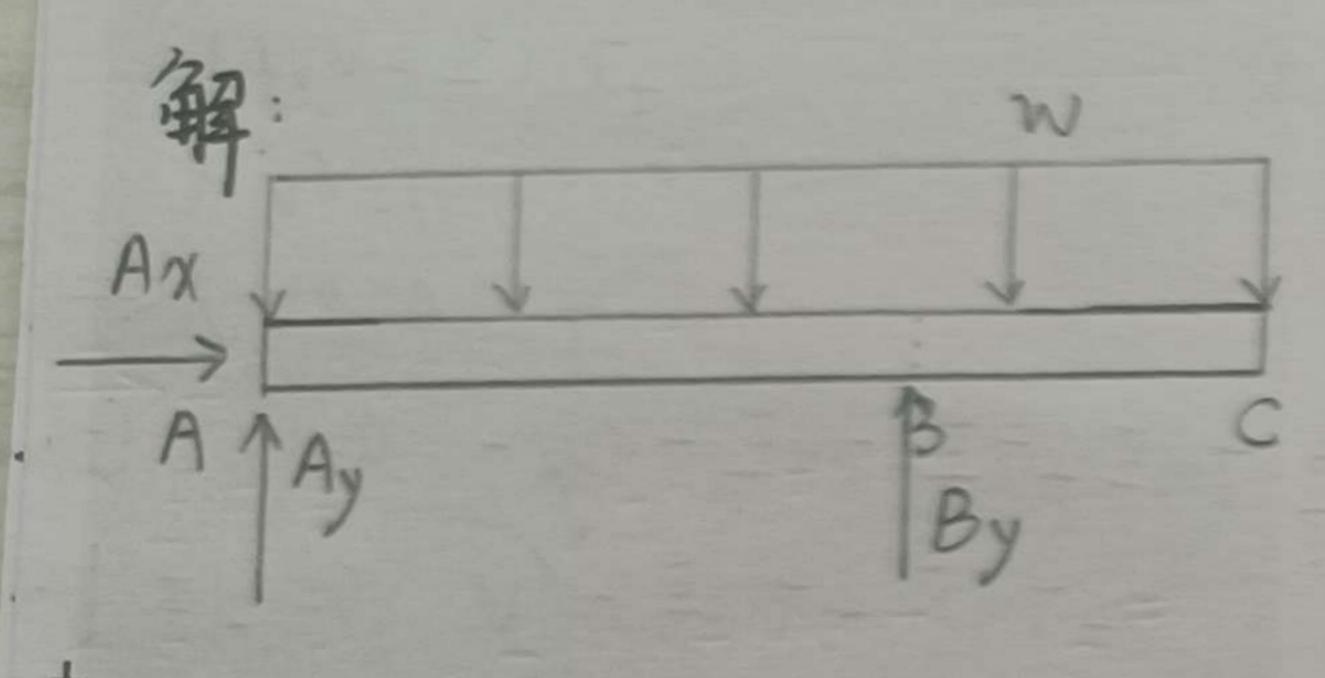
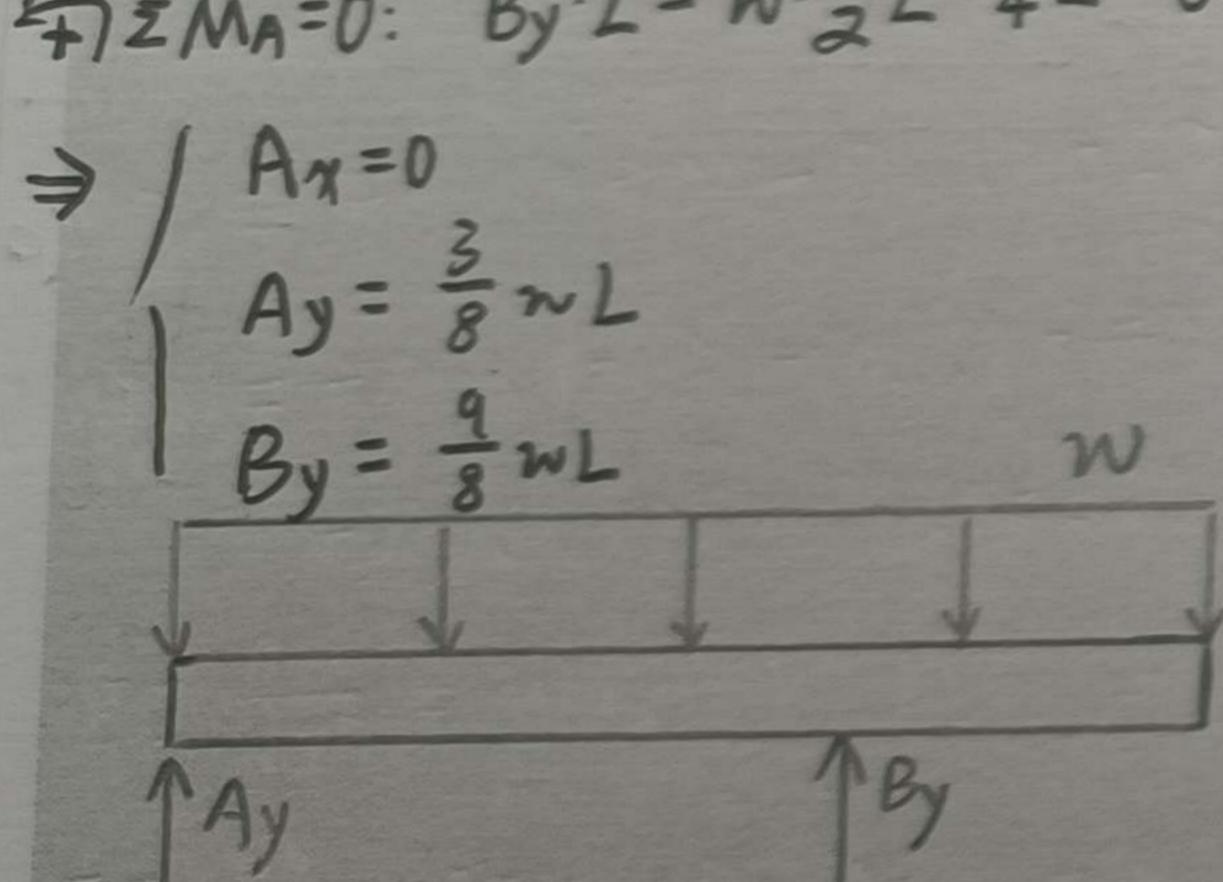
Problem 2



