

## Tutorial Class 4 – Week 5

### Aims:

Upon successfully completing these tutorial exercises, students should be able to:

- Understand relationships between crack size, applied stress and fracture toughness
- Perform calculations relating to fracture toughness of materials
- Understand relationships between stress amplitude, mean stress, fatigue lifetime and fatigue strength
- Perform calculations relating to fatigue behavior of materials

## Fracture Toughness

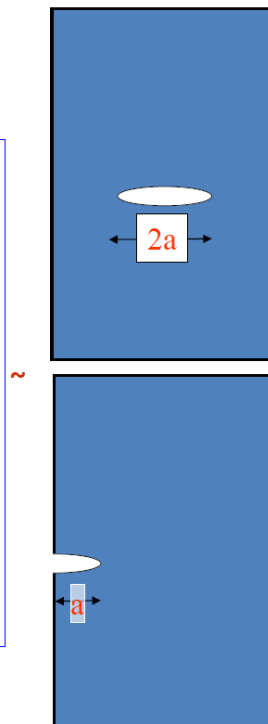
### Additional Crack characteristic information

#### Characteristics of Cracks

Cracks can be characterized looking into the following aspects.

- Its connection with the external free surface:
  - (i) completely internal,
  - (ii) internal cracks with connections to the outer surfaces,
  - (iii) Surface cracks.
- Cracks with some contact with external surfaces are exposed to outer media and hence may be prone to oxidation and corrosion (cracking).
- Crack length
- Crack tip radius Crack tip radius is dependent of the type of loading and the ductility of the material.
- Crack orientation with respect to geometry and loading.

$$K_{Ic} = Y\sigma\sqrt{\pi a}$$



**Exercise 4.1** A 50mm wide sample plate of 7074-T8 aluminium alloy contains a central through-crack of length 2a. For 7074-T8:  $K_{Ic} = 22.2 \text{ MN m}^{-3/2}$ ;  $\sigma_y = 520 \text{ MPa}$

- an applied stress of 200 MPa, determine if the plate will fail by fracture with a crack half-length a of:
  - 1 mm; 5 mm; 10 mm
- Determine the limiting crack size 2a below which the plate will fail by yielding (assume  $Y = 1$ )

**Exercise 4.2** In a component  $\sigma_y = 800 \text{ MPa}$ ,  $K_{Ic} = 85 \text{ Mpa.m}^{0.5}$ .

What is the maximum allowable size of a fully internal crack (Assume  $Y=0.95$ )?

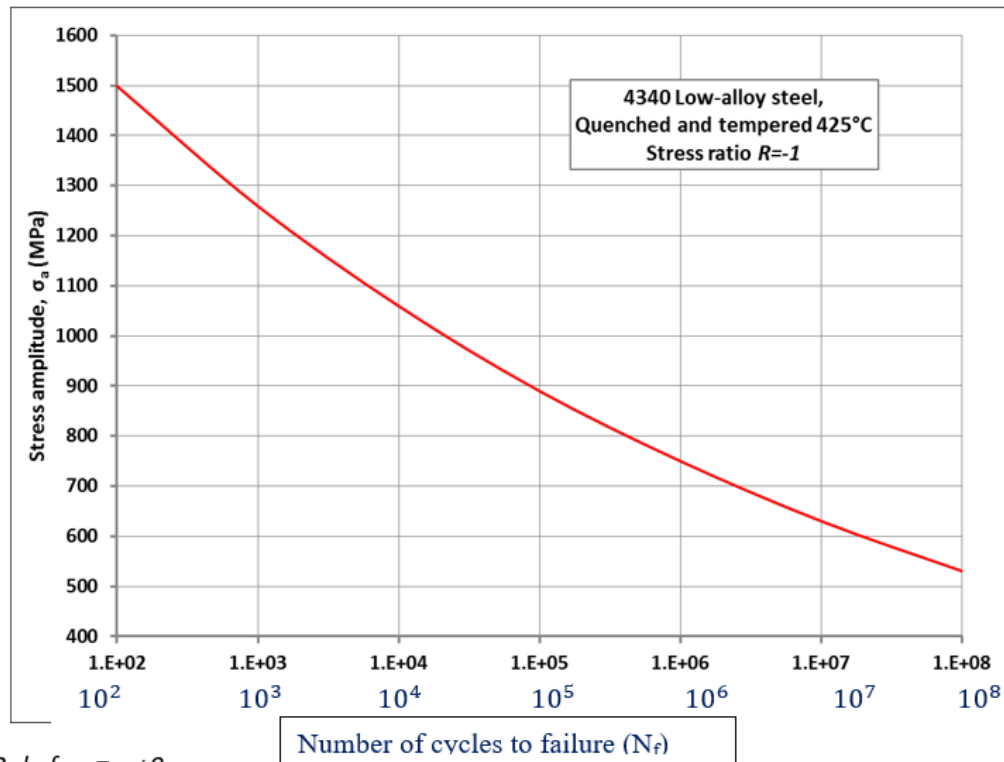
**Note:** For a fully internal crack, the crack size is 2a.

