

Laplace Transforms

General

Definition:  $F(s) = L[f(t)] = \int_0^\infty f(t)e^{-st} dt$

Attribute: Linear Attribute

Differentiation:

$L\left[\frac{df(t)}{dt}\right] = sF(s) - f(0)$

$L\left[\frac{d^nf(t)}{dt^n}\right] =$

$s^n F(s) - \left[s^{n-1}f(0) + s^{n-2}\frac{df(t)}{dt}\Big|_{t=0} + \dots + \frac{d^{n-1}f(t)}{dt^{n-1}}\Big|_{t=0}\right]$

Integration:  $L\left[\int_0^t f(\tau)d\tau\right] = \frac{F(s)}{s}$

Initial / Final Value:

$f(0) = \lim_{s \rightarrow \infty} sF(s)$

$\lim_{t \rightarrow \infty} f(t) = \lim_{s \rightarrow 0} sF(s)$

Convolution:

$f_1(t) \otimes f_2(t) = f_2(t) \otimes f_1(t) = \int_0^t f_1(t-\tau)f_2(\tau)d\tau$

$L[f_1(t) \otimes f_2(t)] = L[f_1(t)]L[f_2(t)] = F_1(s)F_2(s)$

Inverse Laplace Transform

Definition: [Not required]  $f(t) = L^{-1}[F(s)] = \frac{1}{2\pi j} \int_{c-j\infty}^{c+j\infty} F(s)e^{st} ds$

Feasible methods:

If  $F(s) = F_1(s) + F_2(s) + \dots + F_n(s)$ , then  $f(t) = f_1(t) + f_2(t) + \dots + f_n(t)$

If  $F(s) = \frac{a_0 + a_1s + \dots + a_ms^m}{\frac{c_1}{(s-\lambda_1)} + \frac{c_2}{(s-\lambda_2)} + \dots + \frac{c_n}{(s-\lambda_n)}}$ , then it can be break down to  $F(s) =$

$\frac{c_1}{(s-\lambda_1)} + \frac{c_2}{(s-\lambda_2)} + \dots + \frac{c_n}{(s-\lambda_n)}$ , where each item in this statement can be found in the Laplace table.

Using Laplace Transform to differential equation

- 1. Apply Laplace Transform to both sides on a differential equation.
- 2. Solve the Laplace equation.
- 3. Apply inverse Laplace Transform to the solution.

Transfer Function

Definition:  $G(s) = \frac{Y(s)}{U(s)}$ , where  $Y(s)$  is the Laplace Transform of the output  $y(t)$  and  $U(s)$  is that of the input  $u(t)$ .

Pole: Points that makes  $G(s)$  go towards  $\infty$ .

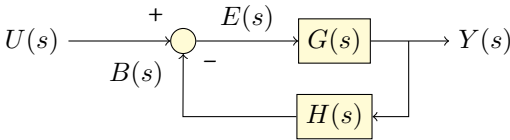
For  $\lim_{s \rightarrow -p} G(s)(s+p)^n = C \neq 0 \quad n = 1, 2, \dots$ , we call  $s = -p$  as the  $n^{th}$  rank pole.

For  $s = -r$  that makes  $G(s) = 0$ , it is a zero point.  $r$  can be  $\infty$ .

Steps to get the transfer function of a system:

- 1. Write down the differential equation of the system.
- 2. Assume all the beginning values are zero, apply Laplace Transform.
- 3. Get  $\frac{Y(s)}{U(s)}$ .

Close Ring System



Meaning of the figure:

$B(s) = H(s)Y(s)$

$Y(s) = E(s)G(s)$

$E(s) = U(s) - B(s)$

Every “block”(Amplifier)’s output is the **product** of itself and **its input**.

Forward Transfer Function:  $G_F(s) = \frac{Y(s)}{E(s)} = G(s)$

Open Ring Transfer Function:  $G_o(s) = \frac{B(s)}{E(s)} = G(s)H(s)$

Close Ring Transfer Function  $G_C(s) = \frac{Y(s)}{U(s)} = \frac{Y(s)}{E(s) + B(s)}$

State Equation

State variable:  $x_i(t), i = 1, 2, \dots, n$

State vector:  $X(t) = [x_1(t) \quad x_2(t) \quad \dots \quad x_n(t)]^T$

Standard description of a linear system:

Continuous:

$\begin{cases} \dot{x}(t) = A(t)x(t) + B(t)u(t) \\ y(t) = C(t)x(t) \end{cases}$

Discrete:

$\begin{cases} x(k+1) = A(k)x(k) + B(k)u(k) \\ y(k) = C(k)x(k) \end{cases}$

$A(t), B(t), C(t)$  or  $A(k), B(k), C(k)$  can be constant  $A, B, C$

Defining the State Equation

If the kinematic equation is  $y^{(n)} + a_{n-1}y^{(n-1)} + \dots + a_1\dot{y} + a_0y = u$

Let  $x_1(t) = y(t), x_2(t) = \dot{y}(t), \dots, x_n(t) = y^{(n-1)}(t)$

Then  $\begin{cases} \dot{x}_1(t) = x_2(t) \\ \dot{x}_2(t) = x_3(t) \\ \vdots \\ \dot{x}_{n-1}(t) = x_n(t) \\ \dot{x}_n(t) = -a_0x_1(t) - \dots - a_{n-1}x_n(t) + u(t) \end{cases}$

That is:

$\dot{x}(t) = \begin{bmatrix} 0 & 1 & 0 & \dots & 0 \\ 0 & 0 & 1 & 0 & 0 \\ \vdots & \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & 0 & \dots & 1 \\ -a_0 & -a_1 & -a_2 & \dots & -a_{n-1} \end{bmatrix} x(t) + \begin{bmatrix} 0 \\ 0 \\ \vdots \\ 0 \\ 1 \end{bmatrix} u(t)$

$y(t) = \begin{bmatrix} 1 & 0 & 0 & \dots & 0 \end{bmatrix} x(t)$

If the kinematic equation is  $y^{(n)} + a_{n-1}y^{(n-1)} + \dots + a_1\dot{y} + a_0y = b_mu^{(m)} + \dots + b_1\dot{u} + b_0u$

Apply Laplace Transform:  $G(s) = \frac{Y(s)}{U(s)} = \frac{b_ms^m + \dots + b_1s + b_0}{s^n + \dots + a_1s + a_0}$

Let  $Z(s)$  be an intermediate variable.