=> \(\bar{z} \in \bar{x} = 0 : \An = 0

1 = 0: +Ay+By-W-===0

年 ZMA=0: By-L-w- 3L- 4L=0



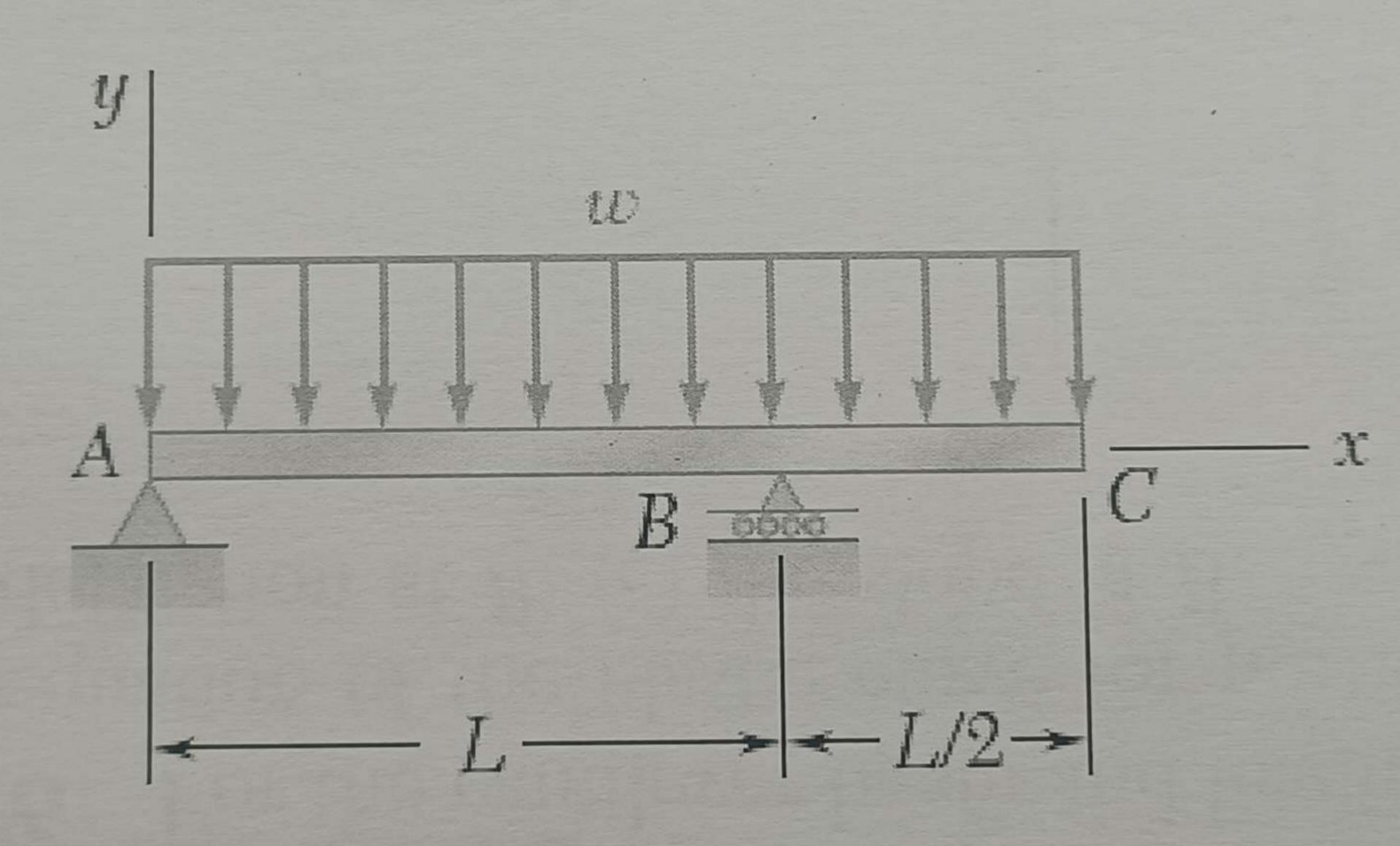
 $V(x) = +Ay(x-0)^{0} + By(x-1)^{0} - w(x-0)^{1}$ $M(x) = Ay(x-0)^{1} + By(x-1)^{1} - w \frac{1}{2}(x-0)^{2}$

