

# Two-mode squeezed states in cavity optomechanics via engineering of a single reservoir

PhD course - Quantum coherent phenomena  
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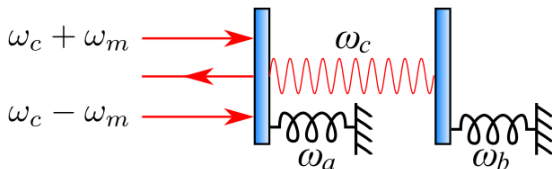
# Outline

- ① Introduction, system and Hamiltonian
- ② Implementation strategies
- ③ Observable quantities
- ④ Experimental observability
- ⑤ Conclusions

# Introduction

## System representation

- Two mechanical oscillators with resonance frequencies  $\omega_a, \omega_b$
- Dispersively coupled with rates  $g_a, g_b$  to a common cavity  $\omega_c$
- Apply radiation pressure forces inside the cavity leading to entangled motion of the mirrors become



# Introduction

## System and Hamiltonian

Quantum optomechanics  $\rightarrow$  describing optical and mechanical modes with same formalism

$$\begin{aligned}\hat{\mathcal{H}} = & \omega_a \hat{a}^\dagger \hat{a} + \omega_b \hat{b}^\dagger \hat{b} + \omega_c \hat{c}^\dagger \hat{c} + g_a (\hat{a} + \hat{a}^\dagger) \hat{c}^\dagger \hat{c} \\ & + g_b (\hat{b} + \hat{b}^\dagger) \hat{c}^\dagger \hat{c} + \hat{H}_{\text{drive}} + \hat{H}_{\text{diss}},\end{aligned}$$

under usual approximations, obtain the master equation

$$\begin{aligned}\dot{\rho} = & -i[\hat{\mathcal{H}}', \rho] + \gamma_a (\bar{n}_a + 1) \mathcal{D}[\hat{a}] \rho + \gamma_a \bar{n}_a \mathcal{D}[\hat{a}^\dagger] \rho \\ & + \gamma_b (\bar{n}_b + 1) \mathcal{D}[\hat{b}] \rho + \gamma_b \bar{n}_b \mathcal{D}[\hat{b}^\dagger] \rho + \kappa \mathcal{D}[\hat{c}] \rho,\end{aligned}$$

Being  $\mathcal{H}' = \mathcal{H} - \mathcal{H}_{\text{diss}}$ , and  $\mathcal{D}[\hat{c}]$  the dispersive superoperator

Only dissipation term for  $\hat{c} \rightarrow$  Assuming zero thermal occupation

# Reservoir engineering strategies

## Bogoliubov operators

Define the Bogoliubov mechanical modes in terms of the modes  $\hat{a}, \hat{b}$

$$\begin{aligned}\hat{\beta}_1 &= \hat{a} \cosh r + \hat{b}^\dagger \sinh r, \\ \hat{\beta}_2 &= \hat{b} \cosh r + \hat{a}^\dagger \sinh r.\end{aligned}$$

Where  $r$  is the squeezing parameter.

Work in rotating frame.

# Reservoir engineering strategies

## Hamiltonian

Hamiltonian in terms of the Bogoliubov modes

$$\hat{\mathcal{H}} = \Omega(\hat{\beta}_1^\dagger \hat{\beta}_1 - \hat{\beta}_2^\dagger \hat{\beta}_2) + \mathcal{G}[(\hat{\beta}_1^\dagger + \hat{\beta}_2^\dagger)\hat{c} + \text{H.c.}] + \hat{H}_{\text{diss}},$$

where  $\Omega$  is the effective oscillation frequency and  $\mathcal{G}$  an effective optomechanical coupling.

Written in terms of the original operators,

$$\begin{aligned}\hat{\mathcal{H}} = & \Omega(\hat{a}^\dagger \hat{a} - \hat{b}^\dagger \hat{b}) + G_+[(\hat{a} + \hat{b})\hat{c} + \text{H.c.}] \\ & + G_-[(\hat{a} + \hat{b})\hat{c}^\dagger + \text{H.c.}] + \hat{H}_{\text{diss}}.\end{aligned}$$

with couplings related by  $\mathcal{G} \equiv \sqrt{G_-^2 - G_+^2}$  and  $\tanh r \equiv G_+/G_-$

