control

Robótica Móvel e Inteligente / Mobile and Intelligent Robotics

Academic year 2022-23

Departamento de Electrónica, Telecomunicações e Informática
Universidade de Aveiro

Control systems

- **Objective: to impose a given value of some physical quantity in a system by acting on some other physical quantity**
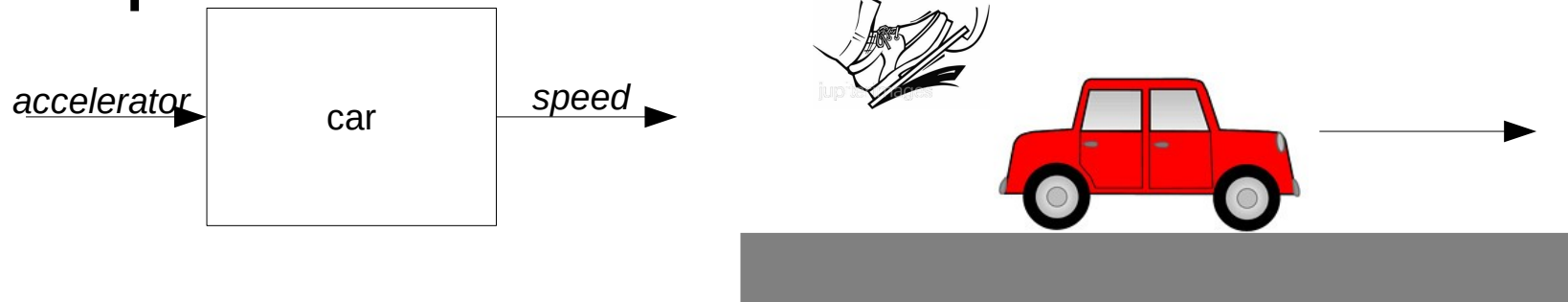

http://blog.caranddriver.com/nissan-develops-fully-electric-steer-by-wire-system-will-go-on-sale-next-year/

basic concepts

- **Systems approach:**
  - Input signal
  - Output signal
  - Process, transforming input into output

- **Objective: to impose a given value at a system's output, by acting in its input**
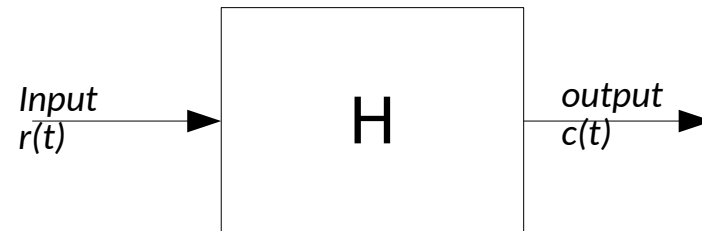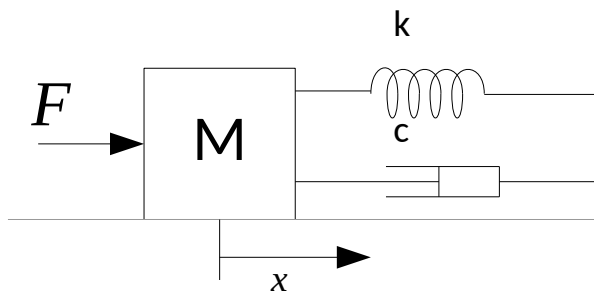

Input r(t) → H → output c(t)

- **Example:**

accelerator → car → speed

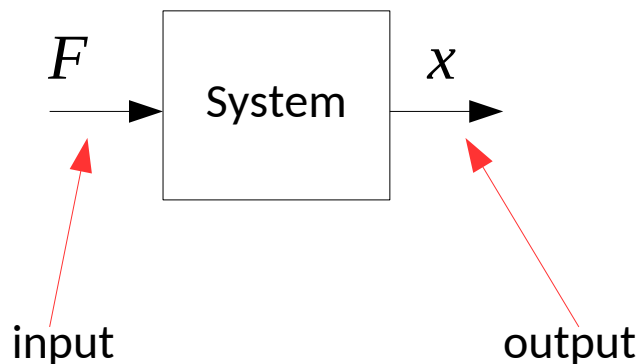# Input / output relationship

- **General case:**
  - input r*(t)* and output c*(t)* are related by differential equations
    - this is the "default" in physical systems...

Mathematical relation
between input and output

$$F(t) = M\,a(t) + c\,v(t) + k\,x(t)$$

$$F(t) = M\,\frac{\mathrm{d}^2 x(t)}{\mathrm{d}\,t^2} + c\,\frac{\mathrm{d}\,x(t)}{\mathrm{d}\,t} + k\,x(t)$$

input
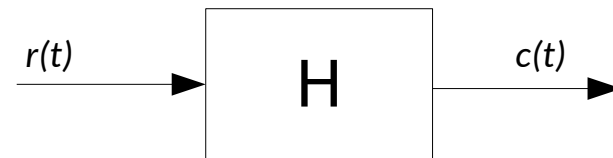
output

Input / output relationship

- **General case:**
  - input r*(t)* and output c*(t)* are related by differential equations

$$a_n \frac{d^n c(t)}{dt^n} + a_{n-1} \frac{d^{n-1} c(t)}{dt^{n-1}} + \ldots + a_0 c(t) = b_m \frac{d^m r(t)}{dt^m} + b_{m-1} \frac{d^{m-1} r(t)}{dt^{m-1}} + \ldots + b_0 r(t)$$

Combination of $c(t)$ and its derivatives

Combination of $r(t)$ and its derivatives

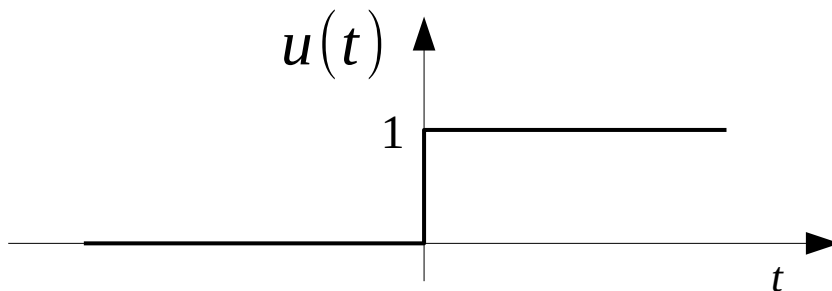Difficult to solve and convert to a systems perspective

r(t) → [H] → c(t)

Laplace transform

- **Differential equations are simplified by the use of Laplace transforms.**

$$L\{f(t)\}=F(s)=\int_{0}^{+\infty} e^{-st} f(t)\, dt$$

$$L^{-1}\{F(s)\}=\frac{1}{2\pi j}\int_{\sigma-j\infty}^{\sigma+j\infty} e^{st} F(s)\, dt = f(t)\cdot u(t)$$

$u(t)$ is the unit step function. We only consider the function $f(t)$ to have non-null values for $t>0$.

Computing the Laplace transform

- **Example for u(t)**

$$L\{u(t)\} = \int_0^{+\infty} e^{-st} u(t)\, dt$$

$$= \int_0^{+\infty} e^{-st}\, dt$$

$$= \left[-\frac{1}{s} e^{-st}\right]_0^{+\infty}$$

$$= 0 - \left(-\frac{1}{s}\right)$$

$$= \frac{1}{s}$$

Laplace transform

TABLE 2.1   Laplace transform table

Item no.	$f(t)$	$F(s)$
1.	$\delta(t)$	1
2.	$u(t)$	$\dfrac{1}{s}$
3.	$tu(t)$	$\dfrac{1}{s^2}$
4.	$t^n u(t)$	$\dfrac{n!}{s^n + 1}$
5.	$e^{-at} u(t)$	$\dfrac{1}{s + a}$
6.	$\sin \omega t u(t)$	$\dfrac{\omega}{s^2 + \omega^2}$
7.	$\cos \omega t u(t)$	$\dfrac{s}{s^2 + \omega^2}$

Laplace transform

Laplace transform theorems

Theorem	Name
$\mathcal{L}[f(t)] = F(s) \ = \int_{0-}^{\infty} f(t)e^{-st}dt$	Definition
$\mathcal{L}[kf(t)] \qquad = kF(s)$	Linearity theorem
$\mathcal{L}[f_1(t) + f_2(t)] = F_1(s) + F_2(s)$	Linearity theorem
$\mathcal{L}[e^{-at}f(t)] \qquad = F(s+a)$	Frequency shift theorem
$\mathcal{L}[f(t-T)] \qquad = e^{-sT}F(s)$	Time shift theorem
$\mathcal{L}[f(at)] \qquad = \dfrac{1}{a}F\left(\dfrac{s}{a}\right)$	Scaling theorem
$\mathcal{L}\left[\dfrac{df}{dt}\right] \qquad = sF(s) - f(0-)$	Differentiation theorem
$\mathcal{L}\left[\dfrac{d^2f}{dt^2}\right] \qquad = s^2F(s) - sf(0-) - f'(0-)$	Differentiation theorem
$\mathcal{L}\left[\dfrac{d^nf}{dt^n}\right] \qquad = s^nF(s) - \sum_{k=1}^{n} s^{n-k}f^{k-1}(0-)$	Differentiation theorem
$\mathcal{L}\left[\int_{0-}^{t} f(\tau)d\tau\right] = \dfrac{F(s)}{s}$	Integration theorem
$f(\infty) \qquad = \lim_{s \to 0} sF(s)$	Final value theorem[1]
$f(0+) \qquad = \lim_{s \to \infty} sF(s)$	Initial value theorem[2]

