

# EE2023 Signals & Systems

## Chapter 9 – Unit Step Responses of LTI Systems

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# Overview : Useful input-output relationships of LTI Systems

Three useful types of input-output mappings are :

- ▶ **Unit Impulse Response**,  $y_{\delta}(t)$  is the output signal when the input is an unit impulse function.
- ▶ **Unit Step Response**,  $y_{step}(t)$  is the output response when the input is a step signal.
- ▶ **Sinusoidal Response**,  $y_{ss}(t)$  is the output response when  $t \rightarrow \infty$  and the input is a sinusoidal signal .

Each of these responses is important to LTI systems because they define certain behaviours that can be generalized for such systems.

- ▶ Unit impulse response characterizes a LTI system in time-domain.
- ▶ Unit step response is very commonly encountered in practice and they give info about some physical parameters in the LTI system.
- ▶ Sinusoidal response is related to the frequency response of the system.

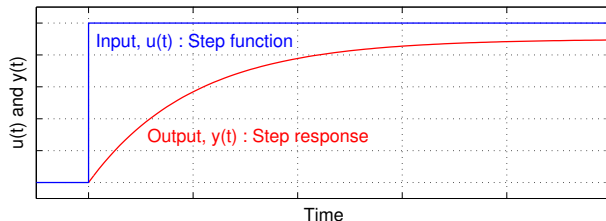
# Unit Step Response : Definition

The **unit step response**,  $y_{step}(t)$ , of a continuous LTI system is defined as the response of the system when the input,  $x(t)$ , is the unit step function,  $U(t)$ .



Step response is a mathematical expression that describes the output when the input is system changed from the current value to a new level. Many daily tasks/actions are examples of step responses e.g.

- ▶ Switching on a fan/kettle
- ▶ F1 driver pressing the pit lane speed limiter
- ▶ Student receiving a piece of CA homework



When the input  $x(t)$  is the unit step function  $u(t)$ , then  $X(s) = \mathcal{L}\{x(t)\} = \mathcal{L}\{u(t)\} = \frac{1}{s}$ .

If the system transfer function is  $G(s)$ , then  $Y_{step}(s) = G(s) \times \frac{1}{s}$ .

$$\text{Unit Step response, } y_{step}(t) = \mathcal{L}^{-1} \left\{ G(s) \cdot \frac{1}{s} \right\}$$

Let  $G(s) = K' \frac{(s + z_1)(s + z_2) \dots (s + z_m)}{(s + p_1)(s + p_2) \dots (s + p_n)}$ , then

$$Y_{step}(s) = K' \frac{(s + z_1)(s + z_2) \dots (s + z_m)}{(s + p_1)(s + p_2) \dots (s + p_n)} \times \frac{1}{s}$$

$$y_{step}(t) = \mathcal{L}^{-1} \left\{ \frac{\alpha_1}{s + p_1} + \dots + \frac{\alpha_n}{s + p_n} + \frac{\beta}{s} \right\} = \left[ \sum_{i=1}^n \alpha_i e^{-p_i t} + \beta \right] u(t)$$

► Note that system poles are  $-p_1, \dots, -p_n$  and  $s = 0$  is an input pole.

► If system is BIBO stable, then  $\sum_{i=1}^n A_i e^{-p_i t} = 0$  when  $t \rightarrow \infty$  and  $\lim_{t \rightarrow \infty} y_{step}(t) = \text{constant}$ .

# Relationship between Unit Step and Unit Impulse Responses

The  $s$ -domain expression for the unit step response of a LTI system,  $G(s)$ , is

$$Y_{step}(s) = G(s) \frac{1}{s}$$

Applying the following concepts to  $Y_{step}(s) = \frac{G(s)}{s}$ ,

► the impulse response,  $\mathcal{L}\{y_{\delta}(t)\} = G(s)$

► Transform of an Integral Property states  $\mathcal{L}\left\{\int_0^t f(\tau) d\tau\right\} = \frac{F(s)}{s}$  where  $\mathcal{L}\{f(t)\} = F(s)$

the relationship between the unit step and unit impulse responses of a LTI system is

$$\begin{aligned} y_{step}(t) &= \int_0^t y_{\delta}(\tau) d\tau \\ y_{\delta}(t) &= \frac{dy_{step}(t)}{dt} \end{aligned}$$

