



南方科技大学

SOUTHERN UNIVERSITY OF SCIENCE AND TECHNOLOGY

MAE5009

Continuum Mechanics B

Session 07: Tensor Notation

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Notation

- Tensor notation, indicial notation:
 - If one of the equations is written, the other expressions may be derived by cyclic permutation
 - We could use the tensor notation to condense the governing equations
- Symbolic notation:
 - Vector-dyadic notations, vector notation

Equilibrium equations

$$\begin{cases} \frac{\partial \sigma_x}{\partial x} + \frac{\partial \tau_{yx}}{\partial y} + \frac{\partial \tau_{zx}}{\partial z} + f_x = 0 \\ \frac{\partial \tau_{xy}}{\partial x} + \frac{\partial \sigma_y}{\partial y} + \frac{\partial \tau_{zy}}{\partial z} + f_y = 0 \\ \frac{\partial \tau_{xz}}{\partial x} + \frac{\partial \tau_{yz}}{\partial y} + \frac{\partial \sigma_z}{\partial z} + f_z = 0 \end{cases}$$

Compatibility equations

$$\begin{cases} \frac{\partial^2 \epsilon_x}{\partial y^2} + \frac{\partial^2 \epsilon_y}{\partial x^2} = \frac{\partial^2 \gamma_{xy}}{\partial x \partial y} \\ \frac{\partial^2 \epsilon_y}{\partial z^2} + \frac{\partial^2 \epsilon_z}{\partial y^2} = \frac{\partial^2 \gamma_{yz}}{\partial y \partial z} \\ \frac{\partial^2 \epsilon_z}{\partial x^2} + \frac{\partial^2 \epsilon_x}{\partial z^2} = \frac{\partial^2 \gamma_{zx}}{\partial z \partial x} \end{cases}$$

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Indicial notation and vector transformations

- Using x_1, x_2 and x_3 to represent the three Cartesian coordinates:

x

y

z

\rightarrow

x_1

x_2

x_3

- For a vector transformed to another coordinate system x'_1, x'_2 and x'_3 :

$\begin{bmatrix} x'_1 \\ x'_2 \\ x'_3 \end{bmatrix} = \begin{bmatrix} a_{11} & a_{12} & a_{13} \\ a_{21} & a_{22} & a_{23} \\ a_{31} & a_{32} & a_{33} \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}$

\rightarrow

$\begin{bmatrix} x'_1 \\ x'_2 \\ x'_3 \end{bmatrix} = \mathbf{R} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}$

Direction cosines
(Transformational matrix \mathbf{R})

	x_1	x_2	x_3
x'_1	a_{11}	a_{12}	a_{13}
x'_2	a_{21}	a_{22}	a_{23}
x'_3	a_{31}	a_{32}	a_{33}

$$\begin{cases} x'_1 = x_1 a_{11} + x_2 a_{12} + x_3 a_{13} \\ x'_2 = x_1 a_{21} + x_2 a_{22} + x_3 a_{23} \\ x'_3 = x_1 a_{31} + x_2 a_{32} + x_3 a_{33} \end{cases}$$

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Indicial notation and vector transformations

The three equations can be condensed to:

$$\begin{cases} x'_1 = x_1a_{11} + x_2a_{12} + x_3a_{13} \\ x'_2 = x_1a_{21} + x_2a_{22} + x_3a_{23} \\ x'_3 = x_1a_{31} + x_2a_{32} + x_3a_{33} \end{cases} \rightarrow \begin{cases} x'_1 = \sum_{i=1}^3 x_i a_{1i} \\ x'_2 = \sum_{i=1}^3 x_i a_{2i} \\ x'_3 = \sum_{i=1}^3 x_i a_{3i} \end{cases} \rightarrow x'_j = \sum_{i=1}^3 x_i a_{ji}, j=1,2,3$$

题目: 学会转换

$$x'_j = x_i a_{ji} \quad i, j = 1, 2, 3$$

Tensor notation:
If a repeated alphabetic subscript appears in one monomial, automatic summation over the range of this subscript is required

$$\textcircled{1} b_{ij}b_{ik} + b_{iz}b_{zk} + b_{iz}b_{zk} \quad \text{给定 } j, i, k \Rightarrow 3 \times 9$$

☆ $b_{ij}b_{ik}$
必考

dummy index: $j=1,2,3$ 加起来

free index: $V_i = V_1, V_2, V_3$ 3个

$V_{ij}: 3 \times 3$

$V_{ijklmn}: 3^6$

i is dummy index 重复的 (哑标)
 j is free index 不重复的

Cartesian tensor notation

- A repeated subscript is called **dummy** index
- Non-repeated subscript is called **free** index
- Both dummy and free index can be arbitrary
- A subscript may appear no more than twice in each monomial
- If a subscript appears only once in a monomial, it must appear just once in each other monomial

