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SOUTHERN UNIVERSITY OF SCIENCE AND TECHNOLOGY



地球与空间科学系
DEPARTMENT OF EARTH AND SPACE SCIENCES

MAE5009
Continuum Mechanics B
Session 08: Symbolic Notation

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
1

Symbolic (vector and dyadic) notation

- For vector (1st order tensor) quantities, the symbolic notation is most suitable
- For 2nd order tensor quantities, both symbolic and tensor notations could be used
- For higher-order tensors, tensor notation is more applicable
- Tensor notation is coordinate system based, equations derived with tensor notation cannot be changed readily into curvilinear coordinates
- For the purpose of treating linear elasticity in curvilinear coordinates, the symbolic notation is more convenient than the tensor notation
- Both tensor and symbolic notations are being used in the literature today

2

Basic vector analysis

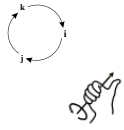
- Vector:
 $A = (A_x, A_y, A_z)$ magnitude $|A|$
Unit vector $|i| = |j| = |k| = 1$
 $A = A_x i + A_y j + A_z k$
- Scalar (dot) product:
 $A \cdot B = |A||B|\cos(A, B)$
 $A \cdot B = (A_x i + A_y j + A_z k) \cdot (B_x i + B_y j + B_z k)$
 $= A_x B_x + A_y B_y + A_z B_z$
 $A \cdot B = B \cdot A$
- Vector (cross) product:
 $A \times B = |A||B|\sin(A, B)n$


3

Basic vector analysis

- Vector (cross) product:

$$\begin{aligned} \mathbf{i} \times \mathbf{j} &= \mathbf{k} & \mathbf{j} \times \mathbf{k} &= \mathbf{i} & \mathbf{k} \times \mathbf{i} &= \mathbf{j} \\ \mathbf{j} \times \mathbf{i} &= -\mathbf{k} & \mathbf{k} \times \mathbf{j} &= -\mathbf{i} & \mathbf{i} \times \mathbf{k} &= -\mathbf{j} \\ \mathbf{i} \times \mathbf{i} &= \mathbf{j} \times \mathbf{j} = \mathbf{k} \times \mathbf{k} = 0 \end{aligned}$$



$$\begin{aligned} \mathbf{A} \times \mathbf{B} &= (A_x \mathbf{i} + A_y \mathbf{j} + A_z \mathbf{k}) \times (B_x \mathbf{i} + B_y \mathbf{j} + B_z \mathbf{k}) \\ &= \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ A_x & A_y & A_z \\ B_x & B_y & B_z \end{vmatrix} = (A_y B_z - B_y A_z) \mathbf{i} + (A_z B_x - B_z A_x) \mathbf{j} + (A_x B_y - B_x A_y) \mathbf{k} \\ \mathbf{A} \times \mathbf{B} &= -\mathbf{B} \times \mathbf{A} \end{aligned}$$

4

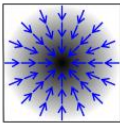
Basic vector analysis

- Vector operator ∇ (del or nabla), or gradient operator:

$$\nabla = \mathbf{i} \frac{\partial}{\partial x} + \mathbf{j} \frac{\partial}{\partial y} + \mathbf{k} \frac{\partial}{\partial z}$$

$$\text{grad } A = \nabla A = \mathbf{i} \frac{\partial A}{\partial x} + \mathbf{j} \frac{\partial A}{\partial y} + \mathbf{k} \frac{\partial A}{\partial z} \quad (A = A(x, y, z))$$

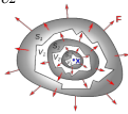
The gradient of a scalar field is a vector field, which gives the maximum rate of change of A at a point



- Divergence of a vector field is a scalar field:

$$\text{div } \mathbf{A} = \nabla \cdot \mathbf{A} = \left(\mathbf{i} \frac{\partial}{\partial x} + \mathbf{j} \frac{\partial}{\partial y} + \mathbf{k} \frac{\partial}{\partial z} \right) \cdot (A_x \mathbf{i} + A_y \mathbf{j} + A_z \mathbf{k}) = \frac{\partial A_x}{\partial x} + \frac{\partial A_y}{\partial y} + \frac{\partial A_z}{\partial z}$$

The divergence represents the volume density of the outward flux of a vector field from an infinitesimal volume around a given point.



