

**MAE5009: Continuum Mechanics B**  
**Assignment 04: Formulation of Problems in Elasticity**  
**Due November 2, 2021**

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1. Verify the following equations for plane strain problems with constant  $f_x$  and  $f_y$ :

Solution: (a)  $\frac{\partial}{\partial y} \nabla^2 u = \frac{\partial}{\partial x} \nabla^2 v$

According to the displacement formulation of the plane strain problem, we can know,

$$G \nabla^2 u + (\lambda + G) \frac{\partial}{\partial x} \left( \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} \right) + f_x = 0$$

$$G \nabla^2 v + (\lambda + G) \frac{\partial}{\partial y} \left( \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} \right) + f_y = 0$$

Then:

$$G \frac{\partial}{\partial y} \nabla^2 u = -(\lambda + G) \frac{\partial^2}{\partial x \partial y} \left( \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} \right)$$

$$G \frac{\partial}{\partial x} \nabla^2 v = -(\lambda + G) \frac{\partial^2}{\partial x \partial y} \left( \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} \right)$$

Thus:

$$\frac{\partial}{\partial y} \nabla^2 u = -\frac{\lambda + G}{G} \frac{\partial^2}{\partial x \partial y} \left( \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} \right) \Rightarrow \frac{\partial}{\partial y} \nabla^2 u = \frac{\partial}{\partial x} \nabla^2 v$$

$$\frac{\partial}{\partial x} \nabla^2 v = -\frac{\lambda + G}{G} \frac{\partial^2}{\partial x \partial y} \left( \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} \right)$$

(b)  $\frac{\partial}{\partial x} \nabla^2 u + \frac{\partial}{\partial y} \nabla^2 v = 0$

Solution:

According to (a)

$$\nabla^2 u = -\frac{\lambda + G}{G} \frac{\partial}{\partial x} \left( \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} \right) - \frac{f_x}{G}$$

$$\nabla^2 v = -\frac{\lambda + G}{G} \frac{\partial}{\partial y} \left( \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} \right) - \frac{f_y}{G}$$

Then:

$$\frac{\partial}{\partial x} \nabla^2 u = -\frac{\lambda + G}{G} \frac{\partial^2}{\partial x^2} \left( \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} \right)$$

$$\frac{\partial}{\partial y} \nabla^2 v = -\frac{\lambda + G}{G} \frac{\partial^2}{\partial y^2} \left( \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} \right)$$

And:

$$\frac{\partial}{\partial x} \nabla^2 u + \frac{\partial}{\partial y} \nabla^2 v = -\frac{\lambda + G}{G} \left( \frac{\partial}{\partial x} \nabla^2 u + \frac{\partial}{\partial y} \nabla^2 v \right)$$

Since  $-\frac{\lambda + G}{G} \neq 0$

So  $\frac{\partial}{\partial x} \nabla^2 u + \frac{\partial}{\partial y} \nabla^2 v = 0$

(c)  $\nabla^4 u = \nabla^4 v = 0$ , where  $\nabla^4 = \nabla^2 (\nabla^2)$ .

Solution:

$$\nabla^4 u = \nabla^2 (\nabla^2 u) = \frac{\partial^2}{\partial x^2} \Delta^2 u + \frac{\partial^2}{\partial y^2} \Delta^2 u$$

$$\nabla^4 u = \nabla^2 (\nabla^2 v) = \frac{\partial^2}{\partial x^2} \Delta^2 v + \frac{\partial^2}{\partial y^2} \Delta^2 v$$

According to (a)

$$\frac{\partial}{\partial y} \nabla^2 u = \frac{\partial}{\partial x} \nabla^2 v, \text{ then } \frac{\partial^2}{\partial y^2} \nabla^2 u = \frac{\partial^2}{\partial x \partial y} \nabla^2 v, \frac{\partial^2}{\partial x^2} \nabla^2 v = \frac{\partial^2}{\partial x \partial y} \nabla^2 u$$