$$x'_k = x_i a_{ki} \quad i, k = 1, 2, 3$$

$$x'_j = x_i a_{ji} \quad i, j = 1, 2, 3$$

$$b_{ij}b_{jk} \quad i, j, k = 1, 2, 3 \rightarrow \begin{matrix} i=1, k=1: b_{11}b_{11} + b_{12}b_{21} + b_{13}b_{31} & i=2, k=1 & i=3, k=1 \\ k=2: b_{11}b_{12} + b_{12}b_{22} + b_{13}b_{32} & k=2 & k=2 \\ k=3: b_{11}b_{13} + b_{12}b_{23} + b_{13}b_{33} & k=3 & k=3 \end{matrix}$$

$$b_{11}b_{1k} + b_{12}b_{2k} + b_{13}b_{3k}$$

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Cartesian tensor notation

- For a vector transformed back from x'_1, x'_2 and x'_3 to coordinate system x_1, x_2 and x_3 :

$$\begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} = \begin{bmatrix} a_{11} & a_{21} & a_{31} \\ a_{12} & a_{22} & a_{32} \\ a_{13} & a_{23} & a_{33} \end{bmatrix} \begin{bmatrix} x'_1 \\ x'_2 \\ x'_3 \end{bmatrix} \rightarrow \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} = \mathbf{R}^T \begin{bmatrix} x'_1 \\ x'_2 \\ x'_3 \end{bmatrix}$$

Direction cosines
(Transformational matrix \mathbf{R}^T)

	x'_1	x'_2	x'_3
x_1	a_{11}	a_{21}	a_{31}
x_2	a_{12}	a_{22}	a_{32}
x_3	a_{13}	a_{23}	a_{33}

$$\begin{cases} x_1 = x'_1a_{11} + x'_2a_{21} + x'_3a_{31} \\ x_2 = x'_1a_{12} + x'_2a_{22} + x'_3a_{32} \\ x_3 = x'_1a_{13} + x'_2a_{23} + x'_3a_{33} \end{cases}$$

Direction cosines
(Transformational matrix \mathbf{R})

	x_1	x_2	x_3
x'_1	a_{11}	a_{12}	a_{13}
x'_2	a_{21}	a_{22}	a_{23}
x'_3	a_{31}	a_{32}	a_{33}

$$x_j = x'_i a_{ij} \quad i, j = 1, 2, 3$$

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Cartesian tensor notation

- For a vector transformed back from x'_1, x'_2 and x'_3 to coordinate system x_1, x_2 and x_3 :

$x_j = x'_i a_{ij} \quad i, j = 1, 2, 3$

$x_j = x'_i a_{ij} = x'_i a_{ik} a_{kj} \quad i, j, k = 1, 2, 3$

$a_{ik} a_{kj} = \mathbf{I} \quad i, j, k = 1, 2, 3$

$a_{ii} a_{ij} = 1 \quad \text{if } i = j$
 $a_{ii} a_{ij} = 0 \quad \text{if } i \neq j$

$a_{11}^2 + a_{21}^2 + a_{31}^2 = 1 \quad a_{11} a_{12} + a_{21} a_{22} + a_{31} a_{32} = 0$
 $a_{12}^2 + a_{22}^2 + a_{32}^2 = 1 \quad a_{11} a_{13} + a_{21} a_{23} + a_{31} a_{33} = 0$
 $a_{13}^2 + a_{23}^2 + a_{33}^2 = 1 \quad a_{12} a_{13} + a_{22} a_{23} + a_{32} a_{33} = 0$

Direction cosines
(Transformational matrix **R**)

	x_1	x_2	x_3
x'_1	a_{11}	a_{12}	a_{13}
x'_2	a_{21}	a_{22}	a_{23}
x'_3	a_{31}	a_{32}	a_{33}

$\begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} = \mathbf{R}^T \mathbf{R} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}$

$\mathbf{R}^T \mathbf{R} = \mathbf{I}$

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Definition of vector

- A set of three quantities F_i referred to a coordinate system x_i and transformed to another coordinate system x'_i by the following equation is defined as a vector:

$F'_j = F_i a_{ji} \quad i, j = 1, 2, 3$

Not any set of three scalar quantities attached to the x_1, x_2, x_3 system can be called a vector

Direction cosines
(Transformational matrix **R**)

	x_1	x_2	x_3
x'_1	a_{11}	a_{12}	a_{13}
x'_2	a_{21}	a_{22}	a_{23}
x'_3	a_{31}	a_{32}	a_{33}

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Higher order tensors

- Consider any two vectors u_i and $v_{k'}$ in another coordinate system their coordinates are:

$u'_j = u_i a_{ji}$

$v'_j = v_k a_{jk}$

$u'_j v'_i = a_{ji} a_{ik} u_j v_k \quad A'_{ji} = a_{ji} a_{ik} A_{ik}$

Any group of nine scalar quantities A_{ik} referred to a coordinate system x_i and which transforms to a group of nine quantities referred to another coordinate system x'_i by the above rule is called a tensor of second order

Third order tensor: $w'_{ijk} = a_{ip} a_{jq} a_{kr} w_{pqr}$

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Gradient of scalar and vector field