$\frac{Z(s)}{U(s)} = \frac{1}{s^n + \dots + a_1s + a_0}$

$\frac{Y(s)}{Z(s)} = b_ms^m + \dots + b_1s + b_0$

Eliminate the denominator and apply the Inverse Laplace Transform:

$z^{(n)} + a_{n-1}z^{(n-1)} + \dots + a_1\dot{z} + a_0z = u$

$y = b_mz^{(m)} + b_{m-1}z^{(m-1)} + \dots + b_1\dot{z} + b_0z$

[This can also be used to *memorize* the place of  $Z(s)$ :  $Z(s)$  should always combine with  $\triangle s^\triangle$  after simplify the equation]

Choose  $x_1(t) = z(t), x_2(t) = \dot{z}(t), \dots, x_n(t) = z^{(n-1)}(t)$  as the state variable.

Then

$\begin{cases} \dot{x}_1(t) = x_2(t) \\ \dot{x}_2(t) = x_3(t) \\ \vdots \\ \dot{x}_{n-1}(t) = x_n(t) \\ \dot{x}_n(t) = -a_0x_1(t) - \dots - a_{n-1}x_n(t) + u(t) \end{cases}$

$y(t) = b_0x_1(t) + \dots + b_mx_{m+1}(t)$

That is:

$\dot{x}(t) = \begin{bmatrix} 0 & 1 & 0 & \dots & 0 \\ 0 & 0 & 1 & 0 & 0 \\ \vdots & \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & 0 & \dots & 1 \\ -a_0 & -a_1 & -a_2 & \dots & -a_{n-1} \end{bmatrix} x(t) + \begin{bmatrix} 0 \\ 0 \\ \vdots \\ 0 \\ 1 \end{bmatrix} u(t)$

$y(t) = \begin{bmatrix} b_0 & \dots & b_m & 0 & \dots & 0 \end{bmatrix} x(t)$

The diagram shows two block representations. On the left, an 'Integrator' block is represented by a yellow triangle containing the integral symbol  $\int$ . The input is  $\dot{x}(t)$  and the output is  $x(t)$ . On the right, a 'Delay' block is represented by a yellow triangle containing the letter  $D$ . The input is  $x(k+1)$  and the output is  $x(k)$ . Below the diagrams, a note states: 'For adder and amplifier, refer to **Close Ring System**.'

If the coefficient is constant:

$$\begin{cases} \dot{x}(t) = Ax(t) + Bu(t) \\ y(t) = Cx(t) \end{cases}$$

Assume that the initial value of the system is zero, apply Laplace Transform.

$$sX(s) = AX(s) + BU(s)$$
$$Y(s) = CX(s)$$

Recall that  $G(s) = \frac{Y(s)}{U(s)}$ , we can get:

$$G(s) = C(Is - A)^{-1}B.$$

Firstly the original Transfer Function is:

$$G(s) = \frac{Y(s)}{U(s)} = \frac{b_m s^m + \dots + b_1 s + b_0}{s^n + \dots + a_1 s + a_0}$$

Let  $V(s)$  be an intermediate function.

$$\frac{Y(s)}{V(s)} = b_m s^m + \dots + b_1 s + b_0$$

$$\frac{V(s)}{U(s)} = \frac{1}{s^n + \dots + a_1 s + a_0}$$

[Resembling *Defining the state equation* part.]

Apply Inverse Laplace Transform,

let  $x_i(t) = v^{(i-1)}(t), i = 1, 2, \dots, n$ :

$$\begin{cases} \dot{x}_1(t) = x_2(t) \\ \dot{x}_2(t) = x_3(t) \\ \vdots \\ \dot{x}_{n-1}(t) = x_n(t) \\ \dot{x}_n(t) = -a_0 x_1(t) - \dots - a_{n-1} x_n(t) + u(t) \end{cases}$$

$$y(t) = b_0 x_1(t) + \dots + b_m x_{m+1}(t)$$

That is:

$$\dot{x}(t) = \begin{bmatrix} 0 & 1 & 0 & \cdots & 0 \\ 0 & 0 & 1 & 0 & 0 \\ \vdots & \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & 0 & \cdots & 1 \\ -a_0 & -a_1 & -a_2 & \cdots & -a_{n-1} \end{bmatrix} x(t) + \begin{bmatrix} 0 \\ 0 \\ \vdots \\ 0 \\ 1 \end{bmatrix} u(t)$$

$$y(t) = \begin{bmatrix} b_0 & \cdots & b_m & 0 & \cdots & 0 \end{bmatrix} x(t)$$
$$\begin{aligned} \text{Factorize } G(s) \\ G(s) &= \frac{b_0 + b_1 s + \dots + b_m s^m}{a_0 + a_1 s + \dots + s^n} \\ &= \frac{a_0 + a_1 s + \dots + a_m s^m}{(s - \lambda_1)(s - \lambda_2) \dots (s - \lambda_n)} \end{aligned}$$

If the denominator does not have multiple root:

$$G(s) = \frac{c_1}{(s - \lambda_1)} + \frac{c_2}{(s - \lambda_2)} + \dots + \frac{c_n}{(s - \lambda_n)}, \text{ where } c_i \text{ can be}$$

derived by fraction reduction.

Let:

$$X_1(s) = \frac{1}{(s - \lambda_1)} U(s)$$

$$X_2(s) = \frac{1}{(s - \lambda_2)} U(s)$$

$$\dots$$

$$X_n(s) = \frac{1}{(s - \lambda_n)} U(s)$$

$$Y(s) = c_1 X_1(s) + c_2 X_2(s) + \dots + c_n X_n(s)$$

Apply Inversed Laplace Transform,

$$\begin{aligned} \dot{x}_1(t) &= \lambda_1 x_1(t) + u(t) \\ \dot{x}_2(t) &= \lambda_2 x_2(t) + u(t) \\ &\dots \\ \dot{x}_n(t) &= \lambda_n x_n(t) + u(t) \\ y(t) &= c_1 x_1(t) + c_2 x_2(t) + \dots + c_n x_n(t) \end{aligned}$$

That is,

$$\begin{cases} \dot{x}(t) = \begin{bmatrix} \lambda_1 & \cdots & 0 \\ \vdots & \ddots & \vdots \\ 0 & \cdots & \lambda_n \end{bmatrix} x(t) + \begin{bmatrix} 1 \\ \vdots \\ 1 \end{bmatrix} u(t) \\ y(t) = [c_1 \ \cdots \ c_n] x(t) \end{cases}$$

Definition:  $\Phi(k, l) = A(k-1)A(k-2) \cdots A(l+1)A(l) \quad k > l$   
Attribute:  
 $x(k) = \Phi(k, l)x(l) \quad k > l$   
 $\Phi(l, l) = I$

Its state function is  $x(k+1) = A(k)x(k)$   
 Then  $x(k) = A(k-1)x(k-1) = A(k-1)A(k-2) \cdots A(1)A(0)x(0)$   
 Thus,  $\Phi(k, l) = A(k-1)A(k-2) \cdots A(l+1)A(l) \quad k > l$   
 Let  $x^1(k), \dots, x^n(k)$  represent  $n$  solutions to the state equation, which  
 ins that  $x^i(k+1) = A(k)x^i(k)$  holds true for all  $k$ .  
 If all  $x^i(k)$  are linearly independent,  
 $X(k) = [x^1(k) \quad x^2(k) \quad \cdots \quad x^n(k)]$  is called Fundamental Solu-  
 Matrix.

