarrow

Poles: values for which $|sI - A| = 0$

Eigenvalue: characteristic (Equation) roots
 $|sI - A| = 0$

LINEARIZATION



For a nonlinear function $f(x)$, given x_0 , we can approximate $f(x)$ in the neighborhood of x_0 by a linear function of the form:

$$\delta f(x) = a \delta x$$

$$f(x) - f(x_0) = a(x - x_0)$$

$$a = \left. \frac{df}{dx} \right|_{x=x_0}$$

In general, for multivariable functions

$$x = \begin{bmatrix} x_1 \\ x_2 \\ \vdots \\ x_n \end{bmatrix} \in \mathbb{R}^n, \quad f(x) = \begin{bmatrix} f_1 \\ f_2 \\ \vdots \\ f_n \end{bmatrix} \in \mathbb{R}^q$$

$$\delta f(x) = \underbrace{\nabla_x f}_{\text{Jacobian (q} \times \text{n matrix)}} \delta x$$

Jacobian (q \times n matrix)

$$\nabla_x f = \begin{bmatrix} \frac{\partial f_1}{\partial x_1} & \frac{\partial f_1}{\partial x_2} & \dots & \frac{\partial f_1}{\partial x_n} \\ \frac{\partial f_2}{\partial x_1} & \frac{\partial f_2}{\partial x_2} & \dots & \frac{\partial f_2}{\partial x_n} \\ \vdots & \vdots & \ddots & \vdots \\ \frac{\partial f_q}{\partial x_1} & \dots & \dots & \frac{\partial f_q}{\partial x_n} \end{bmatrix}$$

Hence, given $\dot{x} = f(x, u)$ $x \in \mathbb{R}^n$ state
 $y = h(x, u)$ $u \in \mathbb{R}^m$ inputs
 $y \in \mathbb{R}^p$ output

$$f = \begin{bmatrix} f_1(x, u) \\ f_2(x, u) \\ \vdots \\ f_n(x, u) \end{bmatrix}$$

$$h = \begin{bmatrix} h_1(x, u) \\ \vdots \\ h_p(x, u) \end{bmatrix}$$

We can pick the point $\begin{Bmatrix} x_e \\ u_e \end{Bmatrix}$, and define the deviation variable

$$\delta x = x - x_e$$

$$\delta u = u - u_e$$

The linearized state space can then be written as

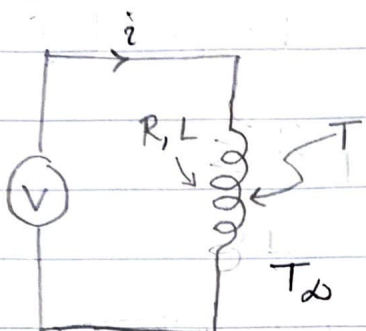
$$\frac{d}{dt}(\delta x) = \left. \nabla_x f \right|_{\substack{x=x_e \\ u=u_e}} \delta x + \left. \nabla_u f \right|_{\substack{x=x_e \\ u=u_e}} \delta u$$

$$\delta y = \left. \nabla_x h \right|_{\substack{x=x_e \\ u=u_e}} \delta x + \left. \nabla_u h \right|_{\substack{x=x_e \\ u=u_e}} \delta u$$

for short: $\frac{d}{dt}(\delta x) = A \delta x + B \delta u$

$$\delta y = C \delta x + D \delta u$$

Ex 1: Electric Heater



$$C\dot{T} = i^2 R + K(T_{\infty} - T)$$

$$L \frac{di}{dt} + iR = v$$

input

$T \leftarrow$ Output

Linearize about

$$V = V_0$$

$$T = T_0$$

$$i = i_0$$

$$T_{\infty} \equiv 0$$

Step 1: Write ODE's in standard state-space form

$$x_1 = T, \quad x_2 = i, \quad u = V$$

$$\dot{x}_1 = \frac{R}{C} x_2^2 + \frac{K}{C} T_{\infty} - \frac{K}{C} x_1$$

$$\dot{x}_2 = -\frac{R}{L} x_2 + \frac{u}{L}$$

$$y = x_1$$

$$f = \begin{bmatrix} -\frac{K}{C} x_1 + \frac{R}{C} x_2^2 \\ -\frac{R}{L} x_2 + \frac{u}{L} \end{bmatrix}$$