Laplace transform

$$a_n \frac{d^n c(t)}{dt^n} + a_{n-1} \frac{d^{n-1} c(t)}{dt^{n-1}} + \ldots + a_0 c(t) = b_m \frac{d^m r(t)}{dt^m} + b_{m-1} \frac{d^{m-1} r(t)}{dt^{m-1}} + \ldots + b_0 r(t)$$

Computing the
Laplace transform

$$L\left\{ a_n \frac{d^n c(t)}{dt^n} \right\} \quad = \quad a_n L\left\{ \frac{d^n c(t)}{dt^n} \right\} \quad \text{Linearity}$$

$$= \quad a_n s^n L\{c(t)\} \quad \text{Differentiation theorem}$$

$$= \quad a_n s^n C(s) \quad \text{Definition of Laplace transform}$$

Laplace transform

$$a_n \frac{d^n c(t)}{dt^n} + a_{n-1} \frac{d^{n-1} c(t)}{dt^{n-1}} + \ldots + a_0 c(t) = b_m \frac{d^m r(t)}{dt^m} + b_{m-1} \frac{d^{m-1} r(t)}{dt^{m-1}} + \ldots + b_0 r(t)$$
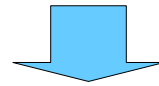
$$a_n s^n C(s) + a_{n-1} s^{n-1} C(s) + \ldots + a_0 C(s) = b_m s^m R(s) + b_{m-1} s^{m-1} R(s) + \ldots + b_0 R(s)$$

$$\left(a_n s^n + a_{n-1} s^{n-1} + \ldots + a_0 \right) C(s) = \left(b_m s^m + b_{m-1} s^{m-1} + \ldots + b_0 \right) R(s)$$

$$\frac{C(s)}{R(s)} = \frac{b_m s^m + b_{m-1} s^{m-1} + \ldots + b_0}{a_n s^n + a_{n-1} s^{n-1} + \ldots + a_0} = G(s)$$

Transfer function

$$\frac{C(s)}{R(s)} = \frac{b_m s^m + b_{m-1} s^{m-1} + \ldots + b_0}{a_n s^n + a_{n-1} s^{n-1} + \ldots + a_0} = G(s)$$

Transfer function

$$C(s) = G(s) \cdot R(s)$$

$R(s) \longrightarrow \boxed{G(s)} \longrightarrow C(s)$

input   system   output
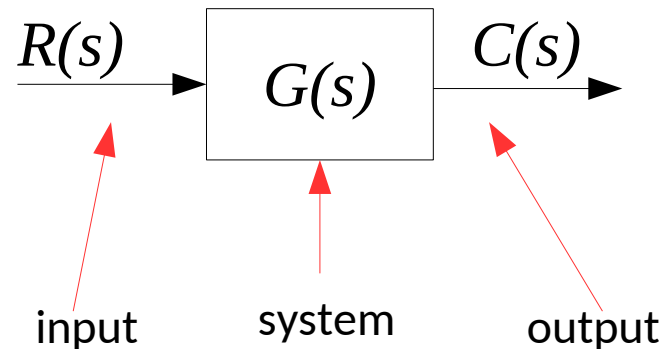
- – A relation expressed originally in terms of a differential equation is expressed as a product
- – the physical nature of input/output relationship is irrelevant; only mathematical relationship matters --> **abstraction**

Block diagram algebra
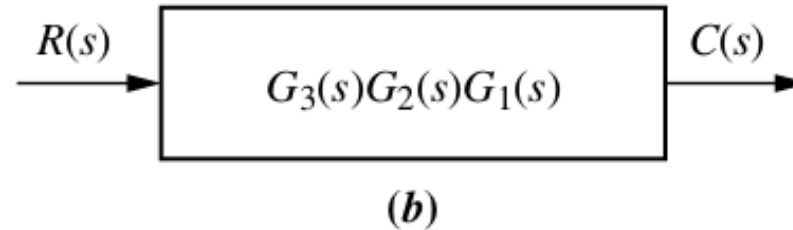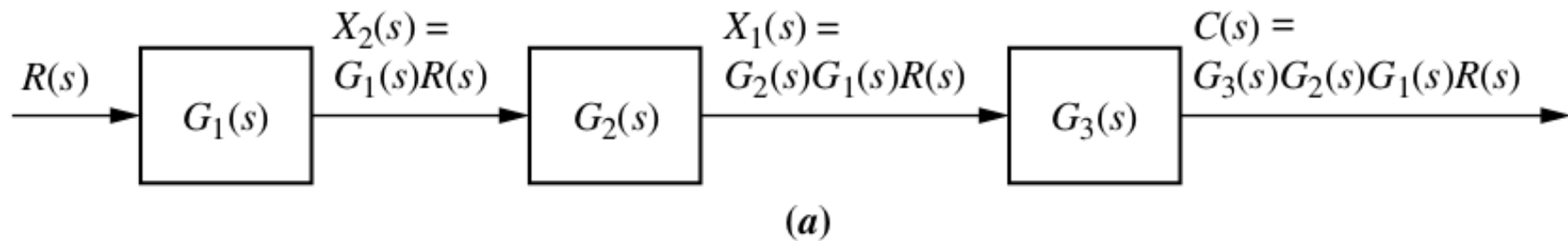
$$X_2(s) = G_1(s)R(s)$$

$$X_1(s) = G_2(s)G_1(s)R(s)$$

$$C(s) = G_3(s)G_2(s)G_1(s)R(s)$$

(a)

(b)

Block diagram algebra

$$X_1(s) = R(s)G_1(s)$$

$$X_2(s) = R(s)G_2(s)$$

$$C(s) = [\pm G_1(s) \pm G_2(s) \pm G_3(s)]R(s)$$

$$X_3(s) = R(s)G_3(s)$$

(a)

$$\pm G_1(s) \pm G_2(s) \pm G_3(s)$$

(b)

Block diagram algebra

(a)

(b)

Block diagram algebra

(a)