Assume  $x^i(0)$  are  $n$  solutions for the State Equation.

Iterate  $x^i(k+1) = A(k)x^i(k)$  to find the pattern (or expression) for  $x^i(k)$ .

Then,  $X(k) = \begin{bmatrix} x^1(k) & x^2(k) & \dots & x^n(k) \end{bmatrix}$ .

And  $X(0) = \begin{bmatrix} x^1(0) & x^2(0) & \dots & x^n(0) \end{bmatrix}$ .

$\Phi(k, 0) = X(k)X^{-1}(0)$  **[Formula]**

Usually, let  $X(0)$  be  $I$ , then  $X^{-1}(0)$  is  $I$  as well, which simplifies the calculation.

If  $A$  is constant:  $\Phi(k, l) = A^{k-l} \quad k \geq l$ ,  
so  $\Phi(k, 0) = A^k \quad k \geq 0$   
Methods of calculating  $A^k$ :

1.  $A^k = (I + B)^k = I^k + \binom{k}{1} I^{k-1} B + \binom{k}{2} I^{k-2} B^2 + \dots + B^k$ , where  $\binom{k}{i} = \frac{k!}{(k-i)!i!}$  and  $B^k$  is easy to obtain
2. Diagonalize  $A = PBP^{-1}$ , then  $A^k = PB^kP^{-1}$

Solution for  $x(k+1) = A(k)x(k) + B(k)u(k)$  is:

$$x(k) = \Phi(k, 0)x(0) + \sum_{l=0}^{k-1} \Phi(k, l+1)B(l)u(l)$$

To obtain  $\Phi(k, 0)$ , simply ignore  $B(k)u(k)$  and apply the methods in *Deriving  $\Phi(k, 0)$  from State Equation*.

Attribute:

$$\dot{x}(t) = A(t)\Phi(t, \tau)x(\tau)$$
$$\frac{d\Phi(t, \tau)}{dt} = A(t)\Phi(t, \tau)$$

$$^1u(k) = 0$$

$$\begin{aligned}\Phi(\tau, \tau) &= I \\ \Phi(t_2, t_0) &= \Phi(t_2, t_1) \Phi(t_1, t_0) \\ \Phi(t_1, t_0) &= \Phi^{-1}(t_0, t_1)\end{aligned}$$

### Linear Continuous Free System

Let  $x^1(k), \cdots, x^n(k)$  represent  $n$  solutions to the state equation, which means that

$$X(t) = \begin{bmatrix} x^1(t) & x^2(t) & \cdots & x^n(t) \end{bmatrix} \text{ satisfies } \dot{X}(t) = A(t)X(t)$$

$$\text{Then, } \Phi(t, \tau) = X(t)X^{-1}(\tau)$$

#### Constant A

It can be derived that

$$x(t) = \left( I + At + \frac{1}{2}A^2t^2 + \cdots + \frac{1}{k!}A^kt^k + \cdots \right) x(0) = e^{At}x(0)$$

$$\text{That is, } \Phi(t, 0) = e^{At}$$

$$\Phi(t, \tau) = e^{At}e^{-A\tau} = e^{A(t-\tau)}$$

#### Computing $e^{At}$

If  $A$  is diagonal:

$$A = \begin{bmatrix} a_1 & 0 & \cdots & 0 \\ 0 & a_2 & \cdots & 0 \\ \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & \cdots & a_n \end{bmatrix}$$

Then

$$e^A = \begin{bmatrix} e^{a_1} & 0 & \cdots & 0 \\ 0 & e^{a_2} & \cdots & 0 \\ \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & \cdots & e^{a_n} \end{bmatrix}$$

If  $A$  is diagonalizable:

$$A = P\Lambda P^{-1}, \text{ where } \Lambda \text{ is diagonal, then } e^A = Pe^{\Lambda}P^{-1}.$$

If  $A$  is not diagonalizable:

$$A = PJP^{-1}, \text{ where } J \text{ is the Jordan Standard Form.}$$

Then use the definition:

$$e^{At} = \left( I + At + \frac{1}{2}A^2t^2 + \cdots + \frac{1}{k!}A^kt^k + \cdots \right)$$

and the **taylor series formula**.

$$e^x = \sum_{n=0}^{\infty} \frac{x^n}{n!} = 1 + x + \frac{x^2}{2!} + \frac{x^3}{3!} + \cdots$$

About matrix diagonalization and Jordan Standard form, refer to other materials.

Or, recite this:

$$\text{For } A = \begin{pmatrix} \lambda & 1 & & \\ & \lambda & 1 & \\ & & \lambda & 1 \\ & & & \lambda \end{pmatrix}$$

$$A^k = \begin{pmatrix} \lambda^k & k\lambda^{k-1} & \frac{k(k-1)}{2}\lambda^{k-2} & \frac{k(k-1)(k-2)}{6}\lambda^{k-3} \\ & \lambda^k & k\lambda^{k-1} & \frac{k(k-1)}{2}\lambda^{k-2} \\ & & \lambda^k & k\lambda^{k-1} \\ & & & \lambda^k \end{pmatrix}$$

$$e^{At} = \begin{pmatrix} e^{\lambda t} & te^{\lambda t} & \frac{t^2}{2}e^{\lambda t} & \frac{t^3}{6}e^{\lambda t} \\ & e^{\lambda t} & te^{\lambda t} & \frac{t^2}{2}e^{\lambda t} \\ & & e^{\lambda t} & te^{\lambda t} \\ & & & e^{\lambda t} \end{pmatrix}$$

### Linear Continuous System with $B(t)u(t)$

$$\text{For } \dot{x}(t) = A(t)x(t) + B(t)u(t),$$

Solution is $x(t) = \Phi(t, 0)x(0) + \int_0^t \Phi(t, \tau)B(\tau)u(\tau)d\tau$
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#### Summary

$$\text{For } x(k+1) = Ax(k) + Bu(k), \Phi(k, l) = A^{K-l}$$

$$\text{For } \dot{x}(t) = Ax(t) + Bu(t), \Phi(t, \tau) = e^{A(t-\tau)}$$

$$\text{For } x(k+1) = Ax(k)$$

The eigenvalues and eigenvectors are correspondingly:

$$\lambda_1, \lambda_2, \dots, \lambda_n$$

$$e_1, e_2, \dots, e_n$$

★ If  $x(0) = \alpha e_i$  ( $\alpha$  here is a **scalar**), then

$$x(1) = Ax(0) = A\alpha e_i = \lambda_i\alpha e_i = \lambda_ix(0)$$

$$x(2) = Ax(1) = A\lambda_i\alpha e_i = \lambda_i^2\alpha e_i = \lambda_i^2x(0)$$

$$\cdots$$

$$x(k) = Ax(k-1) = A\lambda_i^{k-1}\alpha e_i = \lambda_i^k\alpha e_i = \lambda_i^kx(0)$$

