Heat Transfer HW5

Basic information as follows:

```
Heat_Transfer_HW5.m × +
 1 -
       clc
2-
       clear all
 3
 4
       %basic infomation
 5-
       L=0.05; %length L=0.05m
       b=0.01; %base thickness b=1cm
 7-
       w=1; %width w=1m
       h=15; %heat transfer coefficient h=15W/(m^2*K)
       k=180; %thermal conductivity k=180W/(m*K)
10 -
       T_infinity=25; %T_∞=25°C
11 -
       T_base=200; %T_base=200°C
       sin_theta=0.0995; %angle sin theta
13 -
       cos theta=0.9950;
```

Energy balance euqtion:

$$\left[1-\left(m-rac{1}{2}
ight)rac{\Delta x}{L}
ight](T_{m-1}-T_m)+\left[1-\left(m+rac{1}{2}
ight)rac{\Delta x}{L}
ight](T_{m+1}-T_m)+rac{h\Delta x^2}{\lambda L\sin heta}(T_{\infty}-T_m)=0$$

Rearrange:

$$\left[1-\left(m-\frac{1}{2}\right)\frac{\Delta x}{L}\right]T_{m-1}+\left[-2+2\frac{m\cdot\Delta x}{L}-\frac{h(\Delta x)^2}{\lambda L\sin\theta}\right]T_m+\left[1-\left(m+\frac{1}{2}\right)\frac{\Delta x}{L}\right]T_{m+1}=-\frac{h(\Delta x)^2}{\lambda L\sin\theta}T_{\infty}$$

以此来构建矩阵方程, A·T=C

若 M=101. 则 A 为 100*100 的方阵

矩阵 A 的构建:

For m=1

$$\left(1-rac{1}{2} imesrac{\Delta x}{L}
ight)T_{\mathrm{base}} \,+\left[-2+2rac{\Delta x}{L}-rac{h(\Delta x)^2}{\lambda L\sin heta}
ight]T_1+\left[1-\left(1+rac{1}{2}
ight)rac{\Delta x}{L}
ight]T_2\,=-rac{h(\Delta x)^2}{\lambda L\sin heta}T_\infty$$

Rearrange:

$$\overline{\left[-2+2rac{\Delta x}{L}-rac{h(\Delta x)^2}{\lambda L\sin heta}
ight]}T_1+\left[1-\left(1+rac{1}{2}
ight)rac{\Delta x}{L}
ight]T_2} = -rac{h(\Delta x)^2}{\lambda L\sin heta}T_\infty-T_{
m base}\left(1-rac{1}{2} imesrac{\Delta x}{L}
ight)$$



%row1

$$A(1, 1) = -2 + 2 \times 1 \times \det_x/L - h \times \det_x^2/(k \times L \times \sin_t h + a)$$

 $A(1, 2) = 1 - (1 + 1/2) \times \det_x/L$:

From m=2 to 99:

$$\left[1-\left(m-rac{1}{2}
ight)rac{\Delta x}{L}
ight]T_{m-1}+\left[-2+2rac{m\cdot\Delta x}{L}-rac{h(\Delta x)^2}{\lambda L\sin heta}
ight]T_m+\left[1-\left(m+rac{1}{2}
ight)rac{\Delta x}{L}
ight]T_{m+1}
ight]=-rac{h(\Delta x)^2}{\lambda L\sin heta}T_{\infty}$$



%row2 to row 99 (98*98)

For m=100, consider the boundary condition equation:

$$\lambda A_{ ext{left}} rac{T_{99} - T_{100}}{\Delta x} + h A_{ ext{conv}} (T_{\infty} - T_{100}) = 0 \ \lambda \cdot 2w rac{\Delta x}{2} an heta \cdot rac{T_{99} - T_{100}}{\Delta x} + h \cdot 2w rac{\Delta x}{2 ext{cos } heta} \cdot (T_{\infty} - T_{100}) = 0 \ \left[\lambda T_{99} + \left(-\lambda - rac{h \Delta x}{ ext{sin } heta}
ight) T_{100}
ight] = rac{-h \Delta x}{ ext{sin } heta} T_{\infty}$$

%row100 is based on the Boundary Condition

$$A(M-1, M-2) = k;$$

$$A(M-1, M-1) = -k-h*\det x/\sin theta;$$

矩阵C的构建

For
$$m = 1$$
,

$$\left[-2+2rac{\Delta x}{L}-rac{h(\Delta x)^2}{\lambda L\sin heta}
ight]T_1+\left[1-\left(1+rac{1}{2}
ight)rac{\Delta x}{L}
ight]T_2 \ = \left[-rac{h(\Delta x)^2}{\lambda L\sin heta}T_\infty-T_{
m base}\left(1-rac{1}{2} imesrac{\Delta x}{L}
ight)
ight]$$

%row 1

 $C(1,1) = -h*\det_x^2/(k*L*\sin_t + 2\pi) = -h*\det_x^2/(k*L*\cos_t + 2\pi) = -h*\det_x^2/(k*L*\sin_t + 2\pi) = -h*\det_x^2/(k*L*\cos_t + 2\pi) = -h*\det_x^2/(k*L*C*\cos_t + 2\pi) = -h*\det_x^2/(k*L*C*\cos_t + 2\pi) = -h*\det_x^2/(k*L*C*\cos_t + 2\pi) = -h*\det_x^2/(k*L*C*\cos_t + 2\pi) =$