(b)
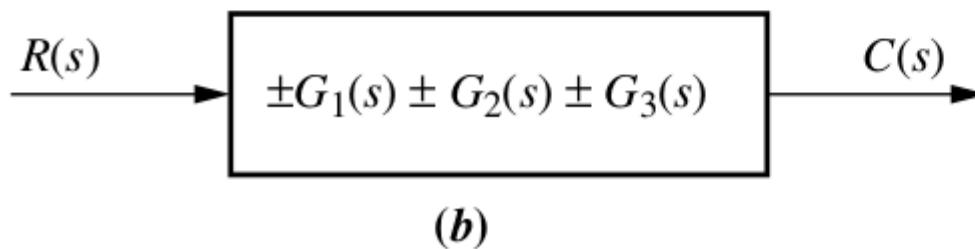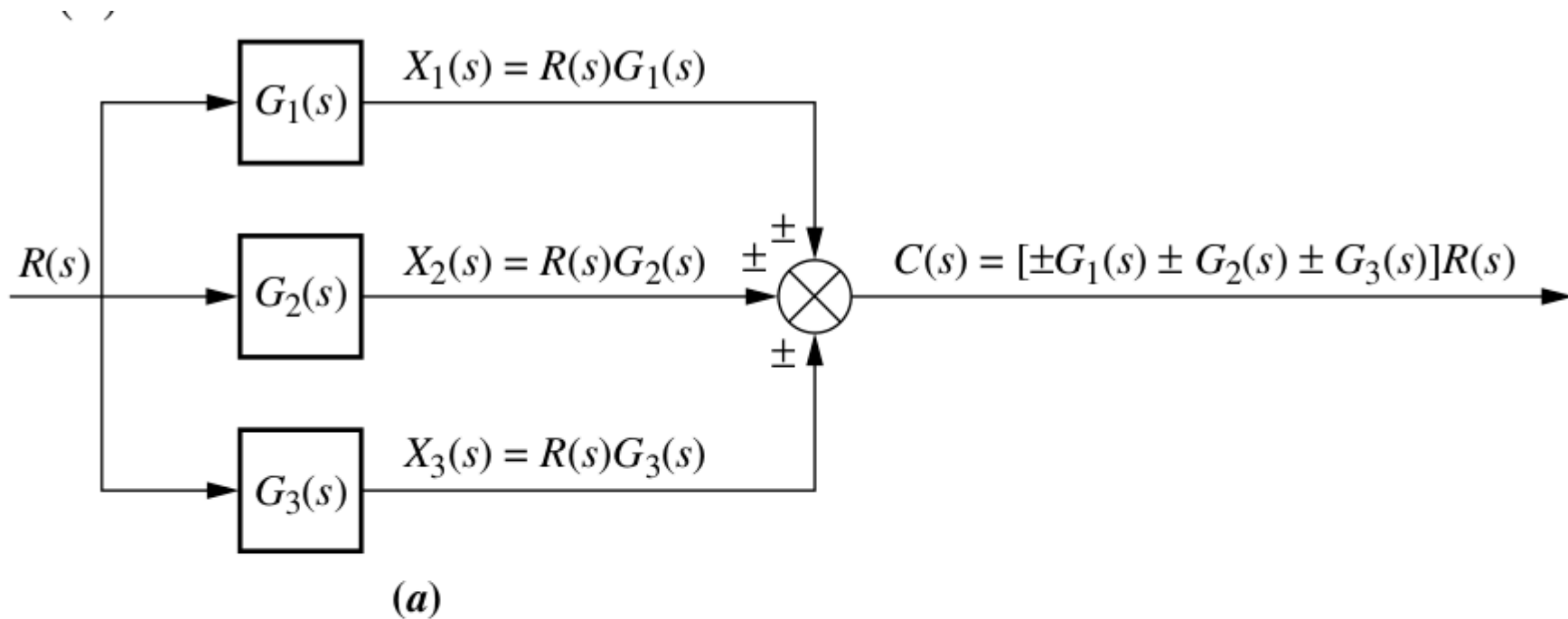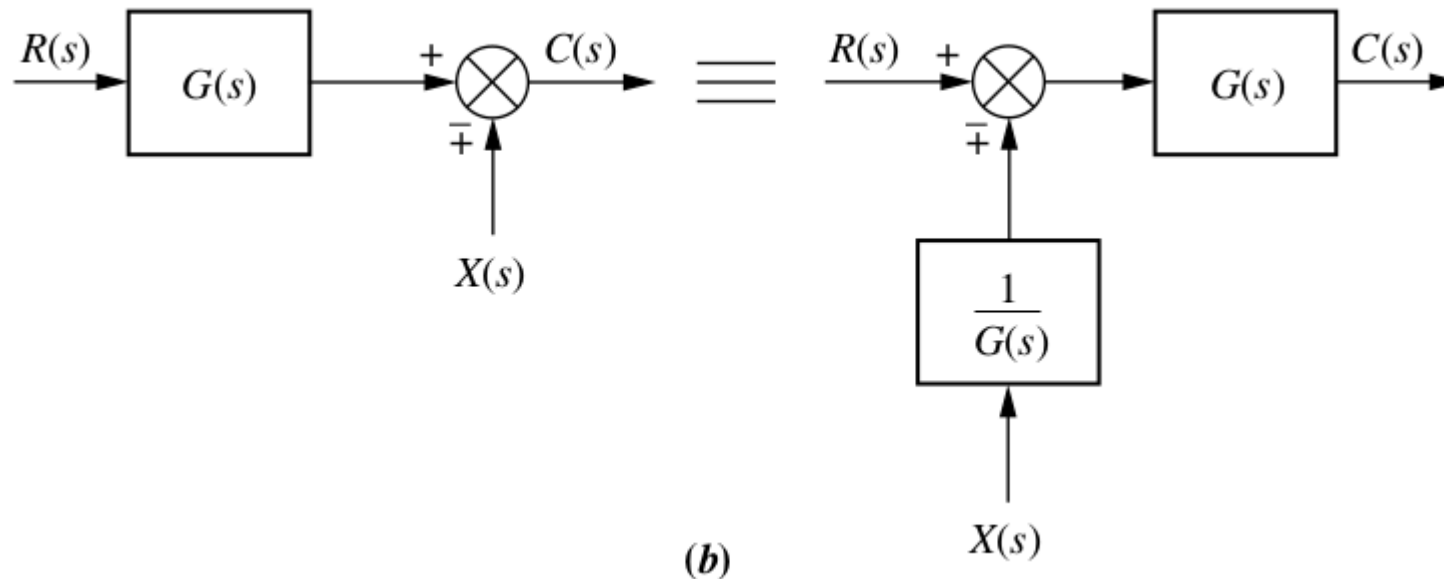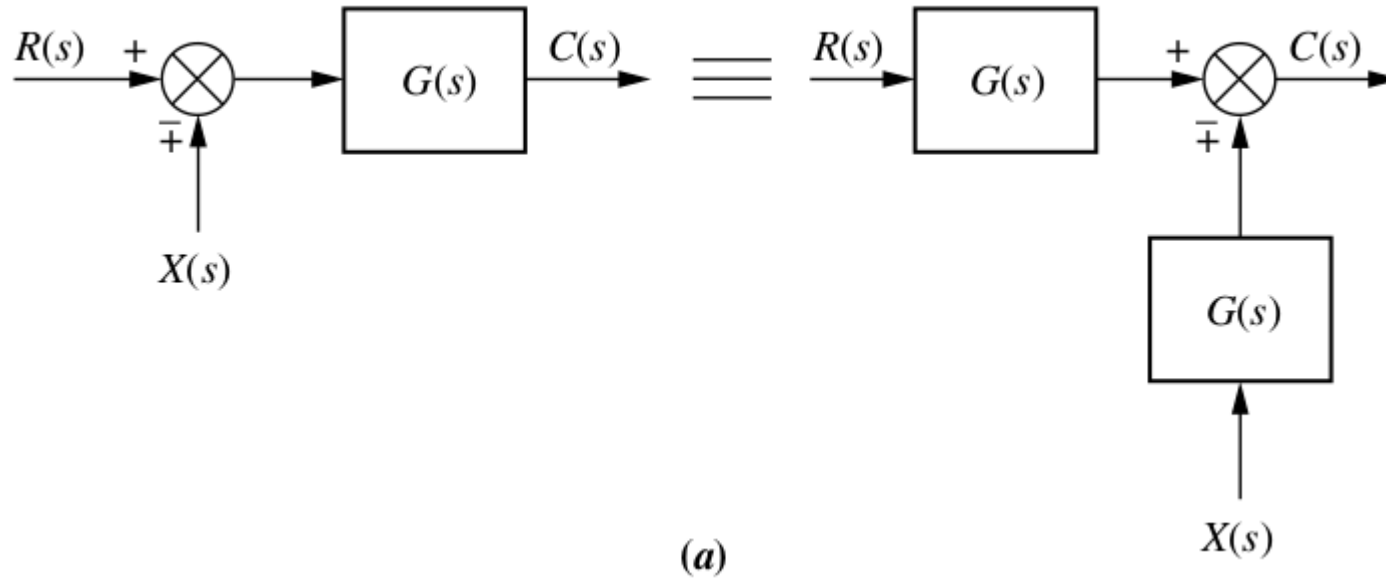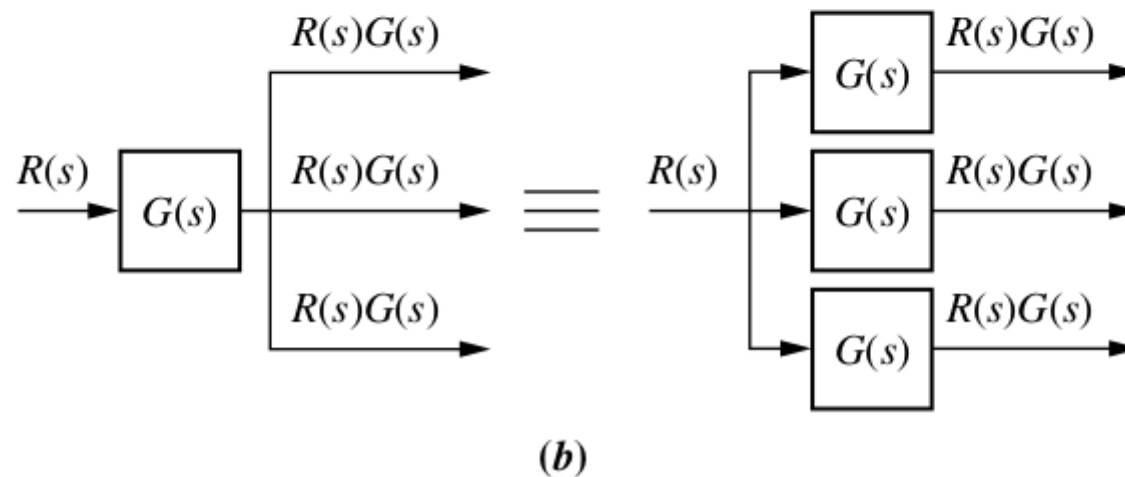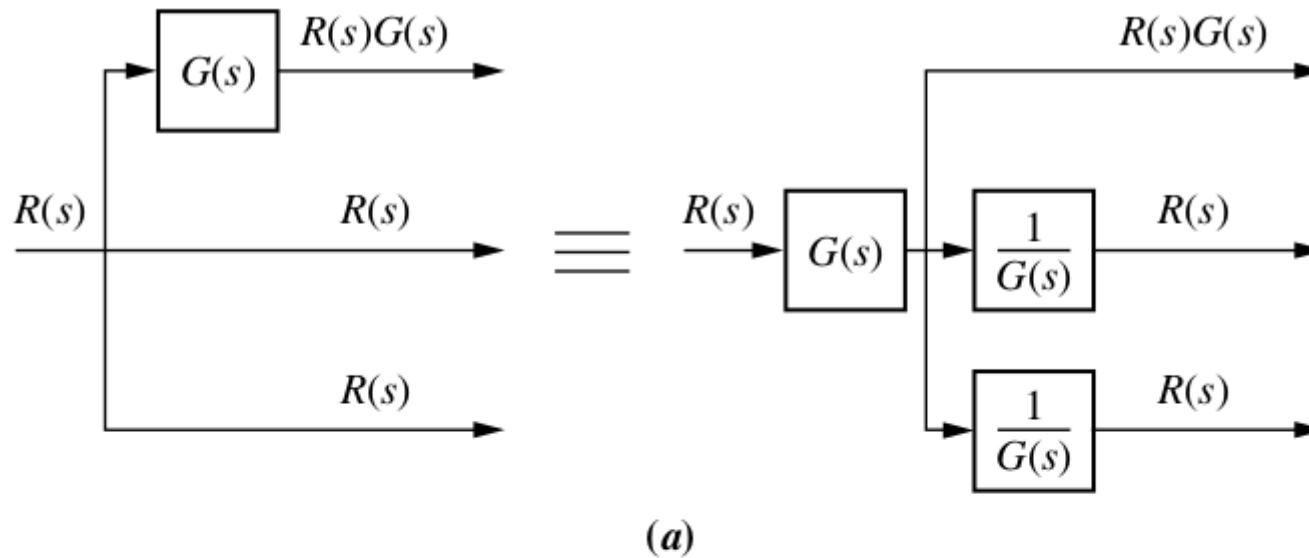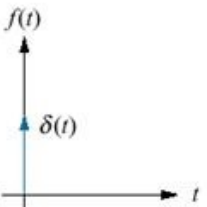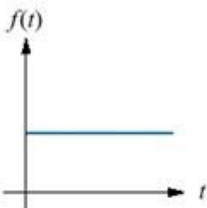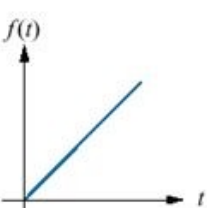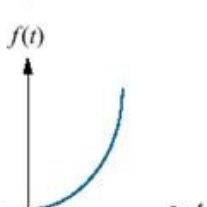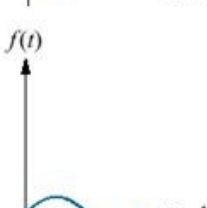
Block diagram algebra