[*Mind that the  $x(k)$  expressed like this are solutions to the differential function.*]

$$\text{Let a variable } z(k) \text{ satisfy } x(k) = z(k)e_i, \text{ then } z_i(k+1) = \lambda_iz_i(k).$$

#### Diagonalization of a Discrete System

As the  $e_i$  are linearly independent, any  $x(k)$  can be expressed as

$$x(k) = z_1(k)e_1 + z_2(k)e_2 + \cdots + z_n(k)e_n$$

$$\text{It satisfies } x(k+1) = z_1(k+1)e_1 + z_2(k+1)e_2 + \cdots + z_n(k+1)e_n$$

$$\text{Also, } z_i(k+1) = \lambda_iz_i(k).$$

$$\text{Let } x(k) = Mz(k) \text{ [Formula],}$$

$$\text{where } M = [e_1 \quad \cdots \quad e_n], z(k) = \begin{bmatrix} z_1(k) \\ \vdots \\ z_n(k) \end{bmatrix}$$

$$x(k+1) = Mz(k+1) = Ax(k) = AMz(k)$$

$$\Rightarrow z(k+1) = M^{-1}AMz(k) \text{ [Formula]}$$

$$\text{It's easy to proof: } M^{-1}AM = \Lambda, z(k+1) = \begin{bmatrix} \lambda_1 & \cdots & 0 \\ \vdots & \ddots & \vdots \\ 0 & \cdots & \lambda_n \end{bmatrix} z(k)$$

$$\text{Also, } \Phi(k, 0) = A^k = M\Lambda^kM^{-1} \text{ for constant } A.$$

#### Diagonalization of a Continuous System

$$\text{For } \dot{x}(t) = Ax(t)$$

The eigenvalues and eigenvectors are correspondingly:

$$\lambda_1, \lambda_2, \dots, \lambda_n$$

$$e_1, e_2, \dots, e_n$$

$$\text{Let } M = [e_1, e_2, \dots, e_n] \text{ and } x(t) = Mz(t)$$

$$\text{Then, } \dot{z}(t) = M^{-1}AMz(t)$$

$$\text{and } \dot{z}(t) = \begin{bmatrix} \lambda_1 & \cdots & 0 \\ \vdots & \ddots & \vdots \\ 0 & \cdots & \lambda_n \end{bmatrix} z(t)$$

$$\text{Also, } \Phi(t, 0) = e^{At} = Me^{\Lambda t}M^{-1}, \text{ and } e^{\Lambda t} = \begin{bmatrix} e^{\lambda_1 t} & \cdots & 0 \\ \vdots & \ddots & \vdots \\ 0 & \cdots & e^{\lambda_n t} \end{bmatrix}$$

#### Left Eigenvectors and Right Eigenvectors

$$\text{For } x(k+1) = Ax(k),$$

Vector  $f_i$  that satisfies  $f_j^T A = \lambda_j f_j^T$  are the left eigenvectors of  $A$ , denoted as  $f_1, f_2, \dots, f_n$ .

Attributes:

$$f_i^T e_j = 0 \quad \text{if} \quad i \neq j$$

$$f_i^T e_i = 1 \text{ (After it's standardized)}$$

$$f_i^T x(k) = z_i(k)$$

#### Multiple Eigenvalues

If there are multiple eigenvalues for  $A$ , then it cannot be diagonalize.  $A$  can be transformed into Jordan Normal Form.

#### Equilibrium Points

Definition, a vector  $\bar{x}$  that satisfies: once the system is  $\bar{x}$ , then the system will always be  $\bar{x}$ .

#### Free System

For both discrete and continuous free system,

$\bar{x} = 0$  is **always** a equilibrium point.

#### Eigenvectors of a System

**Discrete Free System**

$x(k + 1) = x(k) = \bar{x}$   
That is,  $\bar{x} = A\bar{x}$   
As  $Ae = \lambda e$ , so if any eigenvalue  $\lambda = 1$ , its corresponding eigenvector  $e$  is its equilibrium point.

**Discrete System**

For  $x(k + 1) = Ax(k) + b$ ,  
 $x(k + 1) = x(k) = Ax(k) + b \Rightarrow \bar{x} = A\bar{x} + b$   
That is,  $(I - A)\bar{x} = b, \bar{x} = (I - A)^{-1}b$   
If  $(I - A)^{-1}$  does not exist, there is no equilibrium.

**Continuous Free System**

For  $\dot{x}(t) = Ax(t)$ ,  
If a system is at its equilibrium, then its state does not change,  $\dot{x}(t) = 0$ .  
If  $x(t) = e_i$  and this  $e_i$ 's corresponding  $\lambda_i = 0$ ,  
then  $\lambda_i x(t) = Ax(t) = \dot{x}(t) = 0$ , which is an equilibrium.

**Continuous System**

For  $\dot{x}(t) = Ax(t) + b$   
If a system is at its equilibrium, then its state does not change,  $\dot{x}(t) = 0$ .  
 $0 = A\bar{x} + b \Rightarrow \bar{x} = A^{-1}b$   
If  $A^{-1}$  does not exist, there is no equilibrium.

**Stability**

Definition: For any initial status, the state vector approaches the equilibrium as the time goes. Then the equilibrium is asymptotically stable.

**Convert Controlled System to Free System**

$x(k + 1) - \bar{x} = A(x(k) - \bar{x})$   
Let  $z(k) = x(k) - \bar{x}, x(k + 1) - \bar{x} \Rightarrow z(k + 1) = Az(k)$   
Thus,  $x(k) \rightarrow \bar{x} \Leftrightarrow z(k) \rightarrow 0$ , controlled system's approaching the equilibrium is equivalent to a free system, with the same  $A$  matrix, approaching 0.

**Discrete System**

The equivalent condition of discrete system's asymptotically stability is all its  $A$ 's  $|\lambda_i| < 1$ . That is, the norm of all the eigenvalues are less than one, or all the eigenvalues are in the unit circle.

When the largest norm of all the eigenvectors is 1, if there is no multiple eigenvalue, it's called critical stable. If there are multiple eigenvalues, it's not stable.