# Reservoir engineering strategies

## 2-mode squeezed state

Define the **2-mode squeezed state** as  $|r\rangle_2 = \hat{S}_2(r) |0, 0\rangle$ , being the squeezing operator

$$\hat{S}_2(r) \equiv \exp[r(\hat{a}\hat{b} - \hat{a}^\dagger\hat{b}^\dagger)]$$

such that  $[\hat{S}_2(r)\hat{a}\hat{S}_2^\dagger(r)]|r\rangle_2 = [\hat{S}_2(r)\hat{b}\hat{S}_2^\dagger(r)]|r\rangle_2 = 0$

Therefore,  $\hat{\beta}_1 = \hat{S}_2(r)\hat{a}\hat{S}_2^\dagger(r)$ ,  $\hat{\beta}_2 = \hat{S}_2(r)\hat{b}\hat{S}_2^\dagger(r)$  and their ground state is the two-mode squeezed state with squeezing parameter  $r$

# Reservoir engineering strategies

## Note on Quantum Optomechanics

Linearized Hamiltonian with 2-tone laser with amplitudes  $\alpha_+$  and  $\alpha_-$

$$\mathcal{H} = \hbar g_+ (a^\dagger b^\dagger + ab) + \hbar g_- (a^\dagger b + ab^\dagger)$$

being  $g_\pm = g_0 \alpha_\pm$

Study different cases

- $g_- = 0 \longrightarrow$  Sideband blue  $\mathcal{H} = \hbar g (a^\dagger b^\dagger + ab)$  "2 - mode squeezing"
- $g_+ = 0 \longrightarrow$  Sideband red  $\mathcal{H} = \hbar g (a^\dagger b + ab^\dagger)$  "beam - splitter"
- $g_- = g_+ = g \longrightarrow \mathcal{H} = \hbar g (a + a^\dagger)(b + b^\dagger)$  "back-action evading"



# Implementation

## Different cases

$$\hat{\mathcal{H}} = \Omega(\hat{a}^\dagger \hat{a} - \hat{b}^\dagger \hat{b}) + G_+[(\hat{a} + \hat{b})\hat{c} + \text{H.c.}] \\ + G_-[(\hat{a} + \hat{b})\hat{c}^\dagger + \text{H.c.}] + \hat{H}_{\text{diss}}.$$

already implemented in conventional optomechanical setups.

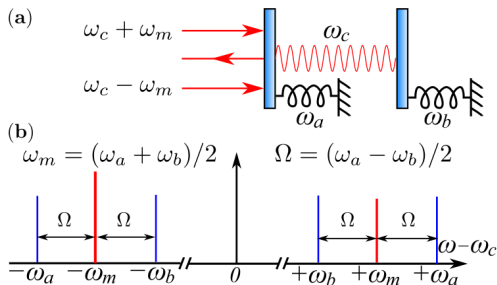
Different cases depending on the optomechanical couplings relation:

- Two-tone driving ( $g_a = g_b$ )  $\longrightarrow$  two cavity drives are required
- Four-tone driving ( $g_a \neq g_b$ )  $\longrightarrow$  four cavity drives are required
- Case similar ( $g_a \sim g_b$ )  $\longrightarrow$  approximate with two cavity drives

# Implementation

2 - tone driving ( $g_a = g_b$ )

Driving tones at  $\omega_c \pm \omega_m$  being  $\omega_m = (\omega_a + \omega_b)/2$



Apply our drive Hamiltonian

$$\hat{H}_{\text{drive}} = (\mathcal{E}_+^* e^{+i\omega_m t} + \mathcal{E}_-^* e^{-i\omega_m t}) e^{+i\omega_c t} \hat{c} + \text{H.c.}$$

# Implementation

2 - tone driving ( $g_a = g_b$ )

Interaction picture with respect to  $\hat{\mathcal{H}}_0 = \omega_m(\hat{a}^\dagger \hat{a} + \hat{b}^\dagger \hat{b}) + \omega_c \hat{c}^\dagger \hat{c}$  leads back to our desired Hamiltonian

$$\begin{aligned}\hat{\mathcal{H}} = & \Omega(\hat{a}^\dagger \hat{a} - \hat{b}^\dagger \hat{b}) + G_+[(\hat{a} + \hat{b})\hat{c} + \text{H.c.}] \\ & + G_-[(\hat{a} + \hat{b})\hat{c}^\dagger + \text{H.c.}] + \hat{H}_{\text{diss}}.\end{aligned}$$