 $EI = \frac{dy}{dx^2} = M(X)$

 $EI \frac{dy}{dx} = EI\theta = \frac{1}{2}Ay(x-0)^{2} + \frac{1}{2}By(x-1)^{2} - \frac{1}{6}w(x-0)^{3} + C_{1}$

EI $y = \frac{1}{6} Ay (x-0)^3 + \frac{1}{6} By (x-1)^3 - \frac{1}{24} w (x-0)^4 + C_1 x + C_2$

9.7 For the beam and loading shown, determine (a) the equation of the elastic curve for portion AB of the beam, (b) the slope at A, (c) the slope at B.



b) $\theta_{A} = \frac{1}{EI} (0+0+0-\frac{1}{48}wL^{3})$ $= -\frac{wL^{3}}{48EI}$

c) $\theta_B = \frac{1}{EI} (\frac{1}{2} \times \frac{3}{8} w L \cdot L^2 + 0 - \frac{1}{6} w L^3)$ $= \frac{1}{EI} \cdot (0 w L^3) = 0$

 $A: \Lambda = 0, y = 0, M = 0$

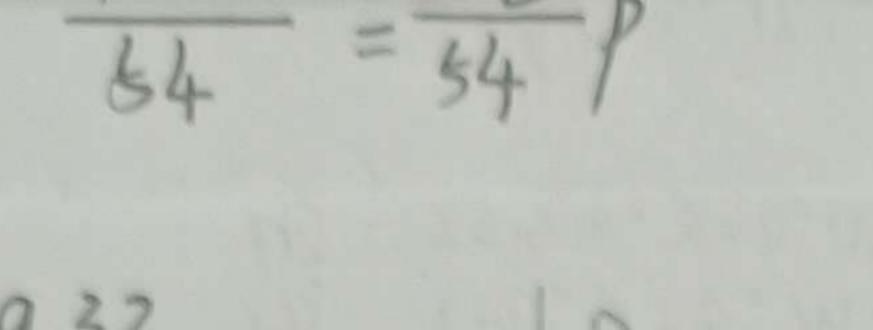
B: N=L, Y=0, M=0