**Continuous System**

The equivalent condition of continuous system's asymptotically stability is all its  $A$ 's  $\text{Re}(\lambda_i) < 0$ . That is, the real part of all the eigenvalues are less than zero, or all the eigenvalues are to the left of the y-axis on the complex plane.

When the largest real part of all eigenvectors is 0, if there is no multiple eigenvalue, it's called critical stable. If there are multiple eigenvalues, it's not stable.

**Summary**

Linear constant system is asymptotically stable if:  
Discrete System: All eigenvalues of  $A$  are in the unit circle.  
Continuous System: All eigenvalues of  $A$  are in the left side of the complex panel.

**Oscillation**

**Discrete System**

Oscillate if there is imaginary or negative eigenvalue.

**Continuous System**

Oscillate if there is imaginary eigenvalue.

**Dominant Mode**

Defining the major behavior of the system in the long run.

**Discrete System**

The dominant eigenvalue is the  $\lambda_1$  which has the largest norm.

$\lambda_1$  &  $e_1$  is called the dominant mode.

The speed that the system converge to the equilibrium point is decided by  $\lambda_1$ .

If  $\lambda_1$  is 1, the speed is decided by  $\lambda_2$ , whose norm is the second largest.

For discrete free system, there must be an eigenvalue = 1 (refer to "Equilibrium Points - Discrete Free System"), so its converging speed is decided by the second largest eigenvalue  $\lambda_2$ .

**Continuous System**

The dominant eigenvalue is the  $\lambda_1$  which has the largest real part.  
 $\lambda_1$  &  $e_1$  is called the dominant mode.

The speed that the system converge to the equilibrium point is decided by  $\lambda_1$ .

If  $\lambda_1$  is 0, the speed is decided by  $\lambda_2$ , whose real part is the second largest.

For continuous free system, there must be an eigenvalue = 0 (refer to "Equilibrium Points - Continuous Free System"), so its converging speed is decided by  $\lambda_2$ .

**Positive Linear System**

Definition: The linear system whose state variable are always positive (or at least non-negative).

It is related to the positive matrix.

**Positive Matrix**

Definition:  
Let  $A = [a_{ij}]$  be a matrix,  
• If  $\forall i, j, a_{ij} > 0$ ,  $A$  is strictly positive,  $A > 0$   
• If  $\forall i, j, a_{ij} \geq 0$  and  $\exists k, l, a_{kl} > 0$ ,  $A$  is strictly non-negative or positive,  $A \geq 0$   
• If  $\forall i, j, a_{ij} \geq 0$ ,  $A$  is non-negative,  $A \geq 0$

Define:  
•  $A \geq B \Leftrightarrow A - B \geq 0$   
•  $A > B \Leftrightarrow A - B > 0$   
•  $A \geq B \Leftrightarrow A - B \geq 0$

**Frobenius-Perrion theorem**

**Theorem 1**

If  $A > 0$ , then there exists such  $\lambda_0 > 0$  and  $X_0 > 0$ , which:  
•  $AX_0 = \lambda_0 X_0$   
• For any other eigenvalue  $\lambda \neq \lambda_0, |\lambda| < \lambda_0$ . That is,  $\lambda_0$  is the dominant eigenvalue  
•  $\lambda_0$  is a single eigenvalue (no duplication)

**Theorem 2**

If  $A \geq 0$  and there exists such  $m > 0$  that  $A^m > 0$ , then Theorem 1 holds true.

**Theorem 3**

If  $A \geq 0$ , there exists  $\lambda_0 \geq 0$  and  $X_0 \geq 0$  such that:  
•  $AX_0 = \lambda_0 X_0$   
• For any other eigenvalue  $\lambda \neq \lambda_0, |\lambda| \leq \lambda_0$   
Note that this theorem does not ensure  $\lambda_0$  to be the single eigenvalue.

**The  $\lambda_0$**

The above-mentioned  $\lambda_0$  is called the Frobenius-Perron eigenvalue.  
It can be estimated as follows:  
Let  $a_i$  be the sum of each row, then  $\min(a_i) \leq \lambda_0 \leq \max(a_i)$   
Let  $b_i$  be the sum of each column, then  $\min(b_i) \leq \lambda_0 \leq \max(b_i)$

**Positive Discrete Linear System**

**Free System**

For  $x(k + 1) = Ax(k)$ , if  $A > 0$  or  $A \geq 0$  and  $A^m > 0, m > 1$ .  
Because of the Frobenius-Perron Theorem, when  $k$  is large enough, the system will approach  $x(k) = \alpha \lambda_0^k x_0$ ,  $\alpha$  is some certain constant.

**Forced System**

For  $x(k + 1) = Ax(k) + b$ .

Lemma

If all the eigenvalue of positive matrix  $A$  are strictly in the unit circle,  $(I - A)^{-1} = I + A + A^2 + A^3 + \dots$

Theorem 1

If  $A \geq 0$  and its Frobenius-Perron eigenvalue is  $\lambda_0 \geq 0$ , then:  $(\lambda I - A)^{-1}$  exists and  $(\lambda I - A)^{-1} \geq 0 \Leftrightarrow \lambda > \lambda_0$

Theorem 2

Given  $A \geq 0, b > 0$ : All the eigenvalues of  $A$  are strictly in the unit circle  $\Leftrightarrow$  There exists a  $\bar{x} \geq 0$  such that  $\bar{x} = A\bar{x} + b$

It means that for a positive system, the existence of a non-negative equilibrium  $\Leftrightarrow$  the system is asymptotically stable.

Positive Continuous Linear System

Free System

If  $A$  is a Metzler Matrix, then the system is positive.

Metzler Matrix

Definition: All  $a_{ij} \geq 0, i \neq j$

Theorem 1

If  $A$  is a Metzler Matrix, then there exists a real number  $\mu_0$  and vector  $X_0 \geq 0$ , such that

- $Ax_0 = \mu_0 x_0$
- If  $\mu \neq \mu_0$  is any other eigenvalue of  $A$ ,  $\text{Re}(\mu) < \mu_0$

Forced System

For  $\dot{x}(t) = Ax + b$ :

Theorem 2

If  $A$  is a Metzler Matrix, then:  $-A^{-1}$  exists and is positive  $\Leftrightarrow$  All the eigenvalues of  $A$  are strictly in the left half of the complex panel.

Theorem 3

For a Metzler Matrix  $A$  and vector  $b > 0$ : All the eigenvalues of  $A$  are strictly in the left half of the complex panel  $\Leftrightarrow$  There exists a vector  $\bar{x} \geq 0$  that  $0 = A\bar{x} + b$

That is, the asymptotical stability  $\Leftrightarrow$  the existence of the non-negative equilibrium.