According to (b)

$$\frac{\partial}{\partial x} \nabla^2 u = -\frac{\partial}{\partial y} \nabla^2 v, \text{ then } \frac{\partial^2}{\partial x^2} \nabla^2 u = -\frac{\partial^2}{\partial x \partial y} \nabla^2 v, \frac{\partial^2}{\partial y^2} \nabla^2 v = -\frac{\partial^2}{\partial x \partial y} \nabla^2 u$$

Thus:

$$\nabla^4 u = -\frac{\partial^2}{\partial x \partial y} \nabla^2 v + \frac{\partial^2}{\partial x \partial y} \nabla^2 v = 0$$

$$\nabla^4 v = \frac{\partial^2}{\partial x \partial y} \nabla^2 u - \frac{\partial^2}{\partial x \partial y} \nabla^2 u = 0$$

$$\therefore \nabla^4 u = \nabla^4 v = 0$$

2. A bar of constant mass density  $\rho$  hangs under its own weight and is supported by the uniform stress  $\sigma_0$  as shown in the figure. Assume that the stresses  $\sigma_x, \sigma_y, \tau_{xy}, \tau_{xz}$  and  $\tau_{yz}$  are all zero,

(a) based on the above assumption, reduce 15 governing equations to seven equations

Solution: in terms of  $\sigma_z, \epsilon_x, \epsilon_y, \epsilon_z, u, v$  and  $w$

From the stress-strain relations:

$$\sigma_x = 2G\epsilon_x + \lambda(\epsilon_x + \epsilon_y + \epsilon_z) = 0$$

$$\sigma_y = 2G\epsilon_y + \lambda(\epsilon_x + \epsilon_y + \epsilon_z) = 0$$

$$\sigma_z = 2G\epsilon_z + \lambda(\epsilon_x + \epsilon_y + \epsilon_z) = E\epsilon_z$$

we can get:

$$\epsilon_x = \epsilon_y = -\epsilon_z, \gamma_{xy} = \gamma_{xz} = \gamma_{yz} = 0$$

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According to the equilibrium equations:

$$f_x = f_y = 0$$

$$\frac{\partial \sigma_z}{\partial z} = f_z$$

For the strain displacement:

$$\epsilon_x = \frac{\partial u}{\partial x}, \epsilon_y = \frac{\partial v}{\partial y}, \epsilon_z = \frac{\partial w}{\partial z}$$

$$\gamma_{xy} = \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} = 0, \gamma_{yz} = \frac{\partial v}{\partial z} + \frac{\partial w}{\partial y} = 0, \gamma_{zx} = \frac{\partial w}{\partial x} + \frac{\partial u}{\partial z} = 0$$

(b) integrate the equilibrium equation to show that

$$\sigma_z = \rho g z$$

where  $g$  is the acceleration due to gravity. Also show that the prescribed boundary conditions are satisfied by this solution

Solution:

Let the cross-section area be  $A$ , the force equilibrium for any  $z$  should be satisfied.

$$\sigma_z \cdot A - mg = 0$$

$$\text{Then } mg = \rho V \cdot g = \rho g \cdot A z \Rightarrow \sigma_z \cdot A - \rho g A z = 0$$

We can prove  $\sigma_z = \rho g z$

$$\text{When } z=l, \sigma_z = \rho g l = \sigma_0$$

$$\text{When } z=0, \sigma_z = 0$$

(c) find  $\epsilon_x, \epsilon_y, \epsilon_z$  from the generalized Hooke's law

Solution:

$$\sigma_x = 2G\epsilon_x + \lambda(\epsilon_x + \epsilon_y + \epsilon_z) = 0$$

$$\sigma_y = 2G\epsilon_y + \lambda(\epsilon_x + \epsilon_y + \epsilon_z) = 0$$

We can get:

$$\epsilon_x = \epsilon_y = -\nu\epsilon_z \quad \sigma_z = 2G\epsilon_z + \lambda(\epsilon_x + \epsilon_y + \epsilon_z) = E\epsilon_z$$