$$\begin{aligned}\text{where} \quad \Omega &= (\omega_a - \omega_b)/2 \\ G_{\pm} &= (g_a + g_b)\bar{c}_{\pm}/2\end{aligned}$$

and  $\bar{c}_{\pm}$  are the steady state amplitudes at the sidebands

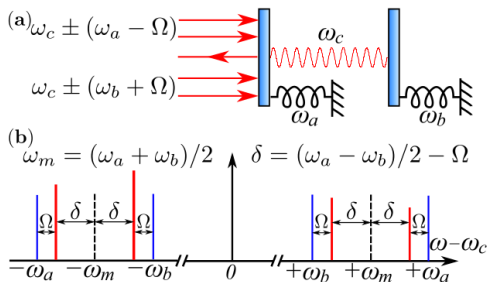
$$\bar{c}_{\pm} \equiv \langle \hat{c}_{\pm} \rangle_{\text{ss}} = \frac{i\mathcal{E}_{\pm}}{\pm i\omega_m - \kappa/2}.$$

# Implementation

4 - tone driving ( $g_a \neq g_b$ )

Four different sideband processes involved

Driving tones applied with detuning of  $\Omega$  from the sidebands at  $\omega_c \pm (\omega_a - \Omega)$  and  $\omega_c \pm (\omega_b + \Omega)$



# Implementation

4 - tone driving ( $g_a \neq g_b$ )

$$\hat{H}_{\text{drive}} = e^{+i\omega_c t} \hat{c}(\mathcal{E}_{1+}^* e^{+i(\omega_a - \Omega)t} + \mathcal{E}_{2+}^* e^{+i(\omega_b + \Omega)t} + \mathcal{E}_{1-}^* e^{-i(\omega_a - \Omega)t} + \mathcal{E}_{2-}^* e^{-i(\omega_b + \Omega)t}) + \text{H.c.}$$

steady-state amplitudes are now

$$\bar{c}_{k\pm} \equiv \langle \hat{c}_{k\pm} \rangle_{\text{ss}} = \frac{i\mathcal{E}_{k\pm}}{\pm i\omega_k - \kappa/2},$$

where we introduced notation for the drive detunings

$$\omega_1 = (\omega_a - \Omega)$$

$$\omega_2 = (\omega_b + \Omega)$$

$$G_{\pm} = (g_a \bar{c}_{1\pm} + g_b \bar{c}_{2\pm})/2$$

# Adiabatic limit

## Adiabatically eliminated master equation

- Assume the system responds fast to mechanical motion ( $k > \Omega$ ,  $G_{\pm}$ )
- Simplify by getting rid of the cavity operator  $\hat{c} = -2i\mathcal{G}(\hat{\beta}_1 + \hat{\beta}_2)/k$
- Obtain adiabatically eliminated master equation

$$\begin{aligned}\dot{\rho} = & -i\Omega[\hat{\beta}_1^\dagger\hat{\beta}_1 - \hat{\beta}_2^\dagger\hat{\beta}_2, \rho] + \gamma_a(\bar{n}_a + 1)\mathcal{D}[\hat{a}]\rho + \gamma_a\bar{n}_a\mathcal{D}[\hat{a}^\dagger]\rho \\ & + \gamma_b(\bar{n}_b + 1)\mathcal{D}[\hat{b}]\rho + \gamma_b\bar{n}_b\mathcal{D}[\hat{b}^\dagger]\rho + \Gamma\mathcal{D}[\hat{\beta}_1 + \hat{\beta}_2]\rho,\end{aligned}$$

with optomechanical damping rate

$$\Gamma \equiv \gamma\mathcal{C} \equiv \frac{4\mathcal{G}^2}{\kappa},$$

Easy to obtain steady state, and to measure entanglement and purity.

