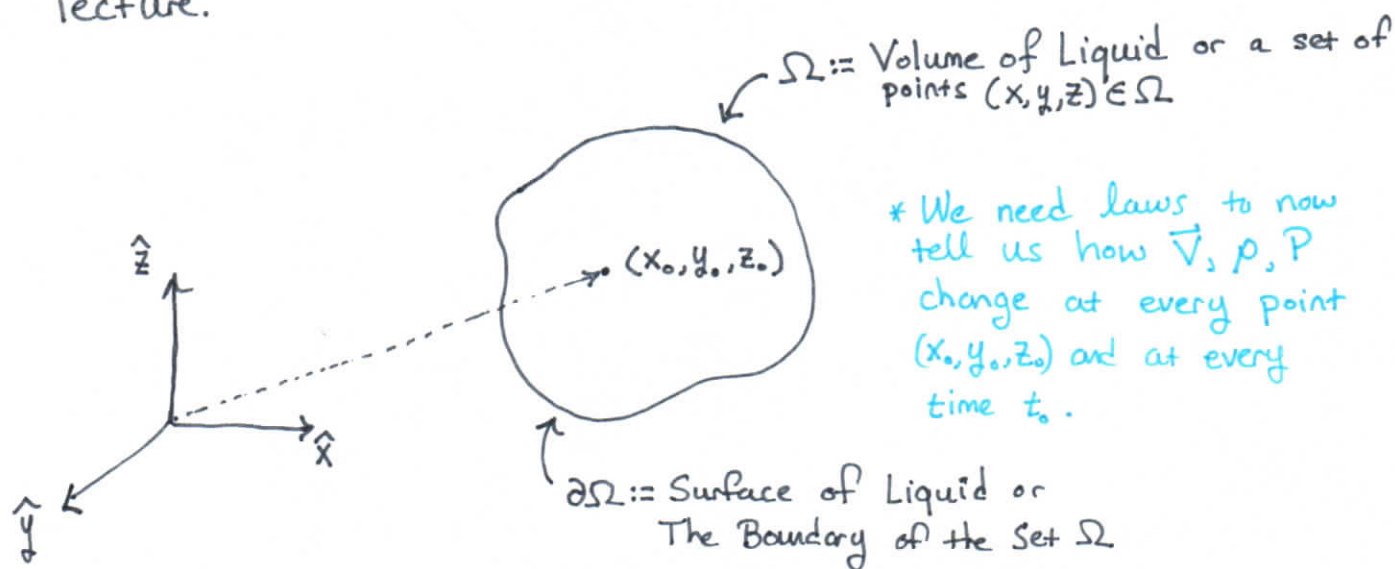


# LECTURE 14 : WHAT IF IT'S GAS FLOWING INSTEAD OF A LIQUID?

Up until now we have been under the assumption that of INCOMPRESSIBLE FLUID. It is natural that we round out this intro to applied fluids by introducing the applications of compressible flow. How do you tell your problem is compressible? Here's just a list of a few.

- 1) Fluid is a gas
- 2) VERY HIGH Temperature
- 3) Enormous Pressures
- 4)  $Ma \sim \mathcal{O}(1)$

Mathematically COMPRESSIBLE is defined for flow with non-constant density field. Recall our very first lecture.



$$(1) \quad \frac{D\vec{V}}{Dt} = \frac{1}{\rho} \{ \text{Forces} \}$$

$$(2) \quad \frac{D\rho}{Dt} = \{ \text{Prescribe how Density Changes in } \Omega \}$$

$$(3) \quad \frac{DP}{Dt} = \{ \text{Prescribe how } P \text{ is coupled with } \vec{V} \text{ \& } \rho \}$$

## LECTURE 14:

The only thing NEW is (3) which is a thermodynamical Equation of State. We know that (2) is still conservation of mass only this time  $\rho \neq \text{constant}$  so it won't simplify to  $\nabla \cdot \vec{V} = 0$ . Intuitively then, you should understand that Density and Velocity will be coupled. To compare both situations let's write out the governing equations.  $\vec{V} = \langle u(x,y,z,t), v(x,y,z,t), w(x,y,z,t) \rangle \in \mathbb{R}^3$ ,  $\rho = \rho(x,y,z,t) \in \mathbb{R}$  and  $P = P(x,y,z,t) \in \mathbb{R}$ .