Brittle fracture occurs when  $K_{Ic} = Y\sigma\sqrt{\pi a}$

**Exercise 4.3** A rotating shaft in a gearbox is to be made from AISI 4340 quenched and tempered steel which has

a tensile strength of  $\sigma_{TS} = 1820 \text{ MPa}$ . Using the information in the following figure, determine:

- What is the fatigue strength  $\sigma_f$  at  $10^7$  cycles?
- Will the shaft fail by fatigue if it is subjected to 100 cycles with amplitude of 1200 MPa and zero mean stress?
- Will the shaft fail by fatigue if it is subjected to 100 000 cycles with amplitude of 900 MPa and zero mean stress?
- Will the shaft fail by fatigue if it is subjected to 100 000 cycles with amplitude of 800 MPa and mean stress of 300 MPa?
- If cycled between -100 MPa and 1100 MPa, how many cycles will the shaft survive? If the rotation frequency is 8 Hz, how long (in hours) will the shaft survive? (*Hertz = cycles per second*)



Goodman Rule for  $\sigma_m \neq 0$ :

$$\Delta\sigma_{\sigma_m} = \Delta\sigma_{\sigma_0} \left( 1 - \frac{\sigma_m}{\sigma_{TS}} \right) /$$



**Exercise 4.4** A small actuator is made from a polymer material with an ultimate tensile strength of  $\sigma_{ts} = 28$  MPa. The loading of the actuator is fully reversed with a stress amplitude of  $\sigma_a = 11$  MPa. If the loading cycles at a frequency of 3.0 Hz, how many hours would the actuator be expected to survive before failing due to fatigue? The SN curve for the polymer material is shown below.

1 Hz = 1 cycle per second  
3600 Hz = 1 cycle per hour

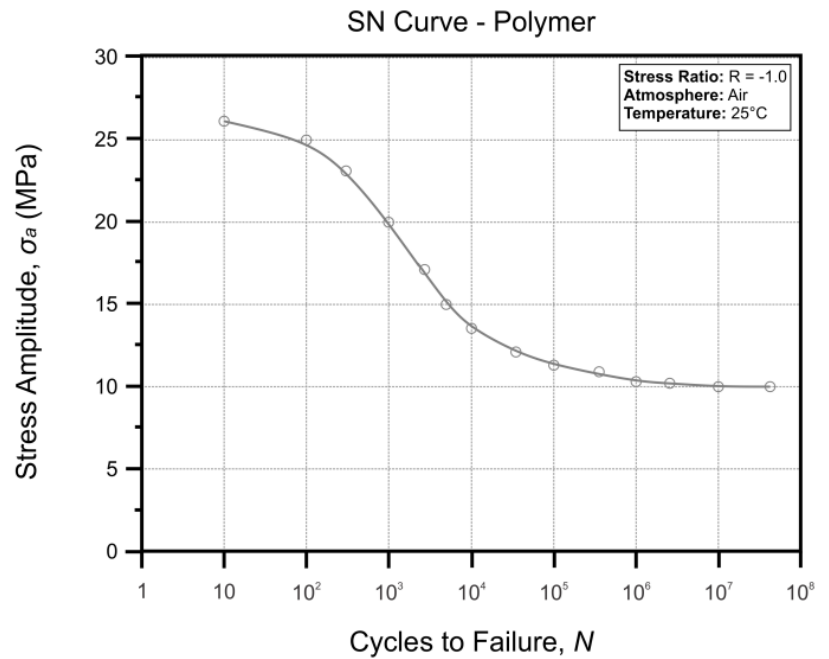


Figure 2. Fatigue behaviour of the polymer material.