- The gradient of a scalar field is a vector field:

$$U = U(x_1, x_2, x_3)$$

- and its gradient in two coordinate systems x_i and x'_i :

$$\frac{\partial U}{\partial x_i}, \frac{\partial U}{\partial x'_j} \rightarrow \frac{\partial U}{\partial x'_j} = \frac{\partial U}{\partial x_i} \frac{\partial x_i}{\partial x'_j} = a_{ij} \frac{\partial U}{\partial x_i}$$

- The gradient of a vector is a second order tensor:

$$u_i, u'_j \rightarrow \frac{\partial u_i}{\partial x_k}, \frac{\partial u'_j}{\partial x'_l}$$

$$\frac{\partial u'_j}{\partial x'_l} = \frac{\partial u'_j}{\partial x_k} \frac{\partial x_k}{\partial x'_l} = a_{kl} \frac{\partial u'_j}{\partial x_k} = a_{kl} \frac{\partial (u_i a_{ji})}{\partial x_k} = a_{ji} a_{kl} \frac{\partial u_i}{\partial x_k}$$

Since: $\frac{\partial a_{ji}}{\partial x_k} = \frac{\partial}{\partial x_k} \left(\frac{\partial x_j}{\partial x'_i} \right) = 0$ a_{ji} is constant

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The Kronecker delta

- The gradient of the position vector x_i is a second order tensor:

$$\begin{aligned} \frac{\partial x_i}{\partial x_k} &\rightarrow \frac{\partial x_1}{\partial x_1} = 1 & \frac{\partial x_1}{\partial x_2} = 0 & \frac{\partial x_1}{\partial x_3} = 0 \\ \frac{\partial x_i}{\partial x_k} &= 1 \quad \text{if } i = k \\ \frac{\partial x_i}{\partial x_k} &= 0 \quad \text{if } i \neq k \end{aligned} \rightarrow \begin{aligned} \delta_{ik} &= 1 \quad \text{if } i = k \\ \delta_{ik} &= 0 \quad \text{if } i \neq k \end{aligned} \quad \text{Kronecker delta}$$
$$\delta_{ik} = \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

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The Kronecker delta

- The Kronecker delta has identical components in any coordinate systems. Applying the transformation:

$$\delta'_{ji} = a_{ij} a_{kl} \delta_{ik} = a_{ij} a_{il}$$
$$\begin{aligned} a_{ij} a_{il} &= 1 \quad \text{if } j = l \\ a_{ij} a_{il} &= 0 \quad \text{if } j \neq l \end{aligned} \rightarrow \begin{aligned} \delta'_{ji} &= 1 \quad \text{if } j = l \\ \delta'_{ji} &= 0 \quad \text{if } j \neq l \end{aligned}$$
$$\delta'_{ji} = \delta_{ji}$$

Tensors with identical components in any coordinate system are called isotropic tensors

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The Kronecker delta

- The Kronecker delta has identical components in any coordinate systems. Applying the transformation:

$\delta_{ik} u_k = \delta_{i1} u_1 + \delta_{i2} u_2 + \delta_{i3} u_3$ ① $\delta_{i1} u_1 + \delta_{i2} u_2 + \delta_{i3} u_3$

$i = 1: \delta_{ik} u_k = u_1$
 $i = 2: \delta_{ik} u_k = u_2$
 $i = 3: \delta_{ik} u_k = u_3$

$\delta_{ik} u_k = u_i$ = u_i

Kronecker delta is sometimes called the substitution tensor

$\frac{\partial x_i}{\partial x_k} = \delta_{ik}$ ✗/3

$\frac{\partial u_j}{\partial x_i} = u_{j,i}$ rector

$\frac{\partial^2 u_j}{\partial x_i \partial x_k} = \delta_{ik} u_{j,ik} = \frac{\partial^2 u_j}{\partial x_i \partial x_k} = u_{j,ik}$

where: $u_j = \frac{\partial u}{\partial x_j}$ $\frac{\partial u_j}{\partial x_k} = u_{j,ik}$

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The alternating tensor

- Another isotropic tensor, ϵ_{ikm}

$\epsilon_{ikm} = \begin{cases} 0 & \text{If any two of the subscripts } i, k, m \text{ are equal} \\ +1 & \text{If the subscripts } i, k, m \text{ are unequal but in cyclic order of } 1\ 2\ 3 \\ -1 & \text{If the subscripts } i, k, m \text{ are unequal and in noncyclic order of } 1\ 2\ 3 \end{cases}$

- The ϵ_{ikm} is a third order tensor:

$\epsilon'_{jln} = a_{ij} a_{kl} a_{mn} \epsilon_{ikm}$

$= a_{1j} a_{2l} a_{3n} + a_{2j} a_{3l} a_{1n} + a_{3j} a_{1l} a_{2n}$

$- a_{1j} a_{3l} a_{2n} - a_{2j} a_{1l} a_{3n} - a_{3j} a_{2l} a_{1n}$

$= \begin{vmatrix} a_{1j} & a_{1l} & a_{1n} \\ a_{2j} & a_{2l} & a_{2n} \\ a_{3j} & a_{3l} & a_{3n} \end{vmatrix}$