PROBLEM: Reduce the block diagram shown in Figure 5.9 to a single transfer function.
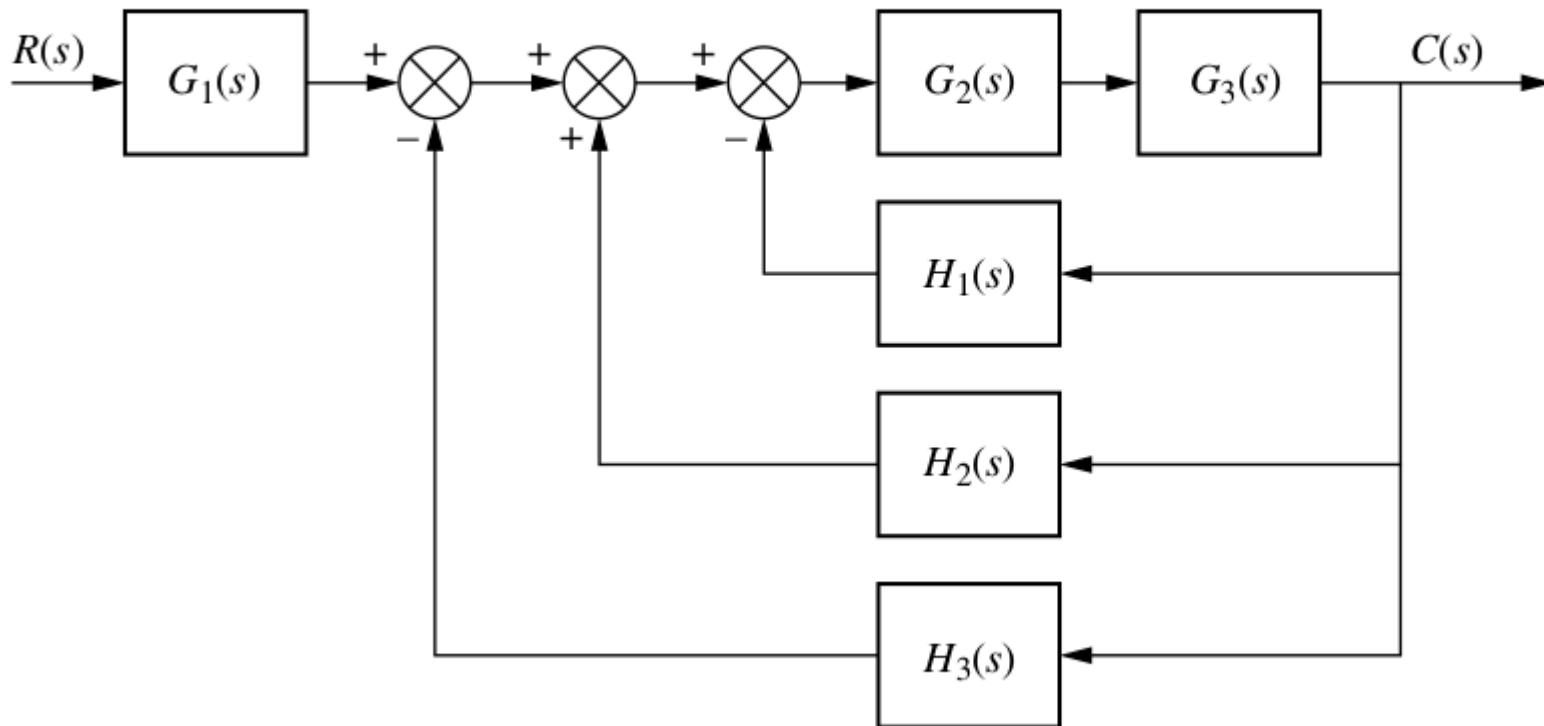
Test waveforms

Input	Function	Description	Sketch	Use
Impulse	$\delta(t)$	$\delta(t) = \infty$ for $0- < t < 0+$ $= 0$ elsewhere $\int_{0-}^{0+} \delta(t)\,dt = 1$		Transient response Modeling
Step	$u(t)$	$u(t) = 1$ for $t > 0$ $= 0$ for $t < 0$		Transient response Steady-state error
Ramp	$tu(t)$	$tu(t) = t$ for $t \geq 0$ $= 0$ elsewhere		Steady-state error
Parabola	$\frac{1}{2}t^2 u(t)$	$\frac{1}{2}t^2 u(t) = \frac{1}{2}t^2$ for $t \geq 0$ $= 0$ elsewhere		Steady-state error
Sinusoid	$\sin \omega t$			Transient response Modeling Steady-state error

