

EQUATION SHEET

Tuesday, October 18, 2022 1:01 AM

- $\mu_{\text{AIR}} = 1.81 \times 10^{-5} [\text{Pa} \cdot \text{s}] @ 15^\circ\text{C}$ | $\rho_{\text{AIR}} = 1.2 [\text{kg}/\text{m}^3] @ 20^\circ\text{C}$
- $\mu_{\text{WATER}} = 1.00 \times 10^{-3} [\text{Pa} \cdot \text{s}] @ 20^\circ\text{C}$ | $\rho_{\text{WATER}} = 1000 [\text{kg}/\text{m}^3]$
- $\gamma_{\text{SHEAR STRESS}} = \mu \left(\frac{dy}{dy} \right)$

SUTHERLAND'S LAW

$$\frac{\mu_2}{\mu_{\text{REF}}} = \left(\frac{T_2}{T_{\text{REF}}} \right)^{3/2} \cdot \left(\frac{T_{\text{REF}} + S}{T_2 + S} \right)$$

- $\mu_{\text{REF}} = 1.716 \times 10^{-5} [\text{Pa} \cdot \text{s}] @ T_{\text{REF}} = 273.15 [\text{K}]$
- $S = 110.9 [\text{K}]$

SPECIFIC HEAT CAPACITIES

$$C_{P\text{AIR}} = 1005 [\text{J}/\text{kg}/\text{K}] \quad | \quad C_{P\text{STEAM}} = 1.87 [\text{kJ}/\text{kg}/\text{K}]$$

$$C_{V\text{AIR}} = 718 [\text{J}/\text{kg}/\text{K}] \quad | \quad C_{V\text{STEAM}} = 1.41 [\text{kJ}/\text{kg}/\text{K}]$$

$$C_{P\text{WATER}} = 4.18 [\text{J}/\text{kg}/\text{K}]$$

- INCOMPRESSIBLE FLOW: $\frac{\rho}{\rho_{\text{REF}}} < 4\% - 5\% \rightarrow C_p = 1 \text{ FOR INCOMP. @ STAGNATION POINT}$

GIBBS'S PHASE RULE: $F = C + 2 - \pi$

- F: # OF INDEPENDENT STATE VARIABLES
- P: # OF PHASES PRESENT (SOLID/LIQUID/GAS)
- C: # OF COMPONENTS (DIFFERENT FLUIDS) $\rightarrow \text{AIR HAS 2 PRIMARY COMPONENTS}$

STATE EQUATIONS (ONLY APPLICABLE @ STATES)

- IDEAL GAS

$$\rightarrow P\forall = nRT, \quad n = \# \text{ OF MOLES}, \quad R_{\text{GAS}} = 8.314 [\text{J}/\text{MOLE}/\text{K}]$$

$$\rightarrow P\forall = mRT, \quad m = \text{MASS}, \quad R = 287 [\text{kJ}/\text{kg}/\text{K}]$$

$$P = \rho RT$$

$$\rightarrow R = \frac{R_{\text{GAS}}}{(\text{MOLAR WEIGHT})}$$

$$\cdot \text{MOLAR WEIGHT} = \sum [(\% \text{ MAKE UP}) \times (\text{INDIVIDUAL MOLAR WEIGHT})]$$

