

# ENGG102 Fundamentals of Engineering Mechanics

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VISIONARY / PASSIONATE / DYNAMIC  
CONNECT:  
UNIVERSITY OF WOLLONGONG

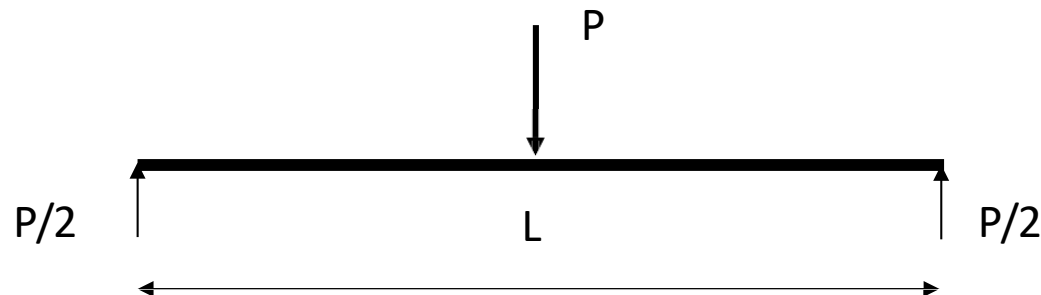
**UOWD**   
University of Wollongong in Dubai

## Outline:

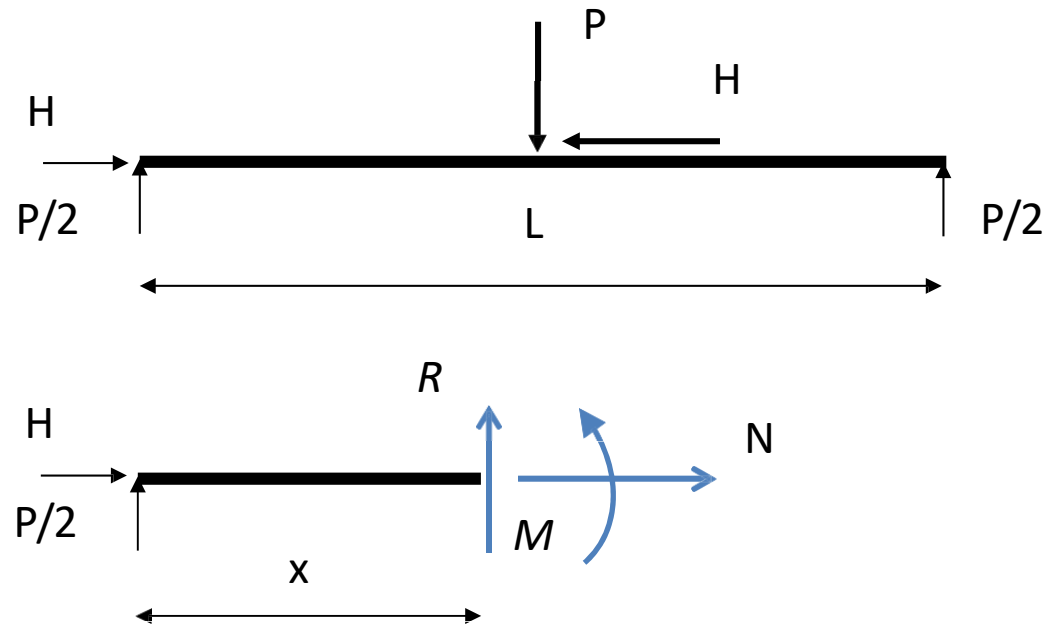
1. Recap on last lecture
2. Revision of Internal Forces in Structural Members
3. Shear and Bending Moment Equations and Diagrams
4. Relationship between Shear and Bending Moment

## Recap on Lecture 3

- Making FBD of part of the beam
- Allows us to determine INTERNAL forces and moments

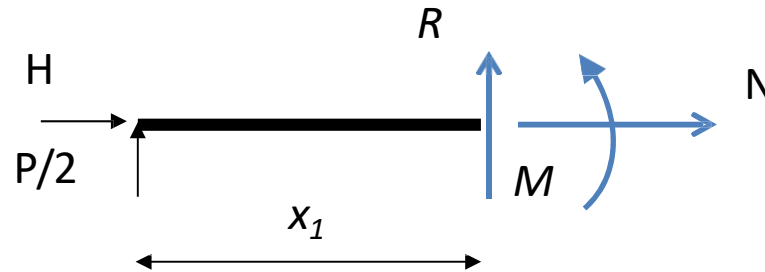


# General Case



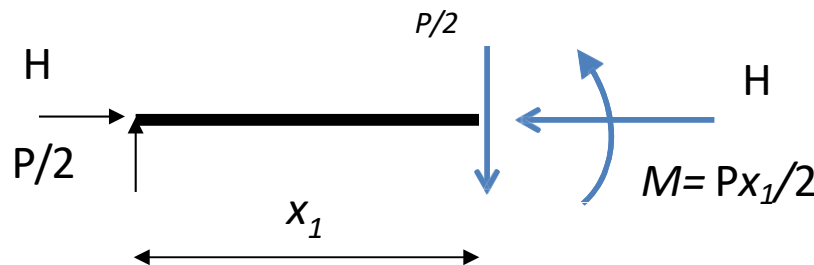
- When we cut a solid beam element we replace the right hand part by THREE actions
- Axial Force, Moment and a transverse force  $R$

# Solving FBD

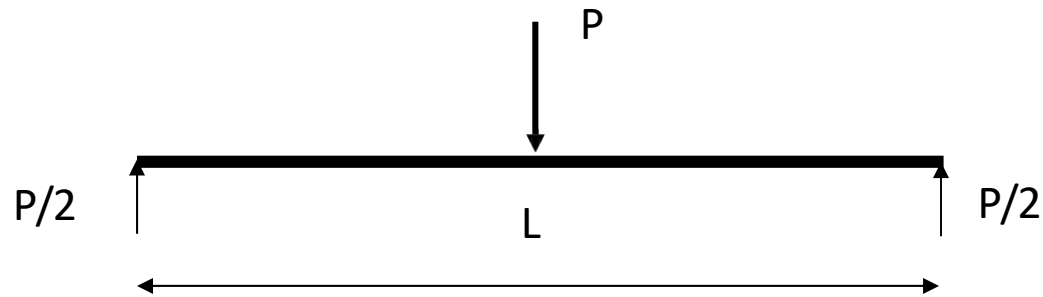


- $\Sigma F_x = 0$ :  $H + N = 0$  implies  $N = -H$
- $\Sigma F_y = 0$ :  $P/2 + R = 0$  implies  $R = -P/2$
- $\Sigma M_{x1} = 0$ :  $-(P/2) \cdot x_1 + M = 0$  implies  $M = Px_1/2$

- The N and R arrows are in the incorrect sense
- M is correct
- Final FBD



# Our Balsa Beam

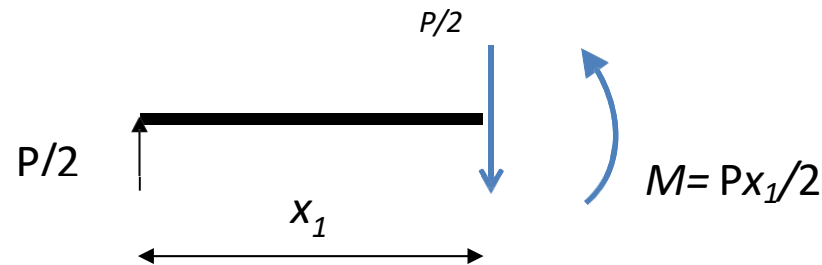


- $\Sigma F_x = 0$ :  $H = N = 0$
- $\Sigma F_y = 0$ :  $P/2 + R = 0$  implies  $R = -P/2$
- $\Sigma M_{x1} = 0$ :  $-(P/2) \cdot x_1 + M = 0$  implies  $M = Px_1/2$

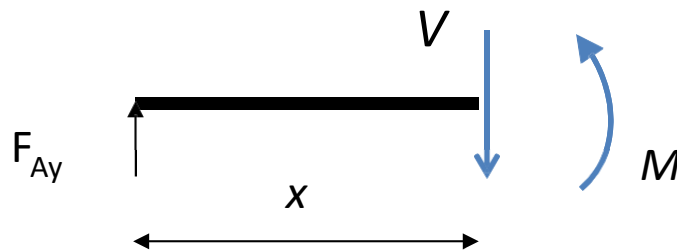
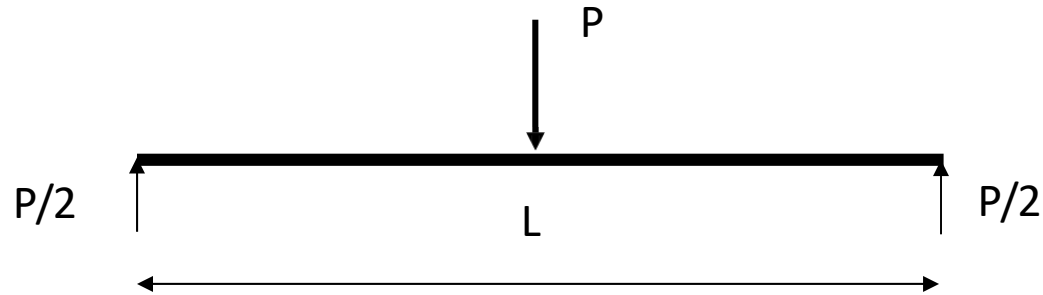
• Final FBD