Poles and step response

$$r(t) = u(t) \qquad G_1(s) = \frac{9}{s^2 + 9s + 9}$$

Overdamped

Poles and step response

ieeta
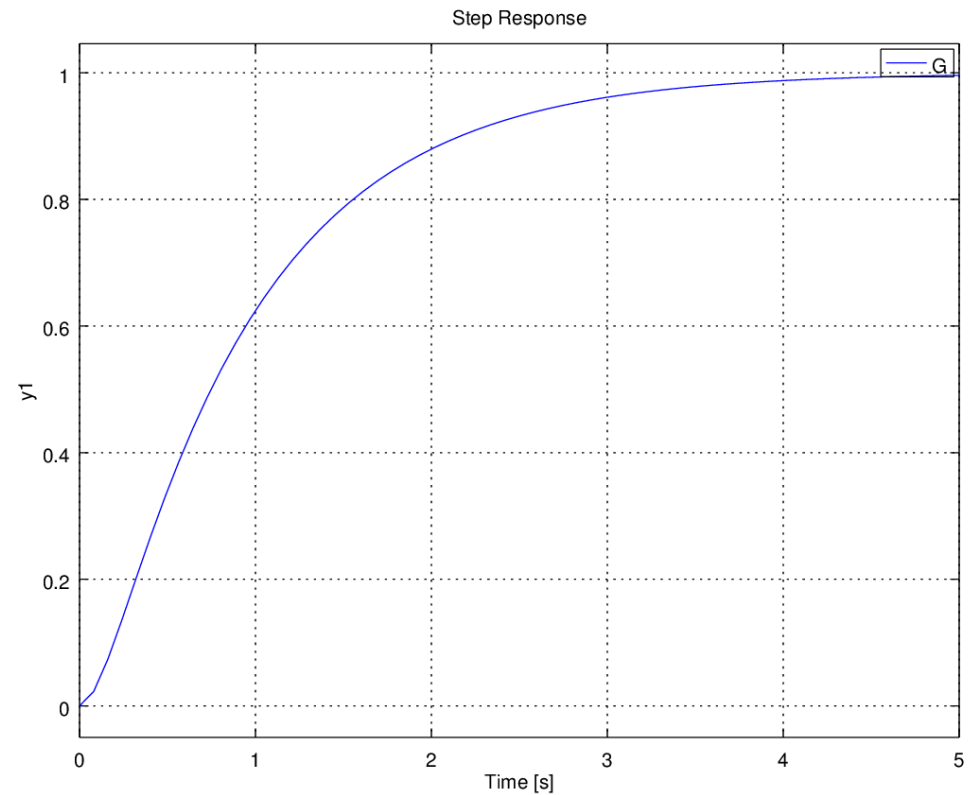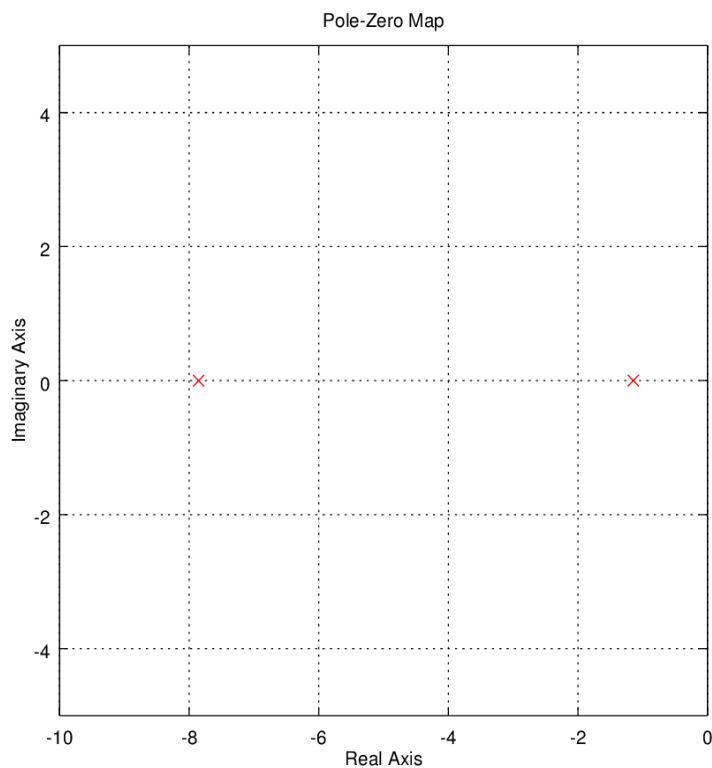
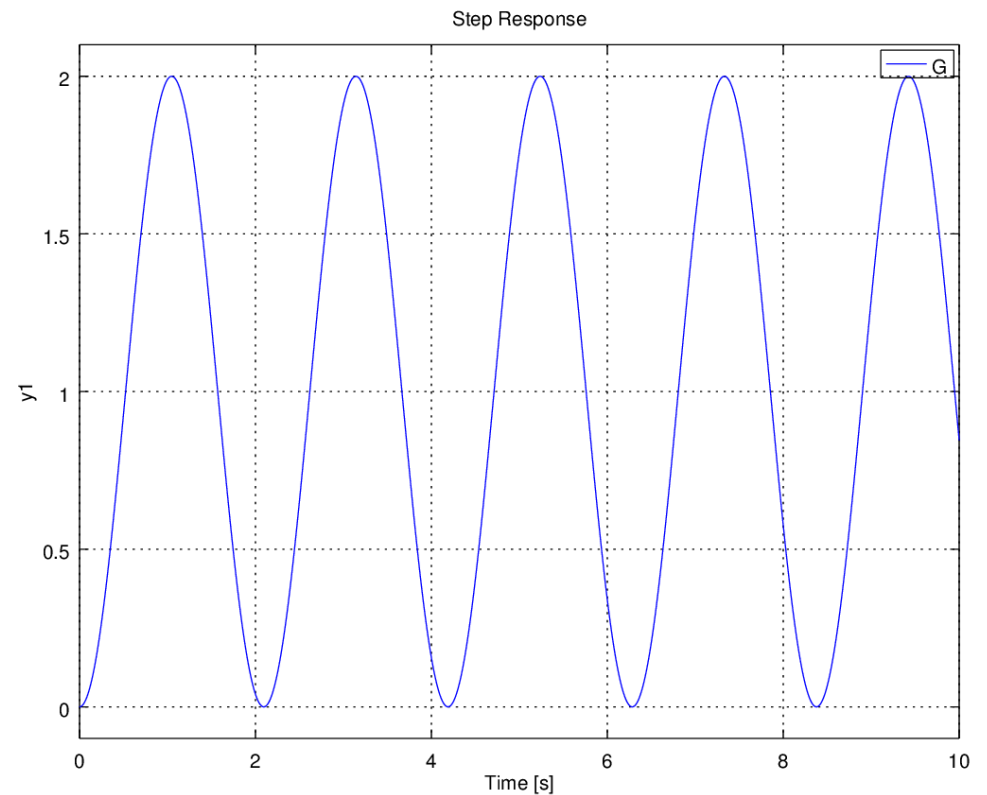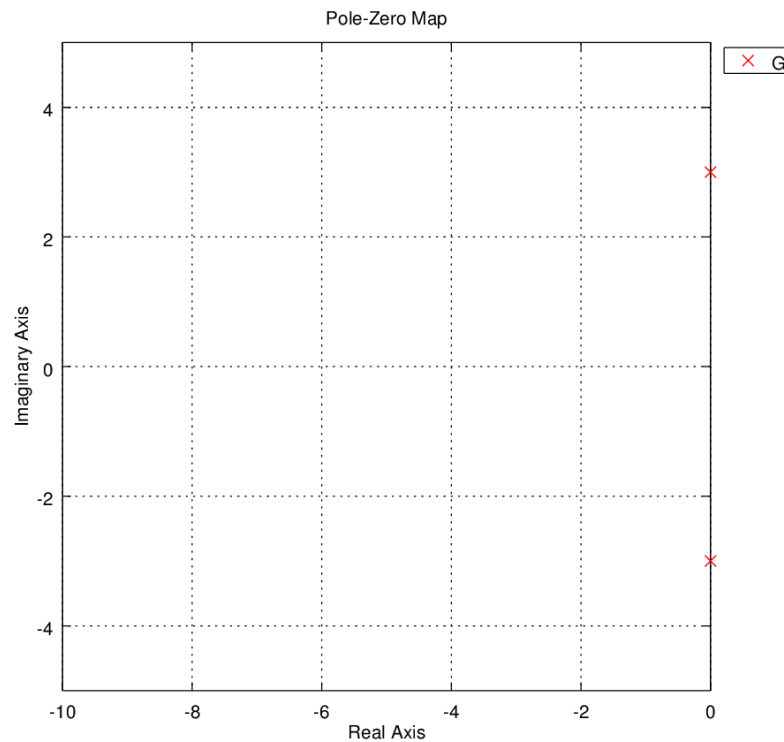$$r(t)=u(t) \qquad G_2(s)=\frac{9}{s^2+2s+9}$$