5

$$\left(\mathbf{i} \frac{\partial}{\partial x} + \mathbf{j} \frac{\partial}{\partial y} + \mathbf{k} \frac{\partial}{\partial z} \right) \times (A_x \mathbf{i} + A_y \mathbf{j} + A_z \mathbf{k})$$

Basic vector analysis

- The curl of a vector field:

$$\text{curl } \mathbf{A} = \nabla \times \mathbf{A} = \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ \frac{\partial}{\partial x} & \frac{\partial}{\partial y} & \frac{\partial}{\partial z} \\ A_x & A_y & A_z \end{vmatrix} = \left(\frac{\partial A_z}{\partial y} - \frac{\partial A_y}{\partial z} \right) \mathbf{i} + \left(\frac{\partial A_x}{\partial z} - \frac{\partial A_z}{\partial x} \right) \mathbf{j} + \left(\frac{\partial A_y}{\partial x} - \frac{\partial A_x}{\partial y} \right) \mathbf{k}$$

The curl is a vector operator that describes the infinitesimal rotation of a vector field in three-dimensional Euclidean space

- Another useful operator:

$$\text{div } \nabla = \nabla \cdot \nabla = \frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2} + \frac{\partial^2}{\partial z^2} = \nabla^2$$

The Laplace operator

6

Dyadic – 2nd order tensor

- Stress dyadic

$$\begin{aligned}\wp &= \mathbf{BC} = \mathbf{B} \otimes \mathbf{C}^T \\ &= B_x C_x \mathbf{ii} + B_x C_y \mathbf{ij} + B_x C_z \mathbf{ik} \\ &\quad + B_y C_x \mathbf{ji} + B_y C_y \mathbf{jj} + B_y C_z \mathbf{jk} \\ &\quad + B_z C_x \mathbf{ki} + B_z C_y \mathbf{kj} + B_z C_z \mathbf{kk}\end{aligned}$$

where

$$\begin{aligned}\mathbf{B} &= B_x \mathbf{i} + B_y \mathbf{j} + B_z \mathbf{k} \\ \mathbf{C} &= C_x \mathbf{i} + C_y \mathbf{j} + C_z \mathbf{k} \\ \text{Tensor notation: } S_{ij} &= B_i C_j\end{aligned}$$

Dyadic can also be written as:

$$\begin{aligned}\wp &= S_{xx} \mathbf{ii} + S_{xy} \mathbf{ij} + S_{xz} \mathbf{ik} \\ &\quad + S_{yx} \mathbf{ji} + S_{yy} \mathbf{jj} + S_{yz} \mathbf{jk} \\ &\quad + S_{zx} \mathbf{ki} + S_{zy} \mathbf{kj} + S_{zz} \mathbf{kk}\end{aligned}$$

Its conjugate dyadic:

$$\begin{aligned}\wp_c &= S_{xx} \mathbf{ii} + S_{yy} \mathbf{jj} + S_{zz} \mathbf{kk} \\ &\quad + S_{yx} \mathbf{ij} + S_{xy} \mathbf{ji} + S_{yz} \mathbf{jk} \\ &\quad + S_{zx} \mathbf{ki} + S_{zy} \mathbf{kj} + S_{xz} \mathbf{ik} \\ &= \mathbf{CB}\end{aligned}$$

If a dyadic is equal to its conjugate, the dyadic is symmetric:

$$\wp = \wp_c$$

7

Dyadic – 2nd order tensor

- If a dyadic is antisymmetric:

$$\begin{aligned}\wp &= -\wp_c \\ S_{xx} = S_{yy} = S_{zz} &= 0 \\ S_{xy} = -S_{yx}, S_{xz} = -S_{zx}, S_{yz} &= -S_{zy}\end{aligned}$$