SPECIFIC HEAT CAPACITIES

$$\text{- RATIOS } \gamma = \frac{C_P}{C_V}$$

$$\rightarrow \text{MONOTOMIC GAS: } \gamma = \frac{5}{3} = 1.67$$

$$\rightarrow \text{DIATOMIC GAS: } \gamma = \frac{7}{5} = 1.4$$

$$\rightarrow \text{TRIATOMIC GAS: } \gamma \approx 1.3$$

ENTHALPY: $h = u + P\forall$

$$\text{- } h = C_P T, \quad u = C_V T, \quad P\forall = R_{\text{GAS}} T$$

$$\text{- } C_P - C_V = R_{\text{GAS}} \text{ (ONLY FOR IDEAL GAS)}$$

DIMENSIONLESS NUMBERS

$$\text{- MACH NUMBER: } M = \frac{U}{C}$$

$$\rightarrow U: \text{LOCAL FLOW VELOCITY}$$

$$\rightarrow C: \text{SPEED OF SOUND IN MEDIUM} = a$$

→ U : LOCAL FLOW VELOCITY

→ C : SPEED OF SOUND IN MEDIUM = a

- REYNOLDS NUMBER: $Re = \frac{\rho U L}{\mu}$

→ ρ : DENSITY OF THE FLUID

→ L : CHARACTERISTIC LINEAR DIMENSION

→ μ : VISCOSITY

- SPEED OF SOUND: $a = \sqrt{\gamma R T} = 340 \text{ m/s}$

→ $\gamma = \frac{C_p}{C_v}$

→ $C_p - C_v = R_{\text{gas}}$

$$C_p = \frac{(P_0 - P_\infty)}{\frac{1}{2} \rho U^2}$$

- KNUDSEN NUMBER: $K_n = \frac{\lambda}{L}$

→ MEAN FREE PATH LENGTH: $\lambda = \frac{\mu \sqrt{\pi}}{\rho \sqrt{2} R T}$

→ RELATES MEAN DISTANCE TRAVELED BY MOLECULES TO AIRCRAFT LENGTH

- FROUDE NUMBER: $\frac{U}{\sqrt{g H}}$ → KINETIC E → POTENTIAL E

MOMENTUM

- FINAL MOMENTUM: $-mU$

- CHANGE IN MOMENTUM: $2mU$

- TIME BETWEEN COLLISIONS: $\frac{2L}{U}$

$$F = \frac{2mU}{\frac{2L}{U}} = \frac{mU^2}{L}$$

$$\hookrightarrow P = \frac{F}{L^2} = \frac{mU^2}{L^3}$$

- $E_{\text{TRANS}} = m C_{v, \text{TRAN}} \Delta T$

$$\hookrightarrow C_{v, \text{TRAN}} = \frac{3}{2} R$$

- PRESSURE: $P_v = RT$

- ATMOSPHERIC PRESSURE: $dP = -\rho g dz$

$$\rightarrow P = \int_{z_1}^{z_2} -\rho g dz$$

$$\rightarrow P = \rho g h$$

- PRESSURE READINGS

→ ABSOLUTE (P_{ABS}): $P_{\text{ABS}} = \rho g h$

→ GAUGE (P_{GAUGE}): $P_{\text{GAUGE}} = P_{\text{GAS}} - P_{\text{ATM}}$

→ VACUUM (P_{VAC}): $P_{\text{VAC}} = P_{\text{AMB}} - P_{\text{GAS}}$

→ U-TUBE:

$$\cdot h = \frac{P}{\rho g}$$

$$\cdot P_{\text{GAS}} = \rho g h + P_{\text{REF}}$$

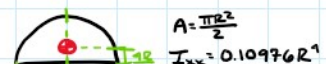
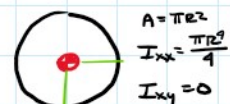
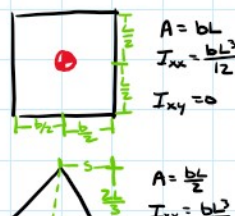
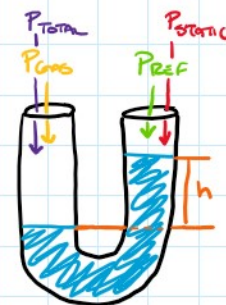
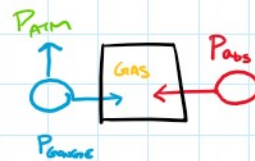
$$\cdot P_{\text{TOTAL}} - P_{\text{STATIC}} = \frac{1}{2} \rho U^2 = \rho g h$$

- PRESSURE FORCES: $F = P \cdot A$

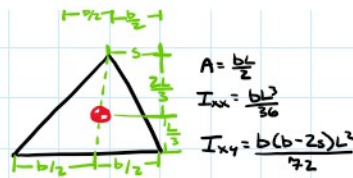
- HYDROSTATIC PRESSURE: $\rho g h$

$$\rightarrow y_{cp} = -y_{swb} \frac{I_{xx}}{\rho_{\text{cen}} A}$$

$$\rightarrow x_{cp} = -y_{swb} \frac{I_{xy}}{\rho_{\text{cen}} A}$$



$$\begin{aligned} \rightarrow y_{cp} &= -y_{swb} \frac{I_{xy}}{\rho_{cp} A} \\ \rightarrow x_{cp} &= -y_{swb} \theta \frac{I_{xy}}{\rho_{cp} A} \\ \rightarrow \sum F &= \int \rho g h dV \\ \rightarrow \sum M &= 0 \end{aligned}$$

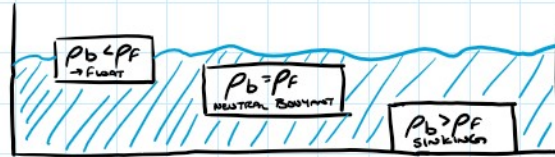


$$\begin{aligned} A &= \frac{bL}{2} \\ I_{xx} &= \frac{bL^3}{36} \\ I_{xy} &= \frac{b(b-z_s)L^2}{72} \\ A &= \frac{\pi R^2}{2} \\ I_{xx} &= 0.10976 R^4 \\ I_{xy} &= 0 \end{aligned}$$