Underdamped
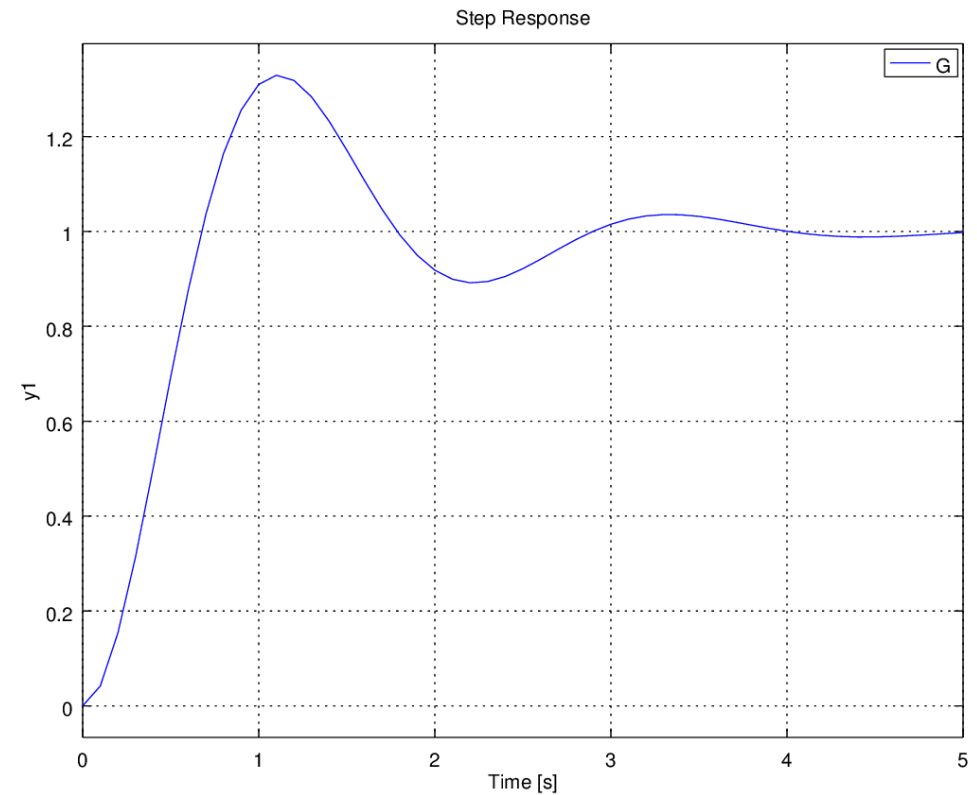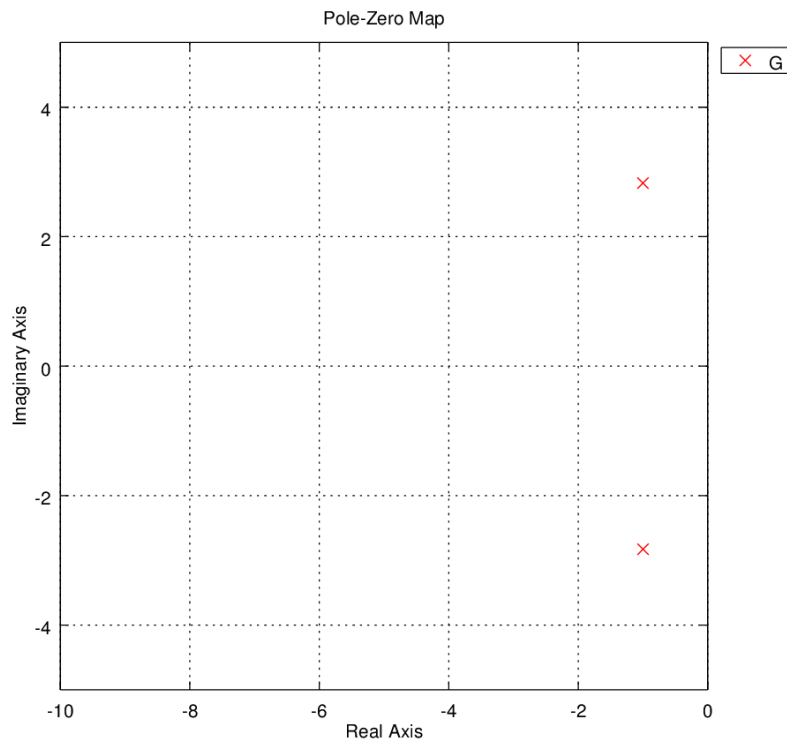
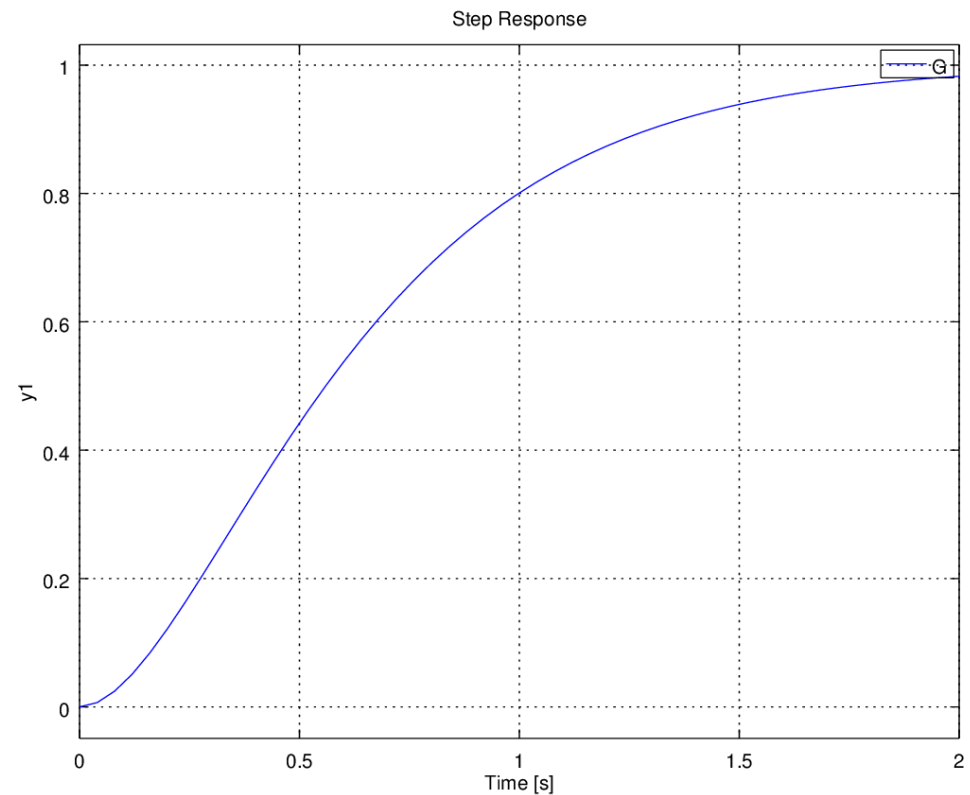# Poles and step response
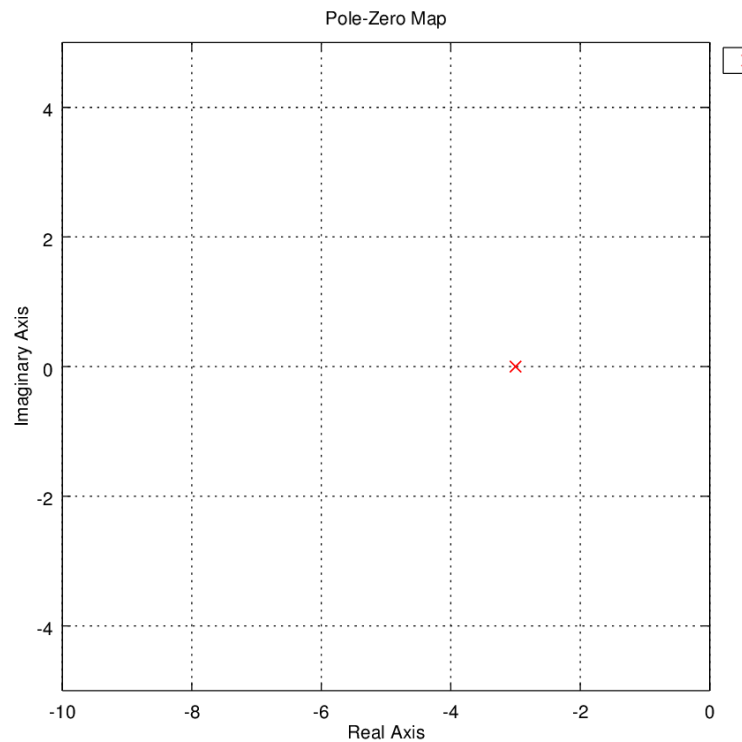
$$r(t)=u(t) \qquad G_3(s)=\frac{9}{s^2+9}$$

Undamped

Poles and step response

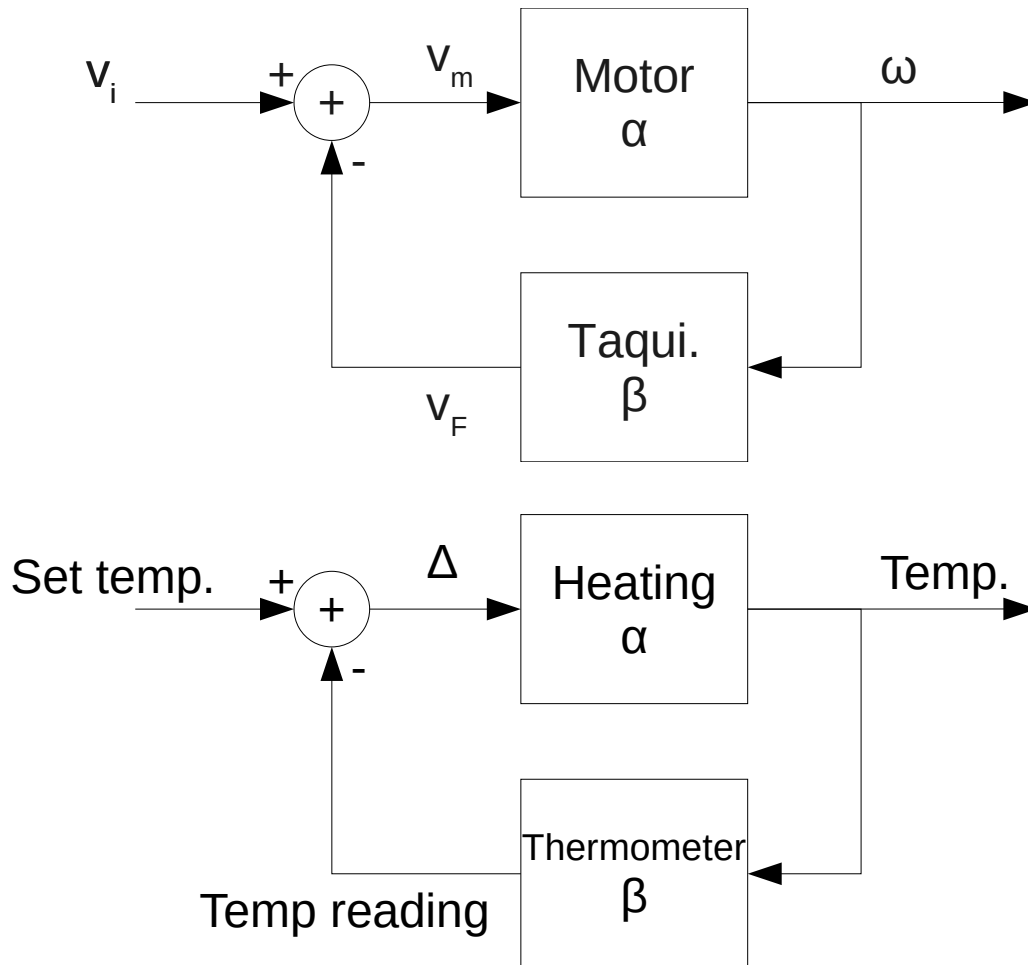$$r(t) = u(t) \qquad G_4(s) = \frac{9}{s^2 + 6s + 9}$$
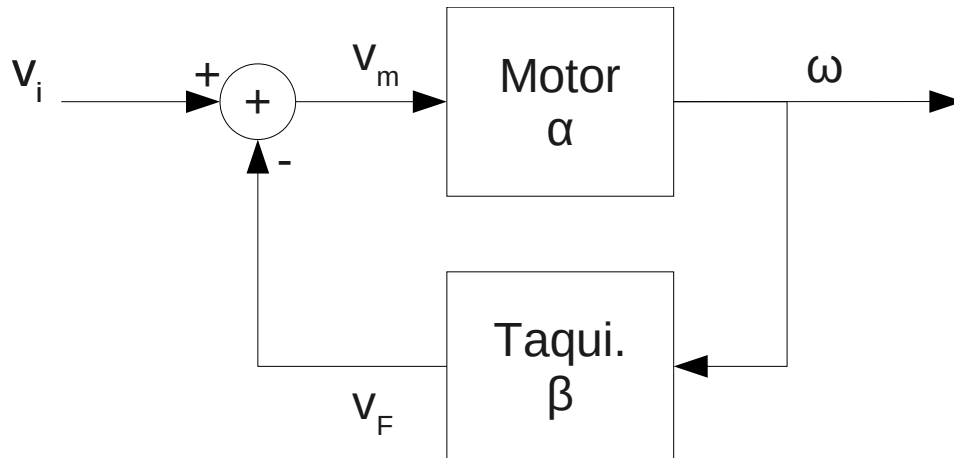
Critically damped