- A dyadic may also be expressed in terms of three vectors:

$$\wp = i\mathbf{S}_x + j\mathbf{S}_y + k\mathbf{S}_z$$

where

$$\begin{aligned}\mathbf{S}_x &= S_{xx} \mathbf{i} + S_{xy} \mathbf{j} + S_{xz} \mathbf{k} \\ \mathbf{S}_y &= S_{yx} \mathbf{i} + S_{yy} \mathbf{j} + S_{yz} \mathbf{k} \\ \mathbf{S}_z &= S_{zx} \mathbf{i} + S_{zy} \mathbf{j} + S_{zz} \mathbf{k}\end{aligned}$$

Its conjugate dyadic:

$$\wp_c = \mathbf{S}_x \mathbf{i} + \mathbf{S}_y \mathbf{j} + \mathbf{S}_z \mathbf{k}$$

8

Vector products between vector and dyadic

$$\begin{aligned}\mathbf{A} \cdot \wp &= \mathbf{A} \cdot \mathbf{BC} = (A_x \mathbf{i} + A_y \mathbf{j} + A_z \mathbf{k}) \cdot (i\mathbf{S}_x + j\mathbf{S}_y + k\mathbf{S}_z) \\ &= A_x \mathbf{S}_x + A_y \mathbf{S}_y + A_z \mathbf{S}_z \\ &= (\mathbf{A} \cdot \mathbf{B}) \mathbf{C} \\ \mathbf{A} \cdot \wp_c &= \mathbf{A} \cdot \mathbf{CB} = (\mathbf{A} \cdot \mathbf{S}_x) \mathbf{i} + (\mathbf{A} \cdot \mathbf{S}_y) \mathbf{j} + (\mathbf{A} \cdot \mathbf{S}_z) \mathbf{k} = (\mathbf{A} \cdot \mathbf{C}) \mathbf{B} = \wp \cdot \mathbf{A} \\ \mathbf{A} \times \wp &= \mathbf{A} \times \mathbf{BC} = \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ A_x & A_y & A_z \\ S_x & S_y & S_z \end{vmatrix} \\ \text{div } \wp &= \nabla \cdot \wp = \left(\frac{\partial}{\partial x} \mathbf{i} + \frac{\partial}{\partial y} \mathbf{j} + \frac{\partial}{\partial z} \mathbf{k} \right) \cdot (i\mathbf{S}_x + j\mathbf{S}_y + k\mathbf{S}_z) \\ &= \frac{\partial}{\partial x} \mathbf{S}_x + \frac{\partial}{\partial y} \mathbf{S}_y + \frac{\partial}{\partial z} \mathbf{S}_z\end{aligned}$$

9

Vector products between vector and dyadic

$$\begin{aligned} \text{div } \wp_c &= \nabla \cdot \wp_c = \nabla \cdot \mathbf{CB} = \left(\frac{\partial}{\partial x} \mathbf{i} + \frac{\partial}{\partial y} \mathbf{j} + \frac{\partial}{\partial z} \mathbf{k} \right) \cdot (\mathbf{S}_x \mathbf{i} + \mathbf{S}_y \mathbf{j} + \mathbf{S}_z \mathbf{k}) \\ &= (\nabla \cdot \mathbf{S}_x) \mathbf{i} + (\nabla \cdot \mathbf{S}_y) \mathbf{j} + (\nabla \cdot \mathbf{S}_z) \mathbf{k} \\ &= (\nabla \cdot \mathbf{C}) \mathbf{B} \end{aligned}$$
$$\text{curl } \wp = \nabla \times \wp = \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ \frac{\partial}{\partial x} & \frac{\partial}{\partial y} & \frac{\partial}{\partial z} \\ S_x & S_y & S_z \end{vmatrix} = \mathbf{i} \times \frac{\partial \wp}{\partial x} + \mathbf{j} \times \frac{\partial \wp}{\partial y} + \mathbf{k} \times \frac{\partial \wp}{\partial z}$$