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State of stress at a point

- Stress can be described as a tensor:

$\tau_{ik} = \begin{bmatrix} \sigma_x & \tau_{xy} & \tau_{xz} \\ \tau_{yx} & \sigma_y & \tau_{yz} \\ \tau_{zx} & \tau_{zy} & \sigma_z \end{bmatrix} = \begin{bmatrix} \tau_{11} & \tau_{12} & \tau_{13} \\ \tau_{21} & \tau_{22} & \tau_{23} \\ \tau_{31} & \tau_{32} & \tau_{33} \end{bmatrix}$

$\tau'_{jn} = a_{ij} a_{kn} \tau_{ik}$

Stress transformation:

$\begin{bmatrix} \sigma'_x & \tau'_{xy} & \tau'_{xz} \\ \tau'_{yx} & \sigma'_y & \tau'_{yz} \\ \tau'_{zx} & \tau'_{zy} & \sigma'_z \end{bmatrix} = \mathbf{R} \cdot \begin{bmatrix} \sigma_x & \tau_{xy} & \tau_{xz} \\ \tau_{yx} & \sigma_y & \tau_{yz} \\ \tau_{zx} & \tau_{zy} & \sigma_z \end{bmatrix} \cdot \mathbf{R}^T$

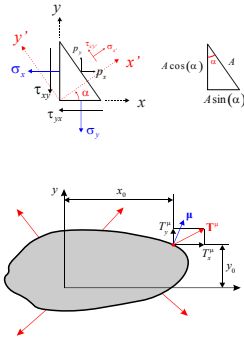
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Utilization of tensor notation

- Stress transformation on an inclined surface:

$$\begin{cases} T_x^\mu = \sigma_{x0}\mu_x + \tau_{yx0}\mu_y + \tau_{zx0}\mu_z \\ T_y^\mu = \tau_{xy0}\mu_x + \sigma_{y0}\mu_y + \tau_{zy0}\mu_z \\ T_z^\mu = \tau_{xz0}\mu_x + \tau_{yz0}\mu_y + \sigma_{z0}\mu_z \end{cases}$$

$T_i^\mu = \tau_{ji}\mu_j$



$\frac{\partial \tau_{11}}{\partial x_1} + \frac{\partial \tau_{21}}{\partial x_2} + \frac{\partial \tau_{31}}{\partial x_3} + f_1 = 0$

$\frac{\partial \tau_{i1}}{\partial x_i} + f_1 = 0$

\Downarrow

$\tau_{ij,i} + f_j = 0$

Principal axes of stress tensor

$$\begin{vmatrix} \tau_{11} - \tau & \tau_{21} & \tau_{31} \\ \tau_{12} & \tau_{22} - \tau & \tau_{32} \\ \tau_{13} & \tau_{23} & \tau_{33} - \tau \end{vmatrix} = 0$$

$$\tau^3 - I_1 \tau^2 + I_2 \tau - I_3 = 0$$

\Downarrow

$$I_1 = \tau_{11} + \tau_{22} + \tau_{33} = \tau_{ii}$$

$$I_2 = \tau_{11}\tau_{22} + \tau_{22}\tau_{33} + \tau_{33}\tau_{11} - \tau_{12}^2 - \tau_{23}^2 - \tau_{31}^2 = \frac{1}{2}(\tau_{ij}\tau_{kk} - \tau_{ik}\tau_{kj})$$

$$I_3 = \tau_{11}\tau_{22}\tau_{33} + 2\tau_{12}\tau_{23}\tau_{31} - \tau_{11}\tau_{23}^2 - \tau_{22}\tau_{31}^2 - \tau_{33}\tau_{12}^2 = \frac{1}{6}(2\tau_{ij}\tau_{jk}\tau_{ki} - 3\tau_{ij}\tau_{ji}\tau_{kk} + \tau_{ii}\tau_{jj}\tau_{kk})$$
$$= \frac{1}{6}(\epsilon_{ijk}\epsilon_{pqr}\tau_{ip}\tau_{jq}\tau_{kr})$$

??

$$\begin{cases} \frac{\partial \sigma_x}{\partial x} + \frac{\partial \tau_{yx}}{\partial y} + \frac{\partial \tau_{zx}}{\partial z} + f_x = 0 \\ \frac{\partial \tau_{xy}}{\partial x} + \frac{\partial \sigma_y}{\partial y} + \frac{\partial \tau_{zy}}{\partial z} + f_y = 0 \\ \frac{\partial \tau_{xz}}{\partial x} + \frac{\partial \tau_{yz}}{\partial y} + \frac{\partial \sigma_z}{\partial z} + f_z = 0 \end{cases}$$

\Downarrow

$$\frac{\partial \tau_{ik}}{\partial x_i} + f_k = 0 \qquad \tau_{ik,i} + f_k = 0$$

Equations of equilibrium

The stress ellipsoid

$$\begin{aligned} p_x &= \sigma_1 a_{11} \\ p_y &= \sigma_2 a_{12} \\ p_z &= \sigma_3 a_{13} \end{aligned}$$