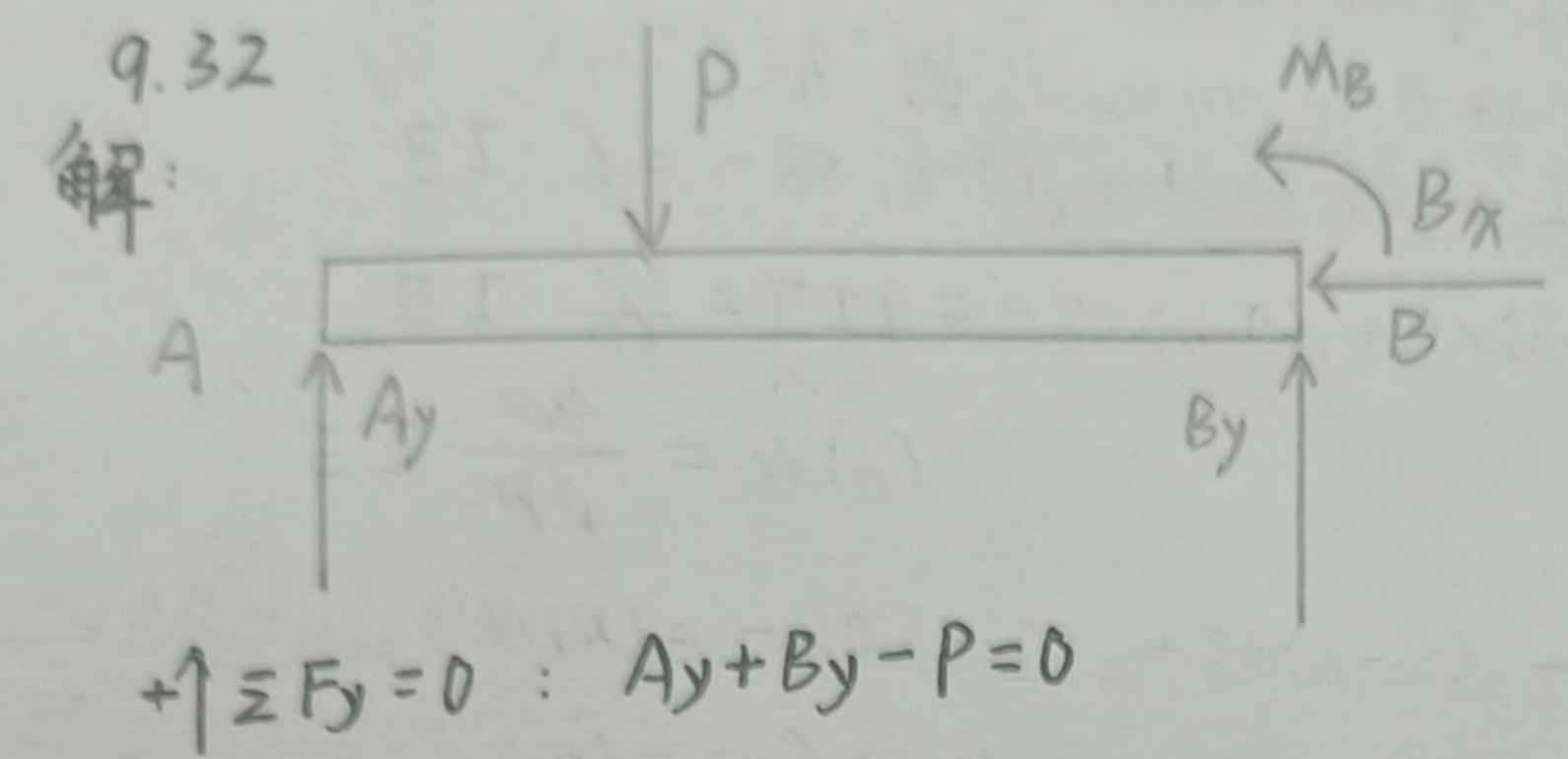
C: $x = \frac{3}{2}L$, y = 0, M = 0

$$\Rightarrow 0 = 0 + 0 - 0 + C_2 \cdot C_2 = 0$$

$$0 = \frac{1}{6} \cdot \frac{2}{8}wL \cdot L^3 + 0 - \frac{1}{24}wL^4 + C_1L + 0 \cdot C_1 = \frac{1}{48}wL^3$$

$$y = \frac{1}{EI} \left(\frac{1}{6} \cdot \frac{2}{8}wL \left(\frac{1}{4} \cdot 0 \right)^3 + \frac{1}{6} \cdot \frac{2}{8}wL \left(\frac{1}{4} \cdot 0 \right)^4 - \frac{1}{48}wL^3 \cdot \frac{1}{48}wL^3$$



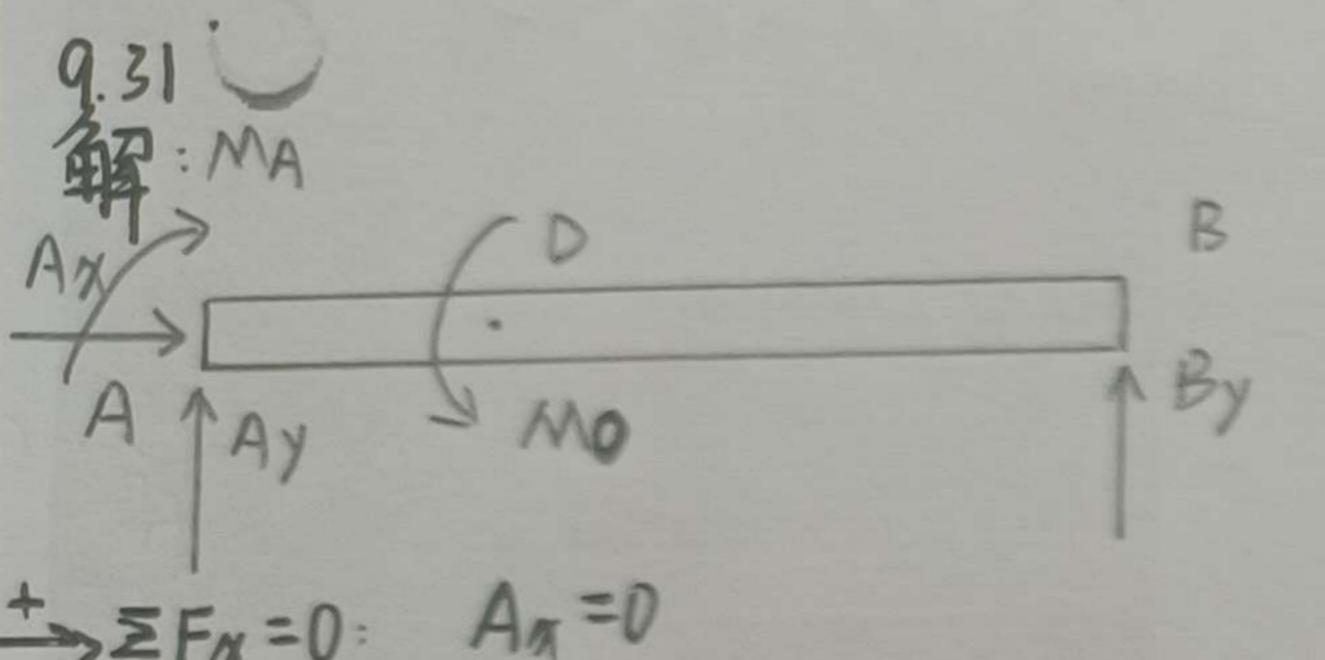


+> EFX=0: -BX=0 TI S.MB=0: MB + P. 3 L-Ay. L=0

By = P - Ay

EI. y = \frac{1}{2} MB \langle (N-0)^2 + \frac{1}{6} By \langle (N-0)^3 - \frac{1}{6} P \langle (N-\frac{2}{3}L)^3 + C_1 N + C_2 JAB = DY + 3PL DOT A: N=L,)=0, M=0