Vector products (dot and cross) between a vector and a dyadic are always made between the vector and the adjacent vector in the dyadic

10

Comparison between tensor and symbolic notations

Quantities & operations	Tensor notation	Symbolic notation
scalar	ϕ	ϕ
1 st order tensor, or vector	u_i	\mathbf{u}
2 nd order tensor, or dyadic	S_{ij}	\wp
Conjugate of a dyadic	S_{ji} for S_{ij}	\wp_c for \wp
Scalar product	$u_i v_i$	$\mathbf{u} \cdot \mathbf{v}$
Vector product	$\epsilon_{ijk} u_i v_j$	$\mathbf{u} \times \mathbf{v}$
Gradient	$\phi_{,i}$	$\nabla \phi$
Divergence	$u_{i,i}$	$\nabla \cdot \mathbf{u}$
Curl	$\epsilon_{ijk} u_{k,j}$	$\nabla \times \mathbf{u}$
Gradient of a vector	$u_{i,j}$	$\nabla \mathbf{u}$
Divergence of a dyadic	$S_{ji,i}$	$\nabla \cdot \wp$
Curl of a dyadic	$\omega_{il} = \epsilon_{ijk} S_{kl,j}$	$\nabla \times \wp$
Laplace	$\phi_{,i,ii}$	$\nabla^2 \phi$
Biharmonic	$\phi_{,i,ijj}$	$\nabla^4 \phi$ or $\nabla^2 \nabla^2 \phi$

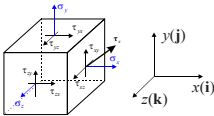
$\nabla^2(\nabla^2 \phi)$

11

Vector representation of stress on a plane

- Stress components acting on the corresponding planes:

$$\begin{aligned} \tau_x &= \sigma_x \mathbf{i} + \tau_{xy} \mathbf{j} + \tau_{xz} \mathbf{k} \\ \tau_y &= \tau_{yx} \mathbf{i} + \sigma_y \mathbf{j} + \tau_{yz} \mathbf{k} \\ \tau_z &= \tau_{zx} \mathbf{i} + \tau_{zy} \mathbf{j} + \sigma_z \mathbf{k} \end{aligned}$$



- The relations between the stress vector and its components are:

$$\begin{aligned} \sigma_x &= |\tau_x| \cos(\tau_x, \mathbf{i}) = \tau_x \cdot \mathbf{i} \\ \tau_{xy} &= |\tau_x| \cos(\tau_x, \mathbf{j}) = \tau_x \cdot \mathbf{j} \\ \tau_{xz} &= |\tau_x| \cos(\tau_x, \mathbf{k}) = \tau_x \cdot \mathbf{k} \end{aligned}$$

12

Equations of transformation of stress

- The unit vector μ_1 normal to the x' plane is:

$$\mu_1 = [\cos(x',x) \quad \cos(x',y) \quad \cos(x',z)]$$
$$= [a_{11} \quad a_{12} \quad a_{13}]$$

- Force equilibrium:

$$A\mathbf{p} = A_x\boldsymbol{\tau}_x + A_y\boldsymbol{\tau}_y + A_z\boldsymbol{\tau}_z$$

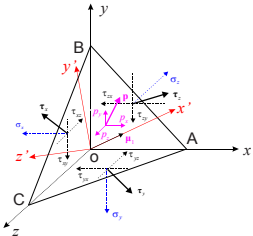
$$\mathbf{p} = \frac{A_x}{A}\boldsymbol{\tau}_x + \frac{A_y}{A}\boldsymbol{\tau}_y + \frac{A_z}{A}\boldsymbol{\tau}_z$$

$$\cos(x',x) = \frac{A_x}{A} = a_{11}$$

$$\cos(x',y) = \frac{A_y}{A} = a_{12}$$

$$\cos(x',z) = \frac{A_z}{A} = a_{13}$$

$$\mathbf{p} = a_{11}\boldsymbol{\tau}_x + a_{12}\boldsymbol{\tau}_y + a_{13}\boldsymbol{\tau}_z$$