$$h = x_1$$

Step 2: linearize

$$\nabla_x f = \begin{bmatrix} -k/c & 2R/c x_2 \\ 0 & -R/L \end{bmatrix}$$

$$\nabla_u f = \begin{bmatrix} \frac{\partial f}{\partial u} \\ \frac{\partial f}{\partial u} \end{bmatrix} = \begin{bmatrix} 0 \\ 1/L \end{bmatrix}$$

$$\nabla_x h = \begin{bmatrix} 1 & 0 \end{bmatrix}$$

$$\nabla_u h = 0$$

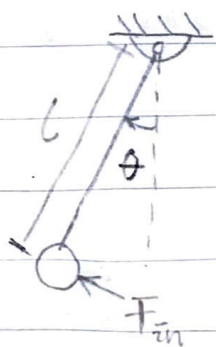
$$A = \nabla_x f \Big|_{\substack{x=x_0 \\ u=u_0}} = \begin{bmatrix} -k/c & 2R/c \dot{x}_0 \\ 0 & -R/L \end{bmatrix}$$

$$B = \nabla_u f \Big|_{\substack{x=x_0 \\ u=u_0}} = \begin{bmatrix} 0 \\ 1/L \end{bmatrix}$$

$$C = \nabla_x h \Big|_{\substack{x=x_0 \\ u=u_0}} = \begin{bmatrix} 1 & 0 \end{bmatrix}$$

$$D = \nabla_u h \Big|_{\substack{x=x_0 \\ u=u_0}} = 0$$

Ex2: Simple Pendulum



$$l F_{in} = \overset{\text{mass}}{m} g l \sin \theta = \overset{\text{moment of inertia}}{I} \ddot{\theta}$$

$y = \theta$ output
 $u = F_{in}$ input

Linearize about equilibrium point

$$\text{Let } x = \theta, \quad x_2 = \dot{\theta}, \quad u = F_{in}$$

$$\dot{x}_1 = x_2$$

$$\dot{x}_2 = \ddot{\theta} = -\frac{mgl \sin \theta}{I} + \frac{l}{I} u = -\frac{mgl \sin x_1}{I} + \frac{l}{I} u$$

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} x_2 \\ \overbrace{-\frac{mgl}{I} \sin(x_1) + \frac{L}{I} u}^{f(x,u)} \end{bmatrix}$$

$$y = u \leftarrow h(x, u)$$

An equilibrium point x_e is a state about which the dynamics are zero

$$x_e \text{ is such that } f(x_e, 0) = 0$$

$$f(x_e, 0) = \begin{bmatrix} x_{e,2} \\ -\frac{mgl}{I} \sin(x_{e,1}) \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix} \Rightarrow \begin{aligned} x_{e,2} &= 0 \\ \sin(x_{e,1}) &= 0 \end{aligned}$$

$$x_{e,1} = k\pi$$

stable - even k
unstable - odd k
equilibrium

$$x_e = \begin{bmatrix} k\pi \\ 0 \end{bmatrix} \quad k \in \mathbb{Z}$$

$$A = \left. \begin{bmatrix} \frac{\partial f_1}{\partial x_1} & \frac{\partial f_1}{\partial x_2} \\ \frac{\partial f_2}{\partial x_1} & \frac{\partial f_2}{\partial x_2} \end{bmatrix} \right|_{x_e = \begin{bmatrix} 0 \\ 0 \end{bmatrix}} = \left. \begin{bmatrix} 0 & 1 \\ -\frac{mgl}{I} \cos(x_1) & 0 \end{bmatrix} \right|_{x_e = \begin{bmatrix} 0 \\ 0 \end{bmatrix}}$$

$$A = \begin{bmatrix} 0 & 1 \\ -\frac{mgl}{I} & 0 \end{bmatrix}$$

$$B = \left. \begin{bmatrix} \frac{\partial f}{\partial u} \\ \frac{\partial f}{\partial u} \end{bmatrix} \right|_{x_e = \begin{bmatrix} 0 \\ 0 \end{bmatrix}} = \begin{bmatrix} 0 \\ \frac{L}{I} \end{bmatrix}$$

$$C = \left[\frac{\partial h}{\partial x_1}, \frac{\partial h}{\partial x_2} \right] \bigg|_{\substack{x = \begin{bmatrix} 0 \\ 0 \end{bmatrix} \\ u=0}} = [1 \quad 0]; \quad D = 0$$

NOTE: $\text{eig}(A) = \pm i \sqrt{\frac{mgL}{I}}$

If we used an odd k , say $k=1$; $\text{eig}(A) = \pm \sqrt{\frac{mgL}{I}}$