↓

$$\left(\frac{p_x}{\sigma_1}\right)^2 + \left(\frac{p_y}{\sigma_2}\right)^2 + \left(\frac{p_z}{\sigma_3}\right)^2 = 1$$

p_x , p_y and p_z are the components of the stress vector on the x_1 plane, and the intercepts on the coordinate axes are $\pm\sigma_1$, $\pm\sigma_2$ and $\pm\sigma_3$

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Displacement and strain

• Displacement:

$$\begin{aligned} u_i &= u_i^0 + \frac{\partial u_i}{\partial x_j} dx_j && \text{Taylor series} \\ &= u_i^0 + \frac{1}{2} \left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right) dx_j + \frac{1}{2} \left(\frac{\partial u_i}{\partial x_j} - \frac{\partial u_j}{\partial x_i} \right) dx_j \\ &= u_i^0 + \varepsilon_{ij} dx_j + \omega_{ij} dx_j \end{aligned}$$

✓ Strain component:

$$\begin{aligned} \varepsilon_{ij} &= \frac{1}{2} \left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right) \\ &= \frac{1}{2} (u_{i,j} + u_{j,i}) \end{aligned}$$

✗ Rotation component:

$$\begin{aligned} \omega_{ij} &= \frac{1}{2} \left(\frac{\partial u_i}{\partial x_j} - \frac{\partial u_j}{\partial x_i} \right) \\ &= \frac{1}{2} (u_{i,j} - u_{j,i}) \end{aligned}$$

$$\begin{aligned} \varepsilon_{11} &= \frac{1}{2} \left(\frac{\partial u_1}{\partial x_1} + \frac{\partial u_1}{\partial x_1} \right) = \varepsilon_x \\ \varepsilon_{12} &= \frac{1}{2} \left(\frac{\partial u_1}{\partial x_2} + \frac{\partial u_2}{\partial x_1} \right) = \frac{1}{2} \gamma_{xy} \\ \varepsilon_{13} &= \frac{1}{2} \left(\frac{\partial u_1}{\partial x_3} + \frac{\partial u_3}{\partial x_1} \right) = \frac{1}{2} \gamma_{xz} \dots \\ \omega_{12} &= -\omega_{21} = \frac{1}{2} \left(\frac{\partial u_1}{\partial x_2} - \frac{\partial u_2}{\partial x_1} \right) \\ \omega_{13} &= -\omega_{31} = \frac{1}{2} \left(\frac{\partial u_1}{\partial x_3} - \frac{\partial u_3}{\partial x_1} \right) \\ \omega_{23} &= -\omega_{32} = \frac{1}{2} \left(\frac{\partial u_2}{\partial x_3} - \frac{\partial u_3}{\partial x_2} \right) \dots \end{aligned}$$

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Generalized Hooke's law

$$\begin{bmatrix} \tau_{11} \\ \tau_{22} \\ \tau_{33} \\ \tau_{12} \\ \tau_{23} \\ \tau_{31} \end{bmatrix} = \begin{bmatrix} c_{11} & c_{12} & c_{13} & c_{14} & c_{15} & c_{16} \\ c_{21} & c_{22} & c_{23} & c_{24} & c_{25} & c_{26} \\ c_{31} & c_{32} & c_{33} & c_{34} & c_{35} & c_{36} \\ c_{41} & c_{42} & c_{43} & c_{44} & c_{45} & c_{46} \\ c_{51} & c_{52} & c_{53} & c_{54} & c_{55} & c_{56} \\ c_{61} & c_{62} & c_{63} & c_{64} & c_{65} & c_{66} \end{bmatrix} \begin{bmatrix} \varepsilon_x \\ \varepsilon_y \\ \varepsilon_z \\ \gamma_{xy} \\ \gamma_{yz} \\ \gamma_{zx} \end{bmatrix}$$

$$C_{ijkl} = a_{im} a_{jp} a_{kq} a_{lr} C_{mnpqr}$$

C_{ijkl} is a fourth order tensor

$$\begin{aligned} \varepsilon_x &= \frac{1}{E} [b_x - \nu (b_y + b_z)] && \gamma_{xy} = \frac{1}{G} \tau_{xy} = \frac{1}{2G} \tau_{xy} = \frac{1+\nu}{E} \tau_{xy} \\ \varepsilon_y &= \frac{1}{E} [b_y - \nu (b_x + b_z)] && \gamma_{yz} = \frac{1}{G} \tau_{yz} = \frac{1}{2G} \tau_{yz} = \frac{1+\nu}{E} \tau_{yz} \\ \varepsilon_{xx} &= \frac{1}{E} [b_{xx} - \nu (b_{yy} + b_{zz})] \\ &= \frac{1}{E} [(1+\nu) b_{xx} - \nu (b_{xx} + b_{yy} + b_{zz})] \end{aligned}$$