• At  $x_1 = L/2$

•  $M = PL/4$

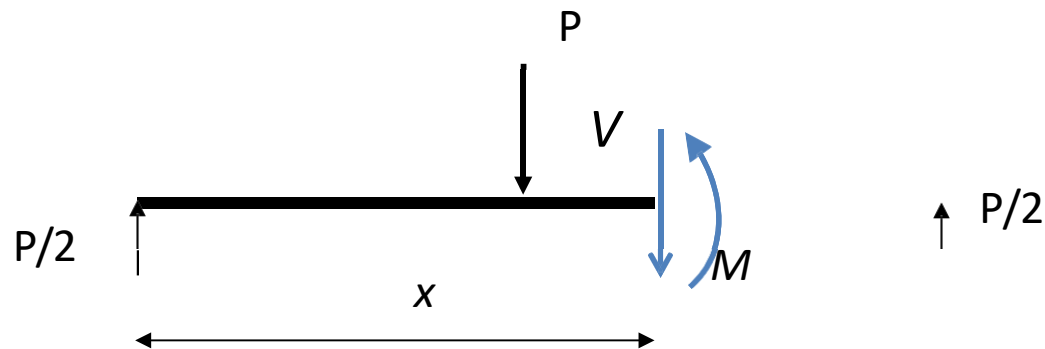
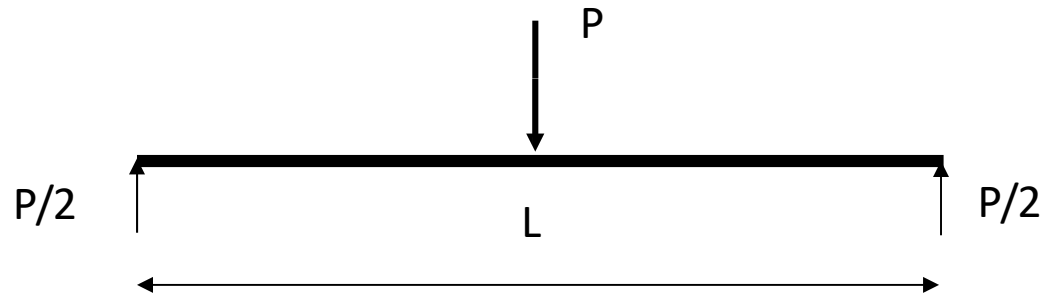


# General FBD for Beam with point load



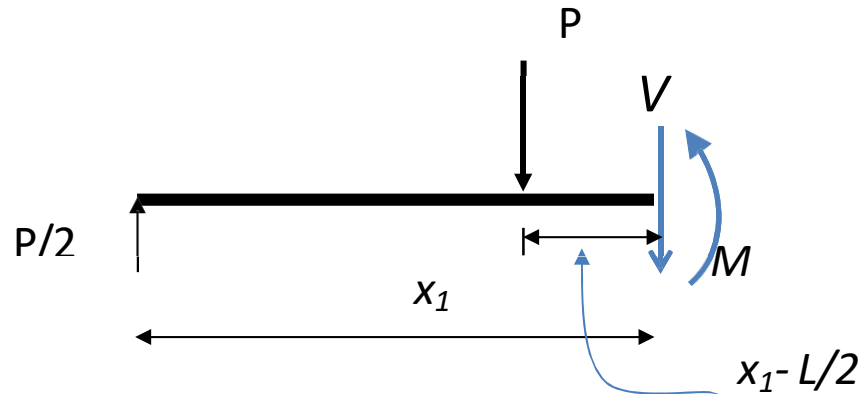
We have a special name for the internal moment: We call it the BENDING MOMENT because it makes the beam bend

What if we made out cut to right of P?





What if we made out cut to right of P?



- $\Sigma F_x = 0$ :  $N = 0$
- $\Sigma F_y = 0$ :  $P/2 - P - V = 0$  implies  $V = -P/2$  (direction wrong)
- $\Sigma M_{x1} = 0$ :  $-(P/2) \cdot x_1 + P(x_1 - L/2) + M = 0$

implies  $M = PL/2 - Px_1/2$  Note When  $x_1 = L/2$   $M = PL/2 - PL/4 = PL/4$

- These equations are true  $\forall x \in [L/2 < x \leq L]$

# Shear and Bending Moment Equations and Diagrams

Bending Moment Diagram is **important** because:

1. Failure occurs where the bending moment is maximum
2. Deflection can be predicted by means of bending moment diagrams

*Beam failure from bending under snow overload.*



17 [www.structurearchives.org](http://www.structurearchives.org)

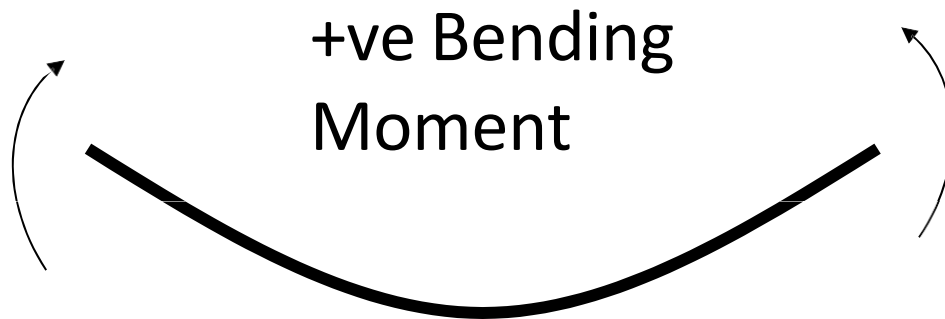
*Beam failure – (in middle again)*



[www.frp distributors.com](http://www.frp distributors.com)

# Shear and Bending Moment Equations and Diagrams

Sign convention for **Bending Moment Diagram**:



+ve Bending  
Moment

Sagging moment

BMD is always a pair of equal  
but opposite moments



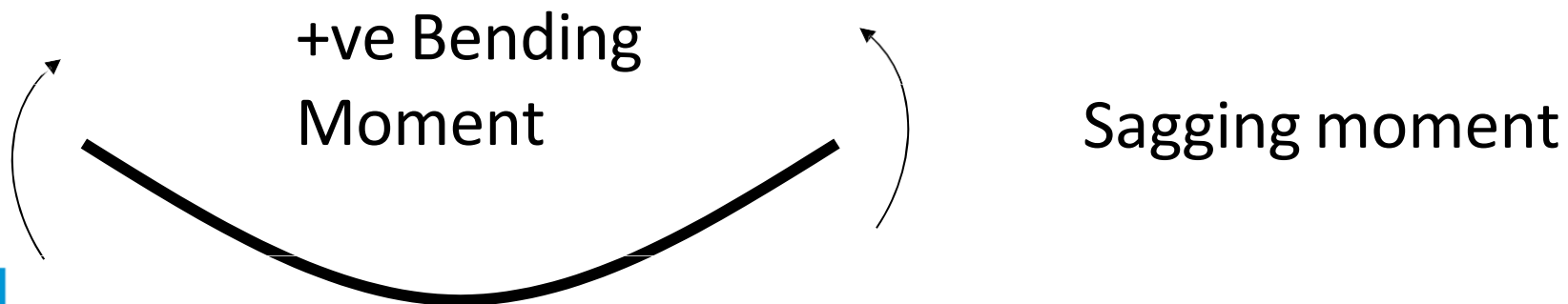
-ve Bending  
Moment

Hogging moment



# Shear and Bending Moment Equations and Diagrams

**You should not confuse the sign convention for Bending Moment Diagram with the sign convention in equilibrium equations.**



In equilibrium equations, the left end moment is normally considered negative as it has a clockwise direction.

# Shear and Bending Moment Equations and Diagrams

In the textbook:

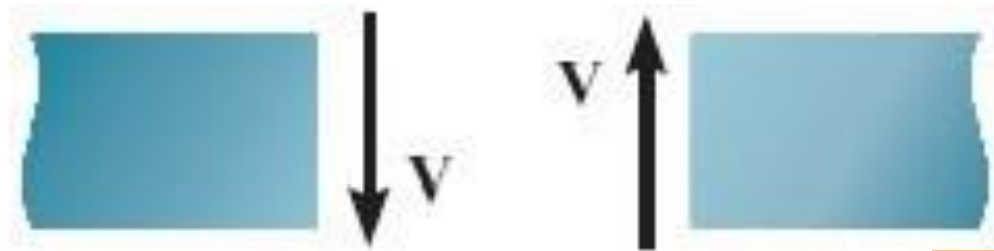


Sagging moment



## 7.2 Shear and Bending Moment Equations and Diagrams

Sign convention for **Shear Force Diagram**:



SFD is always a pair of equal but opposite transverse forces



Positive shear

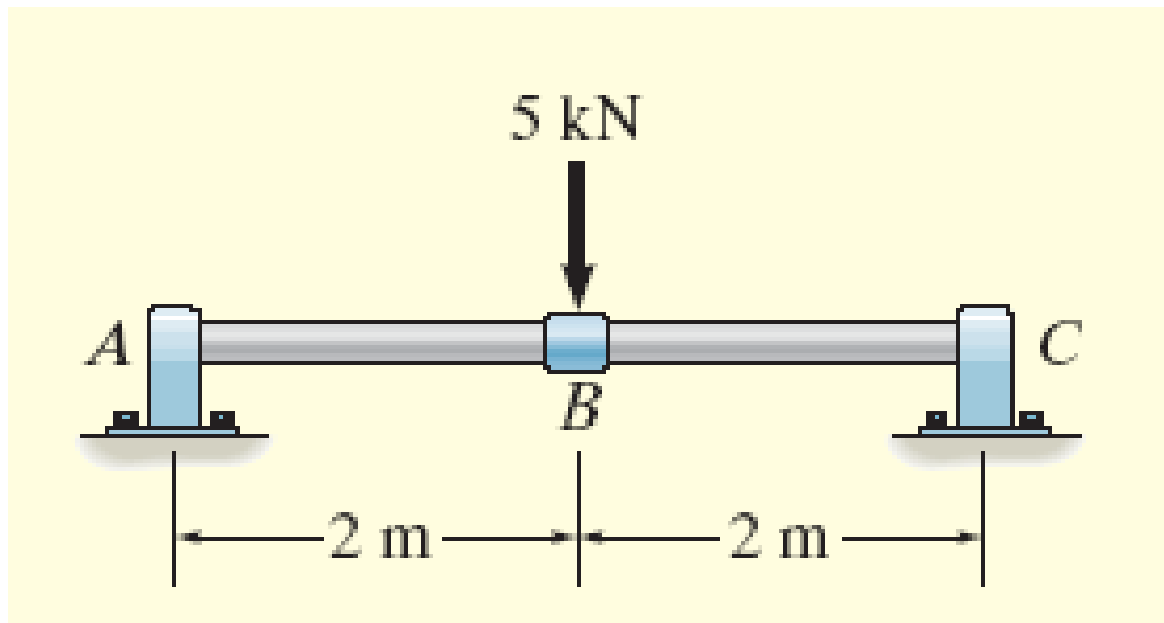
## Shear and Bending Moment Equations and Diagrams



In equilibrium equations, the right end shear force is normally considered negative as it has a downward direction.

## Shear and Bending Moment Equations and Diagrams

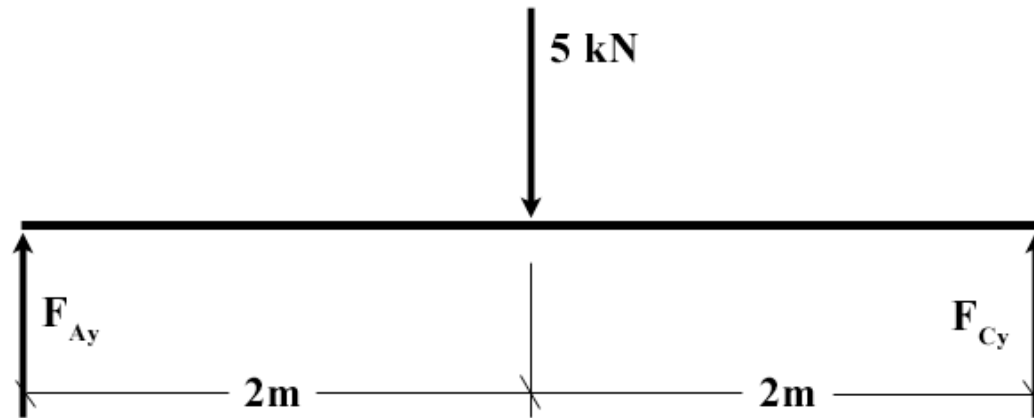
Draw the shear and bending moments diagrams for the shaft. The supports at A and C do not resist any moment.





# Shear and Bending Moment Equations and Diagrams

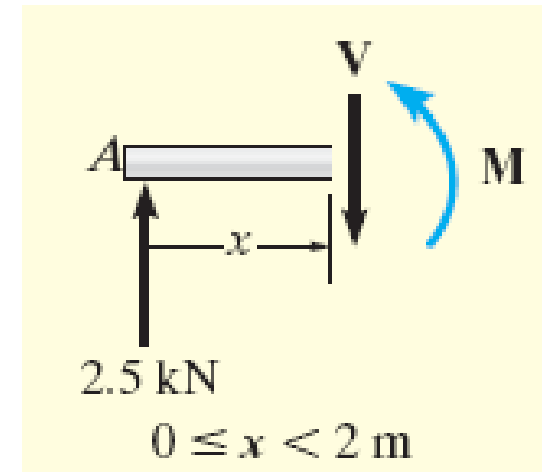
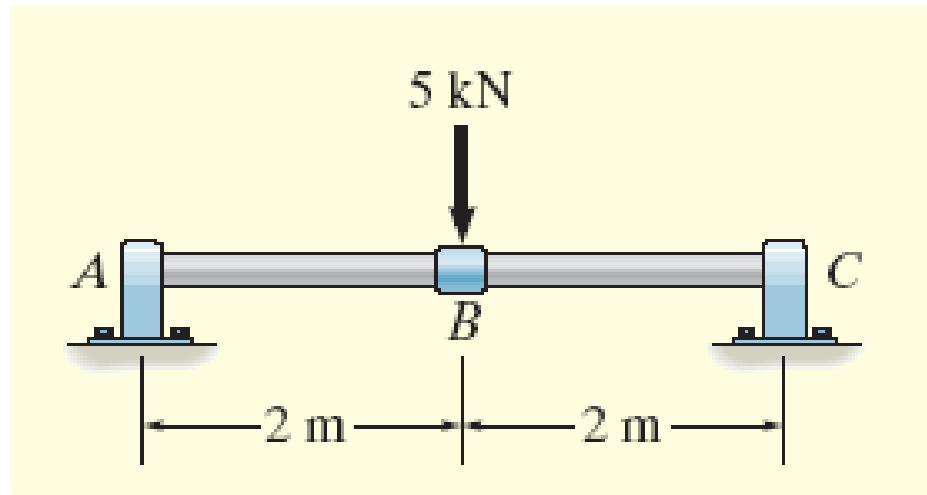
Draw free body diagram and calculate reactions:



$$\Sigma M_A = -(5 \text{ kN})(2 \text{ m}) + F_{Cy}(4 \text{ m}) = 0 : \quad F_{Cy} = 2.5 \text{ kN } (\uparrow)$$

$$\Sigma F_y = F_{Ay} - 5 \text{ kN} + F_{Cy} = 0 : \quad F_{Ay} = 2.5 \text{ kN } (\uparrow)$$

# Shear and Bending Moment Equations and Diagrams



We can also see that due to symmetry, the vertical reaction at A must be half the applied load, i.e. 2.5 kN upwards.

Looking at the FBD above for  $0 \leq x < 2 \text{ m}$ :

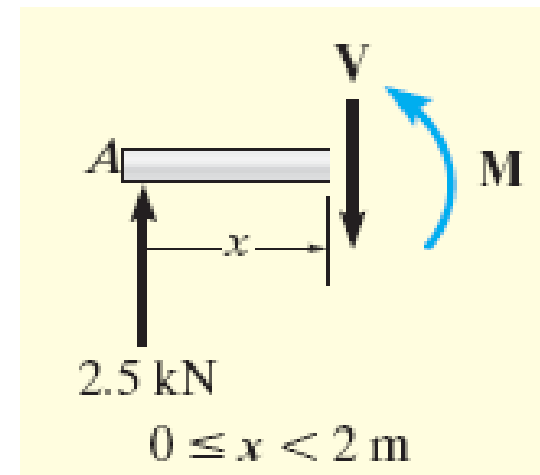
$$\sum F_y = 2.5 \text{ kN} - V = 0 \Rightarrow V = 2.5 \text{ kN}(\downarrow)$$

# Shear and Bending Moment Equations and Diagrams

$$\sum F_y = 2.5 \text{ kN} - V = 0 \Rightarrow V = 2.5 \text{ kN}(\downarrow)$$

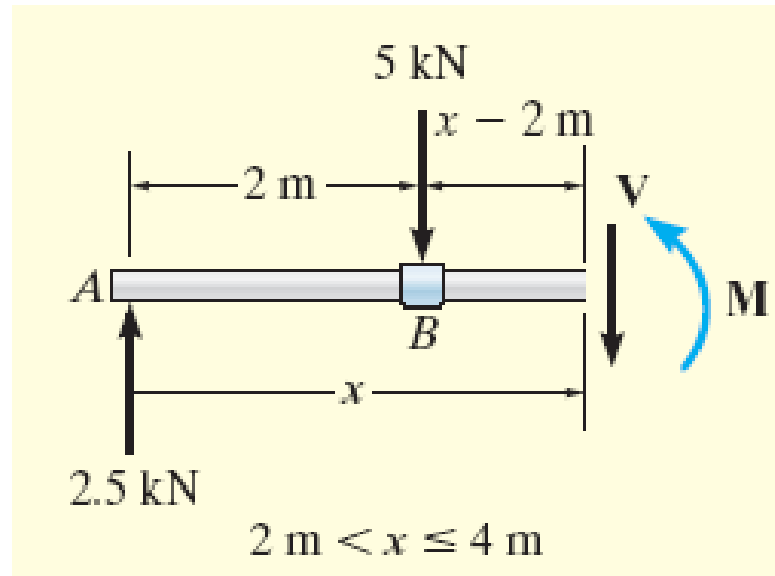
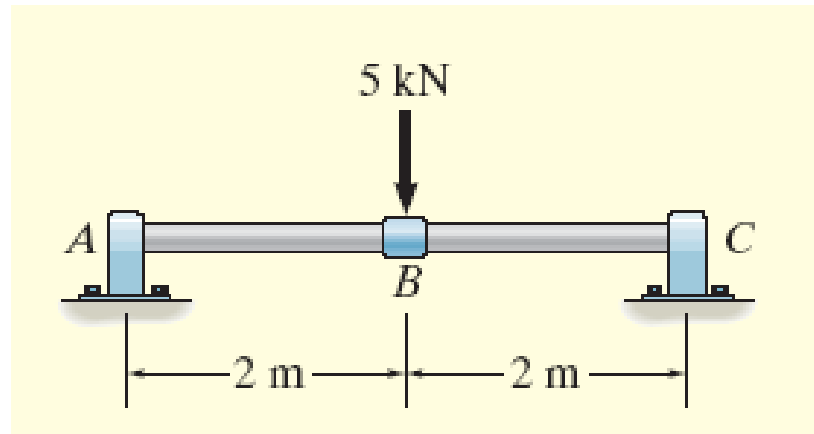


For  $0 \leq x < 2 \text{ m}$ , i.e. between A and B, the internal shear force diagram is *positive*.