Routh Theorem

The system is stable  $\Leftrightarrow$  All the coefficient of the characteristic equation  $\geq 0$ . (The characteristic equation is  $|I_\lambda - A|$ )

Positive System Perturbation Analysis

Let  $\bar{x}, \bar{y}$  be the equilibrium point of the two positive system  $x(k+1) = Ax(k) + b$  and  $y(k+1) = \hat{A}y(k) + \hat{b}$ . The two systems are asymptotically stable, with  $\hat{A} \geq A$  and  $\hat{b} \geq b$ . Then,  $\bar{y} \geq \bar{x}$ .

Also, if  $\hat{a}_{ij} = a_{ij}, \hat{b}_i = b_i$  for all  $j$  and all  $i \neq r$  (which means that the only number changed is in the  $r^{th}$  row), then:

$$\frac{\bar{y}_r}{\bar{x}_r} \geq \frac{\bar{y}_i}{\bar{x}_i} \geq 1$$

# MID-TERM

Controllability

Continuous System

For a continuous system  $\dot{x}(t) = Ax(t) + Bu(t)$

Definition: for a certain state  $\bar{x}$  (not necessarily be an equilibrium), if for any  $T > 0$ , there exists such  $u(t) (0 < t \leq T)$  that can push  $x(0) = \bar{x}$  to  $x(T) = 0$ , then the system is controllable.

If every state is controllable, then the system is fully controllable. If there exists some state that is uncontrollable, then the system is uncontrollable.

Determining the Controllability

A system  $\dot{x}(t) = Ax(t) + Bu(t)$  is fully controllable

$$\Leftrightarrow P_c = \begin{bmatrix} B & AB & A^2B & \dots & A^{n-1}B \end{bmatrix}_{n \times nm}$$

has rank  $n$ .

Discrete System

For a discrete system  $x(k+1) = Ax(k) + Bu(k)$

Definition: if there exists a series  $u(k), u(k+1), \dots, u(N-1)$  that can force the system  $x(k)$  to  $x(N) = 0$  at the  $N^{th}$  step,  $x(k)$  is the controllable state. If for any  $k, x(k)$  is controllable, then the system is fully controllable.

For an  $n$  rank constant discrete system, if no  $u(0), \dots, u(n-1)$  can transform the system to zero at the  $n^{th}$  step, it cannot be transformed to zero in any steps.

Determining the Controllability

A system  $x(k+1) = Ax(k) + Bu(k)$  is fully controllable

$$\Leftrightarrow P_c = \begin{bmatrix} B & AB & \dots & A^{n-1}B \end{bmatrix}$$

has rank  $n$ .

Output Controllability

$$\dot{x}(t) = Ax(t) + Bu(t)$$

For a continuous system  $y(t) = cx(t)$

Definition: if there exists such  $u(t)$  can transform an initial output  $y(t_0)$  to any selected  $y(t_1)$  in a limited time  $(t_1 - t_0)$ , then the system is output controllable.

The equivalent condition is

$$P_{0c} = \begin{bmatrix} CB & CAB & CA^2B & \dots & CA^{n-1}B \end{bmatrix}$$

has rank  $r$ .

Observability

Continuous System

$$\dot{x}(t) = Ax(t) + Bu(t)$$

For a system  $y(t) = Cx(t)$

Definition: for any given input  $u(t)$ , there exists a certain time  $t_1 \geq t_0$ , when the input  $u(t)$  and observation  $y(t)$  can be well-determined. If any time  $t$  is observable, then the whole system is fully observable.

Determining the Observability

$$\dot{x}(t) = Ax(t) + Bu(t)$$

For a system  $y(t) = Cx(t)$ ,

the equivalent condition of its ob-

servability is that

$$P_0 = \begin{bmatrix} C \\ CA \\ \dots \\ CA^{n-1} \end{bmatrix}$$

$\text{Rank}(P_0) = n$

Dual System

$$\text{System} \begin{cases} \dot{x}(t) = Ax(t) + Bu(t) \\ y(t) = Cx(t) \end{cases} \quad \& \quad \begin{cases} z(t) = -A^T z(t) + C^T v(t) \\ w(t) = B^T z(t) \end{cases}$$

are dual systems.

If one of them is observable, then the other is controllable and vice versa.

Determining Controllability & Observability by Jordan Normal Form

$$\text{If a system} \begin{cases} \dot{x}(t) = Ax(t) + Bu(t) \\ y(t) = Cx(t) \end{cases}$$

can be transformed by  $x(t) =$

$Mz(t)$  into  $\begin{cases} \dot{z}(t) = M^{-1}AMz(t) + M^{-1}Bu(t) \\ y(t) = CMz(t) \end{cases}$ , which is a diagonal form or Jordan Normal Form.

Controllability

If  $M^{-1}AM$  is diagonal, then the system is controllable  $\Leftrightarrow$  All rows in  $M^{-1}B$  has at least one non-zero.

If  $M^{-1}AM$  is Jordan Normal Form, then the equivalent condition is that each LAST row of  $M^{-1}B$  corresponding to a Jordan block contains at least non-zero.

Observability

If  $M^{-1}AM$  is diagonal, then the system is controllable  $\Leftrightarrow$  All columns in  $CM$  has at least one non-zero.

If  $M^{-1}AM$  is Jordan Normal Form, then the equivalent condition is that each FIRST column of  $M^{-1}B$  corresponding to a Jordan block contains at least non-zero.

### Typical Decomposition of State Space

#### Decompose to controllable and uncontrollable state

If  $\dot{x}(t) = Ax + Bu(t)$  is not fully controllable.

If the rank of  $P_c$  is K, then a set of vectors  $P_1, P_2, \dots, P_k$  can be found in  $P_c$  who are linearly independent.

Then a set of extra vectors  $P_{k+1}, P_{k+2}, \dots, P_n$  can be found. They can form such  $M = [P_1 \dots P_n]$ . Mind that they should all be linearly independent and this  $M$  is not the same with the syntax  $M$  used before.

Let  $x = M\hat{x}$ , the system will be

$$\begin{bmatrix} \dot{\hat{x}}_1(t) \\ \dot{\hat{x}}_2(t) \end{bmatrix} = \begin{bmatrix} A_{11} & A_{12} \\ 0 & A_{22} \end{bmatrix} \begin{bmatrix} \hat{x}_1(t) \\ \hat{x}_2(t) \end{bmatrix} + \begin{bmatrix} B_1 \\ 0 \end{bmatrix} u(t)$$

$$y(t) = \begin{bmatrix} \hat{C}_1 & \hat{C}_2 \end{bmatrix} \begin{bmatrix} \hat{x}_1(t) \\ \hat{x}_2(t) \end{bmatrix}$$

#### Decompose to observable and unobservable state

The construction of  $M$  starts from  $P_0$  and is similar.  $x = M\hat{x}$  can transform

$$\begin{cases} \dot{x} = Ax + Bu \\ y = Cx \end{cases}$$

into

$$\begin{cases} \dot{\hat{x}} = \begin{bmatrix} A_{11} & A_{12} \\ 0 & A_{22} \end{bmatrix} \begin{bmatrix} \hat{x}_1 \\ \hat{x}_2 \end{bmatrix} + \begin{bmatrix} \hat{B}_1 \\ \hat{B}_2 \end{bmatrix} u \\ y = \begin{bmatrix} 0 & \hat{C}_2 \end{bmatrix} \begin{bmatrix} \hat{x}_1 \\ \hat{x}_2 \end{bmatrix} \end{cases}$$

### Transfer Function and Observability & Controllability

If a transfer function's pole point and zero point offset each other, then either it's uncontrollable or unobservable. If there is no such phenomenon, then the system is fully observable and controllable.

