

Mid-term Test

Wednesday, November 2, 2022 (110 minutes)

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Problem 1. (a) The (displacement) components in a body are given by

$$u = Axy, \quad v = A(x^2 - y^2), \quad w = 0.$$

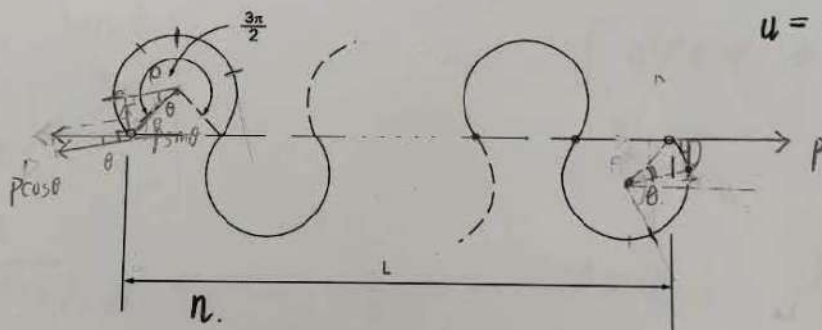
Obtain the (strain components).

(b) Transform the components of the stress state

$$[\sigma] = \begin{bmatrix} 10 & 3 \\ 3 & 2 \end{bmatrix}$$

to a new system rotated (counterclockwise by 30°) from the old system.

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Problem 2. The flat tension spring shown in figure consists of a length of wire of circular cross-section having a diameter d and Young's modulus E . The spring consists of n open loops each of which subtends an angle of $3\pi/2$ radians at its center; the total length between the ends of the spring is L . Considering bending and axial strains only, calculate the stiffness of the spring using the Castigliano's first theorem.

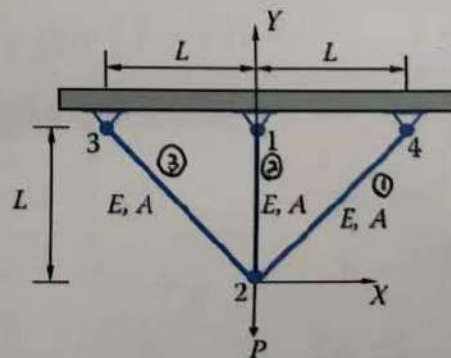


$$u = \frac{\partial U^*}{\partial P}$$

前尾两个.
中间部分.

³⁵
Problem 3. The plane truss is loaded with force P as shown below. Constants E and A for each bar are as shown in the diagram. Determine:

- The nodal displacements;
- The reaction forces;
- The stress in each bar element.



1. 解:

$$a) \varepsilon_{xx} = \frac{\partial u}{\partial x} = Ay$$

$$\varepsilon_{yy} = \frac{\partial v}{\partial y} = -2Ax$$

$$\begin{aligned}\varepsilon_{xy} &= \left(\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \right) \cdot \frac{1}{2} \\ &= (Ax + 2Ax) \cdot \frac{1}{2} \\ &= \frac{3}{2}Ax\end{aligned}$$

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b) counterclockwise $30^\circ \Rightarrow \theta = 30^\circ$, $\cos\theta = \frac{\sqrt{3}}{2}$, $\sin\theta = \frac{1}{2}$

$$[Q] = \begin{bmatrix} \cos\theta & \sin\theta \\ -\sin\theta & \cos\theta \end{bmatrix} = \begin{bmatrix} \frac{\sqrt{3}}{2} & \frac{1}{2} \\ -\frac{1}{2} & \frac{\sqrt{3}}{2} \end{bmatrix}$$

$$\begin{aligned}[\varepsilon'] &= [Q][\varepsilon][Q]^T = \begin{bmatrix} \frac{\sqrt{3}}{2} & \frac{1}{2} \\ -\frac{1}{2} & \frac{\sqrt{3}}{2} \end{bmatrix} \begin{bmatrix} 10 & 3 \\ 3 & 2 \end{bmatrix} \begin{bmatrix} \frac{\sqrt{3}}{2} & -\frac{1}{2} \\ \frac{1}{2} & \frac{\sqrt{3}}{2} \end{bmatrix} \\ &= \begin{bmatrix} 8 + \frac{9}{4}\sqrt{3} & -2\sqrt{3} + \frac{3}{2} \\ -2\sqrt{3} + \frac{3}{2} & 4 - \frac{3}{2}\sqrt{3} \end{bmatrix}\end{aligned}$$