INCOMPRESSIBLE  $\rho = \rho_0$

COMPRESSIBLE  $\rho \neq \text{constant}$

$$(1a) \quad \nabla \cdot \vec{V} = 0$$

$$\frac{D\rho}{Dt} = -\rho(\nabla \cdot \vec{V}) \quad (1b)$$

$$(2a) \quad \rho \frac{D\vec{V}}{Dt} = -\nabla P + \mu \nabla^2 \vec{V} + \rho \vec{F}_b$$

$$\rho \frac{D\vec{V}}{Dt} = -\nabla P + \mu \nabla^2 \vec{V} + \lambda \nabla(\nabla \cdot \vec{V}) + \rho \vec{F}_b \quad (2b)$$

4 equations, (2a) is 3 equations  
1 for each dimension (x,y,z),  
and 4 unknowns

$$\frac{DP}{Dt} = \frac{B}{A} \rho(\nabla \cdot \vec{V}) + \frac{C}{A} \quad (3)$$

5 equations and 5 unknowns

Extra Credit

List the dependent functions in each case,

Show the 5<sup>th</sup> equation is technically still there for incompressible

Show compressible reduces to incompressible if something is zero

Equation 3 is a general form of an equation of State. Commonly we just use a very simple equation of state for air.

$$\rho = \frac{P}{RT}$$

(ideal gas law)

## LECTURE 14

The biggest take away is compressible flow adds terms and equations to the governing equation of motion. The addition of these terms creates a closure problem, (to many unknowns, not enough equations) to close this system of equations we must use THERMODYNAMICS in order to relate ...

$\text{density} \sim f(P, T)$  "Some function of pressure and Temperature"

# Crash Course In Thermo

Sam Style...

0<sup>th</sup> Law : "Allows use to define temperature as a property of a system."

2 systems that are thermal equilibrium are at the same temperature.

1<sup>st</sup> Law : "Allows us to introduce the property energy to a system."

The sum of infinitesimal amounts of heat and work equal the change in energy to the system.

$$\delta Q + \delta W = dE$$

This means "variation" and means that the quantity is path dependent.

Neither of these things are properties of a system yet their algebraic sum is a property of a system.

## LECTURE 14

2<sup>nd</sup> Law: "Allows us to introduce an absolute scale for temperature and define a property called entropy."

The change in entropy is bounded below by the positive ratio of heat exchange and temperature.

$$dS \geq \frac{\delta Q}{T}$$

Equality holds for ideal reversible processes.

//

Let's combine the first and second law and start to build new relations. First we will define Enthalpy.

$$H = U + PV$$

$V := \text{volume}$

per unit  
mass

$$\rightarrow h = u + \frac{P}{\rho}$$

Now using (2) rewrite  $\delta Q$  and we get. if we neglect gravity, motion, electrical, capillary, magnetic effects.

$$TdS - PdV = dU$$

But look, what if we say internal energy is a function of properties.

$$U = U(S, V)$$

$$dU = \left(\frac{\partial U}{\partial S}\right) dS + \left(\frac{\partial U}{\partial V}\right) dV$$

Cool right!



## LECTURE 14

Now we have a new interpretation of  $T$  &  $P$ .

$$T = \left( \frac{\partial U}{\partial S} \right) = \frac{\text{change in internal energy}}{\text{change in system entropy}}$$

$$P = - \left( \frac{\partial U}{\partial V} \right) = \frac{\text{negative change in internal energy}}{\text{change in system volume.}}$$

Let's keep going. More derivatives!!!

$$\left. \begin{aligned} \frac{\partial T}{\partial V} &= \frac{\partial^2 U}{\partial V \partial S} \\ \frac{\partial P}{\partial S} &= - \frac{\partial^2 U}{\partial S \partial V} \end{aligned} \right\} \begin{array}{l} \text{order of } \partial V \partial S \\ \text{doesn't matter.} \end{array}$$

$$\therefore \left( \frac{\partial S}{\partial P} \right) = - \left( \frac{\partial V}{\partial T} \right)$$

Maxwell's  
Relation

Now let's look into our density issue. What do changes in  $\rho(P, T)$  look like.

$$d\rho = \left( \frac{\partial \rho}{\partial T} \right) dT + \left( \frac{\partial \rho}{\partial P} \right) dP$$

are you seeing a "theme" in thermo?