### Feedback Control

#### Output Feedback

$$\text{For the system } \begin{cases} x(t) = Ax(t) + Bu(t) \\ y(t) = Cx(t) \end{cases}$$

Let  $u(t) = Ly(t) + w(t)$ , where  $y(t)$  is the output. <sup>2</sup>

The system is thus

$$\begin{cases} \dot{x}(t) = Ax(t) + Bu(t) \\ = Ax(t) + BLy(t) + Bw(t) \\ = (A + BLC)x(t) + Bw(t) \\ y(t) = Cx(t) \end{cases}$$

$$\text{It can be written as } \begin{cases} \dot{x}(t) = \bar{A}x(t) + Bw(t) \\ y(t) = Cx(t) \end{cases}, \text{ where } \bar{A} = A + BLC$$

#### State Feedback

Let  $u(t) = Kx(t) + w(t)$ , then the system can be represented as

$$\begin{cases} \dot{x}(t) = (A + BK)x(t) + Bw(t) \\ y(t) = Cx(t) \end{cases}$$

#### Attributes

Neither state feedback not output feedback changes the controllability of the system.

Output state does not change the observability of the system. But the **state feedback** may affect observability.

### Pole Point Configuration

The system is fully controllable  $\Leftrightarrow$  linear system  $\dot{x}(t) = Ax(t) + Bu(t)$  can have such state feedback (not output feedback)  $u(t) = Kx(t) + w(t)$  can make the pole point of the system  $\begin{cases} \dot{x}(t) = \bar{A}x(t) + Bw(t) \\ \bar{A} = A + BK \end{cases}$  to be anywhere.

### Designation of K

#### State Feedback

First we have the expect pole point (eigenvalue)  $\lambda_1, \lambda_2, \dots, \lambda_n$ . Then the expected eigenpolynomial is  $f^*(\lambda) = (\lambda - \lambda_1)(\lambda - \lambda_2) \dots (\lambda - \lambda_n)$ .

The system with feedback  $\dot{x} = (A + bK)x + bw$  has a new matrix  $\bar{A} = (A + bK)$ . (Here  $K$  is a one-row- $n$ -column matrix  $(k_1 \ k_2 \ \dots \ k_n)$ , which satisfy that  $bK$  is  $n \times n$ , where  $b$  is  $n \times 1$ )

An eigenpolynomial can be extracted from  $f_k(\lambda) = |I_\lambda - \bar{A}|$ , let the parameters be in accordance with  $f^*$ , we can derive  $k_1, k_2, \dots, k_n$

#### Attribute

If a system with feedback is stable, all of its parameters of the eigenpolynomial should be greater than 0.

#### Output Feedback

The expected eigenvalue and eigenvectors are the same with the state feedback situation.

Consider feedback  $u(t) = Ly(t) + \omega(t)$ ,

$$\text{then system is } \begin{cases} \dot{x} = \bar{A}x + B\omega \\ y = Cx \end{cases}.$$

With  $\bar{A} = A + BLC$ , where L is a output-dimension-row-1-column matrix, making  $BLC$  be the same shape with  $A$ .

Then, we also calculate the eigenpolynomial and compare it with the expected one, getting the parameters  $l_i$ .

#### State Estimation

$$\text{If a system } \begin{cases} \dot{x}(t) = Ax(t) + Bu(t) \\ y(t) = Cx(t) \end{cases} \text{ is completely observable, then}$$

its state can be found by:

$$\dot{\hat{x}}(t) = (A + MC)\hat{x}(t) + Bu(t) - My(t), \hat{x}(0) = \hat{x}_0$$

$M$  is a n-row-one-column matrix which can make MC be n×n.

The pole point can be assigned, so the eigenpolynomial  $f^*(\lambda)$  can be calculated.

$|I_\lambda - (A + MC)|$  can also calculated. Let the parameters be the same, we can solve the  $[m_1 \ m_2 \ \dots \ m_n]^T$ .

### Separation Property

If the state feedback  $u(t) = Kx(t) + w(t)$  of a system

$$\begin{cases} \dot{x}(t) = Ax(t) + Bu(t) \\ y(t) = Cx(t) \end{cases} \text{ can be realized by an observer, then:}$$

$$\begin{cases} \begin{bmatrix} \dot{\hat{x}}(t) \\ \dot{\hat{x}}(t) \end{bmatrix} = \begin{bmatrix} A & 0 \\ -MC & A + MC \end{bmatrix} \begin{bmatrix} x(t) \\ \hat{x}(t) \end{bmatrix} + \begin{bmatrix} B \\ B \end{bmatrix} u(t) \\ y(t) = \begin{bmatrix} C & 0 \end{bmatrix} \begin{bmatrix} x(t) \\ \hat{x}(t) \end{bmatrix} \end{cases}$$

With  $u(t) = K\hat{x}(t) + w(t)$ :

$$\begin{cases} \begin{bmatrix} \dot{\hat{x}}(t) \\ \dot{\hat{x}}(t) \end{bmatrix} = \begin{bmatrix} A & BK \\ -MC & A + MC + BK \end{bmatrix} \begin{bmatrix} x(t) \\ \hat{x}(t) \end{bmatrix} + \begin{bmatrix} B \\ B \end{bmatrix} w(t) \\ y(t) = \begin{bmatrix} C & 0 \end{bmatrix} \begin{bmatrix} x(t) \\ \hat{x}(t) \end{bmatrix} \end{cases}$$

Using the error  $\tilde{x}$  to represent:

$$\begin{cases} \begin{bmatrix} \dot{\hat{x}} \\ \dot{\tilde{x}} \end{bmatrix} = \begin{bmatrix} A + BK & -BK \\ 0 & A + MC \end{bmatrix} \begin{bmatrix} x \\ \tilde{x} \end{bmatrix} w(t) \\ y = \begin{bmatrix} C & 0 \end{bmatrix} \begin{bmatrix} x \\ \tilde{x} \end{bmatrix} \end{cases}$$

### Optimal Control

General description of an optimal control problem:

The system moves along  $\dot{x}(t) = f(x(t), u(t))$ ,  $x(0) = x_0, 0 \leq t \leq T$

Select  $u(t)$  from the allowed set  $U$ ,

so that  $\max_{u \in U} J = \psi(x(T)) + \int_0^T L(x(t), u(t))dt$

### Maximum Principle

<sup>2</sup> $w(t)$  can represent the expected output  $y_r(t)$ , in that case  $u(t) = L(y(t) - y_r(t))$

Let  $u^*(t) \in U$  and  $x^*(t)$  is the solution for the optimal control problem.