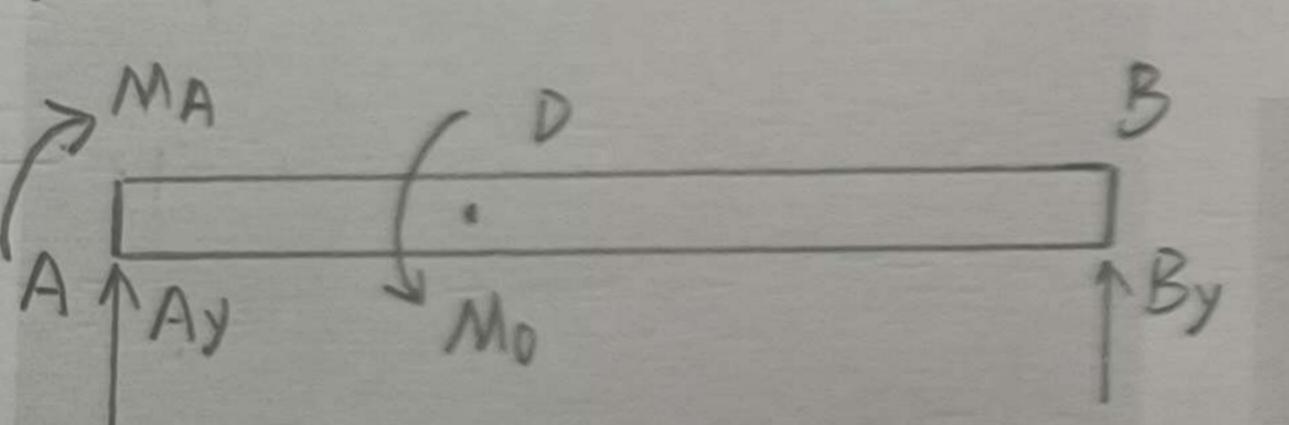
deflection at point D if a is equal to L/3.



-> \(\bullet F_{\pi} = 0:

+1 $\Sigma Fy = 0: Ay + By = 0$

47 = MA = 0: - MA + MO + By L = 0



V(A) = + Ay (M-0) + By (M-L)

M(X) = + MA (1-0) - Mo (1 - =)

+ Ay (A-0) + By (x-1)