Note: When adding all the forces we do not include  $M$  – because it is not a force!

# Shear and Bending Moment Equations and Diagrams



For  $2 \text{ m} < x \leq 4 \text{ m}$ , i.e. between B and C:

$$\sum F_y = 2.5 \text{ kN} - 5 \text{ kN} - V = 0 \Rightarrow V = -2.5 \text{ kN}(\uparrow)$$

# Shear and Bending Moment Equations and Diagrams

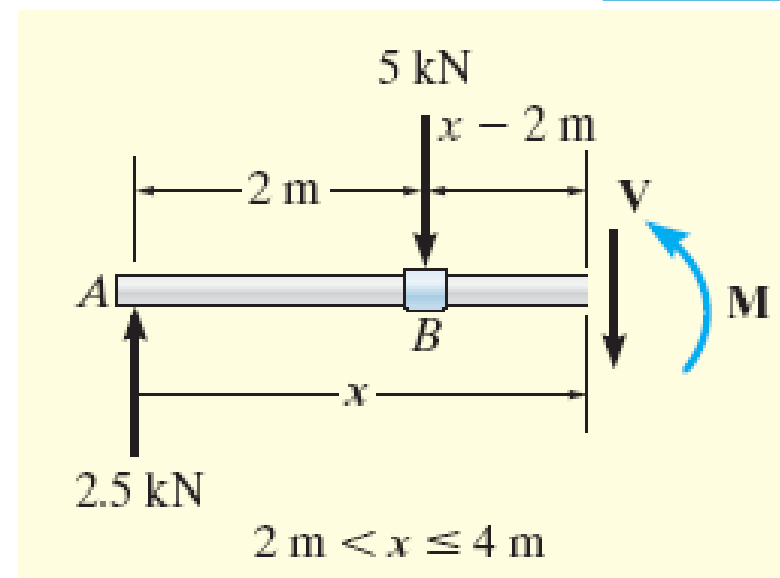
For  $2 \text{ m} < x \leq 4 \text{ m}$ :

$$\sum F_y = 2.5 \text{ kN} - 5 \text{ kN} - V = 0$$

$$V = -2.5 \text{ kN}(\uparrow)$$



Positive shear

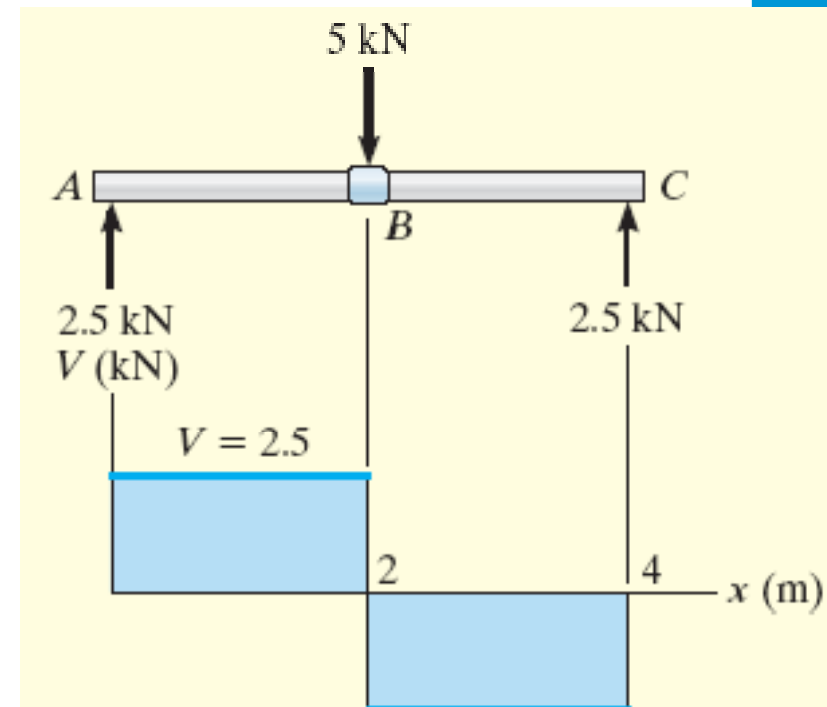


For  $2 \text{ m} < x \leq 4 \text{ m}$ , i.e. between B and C, the internal shear force diagram is *negative*.

## Shear and Bending Moment Equations and Diagrams

For  $0 \leq x < 2$  m, i.e. between A and B, the internal shear force diagram is positive.

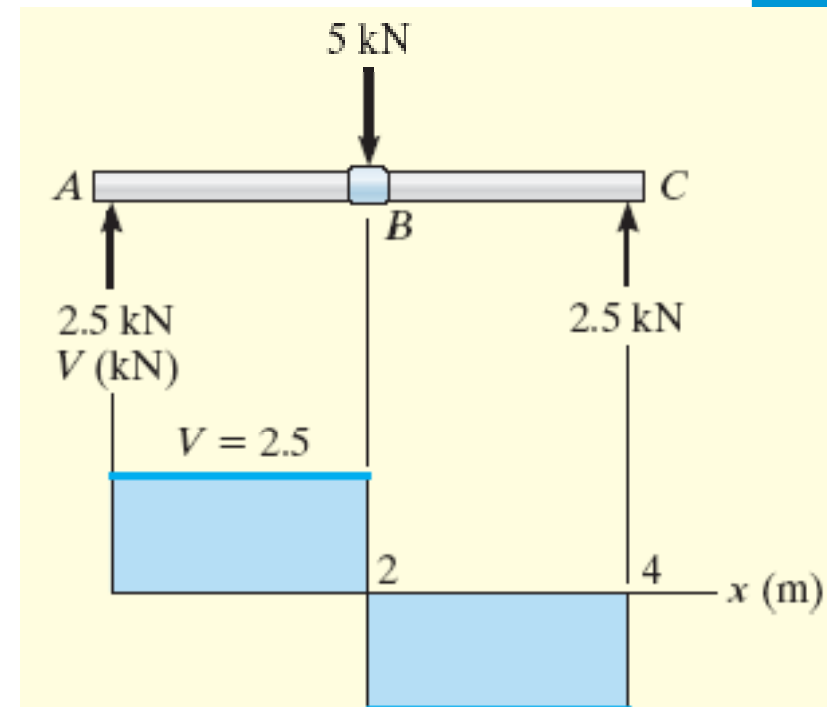
For  $2 \text{ m} < x \leq 4$  m, i.e. between B and C, the internal shear force diagram is negative.



Shear Force Diagram

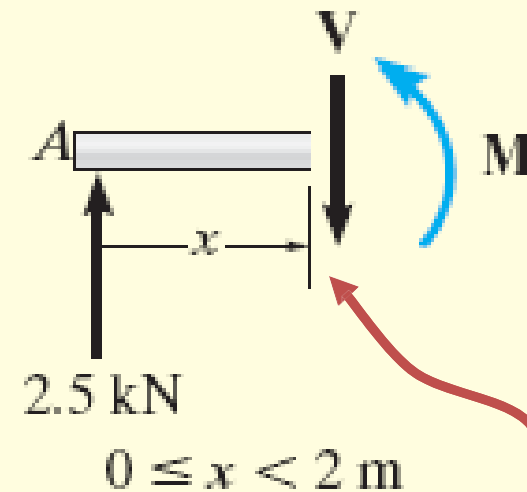
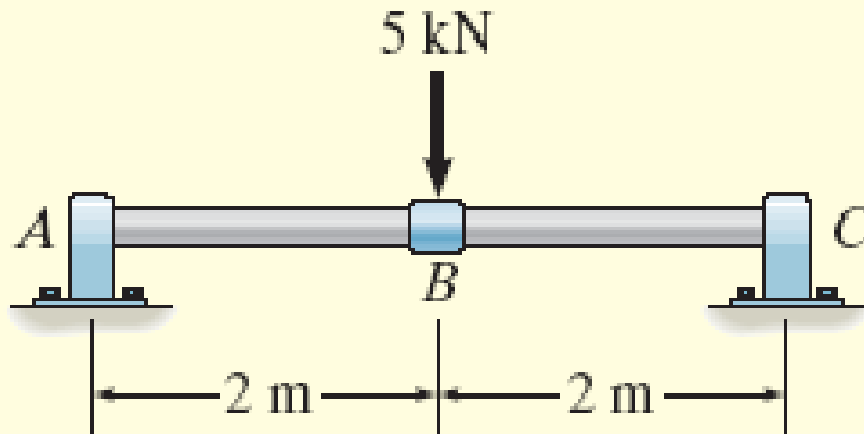
# Shear and Bending Moment Equations and Diagrams

Note that, at B, the internal shear force abruptly changes from 2.5 kN to -2.5 kN, i.e. by the amount of the applied load of -5.0 kN.