Transformational matrix \mathbf{R}

	x	y	z
x'	a_{11}	a_{12}	a_{13}
y'	a_{21}	a_{22}	a_{23}
z'	a_{31}	a_{32}	a_{33}

13

Equations of transformation of stress

- \mathbf{p} can be decomposed into:

$$\mathbf{p} = a_{11}\boldsymbol{\tau}_x + a_{12}\boldsymbol{\tau}_y + a_{13}\boldsymbol{\tau}_z$$

$$p_x = \sigma_x a_{11} + \tau_{yx} a_{12} + \tau_{zx} a_{13} = \boldsymbol{\tau}_x \cdot \boldsymbol{\mu}_1$$

$$p_y = \tau_{xy} a_{11} + \sigma_y a_{12} + \tau_{zy} a_{13} = \boldsymbol{\tau}_y \cdot \boldsymbol{\mu}_1$$

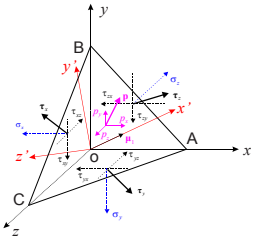
$$p_z = \tau_{xz} a_{11} + \tau_{yz} a_{12} + \sigma_z a_{13} = \boldsymbol{\tau}_z \cdot \boldsymbol{\mu}_1$$

$$\mathbf{p} = p_x \mathbf{i} + p_y \mathbf{j} + p_z \mathbf{k}$$
$$= \mathbf{i}(\boldsymbol{\tau}_x \cdot \boldsymbol{\mu}_1) + \mathbf{j}(\boldsymbol{\tau}_y \cdot \boldsymbol{\mu}_1) + \mathbf{k}(\boldsymbol{\tau}_z \cdot \boldsymbol{\mu}_1)$$
$$= (\mathbf{i}\boldsymbol{\tau}_x + \mathbf{j}\boldsymbol{\tau}_y + \mathbf{k}\boldsymbol{\tau}_z) \cdot \boldsymbol{\mu}_1$$

$$\mathbf{p} = \boldsymbol{\Sigma} \cdot \boldsymbol{\mu}_1$$

where $\boldsymbol{\Sigma} = \mathbf{i}\boldsymbol{\tau}_x + \mathbf{j}\boldsymbol{\tau}_y + \mathbf{k}\boldsymbol{\tau}_z$
Stress dyadic, symbolic representation of stress tensor

$$\boldsymbol{\Sigma} = \sigma_x \mathbf{ii} + \tau_{xy} \mathbf{ij} + \tau_{xz} \mathbf{ik}$$
$$+ \tau_{yx} \mathbf{ji} + \sigma_y \mathbf{jj} + \tau_{yz} \mathbf{jk}$$
$$+ \tau_{zx} \mathbf{ki} + \tau_{zy} \mathbf{kj} + \sigma_z \mathbf{kk}$$
$$\boldsymbol{\Sigma} = \boldsymbol{\Sigma}_c$$



14

Equations of transformation of stress

- The normal stress on the x' plane is:

$$\sigma_{x'} = \boldsymbol{\mu}_1 \cdot \mathbf{p} = \boldsymbol{\mu}_1 \cdot \boldsymbol{\Sigma} \cdot \boldsymbol{\mu}_1$$
$$= a_{11}^2 \sigma_x + a_{12}^2 \sigma_y + a_{13}^2 \sigma_z$$
$$+ 2a_{11}a_{12}\tau_{xy} + 2a_{11}a_{13}\tau_{xz} + 2a_{12}a_{13}\tau_{yz}$$

- Similarly:

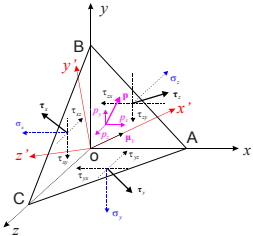
$$\tau_{x'y'} = \boldsymbol{\mu}_2 \cdot \mathbf{p} = \boldsymbol{\mu}_2 \cdot \boldsymbol{\Sigma} \cdot \boldsymbol{\mu}_1$$

$$\tau_{x'z'} = \boldsymbol{\mu}_3 \cdot \mathbf{p} = \boldsymbol{\mu}_3 \cdot \boldsymbol{\Sigma} \cdot \boldsymbol{\mu}_1$$

where $\boldsymbol{\mu}_2$ and $\boldsymbol{\mu}_3$ are unit vectors in the y' and z' directions, respectively