$\epsilon_{ij} = \frac{1+\nu}{E} \sigma_{ij} - \frac{\nu}{E} \Theta \delta_{ij}$

Generalized Hooke's law

$$\begin{cases} \epsilon_x = \frac{1}{E}(\sigma_x - \nu(\sigma_y + \sigma_z)) \\ \epsilon_y = \frac{1}{E}(\sigma_y - \nu(\sigma_z + \sigma_x)) \\ \epsilon_z = \frac{1}{E}(\sigma_z - \nu(\sigma_x + \sigma_y)) \end{cases} \quad \begin{cases} \gamma_{xy} = \frac{1}{G} \tau_{xy} \\ \gamma_{yz} = \frac{1}{G} \tau_{yz} \\ \gamma_{zx} = \frac{1}{G} \tau_{zx} \end{cases}$$

$$\begin{cases} \epsilon_x = \frac{1}{E}((1+\nu)\sigma_x - \nu(\sigma_x + \sigma_y + \sigma_z)) \\ \epsilon_y = \frac{1}{E}((1+\nu)\sigma_y - \nu(\sigma_x + \sigma_y + \sigma_z)) \\ \epsilon_z = \frac{1}{E}((1+\nu)\sigma_z - \nu(\sigma_x + \sigma_y + \sigma_z)) \end{cases} \quad \begin{cases} \epsilon_{xy} = \frac{1+\nu}{E} \tau_{xy} \\ \epsilon_{yz} = \frac{1+\nu}{E} \tau_{yz} \\ \epsilon_{zx} = \frac{1+\nu}{E} \tau_{zx} \end{cases}$$

$$\epsilon_{ij} = \frac{1+\nu}{E} \tau_{ij} - \frac{\nu}{E} \delta_{ij} \Theta \quad \epsilon_{ij} = \frac{1}{2G} \left(\tau_{ij} - \lambda \delta_{ij} \frac{\tau_{kk}}{3\lambda + 2G} \right)$$

where $\Theta = \tau_{kk}$

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Generalized Hooke's law

$$\begin{cases} \sigma_x = 2G\epsilon_x + \lambda(\epsilon_x + \epsilon_y + \epsilon_z) \\ \sigma_y = 2G\epsilon_y + \lambda(\epsilon_x + \epsilon_y + \epsilon_z) \\ \sigma_z = 2G\epsilon_z + \lambda(\epsilon_x + \epsilon_y + \epsilon_z) \end{cases} \quad \begin{cases} \tau_{xy} = G\gamma_{xy} \\ \tau_{yz} = G\gamma_{yz} \\ \tau_{zx} = G\gamma_{zx} \end{cases}$$

$$\sigma_x = 2G\epsilon_x + \lambda \Sigma$$

$$\tau_{xy} = G\gamma_{xy} = 2G\epsilon_{xy}$$

$$\tau_{ij} = 2G\epsilon_{ij} + \lambda \epsilon \delta_{ij}$$

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Compatibility equations

$$\frac{\partial^2 \epsilon_x}{\partial y^2} + \frac{\partial^2 \epsilon_y}{\partial x^2} = \frac{\partial^2 \gamma_{xy}}{\partial x \partial y}$$
$$\frac{\partial^2 \epsilon_y}{\partial z^2} + \frac{\partial^2 \epsilon_z}{\partial y^2} = \frac{\partial^2 \gamma_{yz}}{\partial y \partial z}$$
$$\frac{\partial^2 \epsilon_z}{\partial x^2} + \frac{\partial^2 \epsilon_x}{\partial z^2} = \frac{\partial^2 \gamma_{zx}}{\partial z \partial x}$$

$$2 \frac{\partial^2 \epsilon_x}{\partial y \partial z} = \frac{\partial}{\partial x} \left(-\frac{\partial \gamma_{yz}}{\partial x} + \frac{\partial \gamma_{zx}}{\partial y} + \frac{\partial \gamma_{xy}}{\partial z} \right)$$
$$2 \frac{\partial^2 \epsilon_y}{\partial x \partial z} = \frac{\partial}{\partial y} \left(\frac{\partial \gamma_{yz}}{\partial x} - \frac{\partial \gamma_{zx}}{\partial y} + \frac{\partial \gamma_{xy}}{\partial z} \right)$$
$$2 \frac{\partial^2 \epsilon_z}{\partial x \partial y} = \frac{\partial}{\partial z} \left(\frac{\partial \gamma_{yz}}{\partial x} + \frac{\partial \gamma_{zx}}{\partial y} - \frac{\partial \gamma_{xy}}{\partial z} \right)$$

$$\epsilon_{ij,kl} - \epsilon_{jl,ik} + \epsilon_{jk,il} - \epsilon_{ik,jl} = 0$$
$$\epsilon_{il,jl} - \epsilon_{jl,il} + \epsilon_{jj,il} - \epsilon_{ij,jl} = 0$$