Shear Force Diagram

# Shear and Bending Moment Equations and Diagrams



Take moments about here

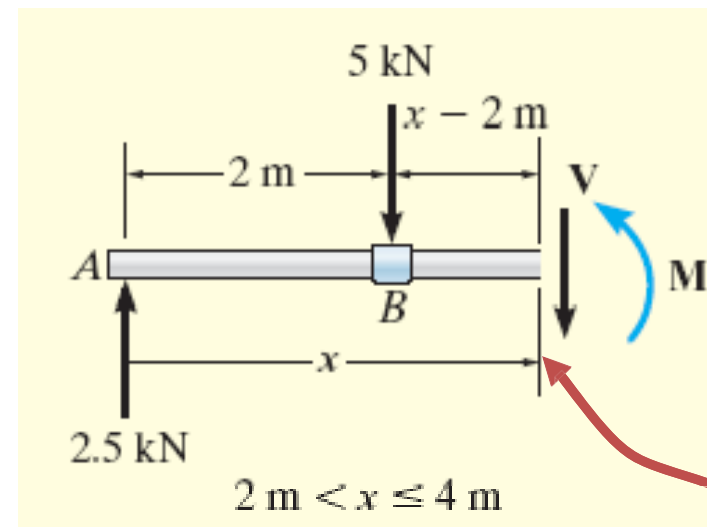
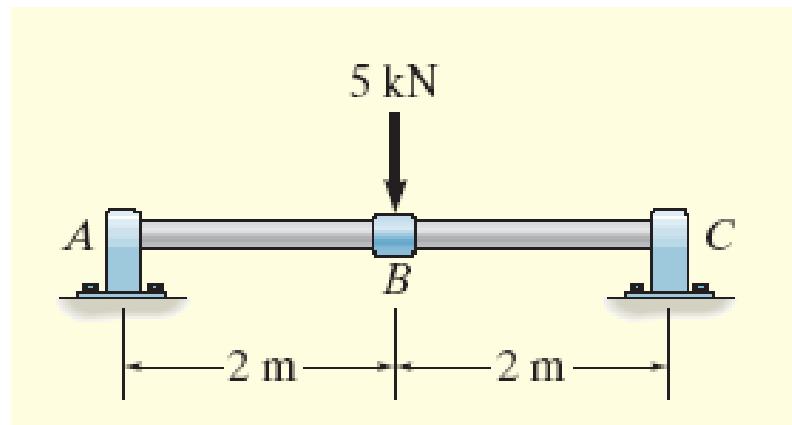
For  $0 \leq x < 2$  m, i.e. between A and B:

$$\sum M_x = -2.5 \text{ kN}(x) + M = 0 \Rightarrow M = 2.5x \text{ kN.m}$$

Note: When adding up the moments, forces times distance = moment. We must add in M at the cut because it is one of the moments in the diagram



# Shear and Bending Moment Equations and Diagrams



For  $2 \text{ m} < x \leq 4 \text{ m}$ :

Take moments about here

$$\sum M_x = M + 5 \text{ kN}(x - 2\text{m}) - 2.5 \text{ kN}(x) = 0$$

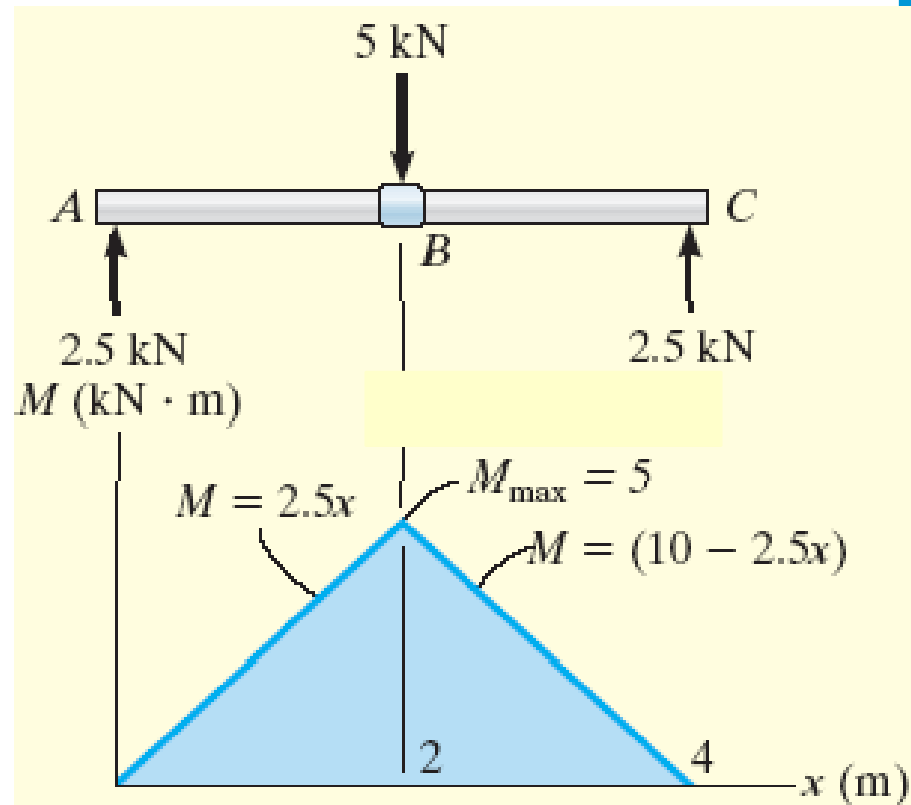
$$M = (10 - 2.5x) \text{ kN.m}$$

# Shear and Bending Moment Equations and Diagrams

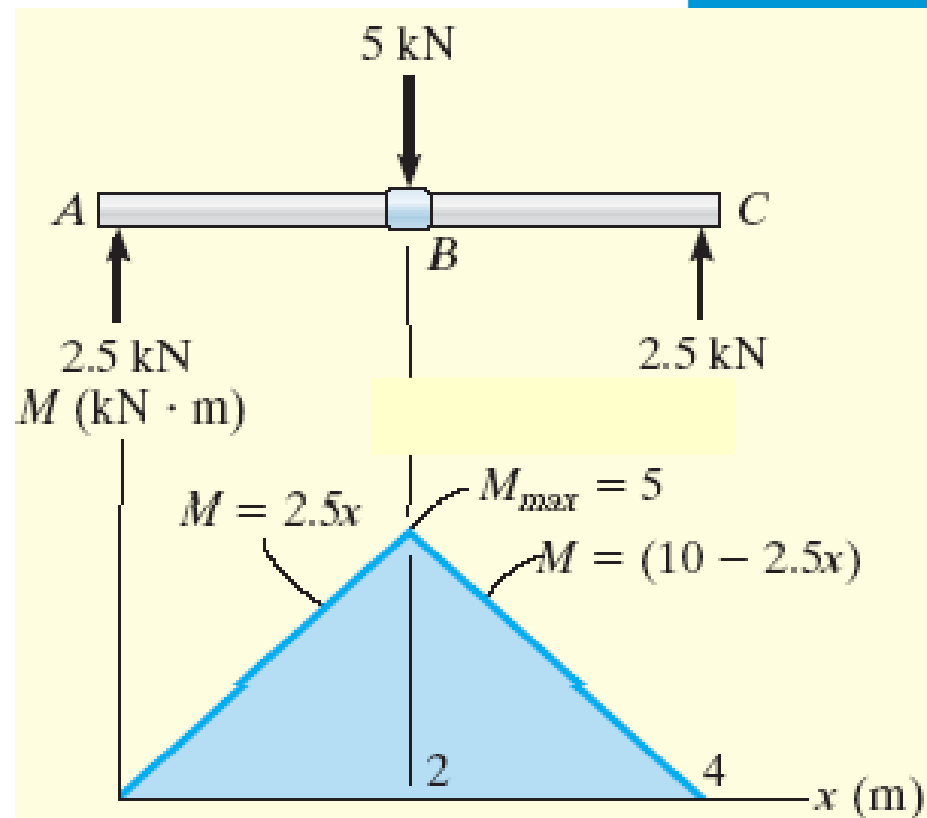
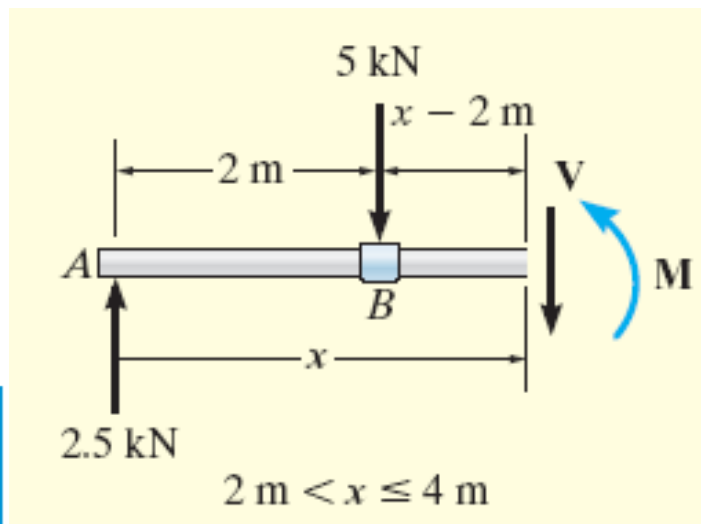
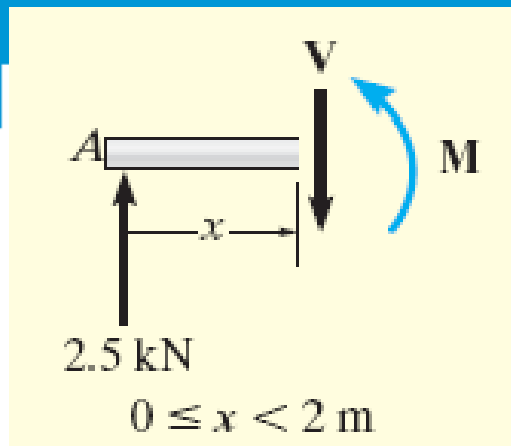
For  $0 \leq x < 2$  m, i.e.  
between A and B:

For  $2 \text{ m} < x \leq 4$  m, i.e.  
between B and C:

$$M = (10 - 2.5x) \text{ kN.m}$$



Sagging Bending  
moment



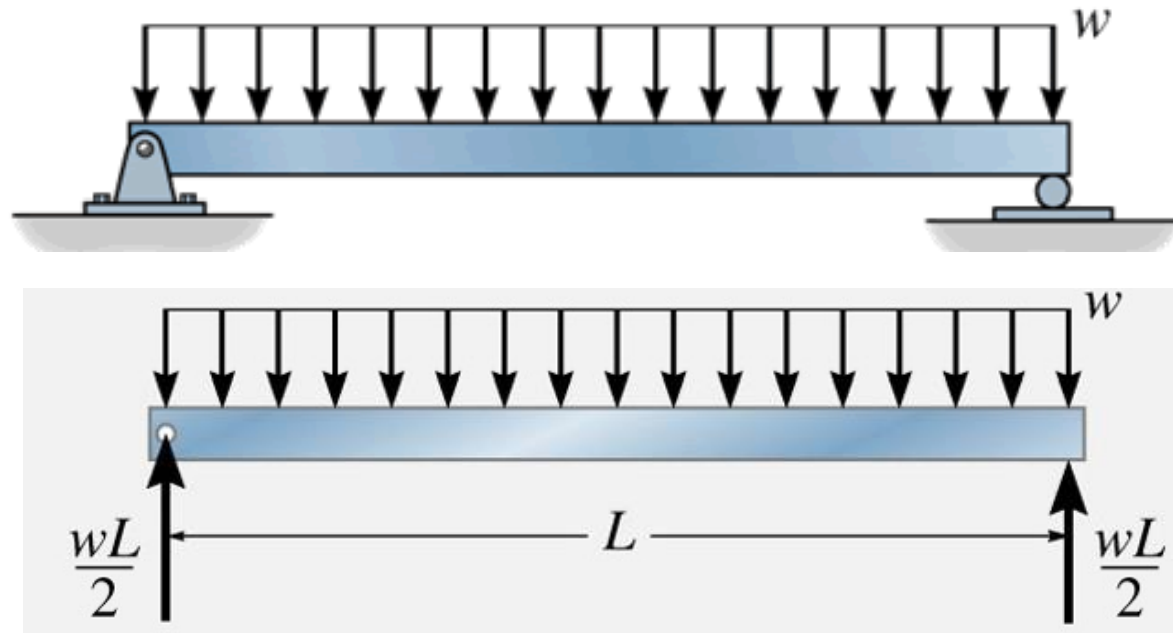
**Sagging Bending moment**

# Shear and Bending Moment Equations and Diagrams

In practice, beams are commonly subject to distributed loads.



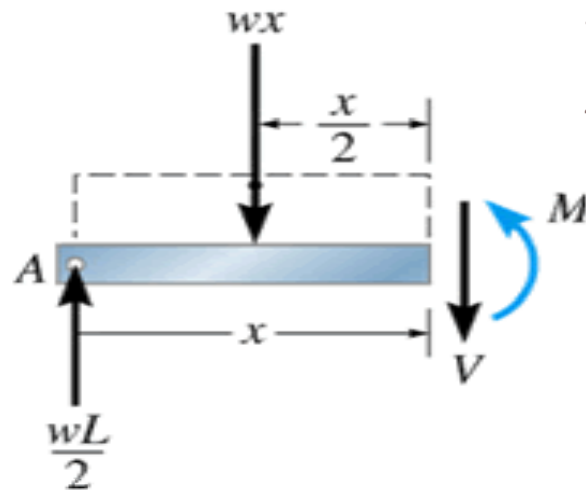
# Shear and Bending Moment Equations and Diagrams



Units for  $w$  are usually N/m or kN/m

Due to symmetry, we know that the vertical reaction at A must be half the applied load, i.e.  $\frac{wL}{2}$  upwards.

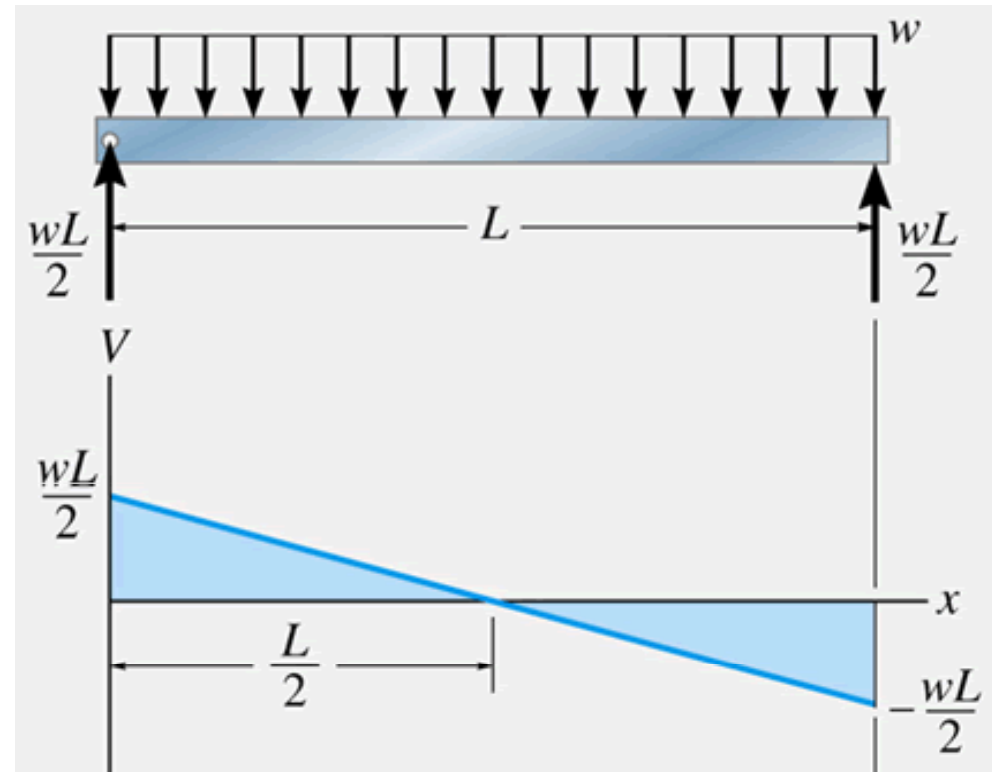
# Shear and Bending Moment Equations and Diagrams



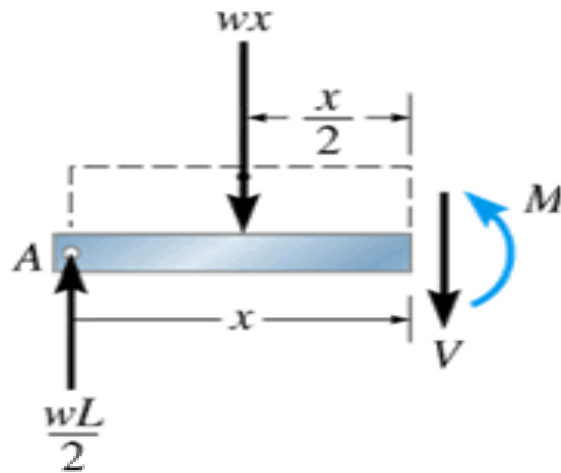
$$\sum F_y = \frac{wL}{2} - wx - V = 0 \Rightarrow V = w\left(\frac{L}{2} - x\right)$$



Shear Force Diagram

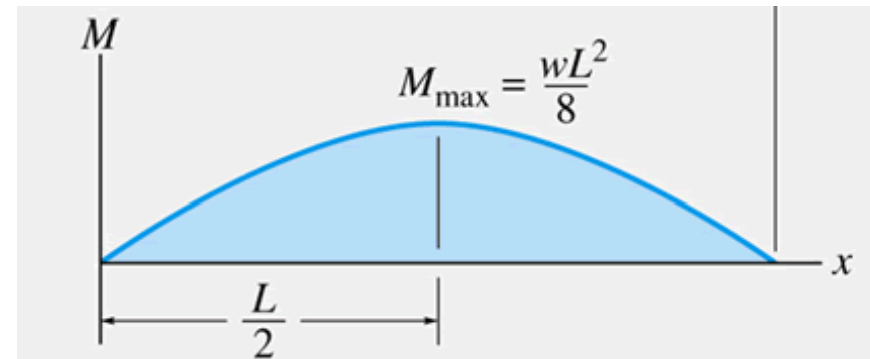
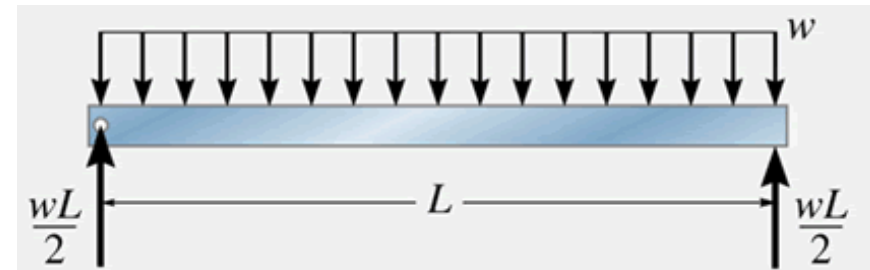