- If x , y and z directions coincides with principal directions:

$$\boldsymbol{\Sigma} = \mathbf{i}\boldsymbol{\tau}_1 + \mathbf{j}\boldsymbol{\tau}_2 + \mathbf{k}\boldsymbol{\tau}_3 = \sigma_1 \mathbf{ii} + \sigma_2 \mathbf{jj} + \sigma_3 \mathbf{kk}$$



Transformational matrix \mathbf{R}

	x	y	z
x'	a_{11}	a_{12}	a_{13}
y'	a_{21}	a_{22}	a_{23}
z'	a_{31}	a_{32}	a_{33}

15

Equilibrium equations

$$\begin{cases} \frac{\partial \sigma_x}{\partial x} + \frac{\partial \tau_{yx}}{\partial y} + \frac{\partial \tau_{zx}}{\partial z} + f_x = 0 \\ \frac{\partial \tau_{xy}}{\partial x} + \frac{\partial \sigma_y}{\partial y} + \frac{\partial \tau_{zy}}{\partial z} + f_y = 0 \\ \frac{\partial \tau_{xz}}{\partial x} + \frac{\partial \tau_{yz}}{\partial y} + \frac{\partial \sigma_z}{\partial z} + f_z = 0 \end{cases} \rightarrow \begin{cases} \frac{\partial \sigma_x}{\partial x} + \frac{\partial \tau_{xy}}{\partial y} + \frac{\partial \tau_{xz}}{\partial z} + f_x = 0 \\ \frac{\partial \tau_{yx}}{\partial x} + \frac{\partial \sigma_y}{\partial y} + \frac{\partial \tau_{yz}}{\partial z} + f_y = 0 \\ \frac{\partial \tau_{xz}}{\partial x} + \frac{\partial \tau_{yz}}{\partial y} + \frac{\partial \sigma_z}{\partial z} + f_z = 0 \end{cases}$$

$$\begin{aligned} \tau_x &= \sigma_x \mathbf{i} + \tau_{xy} \mathbf{j} + \tau_{xz} \mathbf{k} \\ \tau_y &= \tau_{yx} \mathbf{i} + \sigma_y \mathbf{j} + \tau_{yz} \mathbf{k} \\ \tau_z &= \tau_{xz} \mathbf{i} + \tau_{yz} \mathbf{j} + \sigma_z \mathbf{k} \end{aligned} \rightarrow \begin{aligned} \operatorname{div} \tau_x &= \nabla \cdot \tau_x = \frac{\partial \sigma_x}{\partial x} + \frac{\partial \tau_{yx}}{\partial y} + \frac{\partial \tau_{xz}}{\partial z} \\ \operatorname{div} \tau_y &= \nabla \cdot \tau_y = \frac{\partial \tau_{xy}}{\partial x} + \frac{\partial \sigma_y}{\partial y} + \frac{\partial \tau_{yz}}{\partial z} \\ \operatorname{div} \tau_z &= \nabla \cdot \tau_z = \frac{\partial \tau_{xz}}{\partial x} + \frac{\partial \tau_{yz}}{\partial y} + \frac{\partial \sigma_z}{\partial z} \end{aligned}$$

$$\begin{aligned} \operatorname{div} \tau_x + f_x &= 0 \\ \operatorname{div} \tau_y + f_y &= 0 \\ \operatorname{div} \tau_z + f_z &= 0 \end{aligned} \rightarrow \begin{aligned} \mathbf{i}(\operatorname{div} \tau_x + f_x) + \mathbf{j}(\operatorname{div} \tau_y + f_y) + \mathbf{k}(\operatorname{div} \tau_z + f_z) &= 0 \\ \operatorname{div}(\mathbf{i} \tau_x + \mathbf{j} \tau_y + \mathbf{k} \tau_z) + \mathbf{f} &= 0 \\ \operatorname{div} \Sigma + \mathbf{f} &= 0 \quad \text{where } \mathbf{f} = f_x \mathbf{i} + f_y \mathbf{j} + f_z \mathbf{k} \end{aligned}$$