So

$$\epsilon_z = \frac{\sigma_z}{E} = \frac{\rho g z}{E}, \quad \epsilon_x = \epsilon_y = -\nu\epsilon_z = -\nu \frac{\rho g z}{E}$$



(d) if the displacement and rotation components are zero at the point  $(0,0,l)$ , determine the displacement component  $u$  and  $v$

Solution:

According to (a)(b)(c):

$$\epsilon_x = \epsilon_y = \frac{\partial u}{\partial x} = \frac{\partial v}{\partial y} = -\nu \frac{\rho g z}{E} \quad \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = \gamma_{xy} = 0$$

Then we can make assumptions that

$$u = -\nu \frac{\rho g z}{E} x + C_1(y, z) \quad v = -\nu \frac{\rho g z}{E} y + C_2(x, z)$$

$$w = \frac{\rho g}{2E} z^2 + C_3(x, y)$$

Since the displacement and rotation components are zero at the point  $(0,0,l)$ , we can get

$$C_1(y, z)|_{(0,0,l)} = 0, \quad C_2(x, z)|_{(0,0,l)} = 0$$

$$C_3(x, y)|_{(0,0)} = -\frac{\rho g}{2E} l^2$$

According to  $\frac{\partial u}{\partial x} = \frac{\partial v}{\partial y}$  and  $\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$ ,

$$\frac{\partial^2 u}{\partial x^2} = \frac{\partial^2 v}{\partial x \partial y}, \quad \frac{\partial^2 u}{\partial y^2} = -\frac{\partial^2 v}{\partial x \partial y} \Rightarrow \frac{\partial^2 u}{\partial x^2} = -\frac{\partial^2 u}{\partial y^2}$$

Because  $\frac{\partial^2 u}{\partial x^2} = 0$ , then  $\frac{\partial^2 u}{\partial y^2} = 0 = \frac{\partial^2 C_1(y, z)}{\partial y^2}$

The  $\frac{\partial C_1(y, z)}{\partial y}$  is no function of  $y$ . With the

Since  $\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} = 0$ , then  $\frac{\partial C_1(y, z)}{\partial y} = -\frac{\partial C_2(x, z)}{\partial x} = f_1(z)$

$$\frac{\partial C_1(y, z)}{\partial y} \Big|_{(0,0,l)} = 0, \quad f_1(z) \Big|_{z=l} = 0$$

Then  $C_1(y, z) = f_1(z)y$ ,  $C_2(x, z) = -f_1(z)x$

$$u = -\frac{\nu \rho g}{E} x z + f_1(z)y, \quad v = -\frac{\nu \rho g}{E} y z - f_1(z)x$$

With the same way about  $u$  and  $w$ , where

$A_1$  is a constant, So  $f_1(z) = A_1 z - A_1 l$  for the

boundary condition that  $f_1(z)|_{z=l} = 0$

Since:

$$\frac{\partial u}{\partial z} + \frac{\partial w}{\partial x} = \left(-\frac{\nu \rho g}{E} x + A_1 y\right) + \frac{\partial C_3(x, y)}{\partial x} = \frac{\nu \rho g}{2E} x^2 - A_1 xy + f_2(y)$$

$$\frac{\partial v}{\partial z} + \frac{\partial w}{\partial y} = \left(-\frac{\nu \rho g}{E} y - A_1 x\right) + \frac{\partial C_3(x, y)}{\partial y} = \frac{\nu \rho g}{2E} y^2 + A_1 xy + f_3(x)$$

The boundary condition  $C_3(x, y)|_{(0,0)} = -\frac{\rho g}{2E} l^2$

Then  $A_1 = 0$ ,  $f_1(z) = 0$ ,  $f_2(y) = \frac{\nu \rho g}{2E} y^2 - \frac{\rho g}{2E} l^2$ ,  $f_3(x) = \frac{\nu \rho g}{2E} x^2 - \frac{\rho g}{2E} l^2$

$$C_3(x, y) = \frac{\nu \rho g}{2E} x^2 + \frac{\nu \rho g}{2E} y^2 - \frac{\rho g}{2E} l^2$$

same way,  $\frac{\partial C_2(x, z)}{\partial x}$  is not a function of  $x$   $u = -\frac{\nu \rho g}{E} xz$ ,  $v = -\frac{\nu \rho g}{E} yz$

