(G- AI) S=0 Pliggin gives

G= 1+452 ksi 5, = 1/1 in 4z - xx +x =0 4 Z - yx + x =0 4x + 4y - Zx + Z =0 5z = 1 usi $5z = \frac{\sqrt{2}}{2}$ | in $G_{3}=1-4\sqrt{2}$ Ksi $S_{3}=\frac{1}{2}\begin{pmatrix} -1\\ -1\\ 10\end{pmatrix}$ $\mathcal{G}_{e} = \left(\frac{3}{2}\sum_{\beta=1}^{2}\sum_{\alpha=x}^{z}\left(6\sec^{2}\alpha_{\beta}\right)\right)^{2} = \sqrt{\frac{3}{2}}\left(4\right)\left(4^{2}\right) = \sqrt{96}\left(65\right)^{2}$ 0y = 15.6 Ksi, F= 1.5 tresca, yeild if Filmax > 5x Vmax = = = (5, -53) = 4/2 F 7mx = 6 /2 = 8.4853 USI 57 = 7.4 So tresca predicts yeiling

331 HW Z

Von mises yeild if Ge 715y

Ge ≈ 9.198 lesi

So von mises prodicts there will not be yeild

Problem 2 - see motlab

housecleaning

```
clear all
close all
clc
% sympref('FloatingPointOutput',true)
% sympref('FloatingPointOutput',false)
addpath('C:\joshFunctionsMatlab\')
```

Problem 2

```
clear
% Torque
syms T
assume(1<T)
% givens
p = 100/1000; % ksi
r = 25;
t = .1;
A = pi*r^2;
% stress state
sigHoop = p*r/t;
sigAxial = p*r/(2*t);
tau = T/(2*t*A);
sig0 = [[sigAxial, tau, 0]; [tau, sigHoop, 0]; [0, 0, 0]];
% stress state in principle reference frame
[S, sig0] = eig(sig0);
temp = [sig0(1,1), sig0(2,2), sig0(3,3)];
[temp, I] = sort(temp, 'descend');
sig0 = [[temp(1), 0, 0]; [0, temp(2), 0]; [0, 0, temp(3)]];
S = [S(:,I(1)),S(:,I(2)),S(:,I(3))];
clear temp I
% hydrostatic, deviatoric, max shear, effective stress
sig h = (1/3) *trace(sig0);
sig dev = sig0 - sig h*eye(3);
sig e = ((3/2) * sum(sig dev.^2)))^(1/2);
tau max = (1/2)*(sig0(1,1)-sig0(3,3));
% solve for T
sig y = 30; % ksi
eqn1 = tau max == sig y/2;
eqn2 = sig e == sig y;
```

```
sol1 = solve(eqn1,T);
sol2 = solve(eqn2,T);
temp = [sig0(1,1), sig0(2,2), sig0(3,3)];
disp('The principle stresses are in ksi are: ')
disp(temp')
disp('While using the tresca yeild criterion, T can be as high as
 '+string(sol1)+' = '+string(vpa(sol1,5))+' kip-in.')
disp('While using the von mises yeild criterion, T can be as high as
 '+string(sol2)+' = '+string(vpa(sol2,5))+' kip-in.')
disp('We can conclude that tresca yeild condition is more conservative than
von mises yeild condition because ')
The principle stresses are in ksi are:
(9375*pi + (16*T^2 + 9765625*pi^2)^(1/2))/(500*pi)
(9375*pi - (16*T^2 + 9765625*pi^2)^(1/2))/(500*pi)
While using the tresca yeild criterion, T can be as high as
 (625*14^{(1/2)*pi})/2 = 3673.4 \text{ kip-in.}
While using the von mises yeild criterion, T can be as high as
 (625*23^{(1/2)*pi})/2 = 4708.3 \text{ kip-in.}
We can conclude that tresca yeild condition is more conservative than von
mises yeild condition because the yield torque is lower for the tresca critirion than
The von mises
```

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