# Adiabatic limit

## Entanglement

Build a way of identify entanglement on a 2-mode system

**Duan criterion**  $\longrightarrow$  define collective quadratures

$$\hat{X}_{\pm} = (\hat{X}_a \pm \hat{X}_b)/\sqrt{2},$$

$$\hat{P}_{\pm} = (\hat{P}_a \pm \hat{P}_b)/\sqrt{2},$$

as combination of the usual quadrature modes

$$\hat{X}_s = (\hat{s} + \hat{s}^{\dagger})/\sqrt{2}, \quad \hat{P}_s = -i(\hat{s} - \hat{s}^{\dagger})/\sqrt{2}.$$

Duan inequality states that a state for which

$$\langle \hat{X}_+^2 \rangle + \langle \hat{P}_-^2 \rangle < 1$$

is inseparable  $\longrightarrow$  **entangled!**

# Adiabatic limit

## Entanglement

Quadratures can be written as function of the drive asymmetry

$$\begin{aligned}\langle \hat{X}_{\pm}^2 \rangle &= \langle \hat{P}_{\mp}^2 \rangle = \frac{\gamma}{\gamma + \Gamma} (\bar{n} + 1/2) + \frac{\Gamma}{\gamma + \Gamma} \frac{e^{\mp 2r}}{2} \\ &= \frac{\gamma \kappa}{\gamma \kappa + 4(G_-^2 - G_+^2)} (\bar{n} + 1/2) \\ &\quad + \frac{2(G_- \mp G_+)^2}{\gamma \kappa + 4(G_-^2 - G_+^2)}.\end{aligned}$$

Use also logarithmic negativity  $E_{\mathcal{N}} = \max\{0, -\ln 2\eta\}$ , with  $\eta$  factor in terms of the covariance matrix \*



# Adiabatic limit

## Purity

Purity defined as trace of the density matrix

$$\mu \equiv \text{tr}[\rho^2]$$

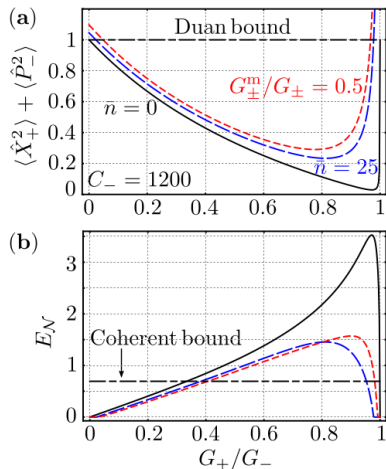
and as function of the covariance matrix \*

$$\mu = \frac{1}{4\sqrt{\det \mathbf{V}}}$$

Again, demanding from experimental point of view.

# Adiabatic limit

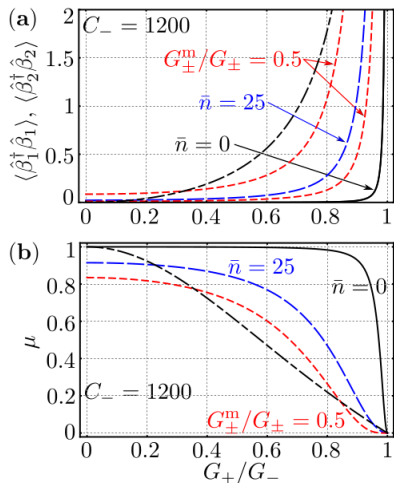
## Entanglement



- Duan quantity and logarithmic negativity as function of the drive asymmetry
- Solid curve with mechanical thermal occupation  $\bar{n} = 0$  and no imperfections on effective coupling
- Thermal occupation and imperfections in the effective coupling lead to less entanglement

# Adiabatic limit

## Entanglement



- Steady state occupations and purity as function of the drive asymmetry
- Solid curve with mechanical thermal occupation  $\bar{n} = 0$  and no imperfections on effective coupling
- Thermal occupation and imperfections in the effective coupling lead to degradation of purity

# Experimental observability

## Output spectrum

- Reconstructing covariance matrix is experimentally demanding
- Directly measuring quadratures is a hard problem
- Seek signature of entanglement in output spectrum

Spectrum as Fourier transform of expected value

$$S[\omega] = \int dt e^{i\omega t} \langle \delta \hat{c}_{\text{out}}^\dagger(t) \delta \hat{c}_{\text{out}}(0) \rangle,$$

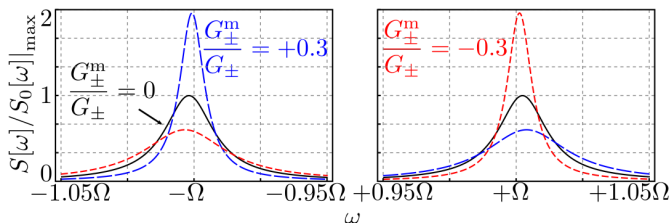
being  $\delta \hat{c}_{\text{out}} = \hat{c}_{\text{out}} - \langle \hat{c}_{\text{out}} \rangle$