Video

feedback

- **a complex system is represented as a collection of interconnected set of simpler systems**
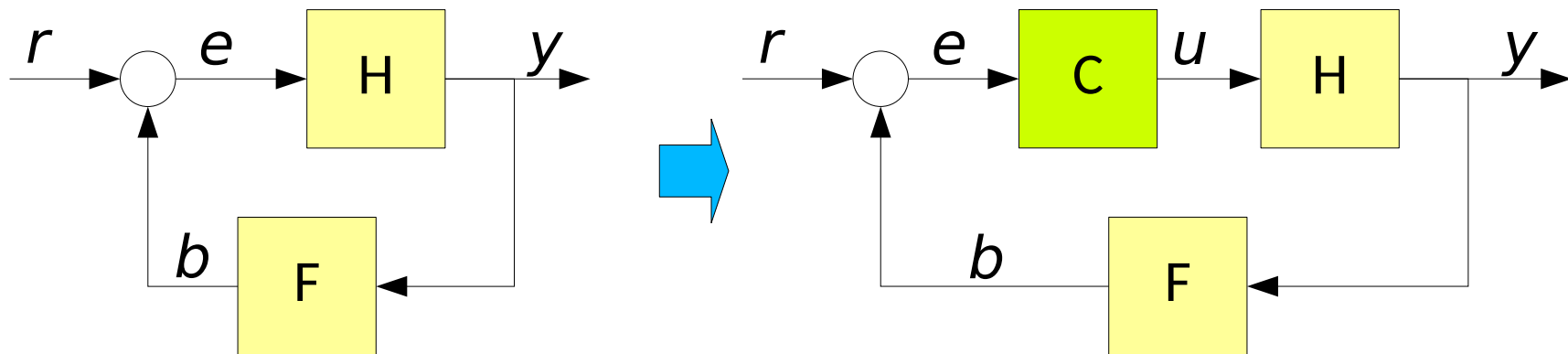 - each simple system has a known transfer function

feedback

$$\begin{cases} v_m = v_i - v_T \\ \omega = \alpha\, v_m \\ v_F = \beta\, \omega \end{cases} \Rightarrow \omega = \frac{\alpha}{1 + \alpha\beta}\, v_i$$

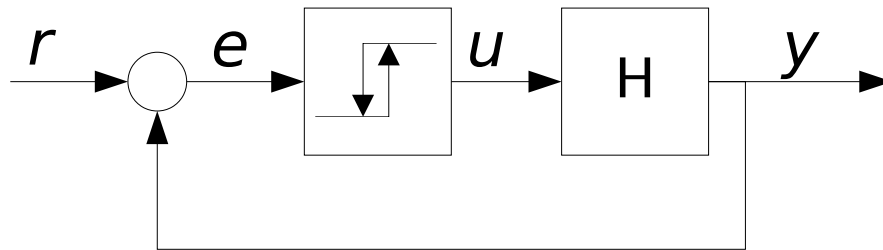$$\lim_{\alpha \to \infty} \frac{\alpha}{1 + \alpha\beta} = \frac{1}{\beta}$$

- **for high values of α, the output value will depend mainly on the feedback**

controller

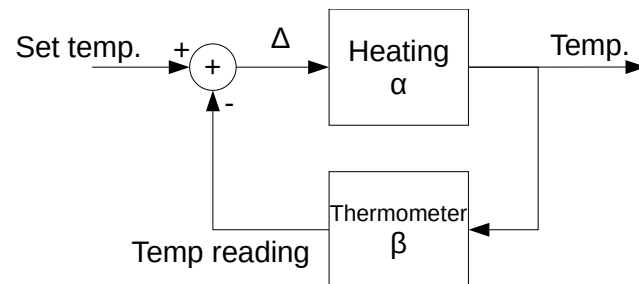- *Controller C: included to improve the system response*

r: reference (input)
e: error
u: system input / control
y: system output
b: feedback

On-Off (Bang-bang)

- **Acting signal *u* goes on and off depending on error**
- **Example:**
  - home temperature control
- **May have hysteresis**
- **Usually not so good behaviour, but…**
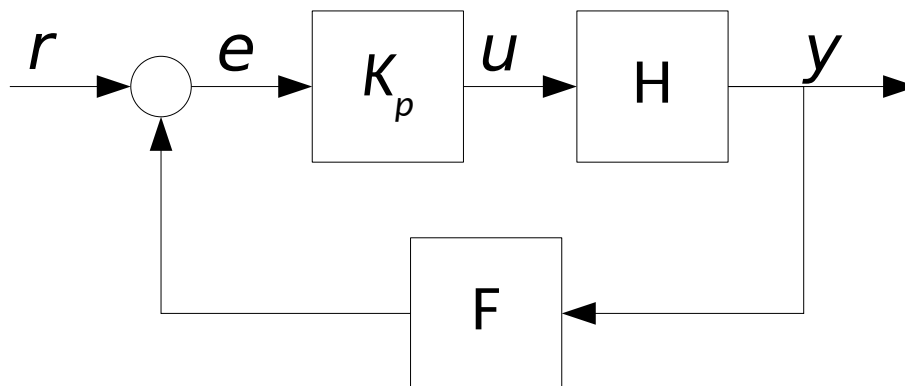- **Easy to implement (switch on and off...)**

P controller

- **P = "proportional". Simplest form of linear controller**
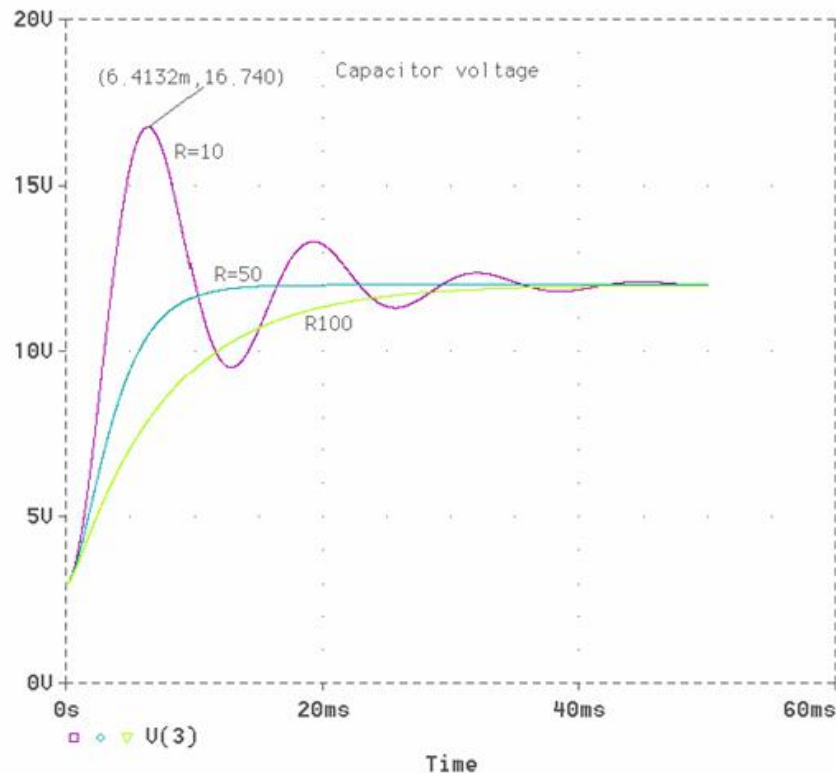
  $u = K_p\ e$

- **Gain: K_p**

The larger K_p, the smaller e for the same output.

- Kp low: soft system
  - it takes a large value of error for system to react
- Kp high: hard system
  - strong reaction, even with small values of e.

$u \neq 0$ iif $e \neq 0$

To reduce error, a high value of Kp is required

P controller

- **Increasing gain Kp reduces error, but...**
- **High values of gain Kp may cause the system to be unstable**
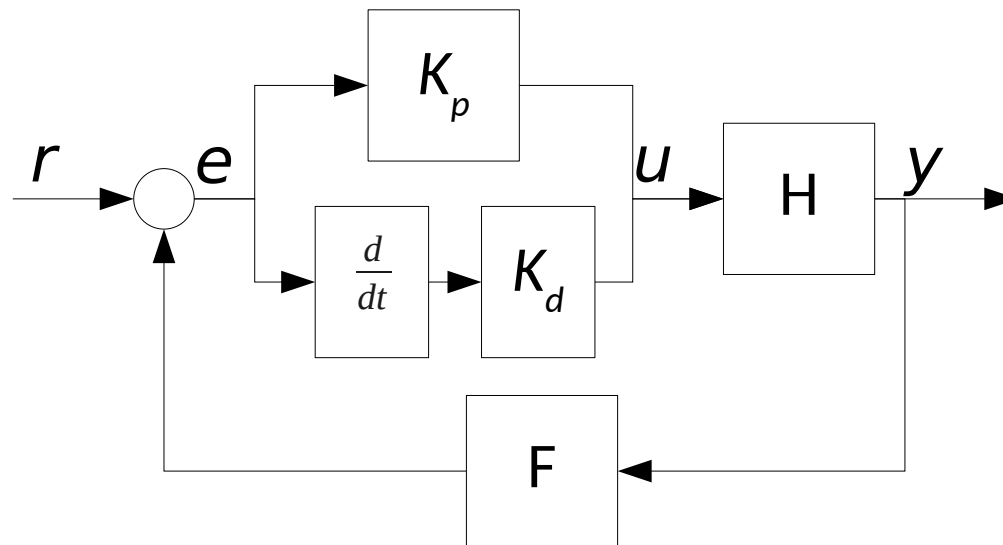
PI controller

- **PI: Proportional + Integral**
  - Adding a term $K_i \int e \, dt$.