(e) prove that

$$w = \frac{\rho g}{2E} (z^2 + \nu x^2 + \nu y^2 - l^2)$$

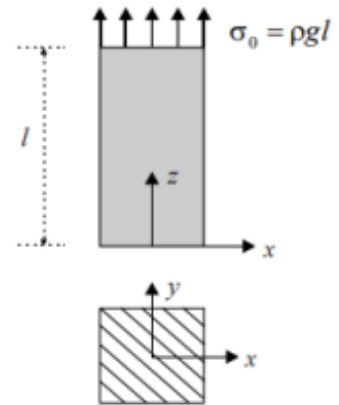
Solution:

According to (d)

$$C_3(x, y) = \frac{\nu \rho g}{2E} x^2 + \frac{\nu \rho g}{2E} y^2 - \frac{\rho g}{2E} l^2$$

Then

$$\begin{aligned} w &= \frac{\rho g}{2E} z^2 + C_3(x, y) \\ &= \frac{\rho g}{2E} z^2 + \frac{\nu \rho g}{2E} x^2 + \frac{\nu \rho g}{2E} y^2 - \frac{\rho g}{2E} l^2 \\ &= \frac{\rho g}{2E} (z^2 + \nu x^2 + \nu y^2 - l^2) \end{aligned}$$



3. Express the boundary conditions for the following plate subjected to plate strain condition. The surface forces are functions of  $x$  and  $y$  only.

Solution:

Surface  $f(x, y)$  for plane strain problem

$$\epsilon_z = \gamma_{yz} = \gamma_{xz} = 0$$

The boundary condition: (The surface is in balance)

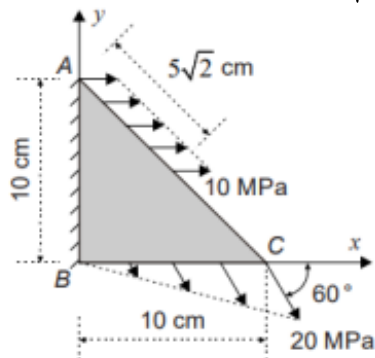
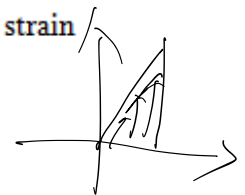
AB:  $\epsilon_x = \epsilon_y = 0$ ,  $x = 0$

$$F_1 = \int_0^{5\sqrt{2}} \sigma_{xx} dy = 50\sqrt{2} \text{ MPa}$$

$$F_2 = \int_0^{10} \sigma_{yy} dx = 100 \text{ MPa}$$

$$\sigma_x = \frac{F_{ABx}}{A} = \frac{F_1}{A} = \frac{50\sqrt{2}}{10} = 5\sqrt{2} \text{ MPa}$$

$$\sigma_y = \frac{F_{AB y}}{A} = \frac{F_2}{A} = 10 \text{ MPa}$$



$$\therefore AB: \begin{cases} \sigma_x = 5\sqrt{2} \text{ MPa}, & x = 0, 0 \leq y \leq 10 \\ \sigma_y = 10 \text{ MPa}, & x = 0, 0 \leq y \leq 10 \end{cases}$$

$$BC: \begin{cases} \sigma_x = 2x \cos 60^\circ, & y = 0, 0 \leq x \leq 10 \\ \sigma_y = 2x \sin 60^\circ, & y = 0, 0 \leq x \leq 10 \end{cases}$$

$$AC: \begin{cases} \sigma_x = 10 \text{ MPa}, & y = 10 - x \\ \sigma_y = 0 \text{ MPa}, & y = 10 - x \end{cases} \quad 0 < x < 5$$

$$AC_2: \begin{cases} \sigma_x = 0 \text{ MPa}, & y = 10 - x \\ \sigma_y = 0 \text{ MPa}, & y = 10 - x \end{cases}$$