# Unit Step Response – First order system

- Unit impulse response of a first order system is  $y_{\delta}(t) = \frac{K}{T} e^{-\frac{t}{T}} u(t)$ . Hence,

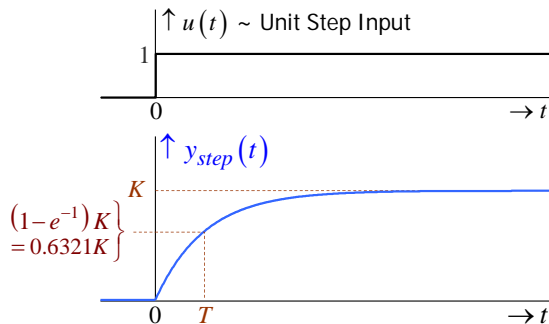
$$y_{step}(t) = \int_0^t y_{\delta}(\tau) d\tau = \int_0^t \frac{K}{T} e^{-\frac{\tau}{T}} u(\tau) d\tau = \left[ -K e^{-\frac{\tau}{T}} \right]_0^t = \left[ K - K e^{-\frac{t}{T}} \right] u(t)$$

- Unit step response of a first order system,

$$G(s) = \frac{K}{Ts + 1} \text{ is:}$$

$$\begin{aligned} Y(s) &= \frac{K}{s(Ts + 1)} \\ &= \frac{K}{s} - \frac{KT}{Ts + 1} \end{aligned}$$

$$y_{step}(t) = \left[ K - K e^{-\frac{t}{T}} \right] u(t)$$



## Behaviour of $y_{\text{step}}(t)$ when $t \rightarrow \infty$

- ▶ Applying Final Value Theorem, “final value” of the unit step response when  $t \rightarrow \infty$  is

$$\lim_{t \rightarrow \infty} y_{\text{step}}(t) = \lim_{s \rightarrow 0} sY_{\text{step}}(s) = \lim_{s \rightarrow 0} G(s) = G(0) = K$$

- ▶ When the input is  $A \cdot u(t)$ , then final value of the step response when  $t \rightarrow \infty$  is

$$\lim_{t \rightarrow \infty} y_{\text{step}}(t) = G(0) \times A = K \cdot A$$

## Relationship between $y_{\text{step}}(t)$ and system pole

- ▶ System pole is  $s = -\frac{1}{T}$ . Hence,  $y_{\text{step}}(t)$  contains a  $\alpha e^{-\frac{t}{T}}$  term.
  - ▶  $\alpha e^{-\frac{t}{T}}$  term decays to zero faster if  $T$  is smaller.
- ▶ When  $t = T$ ,  $y_{\text{step}}(t) = K[1 - e^{-1}] = 0.632K$   
Time constant,  $T$ , can be determined by finding the time taken by the step response to reach 63.2% of the final value.

# Unit Step Response – Second order system

Transfer function of standard second order system is  $\frac{K\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$ .

Hence, unit step response of a second order system may be expressed as

$$y_{step}(t) = \mathcal{L}^{-1} \left\{ \frac{K\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} \times \frac{1}{s} \right\}$$

Initial conditions :  $y_{step}(0) = \dot{y}_{step}(0) = 0$

►  **$y_{step}(t)$  of overdamped ( $\zeta > 1$ ) 2<sup>nd</sup> sys:** Distinct real poles

$$\begin{aligned} y_{step}(t) &= \mathcal{L}^{-1} \left\{ \frac{K\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} \times \frac{1}{s} \right\} \\ &= \mathcal{L}^{-1} \left\{ \frac{K\omega_n^2}{(s+a)(s+b)} \times \frac{1}{s} \right\} \end{aligned}$$

$$a = \zeta\omega_n + \omega_n\sqrt{\zeta^2 - 1} \text{ and } b = \zeta\omega_n - \omega_n\sqrt{\zeta^2 - 1}$$