16

Displacement and strain

- Displacement vector
- The divergence of \mathbf{u} gives the volumetric strain (dilation):

$$\operatorname{div} \mathbf{u} = \nabla \cdot \mathbf{u} = \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = \epsilon_x + \epsilon_y + \epsilon_z = \epsilon$$

- Strain vector:

$$\begin{aligned} \mathbf{e}_x &= \epsilon_x \mathbf{i} + \frac{1}{2} \gamma_{xy} \mathbf{j} + \frac{1}{2} \gamma_{xz} \mathbf{k} \\ &= \frac{\partial u}{\partial x} \mathbf{i} + \frac{1}{2} \left(\frac{\partial v}{\partial x} + \frac{\partial u}{\partial y} \right) \mathbf{j} + \frac{1}{2} \left(\frac{\partial w}{\partial x} + \frac{\partial u}{\partial z} \right) \mathbf{k} \\ &= \frac{1}{2} \left(\nabla u + \frac{\partial \mathbf{u}}{\partial x} \right) \end{aligned} \quad \begin{aligned} \mathbf{e}_y &= \frac{1}{2} \left(\nabla v + \frac{\partial \mathbf{u}}{\partial y} \right) \\ \mathbf{e}_z &= \frac{1}{2} \left(\nabla w + \frac{\partial \mathbf{u}}{\partial z} \right) \end{aligned}$$

17

Displacement and strain

- Strain dyadic:

$$\mathbf{E} = \mathbf{i} \mathbf{e}_x + \mathbf{j} \mathbf{e}_y + \mathbf{k} \mathbf{e}_z$$

$$\mathbf{E} = \frac{1}{2} (\nabla \mathbf{u} + \mathbf{u} \nabla)$$

where

$$\begin{aligned} \nabla \mathbf{u} &= \mathbf{i} \frac{\partial}{\partial x} \mathbf{u} + \mathbf{j} \frac{\partial}{\partial y} \mathbf{u} + \mathbf{k} \frac{\partial}{\partial z} \mathbf{u} \\ \mathbf{u} \nabla &= \frac{\partial}{\partial x} \mathbf{u} \mathbf{i} + \frac{\partial}{\partial y} \mathbf{u} \mathbf{j} + \frac{\partial}{\partial z} \mathbf{u} \mathbf{k} \end{aligned}$$

18

Generalized Hooke's law and Navier's equation

- Stress-displacement relations:

$$\sigma_x = 2G \frac{\partial u}{\partial x} + \lambda \operatorname{div} \mathbf{u} \quad \tau_{xy} = G \left(\frac{\partial v}{\partial x} + \frac{\partial u}{\partial y} \right) \quad \tau_{xz} = G \left(\frac{\partial w}{\partial x} + \frac{\partial u}{\partial z} \right)$$

$$\tau_x = \sigma_x \mathbf{i} + \tau_{xy} \mathbf{j} + \tau_{xz} \mathbf{k} = G \left(\nabla u + \frac{\partial \mathbf{u}}{\partial x} \right) + (\lambda \operatorname{div} \mathbf{u}) \mathbf{i} = 2G \mathbf{e}_x + (\lambda \operatorname{div} \mathbf{u}) \mathbf{i}$$

$$\tau_y = \tau_{yx} \mathbf{i} + \sigma_y \mathbf{j} + \tau_{yz} \mathbf{k} = G \left(\nabla v + \frac{\partial \mathbf{u}}{\partial y} \right) + (\lambda \operatorname{div} \mathbf{u}) \mathbf{j} = 2G \mathbf{e}_y + (\lambda \operatorname{div} \mathbf{u}) \mathbf{j}$$