Then such  $\lambda^*(t)$  exists such that  $\lambda^*(t), u^*(t), x^*(t)$  fulfills:

$$\text{The system is } \begin{cases} \dot{x}(t) = f(x^*(t), u^*(t)) \\ x^*(0) = x_0 \end{cases}$$

$$H(\lambda(t), x(t), u(t)) = \lambda^T(t)f(x(t), u(t)) + L(x(t), u(t)), \text{ where } \lambda^T(t)$$

is a one-row-n-column vector, which enables  $\lambda^T(t)Ax(t)$ .

$$\text{The adjoint differentiate equation is } \begin{cases} \dot{\lambda}(t) = -\frac{\partial H}{\partial x}(\lambda(t), x(t), u(t)) \\ \lambda^*(T) = \frac{\partial \psi}{\partial X(T)} \end{cases}$$

$$\text{For any other } t \in [0, T] \text{ and } u(t) \in U, H(\lambda^*(t), x^*(t), u^*(t)) \geq H(\lambda(t), x(t), u(t))$$

This can be achieved by, if  $u(t)$  is not constrained, let  $\frac{\partial H}{\partial u} = 0$ . Here  $x$  is independent with  $u$ , and  $\lambda$  is calculated and represented by  $x$  and  $u$  according to the adjoint differentiate equation.

## Dynamic Programming

### Quadratic Optimal Control

Definition:

For a discrete system,

$$J = \frac{1}{2}x^T(N)Sx(N) + \frac{1}{2}\sum_{k=0}^{N-1} [x^T(T)Q(k)x(k) + u^T(k)R(k)u(k)]$$

For a continuous system,

$$J = \frac{1}{2}x^T(T)Sx(T) + \frac{1}{2}\int_0^T [x^T(t)Q(t)x(t) + u^T(t)R(t)u(t)] dt$$

where we want to minimize  $J$  and matrix  $S, Q(t)$  are positive semi-definite matrix. Matrix  $R(t)$  is positive definite matrix.

### Solution

For a continuous system  $\dot{x} = Ax(t) + Bu(t), x(0) = x_0$ .

$$\min J = \frac{1}{2}\int_0^T (x^T(t)Qx(t) + u^T(t)Ru(t)) dt$$

We want to get the optimal control  $u^*(t) \quad 0 \leq t \leq T$ .

With Hamilton function  $H(\lambda(t), x(t), u(t))$

$$= -\lambda^T(Ax(t) + Bu(t)) - \frac{1}{2}(x^T(t)Qx(t) + u^T(t)Ru(t))$$

Adjoint differentiate equation is

$$\begin{cases} \dot{\lambda}(t) = -\frac{\partial H}{\partial x(t)} = Qx(t) - A^T\lambda(t) \\ \lambda(T) = 0 \end{cases}$$

To solve the  $u(t)$  which can maximize Hamilton function,

$$-\frac{\partial H}{\partial u(t)} = 0 = -Ru(t) + B^T\lambda(t) \Rightarrow u^*(t) = R^{-1}B^T\lambda(t)$$

Then we get (and must solve):

$$\begin{cases} \dot{x}(t) = Ax(t) + BR^{-1}B^T\lambda(t) & x(0) = x_0 \\ \dot{\lambda}(t) = -A^T\lambda(t) + Qx(t) & \lambda(T) = 0 \end{cases}$$

Assume such solution exists that  $\lambda(t) = -Y(t)x(t)$

where  $Y(t)$  can be obtained by

$$\begin{cases} -\dot{Y}(t) = Y(t)A + A^TY(t) - Y(t)BR^{-1}B^TY(t) + Q \\ Y(T) = 0 \end{cases}$$

With  $Y$  solved,  $\lambda$  can be represented by  $x$  with  $\lambda(t) = -Y(t)x(t)$  so  $u$

can be represented by  $x$  with  $u^*(t) = R^{-1}B^T\lambda(t)$ . Problem solved.

If  $t \rightarrow \infty, Y$  is constant and can be obtained by

$$\bar{Y}A + A^T\bar{Y} - \bar{Y}BR^{-1}B^T\bar{Y} + Q = 0.$$

## Non-Linear System

### Lyapunov Method I

Let the system  $\dot{x}(t) = f(x(t))$  has equilibrium point  $\bar{x}$ .

Let  $\Delta x = x - \bar{x}$ , then  $\Delta\dot{x}(t) = F\Delta x(t)$ , where  $F = \left. \frac{\partial f}{\partial x} \right|_{\bar{x}}$  which is the Jacobian Matrix.

Then the eigenvalue of  $F$  can be used to determine the asymptotic stability. That is, the system is asymptotically stable at this equilibrium point if:

Discrete System: All eigenvalues of  $F$  are in the unit circle.

Continuous System: All eigenvalues of  $F$  are in the left side of the complex panel.

### Lyapunov Method II

Definition:

$V(x)$  is a Lyapunov Function for discrete system at point  $\bar{X}$  if:

1.  $V(x(k))$  is continuous
2.  $V(x(k))$  has the minimum value at  $\bar{X}$
3.  $\Delta V(x) = V(f(x(k))) - V(x(k)) = V(x(k+1)) - V(x(k)) \leq 0$   
 $V(x)$  is a Lyapunov Function for continuous system at point  $\bar{x}$  if:
  1.  $V(x(k))$  is continuous and its partial derivatives are continuous
  2.  $V(x(k))$  has the minimum value at  $\bar{x}$
  3.  $\dot{V}(x) = \frac{\partial V}{\partial x}f(x) \leq 0$ , here  $\dot{V}$  is not differential

If there exists a Lyapunov Function, then the system must be stable. On the other hand, unabling to find one does NOT indicate the system is not stable.

If  $\dot{V}(x)$  or  $\Delta V(x(k))$  (for continuous and discrete system) are strictly negative ( $< 0$ ) except at  $\bar{x}$ , then the system is asymptotically stable.

If  $V(x)$  is defined at the whole state space and  $|x(t)| \rightarrow \infty \Rightarrow V(x(t)) \rightarrow \infty$ , then  $\bar{x}$  is wide-range asymptotically stable.