From m=2 to 99,

$$\left[1-\left(m-\frac{1}{2}\right)\frac{\Delta x}{L}\right]T_{m-1}+\left[-2+2\frac{m\cdot\Delta x}{L}-\frac{h(\Delta x)^2}{\lambda L\sin\theta}\right]T_m+\left[1-\left(m+\frac{1}{2}\right)\frac{\Delta x}{L}\right]T_{m+1}=\boxed{-\frac{h(\Delta x)^2}{\lambda L\sin\theta}T_\infty}$$

%row2 to row99, the energy balance equation constant term

For m=100, the boundary condition equation:

$$\lambda A_{ ext{left}} \, rac{T_{99} - T_{100}}{\Delta x} + h A_{ ext{conv}} (T_{\infty} - T_{100}) = 0 \ \lambda \cdot 2w rac{\Delta x}{2} an heta \cdot rac{T_{99} - T_{100}}{\Delta x} + h \cdot 2w rac{\Delta x}{2 ext{cos} \, heta} \cdot (T_{\infty} - T_{100}) = 0 \ \lambda T_{99} + igg(-\lambda - rac{h \Delta x}{\sin heta} igg) T_{100} = igg(rac{-h \Delta x}{\sin heta} T_{\infty} igg)$$

%row 100, the boundary condition constant term $C(M-1, 1) = -h*\det x/\sin theta*T infinity;$

Calculate the rate of heat transfer from the fin:

$$\phi_{fin} = \frac{hw\Delta x}{cos\theta} \left[(T_{base} - T_{\infty}) + 2 \sum_{i=1}^{M-2=99} (T_i - T_{\infty}) + (T_{M-1} - T_{\infty}) \right]$$

$$\phi_{fin} = \frac{hw\Delta x}{cos\theta} \left[(T_{base} + T_{M-1}) + 2 \sum_{i=1}^{M-2} (T_i) - (2M - 2)T_{\infty} \right]$$

%the rate of heat transfer from the fin

T_total=0; %node temperature sum except the base and tip nodes

```
for i=1:M-2
    T_total=T(i)+T_total;
end
Q_fin=h*w*det_x/cos_theta*(T_base+T(M-1)+2*T_total-(2*M-2)*T_infinity);
```

Fin efficiency:

$$egin{align} \Phi_{ ext{max}} &= h A_{ ext{fin,total}} \left(T_0 - T_\infty
ight) = h rac{2wL}{\cos heta} (T_0 - T_\infty) \ \eta &= rac{\Phi_{fin}}{\Phi_{ ext{max}}} \ . \end{align}$$

Q_max=h*2*w*L*(T_base-T_infinity)/cos_theta;
efficiency=Q fin/Q max;

The solutions are as follows:

- 1) Nodes temperatures: please see the Excel Temperature attached.
- 2) $\emptyset_{fin} = 258.4451488078796W$
- 3) $\eta = 0.979630183100343$

They are quite close to the results on the lecture solution.

As for M=201, the process is similar, and the results are as follows:

- 1) Nodes temperatures: please see the Excel Temperature attached.
- 2) $\emptyset_{fin} = 258.4451433579625W$
- 3) $\eta = 0.979630162442563$

The results of these two cases are very close.

The Matlab code is attached behind.

clear all

%basic infomation

L=0.05; %length L=0.05m

b=0.01; %base thickness b=1cm

w=1; %width w=1m

h=15; %heat transfer coefficient h=15W/(m^2*K)

k=180; %thermal conductivity k=180W/(m*K)

 $T_{infinity}=25$; % $T_{\infty}=25$ °C

T_base=200; %T_base=200°C

sin_theta=0.0995; %angle sin theta

 $cos_{theta}=0.9950;$

M=201; %node number

det_x=L/(M-1); %nodal spacing del_x

%From energy balance equation to obtain 99 equations

A=zeros(M-1);

%row1 to row M-2 are based on the energy balance equation

%row1

 $A(1,1)=-2+2*1*det_x/L-h*det_x^2/(k*L*sin_theta);$

 $A(1,2)=1-(1+1/2)*det_x/L;$

```
%row2 to row M-2 (M-3*M-3)
for m=2:M-2
   A(m,m-1)=1-(m-1/2)*det_x/L;
   A(m,m)=-2+2*m*det_x/L-h*det_x^2/(k*L*sin_theta);
   A(m,m+1)=1-(m+1/2)*det_x/L;
end
%rowM-1 is based on the Boundary Condition
A(M-1,M-2)=k;
A(M-1,M-1)=-k-h*det_x/sin_theta;
%Constant terms
C = zeros(M-1,1);
%row 1
C(1,1) = -h*det_x^2/(k*L*sin_theta)*T_infinity-T_base*(1-(1-1/2)*det_x/L);
%row2 to rowM-2, the energy balance equation constant term
for m=2:M-2
   C(m,1)=-h*det_x^2/(k*L*sin_theta)*T_infinity;
end
%row M-1, the boundary condition constant term
C(M-1,1)=-h*det_x/sin_theta*T_infinity;
%node temperatures
```

 $T=A \setminus C$;

%the rate of heat transfer from the fin

T_total=0; %node temperature sum except the base and tip nodes

for
$$i=1:M-2$$

end

Q_fin=h*w*det_x/cos_theta*(T_base+T(M-1)+2*T_total-(2*M-2)*T_infinity);

Q_max=h*2*w*L*(T_base-T_infinity)/cos_theta; efficiency=Q_fin/Q_max;