# Shear and Bending Moment Equations and Diagrams



$$\sum M_x = -\frac{wL}{2}(x) + wx\left(\frac{x}{2}\right) + M = 0$$

$$M = \frac{w}{2}(Lx - x^2)$$



Sagging moment

+ve Moment



# Bending moment diagram calculators

<http://beamguru.com/online/beam-calculator/>

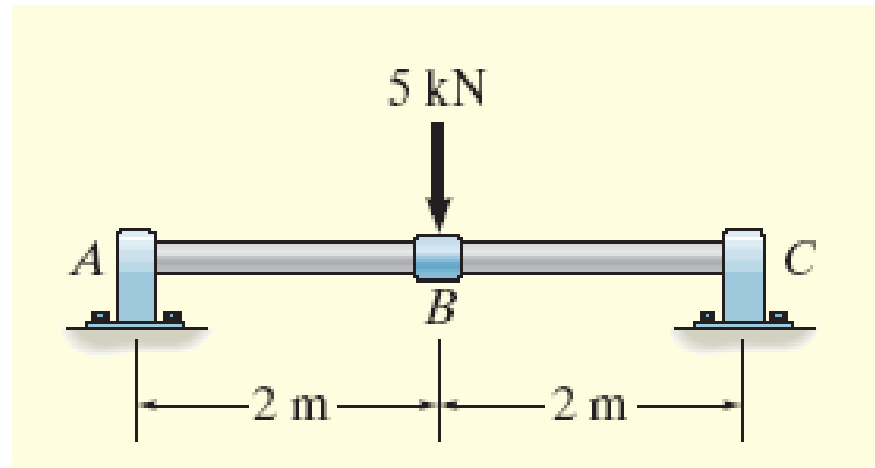
<https://bendingmomentdiagram.com/free-calculator/>

NOT allowed in exams/quizzes.





## Relationship between Shear and Bending Moment



For  $0 \leq x < 2$  m (between A and B):

$$M = 2.5x \text{ kN.m}$$

$$V = 2.5 \text{ kN}$$

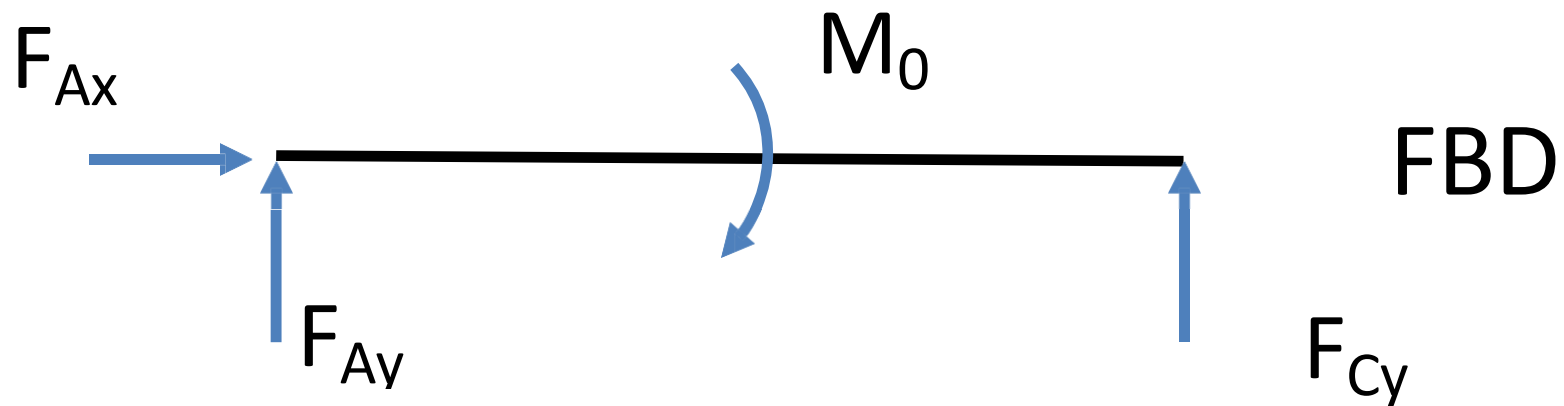
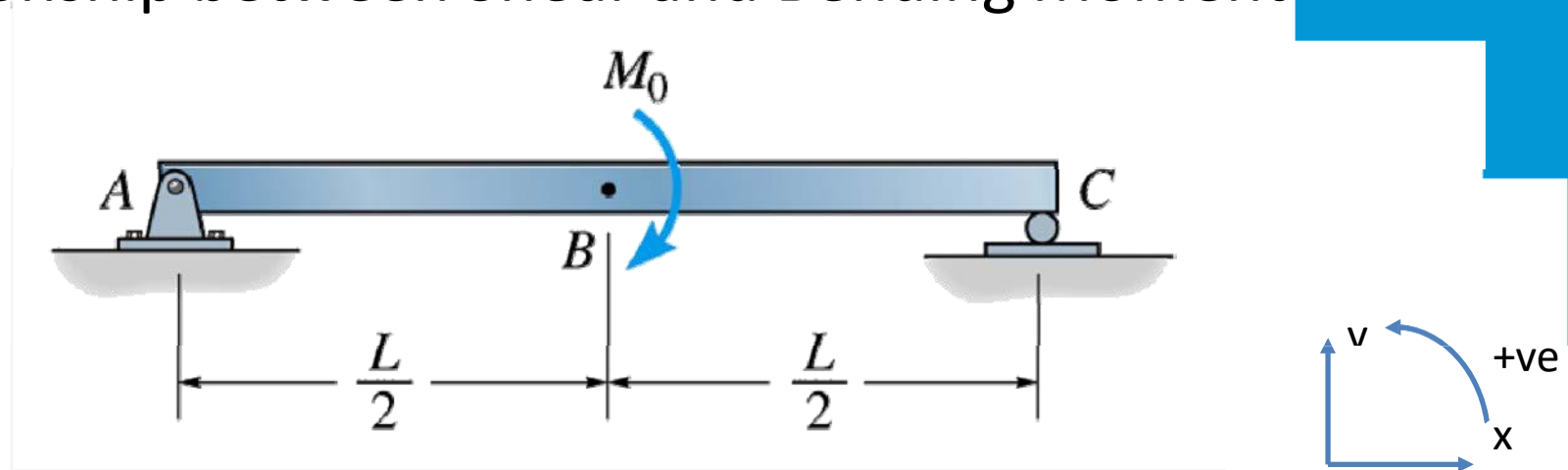
For  $2 \text{ m} < x \leq 4$  m (between B and C):

$$M = (10 - 2.5x) \text{ kN.m}$$

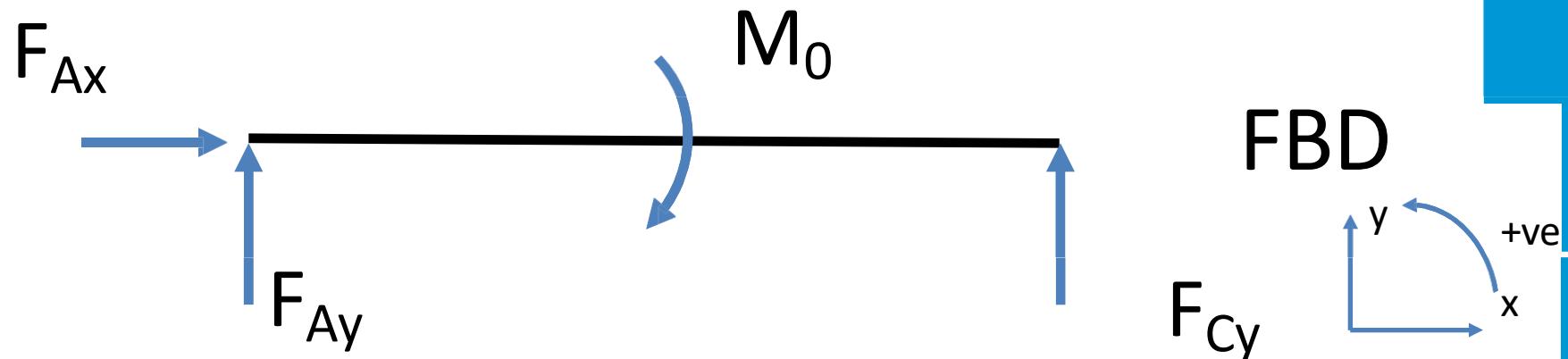
$$V = -2.5 \text{ kN}$$

Notice any pattern between V and M equations?