$$(1) \quad \frac{d\rho}{\rho} = \frac{1}{\rho} \left( \frac{\partial \rho}{\partial T} \right) dT + \frac{1}{\rho} \left( \frac{\partial \rho}{\partial P} \right) dP$$

Same expression but in terms of volume.

$$(2) \quad \frac{dV}{V} = \frac{1}{V} \left( \frac{\partial V}{\partial T} \right) dT + \frac{1}{V} \left( \frac{\partial V}{\partial P} \right) dP$$

## LECTURE 14

Let's go back to densities definition.

$$\rho(T, P) = \frac{m}{V(T, P)}$$

So in order to reconstruct (1) and (2)

$$d\rho = d\left(\frac{m}{V}\right) \quad \& \quad dV = d\left(\frac{m}{\rho}\right)$$

Then using the quotient rule.

$$d\rho = \frac{\cancel{dm} V - m dV}{V^2} = -\frac{m}{V} \frac{dV}{V}$$

conservation of mass (=0)

$$dV = \frac{\cancel{dm} \rho - m d\rho}{\rho^2} = -\frac{m}{\rho} \frac{d\rho}{\rho}$$

=0

Divide both by  $\rho$  and  $V$  respectively.

$$\frac{d\rho}{\rho} = \frac{\cancel{-m}}{\rho V} \frac{dV}{V}$$

=1

$$\frac{dV}{V} = \frac{\cancel{-m}}{V \rho} \frac{d\rho}{\rho}$$

=1

Now by adding (1) and (2) we can derive 2 more properties of fluids and systems.

$$\cancel{\frac{d\rho}{\rho} + \frac{dV}{V}} = \underbrace{\left\{ \frac{1}{\rho} \frac{\partial \rho}{\partial T} + \frac{1}{V} \frac{\partial \rho}{\partial P} \right\}}_{=0} dT + \underbrace{\left\{ \frac{1}{\rho} \frac{\partial V}{\partial T} + \frac{1}{V} \frac{\partial V}{\partial P} \right\}}_{=0} dP$$

=0 by derivation  
and true symmetrically

\* These coefficients must be  
identically zero since  $dT, dP$  are arbitrary.

## LECTURE 14

Coefficient  
of  
Volume  
Expansion

$$\beta \equiv \frac{1}{V} \left( \frac{\partial V}{\partial T} \right) \equiv -\frac{1}{P} \left( \frac{\partial P}{\partial T} \right)$$

$$K \equiv -\frac{1}{V} \left( \frac{\partial V}{\partial P} \right) \equiv \frac{1}{\rho} \left( \frac{\partial \rho}{\partial P} \right)$$

Coefficient of  
Compressibility

We can combine these more to derive a definition for the.

The Bulk  
Modulus  
of  
Elasticity

$$E = \frac{1}{K - \beta \frac{dT}{dP}}$$

for liquids and solids this term is super small so  $E \approx 1/K$ .

- \* For a gas  $\frac{dT}{dP} \gg 1$  so and \*
- \* therefore  $E$  is not a property. \*

- \* Compressibility depends on the \*
- \* process itself not the fluid \*

- \* LET'S USE THE IDEAL GAS LAW \*
- \* FINALLY TO DETERMINE SOME \*
- COMPRESSIBLE FLOW STUFF!

## LECTURE 14:

The equation of state for an ideal perfect gas.

$$\underline{Pv = mRT} \quad (*)$$

Or in terms of density.

$$\rho(T, P) = \frac{P}{RT} \quad (\text{E.S.})$$

This allows us to take derivatives and go further go deeper. So from a perfect gas we go deeper with 2<sup>nd</sup> law!

$$TdS = dU + PdV$$

$$T \left( \frac{\partial S}{\partial V} \right) = \left( \frac{\partial U}{\partial V} \right) + P$$

From here we can show that  $U(T)$  and not  $U(T, V)$ . This is important because for a single var function we can Taylor expand for representations.

If we consider internal energy per unit mass  $u(T)$  Lower case always mean per unit mass

$$u(T) \cong u_0 + \left( \frac{\partial u}{\partial T} \right) \delta T + \frac{\partial^2 u}{\partial T^2} (\delta T)^2$$

assume constant neglect

Same for enthalpy since it's related to internal energy.

$$h(T) = u(T) + \frac{P}{\rho}$$

$$h(T) = h_0 + \left( \frac{\partial h}{\partial T} \right) \delta T + \frac{\partial^2 h}{\partial T^2} (\delta T)^2$$

assume constant value for simplicity neglect



## LECTURE 14

We have arrived at the specific heat variables!

$$C_v = \left( \frac{\partial u}{\partial T} \right)_{v=v.}$$

constant volume  
specific-heat

$$C_p = \left( \frac{\partial h}{\partial T} \right)_{p=p.}$$

constant pressure  
specific-heat.