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Compatibility in terms of stress

$$\begin{cases} \nabla^2 \sigma_x + \frac{1}{1+\nu} \frac{\partial^2 \Theta}{\partial z^2} = -\left(\frac{\nu}{1-\nu}\right) \left(\frac{\partial f_x}{\partial x} + \frac{\partial f_y}{\partial y} + \frac{\partial f_z}{\partial z} \right) - 2 \frac{\partial f_z}{\partial z} \\ \nabla^2 \sigma_y + \frac{1}{1+\nu} \frac{\partial^2 \Theta}{\partial x^2} = -\left(\frac{\nu}{1-\nu}\right) \left(\frac{\partial f_x}{\partial x} + \frac{\partial f_y}{\partial y} + \frac{\partial f_z}{\partial z} \right) - 2 \frac{\partial f_x}{\partial x} \\ \nabla^2 \sigma_z + \frac{1}{1+\nu} \frac{\partial^2 \Theta}{\partial y^2} = -\left(\frac{\nu}{1-\nu}\right) \left(\frac{\partial f_x}{\partial x} + \frac{\partial f_y}{\partial y} + \frac{\partial f_z}{\partial z} \right) - 2 \frac{\partial f_y}{\partial y} \end{cases}$$
$$\begin{cases} \nabla^2 \tau_{xz} + \frac{1}{1+\nu} \frac{\partial^2 \Theta}{\partial y \partial z} = -\left(\frac{\partial f_x}{\partial z} + \frac{\partial f_z}{\partial y}\right) \\ \nabla^2 \tau_{yz} + \frac{1}{1+\nu} \frac{\partial^2 \Theta}{\partial x \partial z} = -\left(\frac{\partial f_y}{\partial z} + \frac{\partial f_z}{\partial x}\right) \\ \nabla^2 \tau_{xy} + \frac{1}{1+\nu} \frac{\partial^2 \Theta}{\partial x \partial y} = -\left(\frac{\partial f_x}{\partial y} + \frac{\partial f_y}{\partial x}\right) \end{cases}$$

Compatibility equations in terms of stress
Or Beltrami-Michell compatibility equations

$$\nabla^2 \tau_{ik} + \frac{1}{1+\nu} \Theta_{,ik} = -\frac{\nu}{1-\nu} \delta_{ik} f_{j,j} - (f_{i,k} + f_{k,i})$$

where $\Theta = \tau_{kk}$ Boundary: $T_i^\mu = \tau_{jk} \mu_j$

$\nabla^2 \tau_{ik} = \tau_{ik,jj}$

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Equilibrium equations in terms of displacement

★要記

$$\begin{cases} (\lambda + G) \frac{\partial \epsilon}{\partial x} + G \nabla^2 u + f_x = 0 \\ (\lambda + G) \frac{\partial \epsilon}{\partial y} + G \nabla^2 v + f_y = 0 \\ (\lambda + G) \frac{\partial \epsilon}{\partial z} + G \nabla^2 w + f_z = 0 \end{cases}$$

Navier's equations

$$(\lambda + G) \epsilon_{,i} + G \nabla^2 u_i + f_i = 0$$

where $\epsilon = \epsilon_{kk} = u_{k,k}$ Boundary: $u_i^0 = u_i^\delta$

$\nabla^2 \tau_{ik} = \tau_{ik,jj}$

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Strain energy density

$$U_0 = \frac{1}{2} (\sigma_x \epsilon_x + \sigma_y \epsilon_y + \sigma_z \epsilon_z + \tau_{xy} \gamma_{xy} + \tau_{yz} \gamma_{yz} + \tau_{zx} \gamma_{zx})$$
$$= \frac{1}{2} \text{vec} \begin{pmatrix} \sigma_x & \tau_{xy} & \tau_{xz} \\ \tau_{yx} & \sigma_y & \tau_{yz} \\ \tau_{zx} & \tau_{zy} & \sigma_z \end{pmatrix} \cdot \text{vec} \begin{pmatrix} \epsilon_x & \epsilon_{xy} & \epsilon_{xz} \\ \epsilon_{yx} & \epsilon_y & \epsilon_{yz} \\ \epsilon_{zx} & \epsilon_{zy} & \epsilon_z \end{pmatrix}$$

In terms of stress components:

$$U_0 = \frac{1}{2} \left(\frac{1}{E} (\sigma_x^2 + \sigma_y^2 + \sigma_z^2) - \frac{2\nu}{E} (\sigma_x \sigma_y + \sigma_x \sigma_z + \sigma_y \sigma_z) + \frac{1}{G} (\tau_{xy}^2 + \tau_{xz}^2 + \tau_{yz}^2) \right)$$

In terms of strain components:

$$U_0 = \frac{1}{2} (\lambda \epsilon^2 + 2G (\epsilon_x^2 + \epsilon_y^2 + \epsilon_z^2) + G (\gamma_{xy}^2 + \gamma_{yz}^2 + \gamma_{zx}^2))$$

↓

$$U_0 = \frac{1}{2} \tau_{ij} \epsilon_{ij}$$
$$U_0 = -\frac{\nu}{2E} \tau_{kk}^2 + \frac{1}{4G} \tau_{ij} \tau_{ij}$$
$$U_0 = \frac{\lambda}{2} \epsilon_{kk}^2 + G \epsilon_{ij} \epsilon_{ij}$$