$$EI \cdot \frac{\partial^2 y}{\partial x^2} = M(x)$$

Fig. P9.31

EI. dy = EI. 0= MA (1-0) - MO (1-3) + Ay = (1-0) + By = (1-2) + C,

EI. y = MA. = (1-0)2 - Mo = (1-1)2+ + Ay (1-0)3+ + By (1-1)3+ C. 4+ C2

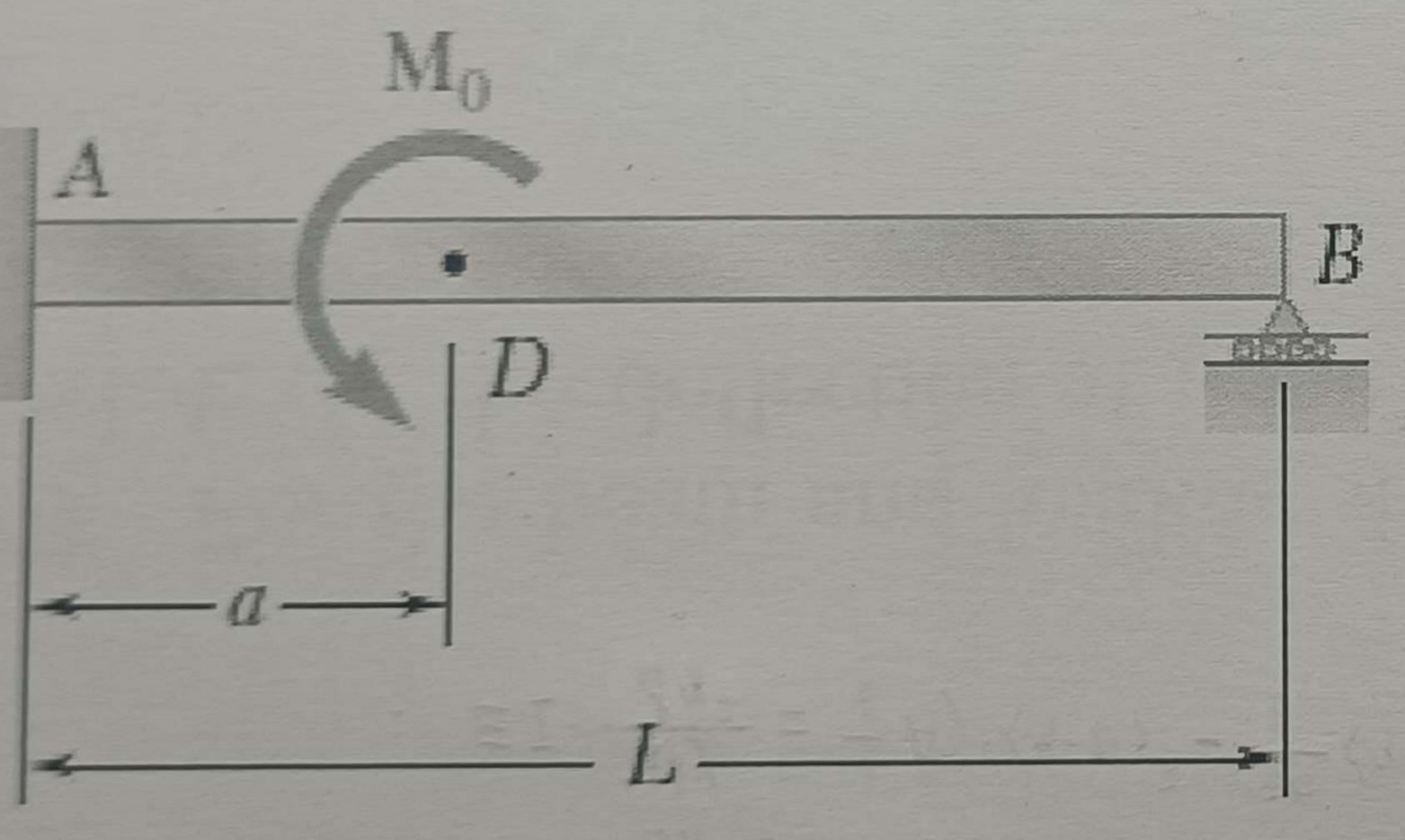
A: 2=0, y=0, 0=0

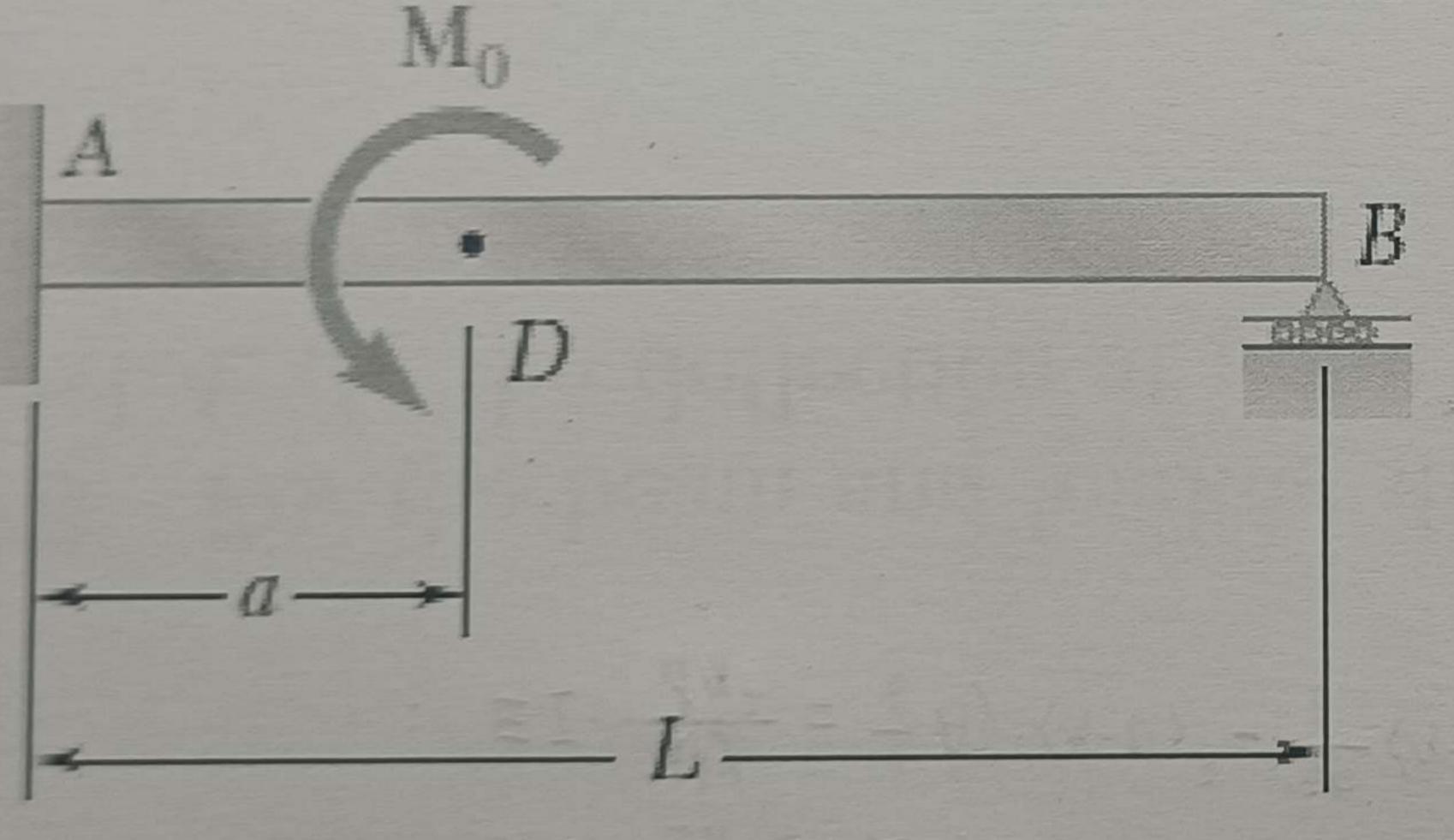
B: N=L, Y=0, M=0

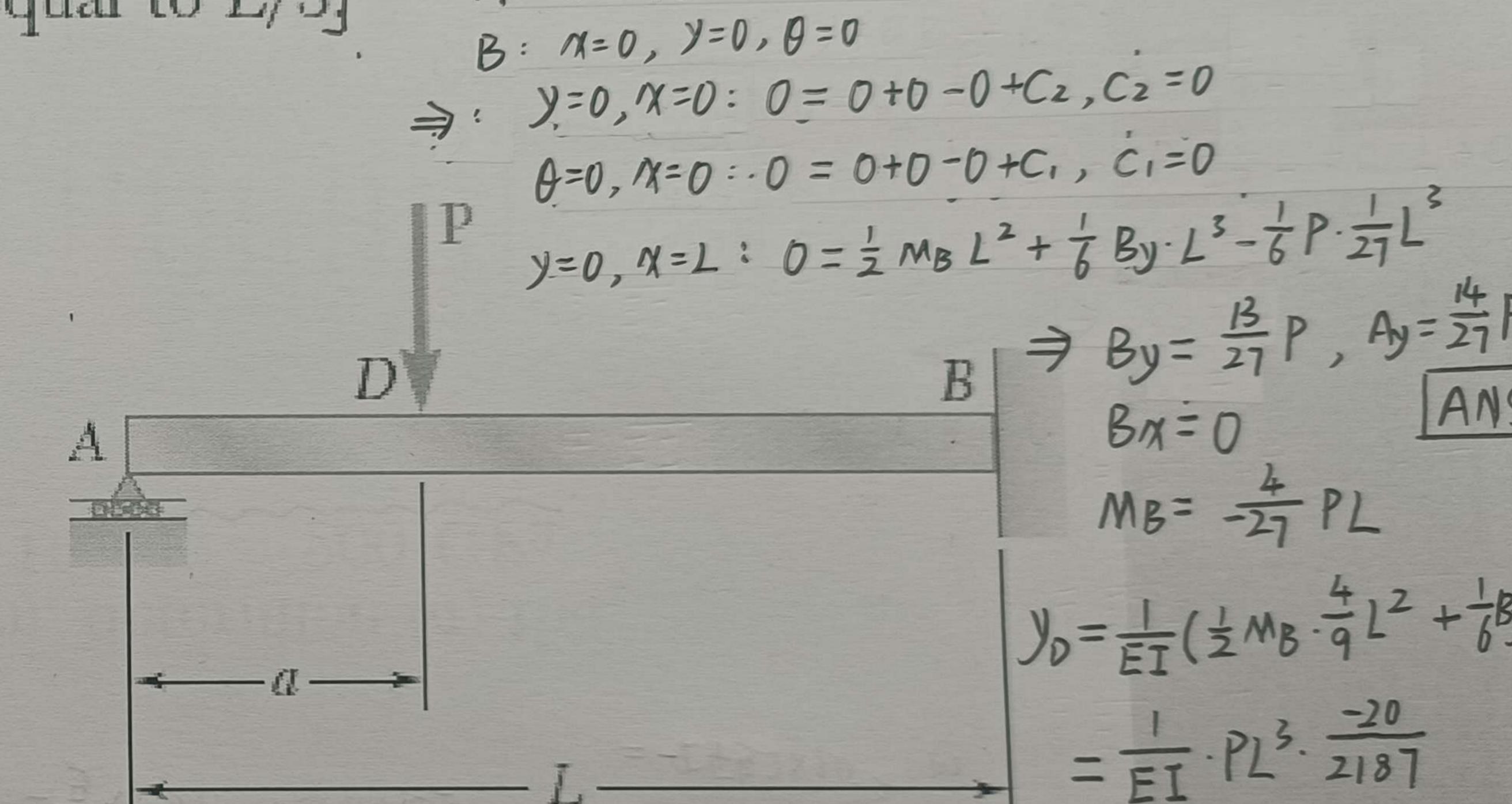
> y=0, x=0: 0=0-0+0+0+cz, c2=0

 $\theta = 0$, $\chi = 0$: $0 = 0 - 0 + 0 + 0 + C_1$, $C_1 = 0$