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2. 解:

Assume load P subjected to the spring horizontally.

$$k \cdot u = P$$

$$k = \frac{P}{u}$$

$$2u = \frac{\alpha U^*_{total}}{\alpha P}$$

Consider one loop

due to bending: $U^* = \frac{1}{2} \int_0^{\frac{3}{2}\pi} \frac{M^2(\theta)}{EI} r d\theta$

$$\begin{aligned} M(\theta) &= P r \sin \theta \\ U^* &= \frac{1}{2} \frac{1}{EI} \int_0^{\frac{3}{2}\pi} P^2 r^3 \sin^2 \theta d\theta \\ &= \frac{P^2 r^3 \cdot 3\pi}{2EI} \\ U^*_{total} &= \frac{3n\pi P^2 r^3}{2EI} \\ k u &= \frac{3n\pi P r^3}{EI \cdot 2} \\ k &= \frac{2EI}{3n\pi r^3} \end{aligned}$$

$$\begin{aligned} M(\theta) &= P r \left[1 - \sin\left(\frac{\pi}{4} - \theta\right) \right] \\ &= P r \left(1 - \frac{\sqrt{2}}{2} \cos \theta + \frac{\sqrt{2}}{2} \sin \theta \right) \end{aligned}$$

$$U^* = \frac{1}{2EI} \int_0^{\frac{3}{2}\pi} M^2(\theta) r d\theta$$

$$= \frac{P^2 r^3}{2EI} \int_0^{\frac{3}{2}\pi} \left(1 - \frac{\sqrt{2}}{2} \cos \theta + \frac{\sqrt{2}}{2} \sin \theta \right)^2 d\theta$$

$$2u = \frac{\alpha U^*}{\alpha P} = \frac{P r^3 \cdot n}{EI} \int_0^{\frac{3}{2}\pi} \left(1 - \frac{\sqrt{2}}{2} \cos \theta + \frac{\sqrt{2}}{2} \sin \theta \right)^2 d\theta$$

$$\therefore k = \frac{P}{u} = \frac{2EI}{r^3 \cdot \int_0^{\frac{3}{2}\pi} \left(1 - \frac{\sqrt{2}}{2} \cos \theta + \frac{\sqrt{2}}{2} \sin \theta \right)^2 d\theta \cdot n}$$

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3. 解:

FEA

for Bar ①:

$$\theta = 45^\circ, L = \cos\theta = \frac{\sqrt{2}}{2}, m = \sin\theta = \frac{\sqrt{2}}{2}$$

$$[k_0] = \begin{bmatrix} k^2 & lm & -l^2 & -lm \\ lm & m^2 & -lm & -m^2 \\ -l^2 & -lm & l^2 & lm \\ -lm & -m^2 & lm & m^2 \end{bmatrix} \frac{AE}{\sqrt{2}L} = \frac{AE}{\sqrt{2}L} \begin{bmatrix} \frac{1}{2} & \frac{1}{2} & -\frac{1}{2} & -\frac{1}{2} \\ \frac{1}{2} & \frac{1}{2} & -\frac{1}{2} & -\frac{1}{2} \\ -\frac{1}{2} & -\frac{1}{2} & \frac{1}{2} & \frac{1}{2} \\ -\frac{1}{2} & -\frac{1}{2} & \frac{1}{2} & \frac{1}{2} \end{bmatrix}$$

for Bar ②:

$$\theta = 90^\circ, L = \cos\theta = 0, m = \sin\theta = 1$$

$$[k_0] = \begin{bmatrix} u_2 & v_2 & u_1 & v_1 \\ 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & -1 \\ 0 & 0 & 0 & 0 \\ 0 & -1 & 0 & 1 \end{bmatrix} \frac{AE}{L} = \frac{AE}{\sqrt{2}L} \begin{bmatrix} u_2 & v_2 & u_1 & v_1 \\ 0 & 0 & 0 & 0 \\ 0 & \sqrt{2} & 0 & -\sqrt{2} \\ 0 & 0 & 0 & 0 \\ 0 & -\sqrt{2} & 0 & \sqrt{2} \end{bmatrix} = \frac{AE}{\sqrt{2}L} \begin{bmatrix} u_1 & v_1 & u_2 & v_2 \\ 0 & 0 & 0 & 0 \\ 0 & \sqrt{2} & 0 & -\sqrt{2} \\ 0 & 0 & 0 & 0 \\ 0 & -\sqrt{2} & 0 & \sqrt{2} \end{bmatrix}$$