Spectrum can be related to the occupation of modes

$$\begin{aligned} \int_{-\infty}^0 S[\omega] d\omega &= \int_0^{+\infty} S[\omega] d\omega \\ &= 8\pi\kappa \frac{\mathcal{G}^2}{4\mathcal{G}^2 + \kappa(\kappa + \gamma)} \langle \hat{\beta}_i^\dagger \hat{\beta}_i \rangle, \end{aligned}$$

# Experimental observability

## Output spectrum



- Output spectrum entered around the detunings from the cavity resonance frequency
- Solid black curve without imperfections
- Steady-state mechanical entanglement can be bounded based on a measurement of the output spectrum
- Experimental work realized in "[Stabilized entanglement of massive mechanical oscillators](#)", Nature, 2018.

# Conclusions

- ① Three-mode optomechanical system such as the steady state includes highly pure and highly entangled two-mode squeezed state is built.
- ② Ways of describing both entanglement and purity are described as function of the drive asymmetry.
- ③ Problem of unequal single-photon optomechanical couplings solved by using four-tone driving scheme.
- ④ Proposal implementable for existing technology.

Thank you!



# Back up

## Bogoliubov operators

Define the **Bogoliubov** mechanical modes in terms of the modes  $\hat{a}, \hat{b}$

$$\begin{aligned}\hat{\beta}_1 &= \hat{a} \cosh r + \hat{b}^\dagger \sinh r, \\ \hat{\beta}_2 &= \hat{b} \cosh r + \hat{a}^\dagger \sinh r.\end{aligned}$$

Where  $r$  is the **squeezing parameter**.

Work in rotating frame with respect to the Hamiltonian:

$$\hat{H}_0 = (\omega_a - \Omega)\hat{a}^\dagger\hat{a} + (\omega_b + \Omega)\hat{b}^\dagger\hat{b} + \omega_c\hat{c}^\dagger\hat{c},$$

where choice of detuning  $\Omega$  is such that collective mechanical quadratures  $\hat{X}_\pm, \hat{P}_\pm$  (defined later) rotate in a non-trivial way.



# Back up

## Generate the 2-mode squeezed state

- i) Two cavity modes to independently cool the Bogoliubov modes (beam splitter  $\hat{\beta}_i^\dagger \hat{c}_i$ )
- ii) Couple the cavity to one Bogoliubov mode ( $\hat{\beta}_1$ ), and then this one to  $\hat{\beta}_2$  through ( $\hat{\beta}_1^\dagger \hat{\beta}_2$ )
- iii) Couple the cavity to sum of the Bogoliubov modes, then the sum to the difference (swap interaction  $\hat{\beta}_{\text{sum}}^\dagger \hat{\beta}_{\text{diff}}$  allows diff to cool).

$$\hat{\beta}_{\text{sum}} = \frac{1}{\sqrt{2}}(\hat{\beta}_1 + \hat{\beta}_2)$$
$$\hat{\beta}_{\text{diff}} = \frac{1}{\sqrt{2}}(\hat{\beta}_1 - \hat{\beta}_2)$$

Cooling  $\hat{\beta}_{\text{sum}}$  and  $\hat{\beta}_{\text{diff}}$  is equivalent to cool  $\hat{\beta}_1$  and  $\hat{\beta}_2$  ✓

Direct coupling not needed, just difference in their resonance frequencies ✓

# Back up

## 4-tone driving

Where we demand the driving strengths of the 4-tone driving are "matched" as

$$\frac{\bar{c}_{1\pm}}{\bar{c}_{2\pm}} = \frac{g_b}{g_a}$$

meaning, asymmetry in steady-state amplitudes is set by the asymmetry in the optomechanical couplings

# Back up

## Effective coupling imperfections

Where we demand the driving strengths of the 4-tone driving are "matched" as

$$G_{\pm}^m = \pm(g_a - g_b)\bar{c}_{\pm}/2 \quad \text{2-tone driving}$$

$$G_{\pm}^m = \pm(g_a\bar{c}_{1\pm} - g_b\bar{c}_{2\pm})/2 \quad \text{4-tone driving}$$

In the 2-tone driving case, imperfections coming from mismatch in the optomechanical couplings

In the 4-tone driving case, imperfection arises from the drives not being weighted precisely according to the matching condition