For ideal gases as well. By definition,

$$C_p - C_v = R$$

Extra Credit show  
this from  $C_p$  &  $C_v$   
defs

A great tool for state analysis is the ratio

$$\gamma = \frac{C_p}{C_v}$$

For air under moderate pressure & temperatures.

$$C_p = 1.0035 \left[ \frac{\text{kJ}}{\text{kg K}} \right]$$

$$C_v = 0.7165 \left[ \frac{\text{kJ}}{\text{kg K}} \right]$$

$$\gamma \cong 1.4$$

We return to the entropy version of the 2<sup>nd</sup> law.

$$dS = \frac{du}{T} + \frac{P}{T} dv$$

## LECTURE 14

$$ds = \frac{dh}{T} - \frac{dP}{\rho T} \quad (\text{Enthalpy Form})$$

For an ideal gas though if  $c_p$  is constant.

$$c_p = \frac{dh}{dT}$$

$$c_p dT = dh$$

$$\therefore c_p \frac{dT}{T} = \frac{dh}{T}$$

Similarly since  $\rho = P/RT$

$$\rho T = \frac{P}{R}$$

$$\therefore \frac{dP}{\rho T} = R \frac{dP}{P}$$

Put it all together and get.

$$ds = c_p \frac{dT}{T} - R \frac{dP}{P}$$

Integrate equation to get relationship based on state  $1 \rightarrow 2$ .

$$\Delta S = c_p \ln\left(\frac{T_2}{T_1}\right) - R \ln\left(\frac{P_2}{P_1}\right)$$

## LECTURE 14

So for any process (even let's say flow) an ideal gas must obey, from state  $1 \rightarrow 2$

Now you know  $C_p$  &  $C_v$  are actually values of entropy  $\rightarrow \Delta S = C_p \ln\left(\frac{T_2}{T_1}\right) - (C_p - C_v) \ln\left(\frac{P_2}{P_1}\right)$

$$\frac{\Delta S}{C_v} = \gamma \ln\left(\frac{T_2}{T_1}\right) - \left(\frac{C_p - C_v}{C_v}\right) \ln\left(\frac{P_2}{P_1}\right)$$

normalized entropy w/ respect to gas

$$\Delta \tilde{S} = \gamma \ln\left(\frac{T_2}{T_1}\right) - (\gamma - 1) \ln\left(\frac{P_2}{P_1}\right)$$

Now if we assume the process is Isentropic  $\Rightarrow \Delta S = 0$ . This means no change in entropy! But this is also called a reversible case process. Recall.

$$dS \geq \frac{\delta Q}{T}$$

Extra credit how can we use the second law to show something is almost isentropic, you need 2- things

If  $dS \approx 0$  we get the isentropic relationships for an ideal-gas (air).

If you didn't know 2 states you could write both down but you'll need to provide more state information

$$\left\{ \begin{array}{l} \frac{T_2}{T_1} = \left(\frac{P_2}{P_1}\right)^{\frac{(\gamma-1)}{\gamma}} \\ \frac{P_2}{P_1} = \left(\frac{\rho_2}{\rho_1}\right)^{\gamma} \\ \frac{T_2}{T_1} = \left(\frac{\rho_2}{\rho_1}\right)^{\gamma-1} \end{array} \right.$$

You have to know 3 vars to "use" one of these equations.

The Gibbs Equations

## LECTURE 14

The last thing is we will derive the "speed of sound" based on "thermodynamical" properties! This is important to do because we need an upper limit of velocity that is based on something that sits outside Kinematics or Classical Mechanics. We need our velocity that normalizes all velocity values to be from a separate theory.