for Bar ③:

$$\theta = 135^\circ, L = -\frac{\sqrt{2}}{2}, m = \frac{\sqrt{2}}{2}$$

$$[k_0] = \frac{AE}{\sqrt{2}L} \begin{bmatrix} u_2 & v_2 & u_3 & v_3 \\ \frac{1}{2} & -\frac{1}{2} & -\frac{1}{2} & \frac{1}{2} \\ -\frac{1}{2} & \frac{1}{2} & \frac{1}{2} & -\frac{1}{2} \\ -\frac{1}{2} & \frac{1}{2} & \frac{1}{2} & -\frac{1}{2} \\ \frac{1}{2} & -\frac{1}{2} & -\frac{1}{2} & \frac{1}{2} \end{bmatrix}$$

Combine above: $\frac{AE}{\sqrt{2}L}$

$$\begin{bmatrix} u_1 & v_1 & u_2 & v_2 & u_3 & v_3 & u_4 & v_4 \\ 0 & 0 & 0 & 0 & -\frac{1}{2} & \frac{1}{2} & -\frac{1}{2} & -\frac{1}{2} \\ 0 & \sqrt{2} & 0 & -\sqrt{2} & \frac{1}{2} & -\frac{1}{2} & -\frac{1}{2} & -\frac{1}{2} \\ 0 & 0 & 1 & 0 & \frac{1}{2} & -\frac{1}{2} & \frac{1}{2} & \frac{1}{2} \\ 0 & -\sqrt{2} & 0 & \sqrt{2} & -\frac{1}{2} & \frac{1}{2} & \frac{1}{2} & \frac{1}{2} \\ & & -\frac{1}{2} & \frac{1}{2} & & & \frac{1}{2} & \frac{1}{2} \\ & & \frac{1}{2} & -\frac{1}{2} & & & \frac{1}{2} & \frac{1}{2} \\ & & -\frac{1}{2} & -\frac{1}{2} & & & \frac{1}{2} & \frac{1}{2} \\ & & -\frac{1}{2} & -\frac{1}{2} & & & \frac{1}{2} & \frac{1}{2} \end{bmatrix} \begin{bmatrix} u_1 \\ v_1 \\ u_2 \\ v_2 \\ u_3 \\ v_3 \\ u_4 \\ v_4 \end{bmatrix} = \begin{bmatrix} F_{1x} \\ F_{1y} \\ F_{2x} \\ F_{2y} \\ F_{3x} \\ F_{3y} \\ F_{4x} \\ F_{4y} \end{bmatrix}$$

BC: $u_1 = v_1 = 0, u_3 = v_3 = 0, u_4 = v_4 = 0, F_{2y} = -P, F_{2x} = 0$

We can solve and get:

$$\begin{cases} u_2 = 0 \\ v_2 = \frac{-\sqrt{2}PL}{(1+\sqrt{2})AE} \end{cases} \begin{cases} F_{1x} = 0 \\ F_{1y} = \frac{\sqrt{2}P}{1+\sqrt{2}} \end{cases}$$

$$\begin{cases} F_{2x} = \frac{-P}{2+2\sqrt{2}} \\ F_{2y} = \frac{P}{2+2\sqrt{2}} \end{cases} \begin{cases} F_{4x} = \frac{+P}{2+2\sqrt{2}} \\ F_{4y} = \frac{P}{2+2\sqrt{2}} \end{cases}$$

$$F_0 = \frac{\sqrt{2}P}{2+2\sqrt{2}}, F_2 = \frac{\sqrt{2}P}{1+\sqrt{2}}, F_3 = \frac{\sqrt{2}P}{2+2\sqrt{2}}$$

$$G_0 = \frac{P}{A(\sqrt{2}+2)}, G_2 = \frac{\sqrt{2}P}{A(\sqrt{2}+1)}, G_3 = \frac{P}{A(\sqrt{2}+2)}$$

ANS

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