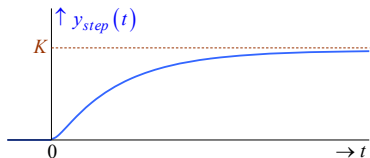
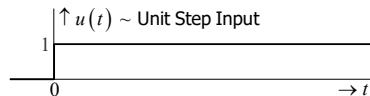


- **$y_{\text{step}}(t)$  of overdamped ( $\zeta > 1$ ) 2<sup>nd</sup> sys:** Distinct real poles (con't)

$$y_{\text{step}}(t) = \mathcal{L}^{-1} \left\{ \frac{K\omega_n^2}{(s+a)(s+b)} \times \frac{1}{s} \right\}$$

$$= \left[ K - \frac{K_1}{a} e^{-at} - \frac{K_2}{b} e^{-bt} \right] u(t)$$

$$K_1 = \frac{K\omega_n^2}{b-a} \text{ and } K_2 = \frac{K\omega_n^2}{a-b}$$

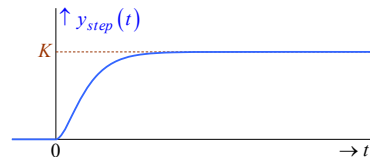


- **$y_{\text{step}}(t)$  of critically damped ( $\zeta = 1$ ) 2<sup>nd</sup> sys:** Repeated real poles

$$y_{\text{step}}(t) = \mathcal{L}^{-1} \left\{ \frac{K\omega_n^2}{(s+\omega_n)^2} \times \frac{1}{s} \right\}$$

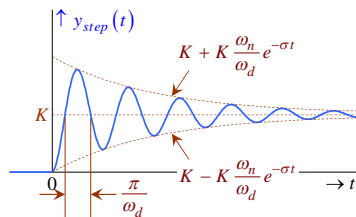
$$= \mathcal{L}^{-1} \left\{ \frac{K}{s} - \frac{K}{s+\omega_n} - \frac{K\omega_n}{(s+\omega_n)^2} \right\}$$

$$= K (1 - e^{-\omega_n t} - \omega_n t e^{-\omega_n t}) u(t)$$



- **$y_{\text{step}}(t)$  of underdamped ( $0 < \zeta < 1$ ) 2<sup>nd</sup> sys:** Complex conjugate poles

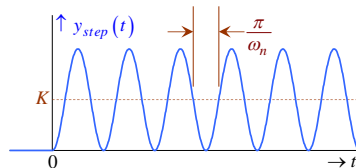
$$\begin{aligned}
 y_{\text{step}}(t) &= \mathcal{L}^{-1} \left\{ \frac{K(\sigma^2 + \omega_d^2)}{[(s + \sigma)^2 + \omega_d^2]} \times \frac{1}{s} \right\} \\
 &= \mathcal{L}^{-1} \left\{ \frac{K}{s} - \frac{K(s + \sigma)}{(s + \sigma)^2 + \omega_d^2} - \frac{K\sigma}{(s + \sigma)^2 + \omega_d^2} \right\}
 \end{aligned}$$



$$y_{\text{step}}(t) = K - Ke^{-\sigma t} \left[ \cos(\omega_d t) + \frac{\sigma}{\omega_d} \sin(\omega_d t) \right] u(t) = K \left[ 1 - \frac{\omega_n}{\omega_d} e^{-\sigma t} \sin(\omega_d t + \phi) \right] u(t)$$

- **$y_{\text{step}}(t)$  of undamped ( $\zeta = 0$ ) 2<sup>nd</sup> sys:** Imaginary conjugate poles

$$\begin{aligned}
 y_{\text{step}}(t) &= \mathcal{L}^{-1} \left\{ \frac{K\omega_n^2}{s^2 + \omega_n^2} \times \frac{1}{s} \right\} = \mathcal{L}^{-1} \left\{ \frac{K}{s} - \frac{Ks}{s^2 + \omega_n^2} \right\} \\
 &= K(1 - \cos \omega_n t) u(t)
 \end{aligned}$$