y=0, x=1: 0= \frac{1}{2}MA. L^2 - Mo. \frac{1}{2}. \frac{4}{9}L^2 + \frac{1}{6}Ay. L^3 + 0 + 0 + 0.







V(X) = +By (X-0) -P. (X-31)

 $EI\frac{dy}{dx^2} = M(x)$

M(x) = +MB(x-0)0 +By(x-0)1-P(x-3L)

 $EI \frac{dy}{dx} = MB(x-0)' + \frac{1}{2}By(x-0)^2 - \frac{1}{2}P(x-\frac{3}{3}L)^2 + C_1 = EI \cdot \theta$

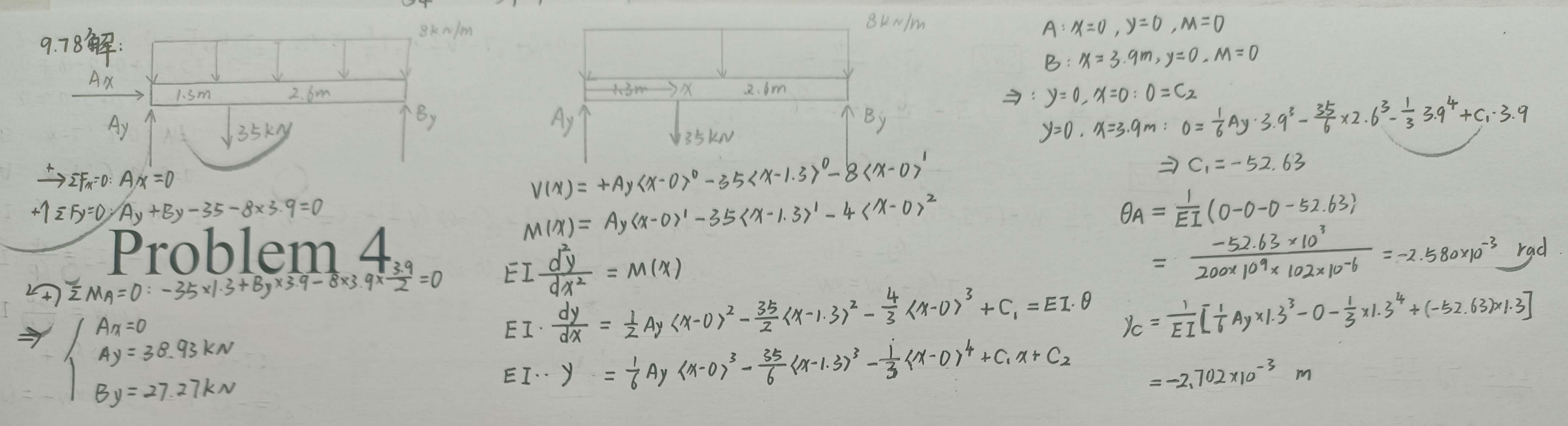
 $B \Rightarrow By = \frac{13}{27}P, Ay = \frac{14}{27}P$ リカ= == (= MB· 4 12 + 18) 8 3 27 4) = FT - PL - 2187