# Relationship between Shear and Bending Moment



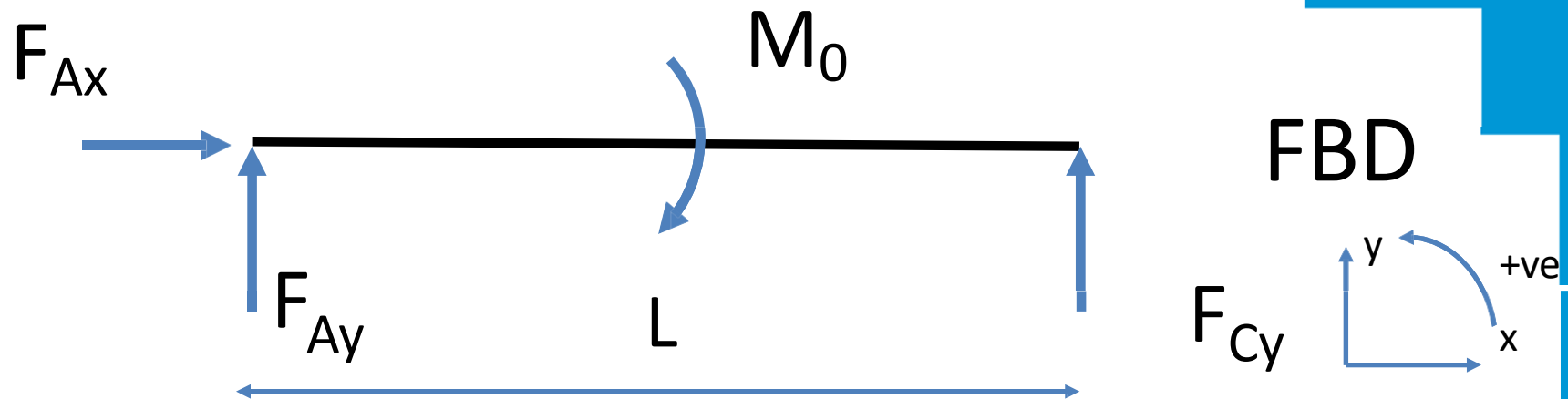
## Relationship between Shear and Bending Moment



$$\sum F_x = 0 \Rightarrow F_{Ax} = 0$$

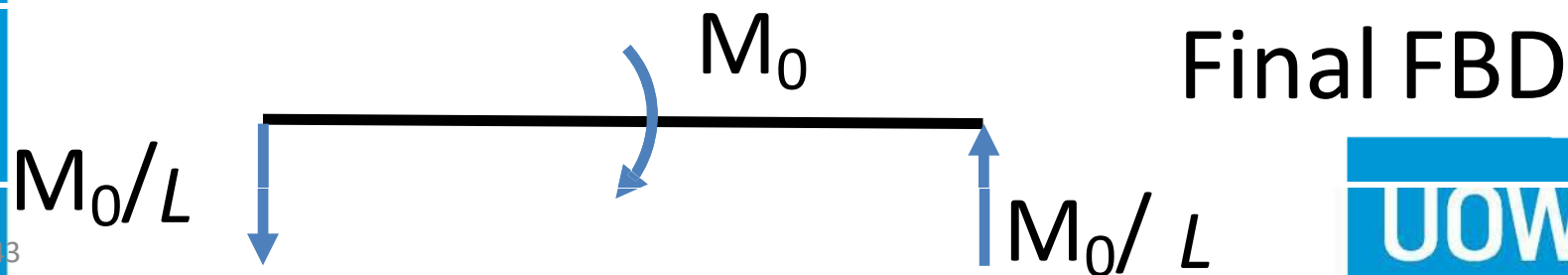
$$\sum F_y = 0 \Rightarrow F_{Ay} + F_{Cy} = 0 \Rightarrow F_{Ay} = -F_{Cy}$$

## Relationship between Shear and Bending Moment

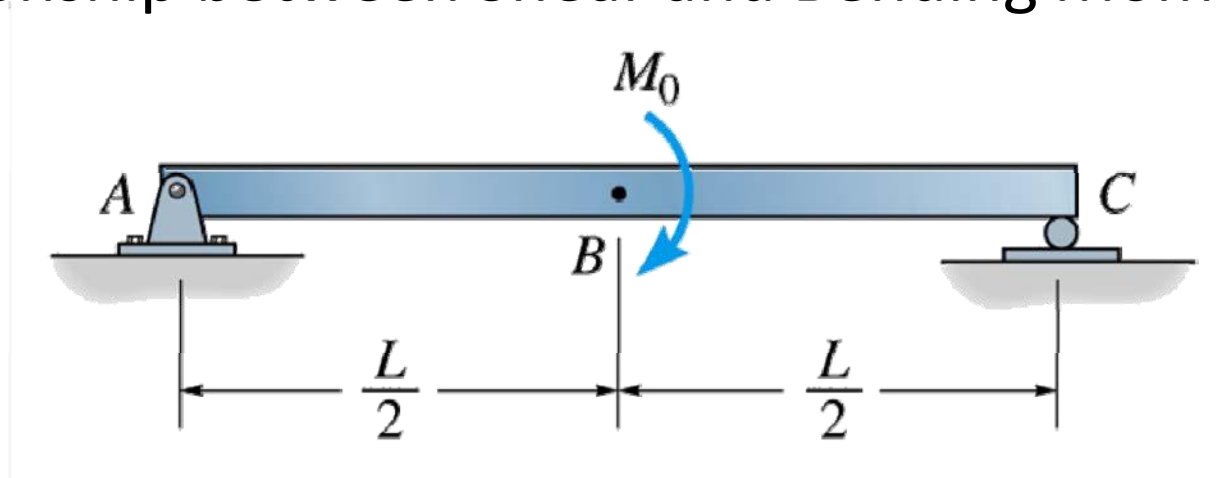


$$\sum M_A = 0 \Rightarrow F_{Cy} * L - M_0 = 0 \Rightarrow F_{Cy} = M_0 / L$$

$$F_{Ay} = -F_{Cy} = -M_0 / L$$



## Relationship between Shear and Bending Moment



For  $0 \leq x < L/2$  (between A and B):

$$M = -M_0 x / L$$

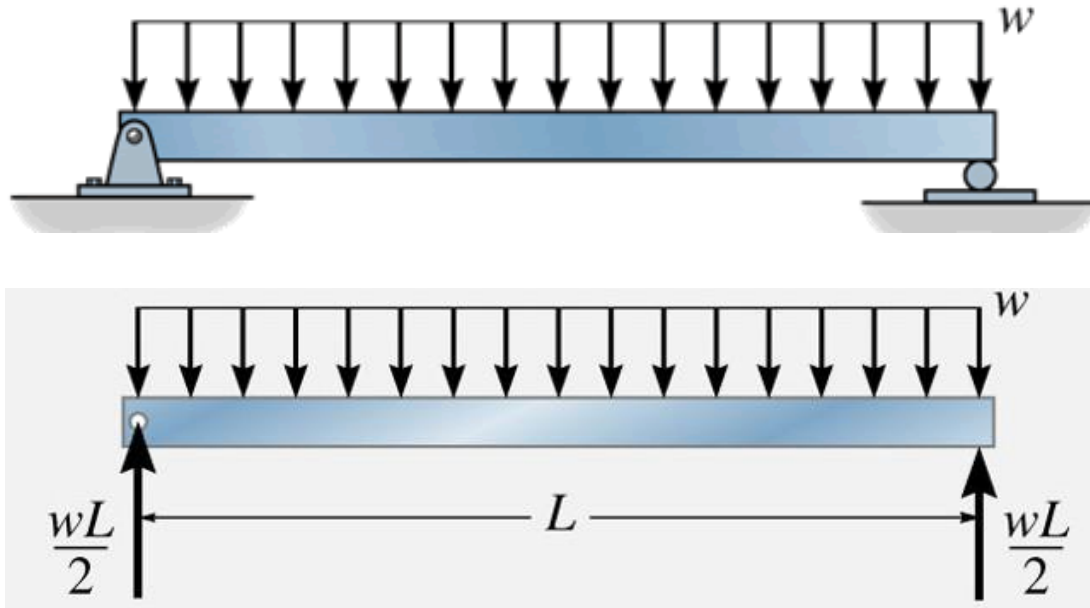
$$V = M_0 / L$$

For  $L/2 < x \leq L$  (between B and C):

$$M = M_0 (1 - x / L)$$

$$V = -M_0 / L$$

## Relationship between Shear and Bending Moment



$$M = \frac{w}{2} (Lx - x^2)$$

$$V = w\left(\frac{L}{2} - x\right) = \frac{w}{2} (L - 2x)$$

## 7.3 Relationship between Shear and Bending Moment

Do you notice a pattern?

$$M = 2.5x \text{ kN.m}$$

$$V = 2.5 \text{ kN}$$

$$M = (10 - 2.5x) \text{ kN.m}$$

$$V = -2.5 \text{ kN}$$

$$M = -\frac{M_0 x}{L}$$

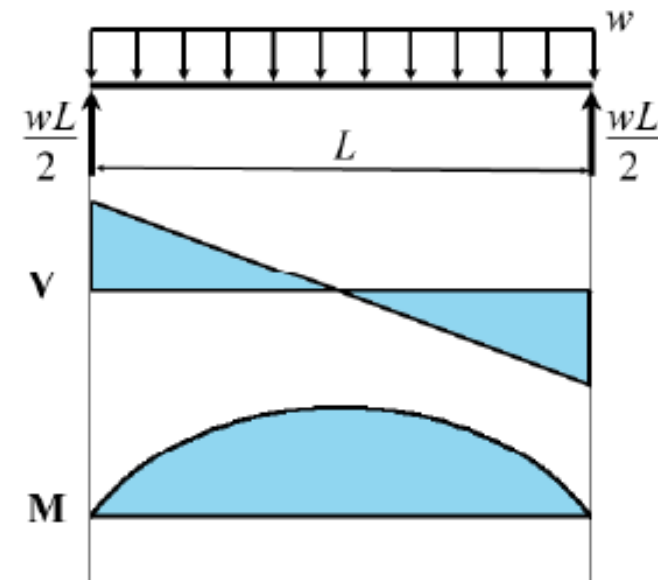
$$V = \frac{M_0}{L}$$

$$M = M_0 \left(1 - \frac{x}{L}\right)$$

$$V = -\frac{M_0}{L}$$

$$M = \frac{w}{2} (Lx - x^2)$$

$$V = w \left(\frac{L}{2} - x\right)$$



## 7.3 Relationship between Shear and Bending Moment



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07\_PH05

In order to design the beam used to support these power lines, it is important to first draw the shear and moment diagrams for the beam.



# YOUR Beam: stress and strength

FAST FORWARD TO SECOND YEAR ENGG251

- Maximum tensile stress allowed in BALSA is for you to find
- In this case the beam will break (mostly) because it is bending
- In that case stress on the bottom fibres in the middle will be  $\sigma_{\max} = M_{\max} h / (2I)$