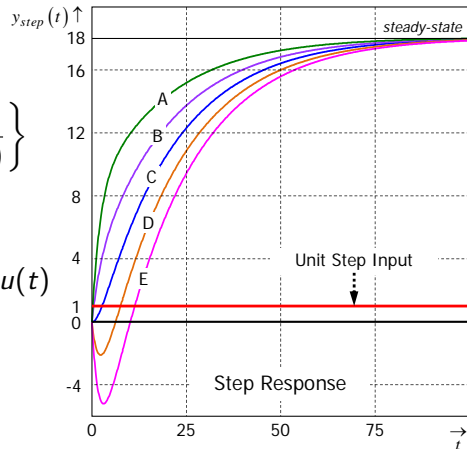


# Unit Step Response – Effect of an Additional Zero

Consider a system  $G(s)$  which has a DC gain of 18, a zero at  $s = -\frac{1}{\gamma}$ , and poles  $s = -0.5$  and  $s = -0.05$ . Derive the unit step response.

$$\begin{aligned}
 G(s) &= \frac{18(\gamma s + 1)}{40s^2 + 22s + 1} \\
 y_{step}(t) &= \mathcal{L}^{-1}\left\{\frac{G(s)}{s}\right\} = \mathcal{L}^{-1}\left\{\frac{18(\gamma s + 1)}{s(40s^2 + 22s + 1)}\right\} \\
 &= \mathcal{L}^{-1}\left\{\frac{18}{s} + \frac{2 - \gamma}{s + 0.5} - \frac{20 - \gamma}{s + 0.05}\right\} \\
 &= [18 + (2 - \gamma)e^{-0.5t} - (20 - \gamma)e^{-0.05t}] u(t)
 \end{aligned}$$

- ▶ Location of system zero does not affect the “final” value.
- ▶  $\lim_{t \rightarrow \infty} y_{step}(t) = \lim_{s \rightarrow 0} sY_{step}(s) = G(0) = 18..$



# Transient and Steady-state Response

Recall that

$$Y_{step}(s) = K' \frac{(s + z_1)(s + z_2) \dots (s + z_m)}{(s + p_1)(s + p_2) \dots (s + p_n)} \times \frac{1}{s}$$

$$y_{step}(t) = \mathcal{L}^{-1} \left\{ \frac{\alpha_1}{s + p_1} + \dots + \frac{\alpha_n}{s + p_n} + \frac{\beta}{s} \right\} = \underbrace{\sum_{i=1}^n \alpha_i e^{-p_i t} u(t)}_{\text{transient}} + \underbrace{\beta u(t)}_{\text{steady-state}}$$

The step response of a LTI system may be decomposed into 2 parts:-

- ▶ **Transient Response** comprising terms that map to the system poles.
  - ▶ If the system is stable, this component decays to zero when  $t \rightarrow \infty$  so it exists only “temporarily”.
  - ▶ If the system is unstable, this component will “blow up”.
- ▶ **Steady-state Response** is a constant if the input is a step function. For an arbitrary input, terms in this component map to the input pole(s) so it is a scaled form of the input.

# Definition of Steady-State/DC/Static Gain

- ▶ Often times, it is useful to know how the step response behaves at steady-state by inspecting  $G(s)$ .
  - ▶ If the system is unstable, then the step response will “blow up”.
  - ▶ If the system is marginally stable, then the steady-state step response is a sinusoid.
  - ▶ If the system is stable, then the “final” value is a constant.
- ▶ Suppose the input  $x(t) = Au(t)$ , i.e. a step function of magnitude  $A$ , is applied to a **stable** LTI system. Then,

$$X(s) = \mathcal{L}\{x(t)\} = \mathcal{L}\{A u(t)\} = \frac{A}{s}$$

Using the Final Value Theorem, the step response of a stable LTI system at steady-state ( $t \rightarrow \infty$ ) can be expressed as

$$\lim_{t \rightarrow \infty} y_{step}(t) = \lim_{s \rightarrow 0} sY_{step}(s) = \lim_{s \rightarrow 0} sG(s) \frac{A}{s} = A \cdot G(0)$$

$G(0)$  is known as the **Steady-state/DC/Static Gain**. It is a transfer function parameter that can be used to ascertain the “final” value of  $y_{step}(t)$  without the need to compute  $y_{step}(t)$  or apply the Final Value Theorem.