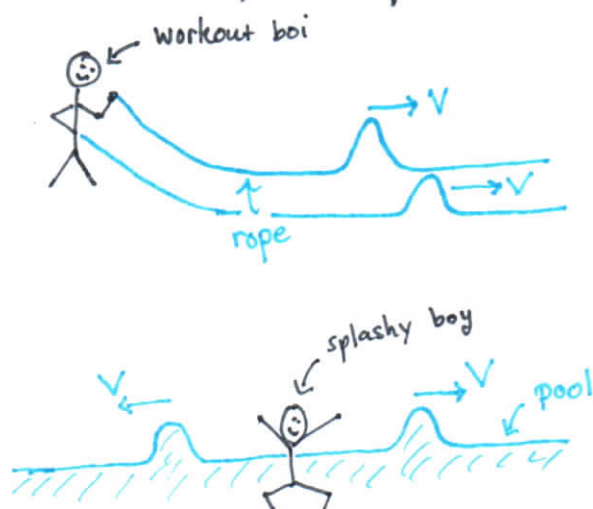
---

### The Speed of Sound "c"

- \* This speed is simple the velocity of a "pressure wave" traveling thru a medium. By pressure wave we mean the compression and expansion of air molecules.

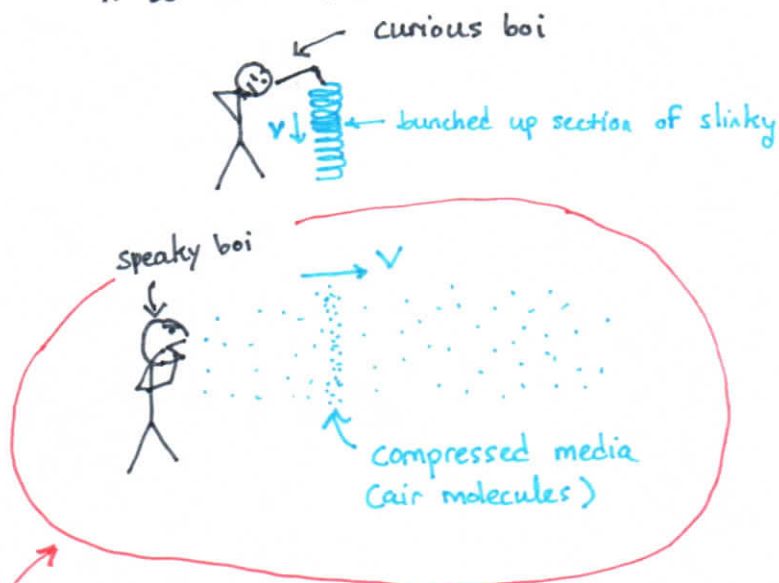
#### Transverse Wave Speed

- \* flicking a rope and watching the bump move away from you
- \* The top of liquid surface



#### Longitudinal Wave Speed

- \* Sending a "pulse" down a slinky
- \* The air pulse from one of those air guns (really a vortex)
- \* a sound!



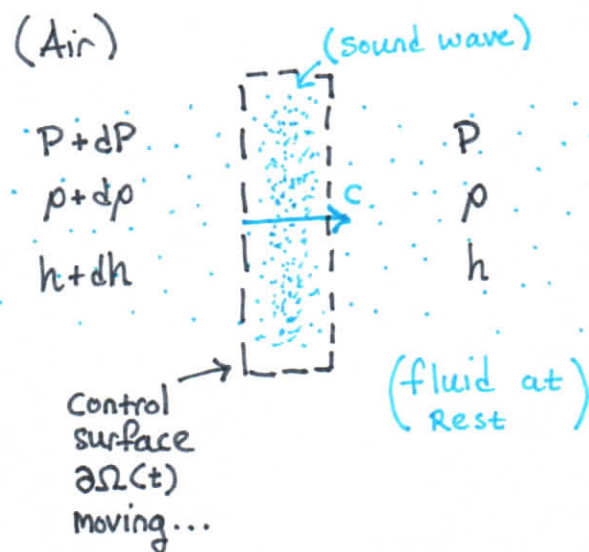
these are the physical waves we're talking about all class.



# LECTURE 14

Well let's look at that section of bunched up air molecules.

\* Since our surface  $\partial\Omega(t)$  is moving with the wave we must switch to a relative view!