- BOUANCY: $F_B = \rho_f g V_{\text{submerged}}$

$$\rightarrow \rho_f g V_{\text{sub}} = \rho_b g V$$

$$\hookrightarrow \frac{V_{\text{sub}}}{V} = \frac{\rho_b}{\rho_f}$$



\rightarrow GASES LIGHTER THAN AIR: HYDROGEN, HELIUM, AIR

- DYNAMIC PRESSURE $= \frac{1}{2} \rho U^2$

- TOTAL PRESSURE OF FLOW: $P_0 = P + \frac{1}{2} \rho U^2 + \rho g z$

$\rightarrow P_0 = P_{\text{tot}}$: TOTAL PRESSURE

$\rightarrow P$: STATIC PRESSURE, HAS TO BE MEASURED

• THERMODYNAMIC PROCESS: $du = \int Q - \int W$

- du : INTERNAL ENERGY OF THE FLUID (TEMP.)

- $\int Q$: HEAT ADDITION/REMOVAL

- $\int W$: WORK ADDITION/REMOVAL

- WORK FLOW: $W = \int_{V_1}^{V_2} P dV = \int_{P_1}^{P_2} V dP = d(PV) = P dV + V dP$

$\rightarrow W = F \cdot \Delta x$

• NOZZEL FLOW: $F = (\rho_2 A_2 U_2^2 - \rho_1 A_1 U_1^2) + (P_1 A_1 - P_2 A_2)$

- $\sum F = 0 = \dot{m} V + (P_1 A_1 - P_2 A_2)$

$\rightarrow \sum F = (\rho_1 A_1 U_1) U_1 - (\rho_2 A_2 U_2) U_2 + (P_1 A_1 - P_2 A_2)$

• ENTHALPY

- $H = U + PV$ [kJ]

$\rightarrow U$: INTERNAL ENERGY

$\rightarrow PV$: FLOW WORK

- $h = u + Pv = C_p T$ [kJ/kg]

• TOTAL ENERGY

- KINETIC ENERGY: $\frac{1}{2} U^2$ [kJ/kg]

- POTENTIAL ENERGY: gh [kJ/kg]

- INTERNAL ENERGY: $h = u + Pv$ [kJ/kg]

- FLOW WORK:

$\rightarrow h_0 = h + \frac{1}{2} U^2 + gz$

$\rightarrow C_p T_0 = C_p T + \frac{1}{2} U^2 + gz$

- TRANSLATIONAL KINETIC ENERGY $\propto \frac{P}{\rho} = \frac{1}{3} (U^2 + V^2 + W^2) = C^2$: MEAN SQUARED SPEED

• ISENTROPIC PROCESS

- 1ST LAW:

\rightarrow CLOSED SYSTEM: $du = \int q - \int w$

\rightarrow OPEN SYSTEM: $dh_0 = \int q - \int W_{\text{shaft}}$

→ CLOSED SYSTEM: $du = \delta q - \delta w$

→ OPEN SYSTEM: $dh_0 = \delta q - \delta w_{\text{shaft}}$

$$- \frac{T_0}{T} = 1 + \frac{(\gamma-1)}{2} M^2 \quad (M = \text{MACH NUMBER})$$

- ADIABATIC: $\delta q = 0$ (ENSURED)

→ CONSTANT PRESSURE, EXPANSION/COMPRESSION

$$\bullet P_V = \text{CONSTANT}$$

- COMPRESSIBLE ($\frac{P}{P_{\text{ref}}} = 1\% - 5\%$) (CONSEQUENCE)

- REVERSIBLE ($\delta s = 0$) (MEASURED)

→ CHANGE IN ENTROPY: $s = \frac{q}{T}$ [J/kgK]

$$\rightarrow Tds = dh - vdp$$

$$\rightarrow \Delta s = C_p \ln\left(\frac{T_2}{T_1}\right) - R \ln\left(\frac{P_2}{P_1}\right)$$

• ICE → STEAM

- PHASE CHANGE: $q = m H$ [J]

→ ENTHALPY FUSION: $H_{\text{fus}} = 333$ ENTHALPY VAPORIZATION: $H_{\text{vap}} = 2257$ [J/g]

- TEMP. CHANGE: $q = m C (T_2 - T_1)$ [J]

- OPEN SYSTEM (CONSTANT PRESSURE)

$$\rightarrow C_{P_{\text{WAT}}} = 4.18, C_{P_{\text{STEAM}}} = 1.87 \text{ [J/g]}$$

- CLOSED SYSTEM (CONSTANT VOLUME)

$$\rightarrow C_{V_{\text{WAT}}} = 4.18, C_{V_{\text{STEAM}}} = 1.41 \text{ [J/g]}$$