$$\tau_z = \tau_{zx} \mathbf{i} + \tau_{zy} \mathbf{j} + \sigma_z \mathbf{k} = G \left(\nabla w + \frac{\partial \mathbf{u}}{\partial z} \right) + (\lambda \operatorname{div} \mathbf{u}) \mathbf{k} = 2G \mathbf{e}_z + (\lambda \operatorname{div} \mathbf{u}) \mathbf{k}$$

$$\Sigma = 2G \mathbf{E} + \lambda \mathbf{E}$$

where $\mathbf{I} = \mathbf{i}\mathbf{i} + \mathbf{j}\mathbf{j} + \mathbf{k}\mathbf{k}$ is the unit dyadic

19

Generalized Hooke's law and Navier's equation

- Stress-displacement relations:

$$\Sigma = 2G \mathbf{E} + \lambda \mathbf{E} = G (\nabla \mathbf{u} + \mathbf{u} \nabla) + \lambda \nabla \cdot \mathbf{u}$$

$$\operatorname{div} \Sigma + \mathbf{f} = 0$$

$$G \nabla^2 \mathbf{u} + G \nabla \cdot (\mathbf{u} \nabla) + \nabla \cdot (\lambda \nabla \cdot \mathbf{u}) + \mathbf{f} = 0$$

$$G \nabla^2 \mathbf{u} + (\lambda + G) \nabla \nabla \cdot \mathbf{u} + \mathbf{f} = 0$$

since $\nabla \cdot (\mathbf{u} \nabla) = \nabla \cdot \mathbf{u} \nabla = (\nabla \cdot \mathbf{u}) \nabla = \nabla (\nabla \cdot \mathbf{u}) = \nabla \nabla \cdot \mathbf{u}$

$$\nabla \cdot (\lambda \nabla \cdot \mathbf{u}) = \nabla (\nabla \cdot \mathbf{u})$$

20

Compatibility equations

- Curl and conjugate curl of dyadic:

$$\nabla \times \varphi = \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ \frac{\partial}{\partial x} & \frac{\partial}{\partial y} & \frac{\partial}{\partial z} \\ \mathbf{F}_x & \mathbf{F}_y & \mathbf{F}_z \end{vmatrix} = \mathbf{i} \times \frac{\partial \varphi}{\partial x} + \mathbf{j} \times \frac{\partial \varphi}{\partial y} + \mathbf{k} \times \frac{\partial \varphi}{\partial z}$$

where $\varphi = \mathbf{i} \mathbf{F}_x + \mathbf{j} \mathbf{F}_y + \mathbf{k} \mathbf{F}_z$

$$\varphi \times \nabla = \frac{\partial \varphi}{\partial x} \times \mathbf{i} + \frac{\partial \varphi}{\partial y} \times \mathbf{j} + \frac{\partial \varphi}{\partial z} \times \mathbf{k}$$

$$\nabla \times \nabla \varphi + \varphi \nabla \times \nabla = 0$$

21

Compatibility equations

- Curl and conjugate curl of dyadic:

$$\mathbf{E} = \frac{1}{2}(\nabla \mathbf{u} + \mathbf{u} \nabla) \quad \mathbf{\Omega} = \frac{1}{2}(\nabla \mathbf{u} - \mathbf{u} \nabla) \quad \text{Rotation dyadic}$$

$$\mathbf{E} + \mathbf{\Omega} = \nabla \mathbf{u} \quad \nabla \times \mathbf{E} + \nabla \times \mathbf{\Omega} = 0$$

$$\nabla \times \mathbf{E} \times \nabla = 0 \quad \text{Compatibility equations}$$

$$\nabla^2 \mathbf{\Sigma} + \frac{1}{1+\nu} \nabla \nabla \Theta = - \frac{\nu}{1-\nu} \mathbf{I} \nabla \cdot \mathbf{F} + \mathbf{F} \nabla + \nabla \mathbf{F} = 0$$

Compatibility equations in terms of stress

22



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23