Conserve Mass in  $\partial\Omega$

$$Q_{in} = Q_{out}$$

$$\rho A c = (\rho + d\rho) A (c - dV)$$

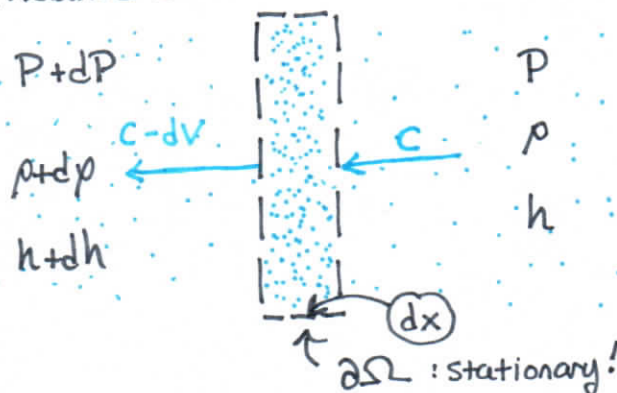
$$= (\rho + d\rho) A c - A d\rho c - \rho A dV + A d\rho dV$$

$$\cancel{\rho A c} = \cancel{\rho A c} - \rho A dV + A d\rho c - \cancel{A d\rho dV}$$

\* The multiplication of  $d\rho dV$  is negligible!  
 $(0.0001)(0.0001) \ll (0.0001)$

$$\boxed{\rho dV = c d\rho}$$

(Relative Control Volume)



Conserve Momentum in  $\partial\Omega$  (this will involve pressure  $P$ )

Work =  $\Delta KE$  (I don't know why but I can only remember the Work Energy Thm)

$$(\Sigma F_{ext}) dx = \{PA - (P + dP)A\} dx = [\rho A c \{ (c - dV) - c \}] dx$$

$$\boxed{dP = \rho c dV}$$

after neglecting  $d(\ )d(\ )$  terms.

Solving for  $c$  we finally arrive at our velocity.

$$dV = \frac{dP}{\rho c} \Rightarrow \cancel{\rho} \frac{dP}{\cancel{\rho} c} = c d\rho \Rightarrow$$

$$c^2 = \frac{dP}{d\rho}$$

\* always positive quantity

## LECTURE 14

Since the right hand side is a derivative of some EQUATION OF STATE, thermodynamical quantities govern this velocity limit.

$$C = \sqrt{\frac{\partial P}{\partial \rho}}$$

Now for an ideal gas we had the relationship

$$P = \rho RT \quad \leftarrow \text{you will be wrong if you just this relationship.}$$

But recall we also showed that for an isentropic process

$$P = C \rho^\gamma$$

Either way we ~~we~~ will get the speed of sound for an isentropic process of an ideal gas.

$$\frac{\partial P}{\partial \rho} = C \gamma \rho^{\gamma-1} = \gamma C \left( \frac{\rho^\gamma}{\rho} \right)$$

$$= \gamma C \left( \frac{P}{C \rho} \right)$$

$$= \gamma \left( \frac{P}{\rho} \right)$$

$$= \gamma RT$$

$$\begin{aligned} \therefore C &= \sqrt{\gamma RT} \sim 50 \sqrt{T} \text{ [ft/s]} \\ &\sim 340 \text{ [m/s]} \text{ @ } T = 15^\circ\text{C} \\ &\sim 20 \sqrt{T} \text{ [m/s]} \end{aligned}$$

## LECTURE 14

- \* This relationship is true for all media to some degree.

$$C = \sqrt{\left. \frac{\partial P}{\partial \rho} \right|_{S=S_0}} \quad (\text{most general})$$

Recall we have derived  $K$  &  $\beta$ , true for all media.

$$K = \frac{1}{\rho} \left( \frac{\partial \rho}{\partial P} \right) \leftarrow \text{for solids and liquids this derivative is too small though!!!}$$

$$C = \sqrt{\frac{1}{\rho K}}$$

Moreover recall we made a derived material prop  $E$ .

$$E = \frac{1}{K - \beta \frac{dT}{dP}} \leftarrow \text{We needed to do this because } \partial \rho / \partial P \rightarrow 0 \text{ and would have predicted } C \rightarrow \infty.$$

For liquids & solids  $dT/dP \approx 0$ . So speed in solid and ~~g~~ liquids is then.

$$C = \sqrt{\frac{E}{\rho}} \sim 1400 \text{ m/s Water}$$
$$\sim 6000 \text{ m/s Steel}$$

The point is we can put all values of Velocity on an absolute scale.

$$Ma = \frac{U_\infty}{C} \leftarrow \begin{array}{l} \text{scale "problem" velocity. This} \\ \text{is the velocities our problem} \\ \text{will be around.} \end{array}$$

$\nwarrow$  an "upperbound" to problem velocity. If  $Ma > 1$  you're travelling waves in the media!?!?