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Solution of 3D problems

• Equilibrium equations (3) $\tau_{ij,j} + f_i = 0$

• Hooke's law (6) $\epsilon_{ij} = \frac{1+\nu}{E}\tau_{ij} - \frac{\nu}{E}\delta_{ij}\Theta$ $\tau_{ij} = 2G\epsilon_{ij} + \lambda\epsilon\delta_{ij}$

• Strain-displacement (6) $\epsilon_{ij} = \frac{1}{2}(u_{i,j} + u_{j,i})$

• Unknowns (15) $u_i, \tau_{ij}, \epsilon_{ij}$

Displacement formulation

$$\tau_{ij} = G(u_{i,j} + u_{j,i}) + \lambda\delta_{ij}u_{k,k}$$
$$\tau_{ij,j} + f_i = 0$$
$$u_i, \tau_{ij}$$

$$(\lambda + G)u_{j,j,i} + G\nabla^2 u_i + f_i = 0$$
$$u_i$$

Stress formulation

$$\tau_{ij,j} + f_i = 0$$
$$\epsilon_{i,j,i} - \epsilon_{j,i,j} + \epsilon_{j,i,i} - \epsilon_{i,j,j} = 0$$
$$\epsilon_{ij} = \frac{1+\nu}{E}\tau_{ij} - \frac{\nu}{E}\delta_{ij}\tau_{kk}$$
$$\tau_{ij}, \epsilon_{ij}$$

$$\tau_{ij,j} + f_i = 0$$
$$\nabla^2 \tau_{ik} + \frac{1}{1+\nu}\tau_{j,j,ik} = -\frac{\nu}{1-\nu}\delta_{ik}f_{j,j} - (f_{i,k} + f_{k,i})$$
$$\tau_{ij}$$

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Solution of plane strain problems

• Equilibrium equations (2) $\tau_{ij,j} + f_i = 0$

• Hooke's law (3) $\epsilon_{ij} = \frac{1+\nu}{E}\tau_{ij} - \frac{\nu}{E}\delta_{ij}\Theta$ $\tau_{ij} = 2G\epsilon_{ij} + \lambda\epsilon\delta_{ij}$

• Strain-displacement (3) $\epsilon_{ij} = \frac{1}{2}(u_{i,j} + u_{j,i})$

• Unknowns (8) $u_i, \tau_{ij}, \epsilon_{ij}$

Displacement formulation

$$\tau_{ij} = G(u_{i,j} + u_{j,i}) + \lambda\delta_{ij}u_{k,k}$$
$$\tau_{ij,j} + f_i = 0$$
$$u_i, \tau_{ij}$$

$$(\lambda + G)u_{j,j,i} + G\nabla^2 u_i + f_i = 0$$
$$u_i$$

Stress formulation

$$\tau_{ij,j} + f_i = 0$$
$$\epsilon_{ij} = \frac{1}{2G}(\tau_{ij} - \nu\delta_{ij}\tau_{kk})$$
$$\epsilon_{11,22} - \epsilon_{22,11} = 2\epsilon_{12,12}$$
$$\tau_{ij}, \epsilon_{ij}$$

$$\tau_{ij,j} + f_i = 0$$
$$\nabla^2 \tau_{ii} = \frac{1}{\nu-1}f_{i,i}$$
$$\tau_{ij}$$

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Solution of plane stress problems

• Equilibrium equations (2) $\tau_{ij,j} + f_i = 0$

• Hooke's law (3) $\epsilon_{ij} = \frac{1+\nu}{E}\tau_{ij} - \frac{\nu}{E}\delta_{ij}\Theta$ $\tau_{ij} = 2G\epsilon_{ij} + \frac{\nu E}{1-\nu^2}\epsilon\delta_{ij}$

• Strain-displacement (3) $\epsilon_{ij} = \frac{1}{2}(u_{i,j} + u_{j,i})$

• Unknowns (8) $u_i, \tau_{ij}, \epsilon_{ij}$

Displacement formulation

$$\tau_{ij} = G(u_{i,j} + u_{j,i}) + \frac{\nu E}{1-\nu^2}\delta_{ij}u_{k,k}$$
$$\tau_{ij,j} + f_i = 0$$
$$u_i, \tau_{ij}$$

$$\frac{E}{2(1-\nu)}u_{j,j,i} + G\nabla^2 u_i + f_i = 0$$
$$u_i$$

Stress formulation

$$\tau_{ij,j} + f_i = 0$$
$$\epsilon_{ij} = \frac{1+\nu}{E}\tau_{ij} - \frac{\nu}{E}\delta_{ij}\tau_{kk}$$
$$\epsilon_{11,22} - \epsilon_{22,11} = 2\epsilon_{12,12}$$
$$\tau_{ij}, \epsilon_{ij}$$

$$\tau_{ij,j} + f_i = 0$$
$$\nabla^2 \tau_{ii} = -(\nu+1)f_{i,i}$$
$$\tau_{ij}$$

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