由里何夕.

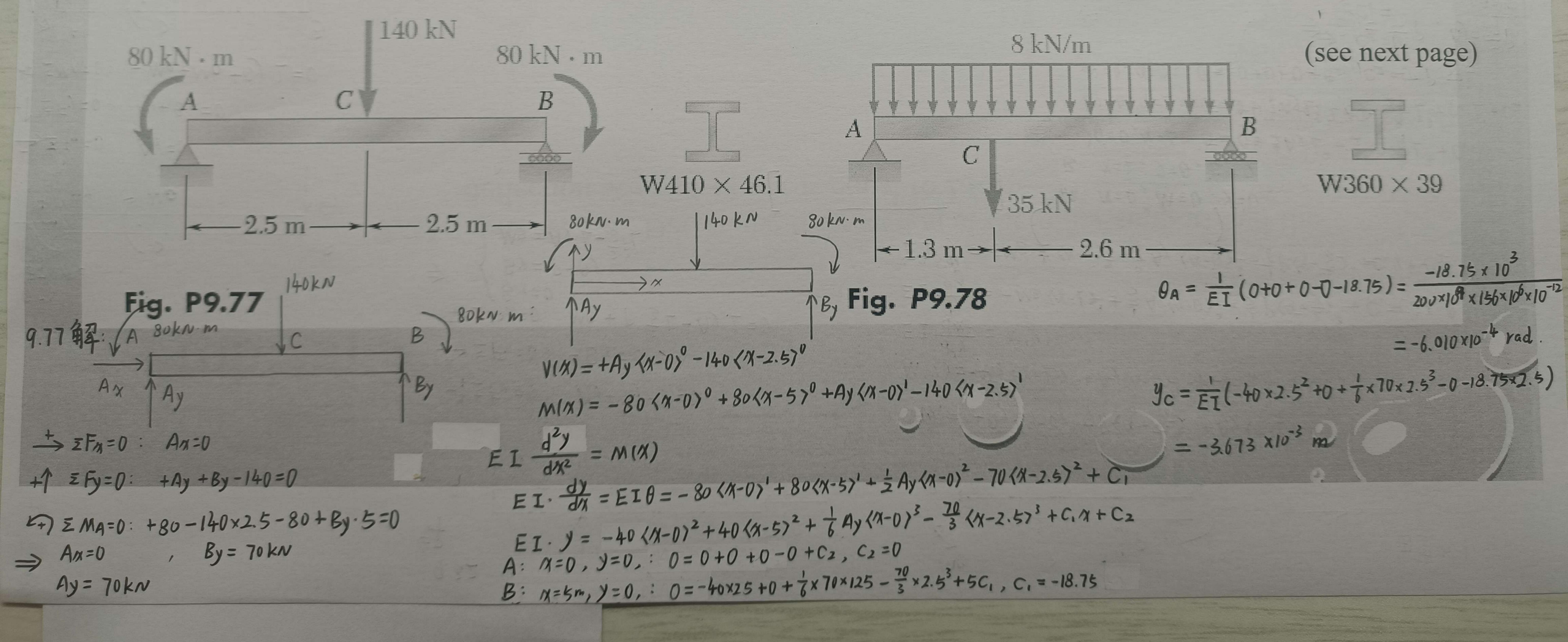
Fig. P9.32

$$MA = M_0 + By \cdot L$$
 $Ay = -By$
 $\Rightarrow 0 = \frac{1}{2}L^2(M_0 + ByL) - M_0 \cdot \frac{2}{9}L^2 + \frac{1}{6}L^3 \cdot (-B_y)$
 $\Rightarrow By = \frac{-5M_0}{6L}$
 ANS
 $y_0 = \frac{1}{2}M_4 \cdot \frac{1}{9}L^2 - 0 + \frac{1}{6}Ay \cdot \frac{1}{27} + 0 + 0 + 0$
 $= \frac{7}{486}M_0L^2$

ANS



9.77 and 9.78 For the beam and loading shown, determine (a) the slope at end A, (b) the deflection at point C. Use E = 200 GPa.



=) C1=-52.63

OA = EI (0-0-0-52.63)

 $=-2,702 \times 10^{-3} \text{ m}$

-52.63 × 10

200x 109x 102x 10-6 = -2.580x 10-3 rad.