  - PI controller allows for systems with $e=0$
  - Problem: inertia (memory effect)
    - with rapid changes in the input, u may be at a value when the error e would require to be different
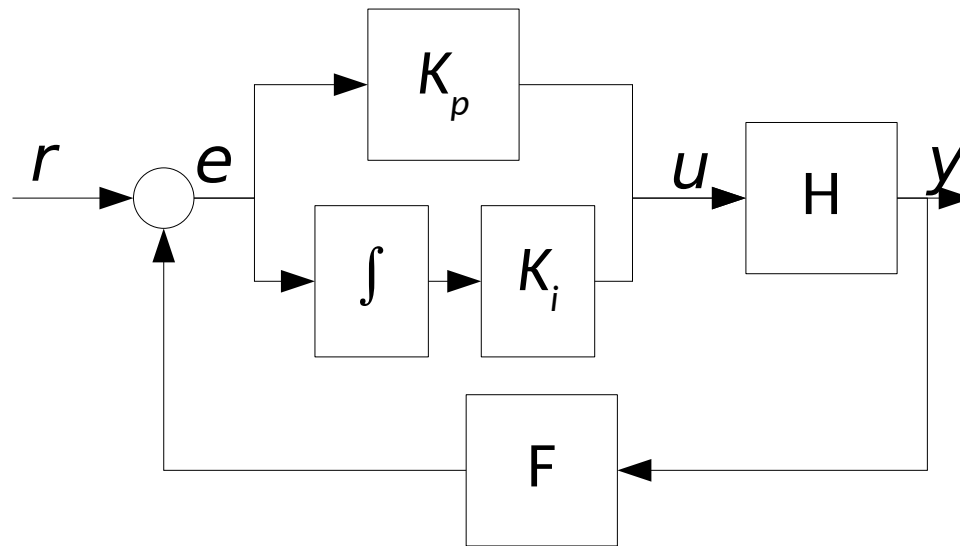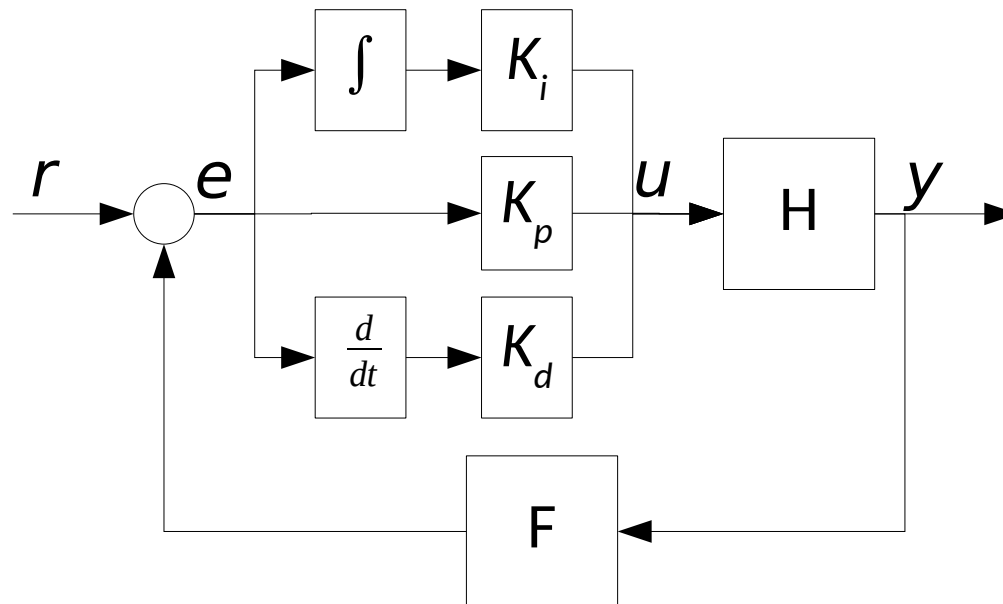
PD controller

- **PD = Proportional + Derivative**
  - adding a term proportional to the error derivative
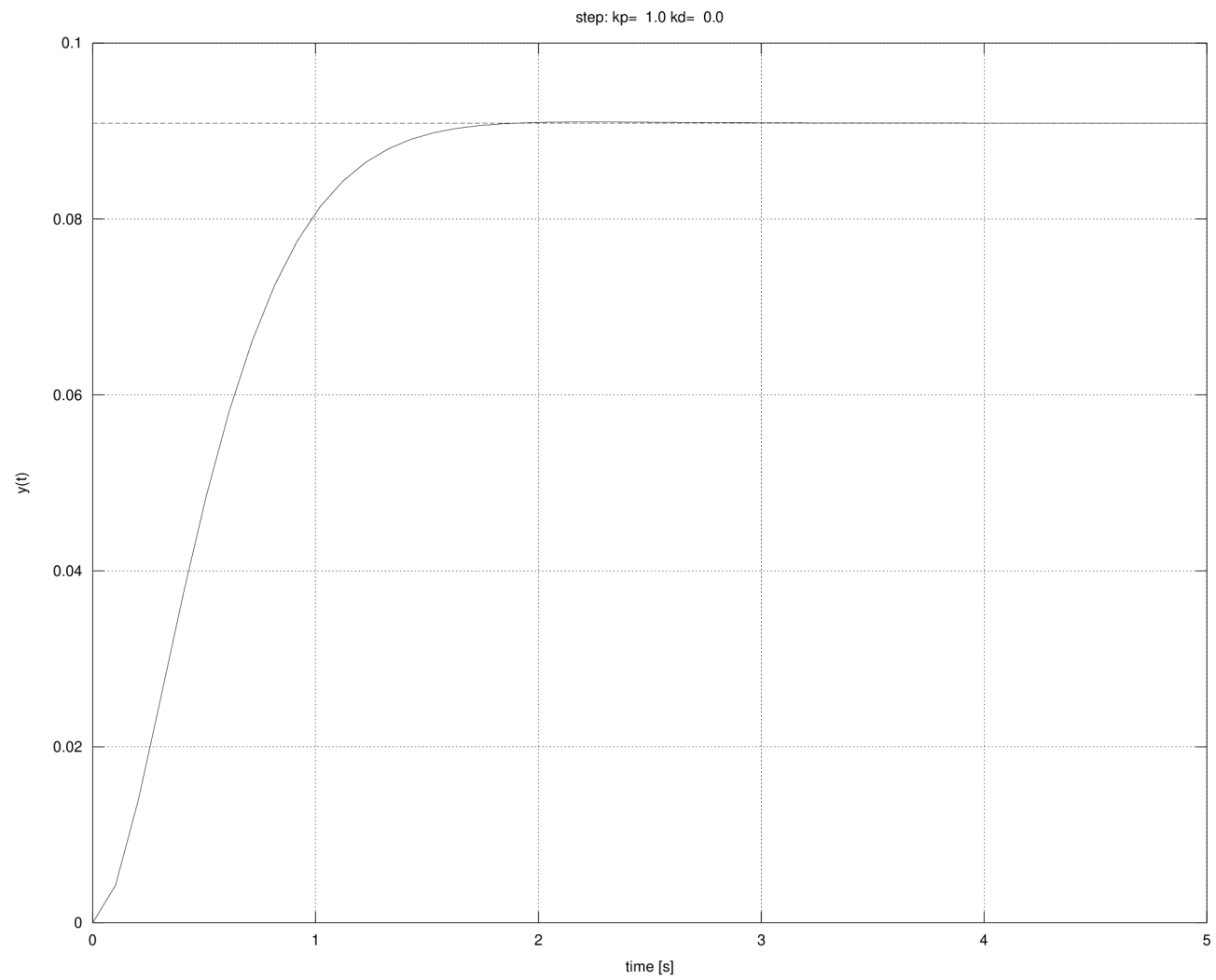 - In general, it has the effect of reducing oscillations (damper)

PID controller

- **PID = Proportional + Integral + Derivative**
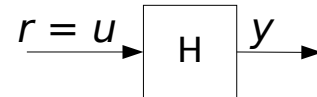 - Reunion of previous controllers
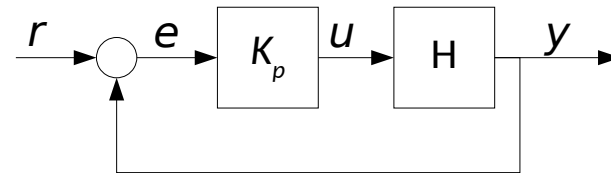 - One of the most popular controllers
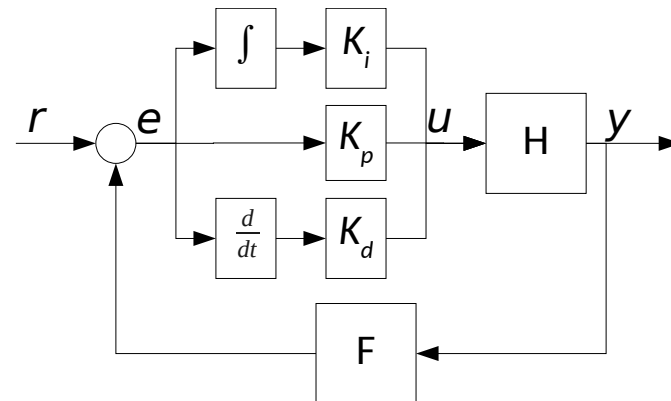
step: kp= 1.0 kd= 0.0

Coding a PID controller

Controller demo

NONE

P

PID

BANG,
BANG2,
BANGH

git@github.com:iris-ua/ciberRatoTools.git
branch: speedc